June 6, 1944. T. D. SEGEBERG 2,350,700

TOOL FOR CUTTING AND THREADING PIPES

Filed Feb. 9, 1943

Fig. 1.

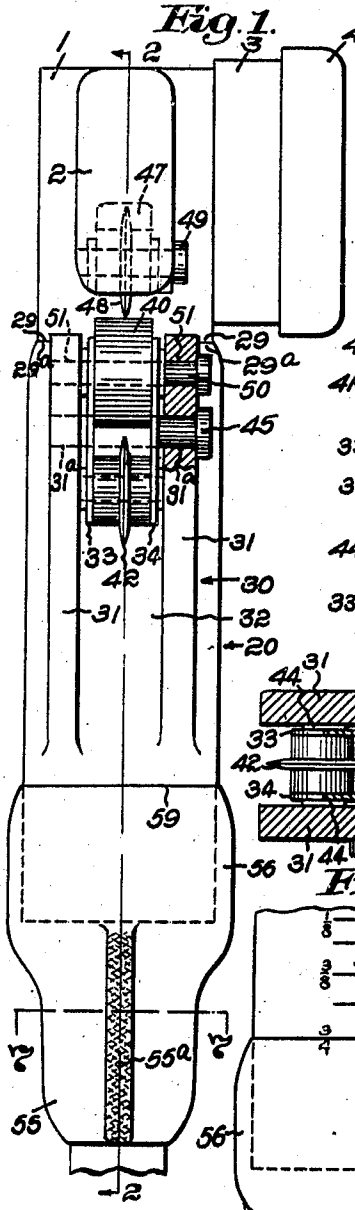


Fig. 2.

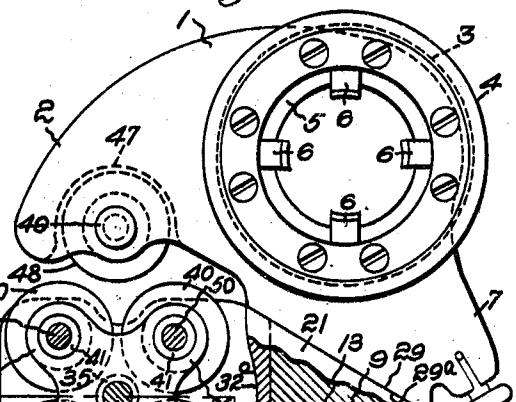


Fig. 3.

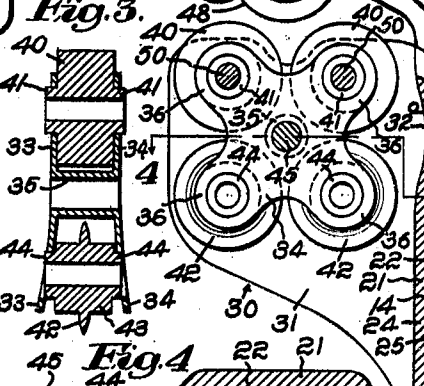


Fig. 4

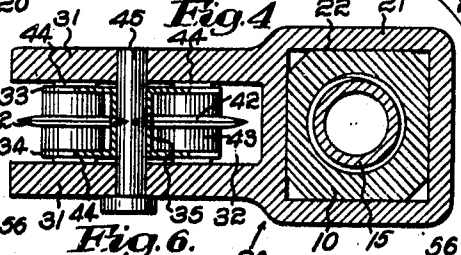


Fig. 6.

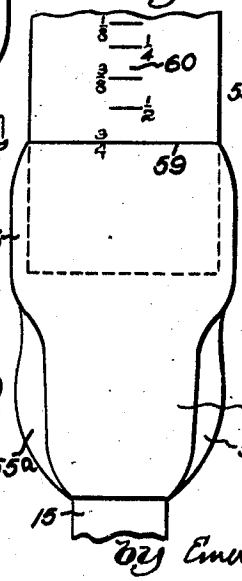


Fig. 5.

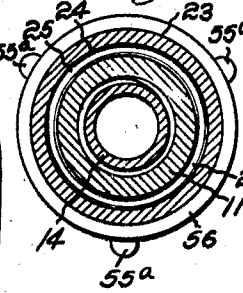
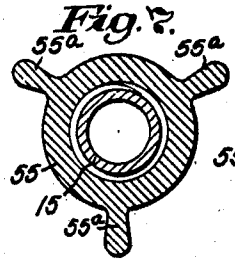


Fig. 7.



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TOOL FOR CUTTING AND THREADING PIPE

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Application February 9, 1943, Serial No. 475,253

11 Claims. (Cl. 30—102)

This invention relates to a tool for pipe-fitting and like purposes. More particularly it aims to provide a compound tool presenting in the one coordinated unit means for threading pipe and the like combined with pipe cutter means, the latter also in accordance with the invention involving simplified and otherwise improved means whereby the tool may readily be utilized either as a single-disk or as a plural-disk cutter selectively.

In the drawing illustrating by way of example one embodiment of the invention:

Fig. 1 is an elevation of the tool as a whole, viewing its disk cutter face or side;

Fig. 2 is a corresponding view, at right angles to Fig. 1, with parts in longitudinal or vertical section as on the line 2—2 of Fig. 1;

Fig. 3 is a detail sectional view of the disk and roll cage or holder;

Figs. 4 and 5 are cross-sections as on the lines 4—4 and 5—5 respectively of Fig. 2, looking down;

Fig. 6 is an elevation corresponding to a lower portion of Figs. 1 and 2, as viewed from the right of Fig. 2 or from the rear of Fig. 1; and

Fig. 7 is a cross-section on the line 7—7 of Fig. 1, also looking down.

Referring to the drawing in more detail, and first to Figs. 1 and 2, the tool comprises an end block, casting or stock 1 having a laterally perpendicular nose or disk-cutter mounting 2 and at right angles to the latter an annular seat 3 for a removable threading die head 4 rotatably carrying a threading die 5 preferably of the reversible ratchet type including a plurality of thread cutting elements 6. This main head or end member 1, when adapted for holding threading die means of the reversible ratchet type may also include a relatively short boss 7 disposed substantially radially of the die head and housing a reversible spring-pressed dog or pawl, not shown, adapted to be adjusted as by the releasable plunger 8, for setting the threading die ratchet for operating in one or the opposite direction. The threader seat is adapted to receive interchangeable die heads with dies of different sizes, as for example ⅛ in. to ¾ in. diameter for a given size of my tool, or 1 in. to 2 in. diameter for a larger tool size.

The block, main body or head element 1 further comprises a preferably integral elongated shank 9 including an upper squared portion 10 and a lower cylindrical portion 11 externally threaded at its bottom or outer end as at 12, Fig. 2. This shank 9 is provided with an internally threaded central longitudinal bore 13, the threading of which desirably is of a relatively low pitch. The shank 9 and its axial bore 13 desirably but not necessarily are so arranged relative to the axis of the die head 4 that the bore 13 is substantially radial of the latter.

The internally threaded shank 9 is adapted to receive the correspondingly threaded upper or inner end 14 of an elongated manipulating or handle member 15, illustrated as a tubular pipe-like element, the lower or handle portion proper of which is broken off on Figs. 1, 2 and 6 merely to conserve space upon the drawing, it being understood that this handle, bar or manipulating lever may be of any convenient length. The external threading along the upper or inner portion of this handle member 15 is of substantial extent downwardly from the extreme extent of the other, over a length in excess of that of the bore 13 of the shank 9, for a purpose shortly to be referred to.

The main body, block, stock, or head element 1 together with its shank 9 are herein specially adapted and arranged to serve also as the supporting and guiding means for a slidable frame or carrier indicated as a whole by the numeral 20. This frame may be stamped, cast, forged or otherwise formed from suitable metal or other material. It comprises a supporting and guiding portion 21 which is preferably of elongated tubular or channeled form adapting it to be received on and at least partially to embrace the shank 9 of the head or body 1. As herein shown the upper part of the bore or channel of this frame portion 21 is squared as at 22, Figs. 2 and 4, similarly as the upper part 10 of the shank 9 and is similarly proportioned as the latter, for guided sliding reception upon it. Below the squared portion 22, the outer or lower part 23 of the supporting frame portion 21 is cylindrical, as is also the corresponding extent of the bore or channel, as indicated at 24, Figs. 2 and 5.

Referring still mainly to Fig. 2, the cylindrical lower portion of the bore or channel 24 of the frame portion 21 is of somewhat larger diameter than the adjacent cylindrical portion 11 of the shank 9. There is thus provided between the two an elongated annular chamber 25 the upper or inner end of which is defined by a shoulder 26 on the frame portion 21 at the junction of the squared and cylindrical parts of its bore or channel. This chamber houses a fairly heavy coil spring 27 surrounding the cylindrical portion 11 of the shank 9 and bearing between said shoulder 26 and the end face of a cap or collar 28 threaded onto the threaded lower extremity 12, previ-

ously mentioned, of said shank portion 11. This cap 28 has an outer diameter adapting it for reception within the cylindrical channel 24 of the frame 21 without interference with the longitudinal sliding movement of the latter on and relative to the shank 9. At its lower end the cap or collar 28 is centrally apertured for free passage of the threaded portion 14 of the handle 15.

Thus it will be seen that the spring 27 normally urges the slidable carrier or frame 20 upwardly or inwardly, toward the outer end or head of the main body or stock 1. The extent of movement of the carrier or frame 20 in said direction is positively limited by lateral shoulders 29, 29 on the carrier 1 at opposite sides of the latter, see particularly Fig. 1, and which define the inner end of the squared portion of the shank 9, these shoulders preferably being inclined or non-perpendicular to the axis of the shank 9, substantially as illustrated in Fig. 2. The adjacent upper end faces of the portion 21 of the frame 20 are laterally disposed along a similar line of inclination, as at 29*a*, 29*a*, Figs. 1 and 2, paralleling the shoulders 29 of the carrier and against which, under the influence of the spring 27, they normally have accurate abutting contact. The inclination of the shoulders 29 and the end faces 29*a* of the frame portion 21 afford somewhat of a lateral camming action between the two whereby side-play between the relatively slidable parts is taken up.

In addition to the described supporting and guiding portion 21 of the carrier or frame 20, the latter is provided, herein integrally, with a laterally projecting upper portion indicated as a whole by the numeral 30. This projecting portion of the frame 20 is recessed or bifurcated to provide a pair of spaced wall members or flanges 31, 31, Figs. 1, 2 and 4, spaced apart by a recess or chamber 32. Within this chamber is mounted holding means for a plurality of rotary elements for pipe-cutting purposes. Said means as herein illustrated comprises a holder or cage including a pair of spaced plates 33, 34, centrally united by a hollow tubular connection or sleeve 35, see particularly Fig. 3. These holder plates desirably are of a somewhat resilient sheet material, enabling them at times to be sprung outwardly in a manner as indicated at the lower portion of Fig. 3, for the purposes of removing and installing or replacing the rotary elements to be referred to, one suitable material being a hardened sheet steel of fairly light gauge. The holder plates 33, 34 may be variously shaped and as herein shown are quatrefoil, presenting four arms as indicated at 36, 36, etc. Each of these arms is centrally apertured to provide rotary bearing support for the rotary elements to be described.

Said rotary elements herein removably associated with the described spider-like cage or holder comprise a pair of rollers 40 having centrally apertured projecting hubs 41 adapted for freely rotatable reception in the bearing recesses of two adjacent arms 36 of the cage. In the position of the latter as illustrated in Figs. 1 to 3, this pair of rollers 40 is uppermost. The other two arms 36 of the cage are adapted similarly to receive a pair of rotary disk cutter elements each comprising a disk or cutter portion proper 42, a cylindrical centrally apertured body 43 and their hubs or bearings 44. By reference particularly to Fig. 3, it will be seen that these rotary cutter elements 43, and similarly also the rollers 40, may readily be removed or inserted with reference to the supporting plates 34 of the cage or holder, merely by springing the plate arms slightly apart, after demounting the collar unit from the tool, while in the assembled condition of the cage or holder unit the plurality of rotary elements including the adjoining pair of rollers 40 and the symmetrically disposed pair of disk cutters 42 are each freely rotatable upon their respective axes.

A further advantage in the conjoint single or plural-disk cutter mechanism, and particularly under the provision for ready removal of any of the cutter disks, is that in the case of breakage of the cutter of the stock or head 1 under circumstances where an extra replacement disk is not immediately at hand, the operator may temporarily rob one cutter from the holder and install it upon the carrier head 1. Thus he is assured of having available at all times at least a single-disk cutter arrangement. Such procedure may be repeated, by utilizing a second cutter from the rotatively adjustable cage or holder, in case of a further breakage of the cutter of the single-disk position.

The described cage or holder unit as a whole is removably mounted in the chamber or recess 32 of the carrier or frame 20, in a manner permitting the holder at desired times to be rotated about the axis of its central sleeve 35. For so mounting the holder, the lateral flanges 31 of the slidable frame 20 are provided with aligned apertures 31*a* for the reception of a removable pivot pin 45, Figs. 1, 2 and 4, which is adapted to pass through and rotatably receive the central holder sleeve 35.

Referring again to the head or stock element 1 of the tool as a whole, the lateral nose 2 of the latter, previously referred to, has formed in its underface a recess 47 for the reception of a rotary disk cutter 48 which may be identical or substantially so to the rotary cutter elements 42 of the described holder, the walls of the recess 47 being themselves recessed at their inner opposite faces to accommodate the cutter hub such as 44 of Fig. 3. Said walls also are apertured in lateral alignment to receive a removable pivot pin 49 of a length to extend through said walls and through the central aperture of the disk received between them thereby removably to support the cutter 48 rotatably in operative position.

By reference particularly to Fig. 2, it will be noted that the axis of the sleeve bearing 35 of the roll and cutter cage is so disposed relative to the axis of the upper cutter disk 48 and with the roll and cutter holder and its supporting frame 20 in their upper or innermost position as represented in Figs. 1 and 2, the axis of said upper cutter 48 and the axes of the two rotary elements then uppermost in the holder are disposed at the points of an isosceles and herein a substantially equilateral triangle. Accordingly the two rotary elements of the pair 40 or the pair 42, whichever is disposed in uppermost position at the given time, are equally spaced from and symmetrically disposed relative to the singular or upper cutter disk 48, thus adapting them for cooperative cutting effort upon a pipe or the like inserted centrally between them.

From the foregoing it will be seen that I have herein provided selectively and rotatively adjustable roll and disk cutter means for my pipe-fitting tool whereby one and the same tool, at the option of the user, may serve alternately as a single disk cutter operating in opposition to a

pair of rolls such as 40, or a plural and herein three-disk cutter wherein the previous single disk and the pair of cutter disks of the adjustable holder are the active cutting means. It also will be particularly noted that the recess 32 of the slidable frame or carrier element of the tool is proportioned and arranged to accommodate the rotary adjusting movement of the cage or holder unit, the inner wall of said recess being arcuately grooved as for example at 32a, Fig. 2, if necessary for clearance.

In Figs. 1 and 2, and correspondingly in Fig. 4, the cutter disk and roll holder is illustrated in its adjusted position for operation of the tool as a single disk cutter. The pair of rolls 40 accordingly are uppermost. For retaining the holder in the selected position of adjustment suitable lock means is provided, herein comprising a pair of self-locking pins or stops 50, 50, Figs. 1 and 2. These are removably inserted through appropriately disposed aligned receiving apertures 51, 51, Fig. 1, in the respective flanges 31 of the slidable carrier or frame 20. With these lock pins 50 installed, the roll and disk holder is firmly held against bodily turning about the axis of its bearing sleeve 35. The single disk and opposed roll pair arrangement is generally preferable for use under circumstances where the tool as a whole may be swung completely about the pipe or bar to be cut.

Under other circumstances, such as for pipes located in corners, ditches or other confined speces, where full revolution of the tool about the work is impossible, a plural or three-disk cutter arrangement is a substantial necessity, so that the tool may be reciprocated through a relatively small angle but still perform a complete cut. Heretofore a separate tool has been required for this purpose. Under my present invention, however, such plural-disk cutter means is readily available at all times in one and the same tool, and preferably also and as herein illustrated and described the pipe threading means as presented by the rotary ratchet threading die head cooperatively embodied in combination with the common carrier and manipulating handle element.

To shift as between the single disk and roll combination of Figs. 1, 2 and 4, and the three-disk arrangement, it is necessary merely to withdraw the lock pins 50 and to rotate the cage or holder bodily upon the axis of its sleeve 35 until the pair of disk cutters 42 are turned up into the uppermost position, adjacent the single disk 46, with their central apertures respectively in line with the locking apertures 51 of the flanges 31. The lock pins 50, of which in some instances but one need be employed, are then reinserted, whereupon the tool is immediately ready for operation as a three-disk cutter.

With an exterior cutter such as here concerned, it will be understood that in order to bring the tool into operative position relative to the pipe or other work, it is first necessary to open or separate the rotary elements, whether these at the given time consist of the single cutter and opposed rolls or a plurality of cutters, so as to bring them into surrounding relation with the work and to enter the latter into the space defined between the adjacent portions of the peripheries of the several rotary elements. Customarily, in the usual single disk cutting tool and also in the separate tool having a plurality of cutters, it is necessary to back off a threaded member which carries one or more of the rotary elements, to provide such opening for the pipe, and then again to turn back the threaded member so as to close the rotary elements onto the work. This generally involves more or less trial and error and is cumbersome and time-consuming. In accordance with my invention, however, it will be seen that such operation of the rotary elements, whether these at the given time are the single disk and roll pair or the plurality of roll disks, is easily and quickly accomplished merely by slidably retracting the entire body or frame 20 together with the roll and disk holder carried by it, against the pressure of the spring 27, sufficiently to afford entrance for the work, upon subsequent release of the frame 20 the spring immediately forces the latter back or up into abutting engagement of the shoulders 29 of the carrier and the end faces 29a of the holder frame. It will further be observed that said shoulders or abutment formations are so calculated and arranged relative to the location of the roll and disk holder and of the single disk cutter upon the head or stock that upon such automatic return of the frame the selected rotary elements are accurately inter-positioned in cooperative contacting relation with the work.

In this connection, a further important advantage of my improved tool resides in its capacity for provision of indicating means whereby the pipe-cutting elements of the tool may be definitely set for operation upon pipe or other cylindrical work of a given diameter, such for example as ⅛, ¼, ⅜, ½, ⅝, ¾ in., etc. (inside diameter). Thus when a number of cutting operations are to be performed upon the same size of work the tool may be set to the given size, as for example ⅛ in. pipe (inside diameter) as assumed in Figs. 1 and 2 or ¾ in. as assumed in Fig. 6. Or the setting may readily otherwise be adjusted at any time, thus facilitating quick shifting as between cutting operations upon work of different sizes.

Referring now more particularly to Figs. 1, 2 and 6, means is provided for variably determining and limiting the extent of downward or outward movement of the roll and disk frame 20 against the compression of the spring 27. For this purpose I have herein provided a nut-like fitting or cupped stop and finger piece 55 fixedly associated with the manipulating element or handle 15. As illustrated this fitting 55 has a central aperture which is internally threaded in conformity with the threaded portion 14 of the handle. In assembling the tool, the fitting 55 is first installed upon the handle 15, prior to threading of the latter into the shank 9. The fitting is turned completely home to the extreme lower end of the threaded portion 14 so as to fix it securely upon the latter.

This fitting 55 comprises a centrally recessed or cupped hub end 56 the recess 57 of which is cylindrical and of a diameter to receive with a guided sliding fit the lower cylindrical portion 23 of the frame 20. The bottom wall 58 of said recess 57 constitutes an abutment stop for the lower end of the frame portion 23, thereby limiting and positively defining the extent to which the frame with its disk and roll holder may be retracted.

A setting index desirably is provided as between the fitting and the frame 20, for setting the tool in the above described manner for the different sizes of work within the capacity of the given tool. Herein for the purpose, I employ the upper edge 59 of the fitting 55, Fig. 6, as an index

or pointer for cooperation with a scale **60** stamped or otherwise marked at any convenient lower portion of the frame member **23**, as for example at the right in Fig. 2, which area is seen foremost in the position of Fig. 6. In the latter figure a setting is represented as for cutting work of ¾ in. size. To effect such setting the handle member **15** is turned in and relative to its supporting shank **9**, the frame **20** remaining in its lowermost position as in Fig. 2, until the desired unit or marking of the scale **60** is brought opposite the index line **59**. In Figs. 1 and 2 the tool is represented as set for one of the smaller work sizes, the frame **20** being disposed for downwardly opening retraction only by the extent represented by the distance between the lower end of the frame member **23** and the opposite bottom wall **58** of the fitting **55**. In proceeding from such setting of Figs. 1 and 2 to that as shown by way of example in Fig. 6, the handle member **15** accordingly is backed out from the shank **9** until the marker or edge **59** of the fitting **55** is brought down to expose or come opposite the ¾ mark of the scale **60**. It will be understood the tool may be made in different sizes or with different ranges for a given tool, such for example as a junior size having a range of say ⅛ to ¾ inch substantially as illustrated, and other sizes for work ranging through the larger diameters. Under any given adjusted setting, such as that at the ¾ mark as represented in Fig. 6, the pipe cutter element of the tool, whether the disk and roll or the plural disk arrangement is to be used at the time, is automatically readied for a cutting operation upon a pipe or like work of the diameter indicated. That is, under the ¾ setting the roll and disk holder frame **20** may be retracted to the extent necessary to open the rotary elements for ready admission of the work of ¾ in. size, and upon release of the retracted frame the latter is automatically returned by the spring so as to close the rotary elements into proper operative contact with the work.

For additional ease in retracting the frame **20** against the spring, it may be provided with finger or hand engaging means, such for example as the protuberance or raised surface **20***a*, Fig. 2. Also manipulation of the handle **15** and fitting **55** fixed thereon may be facilitated, as when making a size setting as above described, as by the provision of one or more longitudinal wing-like formations or ribs **55***a*, of which three are herein provided symmetrically distributed around the fitting; see particularly Fig. 7. These ribs may have knurled or roughened outer surfaces for gripping purposes, as seen at the lower portion of Fig. 1.

Various other advantages of the compound tool of the invention will be evident in addition to those of convenience and time-saving through accomplishing the several different operations with the one tool. These include, for example, the substantial savings in materials, parts and manufacturing cost through the provision for selective adjustment as between single disk and roll, and plural cutter disk arrangement in association with a common supporting body, and through the utilization of a common supporting head stock or body and handle element for both the single disk cutter and the threading die and wherein the shank of such head stock serves at once for the attachment of the one handle element and for the guided slidable support of the plural disk and roll holder frame for cooperation with the single disk cutter of the conjoint disk-cutter and threading-die supporting means or head stock.

My invention is not limited to the particular embodiment thereof as herein illustrated or described, its scope being pointed out in the following claims:

1. In a pipe cutter, in combination, a cutter-disk head or body element and a frame element, one element having an elongated shank and the other a channeled connecting portion adapted to receive the shank and whereby said elements are interengaged for relative sliding movement toward and from each other lengthwise of the shank, said body element having a cutter disk rotatably mounted thereon, said frame element holding a plurality of individually rotatable members symmetrically disposed in opposition to the cutter disk, interengageable means on said elements limiting movement of the frame element toward and from the cutter-disk, and means yieldably urging the frame element toward the cutter-disk.

2. In a pipe cutter, in combination, a head element having an elongated shank, a cutter disk rotatably mounted on said head element in laterally offset relation to the shank thereof, a cage-like holder for a plurality of individually rotatable members cooperable with said cutter disk, a frame supporting said holder for bodily rotary adjustment relative to and while on the frame thereby to present selected rotatable members thereof in opposition to said cutter disk, and means movably interconnecting said frame and the shank of the head element for positioning the selected rotatable members of the frame-supported holder in positions of greater or less spacing relative to the cutter-disk as appropriate respectively to admit and to operate on work of a given size.

3. In a pipe-cutting tool, in combination, an elongated handle, a carrier head at one end of the handle said head including an attaching shank for the handle and a cutter disk rotatably mounted on the carrier head in laterally offset relation to the shank thereof, a frame mounted on said shank for movement longitudinally thereof, said frame having a lateral portion disposed opposite the cutter-disk, a holder carrying a plurality of individually rotary members for selective cooperation with the cutter-disk, means rotatably supporting the holder on the lateral portion of the frame, said frame, holder and its supporting means constructed and arranged for bodily rotative adjustment of the holder relative to and while on the frame selectively between positions presenting different rotary members in operative relation to the cutter-disk, and means for releasably fixing the holder in the selected position of rotative adjustment thereof.

4. In a pipe-cutter, bodily rotatable holder means for a plurality of individually rotary elements including at least one disk cutter and at least one roller, said holder means comprising a pair of spaced plates, a bearing sleeve disposed centrally of and interconnecting the plates, aligned bearing apertures in the plates at symmetrically disposed locations concentric of the plate sleeve, and a plurality of said rotary elements between the plates and having opposed hubs adapted for reception in the respective aligned apertures thereof.

5. In a pipe-fitting tool, in combination, a main body, block or head element having a longitudinal shank and a lateral nose, a rotary cutter disk on the nose, a frame movable lengthwise of the

shank and having a lateral portion opposite said nose, a plurality of rotary members on the lateral frame portion for cooperation with the cutter disk, a manipulating handle connected to the shank in axial extension therefrom, abutment means limiting movement of the frame toward the cutter disk to determine the maximum approach of its rotary members thereto, and spring means urging the frame normally into such maximum approach position, said frame having an opposite retracting movement against the spring means adequate for separation of the rotary members and cutter disk to admit the work between them.

6. In a pipe-fitting tool, in combination, a main body, block or head element having a longitudinal shank and a lateral nose, a rotary cutter disk on the nose, a frame movable lengthwise of the shank and having a lateral portion opposite the cutter disk, a plurality of rotary members on the lateral frame portion for cooperation with the cutter disk, abutment means on the frame and head element limiting approach of the frame and its rotary members toward the cutter-disk, a handle longitudinally adjustably connected to the shank, stop means on the handle engageable with the frame and longitudinally adjustable with the handle so as correspondingly variably to limit movement of the frame and its rotary members away from the cutter disk thereby to set the rotary members with respect to the latter to accommodate different work sizes, and spring means urging the frame and its rotary members toward the cutter disk.

7. In a pipe-fitting tool, in combination, a body or stock having a lateral jaw-like portion, a rotatable cutter-disk on the jaw-like portion, a pair of rotary members for cooperation with the cutter-disk, the body or stock having movably associated with it carrier means for said members presenting them operatively opposite the cutter-disk, said body or stock having a longitudinal threaded portion laterally offset from and substantially paralleling a line radial of the cutter-disk and passing between the rotary members, a handle threaded at one end for connection with said threaded body or stock portion, and means limiting movement of the carrier means in opposite directions relative to the cutter-disk.

8. A compound tool for pipe-fitting purposes, comprising, in combination, a head having a shank and a lateral portion operatively supporting a rotary disk-cutter, a handle connected to the shank, a frame slidably received on the shank, a plurality of rotary elements laterally disposed on the frame for cooperation with the disk-cutter, stop means limiting movement of the frame both toward and from the disk-cutter, and yieldable means urging the frame and its rotary elements toward said disk-cutter.

9. A compound tool for pipe-fitting purposes, comprising, in combination, a head having a shank and a lateral portion operatively supporting a rotary disk-cutter, a handle threadedly connected to the shank for longitudinal adjustment relative to it, a frame slidably received on the shank, a plurality of rotary elements laterally disposed on the frame for cooperation with the disk-cutter, stop means on the handle for variably limiting movement of the frame away from the disk-cutter as determined by adjustment of the handle relative to the stock, abutment means on the stock and frame defining the maximum approach of the frame and its rotary elements toward the disk-cutter, and spring means urging the frame in said approach direction and yieldable for retraction of the rotary-elements for entering work of different sizes between them and the disk-cutter in accordance with the adjusted position of the handle.

10. In a compound tool for pipe-fitting purposes, a stock in the general form of a standard thread-cutting die stock including a rounded head portion and a substantially radially extended shank and handle portion, a cutter disk and supporting means therefor laterally disposed on said head portion of the stock in angularly projecting relation to the axis of the shank and handle portion and having an arcuate outer portion generally conformant to the rounded head portion, a frame movable lengthwise of the shank and handle portion of the stock toward and from the head portion, a plurality of rotary elements laterally disposed on the frame for cooperation with the cutter disk, and means for adjustably positioning the frame relative to the stock to present said rotary elements in selected spaced relation operatively adjacent the cutter disk and with capacity for retraction of the frame for entry of the work between said rotary elements and the cutter disk.

11. In a pipe-fitting tool, a stock including a shank and carrying a rotary cutter disk laterally offset from the shank, a holder for a plurality of individually rotary members for cooperation selectively with the cutter disk, and a support for the holder mounted on the stock and including means positioning the holder for bodily rotary adjustment relative to and while on the support, thereby to present selected rotary members of the holder in opposition to the cutter disk.

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