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(54) **Titre : MACHINE DE PRODUCTION DE PANNEAU A LAME REGLABLE**
(54) **Title: ADJUSTABLE BLADE PANELING MACHINE**

(57) **Abrégé/Abstract:**
A paneling machine (1) is described comprising a counter-blade (3) and a blank holder (5) shaped so as to lock a sheet panel (4) to be bent. The paneling machine (1) further comprises a "C" type blade holder (7), the terminals of which can be coupled to at least a first (9) and a second (11) bending blade. A machine (1) further comprising actuating means (100) of the blade holder structured to orient said second blade (11) in two different working positions (P2, P3). (Figure 4)

"Adjustable blade paneling machine"

* * * *

ABSTRACT

A paneling machine (1) is described comprising a counter-blade (3) and a blank holder (5) shaped so as to lock a sheet panel (4) to be bent. The paneling machine (1) further comprises a "C" type blade holder (7), the terminals of which can be coupled to at least a first (9) and a second (11) bending blade. A machine (1) further comprising actuating means (100) of the blade holder structured to orient said second blade (11) in two different working positions (P2, P3). (Figure 4)

"Adjustable blade paneling machine"

* * * *

The present invention relates to an adjustable blade paneling machine.

Sheet panel bending machines, also known as paneling machines, used to bend sheet panels to a required profile, are known.

The sheet panel inserted in the paneling machine is locked between two members, a lower one named counter-blade, used to support the sheet panel, and a mobile upper member, named blank holder. The blank holder may be actuated vertically between a first upper position, which allows to introduce the sheet panel in the machine between blank holder and counter-blade, and a second lower position to lock the panel and allow the machining thereof; paneling machines are equally known which also include the horizontal movement of the blank holder with respect to the counter-blade to allow further bending combinations.

During the step of bending, the panel is positioned so that one end thereof, the one intended to be bent, protrudes horizontally towards the inside of the machine. The panel end is bent either upwards or downwards by means of the bending blades fitted on an essentially "C"-shaped support, named blade holder, which is actuated by hydraulic pistons or cam-connecting rod systems. Such blade presses on the protruding end of the sheet to bend it by levering on counter-blade and blank holder.

The current solutions display limits in the case of particular bending profiles, characterized, for instance, by a small counter-bend interposed between two much wider bends. Indeed, the known paneling machines may not be able to carry out the described bending sequence for reasons of mechanical interference between panel and bending blades.

In view of the prior art, it is the object of the present invention to provide an adjustable blade paneling machine which can overcome this constraint.

In accordance with the present invention, said object is reached by

means of a paneling machine as disclosed in claim 1.

The features and the advantages of the present invention will be apparent from the following detailed description of practical embodiments thereof, shown by way of non-limitative example in the accompanying drawings, in which:

figure 1 is a side view of an adjustable blade paneling machine according to a first embodiment of the present invention;

figure 2 is a side view of an adjustable blade paneling machine according to a second embodiment of the present invention with blade holder in a first working position;

figure 3 shows the paneling machine according to a second embodiment of the present invention with blade holder in a second working position;

figure 4 shows the paneling machine according to a second embodiment of the present invention with blade holder in a third working position;

figure 5 shows a sheet panel with a given bending profile;

figure 6 shows a part of a known paneling machine in greater detail;

figure 7 shows a part of the paneling machine in figure 4 in greater detail;

figures 8-10 show the kinematism of the paneling machine in figure 1 in working configurations P1-P3;

figures 11a-11b show a bending sequence of a sheet panel with the paneling machine in the second working position;

figures 12a-12b show a bending sequence of a sheet panel with the paneling machine in the third working position;

figure 12c shows a panel which can be made using the bending machine according to the invention, while figure 11c shows the net of the panel in figure 12c;

figures 13a-13b are perspective views of the paneling machine in the

third and second working position;

figures 14a-14c and 15a-15c show two limit cases of downward bending of the sheet panel which can be obtained only with the blade holder rotated with respect to the usual condition.

Figure 1 shows a first embodiment of a paneling machine 1 comprising a frame 2 and a counter-blade 3 integral to said frame 2 which supports a sheet panel 4 to be bent. A blank holder 5 is arranged above said counter-blade 3 and fixed to means 6 for vertically actuating said blank holder 5, said means 6 being adapted to press the blank holder 5 against the blade holder 3 to lock the panel 4. The blank holder 5 may also include a horizontal type adjustment with respect to the blade holder 3 to allow to make further bend combinations of the sheet panel 4.

The paneling machine 1 comprises an essentially "C"-shaped (or "C" type) blade holder 7 with two terminals 8 and 10 and a connection portion 30 of the terminals 8, 10. A first bending blade, or lower blade, 9, adapted to bend the sheet panel 4 upwards, is connected to a first terminal, or lower terminal, 8 of said blade holder, while a second bending blade, or upper blade, 11, adapted to bend downwards, is normally coupled to a second terminal, or upper terminal, 10 of the blade holder 7.

Actuating means 100 of the blade holder 7 configured to rotate and translate the blade holder 7 are provided.

The actuating means 100 of the blade holder 7 include a first member 14, 15, 16, 12, 120 hinged to a terminal of the blade holder 7, preferably to the lower terminal 8 of the blade holder 7 by means of a pin 12, and a second member 20, 21, 13, 130 hinged to the connection portion 30 of the blade holder by means of a pin 13.

The blade holder 7 comprises extensions 120 of the terminal 8 and 130 on the part of the connection 30 for engaging the respective pins 12, 13; connecting rods 14, 20 are engaged on one end to the pins 12, 13, and the other end to the frame 2 of the paneling machine. In particular, the

connecting rod 14 is rotationally connected to a cam comprising a pin 15 arranged in the peripheral part of a motorized disc 16; the latter is preferably arranged in the lower part A of the frame 2 of the paneling machine, preferably in the frame portion 2 beneath the counter-blade 3. The connecting rod 20 is rotationally connected to a pin 21 connected to the frame 2, preferably in the central part A1 of the frame 2, and preferably so that the continuation line 40 of the work top, which corresponds to the sheet panel 4 when said panel 4 is locked on the counter-blade 3 by the blank holder 5, essentially passes through the pin 21. The pin 21 is motorized in the case of the embodiment shown in figure 1. The rotation axes of the blade holder, of the pins 21, 13, 12 and of the disc 16 are orthogonal to the direction of movement of the blade holder 7 and are preferably aligned with the bending direction of the panel 4.

The rotation of the cam 15, 16 allows the rotation of the blade holder 7, while the combination of the rotation of the cam 15, 16 and of the rotation of the motorized pin 21 allows to translate the blade holder 7 itself either upwards/downwards, or rightwards/leftwards.

According to a second embodiment (figures 2-4), the actuating means 100 of the paneling machine 1 comprise a further member 130, 13, 17, 18, 19 hinged to the connection portion 30 of the blade holder by means of the pin 13. The further member comprises a connecting rod 17 which engages the pin 13 on one end and is rotationally connected to a cam on the other end, which comprises a pin 18 arranged on the edge of a motorized disc 19; the latter is preferable arranged in the upper part A2 of the frame 2 of the paneling machine, preferably in the portion of frame 2 above the counter-blade 3. A further member allows a better distribution of the actuating action to be exerted on the blade holder 7. The pin 21 is idle in this case, i.e. not motorized, and only has movement restraint functions. The rotation axes of the pin 18 and of the disc 19 are orthogonal to the direction of movement of the blade holder 7, and preferably aligned with the bending direction of the

panel 4.

In both embodiments, the actuating means 100 of the paneling machine are structured to implement configurations or working positions P1-P3 of figures 2-4, i.e. the positions in which the bending blade 9 or 11 is about to bend the sheet panel 4 with the sheet panel 4 itself locked between blank holder 5 and counter-blade 3, preferably arranged horizontally. In particular, the actuating means 100 are structured to orient the blade 11 in a working position P2 and in another working position P3 different from working position P2.

In both embodiments of the paneling machine 1, the blade 11 preferably has different shape from the known blades; in particular, the blade 11 is wedge-shaped and the end of the wedge coincides with the free end 111 of the blade. The angle Q of the wedge, i.e. the angle between the inner surface 140 (i.e. the surface which faces the connection portion 30 of the blade holder 7) and the outer surface 141 (i.e. the surface which faces the blank holder 5) is preferably comprised between 25° and 35°.

The combination of the rotation or rototranslation motion of the blade holder 7 caused by the actuating means 100 and the particular shape of the blade 11 with inner surface 140 and outer surface 141 allows the blade 11 to be oriented with the inner surface profile 140 or with the outer surface profile 141 essentially perpendicular to the plane identified by the panel 4 in working positions P3 and P2, as shown in figures 3 and 4.

In working configuration P1 (figure 2), the blade holder 7 is rotated by the actuating means 100 to allow the lower blade 9 to bend the sheet panel 4 in working position P1 upwards.

Again by means of the actuating means 100, the paneling machine may actuate working configuration P2 in figure 3, in which the blade holder 7 is rotated, again by the actuating means 100, to allow the upper blade 11 to bend the panel sheet 4 downwards. In both working positions P1 and P2, as shown in figure 8-9, the cam 15, 16 displaces the pin 12 on an arc of

circumference B1 having its centre in the rotation axis of the pin 13; the lower 9 and upper blades 11 are arranged in working positions P1 and P2 with the movement of the cam 15, 16 only, in which the pin 15 is moved without ever crossing the points of singularity identified by the theoretical line R passing through the rotation axes of the disc 16 and the pin 12. The oscillation of the blade holder 7 is due to a light rotation caused by the motorized pin 21 of the first embodiment or by the cam 18, 19 of the further member in case of the second embodiment of the invention; said slight rotation is required to implement small rightward/leftward or upward/downward movements with respect to the blade holder. In working position P2, the profile of the outer surface 141 of the blade 11 is essentially perpendicular to the sheet panel 4.

By means of the actuating means 100 and the particular shape of the blade 11, the paneling machine according to the invention can actuate working configuration P3 in figure 4 and figure 10, in which the blade holder 7 is rotated and translated vertically to make the bending blade 11 rotate with respect to a rotation axis I, orthogonal to the direction of movement of the blade holder 7, passing through the free end 111 thereof; in particular, the bending blade 11 rotates by an angle essentially equal to the angle Q (preferably comprised between 25° and 35°) of the wedge of the upper blade 11 with respect to working position P2 in figure 3. When going from working position P2 to working position P3, the pin 15 of the cam 15, 16 is moved without ever crossing the points of singularity identified by the theoretical line R passing through the rotation axes of the disc 16 and of the pin 12.

The actuating means 100 are adapted to translate the pin 13 vertically, which in figures 2 and 3 is beneath line 40 corresponding to the height of the sheet panel 4 in working position, i.e. beneath the sheet panel 4, while in figure 4 it is above line 40, i.e. above the sheet panel 4, in a position symmetric to the previous one. In this manner, the actuating means 100 are

adapted to translate and rotate the blade holder 7.

The combination of the rototranslation motion of the blade holder 7 caused by the actuating means 100 and the particular shape of the profile of the inner surface 140 of the blade 11 allows the blade 11 itself to be oriented with said inner surface profile 140 essentially perpendicular to the plane identified by the panel 4 in working position P3, as shown in figure 4. The movement of the blade holder 100 for going from working position P2 to working position P3 is due to the movement of the cam 15, 16 and to the rotation of the motorized pin 21, in the case of the first embodiment of the invention, or the movement of the cam 15, 16 and the movement of the cam 18, 19, in the case of the second embodiment of the invention, in which the pin 21 is idle. Again, the combination of the movement of the cam 15, 16 and of the rotation of the motorized pin 21, in the case of the first embodiment of the invention, or the combination of the movement of the cam 15, 16 and of the movement of the cam 18, 19, in the case of the second embodiment of the invention, allow the upper blade 11 to make small vertical or horizontal movements with respect to the blade holder 3.

The paneling machine in accordance with the invention allows to make bending profiles not achievable with paneling machines of the prior art.

Figure 5 shows, for example, a particular bending profile 300 for the panel sheet 4 which may be made by the paneling machine in accordance with the invention; said profile comprises a small width counter-bend “c” interposed between two greater width bends “a” and “b”.

We will assume to intend to make said profile 300 on the sheet panel 4 and that the paneling machine 1 has already made the upward bend “a”. Figure 6 shows the limits of a paneling machine according to the prior art to make the counter-bend “c” if the blade holder 7 is not translated vertically and rotated. Without the combination of a rotation and a vertical translation of the blade holder 7 it is not possible to perform the sequence of bends

according to the profile in figure 5 for reasons of mechanical interference between panel 4 and bending blade 11. Furthermore, again in figure 6, the limits of a paneling machine of the prior art are shown in making wide bend with a high edge sheet panel (shown with a dashed line) once again if the blade holder 7 is not translated vertically and rotated.

Figure 7 shows the overcoming of the constraint with the blade holder 7 in working position P3 translated and rotated by a given angle clockwise with respect to working position P2, and thus with a different orientation from that shown.

Figure 12c shows three orthogonal views of a panel 4, a detail which can only be made using the paneling machine described in this invention. Figure 11c is a flat development of the panel at hand with sequential numbering 150-155 of the bend sequence. In particular, the downward negative bend 155 on the side L4 of the panel may be made only with the blade holder in working position P2 not to interfere with the bends 150 and 151 already made on sides L1 and L2 adjacent to side L4 processed as shown in figures 11a-11b and shown in greater detail in the perspective view in figure 13b. In that case, the blade holder 7, rotated in working position P2, is positioned with the end 111 of the upper blade 11 in contact with the upper surface of the sheet panel 4 so that the profile of the outer surface 141 of the upper blade 11 is essentially perpendicular to the sheet panel 4 in working position. The perpendicularity of the outer surface 141 of the blade 11 avoids collision of the bends 150 151 already made on sides L1 and L2 of the panel.

Again with reference to figures 11c-12c, the counter-bend 153 on side L3 can be made with the bending sequence shown in figures 12a-12b. In this case, the blade holder 7 is translated and rotated by the actuating means 100 so that the end 111 of the upper blade 11 is in contact with the upper surface of the sheet panel 4 and the blade 11 itself is rotated with respect to the rotation axis I passing through the free end 111. The blade 11 performs a

rotation with respect to the working position P2 of the angle Q of the wedge of the blade 11, preferably comprised between 25° and 35°, so that the blade 11 is positioned by the blade holder 7 so that the profile of the inner surface 140 is essentially perpendicular to the sheet panel 4 in working position; in this manner, it is possible to make the bend 153 by pressing the blade 11 downwards on the panel 4. The perspective view in figure 13a shows this in greater detail. By comparing the perspective view in figure 13b the particular blade 11 with its wedge shape and the surfaces 140, 141 which depart from the end 11 form an angle Q preferably comprised between 25° and 35°. Surfaces 140, 141 are preferably flat.

Thus, the inclination of the blade 11 is such to allow the profile of its inner surface 140 to remain essentially perpendicular to the sheet panel 4 arranged on the horizontal plane.

Figures 14b-14c show the bending sequence which allows to make a wide bend "d" on a sheet panel 4 with a high upward edge "l" in working position P3; the blade holder 7 is translated and rotated by the actuating means 100 so that the end 111 of the upper blade 11 is in contact with the upper surface of the sheet panel 4 and the blade 11 itself is rotated with respect to the rotation axis I passing through its free end 111. The blade 11 rotates by the angle Q of the wedge of the blade 11, preferably comprised between 25° and 35° with respect to working position P2, so that the blade 11 is positioned by the blade holder 7 with the profile of the inner surface 140 essentially perpendicular to the sheet panel 4 in working position; so that it is possible to make the bend "d" by pressing the blade 11 downwards on the panel 4. Figure 14a clearly shows the interference condition of the panel blade with the blade holder in the usual position P2. Figures 15a-15c show a similar interference situation and how it can be overcome by rotating the blade holder with bend width "d" extended to the maximum possible.

In the operation of the machine 1 in the first and second embodiment alike (figures 1-4), the sheet panel 4 to be machined is locked between

counter-blade 3 and blank holder 5 but arranged so as to protrude towards the inside of the machine 1 near the blades 9 and 11 to allow bending; the blades 9, 11 work on the locked panel 4 until bending is completed. The blade holder 7 with its rotation movement guides the blades 9, 11 for making upwards and downward bends on the panel 4, respectively. In order to prevent collisions between one of the blades 9, 11 and a previously bent sheet panel 4 with an edge of a given dimension, the actuating means 100 are adapted to translate vertically and rotate the blade holder 7 through the combined movement of the connecting rods 14, 20 so that the blade 11 does not interfere with the high edge of the panel 4.

The actuating means 100 implement a kinematism with three degrees of freedom with only two actuators, i.e. the cam 15, 16 and the motorized pin 12 in the case of the first embodiment of the invention or the cams 15, 16 and 18, 19 in the case of the second embodiment of the invention. The blade holder 7, with the three degrees of freedom, may translate vertically or horizontally with respect to the counter-blade 3.

Having defined D1 and D2 as the distance between pin 21 and pin 13 and the distance between pin 13 and free end 111 of the blade 11, respectively (as shown in greater detail in figures 9-10), when going from working position P2 in figure 3 to working position P3 in figure 4, the motorized connecting rod-cam system 14-16, 17-19 (second embodiment of the invention) or the motorized connecting rod-cam system 14-16 and the motorized pin 12 (first embodiment of the invention) works on the blade holder 7 so that the pin 13 translates vertically upwards thus maintaining the distances D1 and D2 constant. In this manner, a rotation of the upper blade 11 is obtained which performs a rotation of the angle Q, i.e. an angle comprised between 25° and 35° on the rotation axis I passing through the free end 111 going from working position P2 to working position P3. The particular shape of the inner surface of the upper blade 111 allows the profile of the inner surface 140 to be essentially orthogonal to the sheet

panel 4 with the blade 11 in working position P3.

Thus, the actuating means 100 are structured to allow the rotation of the blade holder 7 along the arc of circumference B1 having its centre of gravity in the rotation axis of the pin 13 to go from working position P1 to working position P2 and vice versa; the actuating means 100 are further structured to allow the rotation and translation of the blade holder 7 so as to rotate the blade 11 on the though axis I passing through the end 111 of the angle Q to go from working position P2 to working position P3.

In the operation of the machine 1 (see figures 2, 3 and 4), in its second embodiment, the further connecting rod-cam system 17-19 supports the already present connecting rod-cam system 14-16 and move the blade holder 7 by working on the pin 13. The movement impressed on the blade holder 7 by the actuating means of the second embodiment of the invention remains unchanged with respect to the actuating means 100 of the first embodiment of the invention but exploits a less powerful motor.

The combinations of the movements described above and the possibility of rotating and translating of the blade holder 7 allow the paneling machine 1 to carry out a wide range of bending profiles.

CLAIMS

1. A paneling machine (1) comprising a counter-blade (3) and a blank holder (5) shaped so as to lock a sheet panel (4) to be bent in working position (P1-P3), said paneling machine (1) further comprising a "C"-shaped blade holder (7), which supports at least a first (9) and a second (11) bending blade at the terminals thereof, said machine (1) comprising actuating means (100) of the blade holder (7), said actuating means being structured to orient said second blade (11) in a first working position (P2) and in a second working position (P3) different from the first working position, characterized in that said actuating means of the blade holder (7) are adapted to operate on the blade holder (7) so that said second blade (11) can rotate about an axis (I) passing through the free end (111) thereof to go from the first working position (P2) to the second working position (P3).

2. A paneling machine (1) according to claim 1, characterized in that said second blade (11) is wedge-shaped and said actuating means of the blade holder (7) cause said rotation of the second blade (11) by an angle essentially equal to the angle (Q) of the wedge to go from the first working position (P2) to the second working position (P3).

3. A paneling machine (1) according to any one of the preceding claims, characterized in that said actuating means are adapted to rotate and translate said blade holder (7) and in that said second bending blade (11) is provided with a given conformation of the inner surface (140) which faces the blade holder so that the combination of the rototranslation movement of the blade holder and the given shape of the inner surface of said second blade allows said second blade to be oriented so that the inner surface profile thereof is essentially orthogonal to the sheet panel of said second working position (P3).

4. A paneling machine (1) according to any one of the preceding claims, characterized in that said actuating means are adapted to rotate said blade holder and in that said second bending blade (11) is provided with a

given shape of the outer surface (141) which faces the blank holder so that the combination of the rotation movement of the blade holder and the given conformation of the outer surface of said second blade allows said second blade to be oriented so that the outer surface profile thereof is essentially orthogonal to the sheet panel in said first working position (P2).

5. A paneling machine (1) according to any one of the preceding claims, characterized in that said actuating means (100) implement an actuation system of the blade holder with three degrees of freedom using only two actuators.

6. A paneling machine (1) according to any one of the preceding claims, characterized in that said actuating means of the blade holder (7) comprise at least one first member (12, 120, 14-16) hinged to a terminal (8, 10) of the "C"-shaped blade holder and a second member (20, 21, 13, 130) hinged to the connection portion (30) of the terminals (8, 10) of the "C"-shaped blade holder by means of a pin (13), said actuating means (100) being structured so that said pin can translate vertically from a position beneath the sheet panel (4) to a position above the sheet panel going from the first working position (P2) to the second working position (P3).

7. A paneling machine (1) according to claim 6, characterized in that said first member comprises a connecting rod (14) having one end connected to a motorized cam (15, 16) and the other end rotationally connected to said "C"-shaped terminal (8, 10) of the blade holder.

8. A paneling machine (1) according to claim 7, characterized in that said motorized cam (15, 16) is arranged in the lower part of the frame (2) of the paneling machine (1) beneath the counter-blade (3) and said terminal (8) of the "C"-shaped blade holder is the lower terminal of the blade holder which can be coupled to the first bending blade (9).

9. A paneling machine (1) according to claim 6, characterized in that said actuating means are configured so as to allow a vertical translation of said pin (13) while maintaining the distance (D2) between said pin (13) and

said free end (111) of the second blade (11) constant.

10. A paneling machine (1) according to claim 6, characterized in that said second member comprises a connecting rod (20) having one end connected to a further pin (21) and the other end rotationally connected to said connection portion (30) of the terminals (8, 10) of the "C"-shaped blade holder by means of said pin (13).

11. A paneling machine (1) according to claim 8, characterized in that said further pin (21) is arranged essentially at the height of the sheet panel (4) locked between counter-blade and blank holder.

12. A paneling machine (1) according to claim 11, characterized in that said further pin (21) is motorized.

13. A paneling machine (1) according to claim 10, characterized in that said further pin (13) is idle and said actuating means (100) comprise a third member comprising a connecting rod (17) having one end hinged to the connection portion (30) of the terminals (8, 10) of the "C"-shaped blade holder and the other end connected to a motorized cam (18, 19).

14. A paneling machine (1) according to claim 9, characterized in that said motorized cam (18, 19) of the third member is connected in the upper part (A2) of the frame (2) of the paneling machine (1) above the counter-blade (3).

15. A paneling machine (1) according to claim 6, characterized in that said actuating means (100) are further structured to move said blade holder (7) between said first working position (P2) of the second blade (11) and a further working position (P1) of the first blade (9) and vice versa by rotating the blade holder about an axis passing through said pin (13).

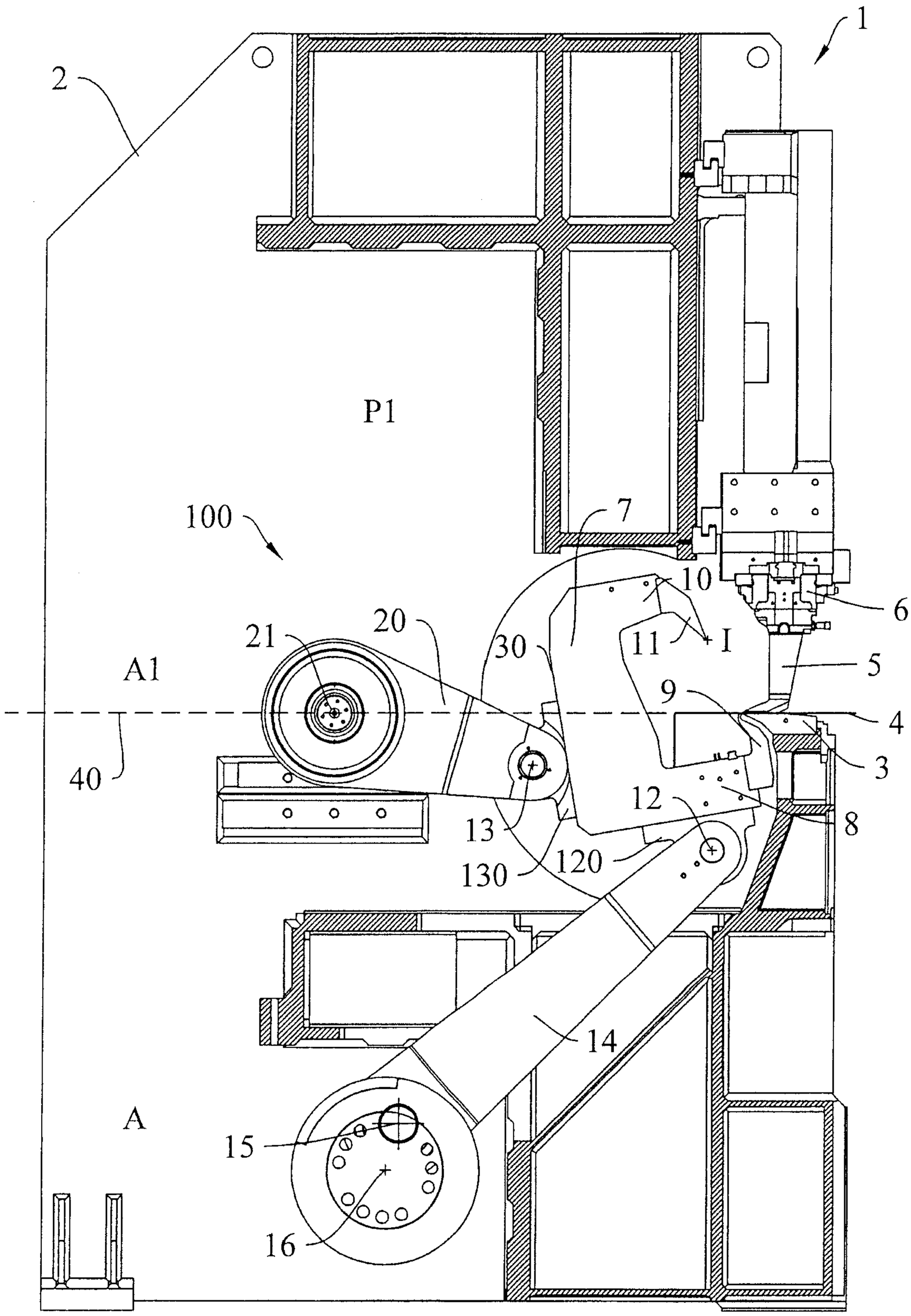


FIG.1

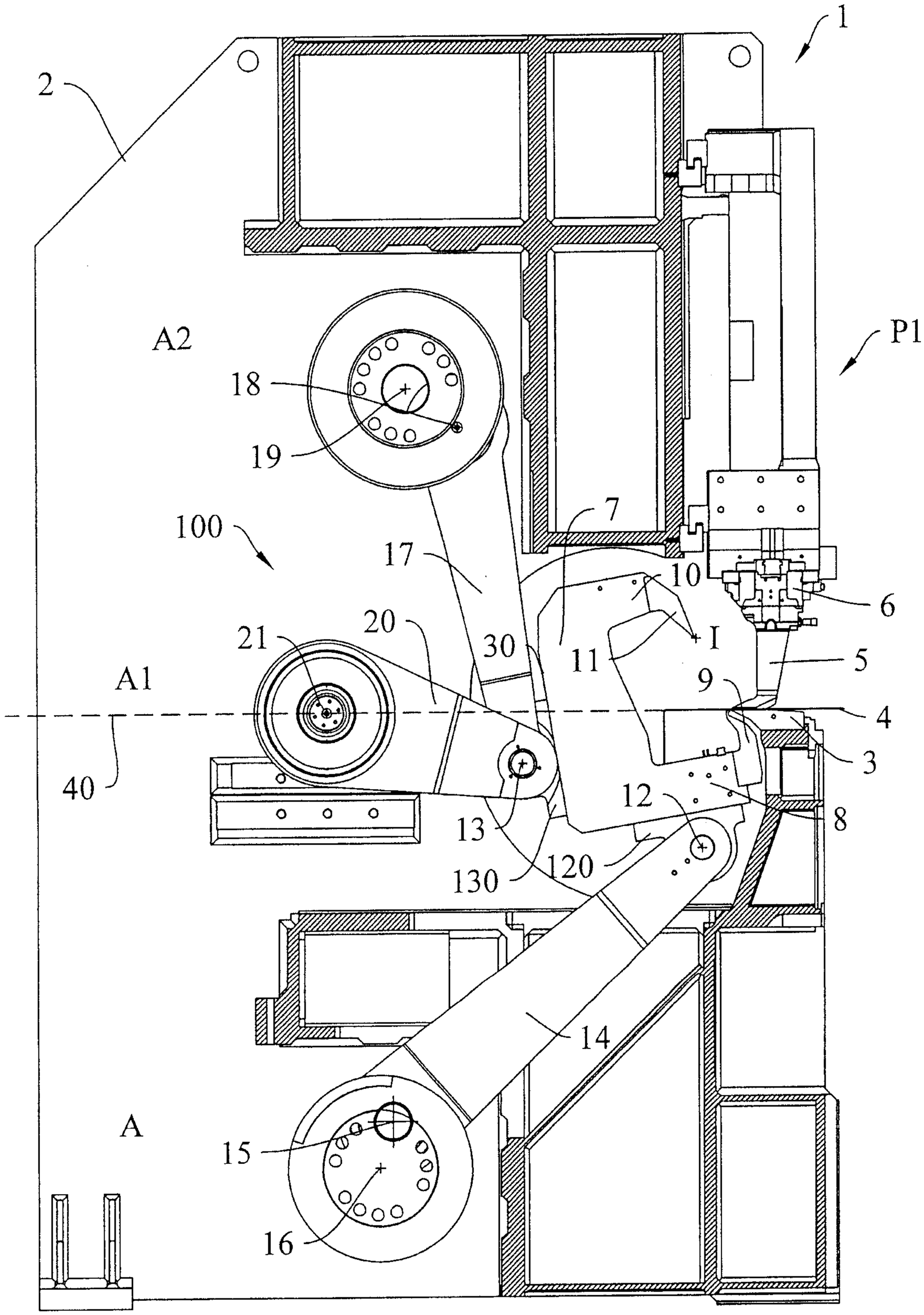

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FIG.2

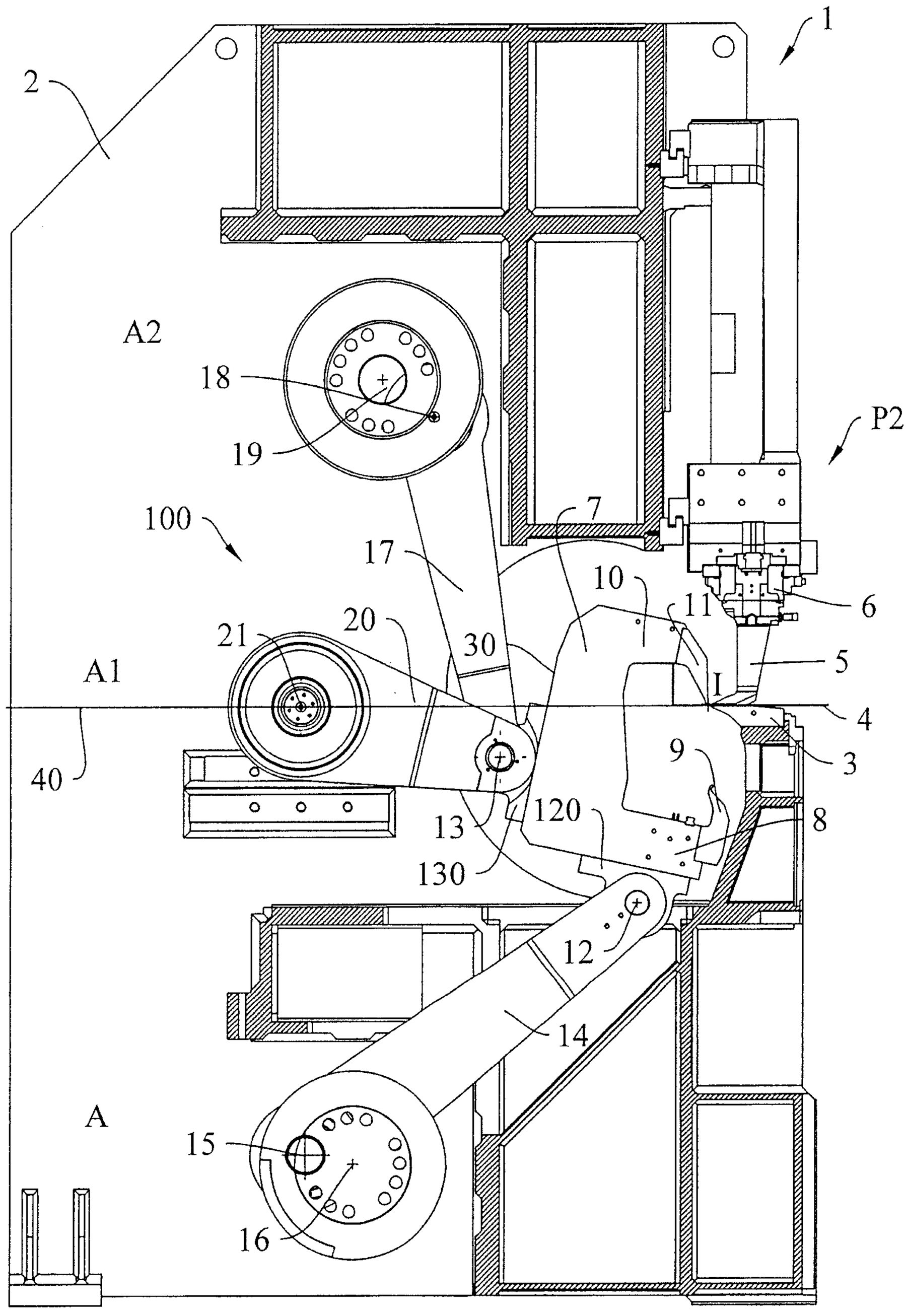

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FIG.3

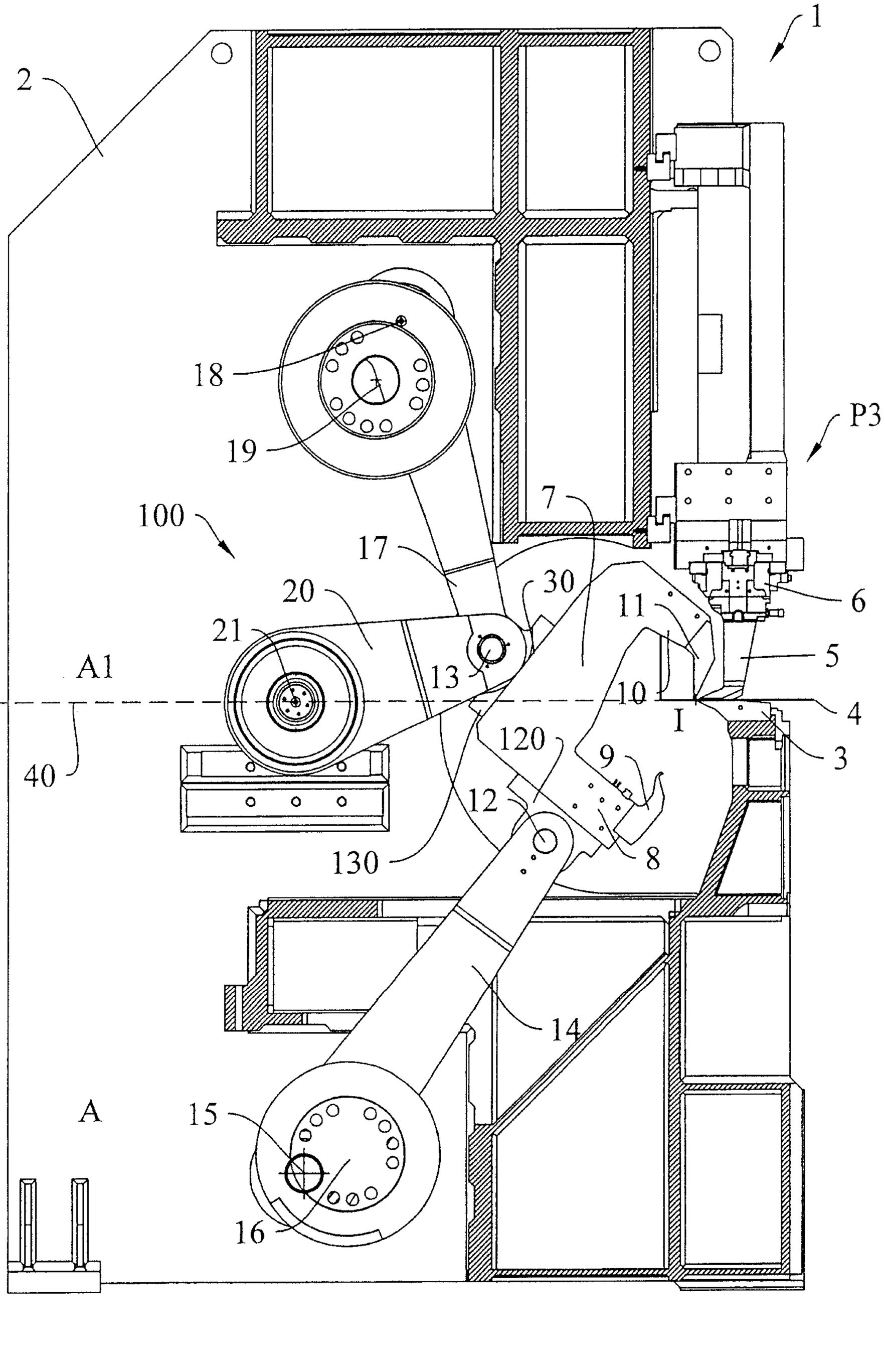

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FIG.4

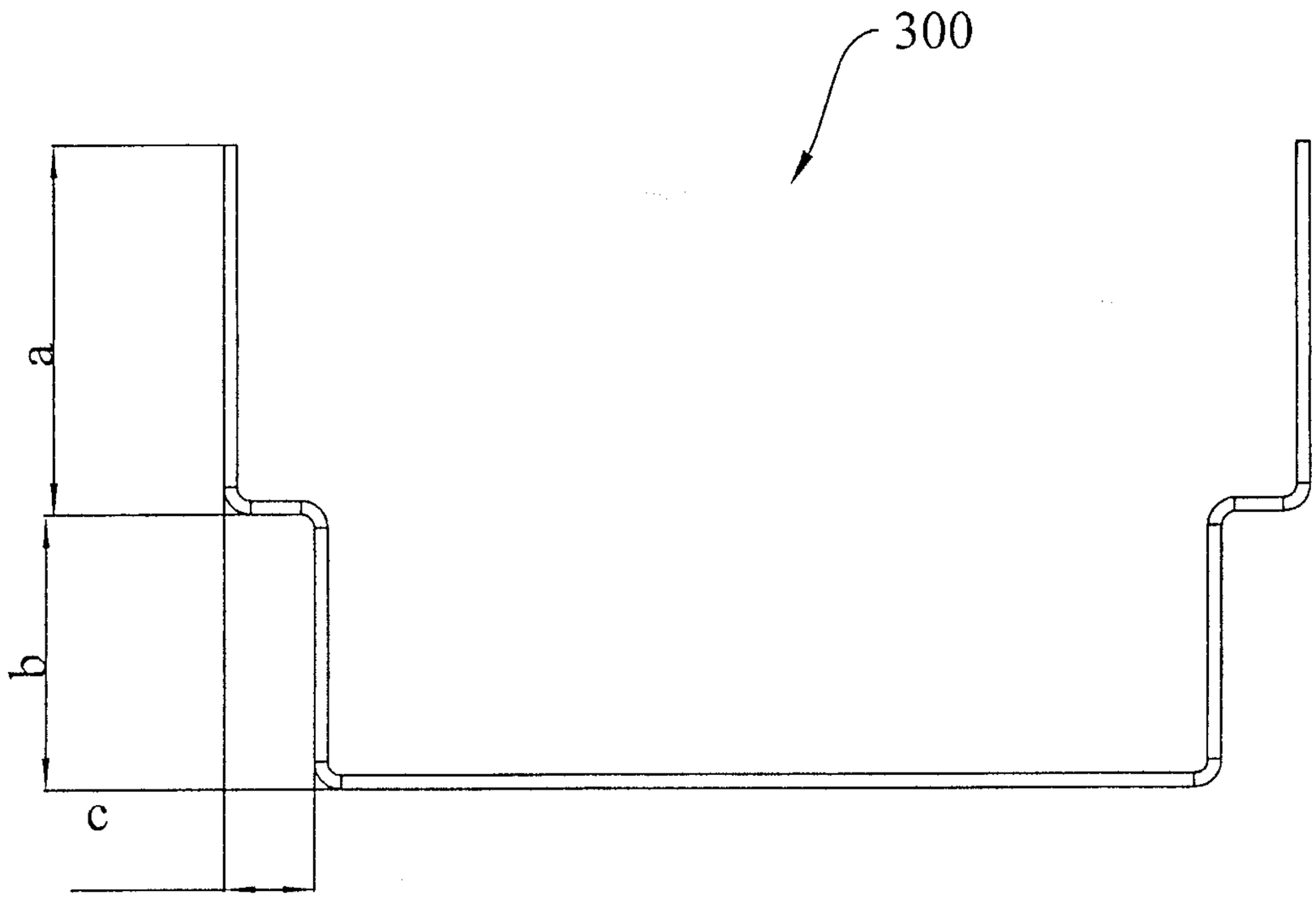


FIG.5

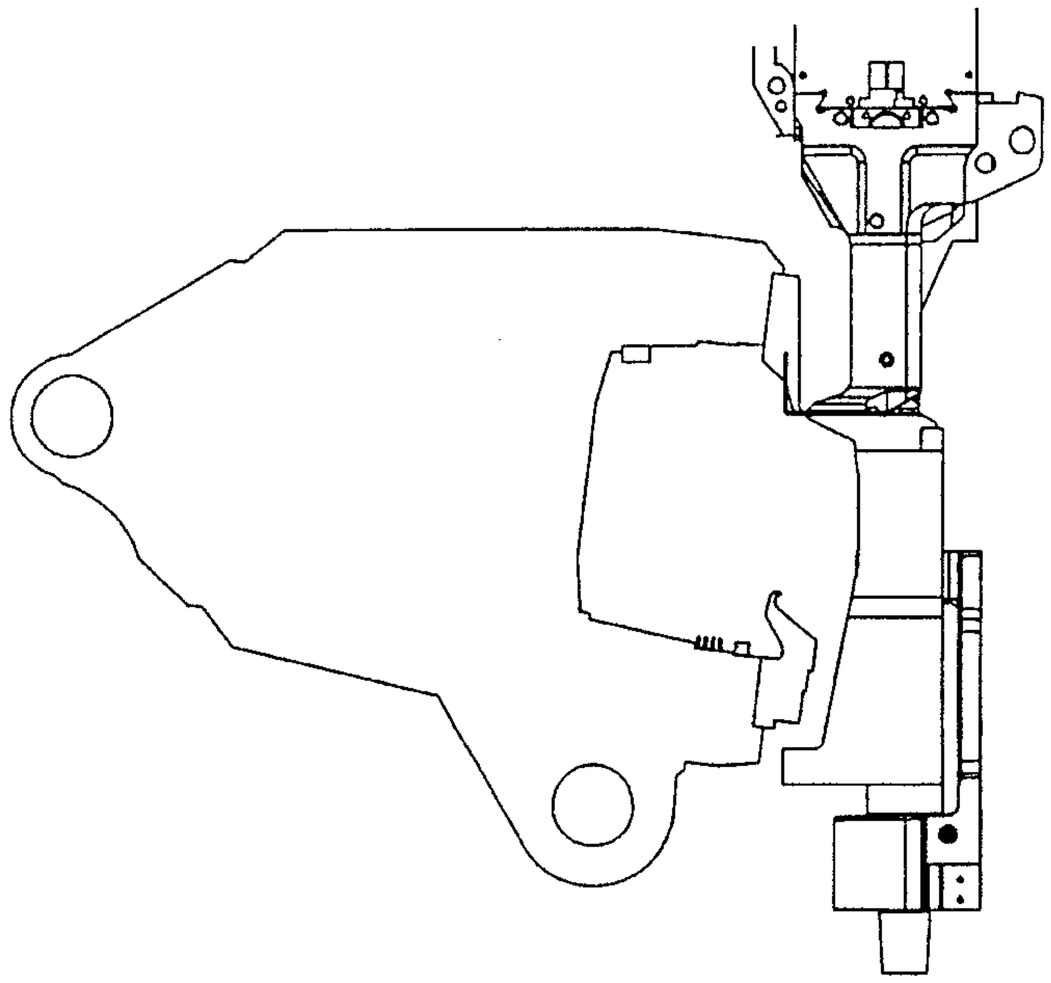


FIG.6

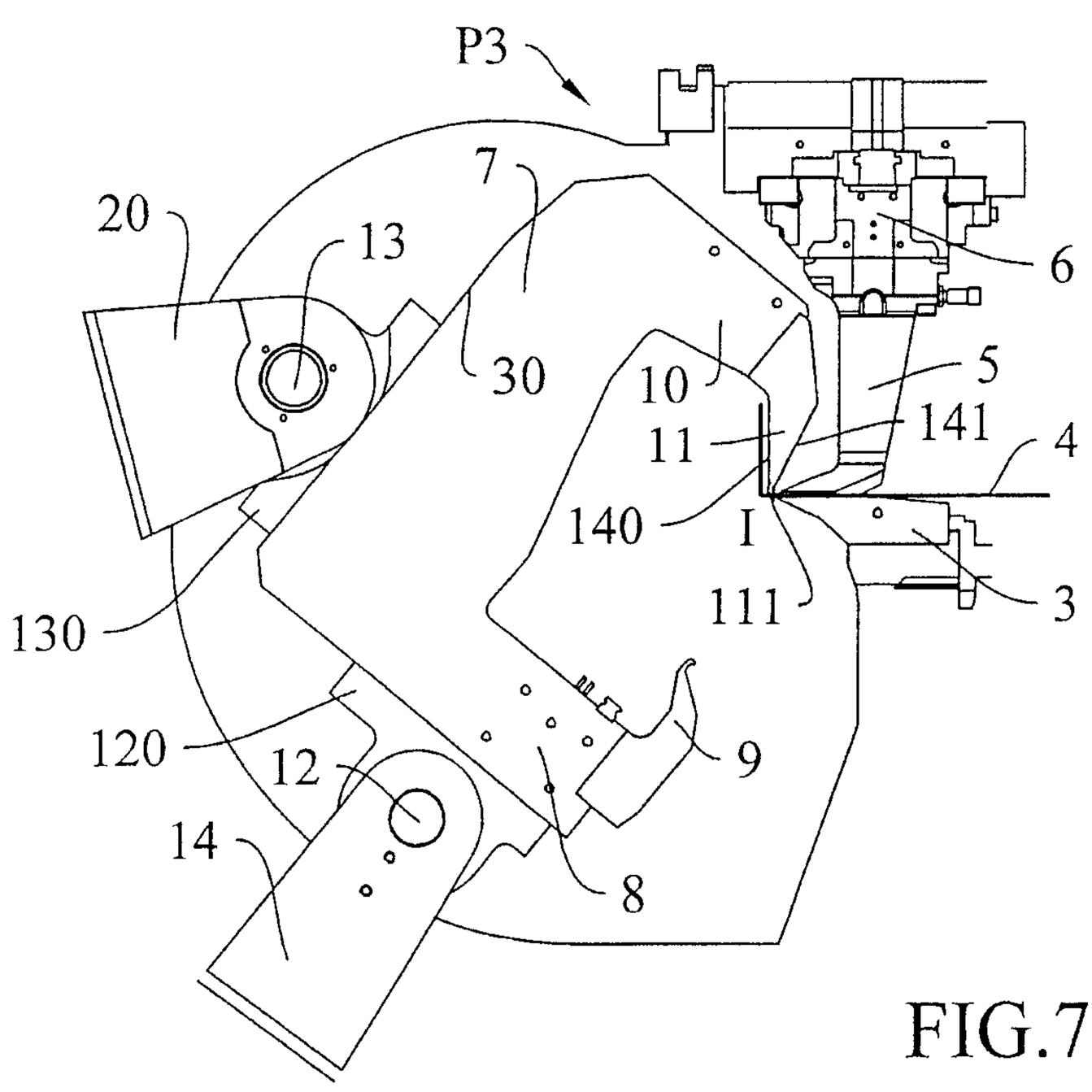


FIG.7

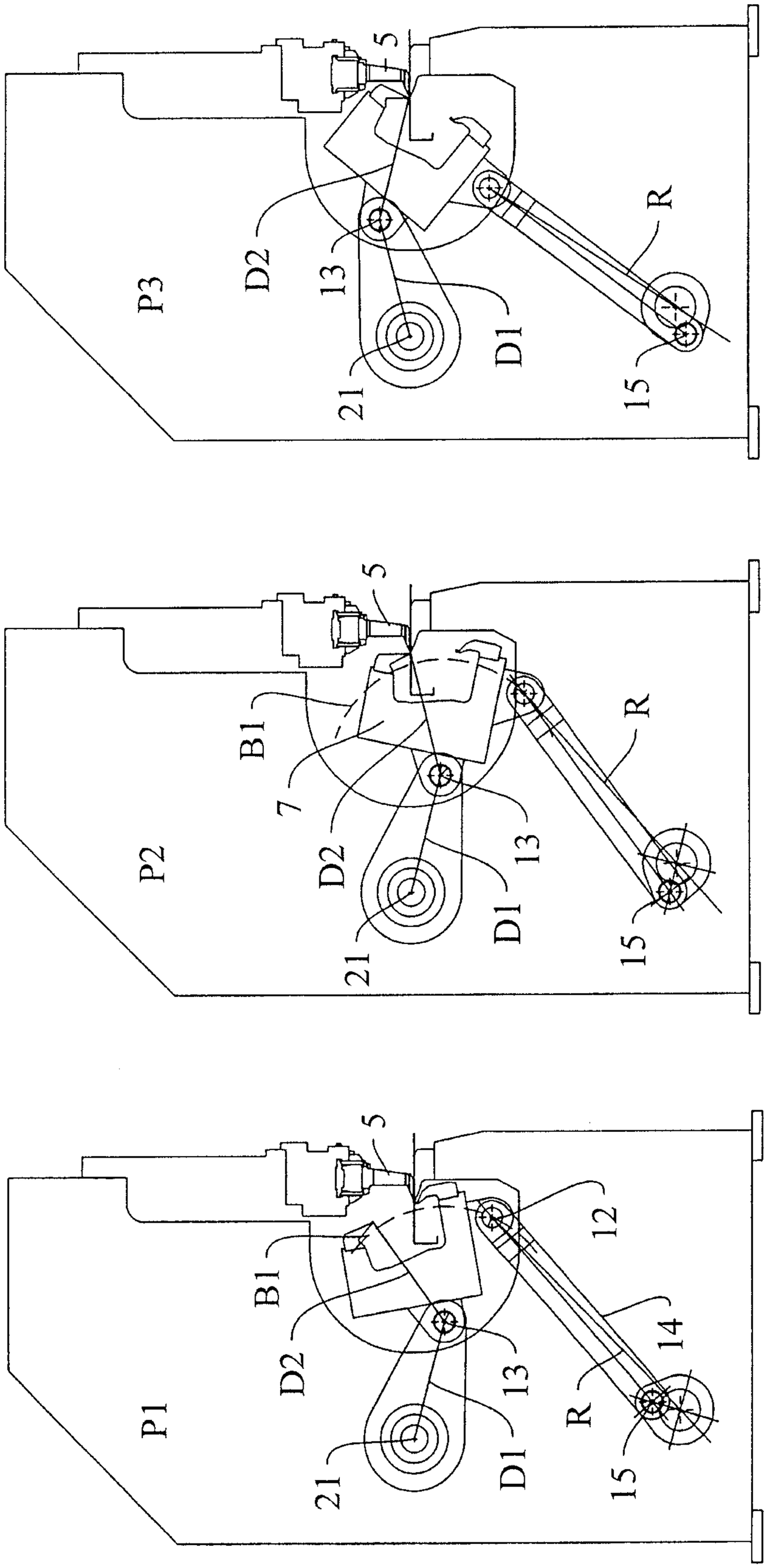


FIG.10

FIG.9

FIG.8

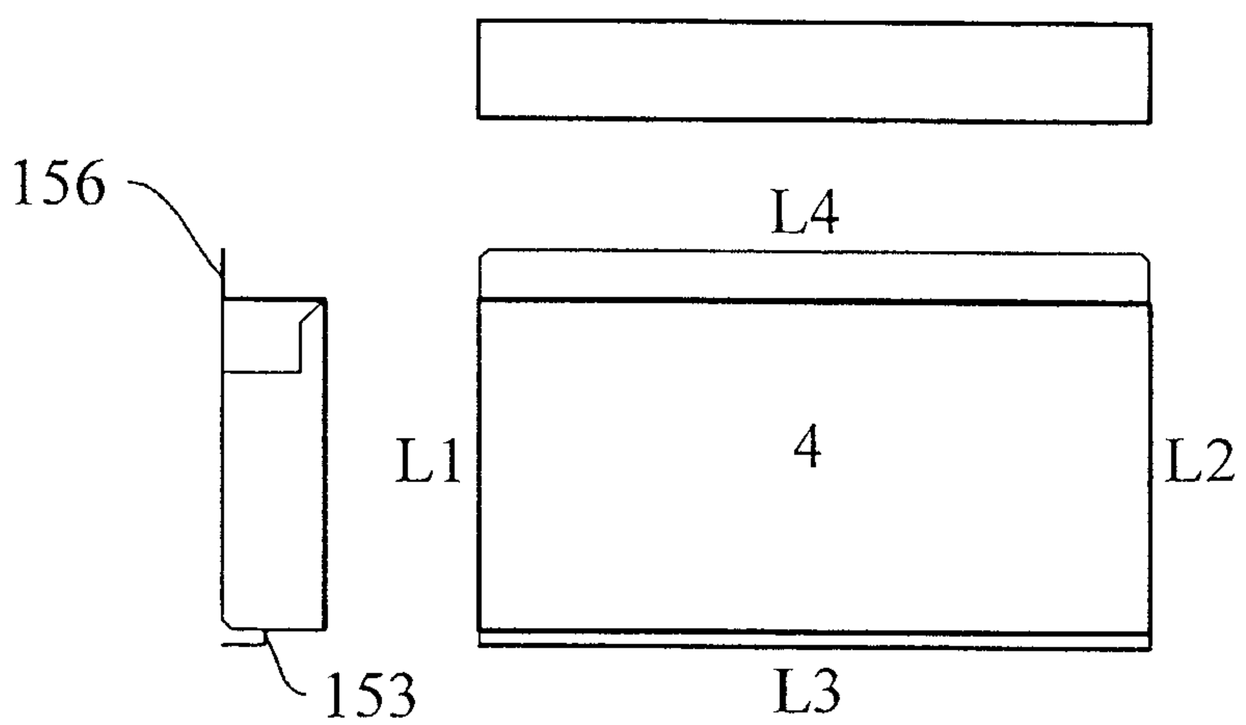


FIG.12c

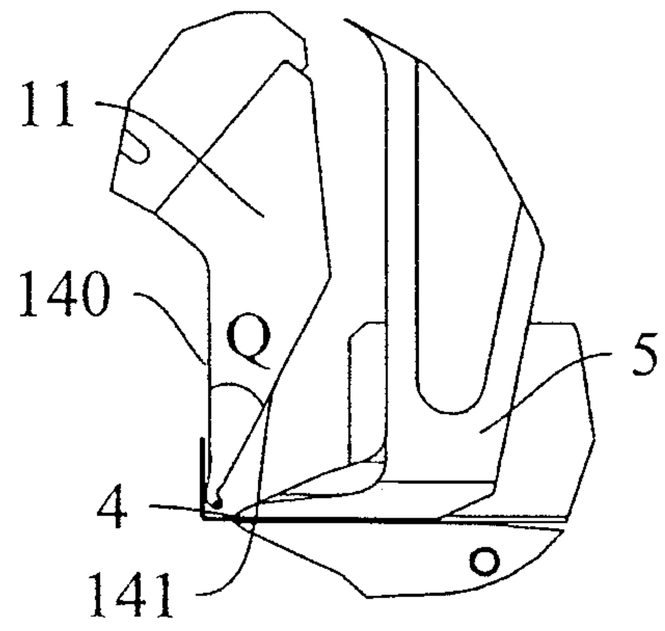


FIG.12a

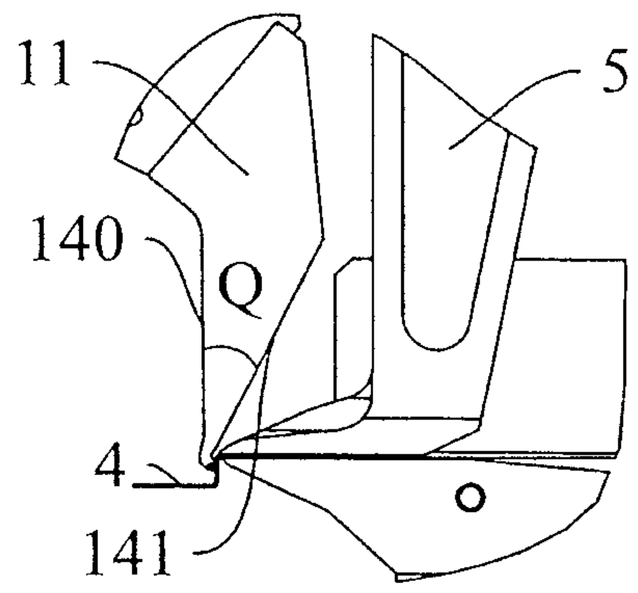


FIG.12b

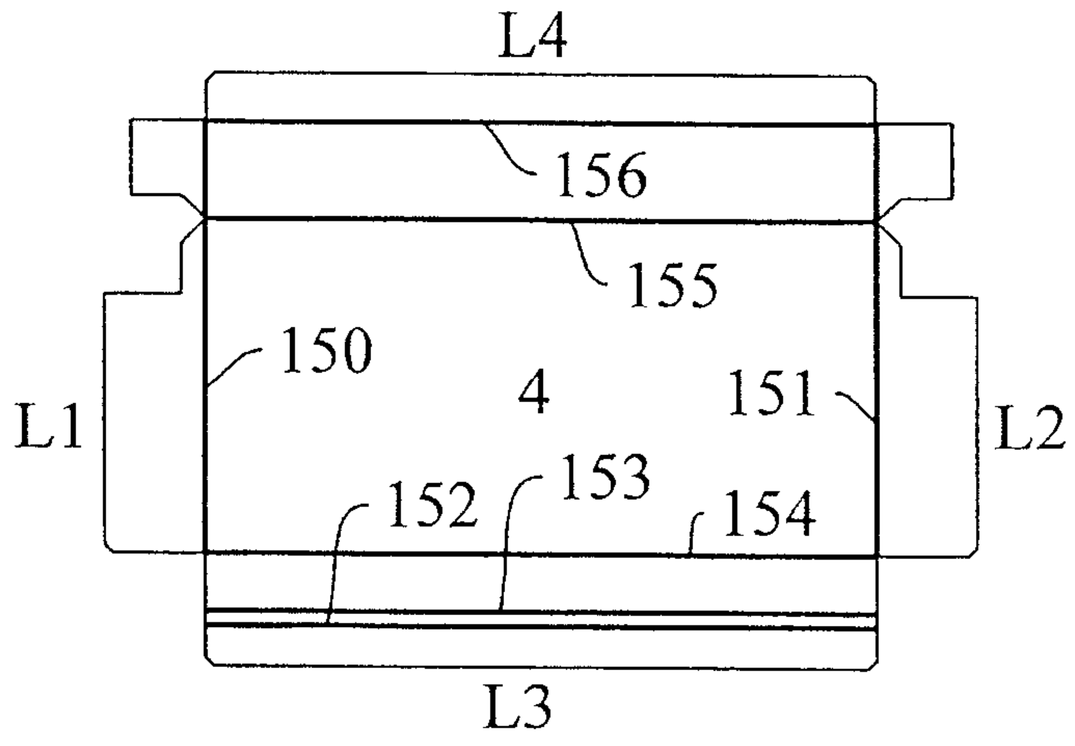


FIG.11c

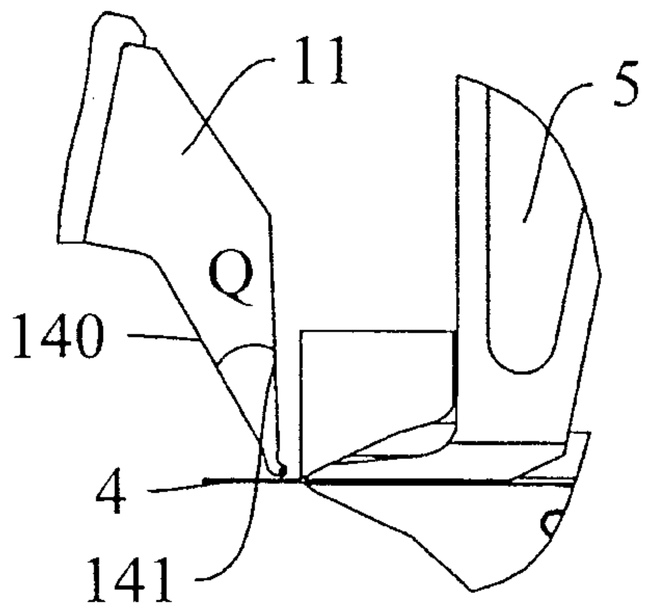


FIG.11a

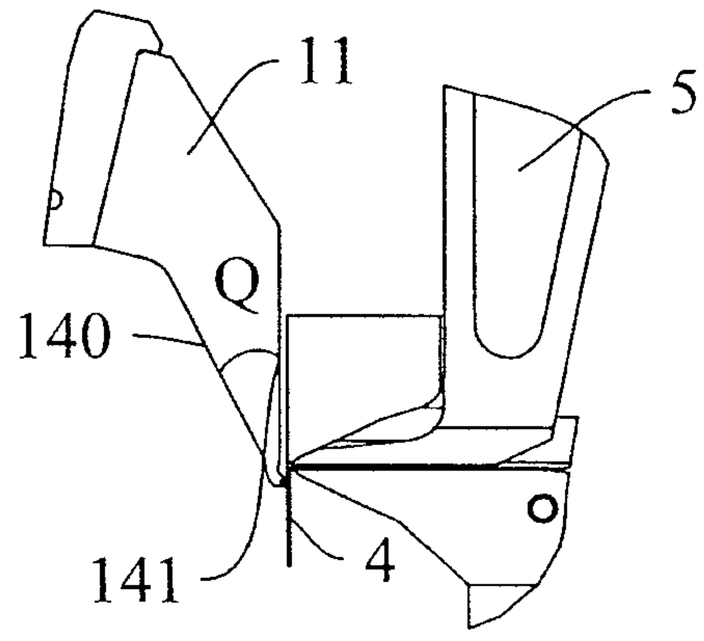


FIG.11b

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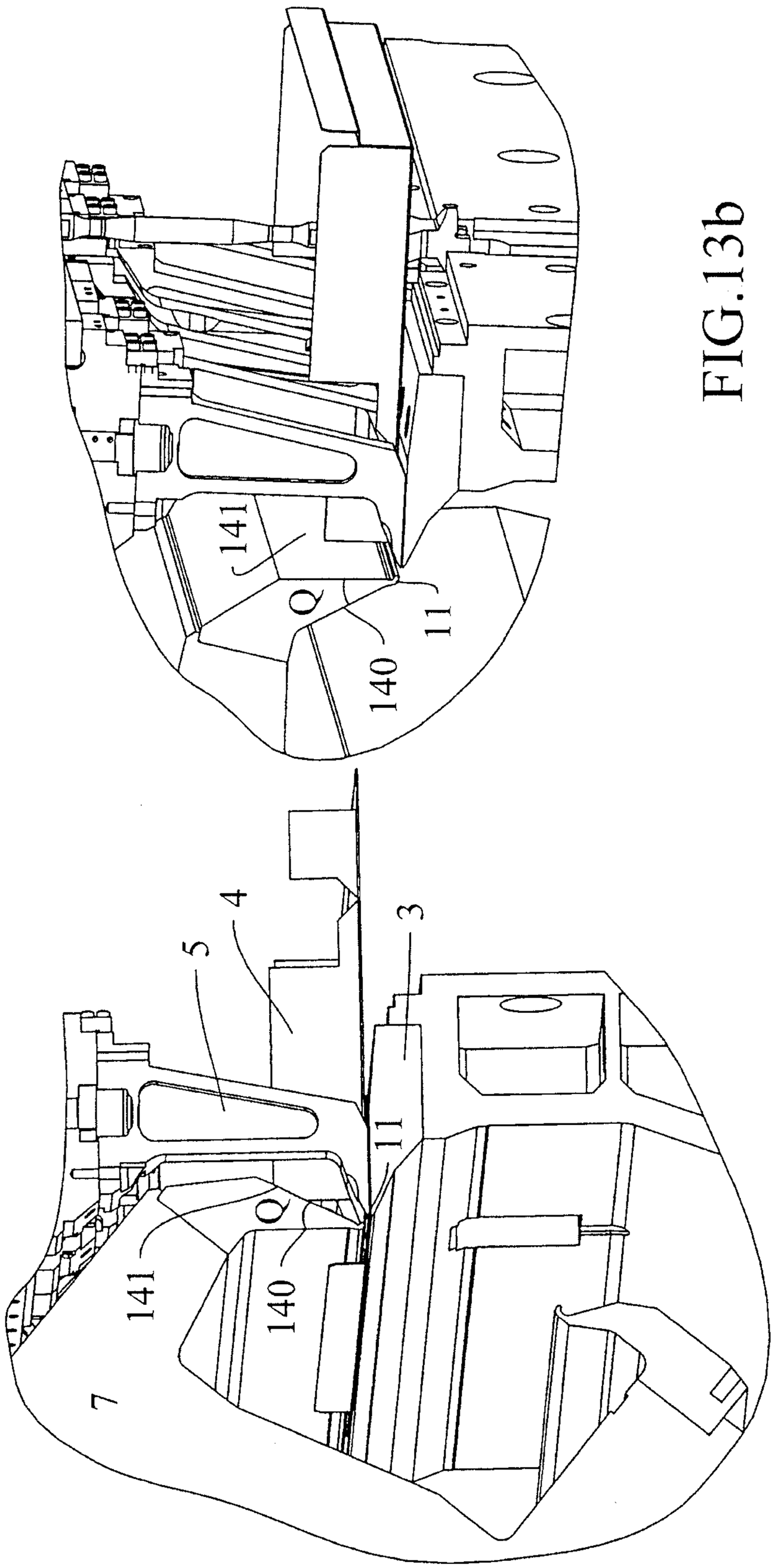


FIG.13a

FIG.13b

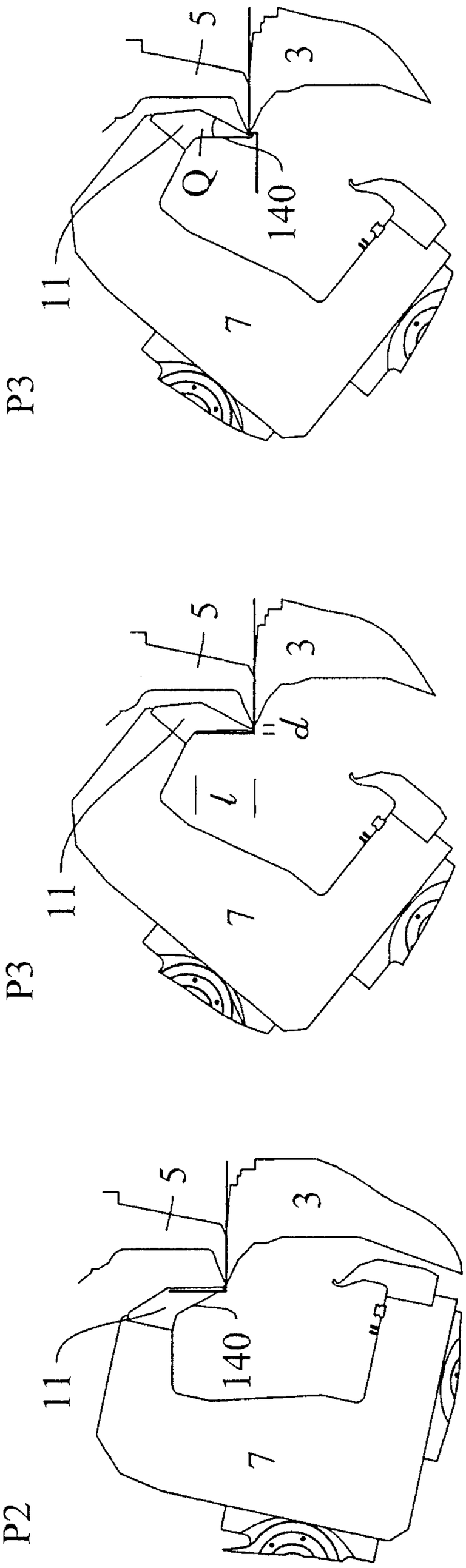


FIG.14c

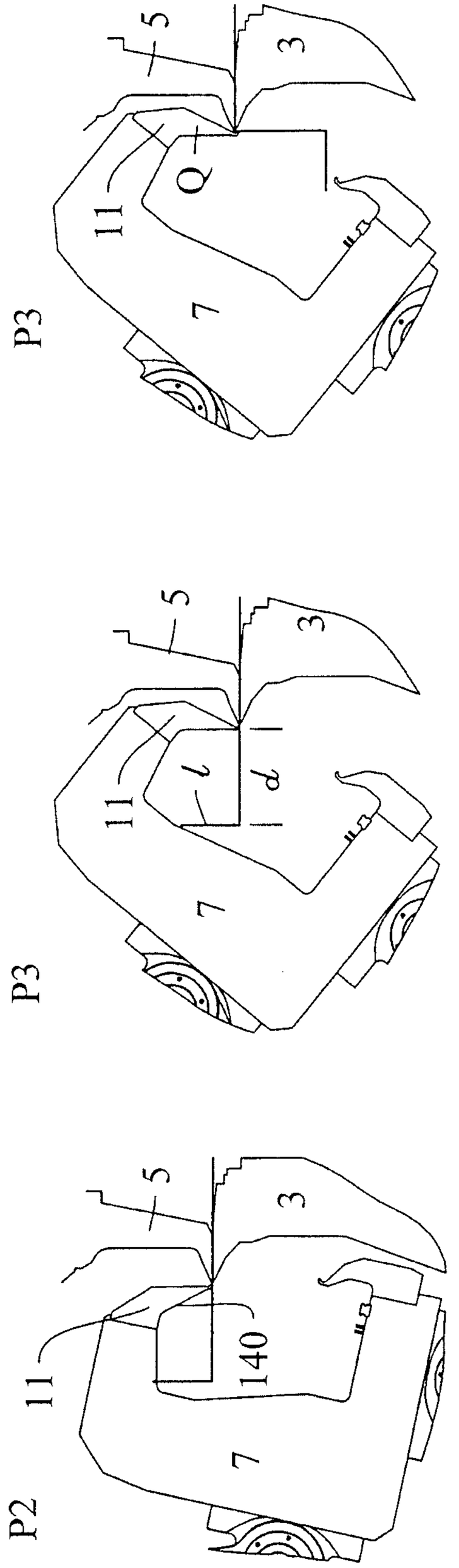


FIG.15c

FIG.14b

FIG.15b

FIG.14a

FIG.15a