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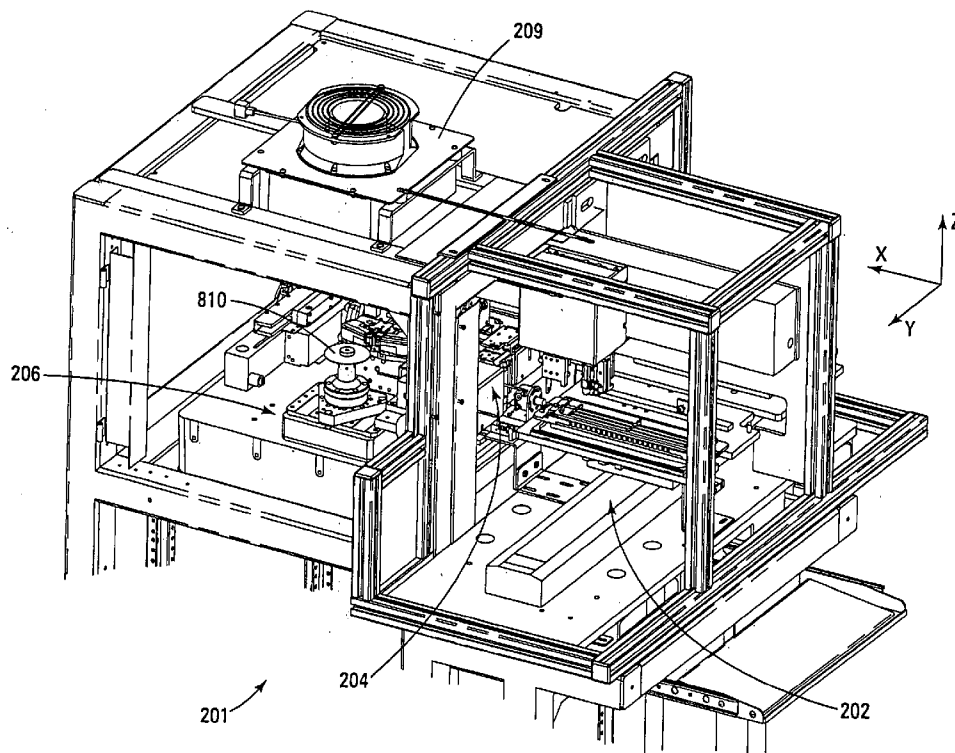
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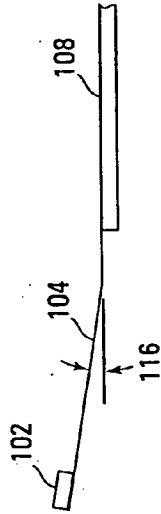
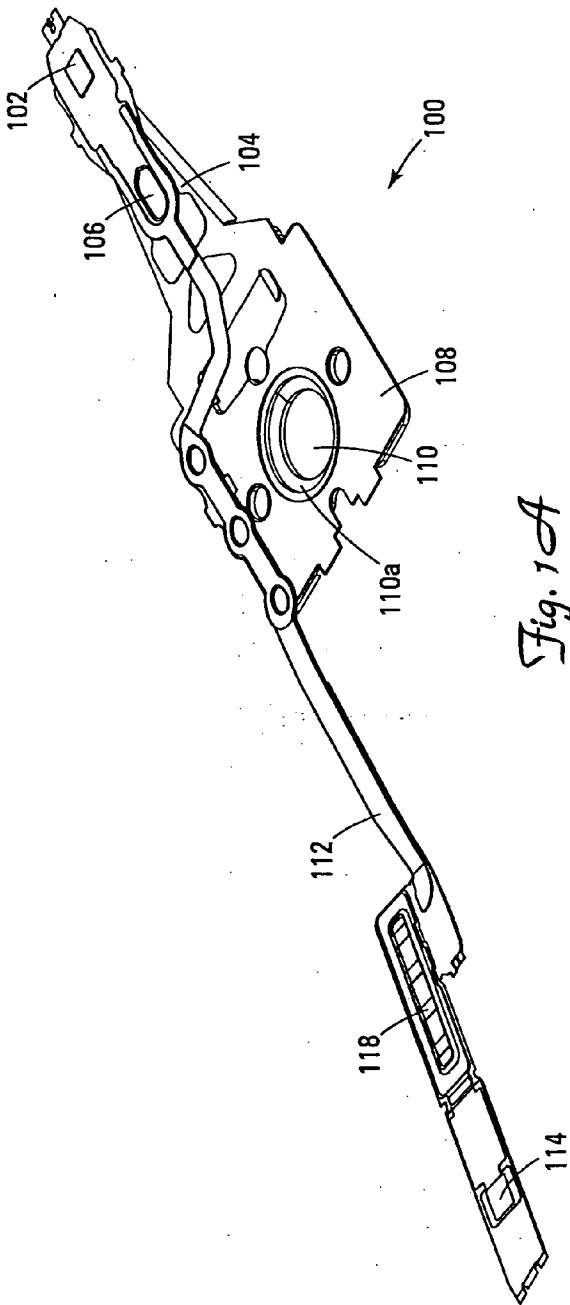
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(57) **ABSTRACT**

A method for dynamic electrical testing of head gimbal assemblies may include initiating an automated continuous process that includes selecting an unmounted head gimbal assembly; aligning the unmounted head gimbal assembly; loading the unmounted head gimbal assembly to a disc; and testing the unmounted head gimbal assembly.





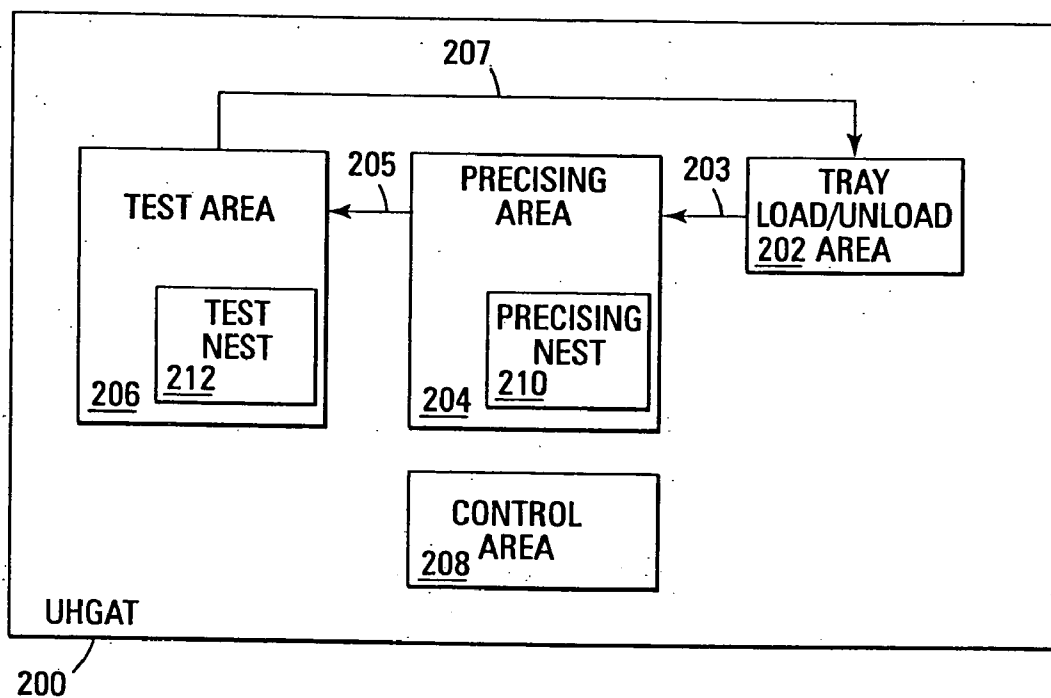


Fig. 2

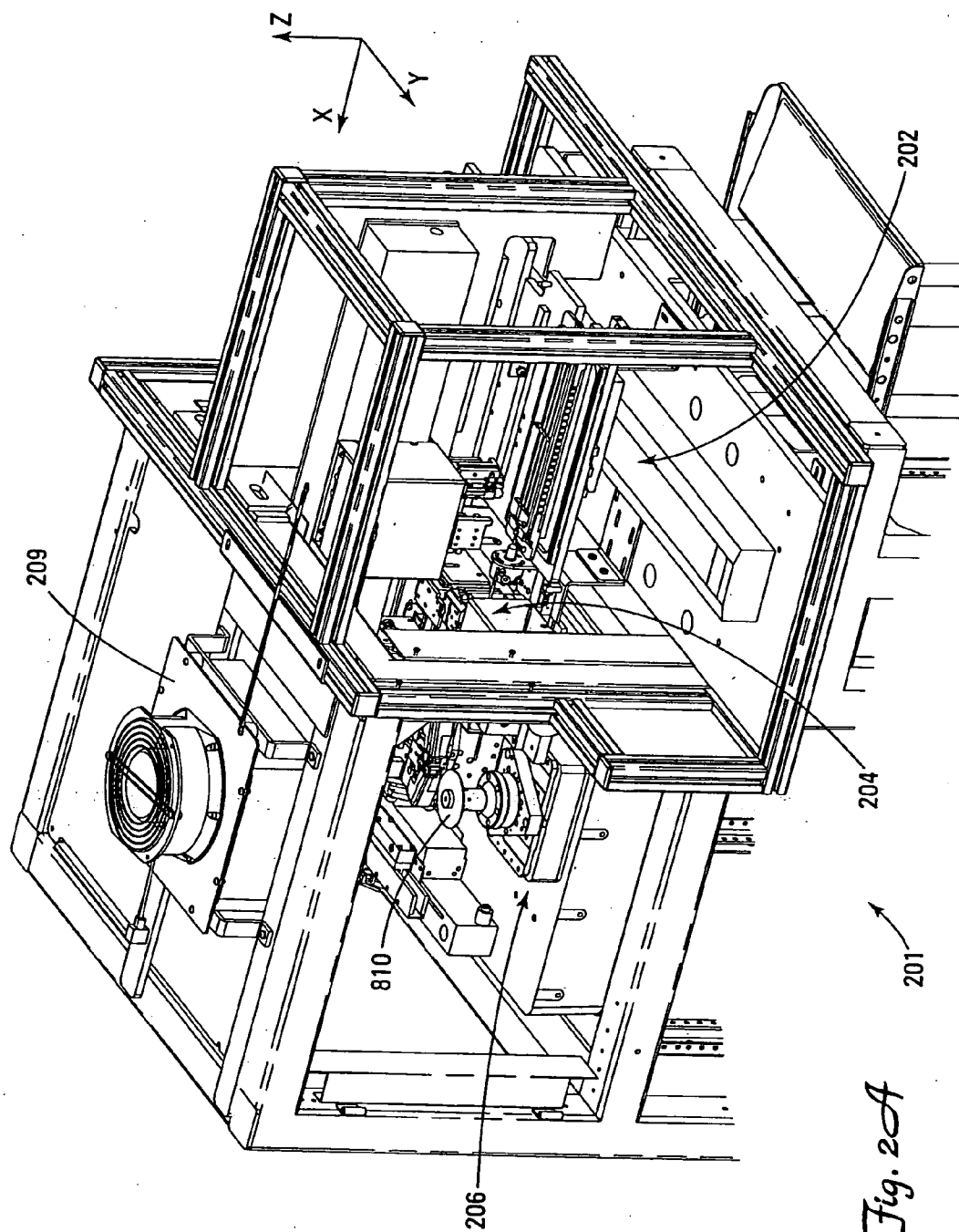


Fig. 2A

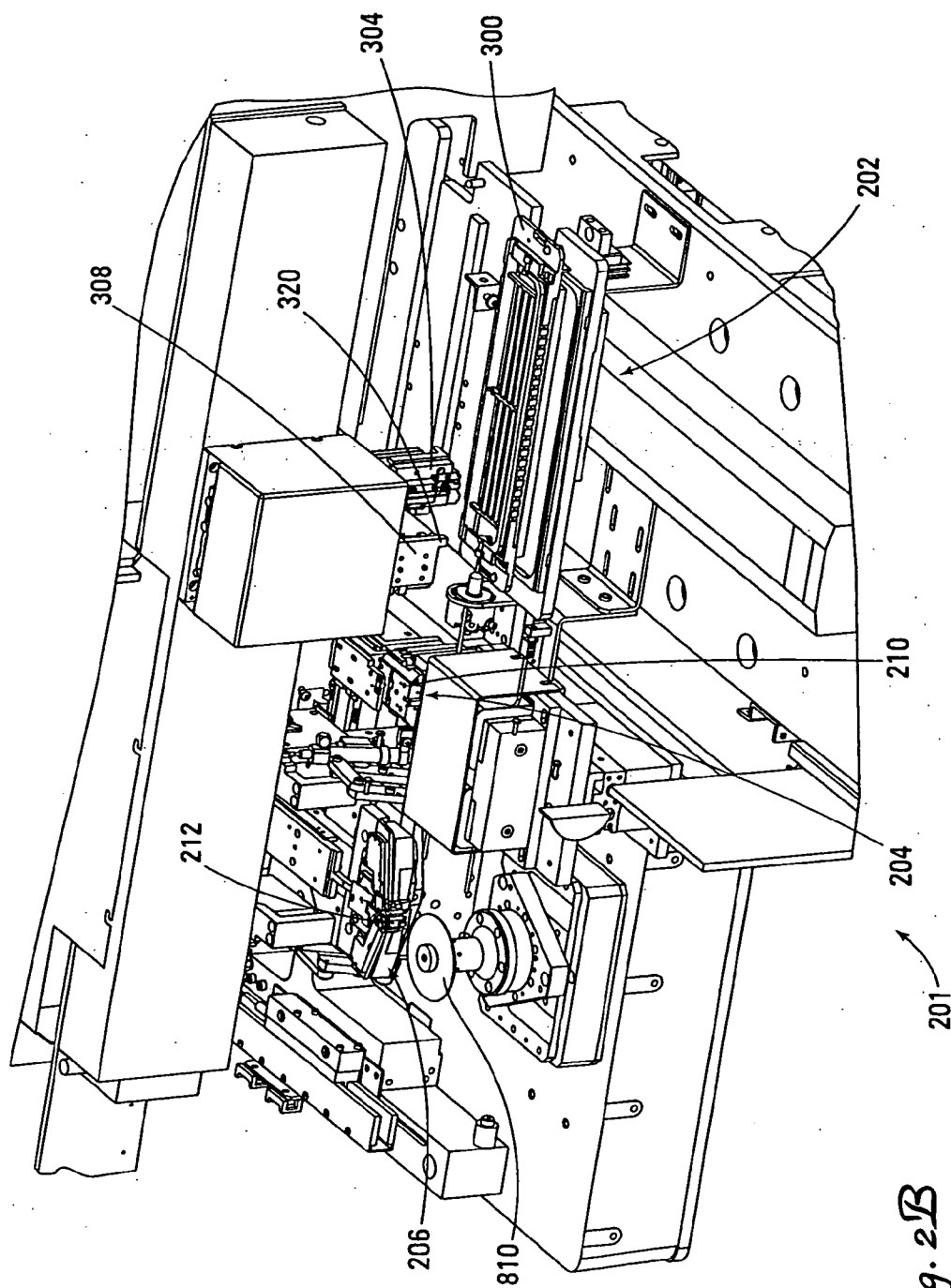


Fig. 2B

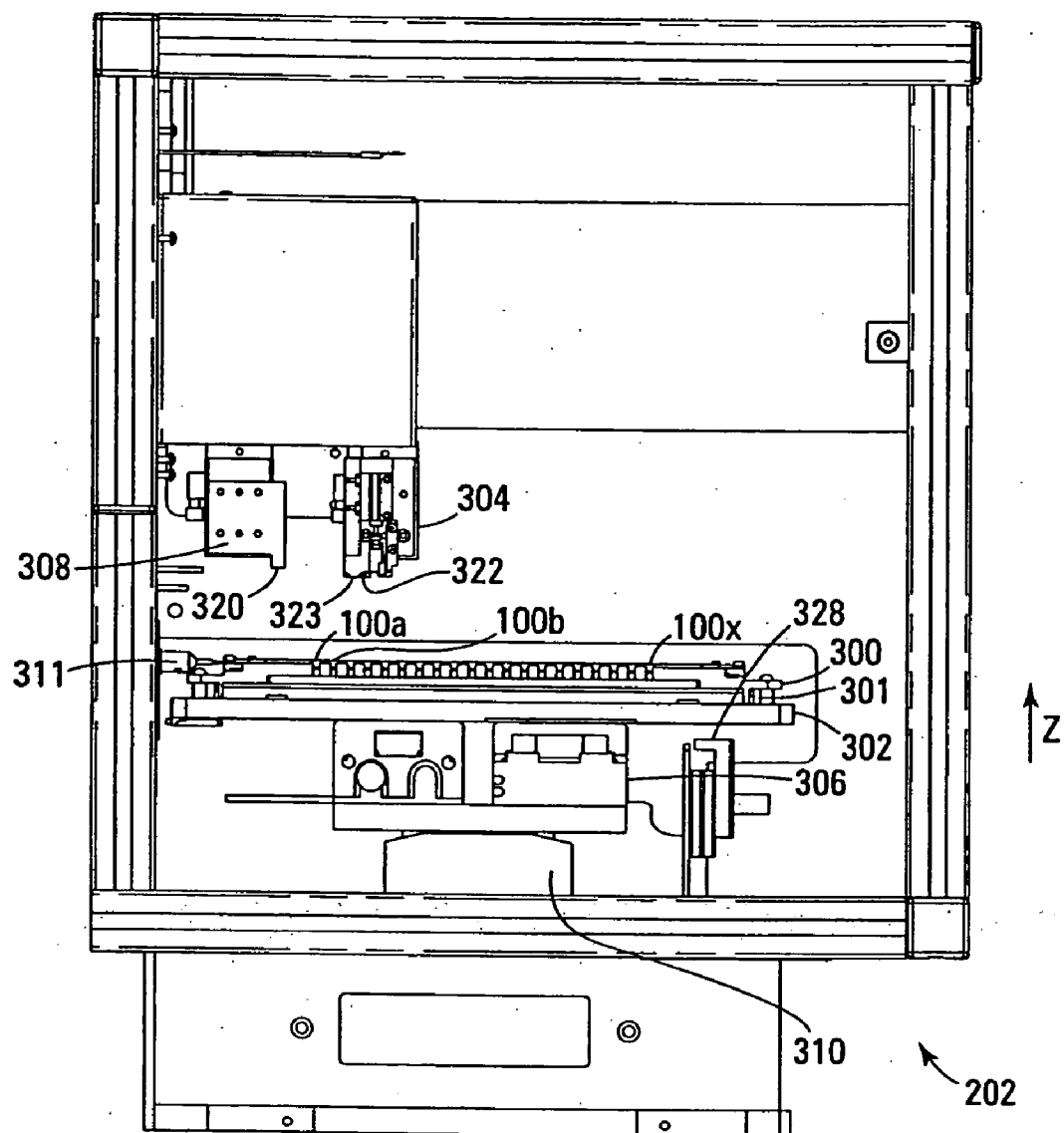


Fig. 3

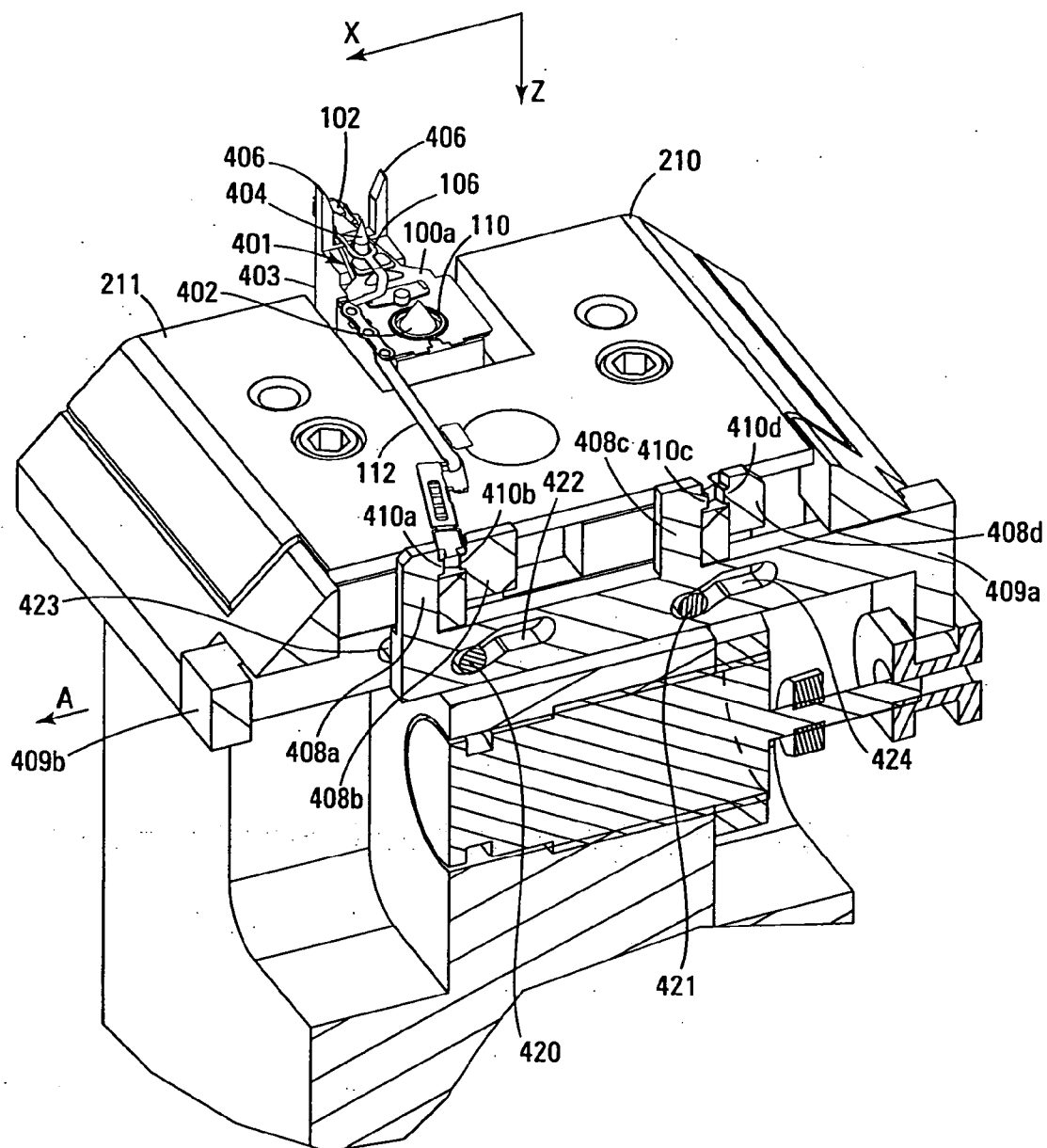


Fig. 4

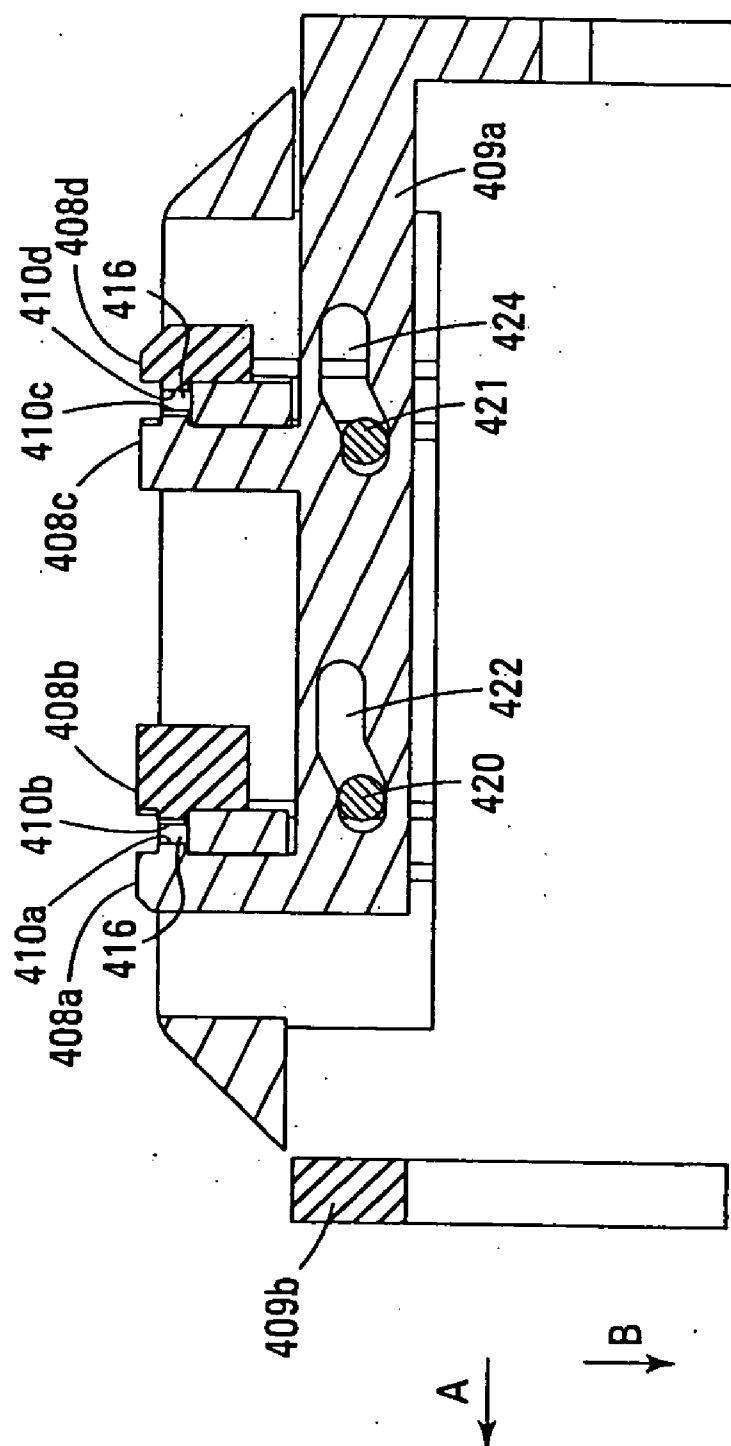


Fig. 4A

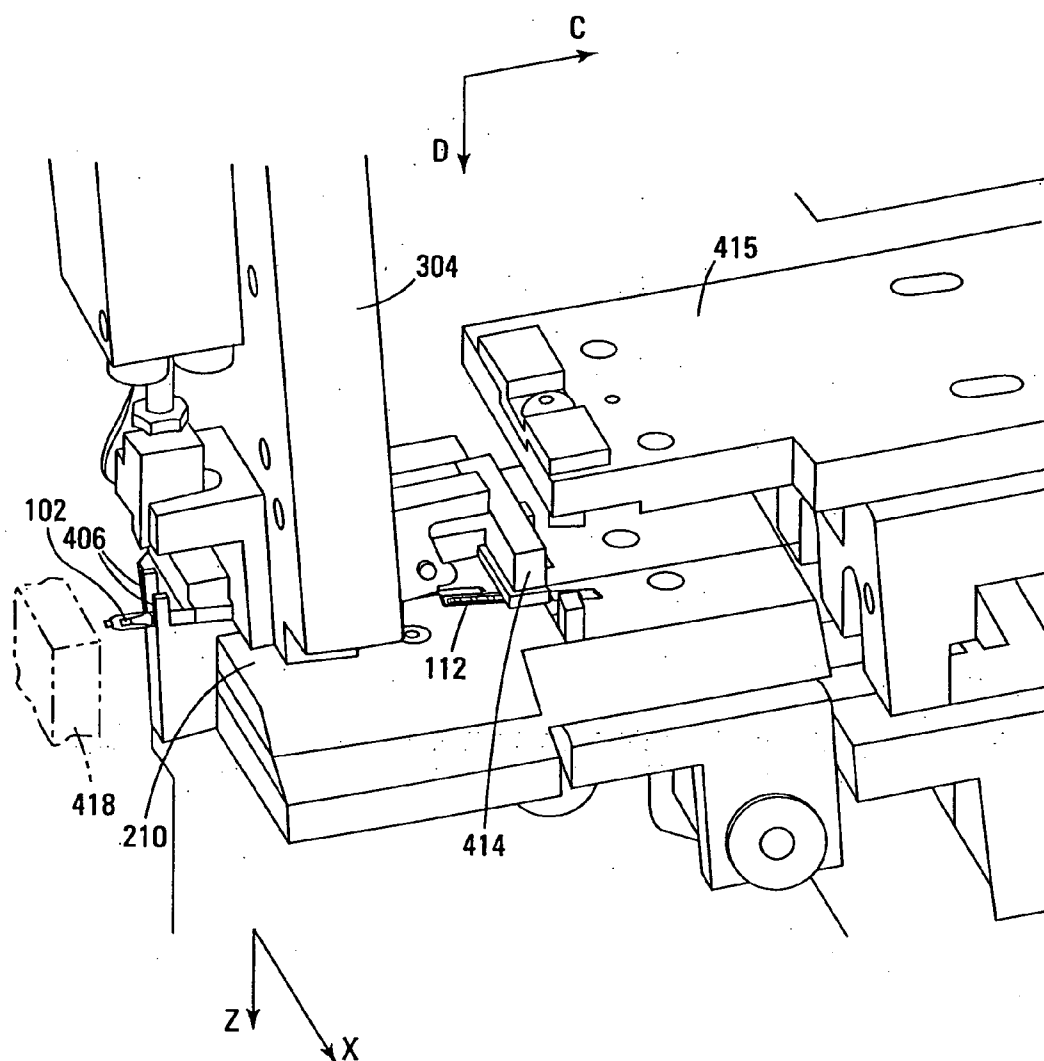
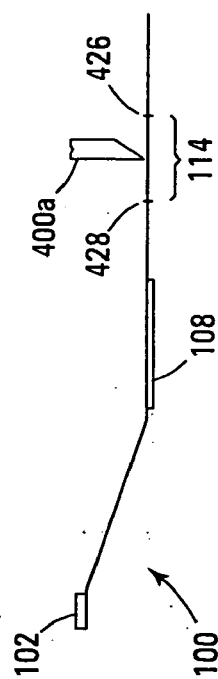
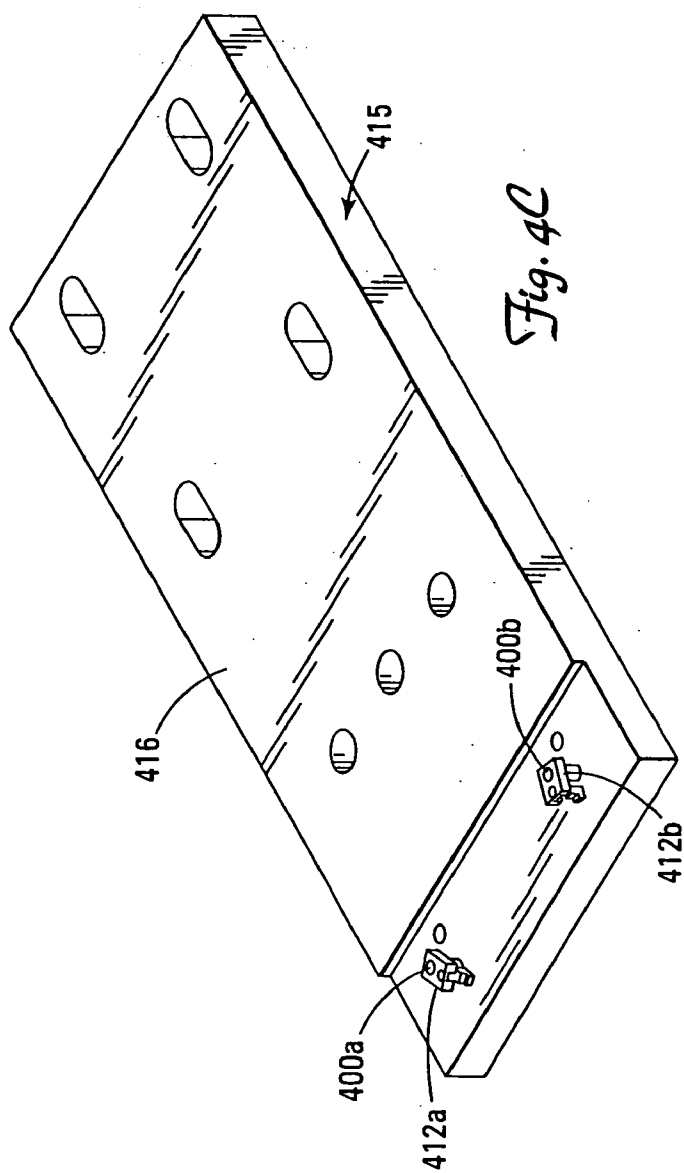


Fig. 4B



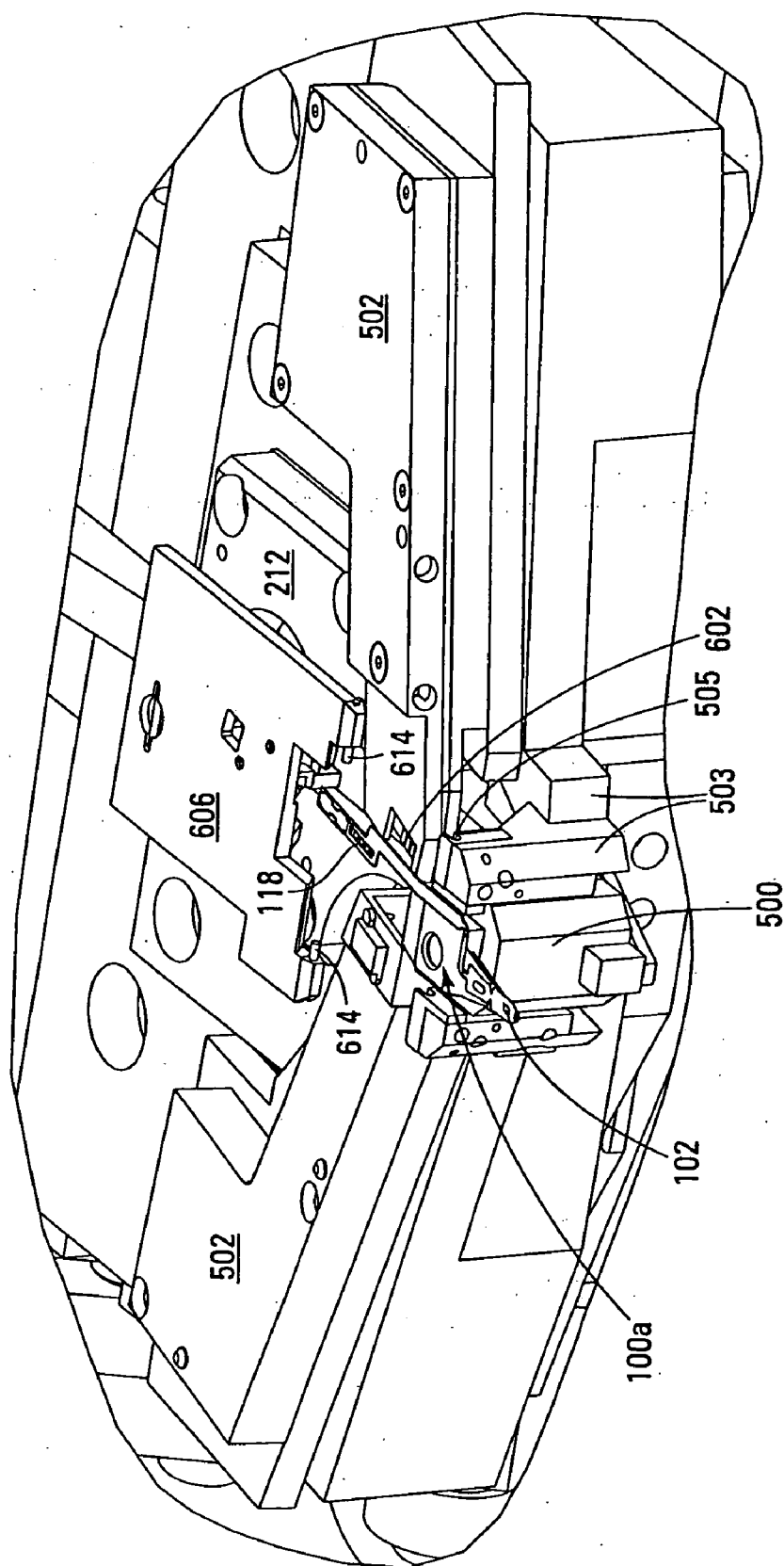


Fig. 5

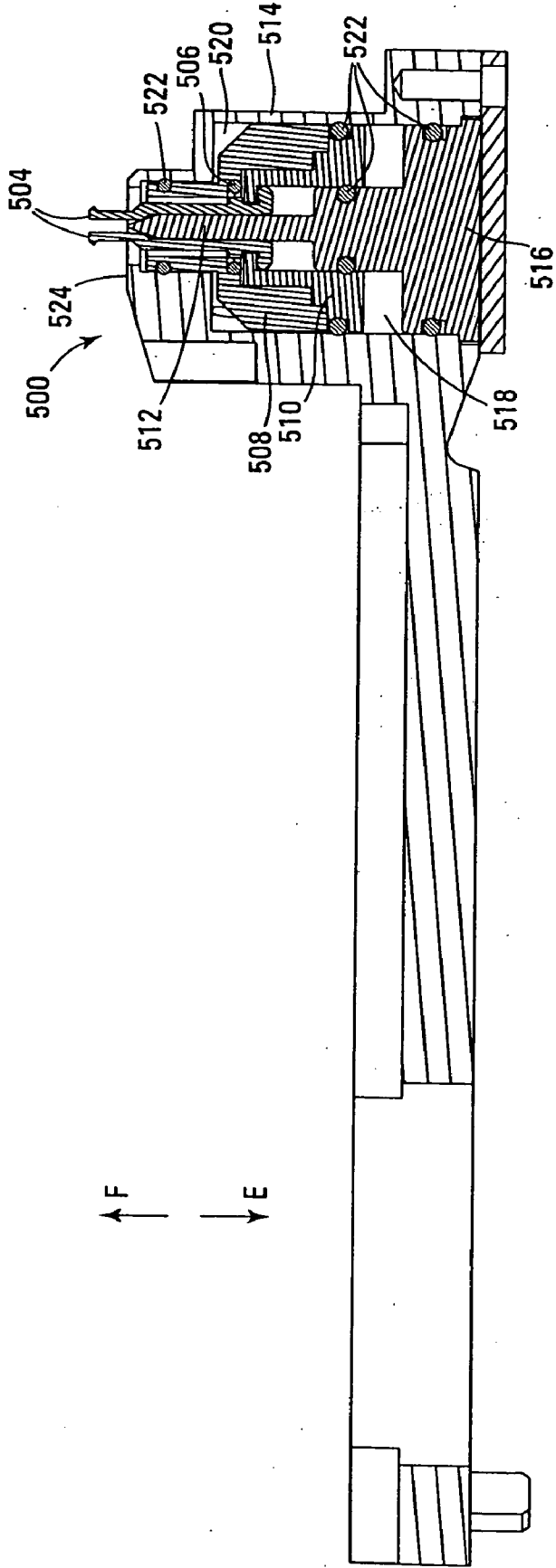


Fig. 5A

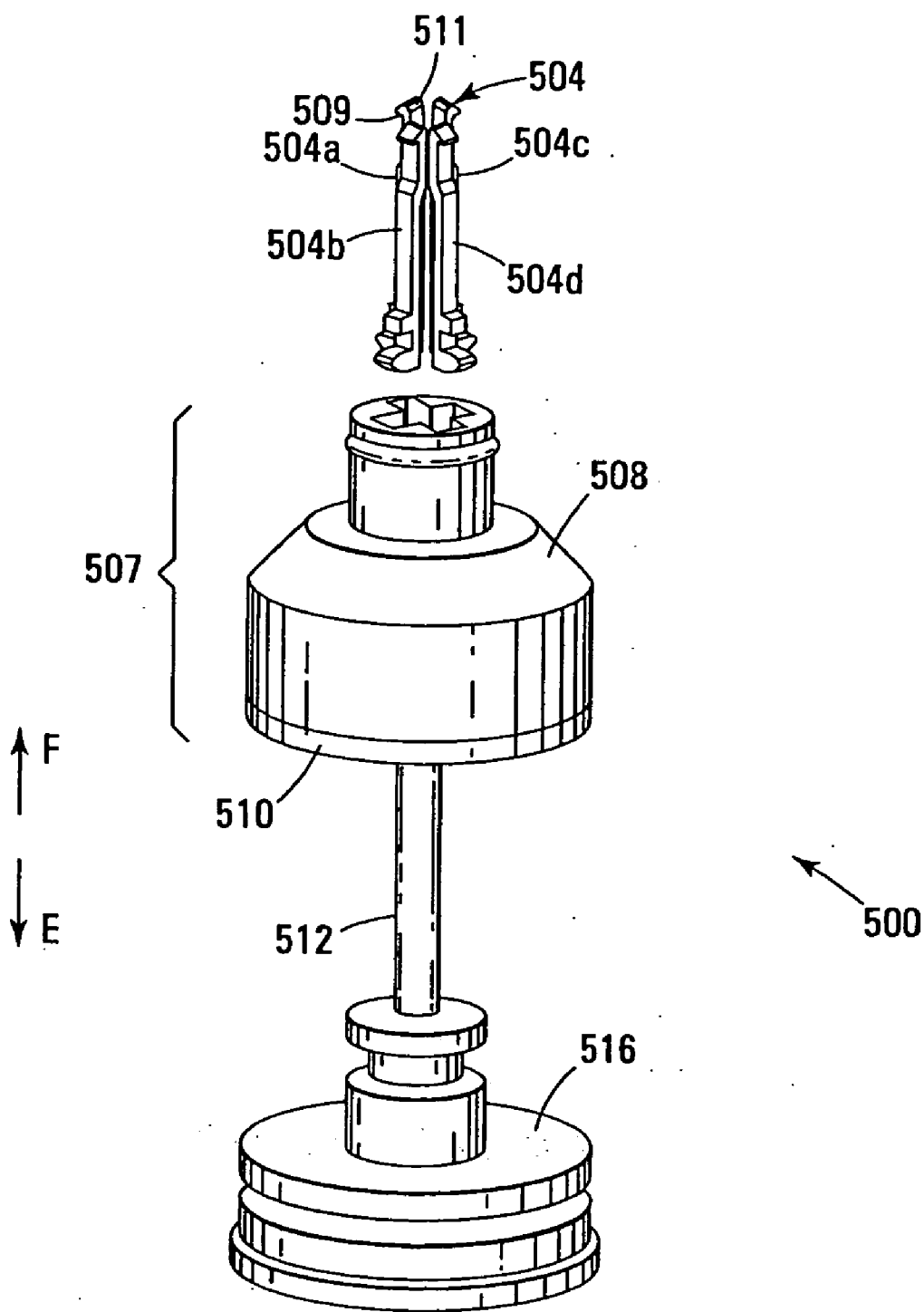
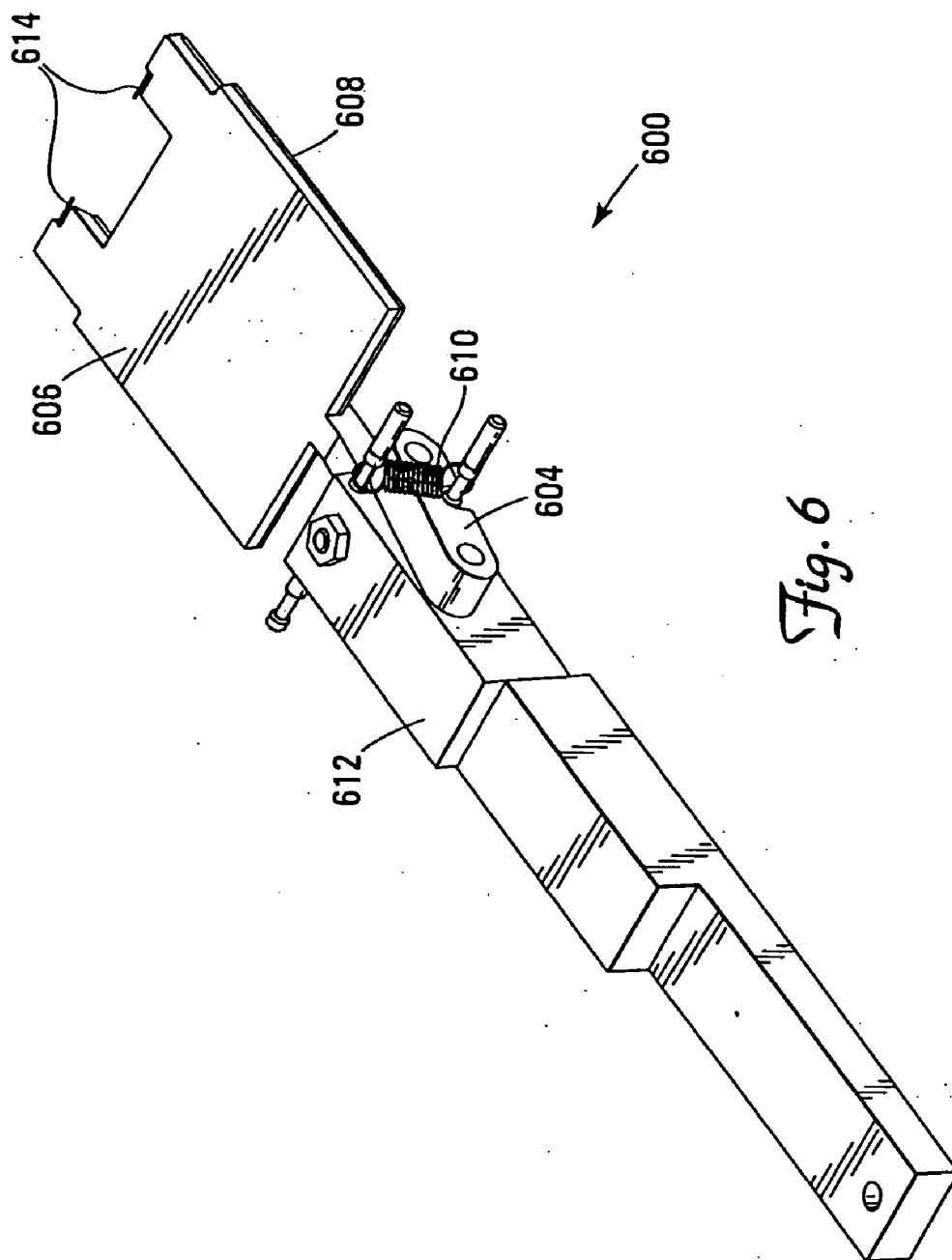


Fig. 5B



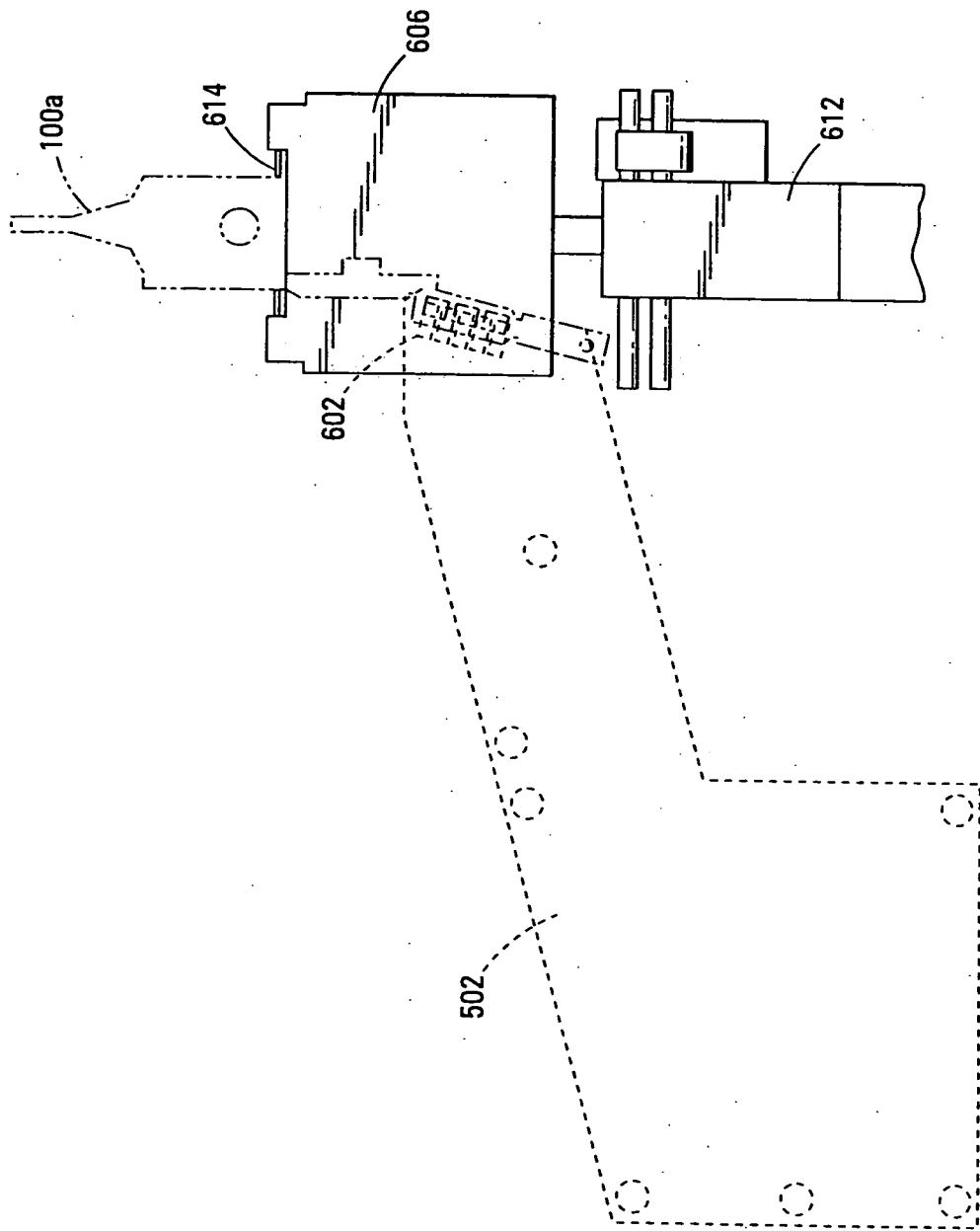


Fig. 6A

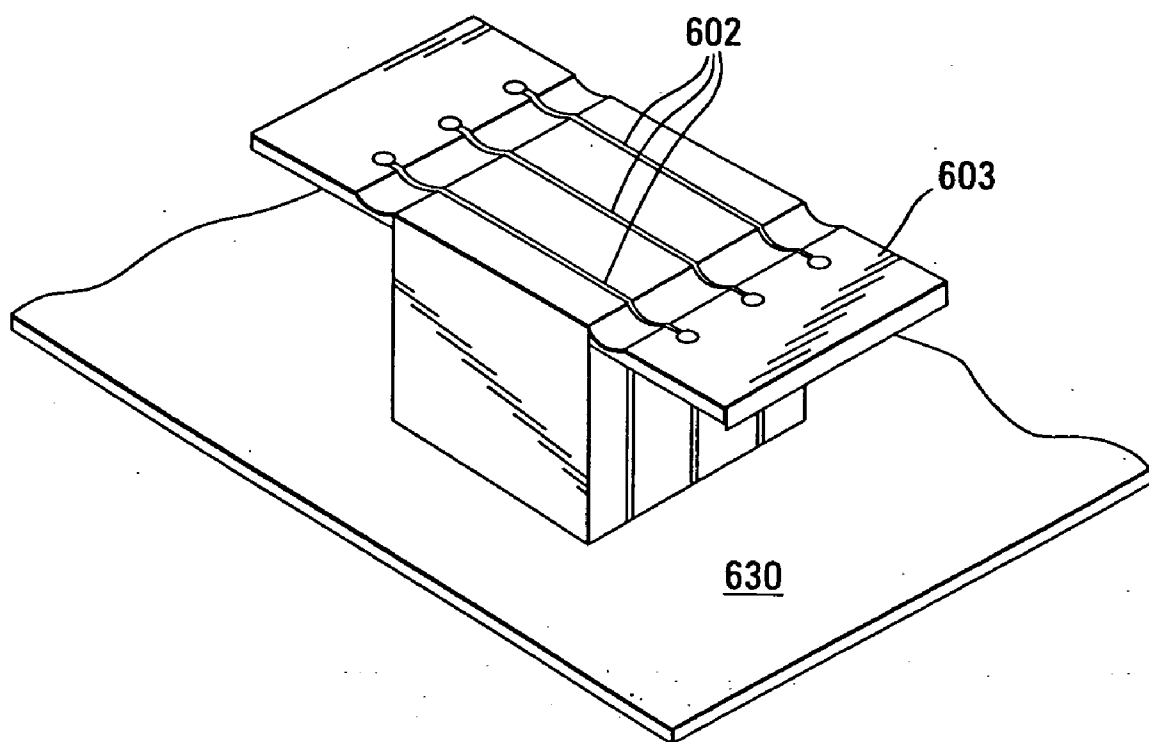


Fig. 6B

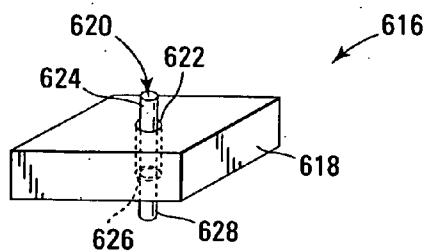


Fig. 6C

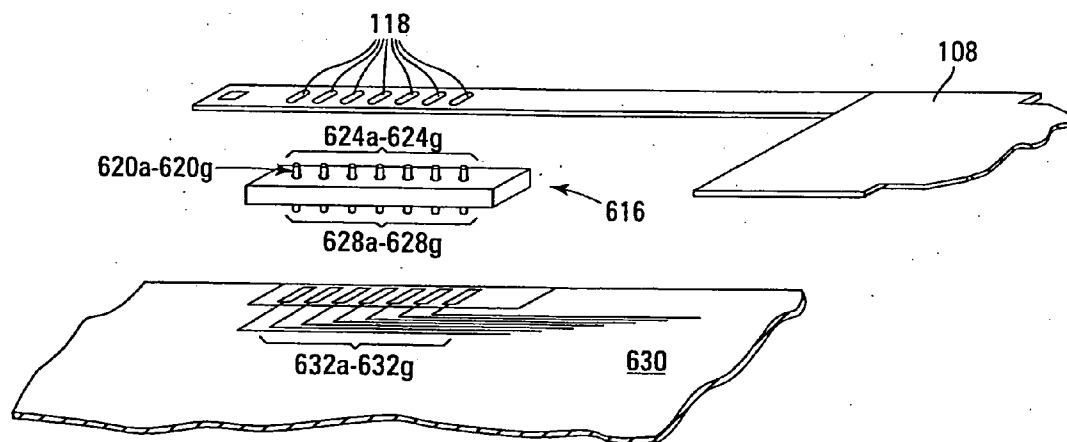


Fig. 6D

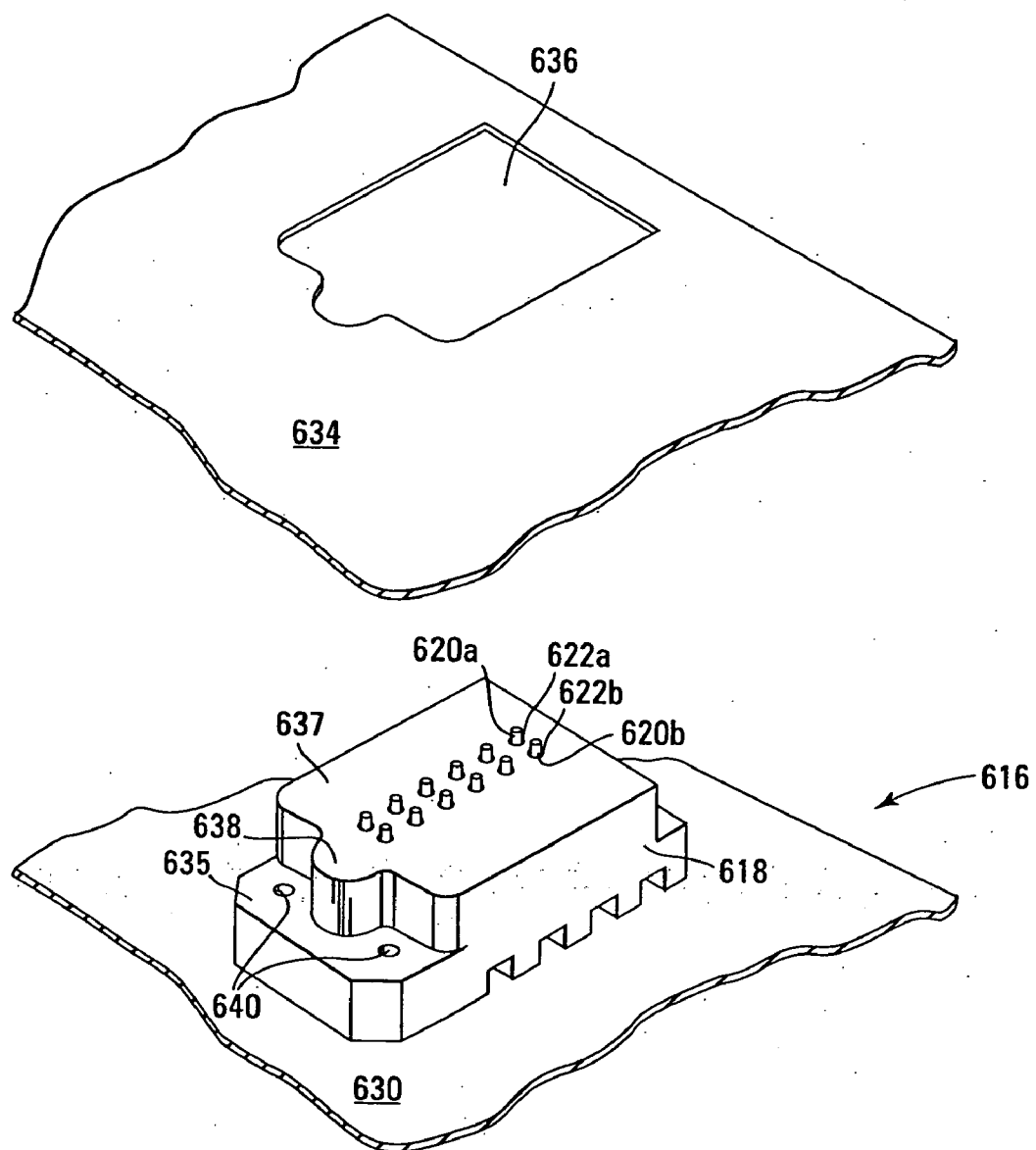
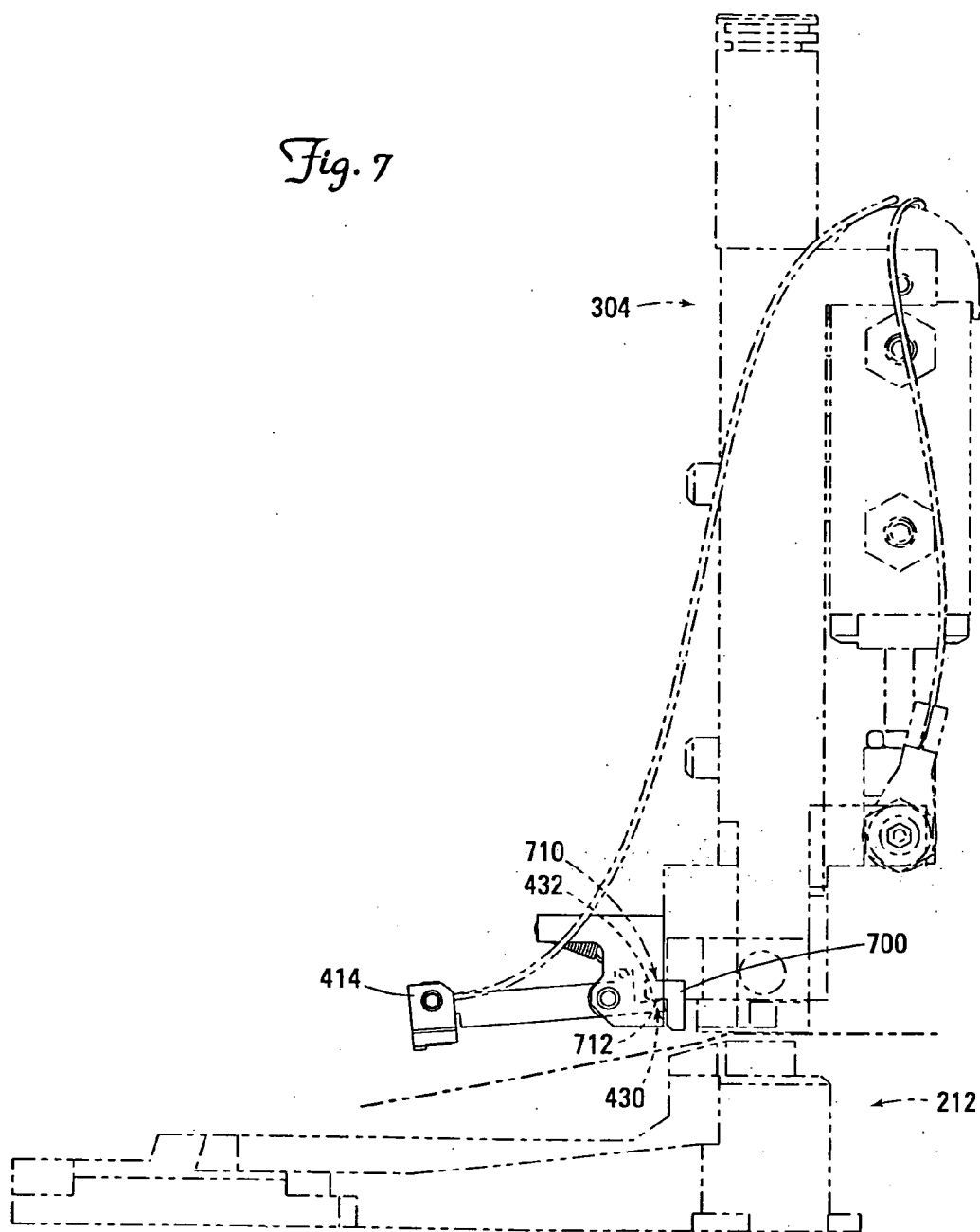


Fig. 6E

Fig. 7



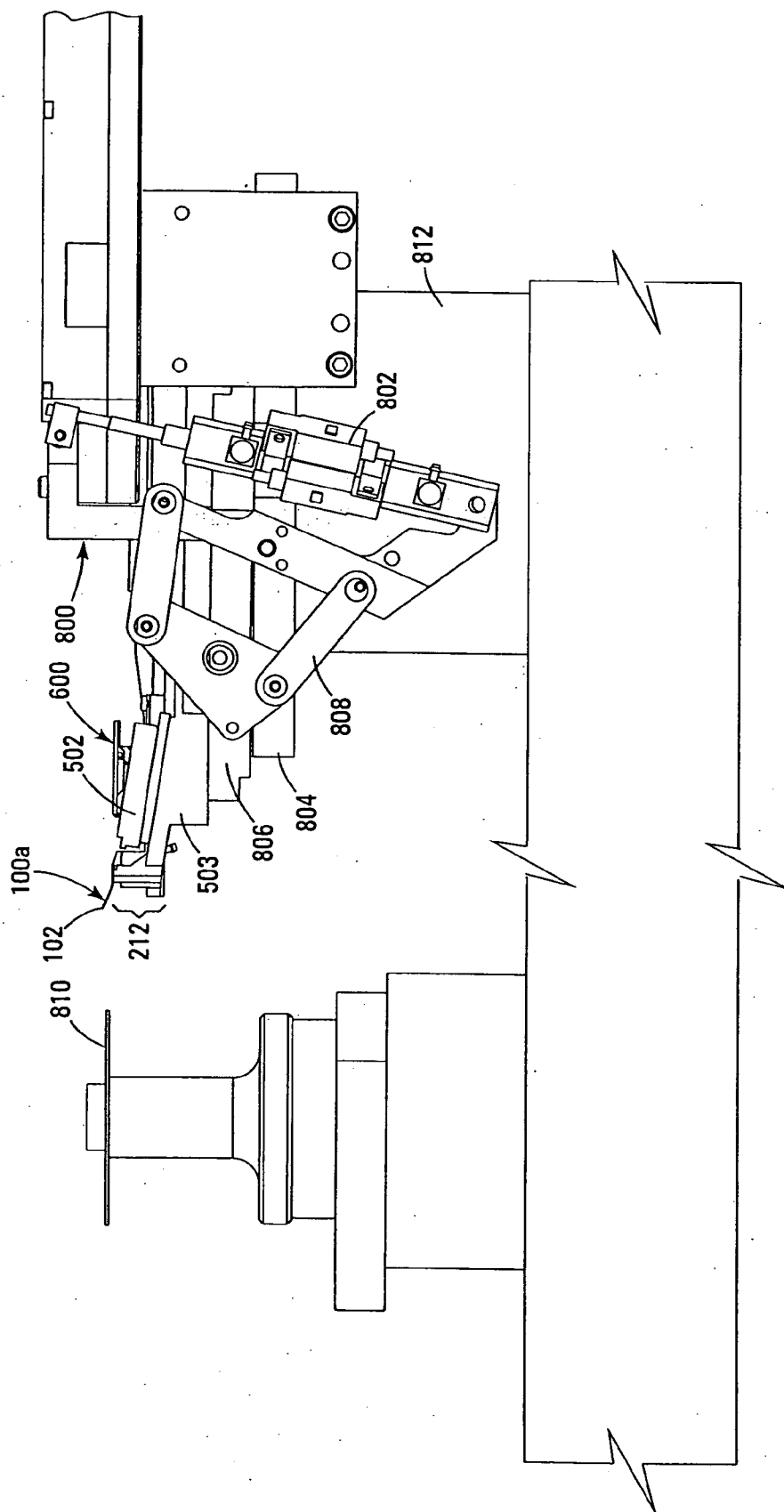


Fig. 8

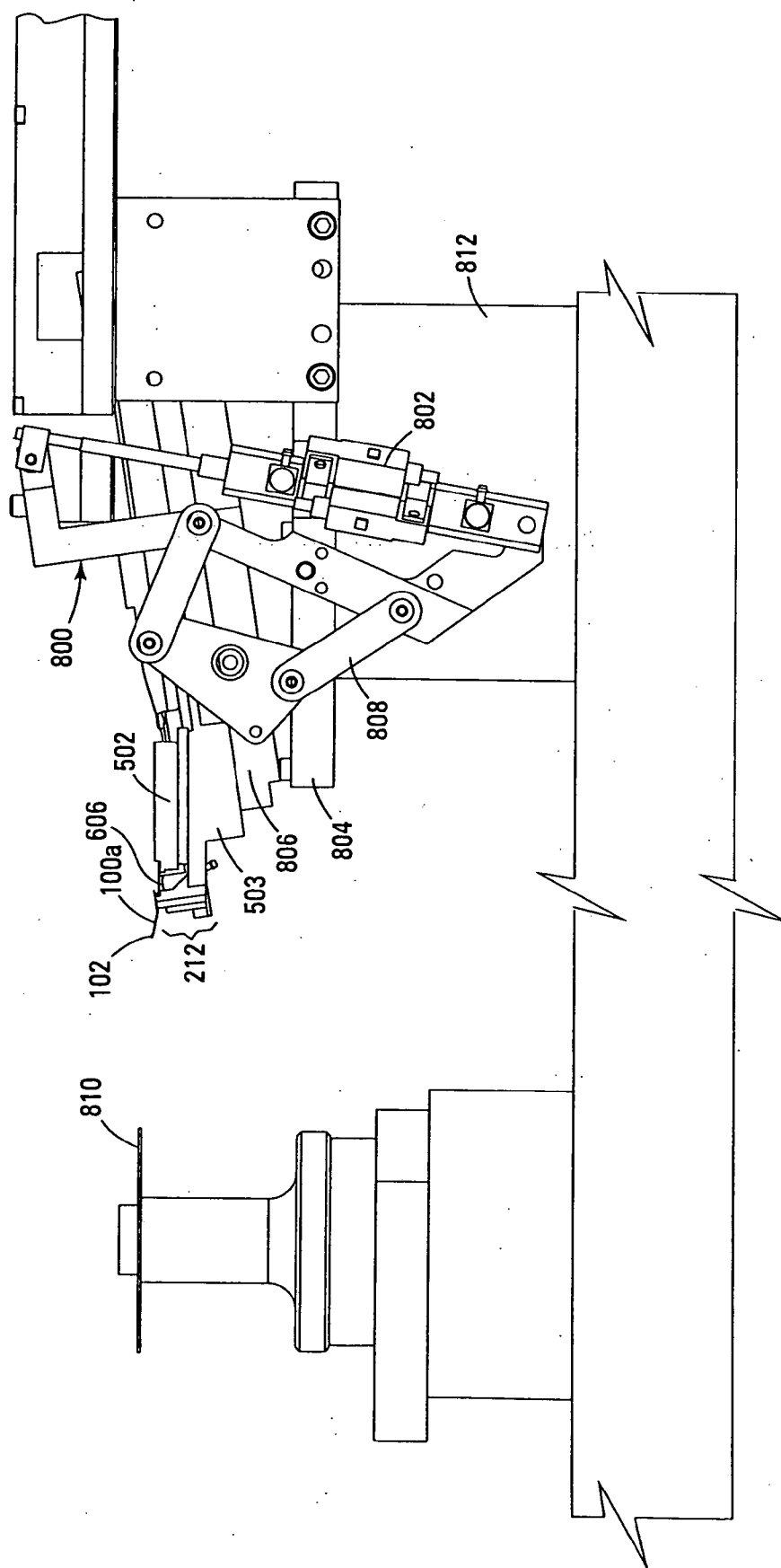


Fig. 8A

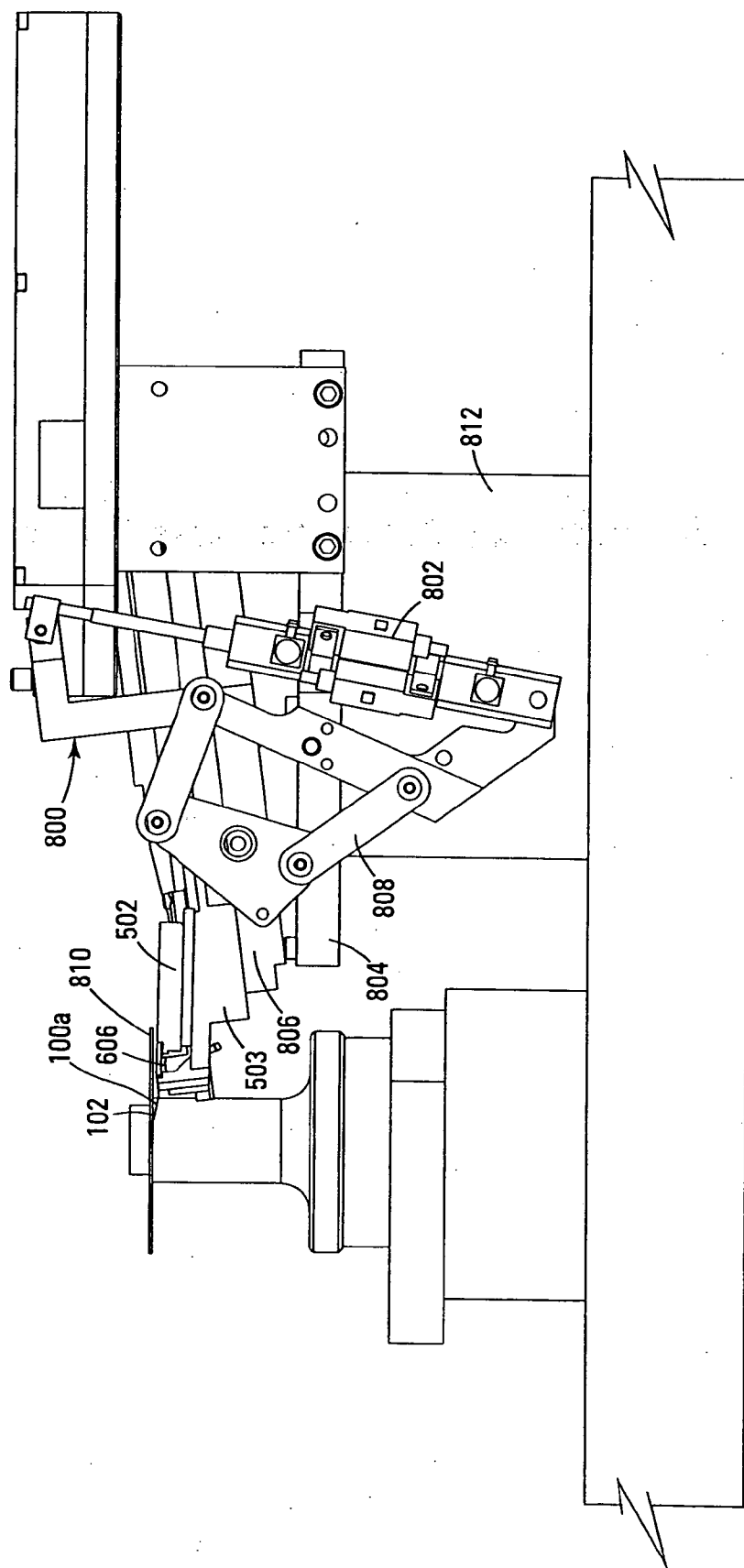


Fig. 8B

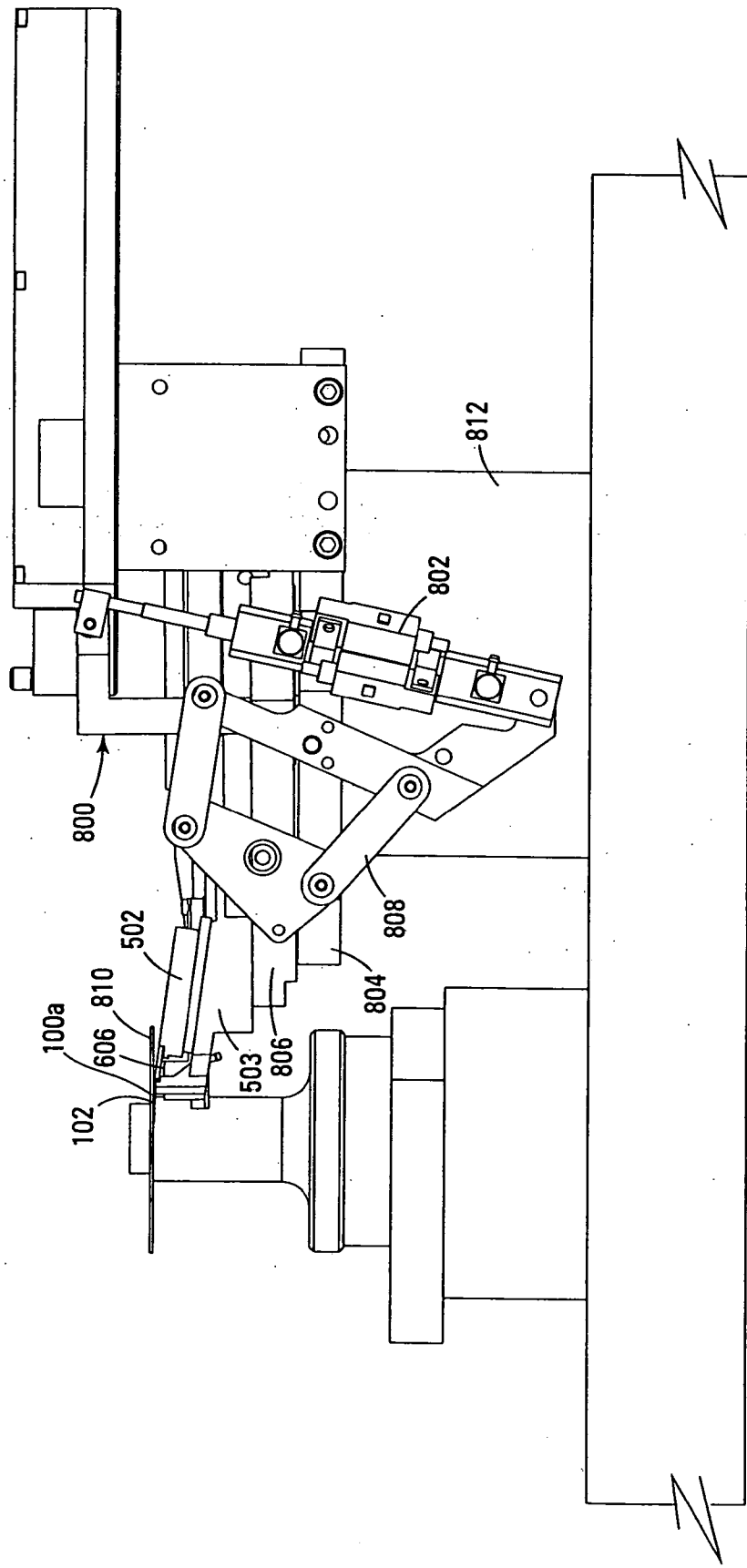
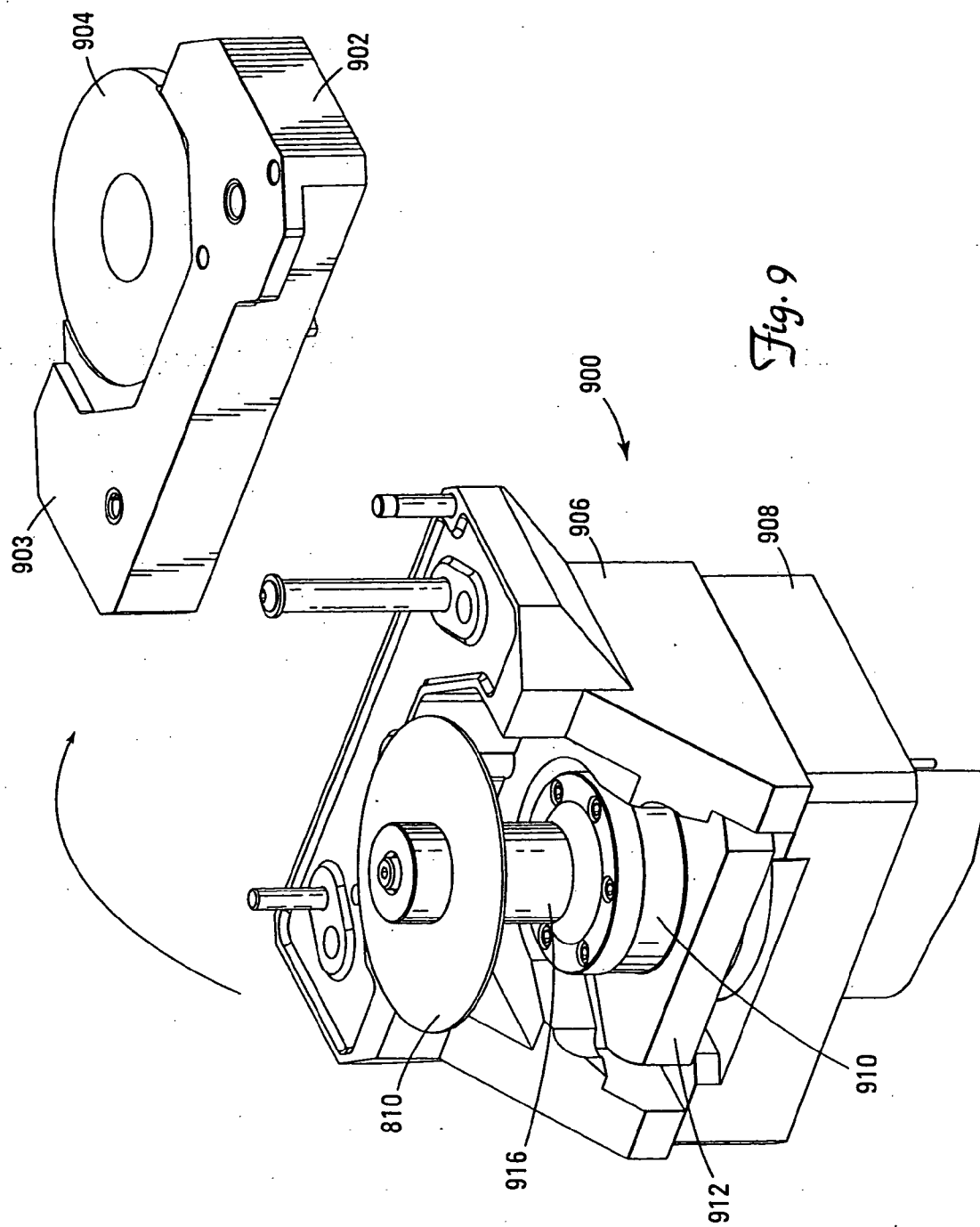


Fig. 8C



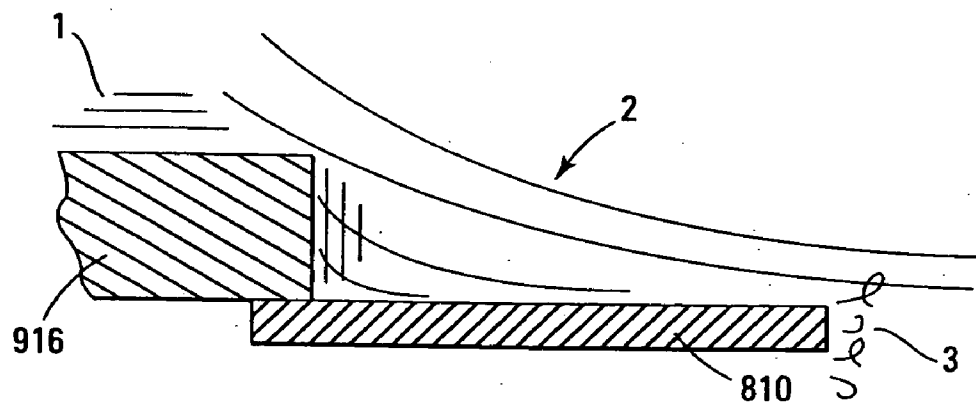


Fig. 11A

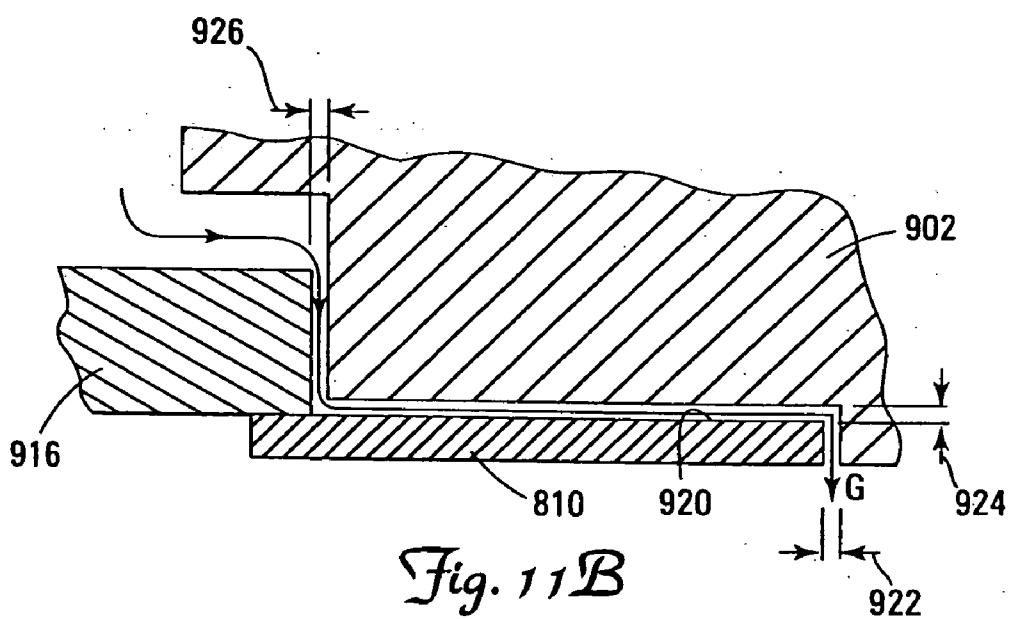


Fig. 11B

FASTENER

[0001] This application is a continuation of U.S. patent application Ser. No. 11/056,337, filed Feb. 11, 2005, which claims the benefit of U.S. Provisional Application No. 60/544,040, filed Feb. 12, 2004, the entire contents of which are incorporated herein by reference.

TECHNICAL FIELD

[0002] The invention relates to methods and apparatus for manipulating, retaining and electrically testing small parts, typically small electronic components. More particularly, the invention relates to methods and apparatus for testing head gimbal assemblies used in computer disc drives.

BACKGROUND

[0003] Prior to attaching a head gimbal assembly (HGA) into a hard disc drive, it is desirable to dynamically test the functionality of the read and write transducers that reside on the head gimbal assembly so that defective HGAs may be identified and sorted. Such testing can include preliminary activities to align, configure, and prepare the HGA for testing, followed by the actual electrical test of the HGA. Because HGAs are typically small, fragile, and contain sensitive electronic components, they are susceptible to mechanical stress, electro-static discharge (ESD), environmental contamination, and other handling-related issues.

[0004] To avoid these handling-related issues, current systems mount the HGA on an intermediate mounting fixture that supports the HGA throughout the testing process. An operator may manually place an HGA into an alignment tool that sets the orientation of the HGA to an intermediate mounting fixture. The alignment of the HGA to the intermediate mounting fixture is important because it helps determine the orientation of the HGA with respect to a disc during dynamic electrical testing. After alignment, a head set operation is performed in which the HGA and the intermediate mounting fixture are manually passed through a magnetic field to properly set the direction of the magnetic domains of the read and write transducers inside the head of the HGA.

[0005] Initially, the HGA's read and write transducers are electrically shorted together with a shunt tab, which resides on a flex circuit of the HGA and protects the HGA from ESD damage by ensuring that the components are held at a common voltage potential. This shunt tab must be broken or removed prior to testing the HGA. In current systems, the shunt tab is manually broken or cut off before the HGA is loaded into the electrical tester. After its removal, the HGA becomes extremely sensitive to ESD damage. Positioning the flex circuit for removal of the shunt tab is challenging because the flex circuit is flexible, and its position can vary over a relatively wide area. Additionally, flex circuits may have an inherent bend or twist, further complicating flex circuit positioning. In current systems, the intermediate mounting fixtures have positioning pins to precisely locate the flex circuit for de-shunting.

[0006] When the HGA is ready for electrical test, an operator can manually pick the HGA from a tray by grasping the intermediate mounting fixture and loading the HGA onto a dynamic electrical tester. During the dynamic electrical test procedure, the HGA's flex circuit makes interconnect

with the dynamic electrical tester's preamplifier, the HGA is loaded to a test disc, and the read and write transducers on the HGA are tested. Using this method of HGA manipulation and electrical testing requires a new intermediate mounting fixture to be designed and fabricated for each new HGA type. The intermediate mounting fixture generally consists of a clamping mechanism to hold the HGA base plate, a set of pins to locate the HGA flex circuit for interconnect, and a set of holes and pins to locate the intermediate mounting fixture at the various operations, including dynamic electrical test.

[0007] The clamping mechanism of the intermediate mounting fixture that holds the HGA base plate during electrical test has several requirements. As the bit density in disc drives increases, in operation the drive heads must fly lower with respect to the disc. This requires tighter tolerances for the HGA's orientation. Errors in the orientation influence Roll Static Attitude (RSA) and Pitch Static Attitude (PSA), which affect the HGA's ability to load to the disc and its fly characteristics after loading. RSA and PSA are effectively the head's static orientation relative to the disc. To ensure that the HGA's performance is consistent for both electrical testing and operation of the drive after installation, it is important the HGA be similarly constrained during both functions.

[0008] During operation of the drive, it is optimal for the HGA's base plate to be pulled down with approximately three-to-seven pounds of force relative to a reference surface and fastened by swaging a boss hole in the HGA to a rotary arm in the drive. In the past, the HGA's base plate has been held by attaching the HGA to an intermediate mounting fixture and then placing the intermediate mounting fixture on the tester. This may require manually screwing the HGA to the intermediate mounting fixture before placing it on the tester. While this is an accurate method of mounting and provides the needed downward force, it is very labor intensive and adds an extra amount of error contributed by the fixture to the testing process.

[0009] Another method of attaching the base plate to an intermediate mounting fixture involves using a flexure clamp jaw that clamps the HGA's boss hole. While this is less labor intensive than manually screwing the HGA in place, it still requires extensive manual handling of the part. It also does not provide any downward force, which leaves the base plate unconstrained and not flat. This negatively affects the test results.

[0010] Still another method includes mounting the HGA to an intermediate mounting fixture that holds the HGA by pinching it with a clamp between the back of the base plate and a pin through a swage hole present in the HGA. This method also does not provide sufficient downward force, but forces the back edge of the base plate to align to the clamp. Because the back edge of the base plate is not a controlled edge, this may cause misalignment during the testing process.

[0011] All of the above-described methods are difficult to automate and have costs associated with loading, purchasing, and maintaining the extra fixtures on which the HGAs are mounted. The intermediate mounting fixtures also create a larger mass and require an additional mechanical interface, both of which create another potential source of error or vibration during the dynamic electric test.

[0012] In current systems, pins on the intermediate mounting fixture align the interconnection pads on the HGA's flex circuit with the dynamic electric tester's preamplifier contacts so that interconnection between the two is achieved. This alignment is necessary because the flex circuit is flexible, which permits the location of the interconnection pads to vary over a relatively wide area. After the flex circuit is constrained for interconnection, the intermediate mounting fixture, which holds the HGA, is loaded onto the tester.

[0013] The set of holes and pins used to align the intermediate mounting fixture for testing the HGA affects Reader Writer Offset (RWO), which is an important measurement in the dynamic electrical test. RWO is the distance a read/write head jogs to read a track that it has just written. The RWO is a function of the skew angle of the head, which is the angle of the head relative to the center of the disc on the x-y plane, the reader-to-writer separation distance, and the reader-to-writer alignment on the head. Because RWO data is used to verify that the reader-to-writer separation and alignment are within the tolerance limits of the process controls, the process of loading the HGA to disc should be carefully controlled so that the position of the HGA's skew angle is both accurate and repeatable.

[0014] Challenges to accurately loading an HGA to a disc include maintaining the HGA's orientation precisely from the moment it is put on the load mechanism until it is loaded to the disc. Also, one must ensure that the structure stiffness of the load mechanism does not contribute to positional error during test. Additionally, the process of loading the HGA must be carefully controlled to prevent damage to the HGA. For instance, the HGA cannot be bent significantly beyond its normal operating state. It also must be presented to the disc at a shallow enough angle to prevent any features from unintentionally contacting the disc during loading.

[0015] The cost effectiveness of the loader is not only measured in direct hardware costs due to damaged parts, but also in its cost effectiveness on the testing process. For instance, costs associated with the loader include the down time for the tester when there are changes in product configuration. Another cost includes the cost of testing media, which is one of the greatest costs in HGA testing. HGAs can crash a disc for several reasons during test including contamination of the media, non-optimal load orientation of the HGA, and HGAs with extreme or out-of-specification roll or pitch values.

[0016] Currently, load mechanisms typically include vertical translating stages, ramp loads, or tilt mechanisms. The vertical translating stage maintains the base plate of the HGA parallel to the disc and lifts the HGA to the disc. New generations of HGAs have features on the load beam beyond the head that contact the disc before the head and can damage the disc. This can result in a limited number of loads before the HGA or media are crashed. The ramp load mechanism works well in the drive, but it is difficult to use in the testing process. The ramp is typically fixed in location at the parameter of the disc, which limits the loading of the HGA to only one radius. Once that radius is crashed, that disc must be discarded. Ramp loading also can result in damage to the HGA from the sliding action across the ramp if the appropriate materials or surface finish are not used. However, use of the ramp enables loading the HGA to the disc at a shallower angle.

[0017] The third and often used loading method utilizes a tilting mechanism. A stage pivots, lowering the HGA below the surface of the disc. Once the HGA is moved into position under the disc surface, the HGA is pivoted up to the disc surface. One of the challenges of this mechanism is where to locate the hinge. The ideal location of the pivot point is near the bend in the HGA. The hinge, however, needs to be in real space and cannot inhibit positioning the HGA at various places on the disc. Miniaturized loaders with small pivot bearings have been used, but it is difficult to achieve the required structural stiffness and maintain all the tight tolerance required with a small structure and still provide room to access the HGA with an electrical interconnection. Though the HGA can be loaded at an angle shallower than the vertical load, it does not sufficiently reduce stresses to the HGA during load.

[0018] Once the HGA's head is loaded to the disc, there are still many sources of positional disturbance that can affect the effective track density during testing. For instance, disc flutter is a result of exciting a disc at the disc's natural resonant frequencies. Internal and external sources, such as spindle motor vibrations or external air turbulence and acoustic vibrations, may create vibrations that excite a disc and create disc flutter. The flutter is primarily a vertical modulation of the disc while the disc is rotating. The modulation creates bends in the disc. The compliance in the HGA load beam allows the head to follow undulations of the disk, but because the base plate of the HGA is held fixed relative to an external reference there may be a small error. This error is significant at current and higher track densities. The radial motion contributes to the total asynchronous runout, which exists when position errors are asynchronous, or do not repeat on each disc revolution.

[0019] Some current systems use devices between the spindle motor and the disc to reduce flutter. In these systems, one must test the HGA's head on the disc side that is opposite of these devices. Common approaches either require testing the top surface of the disc, which necessitates that the spindle protrude down into the test stand, or inverting the spindle that holds the disc so that the testing can be performed on the bottom surface of the disc. Both approaches have disadvantages. Systems that test the top surface are at a disadvantage from a part handling standpoint because testing on the bottom surface is considered more compatible with how the HGA is presented to the HGA tester. Additionally, placing a disc on the spindle is more difficult in current systems that test the top surface because of the close proximity of the flutter reduction device to the seated disc. Current testers may need extra mechanisms, such as guide fingers, to guide the disc to its final position. Current systems that test the bottom surface of the disc also have disadvantages because they typically invert the spindle, which creates a number of structure challenges in order to maintain the required rigidity and access for disc changes and other service needs.

[0020] During dynamic electrical testing, the intermediate HGA mounting fixture also has a large effect on the test and the test results. The intermediate mounting fixture can add to the stack-up tolerance related to the HGA's z-height causing small shifts in fly height. By increasing the mass that the tester micro-positioner must move while testing, the presence of the intermediate mounting fixture can lead to lower dynamic electrical tester track per inch (TPI) capability. The

size of the intermediate mounting fixture also can limit the radii and skew locations that the HGA is loaded onto and unloaded from the disc. By limiting the load radii and skew options, the tester may use more media and take more time to load the HGA, which decreases the number of HGAs tested in a given period of time.

SUMMARY

[0021] In one aspect, the invention is a computer-controlled, automated method and apparatus for loading, aligning, and testing an HGA that is not mounted on an intermediate mounting fixture (referred to herein as an unmounted HGA). Advantages of this apparatus and method include, for example, avoiding time and capital costs associated with testing an HGA mounted on an intermediate mounting fixture; and avoiding time and capital costs associated with maintaining the intermediate mounting fixture. The method and apparatus also improve the efficiency, accuracy, precision, and repeatability of aligning an HGA for testing.

[0022] In another aspect, the invention is a method and apparatus to provide a more efficient method to de-shunt and headset an unmounted HGA. In yet another aspect, the invention is an automated method and apparatus for straightening an unmounted HGA's flex circuit without causing damage during de-shunting.

[0023] Also, the invention is a method and apparatus that provides a computer-controlled, automated method to accurately locate and break the shunt tab on an unmounted HGA.

[0024] The advantages of these methods also include: avoiding contact between the disc and HGA clamping mechanism; providing a clamping force on the HGA similar to the force applied after the HGA is installed in a drive; improving the efficiency of accurately aligning the HGA for testing; and minimizing additional positional error or vibration by lowering the mass required to hold the HGA and eliminating extra mechanical interfaces.

[0025] In yet another aspect, the invention also includes a computer-controlled automated method and apparatus for correcting the positional errors of the flex circuit on an unmounted HGA prior to interconnect with a pre-amplifier. This method and apparatus overcomes errors caused by the differing heights of the pre-amplifier contacts and the flex circuit layers; and improves the clamping mechanism for pressing a flex circuit against a preamplifier's contacts.

[0026] Other aspects include: providing a HGA loader that minimizes head angle to disc during loading; permitting loading of an HGA to a disc at multiple radii; providing lock down of the loading device during test; improving the accuracy and cost efficiency of loading an HGA to a disc; reducing the time required to load an HGA; and providing a device that easily accommodates different HGA geometries.

[0027] In another aspect, the invention includes a method and apparatus for improving suppression of disc flutter; providing more efficient loading of media; providing more efficient access to the underside of the disc during testing; and providing more efficient access to the structures surrounding the disc for servicing.

[0028] The details of one or more embodiments of the invention are set forth in the accompanying drawings and

the description below. Other features, objects, and advantages of the invention will be apparent from the description and drawings, and from the claims.

DESCRIPTION OF DRAWINGS

[0029] FIG. 1A is a perspective view of an unmounted HGA;

[0030] FIG. 1B is a schematic side view of an HGA;

[0031] FIG. 2 is a block diagram of an exemplary device that can test unmounted HGAs;

[0032] FIG. 2A is one embodiment of a device that can test unmounted HGAs;

[0033] FIG. 2B is an enlarged view of the device in FIG. 2A without a portion of its support structure.

[0034] FIG. 3 is one embodiment of a tray load and unload area of the device shown in FIGS. 2A and 2B.

[0035] FIG. 4 is one embodiment of a précising nest;

[0036] FIG. 4A is a more detailed view of flex-on-suspension (FOS) aligners shown in FIG. 4;

[0037] FIG. 4B is one embodiment of a précising area including an electromagnet, the précising nest, and a device for de-shunting the HGA shown in FIGS. 1A and 1B;

[0038] FIG. 4C is a more detailed view of a portion of the de-shunting device shown in FIG. 4B;

[0039] FIG. 4D is a diagram of one embodiment of a de-shunt pin used for de-shunting.

[0040] FIG. 5 is one embodiment of a portion of a test area.

[0041] FIG. 5A is a sectional view of an embodiment of a nest with a collet assembly;

[0042] FIG. 5B is an exploded view of the collet assembly shown in FIG. 5A;

[0043] FIG. 6 is one embodiment of a clamp wing assembly that may be used to hold the FOS against an electrical contact;

[0044] FIG. 6A is one embodiment of the clamp wing assembly in the state of holding the FOS against the electrical contact.

[0045] FIG. 6B is one embodiment of the electrical contact using gold blocks.

[0046] FIG. 6C is another embodiment of the electrical contact using a solderless connector.

[0047] FIG. 6D is a more detailed view of the electrical connector shown in FIG. 6C with optional features.

[0048] FIG. 6E is an alternative embodiment of the electrical connector with pairs of conductors.

[0049] FIG. 7 is one embodiment of a tail pusher and a tail flattener;

[0050] FIG. 8 is one embodiment of a four bar loader;

[0051] FIG. 8A is the device shown in FIG. 8 tilted below the disc in preparation for testing the HGA;

[0052] FIG. 8B is the device shown in FIG. 8 moved under the disc;

[0053] FIG. 8C is the device shown in FIG. 8 loaded to the disc;

[0054] FIG. 9 is one embodiment of a disc flutter control device;

[0055] FIG. 10 is a section view of embodiment shown in FIG. 9;

[0056] FIG. 11A is air flow over the disc without the disc flutter control device shown in FIG. 9; and

[0057] FIG. 11B is air flow over the disc with the disc flutter control device shown in FIG. 9.

[0058] Like reference symbols in the various drawings indicate like elements.

DETAILED DESCRIPTION

[0059] As shown in FIG. 1A, the basic components of an HGA 100 are a head 102, a load beam 104, a tooling hole 106, a base plate 108, a boss hole 110 with an angled surface 110a, and a flex circuit 112 with a flex circuit pads 118 and a shunt tab 114. The head 102 flies above the surface of a disc and contains the read and write transducers. As shown in FIG. 1B, the load beam 104 is a thin, metal structure that has a bend, which provides the spring force to hold the HGA adjacent to the disc during operation. The angle of the bend with respect to the base plate 108 is the free state angle 116.

[0060] FIG. 1A shows the base plate 108, which is retained during testing to permit manipulation and alignment of the HGA assembly, and eventually, is mechanically fastened into a disc drive. The boss hole 110 and the tooling hole 106 are used for aligning the HGA. The flex circuit 112 and its components will subsequently be described in more detail.

[0061] To eliminate the intermediate mounting fixture from the dynamic electrical testing of HGAs, all of the functions that the intermediate mounting fixture completed now must be completed through some means that does not travel along with the HGA. The absence of an intermediate mounting fixture in a process is referred to herein as an unmounted process. However, even in an unmounted process, the test method and apparatus must still pick, align, headset, de-shunt, load onto a test nest, interconnect with a preamplifier, and load the HGA to the disc. The small size and fragility of the HGA makes it necessary that all of those operations be mechanically controlled.

[0062] One embodiment of a device that does not use an intermediate mounting fixture is the unmounted HGA tester (UHGAT) 200 shown in the block diagram of FIG. 2. The UHGAT 200 includes, but is not limited to, four main functional areas. Three of the functional areas are process areas: the tray load/unload area 202, the précising area 204 including a précising nest 210, which aligns, de-shunts, and headsets the HGAs, and the test area 206 including a test nest 212, which interconnects the HGA with a preamplifier and loads the HGA to the disc. The fourth functional area, the control area 208, controls the UHGAT automation and eliminates the need for operator intervention during the UHGAT dynamic electrical test.

[0063] FIG. 2A shows a drawing of one embodiment 201 of the UHGAT device 200 with support structures for a fan 209 and actuators described later in greater detail. The UHGAT device 201 shown in the figure includes the tray load/unload area 202, the précising area 204, and the test area 206. The control area 208 is not shown in FIG. 2A. The control area 208 includes a computer, a display with a user interface, an input device, and circuit boards with embedded controllers. An operator may use the user interface to specify settings for the testing process, such as the gauss level during headset at the précising nest 210, and initiates testing with the UHGAT software installed on the computer. Control of the UHGAT machinery is accomplished through the embedded controllers and the UHGAT software. The UHGAT automation ties together the three processing areas by controlling the transfer of HGAs from the tray load/unload area 202 to the précising area 204, and then to the test area 206 before the automation finally brings the HGA back to the tray load/unload area 202 as illustrated in FIG. 2 by the arrows 203, 205, and 207, respectively.

[0064] FIG. 2B is an enlarged view of the device 201 in FIG. 2A without a portion of its support structure. With the removal of the support structure, one may more clearly see the three processing areas. In the embodiment shown, there may be a tray 300 in the tray load/unload area 202. The précising nest 210 may be in the précising area 204, and the test nest 212 and a disk 810 may be in the test area 206. Additionally, automation that moves the HGAs to and from each area may consist of two end effectors, a first end effector 304 and second end effector 308, that use vacuums to hold the HGAs while they are transferred.

[0065] FIG. 3 shows one embodiment of the tray load/unload area 202 of the device 201. The first and second end effectors 304 and 308 are each attached to a first and second pneumatic linear actuator (not shown in FIG. 3), respectively, which are attached to a main pneumatic linear actuator (not shown in FIG. 3). The pneumatic linear actuators move the end effectors in the 'z' direction and are controlled using conventional techniques with solenoid valves, hall sensors and flow controls. The main pneumatic linear actuator is attached to a linear motor positioning stage (not shown) and is also controlled using conventional techniques by a solenoid valve, hall sensors and flow controls. More expensive and complicated servo controlled actuators could replace any of the pneumatic linear actuators if the higher precision and velocity control could justify the increased cost. The linear motor positioning stage moves the first and second end effectors 304, 308 between the tray load/unload area 202, the précising area 204, and the test area 206. The linear motor positioning stage may be a high precision and high-speed servo-controlled actuator. A very accurate high-speed servo controlled actuator is needed to ensure that the HGA is delivered to the correct position very accurately in a minimum amount of time.

[0066] As shown in FIG. 3, the first processing area is the tray load/unload area 202 where an operator loads and unloads trays containing HGAs. After the tray load/unload area 202 is loaded with the tray 300, which may be done manually by an operator or using an automated placement process, the UHGAT automatically identifies the HGAs by reading a radio frequency (RF) tag 301 associated with or

placed inside the tray. Once the trays have been loaded, the UHGAT picks the HGA from the tray using a vacuum 322 of the first end effector 304.

[0067] When an HGA is removed from the tray it is moved to a second processing area, the précising area 204, where it is aligned on the précising nest 210, de-shunted and headset. After these operations are complete, the HGA is moved using the end effector 304 from the précising area 204 to a third processing area, the test area 206. During transfer from the précising area to the test area, the HGA maintains the alignment set on the précising nest 210. After placement on the test nest 212, the flex circuit 112 (See FIG. 1A) is aligned and moved down to make interconnect with a preamplifier. The preamplifier contacts are large flat gold contacts that cover the tolerance range of the flex circuit's 112 position.

[0068] Next, the linear motion positioning stage moves both first end effector 304 and second end effector 308 back to the tray load/unload area 202. The next HGA is picked from the tray and brought to the précising area 204. Then the first HGA is loaded to the disc using a four bar loader 800 (See, for example, FIG. 8) and dynamic electrical testing proceeds. After electrical testing is complete, second end effector 308 removes the first HGA from the test nest 212. The next HGA, already on first end effector 304, is loaded onto the test nest 212. The same sequence continues until all of the parts in the trays have been tested.

[0069] The following text describes each of the three functional process areas in more detail beginning with the tray load/unload area 202. The steps that may occur include placing a tray in the load/unload area and initiating the automated testing process. The tray's presence is sensed, and the tray's RF tag is read. Based on the read information, untested HGAs are selected for testing and the testing process can be configured based on the type of HGA in the tray. The tray's lid is opened, and an HGA is picked up off the tray. Next, the HGA is moved from the tray load/unload area 202 to the précising area 204.

[0070] FIG. 3 shows a tray 300 in tray load/unload area 202. Other trays may be in the tray load/unload area 202. An operator places the tray 300 on a tray holder 302, wherein the holder 302 can accommodate two trays. Moreover, the holder 302 is attached to a tray pneumatic actuator 306, which, in turn, can be attached to a tray load/unload actuator 310. More than one tray pneumatic actuator may be connected to the tray load/unload actuator 310. For example, the actuator 310 may be connected to two tray pneumatic actuators, wherein each of the tray pneumatic actuators is attached to a tray holder. This configuration allows for independent movement of each tray holder using the separate tray pneumatic actuators and for movement of all the tray holders using the tray load/unload actuator 310.

[0071] The tray 300 includes a tray lid (not shown in FIG. 3) and an RF tag 301. HGAs 100a-100x are located on the tray. In one embodiment, the tray load/unload area 202 accommodates up to four trays and each tray can hold up to twenty HGAs. The trays used in the UHGAT are the same trays that are used in operations previous and subsequent to the UHGAT. The UHGAT may be designed with two trays per side to enable tray load/unload from one side of the UHGAT while testing parts from the other side of the UHGAT. Simultaneous load/unload and testing enables the UHGAT to maximize units per hour.

[0072] FIG. 3 also shows an RF tag reader 328 placed below the tray 300. The RF tag reader 328 is attached to an actuator, which actuates the reader 328 vertically to read the tray's RF tag. The first end effector 304 and second end effector 308 are attached to the automation structure as previously described. The first and second end effector's vacuums 322 and 320 are located on the bottom surface of the first and second end effectors 304 and 308, respectively. An optical sensor (not shown in FIG. 3) is placed below each tray to determine the tray's presence. In one embodiment, the described structures accomplish the steps occurring in the tray load/unload area 202 in the following manner. An operator places tray 300 in tray load/unload area 202, and the optical sensor detects the tray's 300 presence and notifies the UHGAT that the tester can select HGAs from tray 300. During tray loading, the RF tag reader 328 reads the RF tag, which may contain information such as which HGAs should be tested and whether the flex circuit 112 of the HGA is oriented to the left or the right. The orientation information is used to configure the UHGAT for the appropriate HGA testing processes, including HGA placement on the précising and test nests, selection of the appropriate preamplifier to switch on, and the disc spin direction. Additionally, if an HGA fails the testing process, the RF tag may be marked to indicate the HGA's failing status.

[0073] To initiate the automated testing process, an operator uses the user interface on the display attached to the computer in the control area 208. Once the automated testing process begins, it will continue as long as trays are loaded with HGAs to be tested. After initiation, an automated pneumatic screwdriver 311 opens the tray lid. First end effector 304 moves above the HGA 100a to be tested, creates a vacuum with the vacuum 322, and sucks the HGA's base plate 108 against first end effector's bottom surface 323. Next, first end effector 304 transports the HGA 100a to précising area 204.

[0074] The steps that occur in the précising area 204 include moving the HGA 100a above and then lowering it onto the précising nest 210. As the HGA 100a is placed on the précising nest 210, pins align the HGA for eventual testing with the disc. Then the HGA 100a is simultaneously de-shunted and headset at the précising nest 210, and it is subsequently moved to test area 206.

[0075] FIG. 4 shows the précising nest 210 in the précising area 204. HGA 100a is positioned on the précising nest 210 so that the boss hole 110 surrounds a boss hole pin 402 and the HGA's tooling hole 106 surrounds a front alignment pin 404. Both the boss hole pin 402 and the front alignment pin 404 are tapered pins located on a top surface 401 of the précising nest 210. FIG. 4 also shows the HGA's head 102 positioned between a pair of tapered pre-alignment guides 406 and extending beyond a lateral surface 403 of the précising nest 210. For future reference, the HGA's flex circuit 112 is viewed as oriented to the right when the flex circuit is angled to the right relative to an observer facing the précising nest's lateral surface 403 from which the HGA's head 102 protrudes.

[0076] Referring to FIG. 4, the précising nest 210 further includes two sets of air actuated flex-on-suspension (FOS) aligners 408a, 408b and 408c, 408d. Each set of FOS aligners 408a, 408b and 408c, 408d are sliding mechanisms that, starting from an open position, move towards each

other to push the flex circuit **112** to a predetermined de-shunting position, and constrain it from side-to-side motion. Using the nomenclature previously mentioned defining when a flex circuit is right oriented, the right set of FOS aligners **408a**, **408b** moves flex circuits that are oriented to the right (see arrow A in FIG. 4), and the left set of FOS aligners **408c**, **408d** moves flex circuits that are oriented to the left. A flex support is located on each set of the FOS aligners and protrudes from the side designed to contact the flex circuit **112**. The flex supports **410a**, **410b** on the FOS aligners **408a**, **408b** are positioned under the flex circuit **112** after the FOS aligners **408a**, **408b** move the flex circuit **112** to its de-shunting position. A similar set of flex supports **410c**, **410d** are present on the FOS aligners **408c**, **408d** and support the flex circuits that are oriented to the left.

[0077] Referring to FIG. 4a, which shows an enlarged view of the FOS aligners **408a**, **408b** and **408c**, **408d** in the closed position, the FOS aligners move in concert when actuated. The FOS aligners' basic components include two rectangular bars, a first bar **409a** and a second bar **409b**. The set of FOS aligners **408a**, **408c** are part of the first bar **409a**, and the FOS aligners **408b**, **408d** are part of the bar **409b**.

[0078] When the bar **409a** is actuated in the direction of arrow A, the bar **409a** moves along a path constrained by alignment pins **420**, **421** and respective cooperating angled slots **422**, **424**. This movement causes the FOS aligners **408a** and **408c** to move first laterally in the direction of arrow A, and then to move in a direction B generally downward from their original position and normal to the arrow A. Simultaneous to the movement of the bar **409a**, the bar **409b** moves in a direction opposite the arrow A. The movement of the bar **409b** is also constrained by the pins **420**, **421** and respective cooperating angled slot **423** and a slot (not shown in FIG. 4). This constraint causes the FOS aligners **408b** and **408d** to move first laterally in a direction opposite the arrow A, and then to move in the direction B. The lateral and downward movements of the bars **409a** and **409b** may release the flex circuit **112** from its clamped position between the flex supports **410a**, **410b** if the circuit **112** is right aligned or the supports **410c**, **410d** if the circuit is left aligned.

[0079] Conversely, when bar **409a** moves in a direction opposite the arrow A, the FOS aligners **408a** and **408c** move first laterally opposite the arrow A and then upward opposite the direction B. Simultaneous to the movement of the bar **409a**, the bar **409b** moves in the direction of the arrow A, and the FOS aligners **408b** and **408d** move first laterally in the direction of the arrow A and then upward opposite the direction B. These movements by the bars **409a** and **409b** may constrain and clamp the flex circuit **112** between the flex supports **410a**, **410b** if the circuit **112** is right aligned or the supports **410c**, **410d** if the circuit is left aligned.

[0080] The shunt tab **114** resides on the flex circuit **112** (see FIG. 1A) and electrically shorts together an HGA head's readers and writers to protect the HGA from electrostatic discharge damage. The shunt tab **114** must be broken or removed before an HGA can be electrically tested. Breaking the shunt tab **114** must be done without damaging, bending or twisting the flex circuit **112**. Before breaking the shunt tab, it is first moved to the de-shunting position, then clamped between the flex supports **410a**, **410b** or **410c**, **410d** (See FIGS. 4 and 4A). Referring to FIG. 4A, under the positioned shunt tab is a shunt tab hole **416**, which is a hole

in the précising nest **210** that provides clearance for the shunt tab when it is folded down.

[0081] Referring to FIG. 4B, a de-shunt top plate **415** may be attached to a pneumatic linear actuator (not shown) that moves the plate horizontally in the direction of arrow C and another pneumatic linear actuator (not shown) that moves the plate vertically downward in the direction of arrow D. After actuation, the de-shunt top plate **415** is positioned partially above the flex circuit **112**, and a de-shunt pin **400a** (See FIG. 4C) on the underside **416** of the plate **415** is positioned above the shunt tab. Referring to FIG. 4C, a right de-shunt pin **400a** is a small pin with a chisel tip and is precisely located on the underside **416** of the de-shunt top plate **415** to punch the shunt tab **114** on the right-oriented flex circuit after the FOS aligners **408a**, **408b** and flex supports **410a**, **410b** have constrained the flex circuit **112** and the shunt tab in the de-shunting position. In one embodiment, the pin **400a** may be roughly one-third of the length of the shunt tab **114** and approximately the same width. In other embodiments, the pin **400a** may be different lengths and widths. Similarly, a left de-shunt pin **400b** punches the shunt tab on a left-oriented flex circuit retained by FOS aligners **408c**, **408d** and flex supports **410c** and **410d**. A compression foot **412a** is attached to the de-shunt top plate **415** and completely surrounds the de-shunt pin **400a**, while a similar compression foot **412b** surrounds the de-shunt pin **400b**.

[0082] FIG. 4D is a diagram of one embodiment of the de-shunt pin **400a** used for de-shunting. The figure also shows the shunt tab **114** on the HGA, which may have a perforated side **426** farthest from the base plate **108**, and the pin **400a** may be smaller than the shunt tab. This size difference permits variance in the positioning of the pin over the tab for the de-shunting process. For example, the pin may be positioned so that its outer diameter abuts a side **428** of the shunt tab closest to the base plate or abuts the perforated side **426**. When the diameter of the pin abuts the perforated side **426**, the tip of the pin does not contact the shunt tab at the perforated side because the tip is chiseled so that it slopes away from the side **426**. Instead, the tip contacts the shunt tab at a point that may be more toward the middle of the shunt tab. Using a chiseled tip that contacts the shunt tab at a point away from the perforated side prevents distortion that could occur if a flat pin directly contacted the perforated side **426**. Although a flat pin may be used, the shearing force created by the pin on the perforation may deform the HGA.

[0083] FIG. 4B also shows a flex circuit tail flattener **414** that is attached to first end effector **304** and is positioned above the flex circuit **112**. An electromagnet **418** is attached to a pneumatic linear actuator (not shown) and may be actuated to a position where the electromagnet **418** surrounds the HGA head **102** during the headset operation described below.

[0084] Referring again to FIG. 4B, in one embodiment the described structures accomplish the steps occurring in précising area **204** in the following manner. Once the first end effector **304** moves the HGA **100a** from the tray load/unload area **202** along the X direction to a position above the précising nest **210**, it lowers the HGA **100a** along the Z direction onto a set of alignment pins. As shown in FIG. 4, the alignment pins include the front alignment pin **404**, the

boss hole pin **402**, and the front pre-alignment guides **406** that set the skew, x, and y positions of the HGA **100a**.

[0085] As the HGA **100a** is lowered along the Z direction by the first end effector **304**, the HGA's boss hole **110** slips over the boss hole pin **402** and the tooling hole **106** slips over the front alignment pin **404**. As the HGA **100a** travels downward along the Z direction, the taper on the alignment pins **402**, **404** pulls the boss hole **110** and the tooling hole **106** into their proper locations. The tapered pins allow for some misalignment of the HGA **100a** to the précising nest **210**. The HGA alignment is completed while the HGA **100a** is lowered onto the pins causing a relative motion between the HGA's base plate **108** and the first end effector **304**. This relative motion does not harm the HGA **100a** or the first end effector **304** because the only force on the HGA **100a** while being lowered onto the pins is due to the vacuum created by the first end effector **304**. The vacuum force is sufficient to hold the HGA **100a** securely in the Z direction while still allowing horizontal translation along the X direction as the HGA **100a** is pushed into position by the tapered pins. The précising nest's use of the boss hole **110** and tooling hole **106** for alignment makes the HGA testing process more accurate because these features also are used as an alignment datum when the head **102** is attached to the HGA in production. Once the alignment is complete, the HGA **100a** is held firmly against the précising nest **210** by the first end effector **304** to prevent any movement during the subsequent de-shunt and headset operations.

[0086] The précising nest's **210** two pre-alignment guides **406** are also tapered and provide a rough alignment of the HGA **100a** prior to it reaching the front alignment pin during the downward movement along the Z direction of first end effector **304**. The guides are used in the event that the initial position of the HGA **100a** is far enough out of alignment that the tooling hole **106** would not slip over the taper of the front alignment pin **404**.

[0087] The alignment of the HGA is critical for the dynamic electrical test because the position of the HGA on the test nest **212** will affect the test results. Aligning each HGA at the précising nest **210** eliminates misalignment caused by variation in HGA position in the tray and tray-to-tray differences. The précising nest ensures that every HGA is aligned relative to the travel axis of the coarse positioning system in exactly the same way, regardless of the HGA's alignment coming out of the tray.

[0088] After the HGA **100a** is aligned in the précising nest **210**, the de-shunting process begins. Referring to FIG. 4B, the tail flattener **414** is lowered down over the flex circuit **112** by first end effector **304** just in front of the shunt tab **114**. Because the tail flattener **414** is spring loaded, it remains down during first end effector's **304** movement and is in a downward position when placed over the flex circuit **112**. When the tail flattener **414** is in place there is an approximate 0.005 inch gap between it and the précising nest **210**. The tail flattener **414** gently constrains the flex circuit **112** in the vertical direction but the gap allows the flex circuit **112** to float side-to-side without twisting. Without the tail flattener the de-shunt operation could cause significant yield loss because the flex circuit tends to twist and become positioned incorrectly.

[0089] Because the flex circuit **112** can have an inherent twist to it and because the flex circuit **112** is allowed to move

freely, the shunt tab **114** is not always in an accurate, predetermined position on the précising nest **210** for the de-shunting operation. To mitigate this problem, once the tail flattener **414** is in place, the FOS aligners **408a**, **408b** move the flex circuit **112** into position to ensure that the shunt tab **114** is in the correct location for the de-shunting operation.

[0090] In this example, a right-oriented flex circuit is shown in FIG. 4. After moving the flex circuit **112**, the FOS aligners **408a**, **408b** constrain it from further side-to-side motion. Before the FOS aligners **408a**, **408b** begin their movement, they are initially below a top surface **211** of the précising nest **210** where the flex circuit **112** rests. This allows the flex circuit to land or slide freely during alignment without any obstruction on top of the précising nest **210**. As the FOS aligners **408a**, **408b** move toward the flex circuit **112**, the alignment pins **420**, **421** and cooperating angled slots **422**, **424** cause the aligners **408a**, **408b** to rise vertically above the top surface **211** of the précising nest **210** so they can push the flex circuit **112** to its final de-shunting position.

[0091] Referring to FIG. 4B, the de-shunt top plate **415** is then actuated horizontally above shunt tab **114** by a pneumatic linear actuator. Next, the de-shunt top plate **415** is rapidly lowered down by another pneumatic linear actuator causing the de-shunt pin **400a** on the de-shunt top plate **415** to contact the shunt tab **114**. As the de-shunt top plate **415** continues to travel downward, the pressure the de-shunt pin **400a** exerts on the shunt tab **114** causes the shunt tab **114** to break along the perforated side **426** and fold down away from the flex circuit **112**. The shunt tab hole **416** shown in FIG. 4A provides clearance for the shunt tab **114** when it is folded down.

[0092] To ensure that the flex circuit **112** does not bend or get pulled down into the shunt tab hole **416** as the shunt tab **112** is punched, the flex supports **410a**, **410b** are located on the FOS aligners **408a**, **408b** as shown in FIG. 4A. As the FOS aligners **408a**, **408b** slide towards the flex circuit **112**, the flex supports **410a**, **410b** are positioned directly under the sides of the flex circuit **112** around the shunt tab **114**. The flex circuit **112** is also supported in front and back of the shunt tab **114** by the précising nest **210**.

[0093] To prevent buckling of the thin walls of the flex circuit **112** around the shunt tab **114** during the de-shunt operation, the compression foot **412a** is attached to the de-shunt top plate **415** as shown in FIG. 4C. The compression foot **412a** is spring-loaded and can travel in the vertical direction. As the de-shunt top plate **415** travels downward towards the shunt tab **114**, the compression foot **412a** compresses and holds the flex circuit **112** before the de-shunt pin **400a** comes in contact with the shunt tab **114**.

[0094] After de-shunting, the de-shunt top plate **415** and the FOS aligners **408a**, **408b** move back to their unactuated positions. When the FOS aligners **408a** return to their non-actuated position the tail flattener **414** remains in its location but does not hold or impede the flex circuit **112** from moving horizontally to its natural position. If the tail flattener were to keep the flex circuit from moving to its natural position, the précising operation would likely fail due to the force that the constrained flex circuit would apply on the HGA **100a** when the HGA was removed from the alignment pins.

[0095] At the same time the de-shunt operation occurs, the headset operation occurs. Referring to FIG. 4B, a powerful electromagnet 418 attached to a pneumatic linear actuator is actuated to a position where it surrounds a portion of the HGA 100a that extends from the précising nest 210 and creates a magnetic field around the HGA head 102. An operator can control aspects of the headset operation through the UHGAT software such as gauss level, duration, and pre/post-field collapse delays. After the headset is complete, the electromagnet 418 is turned off and moved back to its un-actuated position. The materials of the précising nest 210, the first end effector 304, and the second end effector 308 can be selected to ensure that the electromagnet 418 does not leave a residual magnetic field on the internal components of the UHGAT.

[0096] Although the de-shunt and headset operations occur in parallel, de-shunting may take less time than the headset, which allows the de-shunt operation to add zero additional test time. Performing the operations in this manner can be a benefit compared to manual testing using the intermediate mounting fixture because the latter method typically requires a separate process step for de-shunting. Also, using this method may ensure each HGA achieves a uniform gauss level and exposure time because the headset operation is entirely automated and completed inside the BGHAT.

[0097] Referring again to FIGS. 2-3, after completion of the alignment, de-shunt, and headset operations, the first end effector 304 is raised to move the aligned HGA 100a to the test area 206. As the HGA 100a is moved, the vacuum force created by the vacuum 322 of the first end effector 304 maintains the HGA's alignment that was set on the précising nest 210. In other embodiments, a mechanical device may be substituted for the vacuum 322 used to maintain alignment of the HGA during placement and removal from the précising nest 210. The device may hold the HGA 100a above the précising nest 210 and release it on the alignment pins located on the nest 210. The device may use a variety of methods to mechanically hold the HGA including clamping, hooking, and carrying. After alignment on the nest 210, the device may recapture the aligned HGA. The device may then transport the HGA to the test nest 212 while maintaining the HGA's alignment.

[0098] When the précising nest 210 is first attached to the UHGAT, it is accurately aligned with the test nest 212 using gauges to ensure both nests are in known locations so that the alignment performed at the précising nest 210 will be accurate when the HGA 100a is moved to the test nest 212.

[0099] The steps that may occur in the test area 206 include placing the HGA 100a on the test nest 212 while maintaining the alignment set at the précising nest 210 and preparing the HGA 100a for testing with a disc 810. The HGA's flex circuit 112 is flattened and a connection is made with a preamplifier, and a second HGA 100b is picked up from the tray 300 and moved to the précising area 210. In parallel with the movement of the second HGA 100b, the first HGA's head 102 is loaded to the disc 810. A dynamic electrical test is performed. The second HGA 100b is moved from the précising area 210 to the test area 212, and the first HGA 100a is removed from the test nest 212. Next, the second HGA 100b is placed on the test nest 212, and the first

HGA 100a is returned to the tray 300 in the tray load/unload area 202. This process continues until all the untested HGAs are tested.

[0100] FIG. 5 is one embodiment of a portion of the test area 206. The portion includes the test nest 212, the collet assembly 500, a preamplifier assembly 502, a pivot bracket 503, and a clamp wing 606. The collet assembly 500 and the preamplifier assembly 502 may be attached to the pivot bracket 503. In one embodiment, the preamplifier assembly 502 may have gold contacts 602 that enable interconnection between the preamplifier and the HGA's flex circuit pads 118. In another embodiment, a solderless connector may permit the interconnection. The clamp wing 606 has clamp wing pins 614 that can engage slots 505 in the pivot bracket 503. The clamp wing 606 is part of a clamp wing assembly 600 that is discussed later in greater detail.

[0101] FIG. 5A shows a cross-section of the test nest 212, which includes the collet assembly 500 used to secure the HGA 100a for dynamic electrical testing with the disc 810. It is a small, air driven assembly that applies a downward force in the direction of arrow E on the HGA base plate 108. The test nest 212 is very stiff with a low mass and is attached directly to a micro-actuator (not shown), which maintains alignment of the HGA during clamping and needs no external tooling. The collet assembly 500 contains an integrated air piston 507 (See FIG. 5B), which includes an o-ring 506, a piston top 508 and a retainer 510 that move during actuation (See FIG. 5A). Surrounding the collet assembly 500 is the collet housing 514, and on the top surface of the housing 514 is the mounting area 524, where the HGA 100a is placed. FIG. 5A also shows seals 522 located within the collet assembly 500.

[0102] FIG. 5B shows one embodiment of an exploded view of the collet assembly 500. The assembly may be divided into three functional parts: collet fingers 504, the air piston 507, and a spreader pin 512 on the spreader pin base 516. The collet fingers 504 may extend through the piston top 508 of the air piston 507. In the embodiment shown, the assembly 500 is made up of four individual fingers 504a,b,c,d, however, it could be made with three fingers, or as an integrated single flexure finger assembly, depending on the size and shape of the article to be clamped into position. In this embodiment, an o-ring 506 (shown in FIG. 5A) fits around the base of the fingers and provides the retracting force to keep the fingers 504 from spreading when the air piston 507 is pushed upwards in the direction of arrow F. The retainer 510 holds the fingers 504 in place relative to the piston top 508 and enables their free vertical movement on a spreader pin 512. The spreader pin 512 is stationary and is attached to a hollow spreader pin base 516.

[0103] FIG. 6 shows a clamp wing assembly 600 for making interconnect between the HGA flex circuit 112 and a preamplifier in the test nest 212. The clamp wing assembly 600 is made up of a linkage 604, an arrangement of springs 610, a linkage base 612, a clamp wing 606, and a clamp wing pad 608 on an underside of the clamp wing 606. The linkage 604 has springs 610 on each side of the linkage base 612 that spring load the linkage 604 and hold the clamp wing 606 flat as it is moved over the flex circuit 112. Referring to FIG. 6A, clamp wing pins 614 are located on the front of the clamp wing 606 and engage slots 505 on a pivot bracket 503 (not shown) located near the back edge of the HGA base

plate 108. The pivot bracket 503 may not be attached to the test nest 212, which permits free movement of the nest 212 for micro-positioning during the testing of the HGA. Instead, the pivot bracket 503 may be attached to a pivot plate, which is discussed later in greater detail. In one embodiment, the clamp wing pad 608 is located under the clamp wing 606 and in line with an electrical contact, which may be an arrangement of gold contacts 602 on a preamplifier assembly 502.

[0104] The gold contacts 602 may be cut from a single piece of conductive material, as shown in FIG. 6B. The conductive material may optionally be brass, plated with nickel, and overlaid with gold. To fabricate the contacts shown in FIG. 6B, the piece of conductive materials is cut or molded to form separations between each contact 602. The contacts 602 are maintained in spaced apart relation by thin break-away tabs 603. After the contacts 602 have been soldered to the printed circuit board (PCB) 630, the tabs are broken or cut off, leaving the individual contacts 602 aligned with each other and the PCB 630.

[0105] In another embodiment, the electrical contact comprises a solderless connector that does not need to be soldered to the PCB 630. As shown in FIG. 6C, the connector 616 includes a housing 618 with at least two apertures. A conductor 620, which includes a first end 624 and a second end 628, may be partially contained within the housing 618. The first end 624 may protrude from a first aperture 622 in the housing 618, and the second end 628 may protrude from a second aperture 626. The conductor 620 may be made from an elastic conductive material, such as full-hard beryllium copper. The conductor 620 may be separated from the connector housing 618 by an electrically insulative material. The material selected for the conductor may be an elastic material that does not permanently deform when deflected, but substantially returns to its original form. The conductor 620 may also be gold plated to lower the resistance at the points where the conductor contacts the HGA and the preamplifier assembly's PCB.

[0106] FIG. 6D is a more detailed view of the solderless connector 616 shown in FIG. 6C with optional features. The connector 616 has several conductors 620a-620g with first ends 624a-624g and second ends 628a-628g that may protrude from apertures 622a-622g and 626a-626g, respectively. The flex circuit pads 118 are aligned with the first ends so that each conductor may make contact with an individual pad when the HGA is positioned for interconnection. The second ends are aligned with trace contacts 632a-632g that are on the preamplifier's PCB 630. When the connector is placed on the PCB 630, each second end may contact an individual trace contact. In one embodiment, the first and second ends protrude from apertures to contact the flex circuit pads 118 and the trace contacts, respectively.

[0107] In an alternative embodiment shown in FIG. 6E, more than one conductor may contact the same flex circuit pad and corresponding trace contact. The housing may contain a pair of conductors 620a,b mirrored about a centerline that is aligned with an individual flex circuit pad. The first ends 624a,b of the pair of conductors may protrude from a pair of first apertures 622a,b. The outer circumference of the apertures may be separated by the width of the flex circuit pad less the diameter of each aperture. The spacing permits both conductors to contact the flex circuit pad at the same time when the pad is optimally aligned.

[0108] Use of the two conductors permits the flex circuit pad to contact the first end of a conductor even if the pad is not optimally aligned. This increases the variance permitted in the positioning of the flex circuit pads on the connector. For example, if the flex circuit pad was misaligned in a direction to the upper left of FIG. 6B, the left conductor 620a of the conductor pair 620a,b may make contact with the pad even if the misalignment prevents the right conductor 620b from so doing. A surface of the flex tail 112 that faces the connector 616 and surrounds the flex circuit pads 118 may not be conductive because misalignment of the pads may cause one conductor of the pair to contact the surface. If the surface was conductive, interconnection between the preamplifier assembly and the conductive surface may cause a short to occur.

[0109] The solderless connector 616 does not have to be soldered to the PCB 630 to establish an interconnection between the HGA and the preamplifier assembly 502. Instead, the connector may be placed on the PCB 630 without the use of solder. This makes removal of the connector 616 much simpler if it needs to be replaced. In some embodiments, the connector 616 may be constrained by a device that holds it against the PCB 630. For example, as shown in FIG. 6E, a PCB cover plate 634 may be placed over the solderless connector 616. The cover plate 634 may have a corresponding aperture 636 that permits a portion 637 of the housing 618 to pass through, while preventing another portion 635 from passing.

[0110] The cover plate 634 may be coupled to a structure that supports, surrounds, or is adjacent to the PCB 630 and may constrain or press the other portion 635 of the housing 618 against the PCB 630. In one embodiment, the PCB cover plate 634 may be positioned over the PCB 630 using alignment pins or dowels and corresponding apertures. The pins or dowels may protrude from a surface of the test nest 212, and the cover plate 634 may have apertures that are slipped over the pins or dowels. The cover plate 634 may also be clamped against the PCB 630 and screwed to the surface of the test nest 212 or to a PCB bottom cover plate that is located between the PCB 630 and the test nest 212.

[0111] Referring to FIG. 6E, the housing 618 may have one or more alignment features, such as housing alignment element 638, that enables a small connection form factor and ready replacement. This irregularly shaped element 638 permits precise alignment when the PCB cover plate 634 is used to secure the connector 616 to the PCB 630 and prevents incorrect installation of the connector 616. Correctly aligning the connector 616 on the PCB also permits the connector 616 to be presented in a correct orientation to the flex circuit pads 118 because the pads may be positioned to mate with the connector at a predetermined location. In other embodiments, the housing may include housing alignment apertures 640 that are used to align the connector 616. Alignment pins or dowels may protrude from the PCB 630, the test nest 212, and the PCB cover plate 634, or any combination of these, and the apertures 640 may slip over the pins to correctly locate the housing 618.

[0112] The housing 618 may be different shapes to accommodate different HGA geometries. The housing size may depend on the length of the flex circuit 112 and the number of flex circuit pads 118 on the circuit 112. For instance, if the flex circuit is short, the housing size may be decreased to

permit correct contact. If the flex circuit **112** has many pads **118**, then the housing will have to accommodate the required number of apertures and conductors needed to make contact with each of the pads **118**. The housing may advantageously be configured to accommodate and enclose the internal structure used to support, retain, and guide the conductors within the housing **618**.

[0113] In another embodiment, pogo pins embedded in a cover, such as the PCB cover plate **634**, may be used instead of the solderless connector **616**. The cover plate **634** may be made of a malleable material, such as plastic, to permit embedding the pins. The pins are small metal tubs with a spring inside. On one side of the spring and partially protruding from the tube is a pointed metal tip that digs into the PCB, and on the other side of the spring and partially protruding is a metal portion with prongs that may contact the HGA flex circuit pads. When the cover plate **634** is aligned and placed over the PCB **630**, the metal tip of the pins may contact the PCB trace contacts **632**. The clamp wing **606** may then press the flex circuit pads **118** against the pronged portion of the pins to establish electrical contact.

[0114] FIG. 7 shows the tail pusher **700** and tail flattener **414** both of which are attached to the first end effector **304**. The tail pusher **700** and tail flattener **414** have respective flanges **710** and **430** that contact each other. The lower surface **712** of the tail pusher's flange **710** presses against the upper surface **432** of the tail flattener's flange **430** so that the tail flattener **414** is necessarily raised when the tail pusher **700** is lowered.

[0115] FIG. 8 shows the four bar loader (FBL) **800** that moves the HGA into position adjacent to the disc **810** for dynamic electrical testing of the HGA. The FBL **800** permits loading the head **102** to the disc **810** at a minimal pitch angle only limited by the HGA's geometry and may achieve various radius and skew angles. The FBL **800** includes a pair of plates; one, the fixed plate **804**, is fixed parallel to the disc surface, the other, the pivot plate **806**, is allowed to pivot using a four bar linkage **808**. The pivot plate **806** is actuated with a tilt actuator **802** that connects the four bar linkage **808** and the pivot plate **806** to the fixed plate **804**. The clamp wing assembly **600**, pivot bracket **503**, nest assembly **212**, and fine positioner (not shown in FIG. 8) are attached to the pivot plate **806** and tilt with the FBL **800**. The whole assembly is typically attached to a moving stage **812** to enable movement to and from the disc **810**.

[0116] The FBL **800** enables positional accuracy and structural integrity. The pivot plate **806** is aligned to the fixed plate **804** using precise pins (not shown) allowing for very accurate locating tolerances for the HGA. When the HGA is tested, the pivot plate **806** may be clamped to the fixed plate **804** using a variety of techniques, including but not limited to a mechanical latch or latches or a vacuum pocket drawing the two plates together. This gives excellent rigidity and isolates the pivot mechanisms from contributing to positioning error during test. It is important that the FBL **800** position the HGA base plate **108** parallel to the disc during testing. To do this, the fixed plate **804** and the pivot plate **806** must be machined flat and parallel. In this way, the base plate **108** parallelism is not dependent upon assembly techniques or tolerances.

[0117] Actuation is achieved with the use of pneumatic cylinders, though other types of actuators could be imple-

mented. Pneumatic cylinders can provide fast actuation at a low cost. Tilt velocity and end of travel impact is controlled with air pressure, flow controls, and dampers using conventional techniques. One alternative is to use servomotors, at a higher hardware cost, to provide more control of the tilt velocity.

[0118] In one embodiment, the described structures accomplish the steps occurring in the test area **206** in the following manner. The first end effector **304** moves the HGA **100a** from the precising area **204** to a position above the test nest **212**. It then lowers the HGA **100a** onto the test nest **212** and presses it against the test nest **212**. During this process, the HGA **100a** maintains the alignment set on the precising nest **210**.

[0119] Referring to FIGS. 5 and 5A, when the HGA **100a** is lowered, the boss hole **110** slips over the extended fingers **504** of the collet assembly **500**. In their extended state, the fingers **504** are contracted because the o-ring **506** pulls the fingers together and the spreader pin **512** does not exert pressure through the center of the four fingers when they are in the extended position. When the fingers **504** retract, the spreader pin **512** forces the fingers **504** open, and the fingers grab the base plate **108** and pull it down tight against the surface of the collet housing **514**. As the fingers **504** pull down against the base plate **108**, angled protrusions **509** on their respective tips **511** catch the angled surface **110a** (FIG. 1A) on the inside of the boss hole **110** and pull the base plate **108** tight against the mounting area **524**. This process permits the collet assembly **500** to hold the HGA without disturbing the alignment previously set on the precising nest **210**.

[0120] The application of vertical force is controlled by air pressure. Referring to FIG. 5A, the pressurized air fills a lower space **518** which causes the piston top **508**, retainer **510**, and the fingers **504** to move up. Downward movement occurs when the pressurized air fills an upper space **520** and the air in the lower space **518** is permitted to exhaust to the ambient environment. The downward movement causes the fingers **504** to contact the spreader pin **512**, which creates an outward pressure on the fingers **504** causing them to spread. In this way, the fingers are spread as they are lowered towards the surface of the collet housing **514** that serves as the HGA mounting area **524**. To capture and hold the HGA **100a** and maintain its precise alignment, the fingers **504** need to expand precisely the same amount of distance and with the same amount of force. With the small size of the collet assembly **500**, the independent fingers **504** may permit a better dimensional consistency between the fingers than if the fingers were cut from a single piece of material.

[0121] The collet fingers **504** open up and then pull down, providing the required downward force with minimal radial force. There could be a variety of applications for the collet assembly **500**, and its size may be scaled up or down depending on the size object to be secured. The pneumatics can be replaced with springs for actuation with a finger or other mechanical means. In the embodiment discussed herein, the collet assembly **500** is fabricated with four individual fingers with an o-ring to provide the retracting force. However, the individual fingers may be designed based on what size or type of HGA is being clamped. For instance, in a slightly larger scale, the collet could be one piece, where the fingers were flexures and provided their own retracting force.

[0122] After the collet assembly 500 has secured the HGA 100a to the mounting area 524, the HGA's flex circuit 112 is interconnected with the preamplifier's gold contacts 602 (FIG. 6A). The tail pusher 700 on first end effector 304 is actuated to move the HGA flex circuit 112 down on the gold contacts 602. Here, the tail pusher 700 helps align the flex circuit to the preamplifier's gold contacts 602 for interconnection. As shown in FIG. 7, when the tail pusher 700 is actuated downward, the tail flattener 414 moves up allowing clearance for the clamp wing 606. The tail pusher 700 also pushes the flex circuit 112 down so that the clamp wing 606 does not catch on the circuit 112 as it is actuated horizontally.

[0123] Referring again to FIG. 6A, after the clamp wing 606 is actuated above the flex circuit 112, it presses the flex circuit pads 118 (See FIG. 1A) against the gold contacts 602. The clamp wing assembly 600 may be necessary because making an automated electrical connection requires pulling the flex circuit 112 down against the gold block contacts 602 in a consistent manner. When pulling down the flex circuit 112, the circuit 112 first needs to be contacted near the HGA base plate 108, and pressed down with an even pressure over the length of the flex circuit 112. The clamp wing pins 614 on the front of the clamp wing 606 engage slots 505 on the pivot bracket 503, which may be mounted to the pivot plate 806. The clamp wing 606 is then actuated down against the flex circuit in a rolling motion, where the clamp wing pins 614 are engaged in the slots and the clamp 606 rotates in a downward circular motion around the axis created by the pins 614.

[0124] The clamp wing pad 608 is located under the clamp wing 606 and may be composed of compliant materials including electrostatic dissipative elastomers. The pad 608 is in line with the gold contacts 602 and applies a compliant force to the flex circuit pads 118 to press them against the gold contacts 602. The initial engagement of the pins 614 followed by the clamp wing's downward actuation allows the flex circuit 112 and pads 118 to float during the flattening out of the flex circuit 112, which helps remove a great deal of the flex circuit's 112 positional error. In this way, the clamp wing 606 presses and holds the HGA flex circuit 112 onto the preamplifier's gold contacts 602, which establishes a connection.

[0125] Additionally, when interconnecting the flex circuit 112 with the gold contacts 602 any positional errors of the flex circuit 112 must be corrected or compensated. Another way to overcome the positional errors of the flex circuit 112 is to make the area with the gold contacts 602 much larger than the flex circuit pads 118 that interconnect with the contacts 602. By using large flat gold contacts 602 with an area that covers the tolerance range of the flex circuit location, one may ensure that the gold contacts 602 and the flex circuit pads 118 contact each other when they are pressed together. This type of interconnect aids automation because it is very tolerant to positional flex circuit errors.

[0126] The compressive force may be applied to the flex circuit 112 nominally centered over contact pads 118 with a compliant material, such as the clamp wing pad 608, so that the force can overcome errors in the heights of multiple contacts as well the height differences of the flex circuit layers. The clamp wing pad 608 may be comprised of an array of posts. The size of the posts may be half the size of

the of the contact pads 118 to ensure that at least one full post may press the contact pad 118 against the gold block 602. Using posts of this size maximizes the pressure of the contact pad 118 against the gold block 602 at a minimum overall force to compress the compliant pad 608. The reliability of the electric circuit may depend upon the pressure of the contact pad 118 against the gold block 602.

[0127] In other embodiments, a post with a smaller diameter may be used to increase the post density on the contact pad 118. Larger posts may also be used. Alternatively, ribs may be used instead of posts. The ribs may be alternating stripes of high and low regions of the clamp wing pad 608 that can make lines of contact across the width of the contact pads 118 when the compressive force is applied. In another embodiment, the clamp wing pads 608 may have a waffle pattern, where there may be 2 patterns of ribs aligned perpendicular to each other to create a waffle pattern on the clamping wing pad 608. The waffle pattern clamp wing pad 608 may require a greater compressive force to achieve reliable contact between the contact pad 118 and gold block 602, but may have a lower manufacturing cost.

[0128] The clamp wing assembly 600 may make contact between the connector 616 described in FIGS. 6C-6E and the flex circuit pads 118 with a method similar to that described for the gold block 602. Additionally, when the clamp wing pad 608 applies compressive force to press the flex circuit pads 118 against the first end 624 of the connector's conductor 620, the first end 624 may deflect or bend. The deflection may be flush with the surface of the housing 618 that faces the pads 118 to ensure an adequate connection is established. Using an elastic material for the conductor causes it to oppose the force created by the clamp wing pad 608, and the greater the deflection of the conductor, the greater the opposition force. In turn, the opposition force can permit a reliable contact between the flex circuit pads 118 and the deflected conductor 620. The use of an elastic material in constructing the conductors 620 permits the conductor to return to substantially its original position after the flex circuit 212 is removed, so that the connector 616 may be used many times before it has to be replaced.

[0129] Once the HGA 100a is clamped by the clamp wing assembly 600 and its orientation is set, first end effector 304 releases HGA 100a and returns to the tray load/unload area 202 to pick up a second HGA 100b and moves it to the pre-cising area 210, where the second HGA 100b begins the processes described for the first HGA 100a.

[0130] As shown in FIG. 8, the FBL 800 is used to load the HGA 100a to the disc 810 while the second HGA 100b is picked from the tray load/unload area 202. Referring to FIG. 8A, the FBL 800 tilts the pivot plate 806 down relative to its position shown in FIG. 8 in order to move the HGA 100a below the surface of the disc 810. Next, as shown in FIG. 8B, the moving stage 812 on which the FBL 800 is mounted moves the HGA 100a to a load radius under the disc 810, and as shown in FIG. 8C, the pivot plate 806 is tilted up to a test radius. The HGA is then tested with the disc 810, and the results are read and analyzed by the UHGAT software. After the UHGAT performs the testing on the HGA 100a, the FBL's move stage 812, moves the HGA to an unload radius, the FBL 800 tilts pivot plate 806 down, and the FBL 800 returns to its original location.

[0131] Use of the four bar linkage 808 on the FBL 800 enables the projection of a hinge pivot point into space.

Proper choice of linkage **808** enables putting a virtual pivot point near the bend of the load beam **104** (FIG. 1 B). Additionally, as shown in FIG. 8C, the preamplifier assembly **502**, including the contacts **602**, are not located on the same plane as the base plate **108**. Instead, they are on a plane sloped away from and below the plane of the base plate relative to the disc **810**. In this position, the preamplifier assembly does not limit the loading of the HGA because it is not in a location where it could contact the disc. The projection of the pivot point and the location of the preamplifier's parts enable loading the HGA **100a** with the head **102** at a very shallow angle that is only limited by the geometry of the HGA.

[0132] The FBL design may also be flexible. For instance, the linkages may be sized to locate the pivot where a wide range of HGA geometries can be accommodated. However, if a different style of HGA or HGA attachment mechanism is used, a simple change of the linkages can move the virtual pivot to a new optimum location.

[0133] After the HGA **100a** has been tested, first end effector **304** moves the second HGA **100b** from the pre-cising area **204** to the test area **206**, and the first HGA **100a** is removed from the test nest **212** by the second end effector **308**. The second end effector creates a vacuum force with the vacuum **320** (FIG. 3) to retain the HGA's base plate **108** on the bottom surface of second end effector **308** and the collet assembly releases the HGA **100a** by extending the fingers **504** upward allowing the base plate **108** to slip over them. Then the second end effector **308** removes the first HGA **100a**, and the first end effector **304** places the second HGA **100b** on the test nest **212**. Loading and unloading the HGAs to and from the test nest **212** takes less than 2.5 seconds, which is far faster than any manual operator.

[0134] The first end effector **304** and the second end effector **308** are then actuated back to the tray load/unload area **202** and the first HGA **100a** is returned to the tray **300**. If the HGA **100a** fails, the electrical test the tray's RF tag can be marked to indicate the failure. The described process continues until all the untested HGAs are tested.

[0135] When testing the HGA with disc **810**, an optional disc flutter control device **900** shown in FIG. 9 may be used to mitigate errors caused by disc flutter. Internal or external sources, such as internal spindle vibrations or external windage and acoustic vibrations, may excite the disc **810** and cause disc flutter. The flutter control device **900** reduces disc flutter by including a removable shroud **902** in close proximity to the disc **810**. This feature provides a cushioning force and in effect operates as a stiff virtual spring that connects the disc **810** to the cover, which reduces the results of disc flutter including asynchronous radial runout.

[0136] FIG. 9 shows the optional disc flutter control device **900** in use with the disc **810**. The removable shroud **902** is positioned so that the shroud's underside **903** is shown. The shroud **902** may be removed to permit convenient access to the disc adapter **916** when seating the disc **810** or when the surrounding structures require service. During operation, the circular portion **904** of the shroud **902** may be above the disc **810**, which may require testing HGA heads on the bottom surface of the disc **810**. In operating position, the shroud **902** is attached to a base **906**, which is attached to a riser **908**. A spindle motor **910** is also mounted to the riser **908** via a spindle mounting flange **912**. The

triangular spindle mounting flange **912** may be surrounded on two sides by the base **906**. The side that is not surrounded faces the four bar loader **800** and provides the loader **800** access to the disc **810**. A disc adapter **916** is attached to the spindle motor **910**, and the adapter **916** holds the disc **810** for testing. The disc **810** may be placed on the adapter **916** by passing the disc's center aperture through the protruding portion of the adapter **916**.

[0137] FIG. 10 shows a section view of one half of the disc flutter control device **900**. The figure illustrates the gaps between the structures, which define an air cushion between the disc **810** and the shroud **902**. The side of the disc **810** that defines its outer circumference is the radial disc side **918**, and the disc side that is perpendicular to the radial disc side and that faces the shroud **902** is the axial disc side **920**. Correspondingly, the gap between the radial disc side **918** and the shroud **902** is the radial disc gap **922**, and the gap between the axial disc side **920** and the shroud **902** is the axial disc gap **924**. The gap between the side of the disc adapter **916** and the shroud **902** is the disc adapter gap **926**.

[0138] Referring to FIG. 11A, when the spindle motor **910** rotates the disc **810**, the motor **910** may create vibrations **1** that excite the disc **810** and cause disc flutter. In addition, air flow **2** around the disc **810**, as well as turbulence **3** near the radial side **918** of the disc **810**, create pressure variations and causes random structural excitation forces to be exerted on the disc **810**, which in turn cause excitation of disc modes.

[0139] However, as shown in FIG. 11B, the shroud **902** causes the air flow around the disc **810** to travel in the direction of arrow G. The small size of the gaps **922**, **924**, **926** reduce and smooth air flow near the disc **810**. This reduced flow reduces turbulence near the disc **810**, and the resulting reduced pressure against the disc reduces excitation of disc modes. The narrow gaps **922**, **924**, **926** create an air bearing that acts as a stiff virtual spring between the shroud **902** and the disc **810**. The virtual spring constrains the movement of the disc **810** relative to the shroud **902**, which reduces the amount of disc flutter in the disc **810** caused by spindle motor vibrations, which reduces spindle runout and improves the track per inch performance of the test system.

[0140] To create a stiffer virtual spring that provides a greater reduction in disc flutter, the axial gap **924** should be as small as possible without permitting contact between the shroud **902** and the disc **810**. Moreover, the adapter gap **926** and the radial disc gap **922** also affect the disc flutter. A smaller adapter gap **926** substantially reduces the disc flutter present at the outer edge of the disc **810**, while a smaller radial disc gap **922** substantially reduces the disc flutter present at the inner edge of the disc **810**. By adjusting the thickness of each of the gaps **926** and **922**, one can achieve uniform suppression of disc flutter at both the inner and outer disc edges.

[0141] FIG. 11B also illustrates a vacuum created by the spinning disc **810**. In one embodiment, the vacuum draws external ambient air over the axial disc side **920** as shown by the arrows. In other embodiments, pressurized gas sources could provide the gas present in the gaps. The gas also could dynamically flow over the axial disc side **920** or remain in a static state in the gaps.

[0142] A number of embodiments of the invention have been described. Nevertheless, it will be understood that

various modifications may be made without departing from the spirit and scope of the invention. Accordingly, other embodiments are within the scope of the following claims.

What is claimed is:

1. A method for securing a head gimbal assembly comprising:

placing at least a portion of an unmounted head gimbal assembly on a surface of a collet assembly, wherein the unmounted head gimbal assembly includes a plate with an aperture, and wherein the collet assembly includes collet fingers;

extending the collet fingers from the surface and through the aperture in the plate; and

retracting the collet fingers, wherein the retracting comprises expanding the collet fingers and exerting a clamping force on the plate.

2. The method of claim 1, wherein the collet fingers are controlled by a computer.

3. The method of claim 2, wherein the computer controls the placing.

4. The method of claim 1, further comprising:

loading the head gimbal assembly on a rotating disc; and testing the head gimbal assembly by reading from and/or writing to the rotating disc.

5. The method of claim 1, wherein the collet fingers are individually attached to an actuation mechanism used to extend and retract the collet fingers.

6. The method of claim 5, wherein the actuation mechanism is a pneumatic actuator.

7. The method of claim 1, wherein the clamping force is distributed equally among the collet fingers.

8. A fastening device comprising:

a mounting area forming an aperture;

at least two retractable fingers within the aperture,

wherein distal ends of the retractable fingers are contracted towards each other when the at least retractable fingers are in an extended position relative to the mounting area,

wherein the distal ends of the retractable fingers spread apart when the retractable fingers are in a retracted position relative to the mounting area.

9. The fastening device of claim 8, further comprising an actuation mechanism that moves the retractable fingers between the extended position and the retracted position.

10. The fastening device of claim 9, wherein the actuation mechanism comprises a piston connected to the retractable fingers.

11. The fastening device of claim 8, further comprising a spring that biases the retractable fingers to the retracted position.

12. The fastening device of claim 8, further comprising a spreader pin that forces the distal ends of the retractable

fingers apart when the retractable fingers are in a retracted position relative to the mounting area.

13. The fastening device of claim 12, further comprising an o-ring around the retractable fingers that biases the distal ends of the retractable fingers towards each other.

14. The fastening device of claim 8, wherein the retractable fingers comprise independent retractable fingers attached to a common actuation mechanism.

15. The fastening device of claim 8, wherein each of the retractable fingers expand the same amount of distance and with the same amount of force as each of the other retractable fingers when the retractable fingers are moved from the extended position to the retracted position.

16. The fastening device of claim 8, wherein the retractable fingers pinch against the mounting area when the retractable fingers are in the retracted position.

17. The fastening device of claim 16, wherein the fastening device allows an aperture of a component to fit over the retractable fingers in the extended position and pinches an edge of the aperture of the component between the mounting area and the distal ends of the retractable fingers when the retractable fingers are in the retracted position.

18. The fastening device of claim 17, wherein the component is a head gimbal assembly and the aperture of the component is a boss hole of the head gimbal assembly, wherein the retractable fingers engage an angled adjacent the boss hole.

19. An assembly comprising:

a fastening device including,

a mounting area forming an aperture, and

at least two retractable fingers within the aperture, wherein distal ends of the retractable fingers are contracted towards each other when the at least retractable fingers are in an extended position relative to the mounting area, wherein the distal ends of the retractable fingers spread apart when the retractable fingers are in a retracted position relative to the mounting area; and

a component pinched between the distal ends of the retractable fingers and the mounting area when the retractable fingers are in the retracted position.

20. The assembly of claim 19, wherein the fastening device includes an actuation mechanism that moves the retractable fingers between the extended position and the retracted position.

21. The assembly of claim 19, wherein the fastening device includes a spreader pin that forces the distal ends of the retractable fingers apart when the retractable fingers are in a retracted position relative to the mounting area.

22. The assembly of claim 21, wherein the fastening device includes an o-ring around the retractable fingers that biases the distal ends of the retractable fingers towards each other.

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