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(21) International Application Number: PCT/EP93/00688 (22) International Filing Date: 22 March 1993 (22.03.93) (30) Priority data: 862,033 2 April 1992 (02.04.92) US (71) Applicant (for DE only): FORD WERKE AG [DE/DE]; Werk Köln-Niehl, Henry Ford Strasse, Postfach 60 04 02, D-5000 Köln 60 (DE). (71) Applicant (for FR MC only): FORD FRANCE S.A. [FR/ FR]; B.P. 307, F-92506 Rueil-Malmaison Cédex (FR). (71) Applicant (for GB IE only): FORD MOTOR COMPANY LIMITED [GB/GB]; Eagle Way, Brentwood, Essex CM13 3BW (GB).		(71) Applicant (for all designated States except DE FR GB IE MC): FORD MOTOR COMPANY [US/US]; County of Wayne, Dearborn, MI 48126 (US). (72) Inventor: SMITH, Galvin, Gene ; 8467 Appleton, Dearborn Heights, MI 48127 (US). (74) Agent: MESSULAM, Alec, Moses; A. Messulam & Co., 24 Broadway, Leigh on Sea, Essex SS9 1BN (GB). (81) Designated States: JP, KR, European patent (AT, BE, CH, DE, DK, ES, FR, GB, GR, IE, IT, LU, MC, NL, PT, SE). Published <i>With international search report.</i>
(54) Title: AN AIR CONDITIONING/HEATING MODULE (57) Abstract <p>A space-efficient heating and air conditioning module for a motor vehicle includes radially inwardly directed air handling ducts (12, 14) and a coaxial rotary blower, evaporator, and heater core. A defrost supply duct (38) communicates heated or conditioned air above the evaporator (18) and heater core (24) in a direction 180° from the direction of air flow through the rotary blower (16), evaporator (18), and heater core (24). The air flow is automatically thermally stratified, to provide the warmest air to the floor vent. The module is compact, and is centrally mountable for use in left- or right-hand drive vehicles.</p>		

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AN AIR CONDITIONING/HEATING MODULE

The present invention is directed to a space-efficient air conditioning/heating module. More particularly, the present invention contemplates an air conditioning/heating module which is compact, and may be placed entirely within the central portion of a left- or right-hand drive vehicle dashboard.

Vehicle air conditioning/heating systems are well-known in the prior art, and typically comprise an assemblage of heat exchangers, blowers, temperature sensors, air flow diversion mechanisms, refrigerant flow conduits, controllers, etc. These system elements are generally installed at various locations throughout the vehicle engine compartment and dashboard, and are interconnected via air handling ducts.

The prior art systems for ducting heated and conditioned air to the vehicle interior require large amounts of space to accommodate the above-mentioned system elements. The packaging requirements for climate control systems for vehicles are becoming more important due to the downsizing and weight-reducing programs for modern vehicles. Furthermore, efforts are being directed toward conserving space and weight in all areas of the climate control system. Moreover, manufacturing costs are always an important issue, and the utilisation of modular, space-efficient system components will result in an overall reduction in the cost of manufacturing a motor vehicle.

It would be desirable to produce a space-efficient air conditioning/heating module which could be installed entirely within the dashboard of a vehicle and which could be centre-mounted and therefore useful for both left- or right-hand drive vehicles.

The air conditioning/heating module embodying the present invention is centre-mountable within the dashboard of a vehicle. This air conditioning/heating module can support future styling trend modifications, and can satisfy more demanding customer comfort requirements. The air

conditioning/heating module balances compact packaging requirements and air conditioning and heating performance expectations with noise and electrical power consumption limitations.

5 The air conditioning/heating module embodying the present invention is characterised by a low pressure drop, thereby providing for a high air flow and low noise generation. It incorporates a stacked evaporator and heater core for a straight-through air flow path. This arrangement
10 minimises air flow velocity losses, blower power consumption, and turbulent air flow noise generation. The module may include integral floor, panel, and defrost air distribution ducts. The module may also be used in conjunction with cross-dashboard air handling conduits, to
15 supply heated or conditioned air to outboard registers. The temperature of the air discharged from the module may be controlled via electronic refrigerant and engine coolant control valves, which regulate the flows of refrigerant and engine coolant to the evaporator and heater core,
20 respectively, to obtain the desired amount of reheat to temper the discharged conditioned air.

Fresh air enters the inventive module through a blower assembly. The blower includes an integral
recirculating/outside air door mechanism, which selectively
25 may be driven between a recirculating air position and an outside air position or any combination thereof, by conventional dampers or a conventional rotary door. In the outside position, air enters the blower directly from the centre cowl area. In the recirculating air position, the
30 dampers or door mechanism are positioned to allow air to be drawn into the blower from the passenger compartment of the vehicle. Alternatively, the dampers or door mechanism may be positioned at an intermediate location, thereby allowing a mixture of recirculating air and outside air to be drawn
35 into the blower. The blower is selected to provide a uniform axial flow of recirculating air or outside air, or any combination thereof, to the evaporator. The axial air flow generated by the blower is evenly distributed across

the face of the evaporator, thereby maximising the efficiency of heat exchange between the evaporator and the flowing air stream. The module utilises a thin evaporator core which, when combined with a large face area, will
5 supply conditioned air at a uniform discharge temperature with minimal pressure drop. In a preferred embodiment of the present invention, there is a gap of approximately one inch between the evaporator and a heater core which is positioned downstream therefrom, to allow for condensate
10 drainage and to prevent rehumidification of the air which might otherwise occur if the evaporator and heater core were close enough together to allow airborne condensate to be carried directly from the evaporator into the heater core.

The heater core face area is substantially the same as
15 the face area of the evaporator, to minimise air flow velocity changes due to expansion or contraction. This reduces air flow turbulence and related noise generation. Vehicle engine coolant enters the heater core at the lower header and exits the heater core at the upper header. This
20 engine coolant flow configuration intentionally stratifies the air leaving the heater core in a desirable fashion. Air exiting from the lower portion of the heater core is warmer than the air exiting from the upper portion of the heater core. Thus, in the panel/floor or defrost/floor mode, the
25 warmest air is automatically channelled to the vehicle occupant's feet, while cooler warm air is directed to the remainder of the passenger compartment for improved comfort.

The discharge air temperature may be controlled by means of electrically actuated flow control devices mounted
30 in the engine compartment. Such flow control devices are well-known, and may comprise, for example, a simple butterfly bypass valve. In such a system, refrigerant or engine coolant continuously flows through the valve; only the amount of refrigerant or engine coolant necessary to
35 obtain the desired discharge air temperature is then diverted to the evaporator or heater core and recirculated back to the supply loop.

The discharge air distribution apparatus may be of any type generally known in the art. One particularly useful air distribution apparatus is disclosed in recently allowed U.S. patent application Serial No. 07/628,965 and comprises
5 a flexible film valving system, wherein an apertured film may be driven between various positions, to provide varying amounts of air flow between floor, panel, and defrost ducts.

The invention will now be described further, by way of example, with reference to the accompanying drawings, in
10 which is a perspective view, partially cut away, illustrating an embodiment of the air conditioning/heating module of the present invention.

The present invention is directed to a space-efficient air conditioning/heating module which is centre-mountable
15 within the dashboard of a vehicle. The configuration of the module is substantially identical whether mounted in a left- or right-hand drive vehicle. The module comprises fresh air ducts for admitting radially inwardly directed flows of recirculating air and/or outside air. The inlet ducts
20 change the radially directed flow of incoming air to an axial direction, into the intake of an adjacent rotary blower. Practically the only modification required for either left- or right-hand drive vehicles is the rotation of the intake air ducts to allow air to be drawn either from
25 the outside of the vehicle or generally from the passenger side of the vehicle compartment in the recirculation mode. The rotary blower discharges the air immediately through a series of adjacent heat exchangers. The first heat exchanger is an air conditioning evaporator; the second
30 spaced-apart heat exchanger is an engine coolant heater core. Although an engine coolant heater core is described as the preferred embodiment, it is to be understood that other forms of heat exchange may be used, e.g., a resistance-type ceramic heating element. The evaporator and
35 heater core have substantially the same face areas. Vehicle engine coolant enters the heater core at the lower header and exits the heater core at the upper header. A discharge air distribution device directs the heated or conditioned

air to any or a combination of floor, panel, and defrost air ducts.

Referring now to Figure 1, there is shown generally at 10 an air conditioning/heating module according to the present invention. The module 10 includes an outside air inlet 12 and a recirculating air inlet 14 for providing air to a rotary blower 16. A preferred blower is set forth in U.S. Patent No. 4,900,228 which is incorporated herein in its entirety by reference thereto. It is also contemplated that other rotary blowers may be utilised in the present invention. Air then passes from the blower 16 evenly through an evaporator 18. A seal 20 is provided between an outer duct housing 22 and the evaporator 18, to cause substantially all of the air to pass through the evaporator 18.

Air continues to flow from the evaporator 18 through a heater core 24. A core seal 26 is provided between the heater core 24 and the outer duct housing 22, to prevent air from bypassing the heater core 24. Air from the heater core 24 then passes immediately into a distribution system. The distribution system disclosed in the Figure includes a chamber 28 and a flexible film 30 containing a plurality of apertures 32. The apertures 32 may be moved up or down by winding or unwinding the film 30 around a spring-loaded spindle 34 by means of reversible drive motor 36 attached to a second spindle (not shown). Alternatively, a motor may be attached to each spindle, individually, selectively to drive the flexible film 30 back and forth between the spindles. The movement of the apertures 32 allows the flow of conditioned/heated air to flow into one or a combination of three distribution zones; an upper defrost supply duct 38, a middle dashboard vent 40, or a lower floor duct 42. The characteristics and operation of the illustrated distribution system are fully set forth in allowed U.S. patent application Serial No. 07/628,965, which is incorporated herein in its entirety by reference thereto.

A particularly critical aspect of the present invention is the space-efficient arrangement of the air handling,

heating, and cooling components of the inventive module 10. This arrangement provides compact packaging of the components, making the module 10 centre-mountable for use in either right- or left-hand drive vehicles, while having the ability to supply thermally stratified air without the need for complex damper and ducting systems. Moreover, the configuration of the module 10 provides styling and considerable safety advantages. The evaporator 18 and heater core 24 are arranged to be substantially parallel, with their major surfaces substantially normal to the direction of travel of the motor vehicle. Upon vehicle crash impact, the evaporator 18 and heater core 24 would collapse together, and not present edges which could seriously injure vehicle occupants thrust against the vehicle dashboard.

The axis of the rotary blower 16 is substantially normal to the faces of both the evaporator 18 and heater core 24, which are spaced-apart only to the degree necessary to substantially prevent rehumidification of the air exiting the evaporator 18. Moreover, the evaporator 18 may be tilted slightly from a position parallel to the heater core 24, to assist in preventing the rehumidification of air exiting the evaporator 18. This allows condensate to more easily drain from the evaporator 18. Air enters the module 10 from one or a plurality of radially inwardly directed inlets, e.g., 12 and 14. The air entering the module 10 through any one of the inlets 12 and 14 may be selected by conventional means, such as for example by the use of individually controlled dampers in each inlet duct (not shown), or by the use of a rotary door such as that disclosed in U.S. Patent No. 4,476,773. Moreover, the air distribution system is designed to route the flow of conditioned/heated air to be used for defrosting purposes above the evaporator 18 and heater core 24, in a direction substantially 180° from the direction of the air flow passing through the rotary blower 16, evaporator 18, and heater core 24. Thus, the defrost supply duct 38 lies above the evaporator 18 and heater core 24, and communicates

defrost air in a direction substantially 180° from the direction of the air flow passing through the rotary blower 16, evaporator 18, and heater core 24. In vehicles which include futuristic styling, this allows the introduction of defrost air at the base of the windshield in a quiet and efficient manner.

The module 10 incorporates a heater core 24 which accepts a flow of engine coolant at its lower header and discharges that flow of engine coolant from its upper header, to provide intentionally thermally stratified air to the distribution system. As disclosed previously, other heat exchange elements, e.g., resistance-type ceramic heating elements, may be used and conventionally controlled to provide intentionally thermally stratified air.

The outer duct housing 22 includes a condensate drain tube 44 for eliminating any condensate that is produced within the module 10. The condensate drain tube 44 is disposed at an angle of about 30° to allow drainage, even if the vehicle is parked on a grade.

The outer duct housing 22 additionally is provided with the floor duct 42, with air distribution openings in the bottom thereof (not shown) for directing heated/conditioned air downwardly toward the feet of the vehicle occupants. The floor duct 42 may also have air distribution openings in the sides thereof, as shown. It will be apparent to those ordinarily skilled in the art that more than three duct systems may be supplied with conditioned or heated air from the distribution system. For example, separate ducts may be run to the rear of the passenger compartment to supply conditioned or heated air directly to the rear passengers.

The rotary blower 16 supplies air evenly over the face of the evaporator 18, thence evenly over the face of the heater core 24. Furthermore, refrigerant and engine coolant are supplied to the evaporator 18 and heater core 24, respectively, by means of fluid flow regulating valves well-known in the art. These characteristics allow for the conditioned/heated air to be thermally stratified by supplying engine coolant to the bottom of the heater core 24

via an inlet tube 46, and removing engine coolant from the top of the heater core via an outlet tube 48. Thus, air flowing through the lower portion of the heater core 24 will be substantially warmer than the air flowing through the upper portion of the heater core 24. This provides the ability to direct air toward the floor of the vehicle which is substantially warmer than the air which is directed to the dashboard vent 40 or the defrost duct 38. Such a feature is extremely desirable for passenger comfort.

As will be evident to those ordinarily skilled in the art, it is possible for the air temperatures discharged from various portions of the air distribution system automatically to be controlled by providing temperature sensors within the air flow paths. Such temperature sensors generate signals which may be directed to a feedback loop to activate a microprocessor which manages the flows of refrigerant and engine coolant to the evaporator 18 and heater core 24, respectively.

From the foregoing description, one ordinarily skilled in the art can easily ascertain the essential characteristics of the present invention, and without departing from the spirit and scope thereof, can make changes or modification in the invention to adapt it to various usages and conditions.

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CLAIMS

1. A heating and air conditioning module, comprising:
- 5 A) a plurality of radially inwardly directed air handling ducts (12,14) for selectively receiving a singular or combined flow of air into the module;
- B) a rotary air blower (16) adapted to receive the radially inward flow of air from said air handling ducts and
10 discharge the air in a direction coaxial with the axis of rotation of said rotary air blower;
- C) an evaporator (18) for receiving the flow therethrough of air from the rotary air blower (16) in a direction substantially normal to a face of said evaporator
15 (18), said evaporator (18) adapted to circulate therethrough a quantity of refrigerant for cooling the air flowing therethrough;
- D) a heating element (24) spaced-apart from the evaporator (18), for receiving the flow therethrough of air
20 from said evaporator in a direction substantially normal to a face of said heating element, said heating element (24) adapted to thermally stratify the air flowing therethrough, the face of said heating element (24) being substantially equal in area to the face of said evaporator (18);
- 25 E) means (28,30,32,34,36) for selectively distributing the flow of air from the heating element (24) to any one or a combination of ducts (38,40,42), said distributing means adapted to provide the thermally stratified warmer air from said heating element to the floor duct; and
- 30 F) a defrost supply duct (38) for receiving a flow therethrough of air from said distributing means, for communicating the flow of air above the evaporator and heating element in a direction substantially 180° from the direction of flow of air through said rotary blower,
35 evaporator, and heating element.
2. A heating and air conditioning module according to Claim 1, wherein the radially inwardly directed air handling

ducts comprise an outside air duct and a recirculating air duct.

3. A heating and air conditioning module according to
5 Claim 1, wherein the space between the evaporator and the heater core is sufficient to substantially prevent rehumidification of the air entering said heater core.

4. A heating and air conditioning module according to
10 Claim 1, wherein the heating element is an engine coolant heater core.

5. A heating and air conditioning module according to
Claim 1, wherein the evaporator is tilted slightly from a
15 position parallel to the heating element.

6. A vehicle, including a heating and air conditioning module according to Claim 1 substantially centrally mounted therein.

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7. A heating and air conditioning module, comprising:

A) a plurality of radially inwardly directed air handling ducts for selectively receiving a singular or combined flow of air into the module;

25 B) a rotary air blower adapted to receive the radially inward flow of air from said air handling ducts and discharge the air in a direction coaxial with the axis of rotation of said rotary air blower;

C) an evaporator for receiving the flow therethrough of
30 air from the rotary air blower in a direction substantially normal to a face of said evaporator, said evaporator adapted to circulate therethrough a quantity of refrigerant for cooling the air flowing therethrough;

D) a heater core spaced-apart from the evaporator, for
35 receiving the flow therethrough of air from said evaporator in a direction substantially normal to a face of said heater core, said heater core adapted to circulate therethrough a quantity of engine coolant for heating the air flowing

therethrough, said heater core additionally adapted to receive the engine coolant at the bottom of said heater core and discharge the engine coolant at the top of said heater core to thermally stratify the air flowing therethrough, the
5 face of said heater core being substantially equal in area to the face of said evaporator;

E) means for selectively distributing the flow of air from the heater core to any one or a combination of floor, panel, and defrost ducts, said distributing means adapted to
10 provide the thermally stratified warmer air from said heater core to the floor duct; and

F) a defrost duct for receiving a flow therethrough of air from said distributing means, for communicating the flow of air above the evaporator and heater core in a direction
15 substantially 180° from the direction of flow of air through said rotary blower, evaporator, and heat exchanger.

8. A vehicle, including a heating and air conditioning module according to Claim 7 substantially centrally mounted
20 therein.

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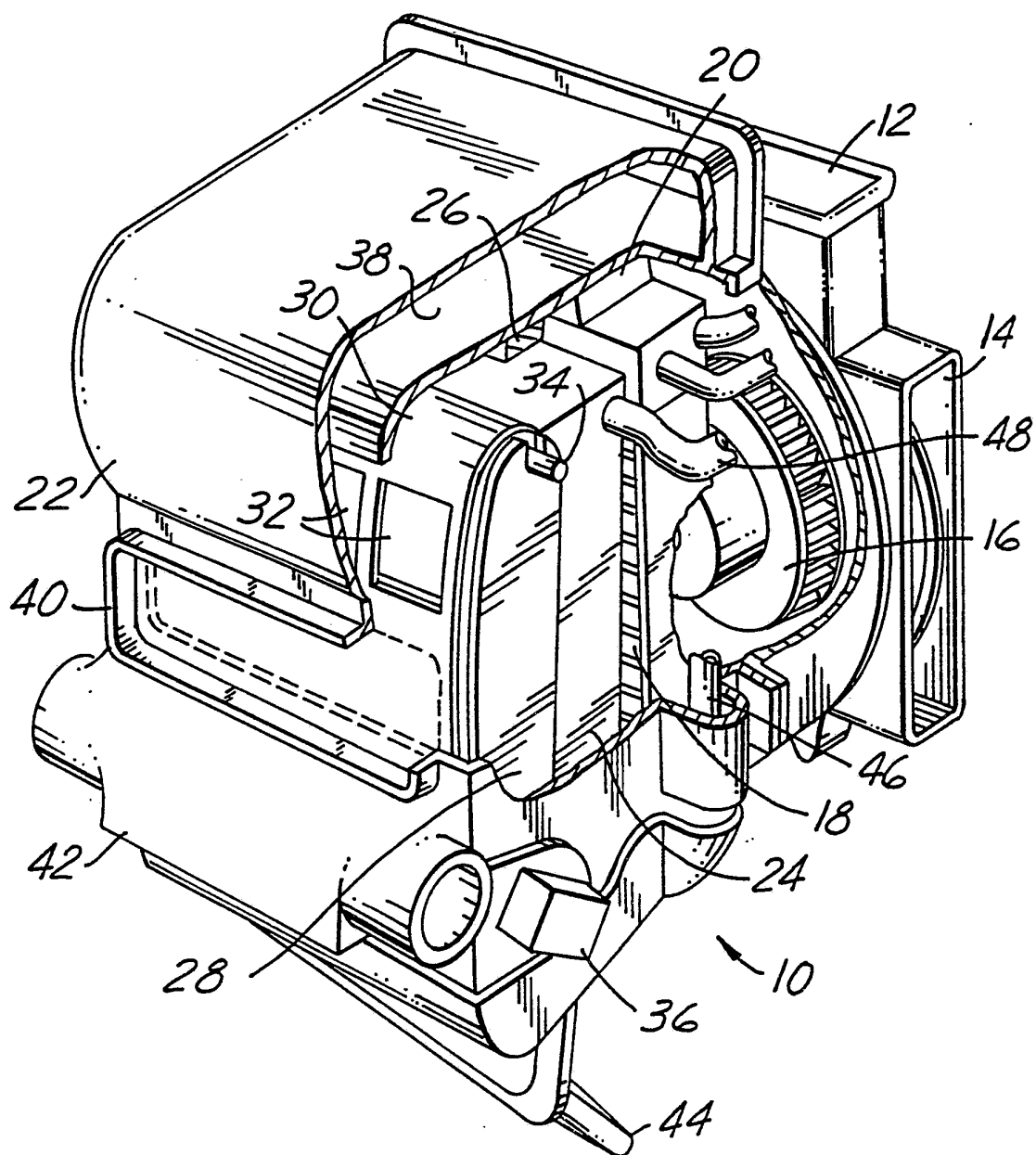


FIG. 1

INTERNATIONAL SEARCH REPORT

International Application No

PCT/EP 93/00688

I. CLASSIFICATION OF SUBJECT MATTER (if several classification symbols apply, indicate all) ⁶		
According to International Patent Classification (IPC) or to both National Classification and IPC Int.Cl. 5 B60H1/00		
II. FIELDS SEARCHED		
Minimum Documentation Searched ⁷		
Classification System	Classification Symbols	
Int.Cl. 5	B60H ; F04D ; F24F	
Documentation Searched other than Minimum Documentation to the Extent that such Documents are Included in the Fields Searched ⁸		
III. DOCUMENTS CONSIDERED TO BE RELEVANT⁹		
Category ¹⁰	Citation of Document, ¹¹ with indication, where appropriate, of the relevant passages ¹²	Relevant to Claim No. ¹³
X,P	EP,A,0 491 466 (FORD MOTOR COMPANY) 24 June 1992 see column 5, line 21 - line 36 see column 6, line 58 - column 7, line 24; figures 1-2 & US,A,5 105 730 cited in the application ---	1-2,7
A	PATENT ABSTRACTS OF JAPAN vol. 9, no. 22 (M-354)(1745) 30 January 1985 & JP,A,59 167 316 (MITSHUBISHI DENKI K.K.) 20 September 1984 see abstract --- -/--	1,6-8
¹⁰ Special categories of cited documents : "A" document defining the general state of the art which is not considered to be of particular relevance "E" earlier document but published on or after the international filing date "L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified) "O" document referring to an oral disclosure, use, exhibition or other means "P" document published prior to the international filing date but later than the priority date claimed "T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention "X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step "Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art. "&" document member of the same patent family		
IV. CERTIFICATION		
Date of the Actual Completion of the International Search 25 MAY 1993		Date of Mailing of this International Search Report 04. 06. 93
International Searching Authority EUROPEAN PATENT OFFICE		Signature of Authorized Officer GONZALEZ-GRANDA C.

III. DOCUMENTS CONSIDERED TO BE RELEVANT (CONTINUED FROM THE SECOND SHEET)		
Category *	Citation of Document, with indication, where appropriate, of the relevant passages	Relevant to Claim No.
A	PATENT ABSTRACTS OF JAPAN vol. 12, no. 432 (M-763)(3279) 15 November 1988 & JP,A,63 166 618 (NIPPON DENSO) 9 July 1988 see abstract ---	1,7
A	EP,A,0 458 705 (NISSAN MOTOR COMPANY) 27 November 1991 see column 3, line 55 - column 4, line 28; figure 2 see column 4, line 40 - line 53; figures 4,5 ---	1-5,7
A	US,A,4 900 228 (YAPP) 13 February 1990 cited in the application see abstract; figures ---	1,7
A	US,A,4 476 773 (FEHR) 16 October 1984 cited in the application see abstract; figures ---	1,7
A	FR,A,2 473 430 (REGIE NATIONAL DES USINES RENAULT) 17 July 1981 see page 2, line 10 - line 16; figures 1-2 -----	6,8

**ANNEX TO THE INTERNATIONAL SEARCH REPORT
ON INTERNATIONAL PATENT APPLICATION NO.**

EP 9300688
SA 72047

This annex lists the patent family members relating to the patent documents cited in the above-mentioned international search report.
The members are as contained in the European Patent Office EDP file on

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Patent document cited in search report	Publication date	Patent family member(s)	Publication date
EP-A-0491466	24-06-92	US-A- 5105730 JP-A- 4266518	21-04-92 22-09-92
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US-A-4476773	16-10-84	DE-A- 3144899 FR-A,B 2516213	19-05-83 13-05-83
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