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DESCRIPTION

FIELD OF THE INVENTION

[0001] The present invention relates to a fluid regulating valve for conditioning plants. The regulating valve of the invention enables controlling a flow internally of a circuit. The valve has a broad field of use, as it contemporaneously combines a high degree of precision and rapidity on the control of the fluid; purely by way of example, a typical application can be made in conditioning plants where an appropriate management of the flow crossing the plant is required so as to keep account of the frequent variations in flow rates to the various users.

PRIOR ART

[0002] As is known, flow regulating valves exist on the market which enable regulating a flow rate, pressure or pressure differential between two distinct points of a plant in which the valve is installed.

[0003] In a first example, illustrated in application US2007/0125432, use is made of a regulating valve with an inlet, an outlet and a membrane for receiving pressure signals and regulating a flow between the inlet and the outlet. The membrane works in opposition to a spring and at least partially checks the flow between the inlet and the outlet, thus determining a pressure difference between inlet and outlet of the valve. The position of the membrane is linked to the pressure of the fluid crossing the valve and to a further pressure signal coming from another point of the plant. The pressure of the fluids in contact with the membrane determines the position of the membrane, which increases or reduces the passage of fluid between inlet and outlet.

[0004] A second example, illustrated in application WO2010/074921 shows a flow control or load loss valve; the valve is provided with a control system for setting a reference value of the desired load loss or the desired flow rate value and for automatically regulating the position of an obturator.

[0005] A third example, illustrated in Korean application KR100835707, includes use of a flow regulating valve. The valve comprises an inlet opening, an outlet opening and an obturator. On the basis of the difference of pressure between a predetermined point of the plant and the point in which the valve is installed, the position of the obturator is modulated in opposition to the action of a spring directly connected to the obturator. The valve is provided with a system for regulating the spring preload; by regulating the preload it is possible to control the pressure difference between the point at which the pressures are read.

[0006] EP 2 226 536 and US 2003/0192 595 show further examples of regulating valves for plants.

SUMMARY OF THE INVENTION

[0007] A first aim of the present invention is to provide a valve enabling setting a series of values relating to physical parameters of the fluid circulating in the plant and which is able to regulate the flow of fluid on the basis of said settings.

[0008] A further aim of the invention is to provide a valve that is sensitive to pressure or flow variations internally of the circuit in which the valve is predisposed, enabling an automatic control of the fluid flow in at least a branch of the plant. A further aim of the invention is to provide a valve able to increase the sensitivity of the flow control.

[0009] A further aim of the invention is to provide a valve having a long operating path in which the sensitivity to flow variation is not compromised.

[0010] One or more of the above-mentioned aims is substantially attained by a valve according to one or more of the appended claims.

[0011] Further aspects of the invention are described below.

[0012] A 1st aspect includes a regulating valve for fluid plants, in particular conditioning plants, comprising: at least a valve body

exhibiting at least an inlet, at least an outlet and at least a channel which sets the inlet in fluid communication with the outlet, at least a check element of fluid operating in the channel and forming the regulating organ together with the valve body, the fluid check element 42 defining, in cooperation with the valve body 36, a passage opening 43 for fluid between the inlet 37 and the outlet 38 of a variable size according to positions assumed by the check element 42 in relation to the valve body 36 along a predetermined operating path; and at least a control device 25 configured to:

> receive in input at least a first signal relating to an intensity of a physical parameter of the fluid circulating in the first section 22 of the plant and a second signal relating to the intensity of a physical parameter of a fluid circulating in the second section 23 of the plant,

> generate the output signal as a function of the first signal and the second signal, said output signal being used to control the angular position of the check element 42.

[0013] In a 2nd aspect in accordance with the 1st aspect the check element 42 is configured such as to rotate with respect to a rotation axis 44 which extends transversally with respect to a prevalent axis of development 55 of said channel 39.

[0014] In a 3rd aspect in accordance with the 2nd aspect the control device 25 is configured such as to move the check element among a plurality of distinct operating positions which are angularly offset to one another. The angular positions are separate and at each distinct angular position the control device is configured such as to evaluate, on the basis of the output signal, whether an additional angular displacement is required or not. An angular step is defined between an operating position and a next angular operating position. In practice a step is understood to mean the angular distance between the distinct positions of the check element and is configured such as to perform the said evaluation. In a 4th aspect in accordance with the previous aspect, the angular step, at least for a predetermined tract of the operative path, is less than 1° (one degree).

[0015] In a 5th aspect in accordance with any one of the preceding 3rd or 4th aspects, the angular step, at least for the predetermined tract of the operative path, is less than 0.5° (half a degree).

[0016] In a 6th aspect in accordance with any of the above aspects from the 3rd to the 5th, the angular step, at least over a predetermined tract of the operating path, is less than 1°, optionally less than 0.5%; and/or wherein the angular step is not constant over the operating path, optionally wherein the tract in which the step is not constant comprises at least 10% of the operating path.

[0017] In a 7th aspect in accordance with any of the above aspects from the third to the sixth, where the angular step reduces progressively, optionally exponentially, for a stretch that includes at least 10% of the operating path.

[0018] In an 8th aspect according to any of the above aspects from 3rd to 7th, the size of the angular step is a function of the intensity of the physical parameter of the fluid circulating in the first section 22 and the intensity of the physical parameter of the fluid circulating in a second section 23 of the plant. In practice, the control device regulates the angular step as a function of the measurements, for example of pressure, in the two sections. For example the control device is configured such as to reduce the angular step progressively on reduction of a difference between a reference value and a real value given by a difference of intensity of the physical parameter of the fluid circulating in the first section and the intensity of the same physical parameter of the fluid circulating in a second section.

[0019] In a 9th aspect in accordance with any of the above aspects from 3rd to 8th, the tract of the operating path in which the step is not constant and/or is less than 1° comprises an initial tract 45 of the operating path, the initial tract being in turn comprised between an initial position 46 of complete closure of the passage opening 43 and an intermediate position 47 in which the passage opening 43 is open to not more than 40%.

[0020] In a 10th aspect in accordance with any of the above aspects from 3rd to 9th, the tract of the operating path in which the step is not constant and/or is less than 1° comprises an initial tract of the operating path, the final tract being in turn comprised between a final position of complete opening of the passage and an intermediate position 47 in which the passage opening is open to not more than 30%.

[0021] In a 11th aspect in accordance with any of the above aspects from 3rd to 10th, the tract of the operating path in which the step is not constant and/or is less than 1° comprises a final tract 48 of the operating path, the final tract being in turn comprised between a final position 49 of complete opening of the passage opening 43 and an intermediate position in which the passage opening 43 is open to not more than 50%.

[0022] In a 12th aspect in accordance with any of the above aspects from 3rd to 11th, the tract of the operating path in which the step is not constant and/or is less than 1° comprises a final tract 48 of the operating path, the final tract being in turn comprised between an final position 49 of complete closure of the passage opening 43 and an intermediate position in which the passage opening 43 is open to not more than 40%.

[0023] In a 13th aspect in accordance with any of the above aspects, the check element 42 has an external surface rotating about said axis 44 which exhibits a substantially spherical or cylindrical conformation.

[0024] In a 14th aspect in accordance with any of the above aspects, the valve body 36 exhibits at least a wall 36a located transversally of the prevalent development axis 50 of the channel 39 and respectively exhibiting a passage opening, said wall 36a being arranged upstream and/or downstream of the check element 42 such as to define the shape of that passage opening 43. In practice, the wall is a shaped diaphragm and in cooperation with the opening of the check element defines the shape of the passage opening.

[0025] In a 15th aspect in accordance with any one of the preceding aspects, the passage opening 43 is conformed such that following a movement of the check element 42 over said initial and/or a final tract 45, 48 of said operating path, the incremental ratio between the percentage variation of the area of the passage opening 43 and the percentage displacement of the check element 42 exhibits an absolute value comprised between 0 and 4, wherein the percentage variation of the area of the passage opening 43 comprises the ratio between the variation of the passage area following a displacement of the check element 42 and a reference area, and wherein said percentage displacement comprises the ratio between the displacement of the check element 42 and the operating path.

[0026] In a 16th aspect in accordance with any one of the preceding aspects, the passage opening 43 is shaped such that following a movement of the check element 42 over said initial and/or a final tract 45, 48 of the operating path, the incremental ratio between the percentage variation of the area of the passage opening 43 and the percentage displacement of the check element exhibits an absolute value comprised between 0 and 2.5.

[0027] In a 17th aspect in accordance with any one of the preceding aspects, the passage opening 43 is shaped such that following a movement of the check element 42 over an initial and/or a final tract 45, 48 of the operating path, the incremental ratio between the percentage variation of the area of the passage opening and the percentage displacement of the check element 42 exhibits an absolute value comprised between 0 and 1.5.

[0028] In a 18th aspect in accordance with any one of the preceding aspects from 15th to 17th, the incremental ratio between the percentage variation of the area of the passage opening 43 and the percentage displacement of the check element 42, over the initial tract comprising not more than 30% of the operating path, exhibits an absolute value comprised between 0 and 4.

[0029] In a 19th aspect in accordance with any one of the preceding aspects from 15th to 18th the incremental ratio between the percentage variation of the area of the passage opening 43 and the percentage displacement of the check element 42, over the initial tract comprising not more than 30% of the operating path, exhibits an absolute value comprised between 0 and 2.5.

[0030] In a 20th aspect in accordance with any one of the preceding aspects from 15th to 19th the incremental ratio between the percentage variation of the area of the passage opening 43 and the percentage displacement of the check element 42, over the final tract 48 comprising not more than 30% of the operating path, exhibits an absolute value comprised between 0 and 4.

[0031] In a 21st aspect in accordance with any of the preceding aspects from 15th to 20th the incremental ratio between the percentage variation of the area of the passage opening 43 and the percentage displacement of the check element 42, over the final tract 48 comprising not more than 40% of the operating path, exhibits an absolute value comprised between 0 and 2.5.

[0032] In a 22nd aspect in accordance with any of the preceding aspects from 15th to 21st the incremental ratio between percentage variation of the area of the passage opening 43 and the percentage displacement of the check element 42 exhibits an absolute value which is substantially constant over an intermediate tract 47 of the operating path comprised between the initial tract and the final tract, and wherein the intermediate tract comprises between 20% and 40% of the operating path.

[0033] In a 23rd aspect in accordance with any of the preceding aspects said first and second signal are related to the intensity of the fluid pressure in said first and second section 22, 23 of the plant.

[0034] In a 24th aspect in accordance with any one of the preceding aspects generating an output signal comprises determining a control differential signal comprising a difference or a ratio between the intensity of the first signal and the second signal.

[0035] In a 25th aspect in accordance with any one of the preceding aspects, generating the output signal comprises determining a control differential signal comprising a difference or a ratio between the intensity of the first signal and the second signal, wherein the control differential signal comprises the difference or ratio between the intensity of the fluid pressure in said first section 22 and the intensity of the fluid pressure in the second section 23, optionally wherein the control device 25 comprises a pressure differential gauge 55 which receives in input the first signal and the second signal and generates in output the control differential signal.

[0036] In a 26th aspect in accordance with any one of the preceding aspects, the control device 25 is configured to:

enable setting at least a reference value,

compare the reference value with the value of the control differential signal, generate the output signal as a function of the comparison.

[0037] In a 27th aspect in accordance with any one of the preceding aspects, the control device 25 comprises an actuator 56, for example an electrical actuator, acting on the check element 42, said actuator 56 enabling displacement of the check element 42 along the operating path.

[0038] In a 28th aspect in accordance with the preceding aspect, the control device 25 is connected with said actuator 56, the control device 25 further being configured such as to command a movement of the check element 42 as a function of the output signal and to position the check element 42 according to the plurality of positions along the operating path.

[0039] In a 29th aspect in accordance with any one of the preceding aspects 27th or 28th, the control device controls the size of the angular step according to which the actuator moves the check element in accordance with the value of the control differential signal and a reference value, optionally wherein the control device controls the size of the angular step according to which the actuator moves the check element in accordance with the difference between the control differential signal and the reference value.

[0040] In a 30th aspect in accordance with any one of the preceding aspects, said regulating valve 1 comprises acoustic signalling means 59 and/or optical signalling means 60, said acoustic and/or the optical signalling means 59, 60 being connected to the control device 25 which is configured such as to command said signalling means to provide:

an optical and/or acoustic representation of an instant value of the output signal, and/or

an optical and/or acoustic representation of displacement of an instant value of the output signal with respect to a reference value.

[0041] In a 31st aspect in accordance with any one of the preceding aspects, the control device 25 is configured to control a movement of the check element 42 thereby enabling a reduction in the difference between the reference value and the value of the control differential signal.

[0042] In a 32nd aspect in accordance with any one of the preceding aspects, said control device 25 includes setting means 57, said setting means 57 being connected to the control device 25, the control device being configured such as to enable a setting of a predetermined number of control parameters that define the working conditions of the valve.

[0043] In a 33rd aspect in accordance with any one of the preceding aspects, said control parameters comprising at least a parameter selected from the group having:

a first control parameter relating to a type of movement of the check element,

a second control parameter relating to a maximum excursion of the values of the first signal and the second signal or of the value of the difference between the first signal and the second signal,

a third control parameter relating to the type of function connecting the output signal and the first signal and the second signal,

a fourth control parameter relating to a velocity of convergence between the real value and the reference value.

[0044] In a 34th aspect in accordance with preceding aspect 33rd, said second control parameter comprises bottom scale values of the pressure differential gauge 55. In a 35th aspect in accordance preceding aspects 33rd or 34th, in relation to the third parameter, said function comprises a link of a proportional or integral or derivate type, or a combination thereof.

[0045] In a 36th aspect in accordance with the 33rd or 34th or 35th aspects said fourth control parameter specifies a convergence time defined as a time transitory in which the value of the control differential signal converges to the reference value.

[0046] In a 37th aspect in accordance with 33rd or 34th, 35th or 36th aspects said setting means 57 enable memorisation of a predetermined number of configurations relating to values of the control parameters which are useful for managing the control device 25, optionally wherein the setting means 57 comprise micro switches.

[0047] In a 38th aspect in accordance with any one of the preceding aspects, said regulating valve 35 further comprises a temperature sensor positioned on the valve body 36, the temperature sensor enabling detection of the temperature at the second section 23.

[0048] In a 39th aspect in accordance with any one of the preceding aspects, said control device 25 further comprises input means 37, said input means 37 enabling setting at least a reference value, optionally enabling setting a reference pressure differential between the first and the second section 22, 23.

[0049] In a 40th aspect in accordance with any one of the preceding aspects said control device 25 comprises a memory and is configured such as to enable memorising a plurality of predetermined reference values and for enabling selection of at least one of those values on the part of a user.

[0050] In a 41st aspect in accordance with one of the preceding aspects said check element 42 has a lateral surface of rotation of cylindrical or spherical shape. In a 42nd aspect in accordance with any of the one of the preceding aspects said passage opening 43 comprises a frontal shaping 51 exhibiting two substantially symmetrical parts wherein each of the parts exhibits a first portion shaped 52 as an arc of a circle, a second portion shaped 53 as an arc of a circle having a smaller radius than the arc of said first portion 52, a connecting inflection 54 which connects said first portion 52 with the second portion 53.

[0051] In a 43rd aspect in accordance with any one of the preceding aspects said passage opening 43 comprises a longitudinal shaping 61, relating to a longitudinal section with respect to the prevalent development axis 50 of the channel 39, exhibiting a progressively growing section from said inlet 37 to said outlet 38 of the valve body 36.

[0052] In a 44th aspect in accordance with any one of the preceding aspects wherein said control device is configured such as to receive the control parameters and, depending on the value thereof, adjust the rotation direction of the check element 42 according to said axis 44 which extends transversally to the axis of the prevalent development axis 50.

[0053] In a 45th aspect in accordance with any with any one of the preceding aspects said first section 22 is defined at the valve body 36, said regulating valve 36 comprising a passage 62 which places said channel 39 in fluid communication with the differential pressure gauge 55.

[0054] In a 46th aspect in accordance with any with any one of the preceding aspects said check element 42 operating in said channel 39 of the valve body 36 is configured such as to rotate according to a movement angle with respect to a rotation axis 44 which extends transversally with respect to a prevalent development axis 50 of said channel 39, optionally in which said check element 42 exhibits a lateral rotation surface having a cylindrical or spherical conformation.

[0055] In a 47th aspect in accordance with the preceding aspect the incremental ratio between the percentage variation of the area of the passage opening 43 and the percentage displacement of the check element 42, for the initial tract comprised between 20 % and 30% of the operating path, falls within a range of between 0 and 1.5.

[0056] In a 48th aspect according to any of aspects 46th or 47th, the incremental ratio between the percentage variation of the area of the passage opening 43 and the percentage displacement of the check element 42, for the final tract comprised between

20 % and 30% of the operating path, falls within a range of between 0 and 1.5.

[0057] A 49th aspect relates to a fluid distribution plant to a plurality of users, in which the plant uses one or more of the valves according to any one of aspects 1 to 48th.

[0058] A 50th aspect relates to a conditioning plant 1 comprising: at least a general inlet 2, at least a general outlet 3, a circuit 4 which sets the general inlet 2 in fluid communication with the general outlet 3, a plurality of users 5 arranged on the circuit 4, at least a balancing system 20 of the flow acting on said circuit 4, said balancing system 20 comprising:

➤

at least a sensor 21 for detecting at least a real value depending on a difference of intensity between a first section 22 upstream of the users and a second section 23 downstream of the users 5, of at least a physical parameter of the fluid,

➤

at least a flow regulating organ 24,

➤

at least a control device 25 connected to the sensor 21, acting on the flow regulating organ 24 and configured for enabling memorisation of at least a reference value of a same physical parameter of the fluid, comparing the reference value with the real value, commanding the regulating organ 24 via an output signal in order to regulate the fluid internally of the circuit such as to maintain the real value substantially aligned with the reference value, or such as to reduce a difference between real value and reference value.

[0059] In a 51st aspect according to the 50th aspect, the plant comprises a first detecting line 26 which places a first point 27 of the circuit 4 situated upstream of each user 5 in fluid communication with the sensor 21 of the balancing system 20, and a second detection line 28 which sets a second point 29 of the circuit 4 situated downstream of each user 5 in fluid communication with the sensor 21 of the balancing system 20.

[0060] In a 52nd aspect according to aspect 51st, said second point 29 is situated substantially on said regulating organ 24, said regulating organ 24 being situated downstream of each user 5, directly connected on the circuit 4 and in fluid communication with the sensor 21.

[0061] In a 53rd aspect according to aspects 50th, or 51st, or 52nd the sensor 21 is a differential sensor, which enables detecting the intensity difference between upstream and downstream of the users 5 of the physical parameter of the fluid, and wherein said sensor 21 detects a difference between a first real value relative to the intensity of the physical parameter at the first point 27 of the circuit 4, and a second real value relative to the intensity of the physical parameter relative to the second point 29 of the circuit 4.

[0062] In a 54th aspect according to aspects 50th, or 51st, or 52nd said sensor 21 comprises a first sensor 30 which is in fluid connection with the first detecting line 26 and enables detecting the intensity of the physical parameter at the first point 27 of the circuit 4, and a second sensor 31 which is in fluid connection with the second detecting line 28 and enables detecting the intensity of the physical parameter at the second point 29 of the circuit 4.

[0063] In a 55th aspect according to any of the preceding aspects from 50th to 54th, each user 4 comprises a respective delivery channel 6 emerging from the circuit 4, a respective return channel 7 in fluid connection with the delivery channel 6 and arranged such as to return fluid into the circuit 4, at least a user device hydraulically interposed between the delivery channel and the return channel 7. In a 56th aspect according to any of the above aspects from 50th to 55th, each user 5 comprises a respective delivery channel 6 emerging from the circuit 4, a respective return channel 7 in fluid connection with the delivery channel 6 and arranged such as to return fluid into the circuit 4, at least a user device hydraulically interposed between the delivery channel 6 and the return channel 7 and wherein each user 5 exhibits at least a respective closing organ 24, partial or total, arranged either on the delivery channel 6 or on the return channel 7.

[0064] In a 57th aspect according to any one of the preceding aspects 55th or 56th the plant comprises at least a delivery manifold 10 arranged upstream of the users 5, at least a return manifold 11 arranged downstream of the users 5, the delivery

manifold 10 and the return manifold 11 being respectively connected to the general inlet 2 for the distribution of the fluid to the users 5 and to the general outlet 3 for collecting the fluid in outlet from the users 5.

[0065] In a 58th aspect according to the preceding aspect, the regulating organ 24 is engaged to a line 32 interposed between an outlet of the return manifold 11 and the general outlet 3.

[0066] In a 59th aspect according to any of the above aspects from 50th to 57th, regulating organ 24 and the control device 25 are part of a regulating valve 35 operating downstream of the users 5, wherein the regulating valve 35.

[0067] In a 60th aspect according to preceding aspect 59th said regulating valve has the characteristics of any one of the aspects from the first to the forty-eighth.

BRIEF DESCRIPTION OF THE DRAWINGS

[0068] Some embodiments and some aspects of the invention will be described in the following with reference to the accompanying figures of the drawings, provided by way of non-limiting example, in which:

Figure 1 illustrates a first configuration of a conditioning plant, which is not part of the invention

Figure 2 illustrates a second configuration of a conditioning plant, which is not part of the invention

Figure 2a illustrates a third configuration of a conditioning plant, which is not part of the invention

Figure 3 is a perspective view of a valve used in the plants of figures 1-2a;

Figure 4 is a front view of the regulating valve of figure 3 in a first use condition;

Figure 5 is a section view of the regulating valve of figure 3;

Figure 6 is a front view of the regulating valve of figure 3 in a second use condition;

Figure 7 is a front view of the regulating valve of figure 3 in a third use condition;

Figure 8 is a perspective view of the valve body of the valve of figure 3;

Figure 9 is a view from above of an alternative embodiment of a regulating valve used in the plant of figure 1;

Figure 10 is a front view of the valve of figure 9;

Figure 11 is a section along line XI-XI of figure 10; and

Figure 12 is a comparative diagram relating to the characteristics of two valves that relates the percentage area of the passage opening with the percentage displacement of the check element.

DETAILED DESCRIPTION

[0069] With reference to the accompanying figures of the drawings, 1 denotes in its entirety a conditioning plant comprising a general inlet 2, a general outlet 3, a circuit placing the general inlet in fluid communication with the general outlet 3. Both the general inlet and the general outlet avail of an inlet valve 2a and an outlet valve 3a, for example of a sphere or shutter type, which regulate the general flow internally of the circuit 4.

[0070] The circuit supplies a fixed number of user units 5 (herein indicated as users), for example, each comprising one or more heat exchangers and related groups of ventilation (fan coil), which are arranged at or near areas to be conditioned. Heat exchange takes place inside the users 5, enabled by the fluid supply from the general inlet 2.

[0071] The circuit exhibits at least two main channels: a delivery channel 6, for sending the fluid to the users, and a return channel 7 in connection with the delivery channel 6 and arranged such as to receive the fluid downstream of the users. The users 5 are hydraulically interposed between the delivery channel 6 and the return channel 7. Before being distributed to the various

utilities the fluid is intercepted by a filter 8 which prevents impurities from reaching the users.

[0072] The delivery channel 6 and the return channel 7 are respectively connected to a delivery manifold 10 upstream of the users 5 and a return manifold 11 downstream of the users 5. The delivery manifold 10 and the return manifold 11 are connected respectively to the general inlet 2, for the distribution of the fluid to the users 5, and the general outlet 3, for the collection of the fluid in outlet from the users 5.

[0073] The conditioning plant also includes, for each user, at least one respective delivery line 6a exiting from the delivery manifold, and leading to the users; a respective partial or total closing organ 9 acts on each delivery line 6a. The closing organs 9 intercept the fluid in inlet to the users 5. The organs 9 are activated such as to vary the supply parameters of a determined user; in particular each closing organ can comprise an on/off valve which closes or opens supply to each user 5.

[0074] The return manifold 7 receives from each user a respective return line 12; on each return line a balancing valve 13 is placed whose function is to regulate the flow in outlet from the users as a function of temperature requirements required for each environment in which the user 5 is installed.

[0075] The plant also has a connecting circuit 14, which sets the delivery manifold 10 in fluid communication with the return manifold 11, bypassing the users. A shut-off valve 15 of the manifold 10 and a shut-off valve 16 of the return manifold 11 are arranged on the connecting circuit 14. The connecting circuit 14 is in turn in fluid connection with a discharge 17 interposed between the respective closing valves of the delivery manifold closing valve and return valve. The discharge 17 exhibits a general discharge valve 18 configured for discharging excess fluid present in the connecting circuit 14 after the circuit 4 setting. In addition to the general discharge valve 18, the drain circuit 17 exhibits a breather valve 19, positioned in a higher position than the manifolds and useful for eliminating any air bubbles inside the circuit 4.

[0076] The air conditioning plant 1 comprises a flow balancing system 20, active on the circuit 4. The balancing system 20 comprises at least a sensor 21 for detecting a real value which is dependent on the difference of intensity that a same physical parameter, such as a pressure or flow, assumes between a first section 22 upstream of the users 5 and a second section 23 downstream of the users. More specifically, the sensor can measure for example a difference or ratio between the real pressure in the section 22 and the pressure in the section 23, providing an output signal that is proportional to said difference or ratio between the pressures in the two above-cited sections.

[0077] The balancing system further exhibits a flow regulating organ 24 and a control device 25 connected to the sensor 21 and active on the flow regulating organ 24. In practice, the control device comprises a control unit, for example a microprocessor, capable of receiving the input signal from the sensor/s 21 and acting on the regulating organ as a function of the signal. According to one embodiment, the balancing system 20 is placed in connection with two detecting lines. More specifically, a first detecting line 26 places in fluid communication a first point 27, part of the first section 22 of the circuit 4 and located upstream of each user 5, with the sensor 21 of the balancing system 20; a second detecting line 26 places in fluid communication a second point 28, part of the second section 23 of the circuit 4 and located downstream of each user 5, with the sensor 21 of the balancing system 20. The sensor 21, for example a differential sensor, is configured such as to detect the difference in intensity between upstream and downstream of the users 5, the physical parameter of the fluid. In more detail, the sensor 21 detects a difference between a first real value related to the intensity of the physical parameter at the first point 27 of the circuit 4, and a second real value of the intensity of the same physical parameter related to the second point 29 of the circuit 4. The second point 29 is for example located substantially on the regulating organ, which is arranged downstream of each user 5, directly connected to the circuit 4 and in fluid connection with the sensor 21. If the detected parameter is the pressure, the sensor 21 includes a pressure differential sensor which generates an output signal proportional to the intensity or a function of the pressure difference between points 27 and 29.

[0078] Alternatively to the use of a differential sensor, a different configuration of the circuit can be realised in which the sensor 21 comprises a first sensor 30, in fluid connection with the first detecting line 26, which enables detecting the intensity of the physical parameter at the first point 27 of the circuit 4, and a second sensor 31, which is in fluid connection with the second detecting line 28, which detects the intensity of the same physical parameter at the second point 29 of the circuit 4.

[0079] The return channel 7 can comprise a line 32, interposed between a general outlet 33 of the return manifold 11 and the general outlet 3 of the plant 1, on which the regulating organ 24 is engaged.

[0080] As previously mentioned, the first point 27 is located upstream of the users 5, in particular, the first point 27 can be located substantially on the delivery manifold 10, while the second detecting point 29 is located on the regulating organ 24. As for

the control device 25, it is connected to the above-described sensor/s and with the regulating organ. The control device is configured such as to enable a memorising of a reference value for the physical parameter of the fluid being measured with the described sensors. For example, the control device may comprise one or more microprocessor unit/s and associated memories capable of storing a code which when executed by the microprocessor unit/s makes the control device capable of performing the procedures described below. The control device is also configured such as to compare the reference value with the real value in relation to the same physical parameter, and such as to control the regulating organ 24 via an output signal such as to regulate the flow of fluid within the circuit 4 such as to maintain the value substantially aligned with the reference value. For example, the control device is configured to determine the real value as the difference between the values of the physical parameter measured in the first and second section by the sensor/s 21 (for example the difference in pressure between the two sections) and such as to compare the difference with a reference value stored by the control device. Following the comparison, if the difference between the pressures in the two above-mentioned sections is equal to or close to the set reference value, the control device does not exert any action up until a subsequent control cycle that can be activated manually or determined periodically by the control device or determined by the control device on reaching a threshold of maximum deviation between the real value and the reference value.

[0081] The plant also has available a display instrument 34 connected to the control device 25. The control device 25 is configured to control a visualising on the display instrument 34 of the real value (for example, the real difference between the pressures measured in sections 22 and 23), the reference value (for example, a reference pressure difference) or both the values of said physical parameter of the fluid.

[0082] In the example of realization shown, the regulating organ 24 and the control device 25 are part of a regulating valve 35 operating downstream of the users 5. The regulating valve 35 includes: a valve body 36 exhibiting at least an inlet 37, an outlet 38 and a channel 39 which sets the inlet 37 in fluid communication with the outlet 38. The valve body 36 exhibits connecting organs 40 arranged at the inlet 37 and the outlet 38 of the valve which enabling fixing the valve on the circuit 4. The connecting organs can be, for example, threading or rapid attachments. The channel 39 exhibits a seating 41 capable of accommodating a check element 42 which in cooperation with the valve body 36 forms the regulating organ 24 and defines a passage opening 43 between the inlet 37 and outlet 38. The passage opening 43 has variable amplitude as a function of the positions assumed by the check element 42 with regard to the valve body 36. The check element 42 is configured such as to act along a predetermined travel path which includes a predetermined number of operating positions that are angularly distinct or translationally staggered to one another: in practice, the control device is configured such as to move the check element between an operating position and a next operating position in a next step controlled by the control device 25 itself, as is described in more detail below.

[0083] The movement of the check element 42 may be rotary or translational. In the first case the movement takes place by rotation in an angle of movement around an axis of rotation 44 which extends transversally with respect to a prevalent development axis 50 of the channel 5. In the second case, the movement can take place in a straight direction.

[0084] Considering the case where the check element 42 performs a rotary movement: to perform a rotary movement, the check element 42 must have an outer surface that is substantially spherical or cylindrical, as can be seen in the accompanying figures. The various operative positions of the element 42 are angularly offset by one or more angular steps of amplitude that are controlled by the control device: at each angular step the control device is programmed to periodically re-verify the difference between the real value and the reference value, and if this difference is not acceptable, the control device will command a new angular step.

[0085] In the case of a rotary valve, the control device can be configured such that along at least part of the operating path (such as a tract greater than or equal to 10% of the operating path), the pitch angle has a value of less than 1° (one degree), optionally less than 0.5° (half a degree) in order to avoid destabilisation in the control of the valve.

[0086] The control device can also be made to adjust the angular pitch in order to maintain this constant pitch along the operating path.

[0087] Alternatively, the control device can adjust the pitch so that the amplitude thereof is variable along a tract (for example, for a distance greater than or equal to 10% of the operative path) or along the entire operating path. For example, a step can be included that is a function of the deviation between the real value of the physical parameter detected by the sensors and the reference value of the physical parameter itself: for example, the step can be relatively large (for example one or more degrees) if the difference between the real value and the reference value is greater than a certain threshold and may be relatively small (e.g. less than one degree) if the difference between the real and the reference values is below a certain threshold. Alternatively, in case of variable pitch, the angular pitch can be reduced progressively during the opening or closing of the valve: optionally the step may for example be reduced gradually, exponentially as the valve is opened.

[0088] As shown in figure 12, the tract of the operating path where the pace is variable and possibly less than one degree includes an initial tract 45 between an initial position 46 of complete closure of the passage opening 43 and an intermediate position 47 in which the passage opening 43 is open no more than 50%, optionally not more than 40%. The tract of the path in which the pitch is variable and possibly less than a degree also includes a final section 48 between a final position of complete opening of the passage opening 43 and the intermediate position 47 in which the passage opening 43 is open no more than 40%, optionally not more than 30%.

[0089] In an embodiment, illustrated in figures 4, 6, 7 and 10, the passage opening 43 is shaped such that, following a movement of the check element 42 over a first and/or final tract (which may coincide with the lines 45, 48 of figure 12) of the operating path, the incremental relationship between the percentage change of the area of the passage opening 43 and the percentage displacement of the check element 42 exhibits an absolute value that is variable along the path but comprised between 0 and 4, optionally between 0 and 2.5. Thanks to this type of variation the increase in area upon the movement of the check element is not excessive and thus enables an efficient control of the pressure fall (and flow) through the valve, enabling better management of the plant balance (again, a curve characteristic of a profiled-opening valve is represent in figure 12 - see the curve with the continuous tract with reference triangles).

[0090] As can be seen in figure 12, over the initial tract 45 that includes no more than 30% of the operating path, the incremental ratio between the percentage variation of the passage opening 43 area and the percentage displacement of the check element 42 exhibits an absolute value variable along the path but always comprised between 0 and 4, for example between 0 and 2.5. More specifically, the incremental ratio falls in a range between 0 and 1.5 in the initial tract that is between 20 and 30% of the operating path. In turn, over the final tract 48 that includes no more than 40% of the operating path, the ratio between incremental percentage change of the passage opening 43 and the percentage displacement of the check element 42 exhibits an absolute value that is variable but comprised between 0 and 4, for example between 0 and 2.5. More specifically this incremental ratio falls in a range between 0 and 1.5 in a final tract that is between 20 and 30% of the operating path. Finally, still with reference to figure 12, over an intermediate section 47 of the operating path between the initial tract 45 and the final tract 48 comprising between 20% and 40% of the operating path, the incremental ratio between percentage change of the area of the passage opening 43 and the percentage displacement of the check element 42 exhibits a substantially constant absolute value.

[0091] Note that the percentage displacement is defined as the ratio between the displacement of the check element 42 and the operating path.

[0092] In turn, the percentage change of the passage opening 43 is defined as the ratio between the change in the passage area following a displacement of the check element 42 and a reference area, for example the area of complete opening of the passage opening 43.

[0093] As previously described the check element 42 is operating internally of the channel 39 of valve body 36, and is configured to vary the passage opening 43 through its movement. As can be seen from the figures, the check element 42 is configured to rotate at an angle of motion with respect to an axis of rotation 44 which extends transversally with respect to a prevalent development axis 50 of the channel 39. In particular, the check element 42 can be configured such as to perform a displacement that will reduce or increase the passage opening 43. According to a front view, as can be seen in figure 4 the passage opening 43 is characterized by a front profiling 51 presenting two substantially symmetrical parts in which each part has a first portion 52, shaped as an arc of a circle, a second portion 53, shaped as an arc of a circle having a smaller radius than that of the first portion 52, a connecting curve 54 which connects the first portion 52 with the second portion 53. According to the longitudinal view, the passage opening has a longitudinal shaping 61, which is characterized by a progressively increasing section from the inlet 37 to the outlet 38 of the valve body 36. In the example shown, although the section progressively grows, the outline of this section in the front view can remain constant and equal to what described above and shown in figure 3.

[0094] The movement of the regulating organ and therefore the check element 42 is, as mentioned, regulated by the control device 25 which is in particular configured such as to receive at least a first input signal relating to the intensity of the physical parameter of the fluid circulating in the first section of the plant and a second signal relating to the intensity of the same physical parameter of a fluid circulating in the second section of the plant, and then generate an output signal according to the first and second signal. The first and second signals are related to the intensity of fluid pressure respectively in the first and second section of the plant.

[0095] The output signal is control differential signal comprising a difference or ratio between the intensity of the first and second signal; for example the control differential signal includes the difference or the ratio between the intensity of fluid pressure in the

first section and the intensity of fluid pressure in the second section.

[0096] The intensity of the output signal is used to control the position of the check element 34 with a cyclically executed control loop. As a sensor a differential pressure gauge 55 can be used which receives in input the first and second signal and generates in output a control differential signal. The differential pressure gauge 55 is disposed on the valve body 36 of the regulating valve 35 and receives the first signal from the first point 27 and the second signal corresponding to the second point 29 of the plant. The second point is located substantially at the outlet of the regulating valve, and is therefore placed after the check element 42, according to the flow direction. The output 38 of the valve and the pressure differential gauge 55 are in fluid connection via a passage 62.

[0097] The intensity of the value measured by the differential pressure gauge 55 is either proportional to or a function of the pressure difference between the first and second points 27, 29. Although reference has been made just above to a physical parameter corresponding to the pressure, the physical parameter can also be the delivery; in this case the differential sensor may be a differential between the flow rate in the first and the second segment.

[0098] On the section where the valve body is placed a temperature sensor can also be present which measures the temperature of the fluid in outlet from the valve.

[0099] The control device 25 is also configured to allow the setting of at least one reference value, compare the reference value with the value of the differential signal control and generate the output signal as a function of the comparison. The comparison of the signals influences the displacement of the check element 42, which is moved directly by an actuator 56, active on the check element 42, along the operating path. The actuator 56 can be electric or mechanical. The case analyzed shows an actuator 56 of an electrical type that enables placing the check element 42 in a plurality of positions along the operating path. The control device 25 regulates the movement of the check element 42, thereby reducing the difference between the reference value and the value of the control differential signal.

[0100] The control device 25 includes a memory which enables storage of a plurality of predetermined reference values and allows the selection thereof by a user.

[0101] The control device 25 further comprises setting means 57. The setting means 57 are connected to the control device 25 and enable setting a fixed number of control parameters which define working conditions.

[0102] The control device 25 further comprises inlet means 58, which enable a reference value to be set, for example by enabling the setting of a reference differential pressure between the first and second sections.

[0103] The control parameters include: a first control parameter on the type of check element 42, a second control parameter relating to an maximum increase in the values of the first and second signal or in the value of the difference between the first and second signals, a third control parameter relating to the type of function which links the outlet signal to the first and the second signals, a fourth control parameter relative to the velocity of convergence between the real value and the reference value.

[0104] Going into further detail it can be specified that, in the case where the check element 42 is of the circular type, the first parameter determines the direction of rotation of the check element 42. Instead the second parameter comprises the bottom-scale values of the pressure the differential pressure gauge 55 thus defining a range of predetermined values. The third control parameter specifies the type of control function that links the output signal to said first and second signal: for example, this function includes a link of a proportional or integral or derivative type, or a combination of these. The fourth control parameter regulates the speed of convergence of the real value of the reference value set to the control device 25.

[0105] The setting means 57 also enable memorisation of a predetermined number of configurations relating to values of the control parameters useful for the management of the control device 25.

[0106] In the case shown in the figures the setting means are micro-switches which have only two values for each parameter setting control.

[0107] The regulating valve 35 enables monitoring the real value and the reference value by means of an acoustic signal 59 and/or optical signal means 60. These signalling devices are connected to the control device 25 which is configured to control the acoustic and/or optical means 59, 60 such as to provide: an optical and/or an instantaneous value of the output signal, and/or an optical and/or an acoustic displacement of an instantaneous value of the output signal with respect to a reference value.

REFERENCES CITED IN THE DESCRIPTION

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Patentkrav

1. Reguleringsventil (35) til fluider til anlæg, især konditioneringsanlæg, omfattende:

- 5 mindst et ventillegeme (36) fremvisende mindst et indløb (37), mindst et udløb (38) og mindst en kanal (39), som sætter indløbet (37) i fluidforbindelse med udløbet (38),
mindst et fluidkontraelement (42) opererende i kanalen (39), hvor fluidkontraelementet (42) definerer en passageåbning (43) til fluid mellem
10 indløbet (37) og udløbet (38) som har en variabel størrelse som en funktion af positioner antaget af kontraelementet (42) i forhold til ventillegemet (36) langs en forudbestemt driftsvej, hvor passageåbningen (43) er formet således at følgende en bevægelse af kontraelementet (42) over mindst en initial og/eller endelig strækning af driftsvejen,
15 inkrementelforholdet mellem procentvariation af arealet af passageåbningen (43) og procentforskydningen af kontraelementet (42) udviser en absolut værdi omfattet mellem 0 og 4,00, eventuelt mellem 0,00 og 2,50,

kendetegnet ved at ventilen yderligere omfatter:

- 20 en styringsindretning (25) konfigureret til:
 at modtage i input mindst et første signal som relaterer til en intensitet af et fysisk parameter af en fluid som cirkulerer i et første afsnit (22) af et anlæg og et andet signal som relaterer til intensiteten af det fysiske parameter af en fluid som cirkulerer i det
25 andet afsnit (23) af anlægget,
 at generere et outputsignal som en funktion af det første signal og det andet signal, hvor outputsignalet er anvendeligt til at styre positionen af kontraelementet (42),
hvor passageåbningen (43) af fluidkontraelementet (42) omfatter en
30 langsgående form som relaterer til et langsgående tværafnsnit i forhold til den fremherskende udviklingsakse af kanalen, som udviser et progressivt voksende tværafnsnit fra indløbet til udløbet af ventillegemet.

- 2.** Reguleringsventilen ifølge krav 1, hvor inkrementelforholdet mellem
35 procentvariationen af arealet af passageåbningen (43) og procentforskydningen af

kontraelementet (42), over den initiale strækning, som omfatter ikke mere end 30% af driftsvejen, udviser en absolut værdi omfattet mellem 0 og 4, eventuelt mellem 0 og 2,5;

hvor inkrementelforholdet mellem procentvariation af arealet af passageåbningen (43) og procentforskydningen af kontraorganet (42), over den endelige strækning (48) som omfatter ikke mere end 40% af driftsvejen, udviser en absolut værdi omfattet mellem 0 og 4, eventuelt mellem 0 og 2,5; hvor

inkrementelforholdet mellem procentvariation af arealet af passageåbningen (43) og procentforskydningen af kontraelementet (42) udviser en absolut værdi som er

10 i alt væsentligt konstant over en mellemliggende strækning (47) af driftsvejen omfattet mellem den initiale strækning (45) og den endelige strækning (48), og hvor den mellemliggende strækning (47) omfatter mellem 20% og 40% af driftsvejen,

og hvor procentvariationen af arealet af passageåbningen (43) omfatter forholdet

15 mellem variationen af passagearealet (43) følgende en forskydning af kontraorganet (42) og et referenceareal, og hvor procentforskydningen omfatter forholdet mellem forskydningen af kontraelementet (42) og driftsvejen.

3. Reguleringsventilen ifølge et hvilket som helst af de foregående krav, hvor det

20 første signal og det andet signal relaterer til intensiteten af fluidtrykket i det første afsnit (22) og det andet afsnit (23) af anlægget, og hvor at generere outputsignalet omfatter at bestemme et differentiale styresignal omfattende en forskel eller et forhold mellem intensiteten af det første signal og det andet signal, eventuelt hvor det differentiale styresignal omfatter forskellen eller forholdet

25 mellem intensiteten af fluidtrykket i det første afsnit (22) og intensiteten af fluidtrykket i det andet afsnit (23), endnu mere eventuelt hvor styreindretningen (25) omfatter en trykdifferensmåler (55) som modtager i input det første signal og det andet signal og genererer i output det differentiale styresignal.

30 **4.** Reguleringsventilen ifølge et hvilket som helst af de foregående krav, hvor styreindretningen (25) er konfigureret til:

- at aktivere indstilling af mindst en referenceværdi,
- at sammenligne referenceværdien med værdien af det differentiale styresignal,
- 35 at generere outputsignalet som en funktion af sammenligningen.

5. Reguleringsventilen ifølge et hvilket som helst af de foregående krav, hvor styreindretningen omfatter en aktuator (56), for eksempel en elektrisk aktuator, som virker på kontraelementet (42), hvor aktuatoren (56) muliggør forskydning af kontraelementet (42) langs driftsvejen, hvor styreindretningen (25) er forbundet
- 5 med aktuatoren (56), hvor styreindretningen (25) yderligere er konfigureret til at styre en bevægelse af kontraelementet (42) som en funktion af outputsignalet.
6. Reguleringsventilen ifølge krav 5, hvor styreindretningen er konfigureret til:
- 10 at positionere kontraelementet (42) i henhold til en flerhed af positioner langs driftsvejen, hvor positionerne separeres af et trin, hvor styreindretningen gentager trinnene til sammenligning og generering ved hver af positionerne;
- at styre størrelsen af trinnet med hvilket aktuatoren bevæger kontraelementet som en funktion af værdien af det differentiale styresignal og referenceværdien, eventuelt hvor styreindretningen styrer størrelsen af
- 15 trinnet i henhold til hvilket aktuatoren bevæger kontraelementet som en funktion af forskellen mellem værdien af det differentiale styresignal og referenceværdien.
- 20 7. Reguleringsventilen ifølge et hvilket som helst af de foregående krav, hvor reguleringsventilen (1) omfatter akustisk signaleringsorgan (59) og/eller optisk signaleringsorgan (60), hvor det akustiske signaleringsorgan (59) og/eller optiske signaleringsorgan (60) er forbundet til styreindretningen (25) som er konfigureret til at kommandere signaleringsorganerne (59, 60) til at forsyne:
- 25 en optisk og/eller akustisk repræsentation af en øjebliksværdi af outputsignalet, og/eller en optisk og/eller akustisk repræsentation af forskydning af en øjebliksværdi af outputsignalet i forhold til en referenceværdi.
- 30 8. Reguleringsventilen ifølge et hvilket som helst af de foregående krav, hvor styreindretningen (25) er konfigureret til at styre bevægelsen af kontraelementet (42) og til at bestemme mindskelsen af forskellen mellem referenceværdien og værdien af det differentiale styresignal.

9. Reguleringsventilen ifølge et hvilket som helst af de foregående krav, hvor styreindretningen (25) omfatter indstillingsorgan (57), hvor indstillingsorganet (57) er forbundet til styreindretningen (25) og aktiverer indstilling af et forudbestemt antal af styringsparametre som definerer driftstilstande af ventilen, 5 og hvor styringsparametrene omfatter mindst et parameter valgt fra gruppen med:

et første styringsparameter som relaterer til en type af bevægelse af kontraelementet,
et andet styringsparameter som relaterer til en maksimal afvigelse af 10 værdierne af det første signal og det andet signal eller af værdien af forskellen mellem det første signal og det andet signal,
et tredje styringsparameter som relaterer til typen af funktion som forbinder outputsignalet og det første signal og det andet signal,
et fjerde styringsparameter som relaterer til en konvergensthastighed 15 mellem realværdien og referenceværdien,
hvor det andet styringsparameter omfatter bundskalaværdier af trykdifferensmåleren (55),
hvor i forhold til det tredje parameter funktionen omfatter en forbindelse af en proportional eller integreret eller afledt type, eller en kombination deraf, 20 og hvor det fjerde styringsparameter specificerer en konvergenstid defineret som en kortvarig tid i hvilken værdien af det differentiale styresignal konvergerer til referenceværdien,
og hvor indstillingsorganet (57) aktiverer hukommelse af et forudbestemt antal af konfigurationer som relaterer til værdier af styringsparametrene 25 som er anvendelige til at styre styreindretningen (25), eventuelt hvor indstillingsorganet (57) omfatter mikrokontakter.

10. Reguleringsventilen ifølge et hvilket som helst af de foregående krav, hvor styreindretningen (25) omfatter inputorgan (58), hvor inputorganet (58) aktiverer 30 indstilling af mindst en referenceværdi, eventuelt aktiverer indstilling af en referencetrykdifferens mellem det første afsnit (22) og det andet afsnit (23), og hvor styreindretningen (25) omfatter en hukommelse og er konfigureret således til at aktivere hukommelse af en flerhed af forudbestemte referenceværdier og til at aktivere en bruger til at vælge mindst en af værdierne.

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11. Reguleringsventilen ifølge et hvilket som helst af de foregående krav fra 2 til 10, hvor kontraelementet (42) som opererer i kanalen (39) af ventillegemet (36) er konfigureret til at rotere i henhold til en bevægelsesvinkel i forhold til en rotationsakse (44) som strækker sig tværgående i forhold til en fremherskende udviklingsakse (50) af kanalen (39), eventuelt hvor kontraorganet (42) udviser en lateral rotationsflade med en cylindrisk eller sfærisk form;
5 hvor inkrementelforholdet mellem en procentvariation af arealet af passageåbningen (43) og procentforskydningen af kontraelementet (42), over den initiale strækning omfattet mellem 20% og 30% af driftsvejen, falder inden for et
10 område på mellem 0 og 1,5, og/eller hvor inkrementelforholdet mellem en procentvariation af arealet af passageåbningen (43) og procentforskydningen af kontraelementet (42), over den endelige strækning omfattet mellem 20% og 30% af driftsvejen, falder inden for et område på mellem 0 og 1,5.

15

12. Reguleringsventilen ifølge et hvilket som helst af de foregående krav, hvor passageåbningen (43) omfatter en frontal udformning (51) som udviser to i alt væsentligt symmetriske dele i hvilken hver af delene udviser en første del (52) formet som en bue af en cirkel, en anden del (53) formet som en bue af en cirkel
20 med en mindre radius end buen af den første del (52), en forbindelsesinfleksion (53) som forbinder den første del (52) med den anden del (53), og/eller hvor passageåbningen (43) omfatter en langsgående form (61) som relaterer til et langsgående tværafnsnit i forhold til den fremherskende udviklingsakse (50) af kanalen (39), som udviser et progressivt voksende tværafnsnit fra indløbet (37) til
25 udløbet (38) af ventillegemet (36).

DRAWINGS

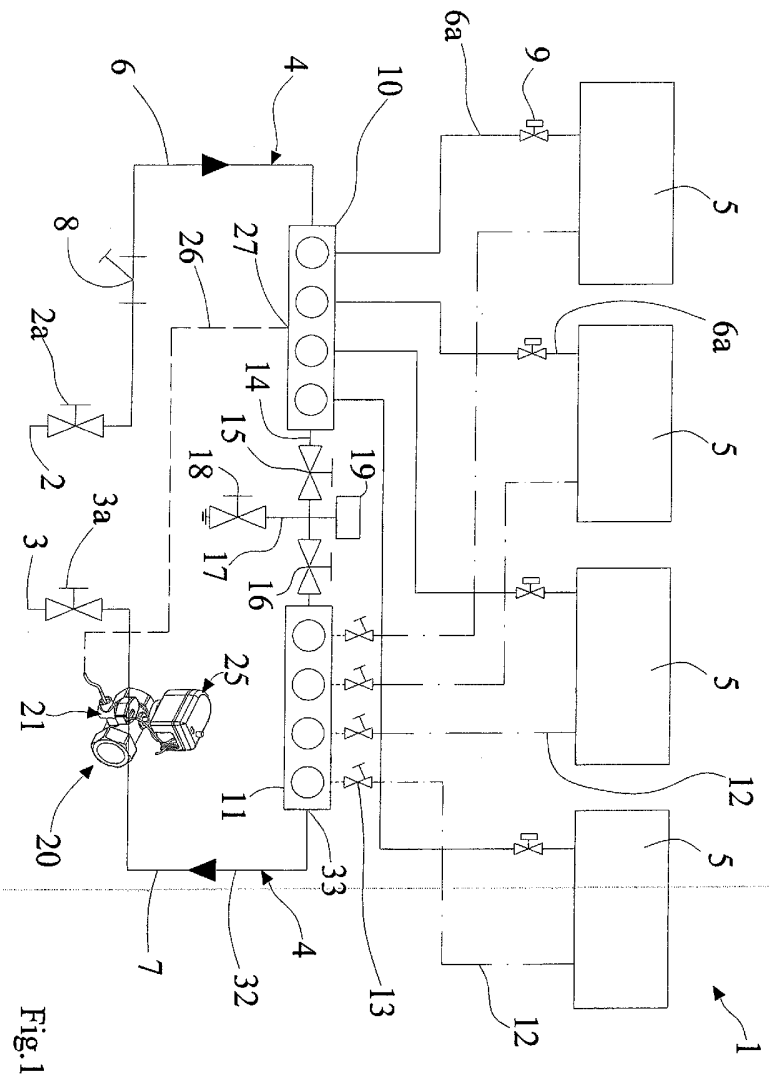


Fig. 1

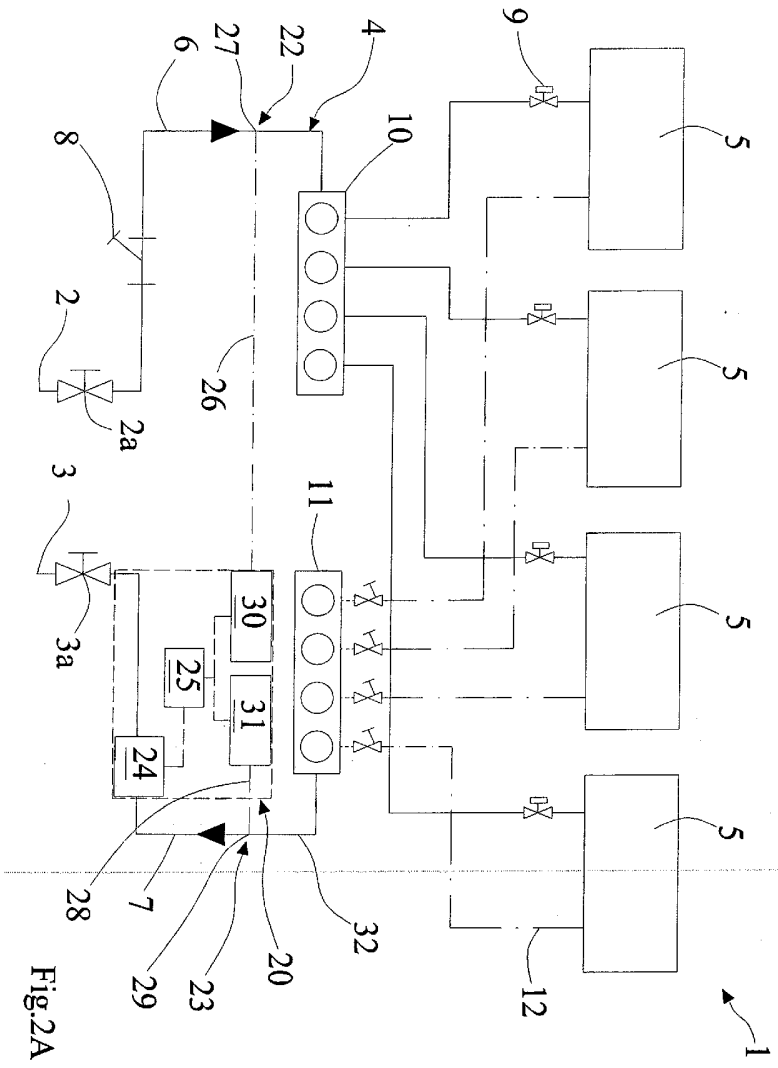
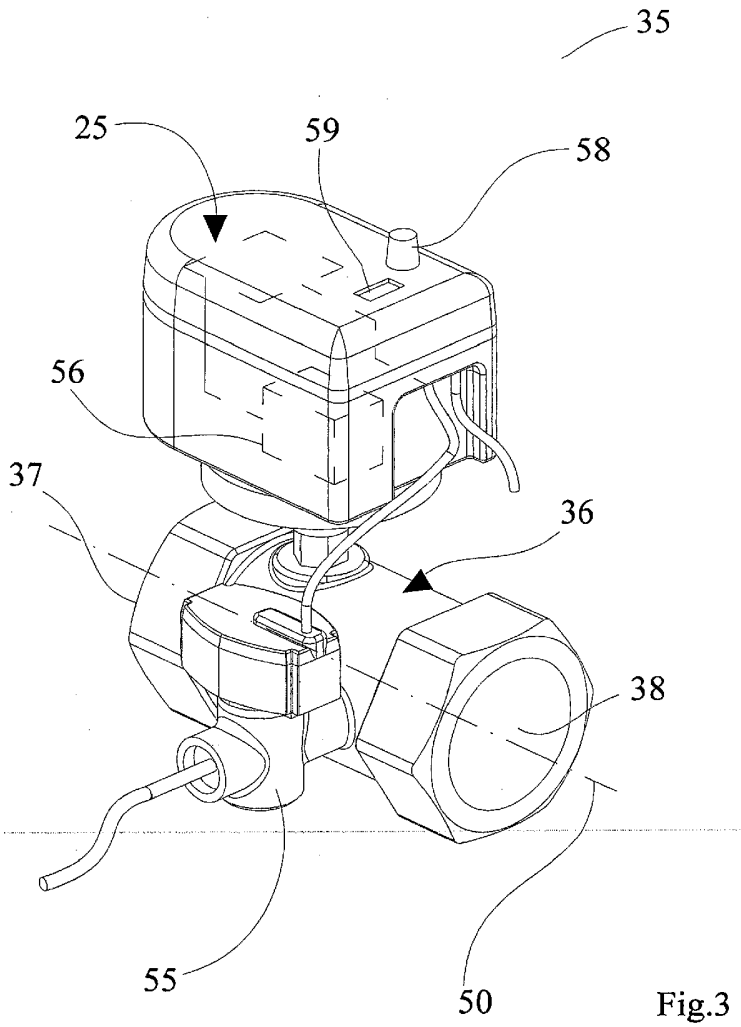


Fig.2A



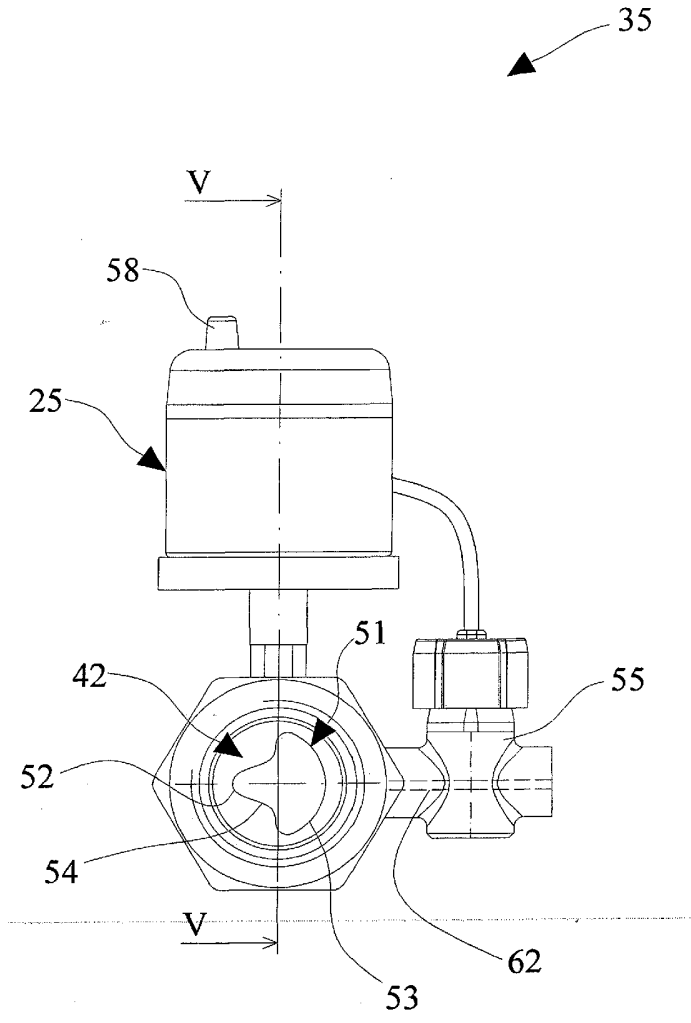
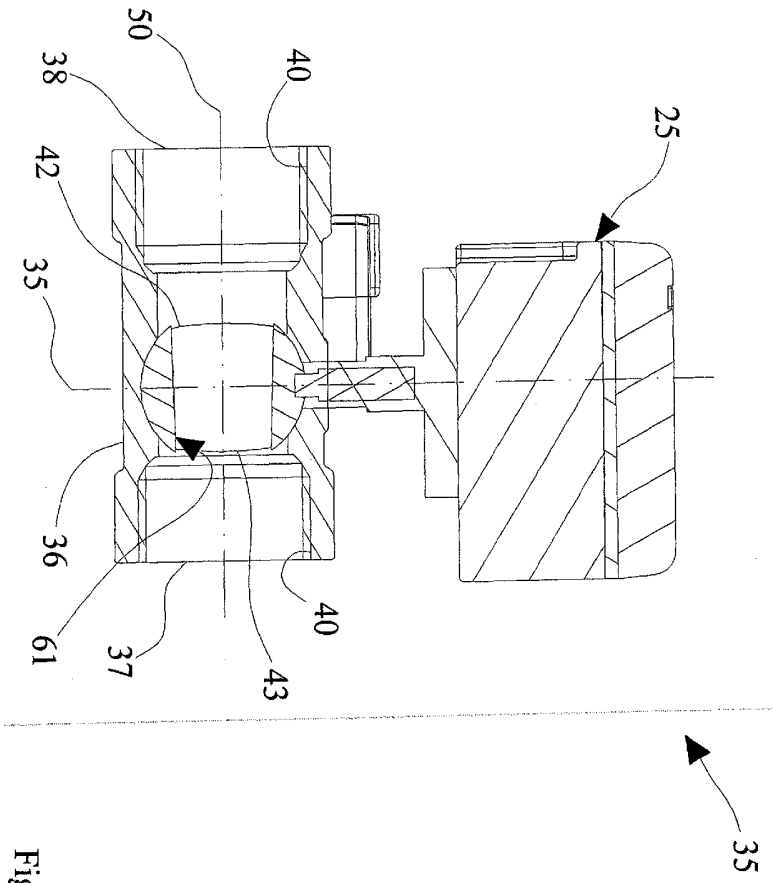


Fig.4



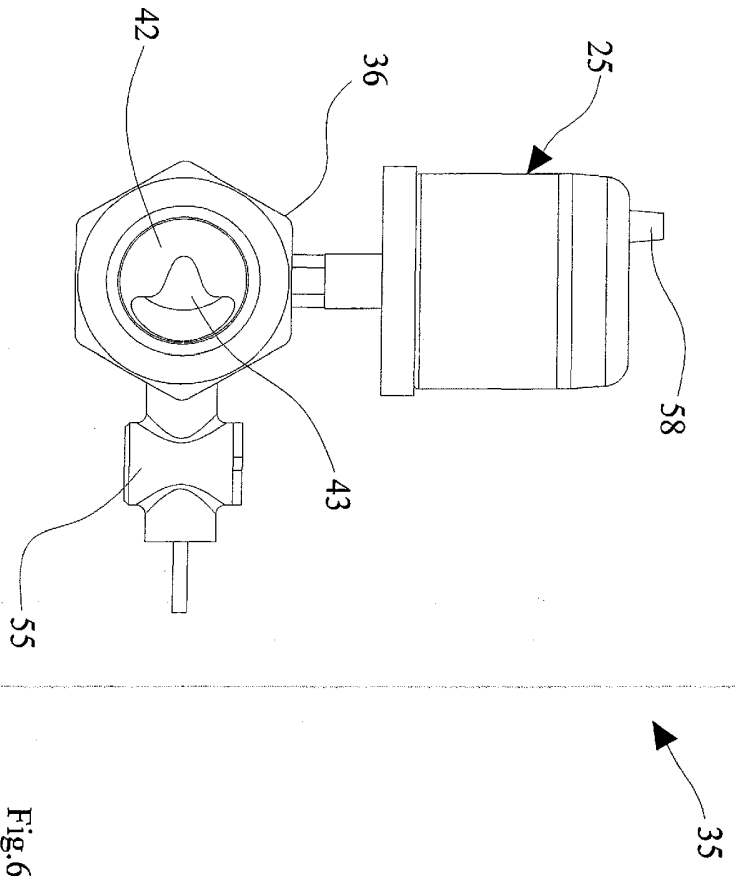


Fig. 6

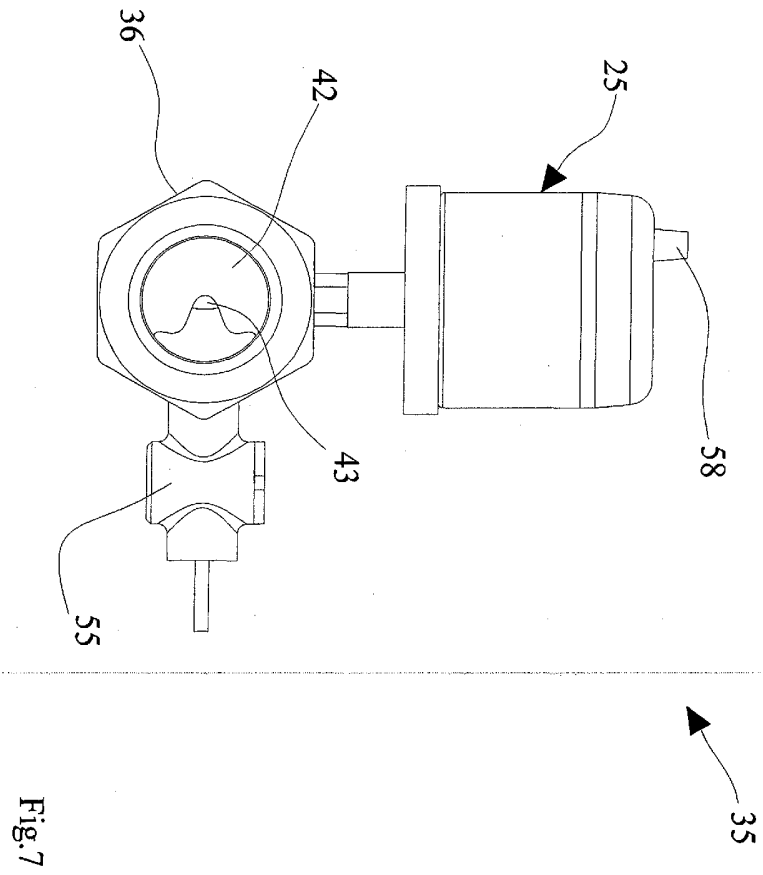


Fig. 7

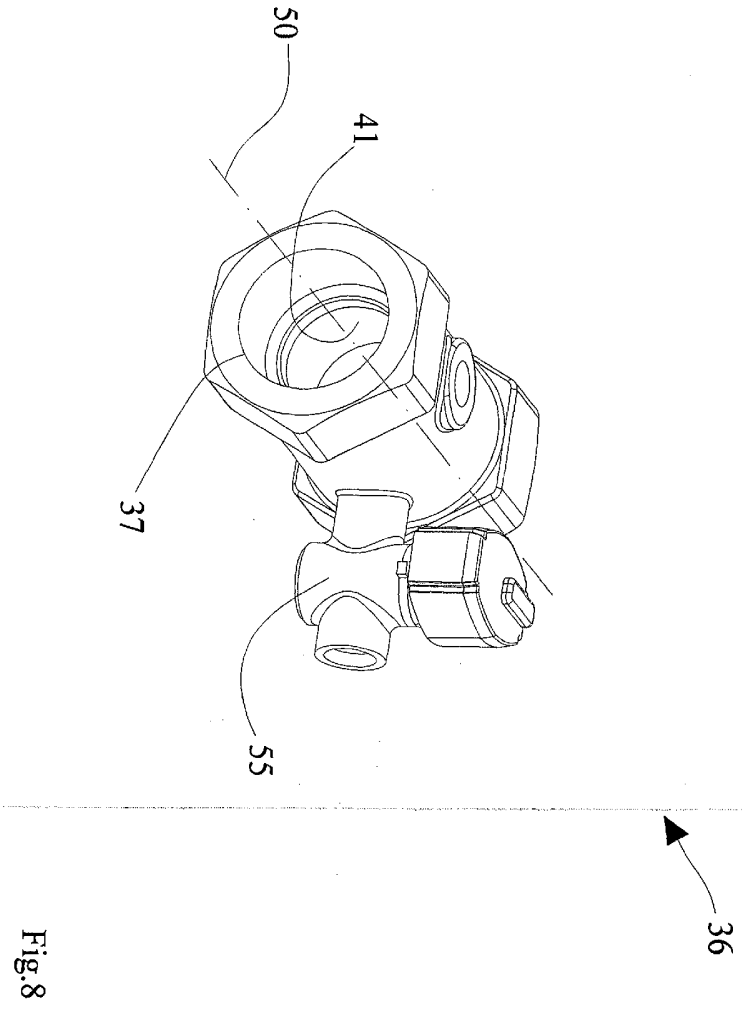


Fig. 8

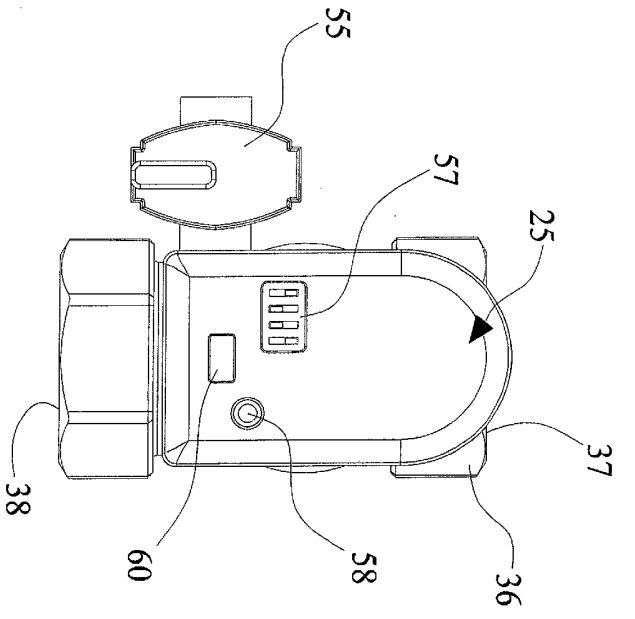


Fig.9

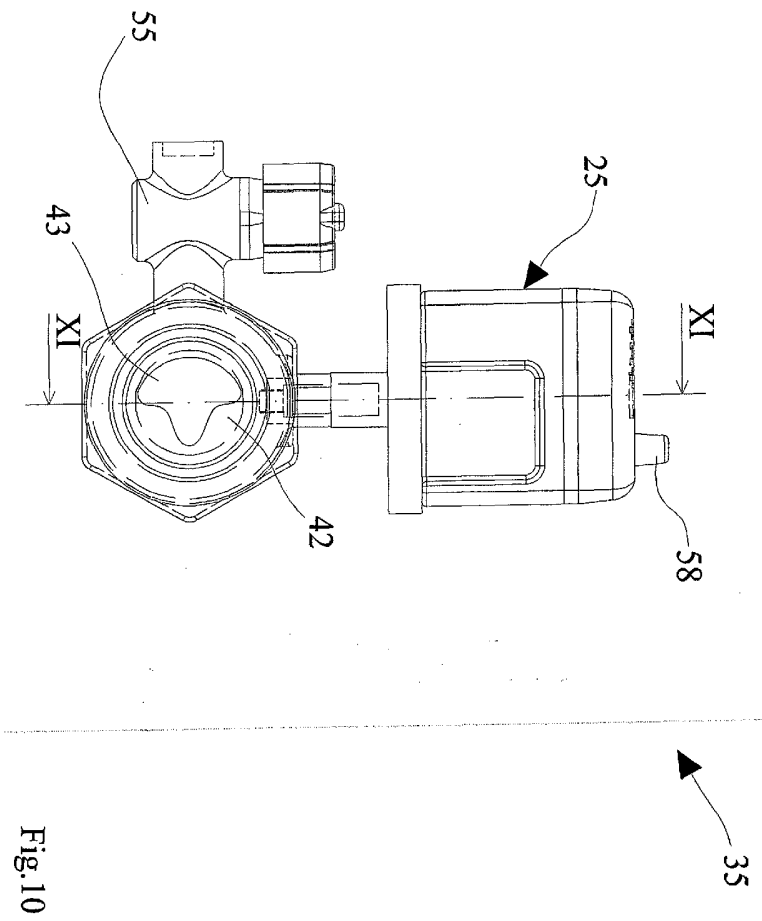
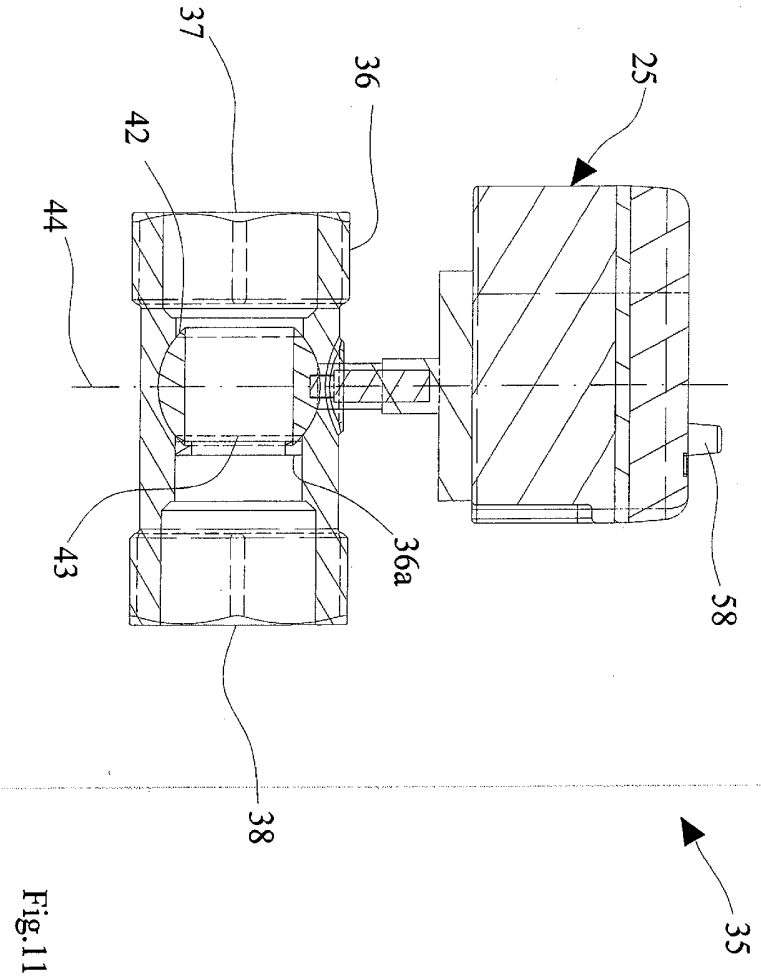


Fig. 10



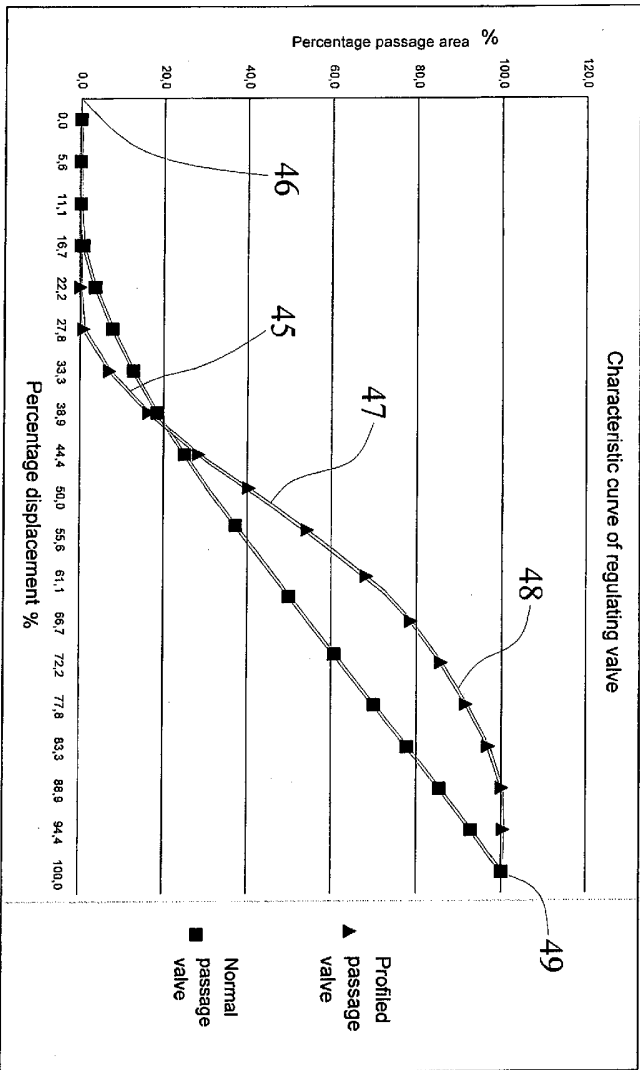


Fig.12