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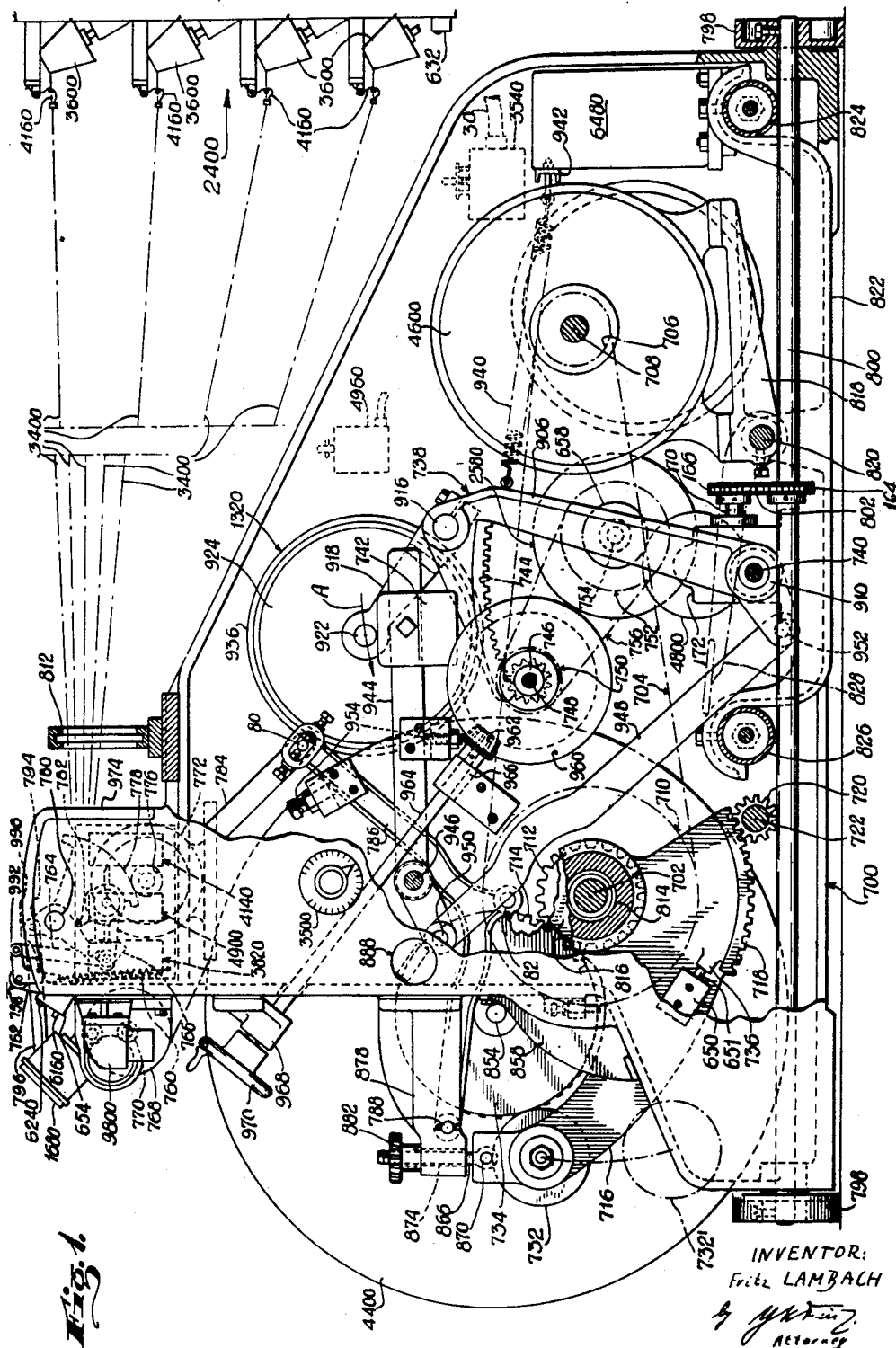
F. LAMBACH

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BEAMER

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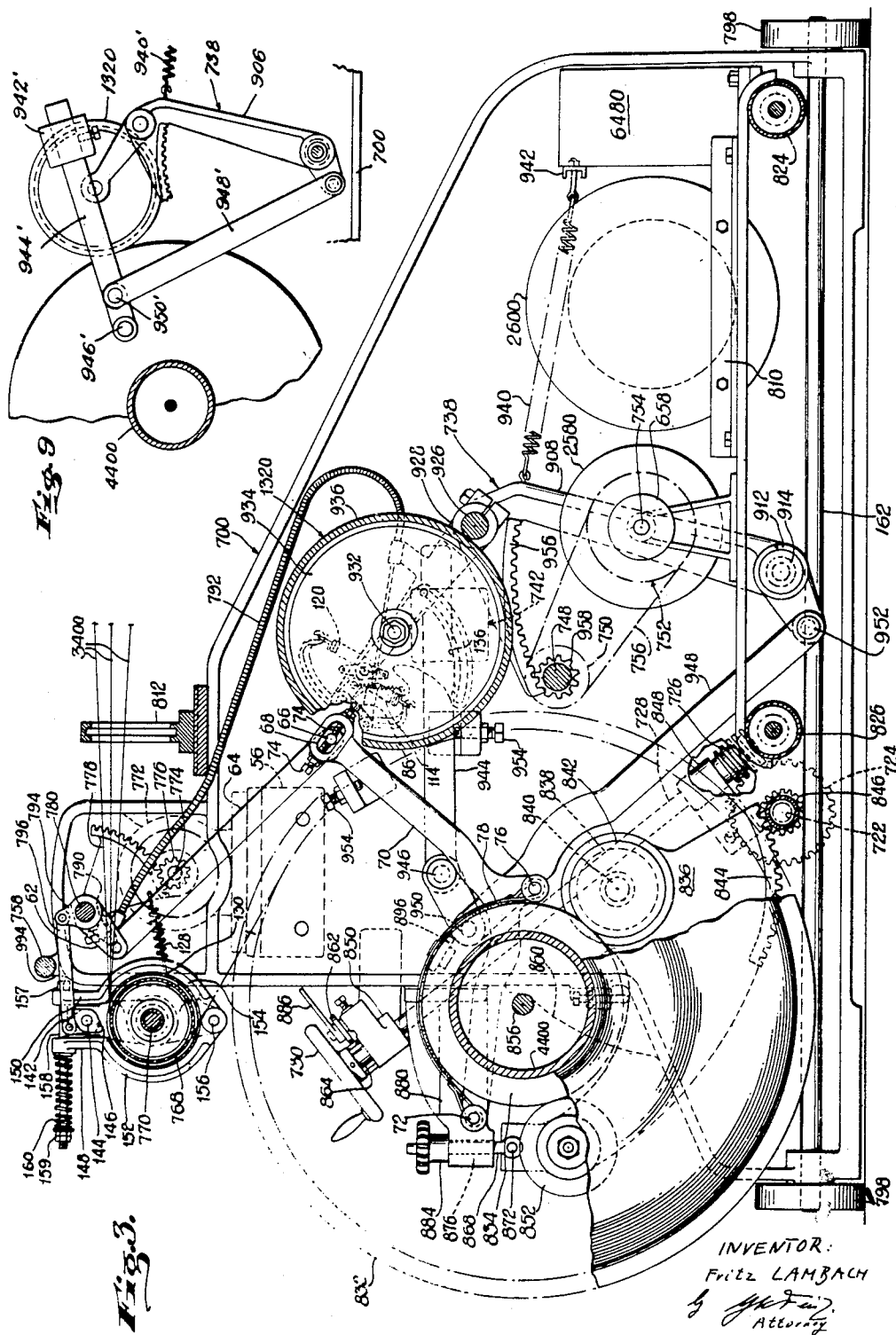
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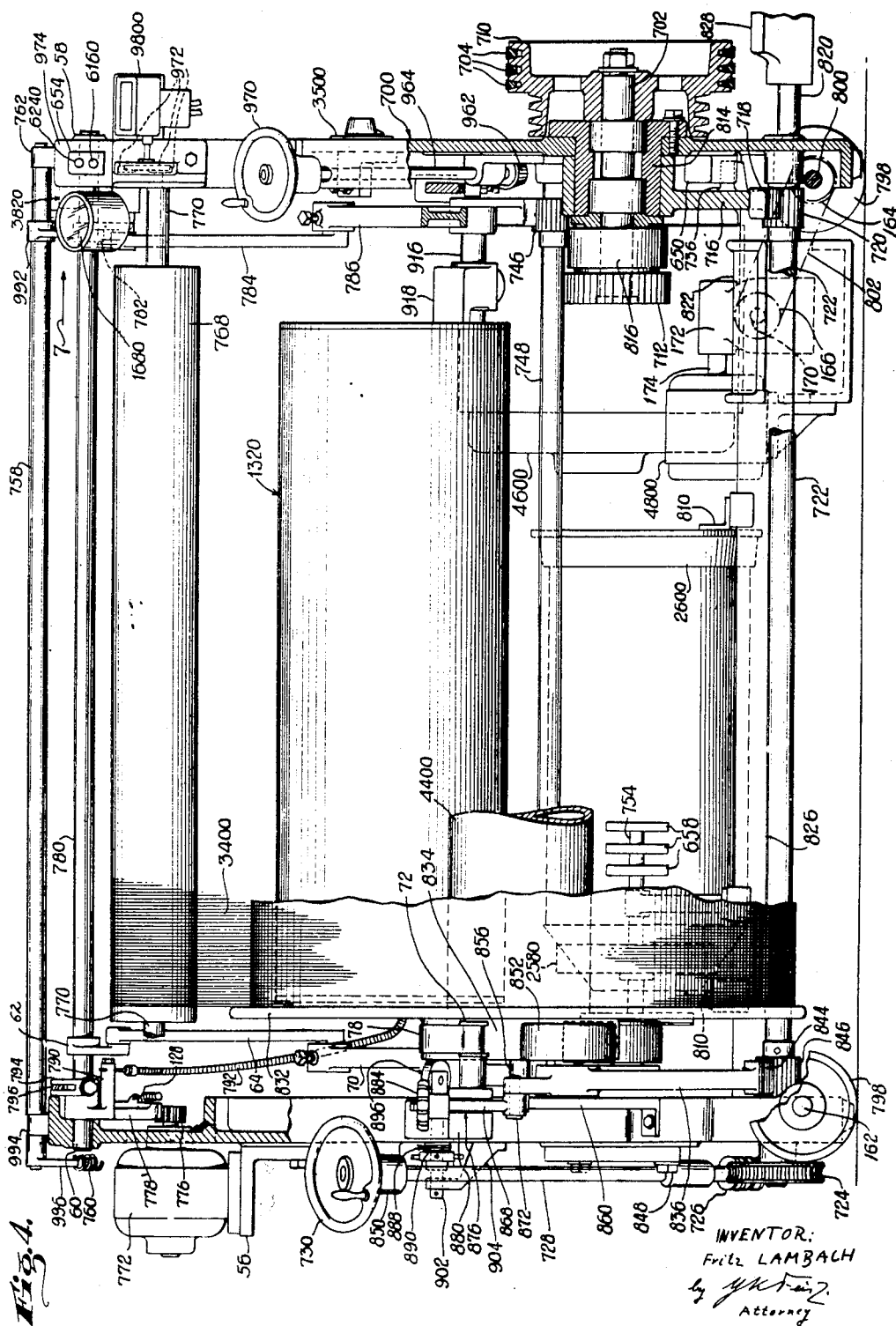
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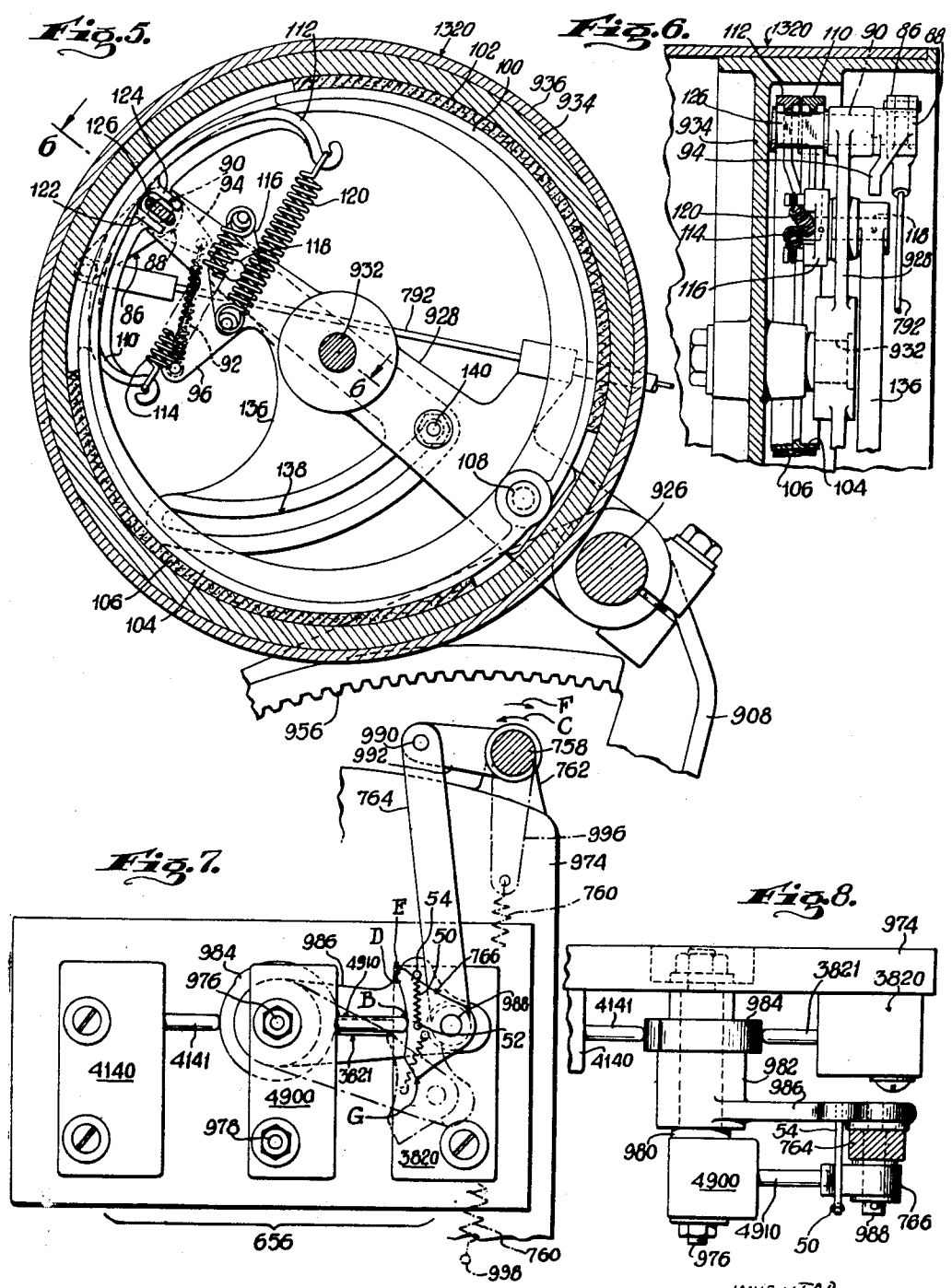
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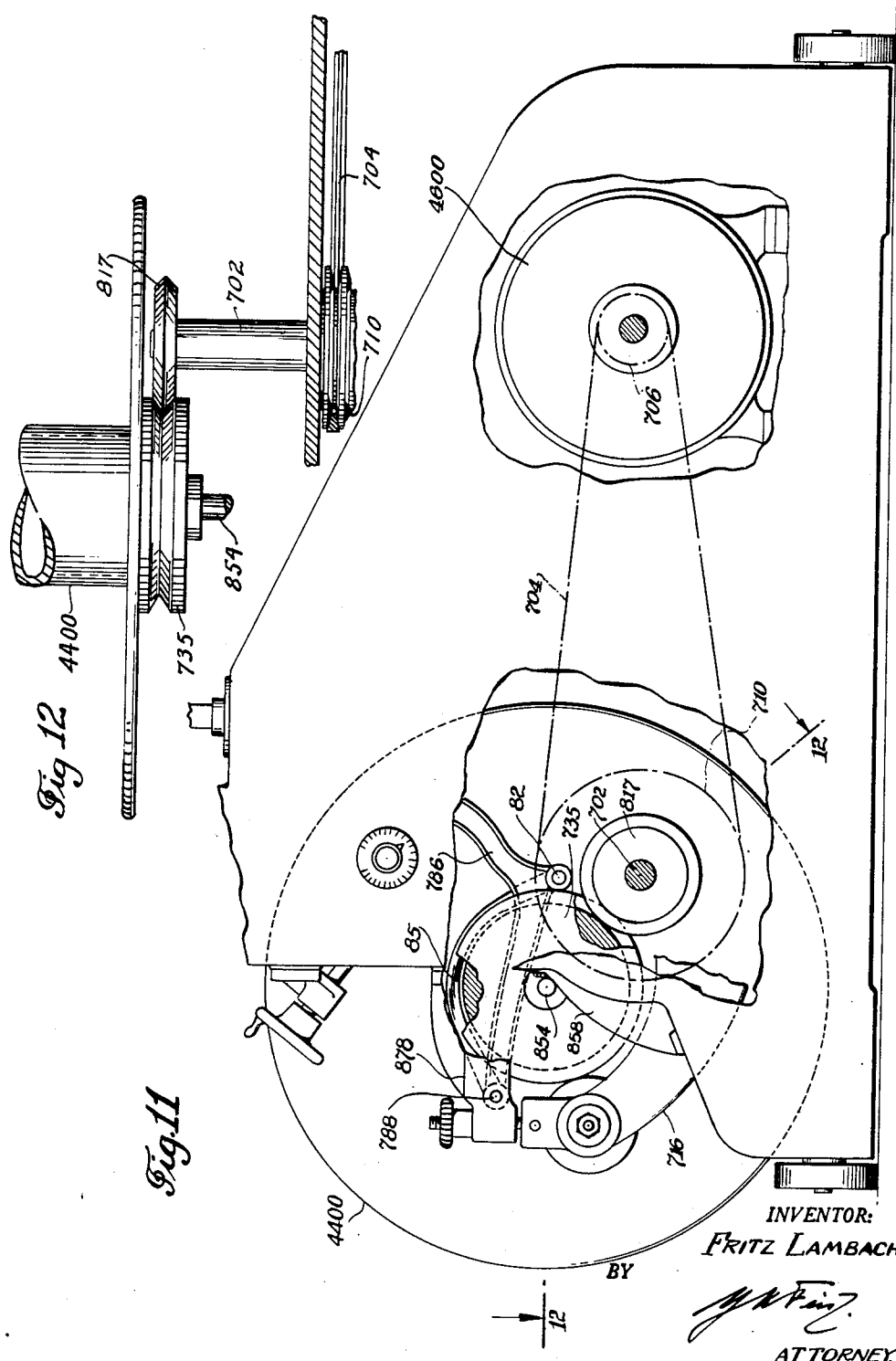
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Patented Sept. 11, 1945

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UNITED STATES PATENT OFFICE

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BEAMER

Fritz Lambach, Tenafly, N. J.

Application September 12, 1942, Serial No. 458,087
In Great Britain July 24, 1942

23 Claims. (Cl. 28—39)

This application is a continuation in part of my patent application Serial #419,833, filed Nov. 21, 1941, for a Beamer.

My invention relates to textile machines, and more particularly to a beamer or the like.

An object of my invention is to improve upon the construction of beamers or the like as now ordinarily made.

My invention consists in certain novel features of construction of my improvements as will be hereinafter fully described.

Further objects and advantages of the invention will be apparent from the following disclosure of an embodiment thereof.

In the accompanying drawings:

Fig. 1 an elevational view of a beaming plant including a beamer and a creel, a portion of the wall of the beamer being broken away, and only the end portion of the creel being shown in the drawing,

Fig. 2 is a top plan view of the beamer, some parts being broken away, and some parts being shown in section,

Fig. 3 a sectional view of the beamer taken on line 3—3 of Fig. 2,

Fig. 4 a front elevational view of the beamer, some parts being broken away and some parts being shown in section,

Fig. 5 a sectional view of the pressure roll taken on line 5—5 of Fig. 2 illustrating the braking mechanism,

Fig. 6 a sectional view of the braking mechanism of the pressure roll taken on line 6—6 of Fig. 5,

Fig. 7 an elevational view of the "Stop-inch-start-station" seen in the direction of the arrow 7 shown in Fig. 4,

Fig. 8 a top plan view of the "Stop-inch-start-station" shown in Fig. 7,

Fig. 9 a somewhat diagrammatic view of a different embodiment of the load acting on the support carrying the pressure roll,

Fig. 10 a diagram of the wiring system for the electrical equipment of the beaming plant, wherein the switches are shown in their normal position when the relays are deenergized,

Fig. 11 is a fragmentary elevational view of a different embodiment of a beamer, a portion of the wall of the beamer being broken away for a better disclosure of the parts exposed thereby, and

Fig. 12 is a sectional view of the beamer according to Fig. 11, taken on line 12—12 of said Fig. 11.

Referring now to Figs. 1—4, 700 generally indi-

cates a beamer, and 2400 generally indicates a creel arranged at a suitable distance from the former. An A. C. to D. C. motor generator set generally indicated by 2600 is mounted in a suitable manner within the casing of the beamer on supporting bars 810. The A. C. to D. C. motor generator set 2600 is supplied with alternating current from an electrical source (not shown) through the line 30 leading to the hand switch 3540 mounted on a wall of the beamer. The A. C. to D. C. motor generator set 2600 delivers direct current to various electrical equipments of the beaming plant through the electrical connections shown in Fig. 10.

During the normal operation of the beaming plant the warp yarns 3400 travel from the bobbins 3600 mounted in the creel 2400 through a reed 812 secured to the casing of the beamer to a measuring roll 768 rotatably mounted on the beamer and thence to a beam 4400 inserted into the beamer and driven by an electromotor 4600 as will be described hereinafter.

The shaft 708 of the electromotor 4600 carries a grooved step pulley 706. A complementary step pulley 710 is secured to one end of a main shaft 702 rotatably arranged in a bearing 814 carried by the frame of the beamer. The other end of said main shaft 702 carries a pinion 712 and a friction pulley 816 keyed thereto. The main shaft 702 may be rotated by the electromotor 4600 through belts 704 trained around complementary portions of the step pulleys 706 and 710. The electromotor 4600 is mounted on a support 818 secured to a rod 820 rotatably arranged on a supporting member 822 resting on two spacing tubes 824 and 826 secured to the walls of the beamer. Owing to the described mounting of the electromotor 4600 the latter tends to swing the support 818 by its weight about the fulcrum 820 in clockwise direction, whereby the belts 704 are automatically tensioned.

A pedal 828 is secured to the end of the rod 820 projecting from the wall of the beamer. If it is desired to change the speed ratio, the support 818 carrying the electromotor 4600 may be swung in counter-clockwise direction by means of the pedal 828, whereby the belts 704 are loosened, so that they may be easily transferred from one step of the complementary step pulleys 706 and 710 to the other step thereof. After the carrying out of this transfer the pedal may be released, so that the belts are automatically tensioned in their new position by the weight of the electromotor 4600 acting on the swingable support 818.

The beam 4400 to be inserted into the beamer

has two flanges 830 and 832. A gear 714 is secured to the beam adjacent the flange 830. Furthermore, discs 734 and 834 serving as brake drums are secured to the beam outside the flanges 830 and 832 respectively. The gear 714 may be used for engagement with the driving pinion 712, and the disc or brake drum 734 may be used for engagement with the driving friction pulley 816.

On each side of the beamer a supporting member or cradle 716 and 836 respectively is swingably arranged for receiving a beam 4400 and lifting same into operating position with the gear 714 in engagement with the pinion 712 and the disc or brake drum 734 in engagement with the friction pulley 816. The supporting member 716 is rotatably mounted on the cylindrical surface of the bearing 814, as best shown in Figs. 1 and 4. Likewise, the supporting member 836 is rotatably mounted on the cylindrical surface of a bearing 838 secured to the other side wall of the beamer, as best shown in Fig. 3. A shaft 840 carrying a bearing roller 842 for engagement with the disc or brake drum 834 of the beam 4400 is rotatably arranged in said bearing 838. Each supporting member 716 and 836 respectively is provided with a gear segment 718 and 844 respectively. The gear segment 718 meshes with a pinion 720, and the gear segment 844 meshes with a pinion 846. The pinions 720 and 846 are keyed to a transverse shaft 722 rotatably arranged in the side walls of the beamer. One end of the shaft 722 projects from the wall of the beamer and carries a worm gear 724 meshing with a self-locking worm 726 secured to the lower end of a spindle 728 rotatable in bearings 848, 850 mounted on the wall of the beamer. The upper end of the spindle 728 carries a hand wheel 730, by means of which the spindle may be rotated for lifting or lowering the supporting member 716 and 836. At the free end of each supporting member a roller 732 and 852 respectively is rotatably arranged for bearing engagement with the discs or brake drums 734 and 834 respectively of the beam 4400.

In order to insert a beam into the beamer, the supporting members 716 and 836 are moved into their lower receiving position by turning the hand wheel 730 in one direction. In said receiving position, the bearing roller 732 is in the position 732' shown in Fig. 1. After a beam 4400 has been placed onto the supporting members, the latter are lifted by turning the hand wheel 730 in opposite direction, until the supporting members and beam reach the operating position shown in full lines in Fig. 1. In said operating position, the gear 714 is in engagement with the driving pinion 712, the disc or brake drum 734 is in contacting engagement with the driving friction pulley 816 and the bearing roller 732, and the disc or brake drum 834 is in contacting engagement with the bearing rollers 842 and 852. During the described lifting or swinging of the supporting members and the beam, axle rods 854 and 856 projecting from each end of the beam are in engagement with stationary guides or cams 858 and 860 mounted on the frame of the beamer. The guides or cams 858 and 860 are of such a shape, that they lift the teeth of the gear 714 to a certain extent above the teeth of the driving pinion 712, whereupon they permit an engagement of the teeth of the gear and pinion in a substantially radial direction. The self-locking worm 726 and a spring loaded pawl 862 cooperating with a ratchet wheel 864 secured to the spindle 728 prevent an undesired return of the supporting members 716 and 836 into the receiving position

under the load of the beam. Furthermore, one end of screw threaded bolts 866 and 868 respectively is pivoted to the free end of each supporting member at 870 and 872 respectively. When the supporting members 716 and 836 are in their upper operating position, the bolts 866 and 868 may be swung upwards into slots 874 and 876 respectively of brackets 878 and 880 respectively secured to the frame of the beamer, whereupon nuts 882 and 884 respectively may be screwed onto the free ends of the bolts for engagement with the brackets. Thus, the supporting members 716 and 836 are firmly held in their upper operating position by the coupling means or latches 866, 882 and 868, 884 connecting the supporting members with the stationary brackets.

As best shown in Figs. 1 and 4, a lug 736 arranged on the swingable supporting member 718 cooperates with a spring loaded control element 651 of a safety switch 650 mounted on a wall of the beamer. As long as the lug 736 is disengaged from the control element 651 the spring (not shown) acting on said element holds the safety switch 650 in open position as shown in the right portion of the wiring diagram of Fig. 10. As will be described hereinafter, the safety switch 650 is arranged in the line 386 of the motor start circuit and motor start holding circuit, so that the electromotor 4600 cannot be operated as long as the safety switch 650 is in open position. The safety switch 650 is closed only upon engagement of the lug 736 with the spring loaded control element 651 when the supporting member 716 is in its upper operating position shown in full lines in Fig. 1. Therefore, the electromotor 4600 can be started and operated for driving the beam 4400 through the belts 704, pinion 712 and pulley 816 only when the beam carried by the supporting members 716 and 836 is in operating position with its gear 714 in mesh with the driving pinion 712 and with its disc or brake drum 734 in contact with the driving friction pulley 816. As soon as, after a disengagement of the bolts 866 and 868 from the slots of the brackets 878 and 880 and after a disengagement of the pawl 862 from the ratchet wheel 864 by means of a lever 886, the hand wheel 730 is rotated for a lowering of the supporting members 716 and 836, the lug 736 is disengaged from the control element 651 of the safety switch 650, so that the latter is opened by its spring and the motor start circuit is interrupted, whereby a starting of the electromotor 4600 is rendered impossible.

As best shown in Figs. 2 and 4, on each side of the beamer adjusting means, generally indicated by 888, are arranged for adjusting the beam 4400 in the direction of its longitudinal axis in a proper position relative to a pressure roll 1320 mounted on the beamer for cooperation with the winding on the beam in a manner to be described hereinafter. Each adjusting means 888 comprises a hollow sleeve 890 screwed into a screw threaded hole of the wall of the beamer. A rod 892 carrying at one end thereof a fork-like portion 894 extends through the bore of the sleeve. A roller 896 is rotatably arranged in the fork 894. The shoulder of the fork abuts against one end of the sleeve 890, and a set collar 898 secured to the rod rests on the other end of the sleeve, so that an inward or outward movement of the sleeve in the direction of its longitudinal axis upon rotation thereof by means of pins 900 secured thereto causes a corresponding inward or outward movement of the rod 892 and the roller 896. The free end of the rod has a square por-

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tion 902 slidably engaged with a slot of a bracket 904 secured to the wall of the beamer, whereby the rod is prevented from rotating in the sleeve 890. When, after the insertion of a beam into the beamer, the beam slidably resting on the rollers of the supporting members has been brought into the operating position, the beam may be shifted in the direction of its longitudinal axis into the proper position relative to the pressure roll by rotating the sleeve 890 of the proper adjusting means 888 for moving the roller 896 associated therewith inwards in contact with the surface of one brake drum of the beam and by continuing the inward movement of the roller, until the beam has been shifted by the roller into the desired position. Then, the roller of the other adjusting means 888 is moved inwards in contact with the surface of the other brake drum of the beam. Thus, the beam is guided by the two rollers of the two adjusting means during its rotation, whereby an undesired oscillation in the direction of its longitudinal axis is eliminated.

The pressure roll 1320 is rotatably mounted on a swingable assembly generally indicated by 738. The assembly 738 comprises two double-armed levers 906 and 908, one on each side of the beamer. The hub 910 of the lever 906 is swingably mounted on a stud 740 secured to a side wall of the beamer, and the hub 912 of the lever 908 is swingably mounted on a stud 914 secured to the opposite side wall of the beamer. A rod 916 is rigidly connected to the upper end of the long arm of the lever 906. One end of an arm 918 is secured to said rod 916, the other end of said arm 918 carries a roller bearing 920, wherein a shaft 922 secured to an end disc 924 of the pressure roll 1320 is rotatably arranged. Likewise a rod 926 is rigidly connected to the upper end of the long arm of the lever 908, and one end of an arm 928 is secured to said rod 926, while the other end of said arm 928 carries a roller bearing 930, wherein a shaft 932 secured to the opposite end disc 934 of the pressure roll 1320 is rotatably arranged. The two end discs 924 and 934 of the pressure roll 1320 are connected with each other by a cylindrical portion 936. Thus, the pressure roll 1320 is rotatable about the pivots 922 and 932 and is swingable to and from the beam 4400 about the pivots 740 and 914.

The pivots 740 and 914 are arranged in such a relationship to the center of the pressure roll 1320, that the pressure roll tends to move by its own weight toward the beam 4400 in the direction of the arrow A shown in Fig. 1, whereby the pressure roll subjects the winding on the beam to a certain pressure. The position of the pressure roll varies in dependence on the diameter of the winding on the beam. When the pressure roll is in the position shown in Fig. 1, the beam is full and the diameter of the winding is large. At the beginning of the beaming operation, however, the diameter of the winding on the beam is considerably smaller, and the center of the pressure roll is considerably nearer to the center of the beam. Owing to this change in the position of the center of the pressure roll the latter would exert a considerably greater pressure on the winding on the beam at the beginning of the beaming operation, if there were no means provided for acting on the assembly 738 to compensate said difference in pressure. As best shown in Figs. 1-3 this compensating means comprises a tension spring 940 stretched between the long arm of the double-armed lever 908 and a stationary part 942 of the beamer. The spring 940

counteracts and compensates the action of the weight of the pressure roll on the winding on the beam, i. e. the nearer the pressure roll to the center of the beam and the greater the action of the weight of the pressure roll, the greater the expansion of the spring and the greater its counteraction. The spring 940 is so designed and arranged, that, during the entire beaming operation, the pressure roll subjects the winding on the beam to a substantially constant pressure irrespective of the variation in the position of the pressure roll in dependence on the increase of the diameter of the winding. When the pressure roll is in the position shown in Fig. 1, the spring 940 is contracted, so that it causes no action on the pressure roll assembly.

Furthermore, two counter-weights 742, one on each side of the beamer, counteract the action of the weight of the pressure roll on the winding on the beam, whereby the pressure on the winding is reduced to a low degree as it may be desirable for the beaming of warps of certain materials. Each counter-weight 742 is adjustably mounted on the long arm of a bell crank-lever 944 pivoted to the side wall of the beamer at 946. The short arm of said bell crank lever is pivoted to one end of a link 948 at 950. The other end of said link 948 is pivoted to the short arm of the double-armed lever 906 or 908 respectively at 952. Adjustable set screws 954 serve to limit the extreme position of the lever 944 and of the pressure roll assembly 738, 1320 connected with said lever.

If it is desired to subject the winding on the beam to an appreciable substantially constant pressure during the beaming operation, the arrangement shown in Fig. 9 may be used. The tension spring 940' acts on the assembly 738 carrying the pressure roll 1320 in the same manner as the tension spring 940 shown in Fig. 3. The weight 742', however, does not act as a counterweight but causes a pressure on the winding on the beam in addition to the pressure caused by the own weight of the pressure roll. The weight 742' is adjustably mounted on a lever 944' pivoted to the wall of the beamer at 946'. An intermediate point 950' of the lever 944' is pivotally connected with one end of the connecting link 948', the other end of which is pivoted to the short arm of the double-armed lever 906 or 908 respectively.

As best shown in Figs. 1-3, the double-armed lever 906 carries a gear segment 744 meshing with a pinion 746, and the double-armed lever 908 carries a gear segment 956 meshing with a pinion 958. Both pinions 746 and 958 are secured to a cross-shaft 748 rotatably arranged in the walls of the beamer. Said cross-shaft 748 carries a sprocket 750 connected with a sprocket 752 secured to a cross-shaft 754 by means of a chain 756. The cross-shaft 748 is rotatably arranged in suitable bearings and carries the movable finger (not shown) of an automatic motor field rheostat 2580 to be described hereinafter and the movable fingers (not shown) of an automatic three-phase resistor 658 to be described hereinafter. As will be readily understood, during the beaming operation, the pressure roll assembly 1320, 738 is moved in clockwise direction about the pivots 740 and 914 as viewed in Figs. 1 and 3 in accordance with the increase of the diameter of the winding on the beam 4400, so that the automatic motor field rheostat 2580 and the automatic three-phase resistor 658 are adjusted in dependence on the building up of the

winding on the beam through the medium of the gear segments 744, 956, pinions 746, 958 and chain drive 750, 752, 756 for a purpose to be described hereinafter.

As best shown in Figs. 1 and 2, a bevel gear 960 is keyed to the cross-shaft 748 carrying the pinions 746 and 958. This bevel gear is in mesh with a bevel gear 962 secured to the lower end of a spindle 964 rotatably arranged in bearings 966 and 968 mounted on the frame of the beamer. A hand wheel 970 is secured to the upper end of said spindle 964. Thus, the pressure roll assembly 1320, 738 may be swung about the pivots 740 and 914 by hand by means of the hand wheel 970, spindle 964, bevel gears 960 and 962, pinions 746 and 958, and gear segments 744 and 956.

As best shown in Figs. 1 and 3, the warp yarns 3400 are trained around a measuring or yarn roll 768, the shaft 770 of which is rotatably mounted on the frame of the beamer, so that the measuring roll is rotated by the warp yarns during the beaming operation. The measuring or yarn roll 768 drives a shut off counter 9800 and a speedometer 1680 through the medium of a toothed gearing 972. The shut off counter 9800, which is secured to the frame of the beamer and may be of any well known type, serves to cause an automatic stoppage of the beamer as soon as the beam 4400 has performed a predetermined number of revolutions for which the shut off counter has been set. The speedometer 1680 mounted on the frame of the beamer serves to indicate the circumferential speed of the measuring roll 768.

The "Stop-inch-start-station" 656 diagrammatically shown in the upper right portion of the wiring diagram illustrated in Fig. 10 comprises a start switch 3820, an inching speed switch 4900 and a stop switch 4140. The electrical connection of these switches with the circuits of the electrical equipment of the beamer will be described hereinafter in connection with the description of the diagram of Fig. 10. At this portion of the specification it is sufficient to state that a closure of both the start switch 3820 and the inching speed switch 4900 may cause an operation of the electromotor 4600 at an extremely low speed, i. e. a so-called inching speed, that a closure of the start switch 3820 alone may cause an operation of the electromotor 4600 at normal high speed, and that an opening of the stop switch 4140 causes an interruption of the supply of current to the electromotor 4600 for stopping same.

As best shown in Figs. 1 and 4, the switches 3820, 4900 and 4140 of the "Stop-inch-start-station" 656 are arranged on the inside of an upper extension 974 of a side wall of the beamer. As best shown in Figs. 7 and 8, the start switch 3820 and the stop switch 4140 are directly mounted on the inner surface of the extension 974 by screws. The inching speed switch 4900, however, is arranged at a certain distance from the inner surface of the extension 974 on bolts 976 and 978 secured to the extension 974. The inching speed switch 4900 rests against a collar 980 on the bolt 976. A member 982 carrying a cam 984 and a lever 986 is rotatably arranged on said bolt 976 between the collar 980 and the extension 974. The cam 984 cooperates with the spring loaded control elements 3821 and 4141 (the springs are not shown) of the start switch 3820 and the stop switch 4140. The lever 986 carries a stud 988 at its free end. One end of a link 764 is swingably arranged on said stud 988. The other end of

said link 764 is pivotally connected at 990 to the free end of a lever 982 keyed to a shipper rod 758 rotatably mounted in bearings 762 and 994 (Figs. 4 and 7). A lever 996 is secured to the end of the shipper rod 758 projecting from the bearing 994, and a tension spring 760 is stretched between the free end of said lever 996 and a stationary point 998 (Fig. 7) in such a way, that the spring tends to hold the mechanism 758, 992, 764, 986, 984, 766 in an intermediate neutral position shown in full lines in Fig. 7. A cam-like actuating element 766 adapted to cooperate with the spring loaded control element 4910 (the spring is not shown) of the inching speed switch 4900 is swingably mounted on the stud 988 carried by the lever 986. A tension spring 50 stretched between a point 52 of the element 766 and a stop 54 secured to the lever 986 tends to urge said element 766 against said stop 54 as shown in full lines in Fig. 7.

The operation of the mechanism of the "Stop-inch-start-station" is as follows:

When the beamer is at a standstill, the mechanism 758, 992, 764, 986, 984, 766 is in the neutral position shown in full lines in Fig. 7, wherein the spring loaded control elements 3821 and 4141 of the start switch 3820 and stop switch 4140 contact the inactive round portion of the cam 984 and the spring loaded control element 4910 of the inching speed switch 4900 contacts the cam surface of the element 766 at the inactive point B, so that the springs (not shown) acting on the control elements 3821 and 4910 of the start switch 3820 and inching speed switch 4900 hold these switches in the open position shown in Fig. 10, and the spring (not shown) acting on the control element 4141 of the stop switch 4140 holds same in the closed position shown in Fig. 10. In order to start the electromotor 4600 of the beamer, the shipper rod 758 is turned by hand through a certain angle into an intermediate position in the direction of the arrow C shown in Fig. 7. During this rotation of the shipper rod 758 through said limited angle, the cam 984 is rotated in clockwise direction and its elevated portion urges the control element 3821 against the action of its spring to the right, whereby the start switch 3820 is closed. At the same time, the lever 986 is turned in clockwise direction and the active portion B-D of the cam-like surface of the element 766 resting against the stop 54 slides along the control element 4910 and urges same against the action of its spring to the left, whereby the inching speed switch 4900 is closed. Thus, the start switch 3820 and the inching speed switch 4900 are closed and the electromotor 4600 is operated at inching speed as long as the shipper rod 758 is held against the action of the spring 760 in above described position wherein the elevated portion of the cam 984 contacts the control element 3821 and the point D of the element 766 contacts the control element 4910.

If the shipper rod 758 is turned further from the intermediate position in the direction of the arrow C into an extreme active position, the cam 984 and the lever 986 are turned further in clockwise direction. The elevated portion of the cam 984 is of such a length, that, during this continuation of the rotation of the shipper rod 758, the elevated portion of the cam remains in contact with the control element 3821 and holds the start switch 3820 in closed position; the point E of the element 766, however, slips off the free

end of the control element 4910, whereby the control element 4910 is released and the spring acting on said control element 4910 returns same into the position shown in Fig. 7 for opening the inching speed switch. Thus, the start switch 3820 remains in closed position, while the inching speed switch 4900 is opened, so that the electromotor 4600 is operated at normal high speed.

As soon as the shipper rod 758 is released, the spring 760 causes a rotation of the shipper rod in the direction of the arrow F and a return of the mechanism 758, 992, 764, 986, 984, 766 into the neutral position shown in full lines in Fig. 7. During said return of the mechanism, the cam 984 and the lever 986 are rotated in counter-clockwise direction. Shortly before the cam 984 reaches the neutral position, the elevated portion of the cam is disengaged from the control element 3821 and the latter is urged by its spring against the inactive round portion of the cam, whereby the start switch 3820 is returned into open position. Furthermore, during said return of the mechanism, the inching speed switch 4900 is not actuated and remains in its open position for the following reasons: During the return of the lever 986 in counter-clockwise direction, the point E of the swingable element 766 hits the lower surface of the control element 4910, whereby the swingable element 766 is swung against the action of the spring 50 in counter-clockwise direction about the pivot 988 without altering the position of the control element 4910 as indicated in dash and dot lines in Fig. 7. Shortly before the lever 986 reaches the neutral position, the point E of the swingable element 766 slips off the free end of the control element 4910, so that the swingable element 766 is swung by the spring 50 in clockwise direction about the pivot 988 against the stop 54. Thus, no actuation of the inching speed switch 4900 has taken place and all elements of the mechanism 758, 992, 764, 986, 984, 766 are again in the position shown in full lines in Fig. 10 after the return of said mechanism by the spring 760 into the neutral position. Although the start switch 3820 is opened after the release of the shipper rod 758 and the return of the mechanism into the neutral position, the high speed operation of the electromotor 4600 is continued in the embodiment of the beamer shown in the drawings, as the electrical connections include a motor start holding circuit as will be described hereinafter in connection with the description of Fig. 10.

During above described movement of the shipper rod 758 from the neutral position in the direction of the arrow C and back into the neutral position in the direction of the arrow F, the stop switch 4140 is not actuated, as only the round inactive portion of the cam 984 contacts the control element 4141 and does not cause a shifting thereof.

If it is desired to stop the beamer by hand, the shipper rod 758 is turned from the neutral position shown in Fig. 7 in the direction of the arrow F into a second extreme active position. This rotation of the shipper rod causes a rotation of the cam 984 in counter-clockwise direction. As soon as the elevated portion of the cam 984 engages the spring loaded control element 4141, the latter is urged against the action of its spring (not shown) to the left, whereby the stop switch 4140 is opened and the beamer is immediately stopped. Upon release of the shipper rod, the latter is turned in the direction of the arrow C by the spring 760 and the mechanism 758, 992,

764, 986, 984, 766 is returned into the neutral position shown in full lines in Fig. 7, wherein the stop switch 4140 is closed and the start switch 3820 and the inching speed switch 4900 are opened as described above.

During above described movement of the shipper rod 758 from the neutral position in the direction of the arrow F and back into the neutral position in the direction of the arrow C, the start switch 3820 and the inching speed switch 4140 are not actuated. During said movement of the shipper rod, only the round inactive portion of the cam 984 contacts the control element 3821 of the start switch, so that the position thereof remains unchanged. Furthermore, the surface B—G of the element 766 is of such a shape, that it does not cause any movement of the control element 4910 of the inching speed switch 4900, when the lever 986 is swung in counter-clockwise direction from the neutral position and is returned subsequently into said neutral position.

The beam 4400, the pressure roll 1320 and the measuring or yarn roll 768 are provided with brakes, by means of which these elements may be arrested immediately upon an interruption of the supply of current to the electromotor 4600. A single torque motor 772 (Figs. 1-4), i. e. a motor with high starting torque which may be loaded to such a degree that a stoppage occurs without an undue heating of the parts of the motor, is mounted on a bracket 56 secured to a wall of the beamer and is operatively connected with the brakes of the beam of the pressure roll and of the measuring roll in a manner to be described hereinafter for a simultaneous application of said brakes. As will be readily understood, the masses of the pressure roll and measuring roll remain constant during the beaming operation, while the mass of the beam is increased in dependence on the building up of the winding on the beam, i. e. in dependence on the increase of the diameter of the winding on the beam. In order to obtain the same constant braking time of the brakes of the beam the pressure roll and the measuring roll irrespective of the variation of the mass of the beam in dependence on the variation of the diameter of the winding thereon, the braking means and their control are arranged and designed in the following manner:

A pinion 776 secured to the shaft 774 of the torque motor 772 is in mesh with a gear segment 778 keyed to a cross-shaft 780 rotatably mounted in suitable bearings 58 and 60 of the beamer (Figs. 1-4). As best shown in Figs. 3 and 4, on each side of the beamer, an arm 782 and 62 respectively is secured to the cross-shaft 780. The free end of the arm 62 is pivotally connected to one end of a connecting link 64, the other end of which carries a pin 66 engaged with a slot 68 of a braking lever 70 swingably mounted on a stud 72 secured to the bracket 880. The relative position between the link 64 and the braking lever 70 may be adjusted by means of two opposite screws 74 screwed into the end of the lever 70 and engaged with the pin 66. A stud 76 secured to an intermediate point of the braking lever 70 is engaged with a brake band 78 adapted to be applied on the brake drum 834 of the beam 4400. The other end of said brake band 78 is connected with the stud 72 secured to the bracket 880. Likewise, the free end of the arm 782 is pivotally connected to one end of a connecting link 784 (Figs. 1 and 4), the other end of which is adjustably and pivotally con-

nected with one end of a braking lever 786 at 80. The other end of the braking lever 786 is swingably arranged on a stud 788 secured to the bracket 878, and a stud 82 is arranged on an intermediate point of the braking lever 786. As best shown in Fig. 2, one end of a brake band 84 adapted for cooperation with the brake drum 734 of the beam 4400 is engaged with the stud 788, while the other end of said brake band is engaged with the stud 82. Thus, if the torque motor 772 is excited and the gear segment 778 is rotated in counter-clockwise direction into the position shown in Fig. 3, the brake bands 78 and 84 are applied on the brake drums 834 and 734 in accordance with the output of the torque motor. The above described adjusting means 66, 68, 74 and 80 may be used for adjusting a certain braking time of the braking mechanism for a certain mass of the beam and a certain output of the torque motor 772. As pointed out above, during the beaming operation the mass of the beam varies in dependence on the increase of the diameter of the winding on the beam. As it is desired to obtain a constant braking time of the beam brake irrespective of the variation of the mass of the beam in dependence on the increase of the diameter of the winding on the beam, the output of the torque motor should be increased in dependence on the increase of the diameter of the winding on the beam. For that purpose, the three-phase resistor 658 automatically controlled by the pressure roll 1320 through the mechanism 744, 956, 746, 748, 750, 756, 752 in dependence on the increase of the diameter of the winding on the beam is arranged in circuit with the torque motor 772 as shown in Fig. 10. The larger the diameter of the winding and the greater the mass of the beam, the smaller the resistance of the resistor 658 and the higher the output of the torque motor 772. Thus, the output of the torque motor 772 is automatically controlled in dependence on the increase of the diameter of the winding on the beam, so that a substantially constant braking time of the beam brake is obtained.

As best shown in Figs. 3 and 4, a projecting arm 790 integral with the body of the gear segment 778 is connected with one end of a Bowden wire 792, the other end of which is connected to an arm 86 of a bell crank lever 88. As best shown in Figs. 5 and 6, the bell crank lever 88 is secured to a shaft 90 rotatably mounted on the arm 928 pivotally connected with the end disc 934 of the pressure roll 1320 at 932. A tension spring 92 stretched between the other arm 94 of the bell crank lever 88 and an extension 96 of the arm 928 tends to swing the shaft 90 in clockwise direction as viewed in Fig. 5. A brake element 100 provided with a lining 102 and a brake element 104 carrying a lining 106 are pivotally mounted on the arm 928 at 108. The end portion 110 of the brake element 100 and the end portion 112 of the brake element 104 are reduced in width with respect to the main body of the respective brake band and are arranged next to each other. A tension spring 114 stretched between the end 110 of the brake element 100 and an arm of a member 116 secured to a shaft 118 arranged on the arm 928 tends to apply the brake element 100 on the inner cylindrical surface of the end disc 934 of the pressure roll 1320. A tension spring 120 stretched between the end 112 of the brake element 104 and the other arm of the member 116 tends to apply the brake element 104 on the inner cy-

lindrical surface of the end disc 934. An abutment 122 of the brake element 100 and an abutment 124 of the brake element 104 are arranged for cooperation with a spreader 126 formed by a flattened portion of the oscillatable shaft 90. Fig. 5 illustrates the elements in a position, where the brake elements are applied. Normally, however, the spreader 126 is in a position turned in counter-clockwise direction through an angle of about 70° with respect to the position shown in Fig. 5, so that its edges contact the abutments 122 and 124 and spread same further apart from each other, whereby the brake bands 102 and 104 are disengaged from the cylindrical surface of the end disc 934 and the pressure roll 1320 may freely rotate. As soon as the torque motor 772 is excited and the gear segment 778 is moved into the position shown in Fig. 3, the shaft 90 and the spreader 126 are swung in clockwise direction by means of the Bowden wire 792 into the position shown in Fig. 5 with the assistance of the tension spring 92, whereby the edges of the spreader 126 are disengaged from the abutments 122 and 124, so that the brake elements 100 and 104 are applied on the cylindrical surface of the end disc 934 by the action of the tension springs 114 and 120 for an immediate stoppage of the pressure roll 1320. Obviously, the braking action of the braking elements on the constant mass of the pressure roll is caused only by the constant force of the tension springs 114 and 120 after a release of said springs by the spreader or locking means 126 and is entirely independent of the output of the torque motor 772 varying in accordance with the increase of the diameter of the winding on the beam, so that a constant braking time of the pressure roll brake is obtained.

When the torque motor 772 is deenergized for a release of the brakes, the shaft 90 and the spreader 126 are returned through the medium of the Bowden wire 792 in counter-clockwise direction into their normal position at an angle of about 70° to the position shown in Fig. 5 by means of a tension spring 128 stretched between the gear segment 778 and a stationary point 130 of the beamer as shown in Fig. 3. Of course, the spring 128 is so designed that it may overcome the action of the springs 114 and 120 on the spreader 126.

As described above, one end of each of the tension springs 114 and 120 is connected with the member 116. Said member 116 is secured to a shaft 118 rotatably mounted on the arm 928. A rotation of the member 116 by a rotation of the shaft 118 results in a change of the length of the tension springs 114 and 120, whereby the braking action of these springs will be changed. A segment 136 provided with a curved slot 138 is secured to the shaft 118 for an adjustment of the position of the member 116 and the braking action of the springs 114 and 120, i. e. for the adjustment of a certain braking time of the pressure roll brake. The adjusting mechanism 116, 136 may be held in its position by a screw 140, which is arranged on the arm 928 and passes through the curved slot 138 of the segment 136.

As best shown in Figs. 2-4, an arm 794 integral with the body of the gear segment 778 is pivotally connected to one end of a link 796, the other end of which is pivoted to a lever 142 secured to a shaft 144 rotatably arranged on the frame of the beamer. A spreader 146 arranged in the space between the extensions 148 and 150 of two brake shoes 152 and 154 is secured to the shaft

144. The brake shoes 152 and 154 provided with the usual lining are pivotally mounted on the beamer at 156 for cooperation with a brake surface on the measuring roll 768. A bolt 158 with head 157 and nut 159 extends through holes of the extensions 148 and 150, and a compression spring 160 arranged between the nut 159 and the extension 148 tends to apply the brake shoes on the brake surface of the measuring roll 768. Fig. 3 illustrates the spreader 146 in a position disengaged from the extensions 148 and 150, so that the brake shoes 152 and 154 are applied by the spring 160. Normally, however, the spreader is in a position, wherein its edges are engaged with the extensions 148 and 150, so that the brake shoes 152 and 154 are disengaged from the brake surface of the measuring roll 768 and the latter may rotate freely. As soon as the torque motor 772 is excited and the gear segment 778 is swung in counter-clockwise direction into the position shown in Fig. 3, the spreader 146 is swung in counter-clockwise direction by the link 796, whereby its edges are disengaged from the extensions 148 and 150 and the brake shoes 152 and 154 are applied on the brake surface of the measuring roll 768 by the action of the spring 160 for an immediate stoppage of the measuring roll. As soon as the torque motor 772 is deenergized for a release of the brakes, the spreader 146 is swung in clockwise direction by means of the tension spring 128 acting on the gear segment 778 against the action of the compression spring 160, whereby the extensions 148 and 158 are spread apart from each other for a disengagement of the brake shoes 152 and 154 from the brake surface of the measuring roll 768. The adjustment of a certain braking time of the measuring roll brake may be obtained by an adjustment of the force of the compression spring 160 by means of a variation of the position of the nut 159 on the bolt 158. The operation of the braking mechanism of the measuring or yarn roll is similar to the operation of the braking mechanism of the pressure roll. The braking action of the brake shoes 152 and 154 on the constant mass of the measuring roll 768 is caused only by the constant force of the spring 160 after a release of said spring by the spreader or locking means 156 and is entirely independent of the varying output of the torque motor 772, so that a constant braking time of the measuring roll brake is obtained.

As described above, the gear segment 778 driven by the torque motor 772 is operatively connected with the brake of the beam 4400 by the linkage 62, 64, 70 and 782, 784, 786, with the brake of the pressure roll 1320 by the arm 790 and Bowden wire 792, and with the brake of the measuring roll 768 by the arm 794 and the link 796, so that all brakes are applied simultaneously upon an energization of the torque motor 772. Furthermore, the braking time of all brakes is adjustable with respect to each other and the brakes of the beam, the mass of which varies during the beaming operation, is actuated directly through the torque motor by the linkage 62, 64, 70, 784, 786 in dependence on the output of the torque motor varying in accordance with the building up of the winding on the beam, while the brakes of the pressure roll and measuring roll, the masses of which are constant, are actuated indirectly through the torque motor 772 by a release of the springs 114, 120 and 160 exerting constant forces on the brakes, so that the same constant braking time of all brakes may be obtained irrespective of the variation of the mass of the beam in de-

pendence on the increase of the diameter of the winding thereon.

Although in the preferred embodiment of my invention a torque motor is used for the application of the brakes, any other suitable electrical means, for example a brake solenoid acting on the various braking mechanisms could be used, if desired.

The frame of the beamer is supported by rollers 798 secured to longitudinal shafts 800 and 162 journaled in suitable bearings, one on each side of the beamer. A sprocket 164 keyed to the shaft 800 is connected by a chain 802 with a sprocket 166 secured to the shaft 170 of a transmission 172 including a self-locking worm keyed to the shaft 174 of a reversible motor 4800. The casing of the transmission 172 and the reversible motor 4800 are mounted in a suitable manner on the supporting member 822. Thus, the beamer may be moved sidewise in one direction or the other upon an energization of the reversible motor 4800, so that the beamer may be easily placed in registry with different creels arranged parallel to each other.

Referring now to Fig. 10 illustrating the wiring system for the electrical equipment of the beaming plant, the circuits for the operation and the control of certain parts of the beaming plant will be described hereinafter. For the remaining circuits reference is had to the description of said remaining circuits in my Patent #2,324,611 relating to "An electrical control system for a warping or beaming plant" issued on July 20, 1943, wherein said circuits are more fully described.

The A. C. motor 352 of the A. C. to D. C. motor generator set 2600 is supplied with alternating current from an electrical source (not shown) through the lines 30 upon closure of the hand switch 3540. The generator 356 of the A. C. to D. C. motor generator set 2600 produces the direct current.

The generator 356 is arranged in the following main motor circuit: The terminal 358 of the generator 356 is connected with the terminal 360 of the driving D. C. electromotor 4600 of the beamer through the line 362 including the electromagnet 364 of a motor field vibrating relay 366. The terminal 368 of the generator 356 is connected with the terminal 370 of the driving electromotor 4600 through the line 372 including the pole M₁ of the triple pole main switch M controlled by the trip coil magnet 374, the solenoid 376 of an overload circuit breaker and the series field 380 of the electromotor 4600.

As will be apparent from the foregoing, the electromotor 4600 driving the beam inserted into the beamer may be started by closing the pole M₁ of the main switch M. This pole M₁ may be closed by an energization of the trip coil magnet 374, which is arranged in the following motor start circuit: A line 384 leads from the right-hand terminal of the start switch 3820 to the left-hand terminal of the over-load circuit breaker 378, the right-hand terminal of which is connected with the trip coil magnet 374 by the line 386 including the safety switch 650 controlled by the lug 736 on the swingable supporting member 716 in the manner described above. A line 388 leads from the trip coil magnet 374 to the movable finger 3900 of the automatic motor field rheostat 2580 controlled by the movable pressure roll assembly 738, 1320 as described above in dependence on the increase of the diameter of the winding on the beam 4400 during

the operation of the beamer. The coil 3920 of the automatic motor field rheostat 2580 is connected with one end of the shunt field 394 of the electromotor 4600 through a line 396. The other end of the shunt field 394 is connected with the left-hand terminal of the start switch 3820 through a line 398. Current is supplied to above described motor start circuit from the exciter 456 of the A. C. to D. C. motor generator set 2600 by means of a line 452 connecting the terminal 454 with a junction point 448 in the line 388 and by means of a line 478 connecting the terminal 458 with a junction point 476 in the line 398. The safety switch 650 and the start switch 3820 are normally held in their open position by the action of springs (not shown) as described above. If the safety switch 650 is closed upon a movement of the beam into the operating position shown in Fig. 1, and if the start switch 3820 is closed by a rotation of the shipper rod in the direction of the arrow C from the neutral position shown in Fig. 7, the above described motor start circuit is closed, whereby the trip coil magnet 374 is excited for closing the main switch M with its pole M₁, which in turn closes the above described main motor circuit for starting the electromotor 4600. The action of the inching speed switch 4900, which is closed simultaneously with the start switch 3820 and causes a slow speed operation of the motor 4600 will be described hereinafter in connection with the description of the circuits including said inching speed switch.

When the trip coil magnet 374 is excited by closing above described motor start circuit by means of the spring loaded start switch 3820, the pole M₂ of the main switch M is also closed, whereby a motor start holding circuit including the trip coil magnet of the main switch is closed as will be described hereinafter, so that the main switch remains in closed position upon a subsequent release of the spring loaded start switch by a return of the mechanism controlling the start switch into the neutral position shown in Fig. 7. Said motor start holding circuit may be traced as follows starting from the left-hand terminal of the start switch 3820: The line 398, the shunt field 394, the automatic rheostat 2580, the line 388, the trip coil magnet 374 of the main switch M, the line 386, the safety switch 650, the bridge of the overload circuit breaker 378, and the line 400 connecting the left-hand terminal of the circuit breaker 378 with the left-hand terminal of the start switch 3820, said line 400 including the pole M₂ of the main switch M, the switch 402 of a counter stop relay 404, the switch 406 of a warp brake relay 408, the electromagnet 410 of a time relay 412, and the stop switch 4140 normally held in closed position by a spring (not shown).

The main switch M is opened and the supply of direct current to the electromotor 4600 is interrupted at the pole M₁, as soon as the above described motor start holding circuit including the trip coil magnet 374 is interrupted either at the stop switch 4140 or at the circuit opening switch 406 of the warp brake relay 408 or at the circuit opening switch 402 of the counter stop relay 404 or at the safety switch 650. The opening of the spring loaded stop switch 4140 takes place when the beamer is arrested by hand by turning the shipper rod 758 in the direction of the arrow F from the neutral position shown in Fig. 7. The opening of the switch 406 takes place when one or more control needles 4160 in the creel 2400 drop and close a needle control

circuit owing to a breakage of yarn as will be described hereinafter. The opening of the switch 402 takes place when the normally closed switch 4180 of the shut off counter 9800 is opened after the performance of a predetermined number of revolutions of the beam as more fully described in my Patent #2,324,611 issued on July 20, 1943. The opening of the safety switch 650 takes place when the supporting member 716 carrying the beam 4400 moves away from the operating position.

Any of above mentioned interruptions of the motor start holding circuit causes an energization of an electromagnet 672 of a torque motor relay 674, whereby a three-pole switch S is closed for supplying current to the torque motor 772, so that the latter is excited and the brakes of the beam 4400, pressure roll 1320 and measuring roll 768 are applied simultaneously in the manner described above. The electromagnet 672 of the torque motor relay 674 is arranged in the following brake circuit starting from the terminal 804 of the secondary winding 546 of a low voltage transformer 548: A line 676 leading from the terminal 804 of the secondary winding to the left-hand terminal of a brake control switch 430 controlled by the electromagnet 434 of a brake control relay 432; a line 678 connecting the right-hand terminal of the brake control switch 430 with the electromagnet 672; a line 680 connecting the electromagnet 672 with the right-hand terminal of the switch 424 of the time relay 412; and a line 682 connecting the left-hand terminal of the switch 424 with the terminal 806 of the secondary winding 546. The primary winding 574 of the low voltage transformer 548 is connected with two lines of the A. C. supply line in the following manner: a line 428 leading from the junction point 426 to the junction point 510 connected with one end of the primary winding 574 by a line 576; the portion of a line 516 between a junction point 514 and a junction point 517, and a line 578 connecting the junction point 517 with the other end of the primary winding 574. The electromagnet 434 controlling the brake control switch 430 of the brake control relay 432 is arranged in the following brake control circuit: A line 436 leading from the electromagnet 434 to the junction point 438, the portion of the line 388 from said junction point 438 to the trip coil magnet 374, the portion of the line 386 leading from the trip coil magnet 374 to the junction point 440, and the line 442 connecting said junction point 440 with the electromagnet 434. Thus, as soon as the motor start holding circuit including the trip coil magnet 374 is interrupted at the stop switch 4140 or the circuit opening switch 406 or the circuit opening switch 402 or the safety switch 650, the described brake control circuit including the electromagnet 434 of the brake control relay 432 is also interrupted, so that the electromagnet 434 is deenergized and the brake control switch 430 is closed. At the same time, the electromagnet 410 of the time relay 412, which is arranged in the motor start holding circuit including the trip coil magnet 374 is also deenergized. Owing to the delaying action of the time relay 412, however, the switch 424 remains for a short period, for example 2 sec., in the closed position 424' shown in dash lines, so that the brake circuit including the electromagnet 672 of the torque motor relay 674 will be closed and the latter will cause a closing of the switch S for an energization of the torque motor 772 and an application of the various brakes of the beamer, as

soon as the electromagnet 434 of the brake control relay 432 is deenergized and the brake control switch 430 thereof is closed. After said short period of 2 sec., for example, however, the switch 424 of the deenergized time relay 412 comes into the open position shown in full lines, so that the electromagnet 672 of the torque motor relay 674 is deenergized, whereby the switch S is opened and the supply of current to the torque motor 772 is interrupted. Thus, the brakes of the beamer are released after a short period of application corresponding to the delaying action of the time relay. Incidentally, it may be mentioned, that a restarting of the electromotor 4600 causes an energization of the electromagnet 410 of the time relay 412, which is arranged in the motor start holding circuit, so that the switch 424 is again closed so as to render the brake circuit ready for the next braking operation in the manner described above.

The torque motor 772 is connected with the three phases of the A. C. supply line in the following manner: A line 420 leading from the junction point 422 to the junction point 506, a line 660 leading from the junction point 506 to the terminal 662 of the torque motor, said line including the pole S₁ of the triple pole switch S controlled by the electromagnet 672 and a first coil of the adjustable resistor 658 controlled by the pressure roll assembly 738, 1320 in dependence on the increase of the diameter of the winding on the beam; the line 428 leading from the junction point 426 to the junction point 510; a line 664 leading from the junction point 510 to the terminal 666 of the torque motor, said line including the pole S₂ of the switch S and a second coil of the adjustable resistor 658; the portion of the line 516 leading from the junction point 514 to the junction point 517, a line 668 leading from the junction point 517 to the terminal 670 of the torque motor, said line including the pole S₃ of the switch S and a third coil of the adjustable resistor 658. Thus, above described closing of the switch S by the electromagnet 672 causes a supply of alternating current to the torque motor 772, and the output of the torque motor is controlled by the adjustable resistor 658 in dependence on the diameter of the winding on the beam.

In the embodiment shown in Fig. 10, the electrical control system for the beamer is also equipped with an electrical brake, which becomes effective in addition to the mechanical brakes of the beamer actuated by the torque motor. For this purpose an adjustable dynamic braking resistor 684 is connected with the junction points 686 and 688 in the lines 362 and 372 of the main motor circuit by means of a line 690 including a switch 692 of a dynamic resistor relay 694. The electromagnet 696 of said relay 694 is arranged in parallel to the electromagnet 434 of the brake control relay 432 by means of lines 698. Therefore, the electromagnet 696 is excited and the switch 692 controlled by said electromagnet is open as long as the motor start holding circuit is closed and the electromotor 4600 drives the beam. As soon as the motor start holding circuit is interrupted in any of above described manners and the electromagnet 434 of the brake control relay 432 and the electromagnet 696 of the dynamic resistor relay 694 are deenergized, the switch 692 is closed and shorts the electromotor 4600 through the dynamic braking resistor 684, whereby an additional braking of the rotating mass of the beam connected with

the electromotor is obtained. The above described electrical brake including the adjustable dynamic braking resistor 684 may be used for an additional adjustment of the braking time required for bringing the mass of the beam to a standstill. Of course, the electrical brake may be entirely omitted, if desired.

As pointed out above, the automatic motor field rheostat 2580 controlled by the pressure roll assembly 738, 1320 in dependence on the increase of the diameter of the winding on the beam 4400 is arranged in the motor start holding circuit. Said motor field rheostat 2580 serves to cause an automatic reduction of the operating speed of the electromotor 4600 from a predetermined normal degree to a lower degree, so that a substantially constant travelling speed of the warp yarns resulting in a substantially constant tension in the yarns is obtained during the operation of the beamer. The normal travelling speed of the warp yarns is determined by said predetermined normal operating speed of the electromotor 4600 at the beginning of the beaming operation after the termination of the inching speed operation. Different yarns of different size and/or material require different travelling speeds to obtain the proper tension in the yarns. For this purpose, an additional electrical adjusting means or a generator field hand rheostat 3500 is mounted on a side wall of the beamer as shown in Figs. 1 and 4. The generator field hand rheostat 3500 may be used for a manual adjustment of said predetermined normal operating speed of the electromotor 4600 driving the beam and is arranged in the following normal speed control circuit: A line 444 leads from the end 445 of the coil 446 of the generator field hand rheostat 3500 to the pole M₂ of the main switch M, which in turn is connected with the junction point 448 in the line 388 by a line 450. The line 452 leads from said junction point 448 to the terminal 454 of the exciter 456. The terminal 458 of the exciter 456 is connected with one end of the field 460 of the D. C. generator 356 by a portion of the line 478 at the point 470. The other end of the field 460 is connected with the movable finger 464 of the hand rheostat 3500 through a line 466 including the speed control switch 468 of the warp brake relay 408. The end 447 of the coil 448 of the hand rheostat 3500 is connected with the junction point 470 through a line 472 leading to a junction point 474, through the portion of the line 398 connecting the junction 474 with a junction point 476, and through a portion of the line 478 connecting the junction point 476 with the junction point 470. As will be apparent from the described connections, the generator field 460 is energized by the exciter 456 which generates a constant potential. This potential is applied to a circuit comprising the left-hand portion (as viewed in Fig. 10) of the coil 446 of the generator field hand rheostat 3500 in parallel with the generator field 460 and the right-hand portion of the coil 446 of the generator field hand rheostat 3500 in series with these. A manual movement of the finger 464 across the generator field hand rheostat 3500 will cause a variation of the potential applied to the generator field 460 and, consequently, a variation of the current flowing through this generator field during the normal operation of the beamer. If, for example, the finger 464 is moved to the right, the energization of the generator field and the current flowing through said generator field are increased. Therefore, the current flowing

through the generator field is in proportion to the setting of the finger 464 of the hand rheostat 3500. This current, when high, gives a high voltage generated by the generator 356, when low, a low voltage generated, i. e. the potential generated by the generator is variable. The normal operating speed of the electromotor 46 at the beginning of the beaming operation after the termination of the inching speed operation is determined by the voltage of the direct current supplied to the electromotor, and, therefore, the normal operating speed of the electromotor may be easily adjusted by means of the generator field hand rheostat 3500.

As will be apparent from the foregoing, the generator field hand rheostat 3500 serves to adjust a predetermined normal operating speed of the electromotor 4600 of the beamer by hand, while the automatic motor field rheostat 2580 controlled by the pressure roll assembly 738, 1320 serves to automatically reduce the operating speed of the electromotor 4600 from said predetermined normal value to a lower value, so as to obtain a substantially constant tension in the yarns and/or constant travelling speed of the yarns respectively during the beaming operation. For a better explanation of my invention, the operation of the electromotor at said normal operating speed and at said reduced lower operating speed shall be termed the "normal high speed operation" of the electromotor in contrast to the very low speed operation or the so-called "inching speed operation" of the electromotor 4600, which is desirable under certain circumstances, for example, first, after the occurrence of a yarn breakage and the tying of the ends of the broken yarns, until the slack in the yarn or yarns has been taken up, or, second, at the beginning of a beaming operation, until a few layers of the winding have been wound on the beam.

The operation of the electromotor 4600 at "normal high speed operation" or at "inching speed operation" is controlled by the speed control switch 468 of the warp brake relay 408, which in turn is controlled by the control needles 4160 in the creel and/or by the inching speed switch 4900 on the beamer as will be described hereinafter. The speed control switch 468 is arranged in the normal speed control circuit including the generator field 460 and the generator field hand rheostat 3500 as described above. As long as the electromagnet 482 of the warp brake relay 408 is deenergized and the speed control switch 468 is closed, the electromotor 4600 may operate at normal high speed operation as described above. If, however, the speed control switch 468 is opened by an energization of the electromagnet 482 upon the dropping of a control needle 4160 owing to a breakage of yarn or upon the manual closing of the inching speed switch 4900, the generator field hand rheostat 3500 is disconnected from the generator field 460. An adjustable inching resistor 484 is arranged in series with the generator field 460 between the junction points 486 and 488 in the lines 466 and 444 respectively, so that, after the above mentioned disconnection of the generator field hand rheostat 3500 from the generator field 460, the current now may flow through the following inching speed control circuit: exciter 456, line 462, generator field 460, line 466, inching resistor 484, line 444, pole M₃, lines 450, 452. Therefore, under these circumstances and upon closure of the main switch M including the pole M₃, a current proportional to the combined resistance of the generator field

460 and the adjustable inching resistor 484 arranged in series with each other will pass through the generator field, setting up a flux which will produce the low voltage required for the low inching speed and the low torque of the electromotor 4600. It may be mentioned, that this flux, and therefore voltage and torque, are independent of the setting of the generator field hand rheostat 3500, as the latter is disconnected from the generator field 460. On the other hand, the generator field hand rheostat 3500 will be reconnected with the generator field 460 and a further operation of the electromotor 4600 at inching speed will be rendered impossible as soon as the speed control switch 468 controlled by the electromagnet 482 of the warp brake relay 408 will be returned into closing position upon a return of the control needle 4160 and/or the inching speed switch 4900 into open position.

Now, the circuits controlling the electromagnet 482 of the warp brake relay 408 containing the speed control switch 468 will be described.

The electromagnet 482 is arranged in the following warp brake relay control circuit: A line 568 connects the electromagnet 482 with the left-hand terminal of a switch 566 controlled by an electromagnet 556 of a warp brake sensitive intermediate relay 558; a line 572 leads from the right-hand terminal of said switch 566 to a junction point 570 in a line 550; the portion of the line 550 that leads from said junction point 570 to the terminal 804 of the secondary winding 546 of the lower voltage transformer 548; the secondary winding 546; a line 544 leading from the secondary winding 546 to the junction point 542, and the portion of a line 540 that leads from said junction point 542 to the electromagnet 482. Obviously, the electromagnet 482 of the warp brake relay 408 will be excited for opening the speed control switch 468, as soon as the electromagnet 556 of the warp brake sensitive intermediate relay 558 is excited for closing the switch 566.

The electromagnet 556 is arranged in the following needle control circuit: A line 630 connecting the electromagnet 556 with one terminal of a creel terminal base 632 mounted on the creel 2400; the control needle 4160, a line 634 leading from the other terminal of the creel terminal base 632 to the junction point 534; the line 536 connecting the junction point 534 with the junction point 538; the portion of the line 540 that connects the junction point 538 with the junction point 542; the line 544; the secondary winding 546 of the low voltage transformer 548; the line 550 up to the junction point 552; and a line 554 connecting the junction point 552 with the electromagnet 556. Although Fig. 10 diagrammatically illustrates only one control needle 4160 arranged on the creel terminal base 632, the creel carries a plurality of control needles 4160, one for each warp yarn 3400 to be drawn from a bobbin 3600 as shown in Fig. 1.

If one or more control needles 4160 of the creel drop owing to a breakage of yarn, the above described needle control circuit is closed, whereby the electromagnet 556 of the warp brake sensitive intermediate relay 558 is excited, so that the switch 566 is closed which in turn causes an excitement of the electromagnet 482 of the warp brake relay 408 resulting in an opening of the speed control switch 468 and of the circuit opening switch 406. The latter causes an intermediate stoppage of the beamer and application of

the brakes as described above. The brakes are released after a short period of application as described above. The switches 406 and 468, however, remain in open position after the tying of the ends of the broken yarn as long as one or more control needles 4160 are in dropped position due to slack in a yarn or yarns. Thus, the electromotor 4600 can be restarted at inching speed only by closing the start switch 3820 by hand and holding same in closed position, until the slack in the yarns has been taken up; as soon as, upon removal of the slack in the yarns, all control needles 4160 are again in their open position and the switches 406 and 468 are again closed, the start switch 3820 may be released for return into the neutral position shown in Fig. 7 by the action of the spring 760, whereupon the operation of the electromotor 4600 will continue at normal high speed, as now the motor start holding circuit including the switch 406 and the normal speed control circuit including the switch 468 are closed.

The manual inching speed switch 4900 is connected with the warp brake relay control circuit in parallel to the switch 566 of the warp brake sensitive intermediate relay 558 by lines 652. Thus, a closing of the inching speed switch 4900 may also cause an energization of the electromagnet 482 of the warp brake relay 408 and an opening of the switches 406 and 468. As pointed out above, the inching speed switch 4900 is normally in the neutral position shown in Fig. 7, wherein the spring acting on its control element 4910 holds the inching speed switch in open position. If, for the start of the beamer, the shipper rod 758 (Fig. 7) is turned in the direction of the arrow C into the intermediate position described above, both, the start switch 3820 and the inching speed switch 4900 are closed. Although the closing of the inching speed switch 4900 causes an opening of the circuit opening switch 406 arranged in the motor start holding circuit, the electromotor 4600 may be operated as long as the start switch 3820 arranged in the motor start circuit is held in closing position; but, the electromotor 4600 may be operated at inching speed only, as, under these circumstances, the speed control switch 468 is also held in open position by the closing of the inching speed switch 4900. As soon as, after the laying of a few windings on the beam, the shipper rod 758 is turned further from the intermediate position in the direction of the arrow C into the extreme active position, the inching speed switch 4900 is opened, while the start switch 3820 remains in closed position. The opening of the inching speed switch 4900 causes an automatic closing of the speed control switch 468 resulting in an automatic increase of the speed of operation of the electromotor 4600 to normal high speed operation. Furthermore, the opening of the inching speed switch 4900 results in the closing of the motor start holding circuit by the switch 406. Now, the shipper rod 758 may be released for a return into the neutral position by the action of the spring 760; the operation of the electromotor 4600 continues at normal high speed, as the switches 406 and 468 controlled by the electromagnet 482 in response to the opening of the inching speed switch 4900 are in closed position.

The operation of the electromotor 4600 is arrested and the brakes of the beamer are applied, when the shipper rod 758 is turned from the neutral position in the direction of the arrow F (Fig. 7) for an opening of the stop switch 4140, or when a control needle 4160 drops and causes an opening of the switch 406, or when the safety switch 650

is opened; in all these cases the motor start holding circuit is interrupted.

Furthermore, an interruption of the motor start holding circuit resulting in a stoppage of the electromotor 4600 and an application of the brakes takes place, when the normally closed switch 4180 of the shut off counter 9800 is opened after the performance of a predetermined number of revolutions of the beam and causes an opening of the switch 402 of the counter stop relay 404. Reference is had to my Patent #2,324,611 relating to "An electrical control system for a warping or beaming plant," issued on July 20, 1943, for a full description of the circuits connecting the shut off counter 9800 with the counter stop relay 404.

Moreover, reference is had to the same Patent #2,324,611 issued on July 20, 1943 for a detailed description of the "reset station" 654 with the reset push button 6160 and the pilot lamp 6240 and their electrical connections with the electrical control system. Incidentally, it may be mentioned, that the electromagnet 606 of the shut off counter relay 572 is excited and the switch 580 of said relay is opened and the switch 581 of said relay is closed when the electrical control system is set for an operation of the beaming plant upon an actuation of the spring loaded reset push button 6160. The "reset-station" 654 is mounted on the extension 974 of the side wall of the beamer as best shown in Figs. 1 and 4.

The reversible motor 4800 for moving the beamer sidewise in one direction or the other is electrically connected through lines 522, 524, 526 with a hand reversing switch 4960 mounted on the wall of the beamer. The reversing switch 4960 is electrically connected with the junction points 506, 510, 517 through lines 508, 512 and 516.

6480 indicates an electrical control box secured to the frame of the beamer. Said electrical control box may be used for the reception and mounting of various instruments, for example the warp brake relay 408, the warp brake sensitive intermediate relay 558, the shut off counter relay 582, the time delay relay 412, the low voltage transformer 548, the counter stop relay 404, the brake control relay 432 and the motor field vibrating relay 366.

Figs. 11 and 12 illustrate a different mechanism of a beamer for rotating the beam. In these figures, parts of the beamer corresponding to equal parts of the beamer shown in Figs. 1-4 are indicated by the same reference numerals. While according to Figs. 1-4 the beam is equipped with a gear 714 for engagement with a driving pinion 712 and with a disc 734 for engagement with a driving friction pulley 816, the gear and disc as well as the pinion and the pulley being arranged next to each other, the beam 4400 shown in Figs. 11 and 12 has only a grooved friction disc 735 for operative engagement with a driving wedge friction pulley 817 keyed to the driving shaft 702 driven by the electromotor 4600 through the medium of the step pulleys 706 and 710 and the belt 704. The brake-band 85 is connected with the braking lever 786 at 82 and pivoted to the bracket 878 at 788 is wedge-shaped for proper engagement with the groove of the grooved disc or braking drum 735. All the remaining parts of the beamer may be of the same form and arrangement as shown in Figs. 1-4.

When the beam 4400 shown in Figs. 11 and 12 is lifted by the swingable support 716 into the operating position, the cam 858 cooperating with the axle or rod 854 of the beam guides the beam in such a manner, that its grooved friction disc

735 comes into proper engagement with the driving wedge friction pulley 817.

The mechanism shown in Figs. 11 and 12 has the advantage of the elimination of a noisy gear drive. Furthermore, the arrangement of the beamer adjusting means generally indicated by 888 in Figs. 2 and 4 may be omitted, as the engagement of the wedge friction pulley 817 with the grooved friction disc 735 also causes an adjustment of the beam 4400 in the direction of its longitudinal axis. Moreover, the mechanism shown in Figs. 11 and 12 allows for a smaller width of the beamer.

The mechanism shown in Figs. 11 and 12 has only a single driving wedge friction pulley 817 on one side of the beamer for cooperation with the grooved friction disc 735 of the beam. If desired, however, two driving wedge friction pulleys may be keyed to a transverse driving shaft substituted for the stub shaft 702 shown in the drawings, one driving wedge friction pulley being arranged on each side of the beamer. In such a case, the beam is provided with a complementary grooved friction disc at each end thereof.

Although preferably the individual objects of the invention are applied to a beaming plant in the combination described, the individual objects of the invention may be applied individually or in partial combination to beamers. It is emphasized that the merits of the invention are not limited to the described and illustrated combination, but the individual objects per se also have inventive merits.

I have described preferred embodiments of my invention, but it is clear that numerous changes and omissions may be made without departing from the spirit of my invention.

What I claim is:

1. A beamer comprising: a drive for rotating a beam, a pair of spaced supporting members for carrying the beam, each of said supporting members being swingably mounted on the beamer, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the beam is disengaged from said drive, into an upper operative position, wherein the beam is coupled with said drive, and coupling means associated with the supporting members for securing same to a stationary member of the beamer when the supporting members are swung into said upper operative position.

2. A beamer comprising: a drive for rotating a beam, a pair of spaced supporting members for carrying the beam, each of said supporting members being swingably mounted on the beamer, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the beam is disengaged from said drive, into an upper operative position, wherein the beam is coupled with said drive, and a latching mechanism associated with each supporting member, each latching mechanism including a first engaging means arranged on the supporting member and a second engaging means arranged on a stationary member of the beamer, said engaging means of each latching mechanism being capable of engagement with each other for connecting the supporting members with the stationary member of the beamer when the supporting members are swung into said upper operative position.

3. A beamer comprising: a drive for rotating a beam, a pair of spaced supporting members

for carrying the beam, each of said supporting members being swingably mounted on the beamer, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the beam is disengaged from said drive, into an upper operative position, wherein the beam is coupled with said drive, said adjusting means including a self-locking worm, and coupling means associated with the supporting members for securing same to a stationary member of the beamer when the supporting members are swung into said upper operative position.

4. A beamer comprising: a drive for rotating a beam, said drive including a driving shaft carrying a driving element for operative engagement with the beam, said driving shaft being journaled in stationary bearings of the beamer, a pair of spaced supporting members for carrying the beam, each of said supporting members being swingably mounted on a stationary point of the beamer, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the beam is disengaged from said driving element, into an upper operative position, wherein the beam is operatively engaged with said driving element, and a guide arranged for cooperation with the beam for guiding same into the proper engaging position with said driving element during the swinging movement of the supporting members from the lower position into the upper operative position.

5. A beamer comprising: a drive for rotating a beam having an axle, said drive including a driving shaft carrying a driving element for operative engagement with the beam, said driving shaft being journaled in stationary bearings of the beamer, a pair of spaced supporting members for carrying the beam, each of said supporting members being swingably mounted on a stationary point of the beamer, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the beam is disengaged from said driving element, into an upper operative position, wherein the beam is operatively engaged with said driving element, and cam means arranged for cooperation with the axle of the beam for guiding the beam into the proper engaging position with said driving element during the swinging movement of the supporting members from the lower position into the upper operative position.

6. A beamer comprising: a drive for rotating a beam having an axle and a gear, said drive including a driving shaft carrying a driving pinion for operative engagement with said gear, said driving shaft being journaled in stationary bearings of the beamer, a pair of spaced supporting members for carrying the beam, each of said supporting members being swingably mounted on a stationary point of the beamer, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the gear of the beam is disengaged from said driving pinion, into an upper operative position, wherein the gear of the beam is operatively engaged with said driving pinion, and cam means arranged for cooperation with the axle of the beam for guiding the beam into the proper engaging position of its gear with said driving pinion during the swinging movement of the supporting members from the lower position into the upper operative position.

7. In a beamer as claimed in claim 6, said cam means being of such a shape, that, during the movement of the supporting members into the upper operative position, they first lift the teeth of the gear to a certain extent above the teeth of the driving pinion, whereupon they lead the teeth of the gear into engagement with the teeth of the driving pinion in substantially radial direction.

8. A beamer comprising: a drive for rotating a beam, said drive including a driving shaft carrying a driving element for operative engagement with the beam, said driving shaft being journaled in stationary bearings of the beamer, a pair of spaced supporting members for carrying the beam, each of said supporting members being swingably mounted on a stationary point of the beamer, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the beam is disengaged from said driving element, into an upper operative position, wherein the beam is operatively engaged with said driving element, a guide arranged for cooperation with the beam for guiding same into the proper engaging position with said driving element during the swinging movement of the supporting members from the lower position into the upper operative position, and locking means associated with the supporting members for holding same in said upper operative position.

9. A beamer comprising: a drive for rotating a beam, said drive including a driving shaft carrying a driving element for operative engagement with the beam, said driving shaft being journaled in stationary bearings of the beamer, a pair of spaced supporting members for carrying the beam, each of said supporting members being swingably mounted on the beamer and being coaxially arranged with said driving shaft, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the beam is disengaged from said driving element, into an upper operative position, wherein the beam is operatively engaged with said driving element, and locking means associated with the supporting members for holding same in said upper operative position.

10. A beamer comprising: a drive for rotating a beam, said drive including a driving shaft carrying a driving element for operative engagement with the beam, said driving shaft being journaled in stationary bearings of the beamer, a pair of spaced supporting members being swingably mounted on the beamer and being coaxially arranged with said driving shaft, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the beam is disengaged from said driving element, into an upper operative position, wherein the beam is operatively engaged with said driving element, and coupling means associated with the supporting members for securing same to a stationary member of the beamer when the supporting members are swung into said upper operative position.

11. A beamer comprising: a drive for rotating a beam, said drive including a driving shaft carrying a driving element for operative engagement with the beam, said driving shaft being journaled in stationary bearings of the beamer, a pair of spaced supporting members being swingably mounted on the beamer and being coaxially arranged with said driving shaft, adjusting means

associated with said supporting members for swinging same from a lower beam receiving position, wherein the beam is disengaged from said driving element, into an upper operative position, wherein the beam is operatively engaged with said driving element, and a latching mechanism associated with each supporting member, each latching mechanism including a first engaging means arranged on the supporting member and a second engaging means arranged on a stationary member of the beamer, said engaging means of each latching mechanism being capable of engagement with each other for connecting the supporting members with the stationary member of the beamer when the supporting members are swung into said upper operative position.

12. A beamer comprising: a drive for rotating a beam having discs at its ends, said drive including a driving shaft carrying a driving friction pulley for operative engagement with a disc of the beam, said driving shaft being journaled in stationary bearings of the beamer, a pair of spaced supporting members, said supporting members being swingably mounted on the beamer and being coaxially arranged with said driving shaft, each supporting member carrying a bearing roller at its free end for contacting engagement with a disc of the beam, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the disc of the beam is disengaged from said driving friction pulley, into an upper operative position, wherein the disc of the beam is in operative engagement with said driving friction pulley, and locking means for holding the supporting members in said upper operative position.

13. A beamer comprising: a drive for rotating a beam having discs at its ends, said drive including a driving shaft carrying a driving friction pulley for operative engagement with a disc of the beam, said driving shaft being journaled in stationary bearings of the beamer, a pair of spaced supporting members, said supporting members being swingably mounted on the beamer and being coaxially arranged with said driving shaft, each supporting member carrying a bearing roller at its free end for contacting engagement with a disc of the beam, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the disc of the beam is disengaged from said driving friction pulley, into an upper operative position, wherein the disc of the beam is in operative engagement with said driving friction pulley, and coupling means associated with the supporting members for securing same to a stationary member of the beamer when the supporting members are swung into said upper operative position.

14. A beamer comprising: a drive for rotating a beam having discs at its ends, said drive including a driving shaft carrying a driving friction pulley for operative engagement with a disc of the beam, said driving shaft being journaled in stationary bearings of the beamer, a pair of spaced supporting members, said supporting members being swingably mounted on the beamer and being coaxially arranged with said driving shaft, each supporting member carrying a bearing roller at its free end for contacting engagement with a disc of the beam, adjusting means associated with said supporting members for swinging same from a lower beam receiving position,

tion, wherein the disc of the beam is disengaged from said driving friction pulley, into an upper operative position, wherein the disc of the beam is in operative engagement with said driving friction pulley, and a latching mechanism associated with each supporting member, each latching mechanism including a first engaging means arranged on the supporting member and a second engaging means arranged on a stationary member of the beamer, said engaging means of each latching mechanism being capable of engagement with each other for connecting the supporting members with the stationary member of the beamer when the supporting members are swung into said upper operative position.

15. A beamer comprising: a drive for rotating a beam having a gear and having discs at its ends, said drive including a driving shaft carrying a driving pinion for operative engagement with said gear and carrying a driving friction pulley for operative engagement with a disc of the beam, a pair of spaced supporting members, said supporting members being swingable about the axis of said driving shaft, each supporting member carrying a bearing roller at its free end for contacting engagement with a disc of the beam, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the disc of the beam is disengaged from said driving friction pulley, into an upper operative position, wherein the disc of the beam is in operative engagement with said driving friction pulley, and a guide arranged for cooperation with the beam for guiding same into the proper engaging position of its gear and disc with said driving pinion and said driving friction pulley during the swinging movement of the supporting members from the lower position into the upper operative position.

16. A beamer comprising: a drive for rotating a beam having an axle and a gear and having discs at its ends, said drive including a driving shaft carrying a driving pinion for operative engagement with said gear and carrying a driving friction pulley for operative engagement with a disc of the beam, a pair of spaced supporting members, said supporting members being swingable about the axis of said driving shaft, each supporting member carrying a bearing roller at its free end for contacting engagement with a disc of the beam, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the disc of the beam is disengaged from said driving friction pulley into an upper operative position, wherein the disc of the beam is in operative engagement with said driving friction pulley, and cam means arranged for cooperation with the axle of the beam for guiding the beam into the proper engaging position of its gear and disc with said driving pinion and said driving friction pulley, during the swinging movement of the supporting members from the lower position into the upper operative position, said cam means being of such a shape, that, during the movement of the supporting members into the upper operative position, they first lift the teeth of the gear to a certain extent above the teeth of the driving pinion, whereupon they lead the teeth of the gear into engagement with the teeth of the driving pinion in substantially radial direction.

17. In a beamer the combination of: a drive including a driving shaft, a beam, a wedge-friction-gear including two complementary elements, one of said complementary elements being se-

cured to said shaft, the other complementary element being secured to said beam, a pair of spaced supporting members for carrying said beam, each of said supporting members being swingably mounted on a stationary point of the beamer, and adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein said complementary elements are disengaged from each other, into an upper operative position, wherein said complementary elements are in operative engagement with each other.

18. A beamer comprising: a drive for rotating a beam having a grooved friction disc at at least one of its ends, said drive including a driving shaft carrying at least one driving wedge friction pulley for operative engagement with said grooved friction disc, a pair of spaced supporting members for carrying the beam, each of said supporting members being swingably mounted on a stationary point of the beamer, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the grooved friction disc of the beam is disengaged from said driving wedge friction pulley, into an upper operative position, wherein the grooved friction disc of the beam is in operative engagement with said driving wedge friction pulley, and coupling means associated with the supporting members for securing same to a stationary member of the beamer when the supporting members are swung into said upper operative position.

19. In a beamer the combination of: a drive including a driving shaft, a beam, a wedge-friction-gear including two complementary elements, one of said complementary elements being secured to said shaft, the other complementary element being secured to said beam, a pair of supporting members for carrying said beam, each of said supporting members being swingably mounted on a stationary point of the beamer, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein said complementary elements are disengaged from each other, into an upper operative position, wherein said complementary elements are in operative engagement with each other, and a latching mechanism associated with each supporting member, each latching mechanism including a first engaging means arranged on the supporting member and a second engaging means arranged on a stationary member of the beamer, said engaging means of each latching mechanism being capable of engagement with each other for connecting the supporting members with the stationary member of the beamer when the supporting members are swung into said upper operative position.

20. In a beamer the combination of: a drive including a driving shaft, a beam, a wedge-friction-gear including two complementary elements, one of said complementary elements being secured to said shaft, the other complementary element being secured to said beam, a pair of spaced supporting members for carrying said beam, each of said supporting members being swingably mounted on a stationary point of the beamer, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein said complementary elements are disengaged from each other, into an upper operative position, wherein said complementary elements are in operative engagement

with each other, said adjusting means including a self-locking worm, and coupling means associated with the supporting members for securing same to a stationary member of the beamer when the supporting members are swung into said upper operative position.

21. A beamer comprising: a drive for rotating a beam having discs at its ends, at least one of said discs being grooved, said drive including a driving shaft carrying at least one driving wedge friction pulley for operative engagement with the grooved disc of the beam, said driving shaft being journaled in stationary bearings of the beamer, a pair of spaced supporting members, each of said supporting members being swingable about the axis of said driving shaft, each supporting member carrying a bearing roller at its free end for contacting engagement with a disc of the beam, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the grooved disc of the beam is disengaged from said driving wedge friction pulley, into an upper operative position, wherein the grooved disc of the beam is in operative engagement with said driving wedge friction pulley, and a guide arranged for cooperation with the beam for guiding same into the proper engaging position of its grooved friction disc with said driving wedge friction pulley during the swinging movement of the supporting members from the lower position into the upper operative position.

22. A beamer comprising: a drive for rotating a beam, said drive including an electromotor, a pair of spaced supporting members for carrying the beam, each of said supporting members being swingably mounted on the beamer, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the beam is disengaged from said drive, into an upper operative position, wherein the beam is coupled with said drive, a safety switch in circuit with said electromotor for interrupting the supply of current to the electromotor when the safety switch is rendered in-

active and for permitting a supply of current to the electromotor when the safety switch is rendered active, a controlling element for the actuation of said safety switch, said controlling element being associated with a swingable supporting member and being arranged for rendering active said safety switch upon a swinging movement of said supporting member from said lower position into said upper operative position, and coupling means associated with the supporting members for securing same to a stationary member of the beamer when the supporting members are swung into said upper operative position.

23. A beamer comprising: a drive for rotating a beam, said drive including an electromotor and a driving shaft connected with said electromotor, said driving shaft carrying a driving element for operative engagement with said beam, a pair of spaced supporting members for carrying said beam, each of said supporting members being swingably mounted on the beamer and being coaxially arranged with said driving shaft, adjusting means associated with said supporting members for swinging same from a lower beam receiving position, wherein the beam is disengaged from said driving element, into an upper operative position, wherein the beam is coupled with said driving element, a safety switch in circuit with said electromotor for interrupting the supply of current to the electromotor when the safety switch is rendered inactive and for permitting a supply of current to the electromotor when the safety switch is rendered active, a controlling element for the actuation of said safety switch, said controlling element being associated with a swingable supporting member and being arranged for rendering active said safety switch upon a swinging movement of said supporting member from said lower position into said upper operative position, and coupling means associated with the supporting members for securing same to a stationary member of the beamer when the supporting members are swung into said upper operative position.

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