

Nov. 27, 1945.

O. L. CLEVEN ET AL

2,389,612

FILING RECEPTACLE

Filed May 27, 1944

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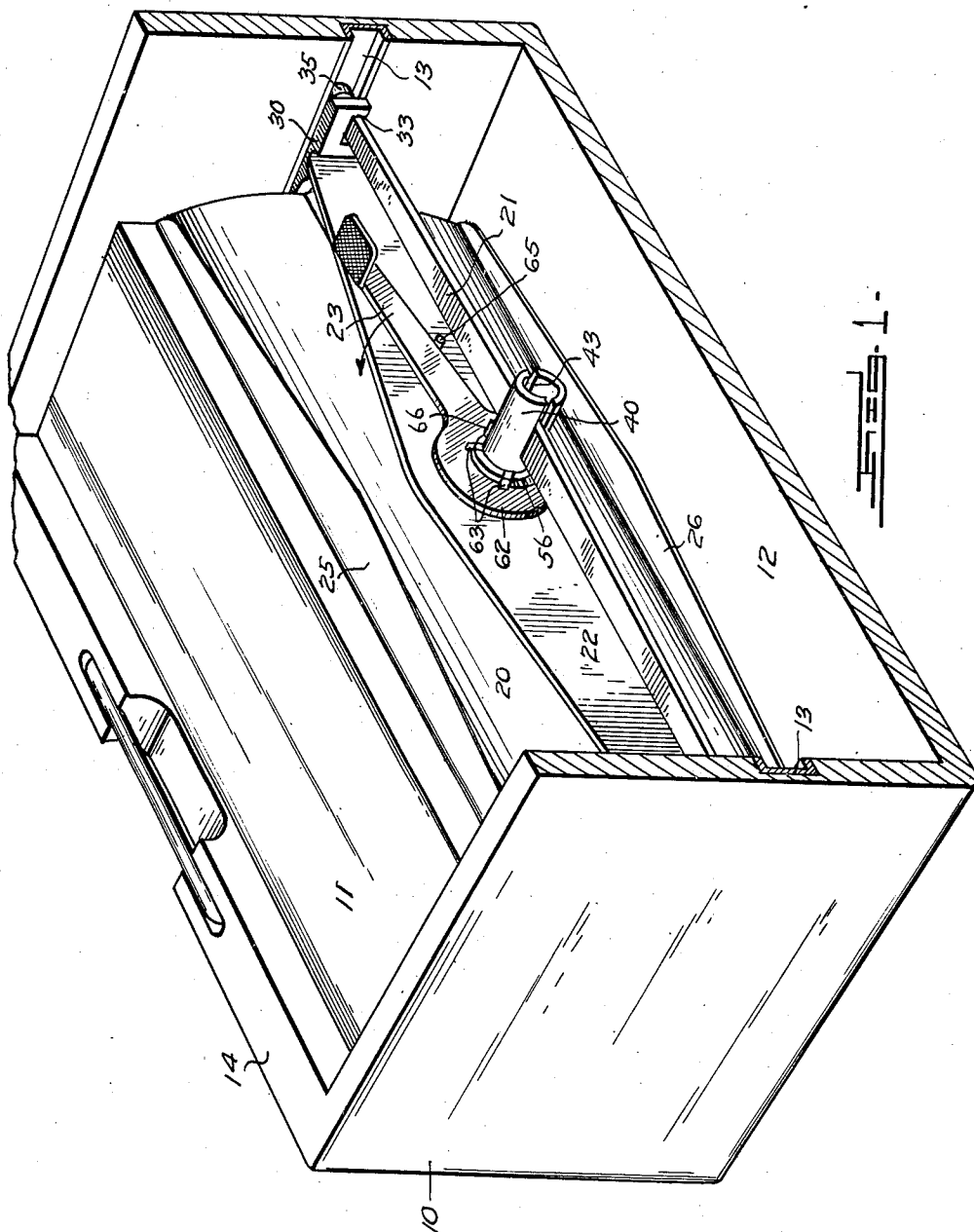


FIG. 1.

INVENTORS
OLUF L. CLEVEN
LAWRENCE A. WILSON,

By *J. F. Motherhead*
attorney.

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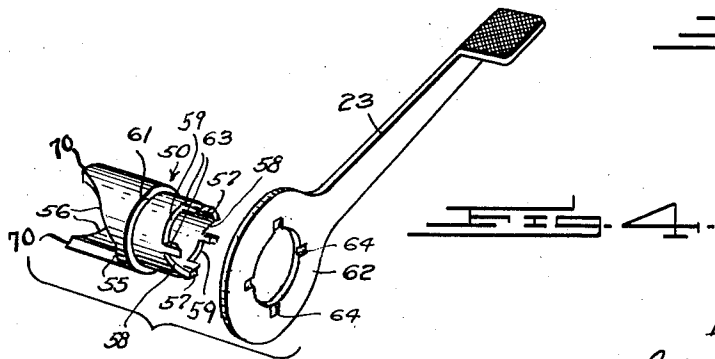
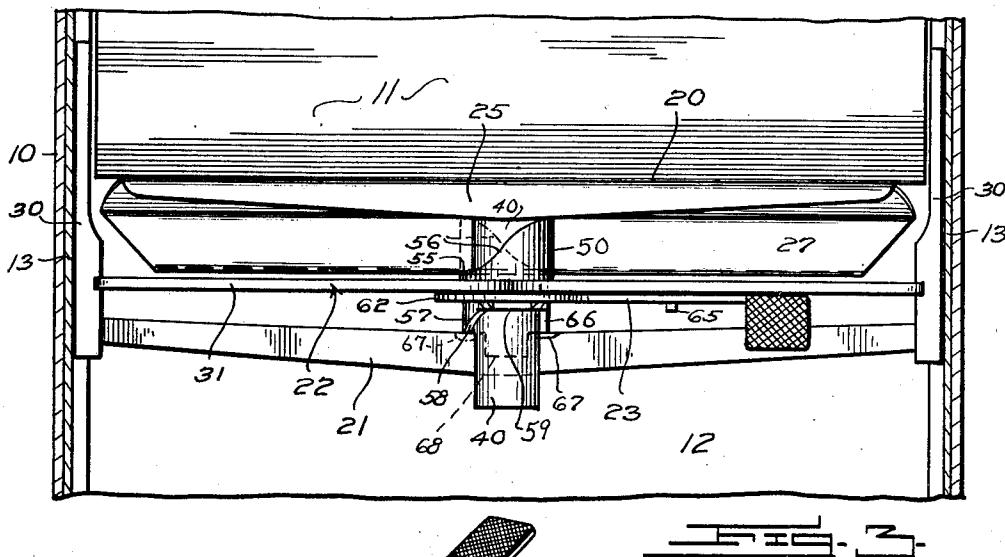
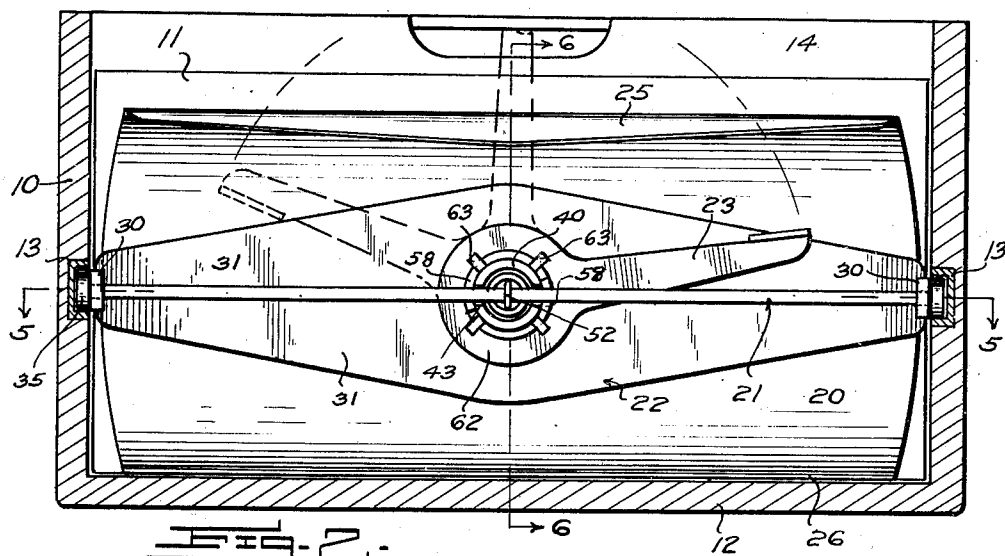
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3 Sheets-Sheet 2



INVENTORS
OLUF L. CLEVEN
LAWRENCE A. WILSON,

By *J. J. Mothushhead*
Attorney

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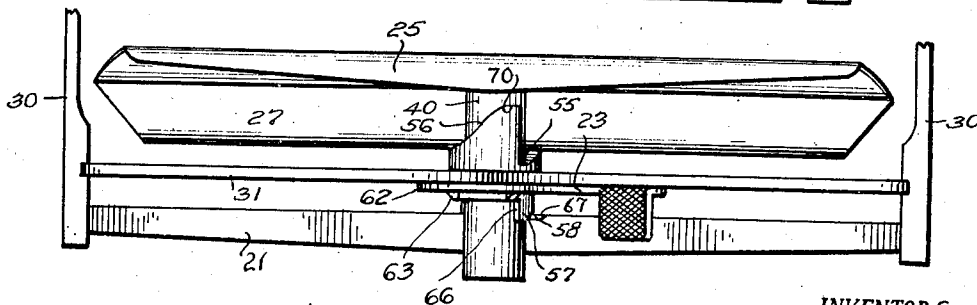
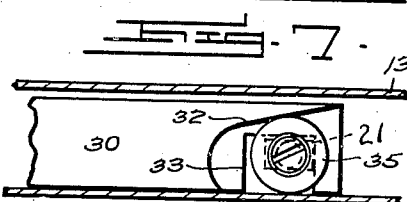
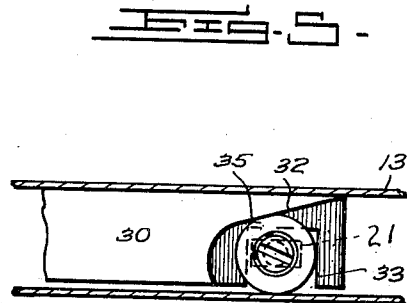
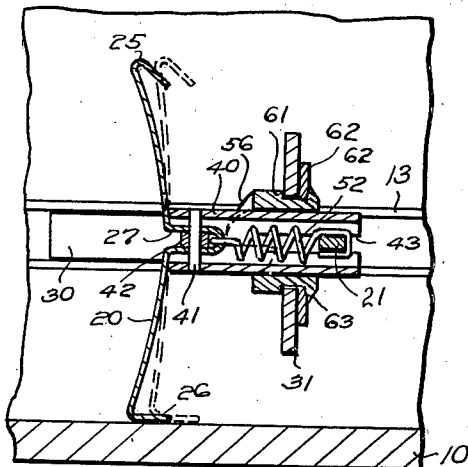
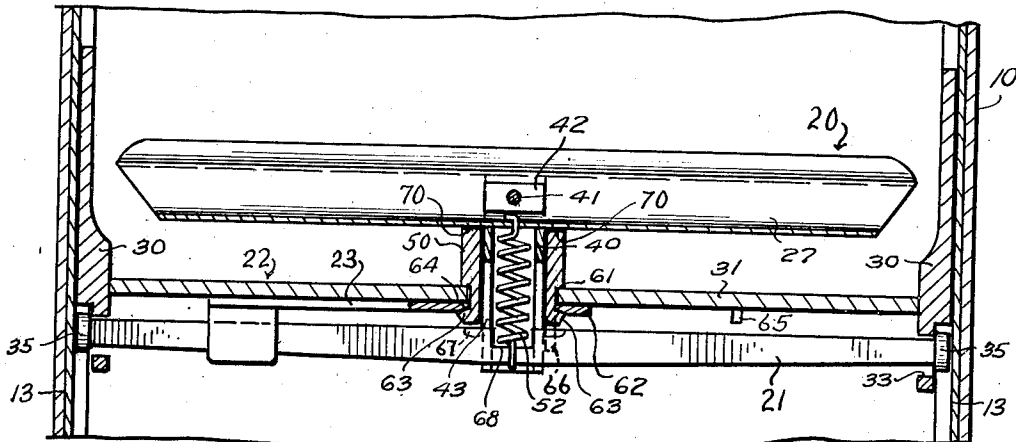
O. L. CLEVEN ET AL

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3 Sheets-Sheet 3



INVENTORS
OLUF L. CLEVEN
LAWRENCE A. WILSON,

By *J. Mothershead*
Attorney.

UNITED STATES PATENT OFFICE

2,389,612

FILING RECEPTACLE

Oluf L. Clevén, Washington, D. C., and Lawrence
A. Wilson, Riva, Md.

Application May 27, 1944, Serial No. 537,666

7 Claims. (Cl. 129—30)

(Granted under the act of March 3, 1883, as
amended April 30, 1928; 370 O. G. 757)

The invention described herein may be manufactured and used by or for the Government of the United States for governmental purposes and without the payment to us of any royalty thereon in accordance with the provisions of the act of April 30, 1928 (Ch. 460, 45 Stat. L. 467).

This invention relates to filing receptacles and more particularly to improvements in followers for vertical card files.

The follower of the present invention is intended to be shiftably mounted in a filing receptacle for adjustment into a position in which its compressor plate engages the files therein. In a preferred embodiment, the follower is adapted in adjusted position to be locked to the filing receptacle and, when locked, to be so operated as to advance its compressor plate toward the engaged files, whereby the files will be compressed and remain in such condition until they are to be removed from the receptacle.

The construction and functions of the follower of the present invention make it well suited to the requirements of files for the cards used in business and other automatic machines. In such machines it is important that the cards be straight and flat, otherwise the machine may not operate in the desired manner.

Experience shows that business machine cards often become bent or warped for various reasons and then, if fed into a machine, interfere with its normal operation. For example, the cards may become bent or warped from handling. The cards will also tend to buckle when exposed to the atmosphere. Loosely arranged cards in a filing receptacle permit air and its moisture to enter the spaces between adjoining cards. This air and moisture is taken up by the cards and causes them to lose their straight and flat condition.

Filing receptacles provided with cards resting on their edges and with a follower embodying the present invention will straighten warped or bent cards and will prevent the warping or buckling of flat, straight cards. This end is accomplished by the compressor plate of the follower which exerts pressure against the cards and maintains them in closely packed condition, the action of the compressor plate being that of a press.

It is therefore an object of this invention to provide a file follower with an improved locking device for securely locking the follower in any position of adjustment in a filing receptacle.

Another object is to provide a follower locking device which provides for locking of the follower

in adjusted position by spring action under the control of a finger lever and for release of the follower by cam action under the control of the finger lever.

Still another object is to provide an improved follower for compressing filed material and maintaining such material in closely packed, compressed condition.

A further object is to provide a relatively simple and inexpensive file receptacle follower having a carriage which is adjustable in a file receptacle, which supports a compressor, and which carries means for locking the carriage in adjusted position and then advancing the compressor by the action of a finger controlled cam which operates against the action of a spring or the like tending to withdraw the compressor to a retracted position.

A still further object is to provide a compressor type of file follower which is characterized by its simplicity in construction, by the ease with which it may be operated, locked and controlled to compress filed material, and which is durable to insure a long, useful life.

Another object is to provide an improved card follower for use with a filing receptacle for machine cards, tabulating cards and the like, which deteriorate or warp if not closely packed when filed, the follower being adapted to compress the filed material at a desired pressure and to maintain the compressed condition of the material as long as desired.

Other objects and advantages will be apparent from the following description, the appended claims, and the accompanying drawings, wherein

Fig. 1 is a perspective view of a card file follower embodying the present invention and installed in a card file receptacle holding material such as cards, photographs or the like.

Fig. 2 is a transverse, vertical sectional view through the receptacle of Fig. 1 and shows a rear elevational view of the follower.

Fig. 3 is a top plan view of the follower of Figs. 1 and 2, the side walls of the filing receptacle being shown in cross-section at the level of the side wall tracks or channel guides.

In Figs. 1, 2 and 3, the finger lever is in such position that the follower carriage is free to slide on the receptacle side wall tracks and the compressor plate is in retracted position.

Fig. 4 is an exploded view of the cam member and finger lever assembly of Figs. 1, 2 and 3.

Fig. 5 is a cross-sectional view taken on line 5—5 of Fig. 2, but with the finger lever in such position that the follower carriage is securely

locked to the receptacle side wall tracks and the compressor plate is in advanced position.

Fig. 6 is a cross-sectional view taken on line 6-6 of Fig. 2, but with the follower having the setting of Fig. 5.

Figs. 7 and 8 are enlarged detail views, Fig. 7 showing a locking bar roller in position to lock the associated follower carriage shoe to its track, and Fig. 8 showing the same roller in such adjusted position that its associated carriage shoe is free to slide in its track.

Fig. 9 is a top plan view corresponding to Fig. 3, but with the finger lever in such position that the compressor plate is in retracted position and the follower carriage is locked to the receptacle side wall tracks.

Referring now to the drawings in providing a description of this invention, the follower may be used with any conventional filing receptacle 10 for holding materials 11, such as tabulating cards, sheets, photographs, files and the like, which are arranged in vertical position with their edges resting on the receptacle bottom 12. The receptacle 10 may be in the form of a file drawer or tray having the usual front, side and rear (not shown) walls. The inner receptacle side walls are provided with guides, which may be channels 13 set in side wall recesses, as illustrated in Figs. 1, 2, 3, 5, 6, 7 and 8. The channels 13 serve as tracks for the follower and extend in parallel relation for the full length of the receptacle 10.

The follower rides on the tracks 13 and is adapted for rectilinear shifting movement toward and away from the receptacle front wall 14 to permit inspection of filed material and to adjust the spacing of the receptacle front wall 14 and the follower according to the amount of filed material 11. The follower comprises a compressor plate 20 and a locking bar 21 which are mounted on and positioned at opposite (front and rear) sides of a carriage 22, as illustrated for example, in Figs. 1, 2 and 3. The carriage 22 also supports a finger lever 23 which is operable to control the bar 21 in its carriage locking and releasing action and to control advancing and retracting movements of the compressor plate 20 relative to the carriage 22.

The compressor plate 20 is preferably flexible in the plane of the paper, referring to Fig. 6, and is preferably formed of sheet metal or other material having the resilient characteristic indicated by this disclosure. The plate 20 (see Fig. 2) has a length slightly less than the inside width of the receptacle 10 and a height slightly less than the height of the cards or material 11 to be filed in the receptacle 10. The flexibility of the compressor plate 20 is limited substantially to the plane of the paper, looking at Fig. 6, by providing longitudinally extending reinforcement at its upper and lower edges and at its median line or center. The reinforcement for the upper and lower edges 25 and 26, respectively, may be secured by rearwardly bending the upper and lower marginal portions of the plate 20 as illustrated more clearly in Fig. 6. The central reinforcement for the plate 20 is secured by a rib or fold 27 extending from end to end thereof, as illustrated in Figs. 3, 5 and 6. The plate 20, except for its reinforcement, is of uniform arcuate shape in vertical cross-section substantially from end to end and as illustrated in solid lines in Fig. 6. The compressor plate 20 thus presents to the cards 11 rounded edges and a substantially concave front face. Consequently, when the follower is pressed against the cards 11, they will

be gripped initially along lines adjacent and parallel to their upper and lower edges and the upper and lower halves of the compressor plate 20 will then yield in the manner of leaf springs until they come to a state of rest and exert the desired pressure on the cards 11.

The follower carriage 22 comprises two shoes 30 rigidly connected by a plate or cross-member 31 extending from side to side of the receptacle 10. The shoes 30, as illustrated in Figs. 2, 3, 5 and 6 rest in the channels or tracks 13 and are proportioned to be freely slidable longitudinally in the tracks 13, although having a sufficiently close fit therein to prevent substantial movement of the shoes 30, and consequently the carriage 22, in any direction excepting longitudinally of the tracks 13. The outer side of the rearward end of each shoe 30 is recessed for a width corresponding substantially to the width of its track 13 (see Fig. 5) and for the purpose of providing a wedge or inclined abutment 32 (see Figs. 7 and 8). The inner side of the rearward end of each shoe 30 is provided with a rectangular recess 33 (see Figs. 5, 7 and 8) extending through the lower shoe edge.

The locking bar 21 is a rigid member extending from side to side of the receptacle 10 and provided at each end with a roller 35 which rides in the corresponding channel or track 13. In a preferred embodiment of this invention, the locking bar 21 is supported by the tracks 13 through the rollers 35. As illustrated in Figs. 5, 7 and 8, the ends of the bar 21 are disposed in the shoe recesses 33 and the rollers 35 underlie the wedges 32. The shoe recesses 33 are of such proportions that they permit limited movement of the locking bar 21 in a direction parallel to the shoes 30 and permit the shoes 30 to rise in the channels 13 and relative to the locking bar 21. When the ends of the bar 21 are positioned rearwardly in the shoe recesses 33 (see Fig. 8), the shoes 30 are free to slide, and the rollers 35 are free to roll, in the tracks 13. If the bar 21 is advanced relatively to the shoes 30 and to the position illustrated in Fig. 7, the rollers 35 and the wedges 32 coact to lock the shoes 30 to the tracks 13 and therefore the carriage 22 to the receptacle 10.

Support for the compressor plate 20 is provided by a tubular member or tube 40 which extends through the carriage plate 31 (see Figs. 5 and 6). The forward end of the tubular member 40 is bifurcated so as to embrace the compressor plate fold 27, as shown in Figs. 3 and 6. The tubular member 40 is rigidly secured to the compressor plate fold 27 by a pin 41. Collapse of the fold 27 is avoided by a small insert 42, which may be seen in Figs. 5 and 6, and through which the pin 41 extends. The rearward end of the tubular member 40 is also bifurcated to provide diametrically opposite slots 43 for slidably receiving the locking bar 21 (Figs. 2, 3, 5 and 6). The sliding connection of the bar 21 and the tubular member 40 is provided for several reasons. The bar 21 may move between the positions illustrated by Figs. 7 and 8 and independently of the tubular member 40. The compressor plate 20 and the tubular member 40 may be advanced and retracted as a unit and relative to the bar 21, but are precluded from having any rotary movement.

The carriage 22 (see Fig. 5) is also provided with a cam member 50, the finger lever or handle 23, and a spring 52 to form operating and control means for the locking bar 21 and the compressor plate 20. The cam member 50 is oper-

ated by the lever 23 to force the compressor plate 20 and the locking bar 21 in opposite directions and against the action of the spring 52.

The cam member 50 is of tubular form (Figs. 4, 5 and 6), is journaled in the carriage plate 31, slidably receives the compressor plate supporting member 40 for rectilinear movement, and is rotatable about the member 40. The forward end of the cam member 50 is provided with two diametrically opposite cut-outs 55 which receive the fold 21 in the retracted position of the compressor plate 20. One of these cut-outs 55 may be seen in Fig. 4. Between these cut-outs 55 and at opposite sides of the member 50, the latter is provided with two cam edges 56 which, during rotary movement of the cam member 50 from the setting of Fig. 3, act similarly and simultaneously on the compressor plate fold 21 and at opposite sides of the tubular member 40 to force the compressor plate 20 toward the front of the receptacle 10 and away from the carriage plate 31. The advanced position of the compressor plate 20 is shown in Figs. 5 and 6. During the reverse rotary movement of the cam member 50, the spring 52 is effective to withdraw the compressor plate 20 from its advanced position of Figs. 5 and 6 to its retracted position of Fig. 3.

The rearward end edge of the cam member 50 has two diametrically opposite extended portions 57 which serve as abutments for the locking bar 21 in the cam position illustrated by Figs. 2 and 3. These abutments 57 are aligned (see Figs. 3 and 4) with the cut-outs 55 in the other end of the member 50. At one side of each of these abutments 57 the end edge of the cam member 50 recedes to provide a cam 58. A relatively long opening 59 in the end edge extends from each extended portion or abutment 57 to the cam 58 adjacent the other abutment, as will be clear from Figs. 3 and 4. When the finger lever 23 is rotated from the full line position to the dotted line position of Fig. 2, the locking bar 21 slides forwardly along the two cams 58 and from the setting of Figs. 1, 2, 3 and 8 to the setting of Figs. 5, 6, 7 and 9 under the action of the spring 52.

The form and manner of assembly of the cam member 50 and the finger lever 23 will be apparent from Figs. 2, 3, 4, 5 and 6. The cam member 50 is of reduced outer diameter at its rearward end to facilitate its insertion into the aperture of corresponding size and shape in the center of the carriage plate 31. The cam member 50 has a shoulder 64 which rests against the marginal portion of the carriage plate 31 surrounding its central aperture. The finger lever 23 has an enlarged, apertured inner end 62 which closely fits the protruding end of the cam member 50 at the rearward side of the carriage plate 31 and engages the latter as illustrated in Figs. 3, 5, 6 and 9. Prongs 63 (Fig. 4) formed on the rearward edge of the cam member 50 are bent into engagement with notches 64 (Fig. 4) in the finger lever 23 whereby the carriage plate 31, the member 50 and the lever 23 are so assembled that the cam member 50 is precluded from axial movement, but is free to be rotated about its axis by the finger lever 23. Looking at Fig. 2, clockwise movement of the finger lever 23 is limited by a pin 65 (Figs. 1 and 3) on the carriage plate 31, and counterclockwise movement is limited by engagement of the surfaces 66 (Figs. 3 and 9) on the cam member 50 with the locking bar 21 as illustrated in Fig. 5. The action of the cams 58 may be aided by providing beveled sections 67 (Figs. 3 and 5) on the engaged portions of the locking bar 21.

The spring 52 is conveniently housed in the tubular member 40 and is connected at its opposite ends to the compressor plate 20 and the locking bar 21 in the manner indicated by Figs. 5 and 6. The locking bar 21 may be notched at 68 (Fig. 5) to accommodate the adjacent coils of the spring 52 when both the compressor plate 20 and the locking bar 21 are fully retracted relative to the carriage plate 31 and in the positions indicated by Fig. 9.

Operation

The most desirable results from the use of the present invention are secured when the receptacle 10 is to receive cards or other flat filing material having a length corresponding closely to the inside width of the receptacle 10 and having such width that when resting with an edge on the receptacle bottom its upper edge is slightly higher in elevation than the upper edge 25 of the compressor plate 20.

Space is provided in the front of the receptacle for the filing of cards 11 by shifting the follower rearwardly, this movement being readily accomplished when the finger lever 23 is in the position of Figs. 1, 2 and 3. When the cards 11 are properly arranged on edge in the receptacle 10, the follower is shifted forwardly and to the position of Figs. 1 and 3, in which its compressor plate 20 engages the adjacent card 11 along horizontal lines adjacent its upper and lower edges. The cards 11 are now somewhat closely packed between the front wall 14 of the receptacle and the compressor plate 20. At this time the cross-sectional shape of the compressor plate 20 is that illustrated in full lines in Fig. 6 and the follower parts are set as represented in Figs. 1, 2, 3 and 8. The compressor plate 20 is in its fully retracted position relative to the carriage plate 31. The locking bar 21 is spaced from the plate 31 by the two abutments 57 (Fig. 3) which equalize the action on the bar 21 at opposite sides of the tube 40 to retain each of the two rollers 35 in the setting illustrated by Fig. 8, whereby the carriage shoes 30 are free to slide in their tracks 13. It will be noted that the spring 52 is effective to yieldingly maintain the compressor plate 20 in its retracted position and the locking bar 21 in engagement with both of the abutments 57.

The operator with one hand may advance the follower in the direction towards the cards 11 until they contact and then, with the same hand, may rotate the finger lever 23 counterclockwise from the full line position of Fig. 2. In its first few degrees of rotary movement, the finger lever 23 reaches the setting of Fig. 9. During this initial lever movement, the two abutments 57 pass to out-of-the-way positions relative to the locking bar 21 and the two cams 58 slide past the locking bar 21, permitting it to be drawn forwardly by the action of the spring 52 from the setting of Figs. 3 and 8 to the setting of Figs. 7 and 9. The extent of the abutments 57 and the cams 58 peripherally of the cam member 50 is limited to secure resetting of the locking bar 21 with a limited movement of the finger lever 23, which movement need not be more than about thirty or forty degrees. Since the cams 58 are identical and engage the bar 21 simultaneously, the spring 52 advances the two rollers 35 similarly and simultaneously to the carriage locking position of Fig. 7. The forward movement of the locking bar 21 is determined by the extent of movement required by the rollers 35 in locking the carriage shoes 30 to their tracks 13. As the locking bar 21 moves

forwardly, it carries with it the rollers 35 from the setting of Fig. 8 to the setting of Fig. 7. The rollers 35 in the Fig. 7 position so co-act with the wedges 32 that the rollers 35 are forced against the lower flanges of the tracks 13 and the carriage shoes 30 are forced against the upper flanges of the tracks 13. This co-action of the rollers 35, the shoes 30 and the tracks 13 is effective to lock securely the follower carriage 22 to the receptacle side walls and to maintain the carriage 22 in the desired position of adjustment. It will also be noted that any force exerted on the carriage 22 in a direction to move it rearwardly (away from the receptacle wall 14) can only result in an increase in the locking effect of the rollers 35 and the wedges 32. The spring 52 is adequate to prevent accidental movement of the locking bar 21 and its rollers 35 to carriage releasing position (Fig. 8).

In its first few degrees of rotary movement, the finger lever 23 also resets the cam member 50 relative to the compressor plate 20, as illustrated in Fig. 9. The front cam edges 56 have reached a position of initial engagement with the compressor plate rib or fold 27. Thus the first operation to result from a counter-clockwise movement of the finger lever 23 from the full line position of Fig. 2 is the locking of the carriage 22 in its position of adjustment.

During the remainder of its rotary movement to the dotted line position of Fig. 2 and the full line position of Fig. 5, the finger lever 23 through the action of the two cam edges 56 on the fold or rib 27 advances the central section of the compressor plate 20 and increases gradually and progressively the pressure exerted by the upper and lower portions of the plate 20 on the cards 11. The plate 20 yields under this cam action and may have face to face contact with the adjacent card 11. The cross-sectional shape of the plate 20 may now be substantially that illustrated in dotted lines in Fig. 6. The advancing movement resulting from operation of the finger lever 23 is effective to highly compress the cards 11 and thereby squeeze out any air disposed between any two adjacent cards 11.

According to the illustrated and preferred embodiment of this invention, locking of the carriage 22 is quickly accomplished in about the first quarter of the arcuate movement of the finger lever 23 between its two positions of Fig. 2 and the advancing movement of the compressor plate rib or fold 27 is gradually accomplished during the last three quarters of such arcuate movement. Thus a powerful thrust is imparted to the compressor plate fold or rib 27 with only a finger pressure on the lever 23, and mechanical compression of the cards 11 is secured by a smoothly acting follower.

The arcuate or rotary movement of the finger lever 23 is limited by engagement of the diametrically opposite cam member surfaces 66 with the upper and lower sides of the locking bar 21. One of these surfaces 66 may be seen in Fig. 5 as engaging the lower side of the bar 21.

When the finger lever 23 is in the dotted line position of Fig. 2 and the position of Figs. 5 and 6, the compressor plate rib or fold 27 rests against the two diametrically opposite abutments 70 at the outer ends of the cam edges 56. These abutments 70, like the abutments 57, are defined by edge surfaces lying in a plane normal to the axis of rotation of the cam member 50. Accidental displacement of the compressor plate 20 from its advanced position is therefore avoided.

It is now clear that with the bar 21 and the plate 20 in the positions of Figs 5 and 6, the follower carriage 22 is locked securely in adjusted position by a wedging action and the compressor plate 20 cooperates with the front receptacle wall 14 to highly compress the cards 11 in a squeezing or pressing action which excludes air and moisture from all but edge surfaces of the cards 11. This action not only presses out wrinkles, bends and other distortions in the cards, but precludes warping and buckling of the cards while not in use.

When the cards 11 are to be removed from the receptacle 10, the finger lever 23 is shifted from the dotted to the full line position of Fig. 2 and the follower is moved toward the rear of the receptacle to facilitate access to the cards 11. Return of the lever 23 to the full line position of Fig. 2 results in the follower being operated in a reverse manner to that heretofore described under the heading "Operation." During the first three quarters of the arcuate movement of the lever 23 to the right (Fig. 2), the spring 52 draws the compressor plate 20 towards the carriage plate 31 until the follower parts have the setting of Figs. 7 and 9. Continued movement of the lever 23 for the last quarter of its travel shifts the cam surfaces 58 into engagement with and then beyond the beveled edge portions 67 of the locking bar 21 to reset the rollers 35, as shown in Fig. 8, and thereby release the follower for sliding action along the tracks 13.

The lever 23 comes to rest (Fig. 3) when it engages the pin 65. The compressor plate 20 is now fully retracted, its rib or fold 27 being received within the cam member cut-outs 55. The locking bar 21 is held in the carriage releasing position by the two cam abutments 57.

The foregoing is to be understood as merely being by way of illustration, since this invention includes all modifications and embodiments thereof falling within the scope of the appended claims.

We claim:

1. In a card file, the combination with a card tray having inner side walls provided with tracks extending between the tray end walls, of a card follower comprising a carriage slidably received by said tracks for rectilinear movement relative to the front end wall of said tray, a card compressor plate, a carriage locking bar, said plate and said bar being supported by said carriage for independent rectilinear movement relative to each other and to said carriage and being respectively positioned forwardly and rearwardly of said carriage, manually operated rotatable cam means supported by said carriage for shifting separately said plate and said bar in opposite directions and away from said carriage, and resilient means urging said plate and said bar toward said carriage, said cam means being operable in one direction to release said locking bar for movement by said resilient means to lock said carriage to said tracks and then to advance said compressor plate relative to the locked carriage and against the action of said resilient means.

2. In a card file, the combination with a card tray having inner side walls provided with tracks extending between the tray end walls, of a card follower comprising a carriage slidably received by said tracks for rectilinear movement relative to the front end wall of said tray, a card compressor plate, a carriage locking bar, said plate and said bar being supported by said carriage for independent rectilinear movement relative to

each other and to said carriage and being respectively positioned forwardly and rearwardly of said carriage, manually operated rotatable cam means supported by said carriage for shifting separately said plate and said bar in opposite directions and away from said carriage, and resilient means urging said plate and said bar toward said carriage, said cam means being operable in one direction to release said compressor plate for movement by said resilient means and then to force said locking bar against the action of said resilient means from carriage locking position to carriage releasing position.

3. In a card file, the combination with a card tray having channels extending between the ends thereof, of a card follower comprising a carriage slidably supported by said channels, an apertured cam member supported by said carriage for rotary movement and having its forward and rearward edges formed with cams, an operating handle for said cam member, a tube extending through and axially of said cam member for axial sliding movement therein, a compressor plate secured to one end of said tube for movement by the cams at the forward edge of said cam member, said tube having a bifurcated rearward end to provide longitudinally extending slots, a locking bar extending through said slots for movement axially of said tube and for engagement with the cams at the rearward edge of said cam member, means carried at the ends of said locking bar and positioned in said channels, said means with said bar in retracted position locking said carriage to said channels and with said bar in extended position releasing said carriage for sliding movement, and resilient means urging said compressor plate and said locking bar toward each other for operative engagement with the adjacent cams of said cam member during rotary movement thereof, said cams being so set on said cam member that in one direction of rotary movement thereof the rearward cams release the locking bar for retracting movement by said resilient means and then the forward cams advance said compressor plate relative to said carriage and that in the other direction of rotary movement thereof the forward cams release the compressor plate for retracting movement by said resilient means and then the rearward cams engage said locking bar to shift said bar from retracted position to extended position for releasing said carriage.

4. In a card file, the combination with a card tray having inner side walls provided with channels extending between the tray and walls, of a card follower comprising a member extending between said tray side walls and having rigidly connected thereto shoes slidably received by said channels, said member and shoes forming a carriage shiftable for rectilinear movement toward and away from the tray front end wall, a ring shaped cam member journaled in said carriage member for rotary movement, a finger lever carried by said cam member for manually operating

said cam member, a tubular member extending through said cam member and said carriage member and axially slidable in said cam member, said tubular member having a bifurcated rearward end, a locking bar slidably received within said bifurcated end for rectilinear movement under the control and action of said cam member and toward and away from said carriage member, means carried at the ends of said locking bar and positioned in said channels, said means with said bar in retracted position forcing said shoes into locking engagement with said channels and with said bar in extended position releasing said shoes for sliding movement in said channels, and a compressor plate secured to the forward end of said tubular member for sliding movement therewith relative to said carriage member under the control and action of said cam member, and resilient means urging said compressor plate and said locking bar toward each other and toward said carriage member.

5. In a card file, the combination with a card tray having inner side walls provided with tracks extending between the tray end walls, of a card follower comprising a carriage slidably supported by said tracks, means associated and movable with said carriage for locking said carriage to said tracks, and a resilient compressor plate carried by said carriage and having rearwardly bent top and bottom edges, said plate having a generally arcuate shape from top to bottom to provide a substantially concave front wall surface, said plate also having a rearwardly projecting reinforcing fold adjacent its median line extending from end to end thereof.

6. In a card file, the combination with a card tray having opposite side walls and an end wall, of a resilient, card compressor plate mounted in said tray for rectilinear, slidable adjustment movement toward and from said end wall, said plate having a generally arcuate shape from top to bottom to provide a substantially concave wall surface facing said tray end wall, said plate also having longitudinally extending integral reinforcement adjacent its median line.

7. In a card file, the combination with a card tray having opposite side walls and an end wall, of a resilient, card compressor plate mounted in said tray for rectilinear, slidable adjustment movement toward and from said end wall, said plate having such length that the side edges thereof are closely adjacent the inner surfaces of said tray side walls, said plate having rearwardly bent top and bottom edges and a generally arcuate shape from top to bottom to provide a substantially concave front wall surface substantially from tray side wall to tray side wall, said wall surface facing said tray end wall, said plate also having a rearwardly projecting reinforcing fold adjacent the plate median line and extending from end to end of said plate.

OLUF L. CLEVEN,

LAWRENCE A. WILSON.