

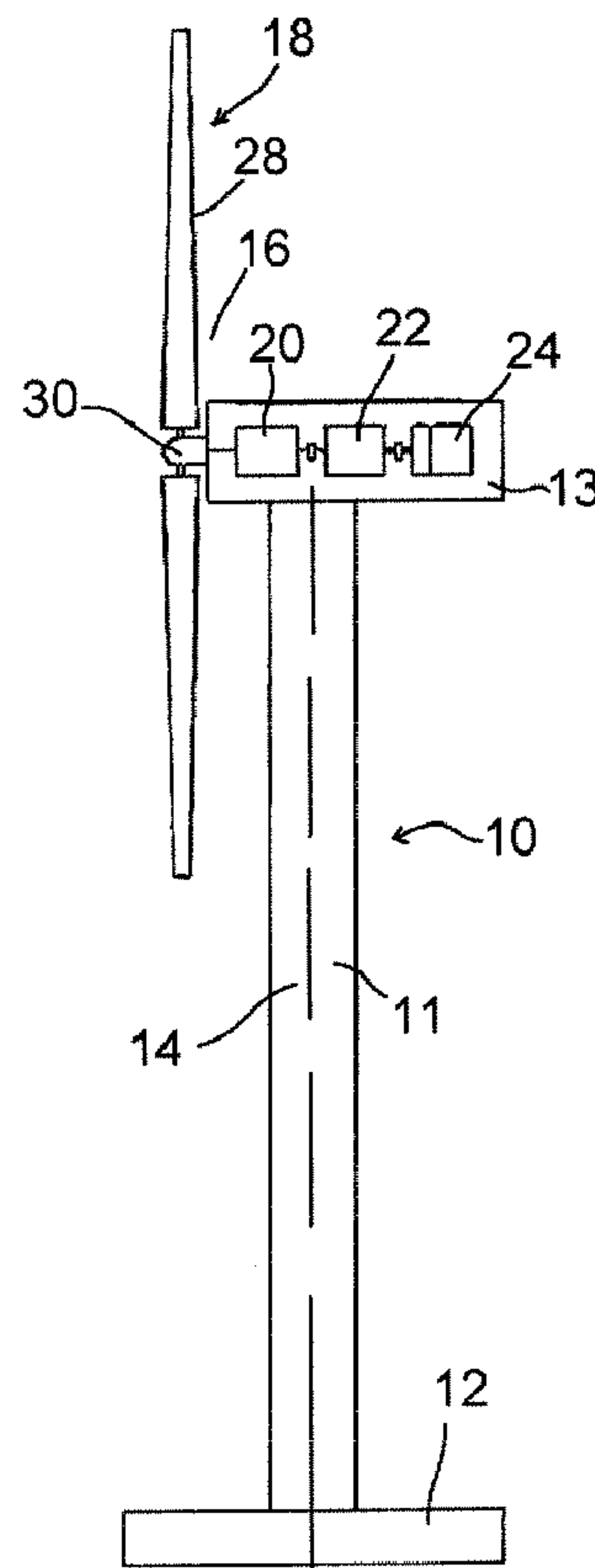


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(54) Title: HYBRID WIND TURBINE SYSTEM, APPARATUS AND METHOD

FIG. 1



(57) Abrégé/Abstract:

A wind turbine assembly and method capable of providing a total firm power output. There is a wind power section which delivers non-firm power from the wind turbine to a generator section. Then there is also an auxiliary power section which is capable of



(57) **Abrégé(suite)/Abstract(continued):**

providing firm power to the same generator section. This can operate in three operating modes, namely an only wind power mode, an only auxiliary power mode, and a combined wind power and auxiliary power mode.

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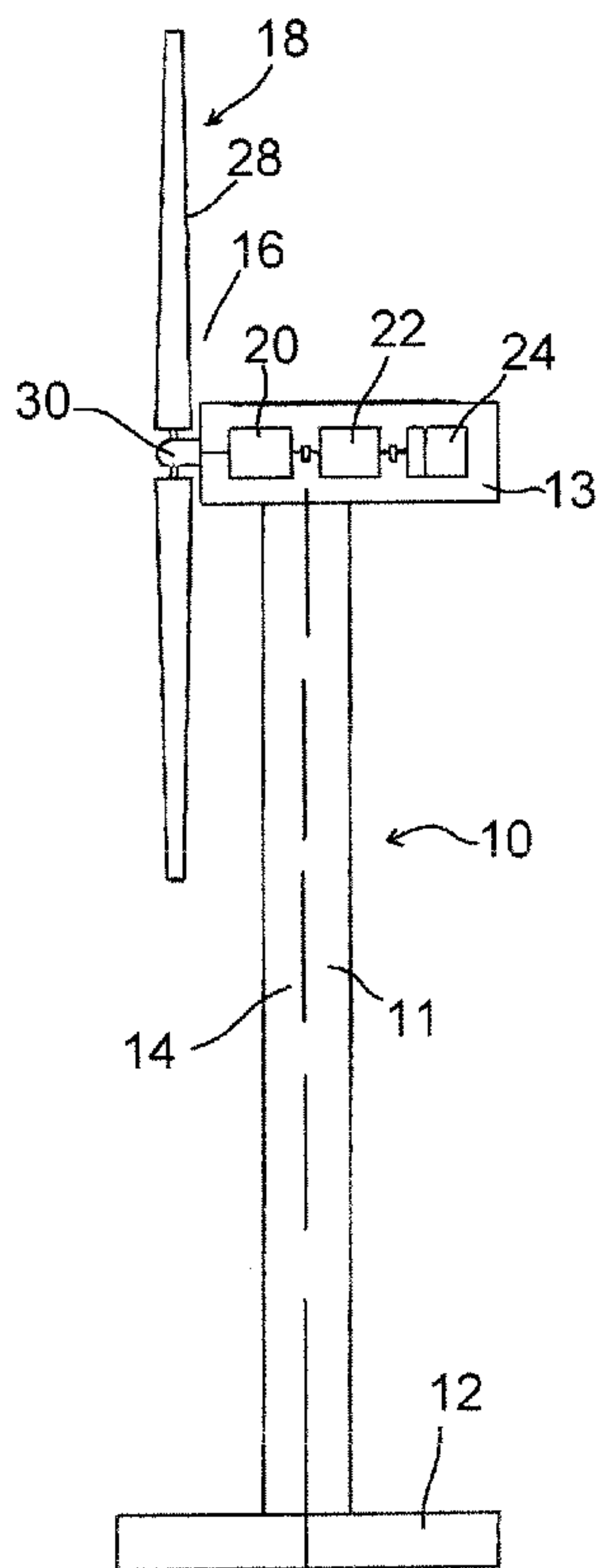
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FIG. 1



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HYBRID WIND TURBINE SYSTEM, APPARATUS AND METHOD

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RELATED APPLICATIONS

This application claims priority benefit of U.S. Serial Number 60/898,619, filed 01/30/2007.

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BACKGROUND OF THE DISCLOSURE

a) Field of the Disclosure

The present disclosure relates in general to wind turbine technology, and more particularly to a system combining the apparatus and method of the wind turbine with other energy sources,

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b) Background Information

While wind turbine power has many advantages as an additional and/or alternative source of energy, it does have the drawback that there are time intervals where it is not able to produce any power at all, or only a small amount of power. Thus, there have been various approaches to combine the wind power source with other independent power sources to be able to produce power more reliably, in the form of "firm power".

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A search of the patent literature has disclosed patents related to solving these problems, and these are summarized in the following text.

U.S. 4,204,126 (Diggs) discloses a "Guided Flow Wind Power Machine With Tubular Fans", which, when powered by the wind, can generate electricity. Also, when there is enough wind power it has the capability of also lifting "massive weights" hydraulically. Then when the wind has subsided, the weights can be permitted to be drop downwardly to supply energy to drive a generator. Figs. 4 and 5 show the weights 114 through 120 arranged in quadrants.

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US 5,740,677 (Vestesen) shows a system which is adapted for use at a location where there is a need for electricity and also fresh water. However, this residential community is also near a source of salt water. There is a wind diesel plant which supplies electricity for various uses and also operates a distillation unit to supply the fresh water. The wind/diesel plant comprises at least an internal combustion engine, a wind turbine, a distillation unit, a first closed fluid circuit containing heating and cooling devices, and a second open fluids circuit.

US 6,127,739 (Appa) issued October 3, 2000, and is the first of three patents which have the same inventor. In this patent, there is a forward front rotor 12 having blades that would cause rotation in one direction, then there is a rear rotor 21 (called a "leeward rotor 21") positioned behind the front rotor 12 and rotating in the opposite direction. This patent states that the various items added to this apparatus would produce a substantially higher "value of energy efficiency factor".

US 6,278,197 (Appa) is the second patent to the inventor and it discloses a wind turbine where there is a forward set of turbine blades which rotate in one direction, and a second set of turbine blades which are in the wake of the first set and which rotate in the opposite direction. The reason given for this is that there is still energy in the air that passes through the first set of turbine blades, and this is utilized in the second set of turbine blades.

US 6,492,743 B1 (Appa) is the third (and more recent) patent to Mr. Appa, and this also shows a basic configuration of wind turbine where there are forward and rear set of blades. There is a heat exchanger having a centrifugal fan to circulate ambient air to cool an alternator in the apparatus, and the hot air is directed to a combustion chamber by means of an air duct in the blades. Natural gas or liquid is also conveyed to the rotating frame. When wind speed is low, fuel will be injected into the combustion chamber and burned with a large mass of air. The hot gasses expand in an exit nozzle to provide thrust to assist wind power.

SUMMARY OF THE EMBODIMENTS

The embodiments of the invention comprise a wind turbine assembly and method capable of providing a total firm power output. There is a wind power
5 section which delivers non-firm power from the wind turbine to a generator section. Then there is also an auxiliary power section which is capable of providing firm power to the same generator section. This can operate in three operating modes, namely an only wind power mode, an only auxiliary power mode, and a combined wind power and auxiliary power mode.

BRIEF DESCRIPTION OF THE DRAWINGS

- Fig. 1 is an elevation view of a hybrid wind turbine system of a first embodiment of the present invention;
- 5 Fig. 2 is an isometric exploded view showing the components of a typical wind turbine apparatus, some or all of which can be combined with the wind turbine apparatus of the embodiments of the present invention;
- Fig. 3 is a somewhat schematic view of a power generating section 16 of the first embodiment;
- 10 Figs. 4, 4A and 4B are views similar to Fig. 3 showing a second embodiment illustrating alternate locations for the auxiliary drives;
- Figs. 5 and 5A are views similar to Figs. 1 and 3, showing a third embodiment which shows a heat recovery section in the support tower;
- Fig. 6 shows yet a fourth embodiment of the invention, where a steam generator and steam turbine are utilized as the auxiliary power source;
- 15 Figs. 7 and 7A are views that show a basic wind turbine system which also utilizes solar energy to add energy to the system;
- Fig. 8 is similar to Fig. 7 in that it shows a basic wind turbine system which also utilizes an engine auxiliary drive with heat recovery and an associated steam turbine auxiliary drive;
- 20 Fig. 9 shows a system for a solar thermal energy system which is independent of the wind turbine power generation system but occupies the same wind turbine structure;
- Fig. 10 is a combination of Figs. 7 and 8 in that it shows a basic wind turbine energy system which utilizes an engine auxiliary drive with heat recovery, an associated steam turbine auxiliary drive, and a solar thermal energy system which uses the same steam turbine; and
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Fig. 11 shows a basic wind turbine system with the addition of the nacelle substructure for housing additional heat recovery and power generation equipment.

DESCRIPTION OF THE EMBODIMENTS OF THE PRESENT INVENTION

It is believed that a clearer understanding of the present invention can be
5 obtained by first reviewing briefly the overall system of a first embodiment of the
present invention, as shown in Fig. 1. This is followed by a more detailed
description of a group of components shown in Fig. 2, which are typically found in
a prior art wind turbine apparatus, and some or all of which can be incorporated
in the embodiments of the present invention. This will then be followed by a
10 more detailed description of the embodiments of the present invention.

A. A General Description Of The First Embodiment.

To proceed now with the more general description of the first embodiment,
as indicated above, this will be done with reference to Fig. 1. There is a wind
15 turbine assembly 10 which comprises a base support section comprising a
vertically aligned tower 11 which is supported by a base 12. At the upper end of
the tower 11, there is a power generating main support structure 13 which is
rotatably mounted to the tower 11 to rotate about a vertical axis of rotation 14
centrally located in the tower 11. This support section 13 provides a support for
20 a power generating section 16 of the first embodiment, and it may be in the
configuration of a nacelle 13 commonly used with wind turbines.

The entire power generating section 16 comprises a blade section 18, a
rotary speed changing drive section 20, a generator section 22, and an auxiliary
power section 24. The blade section 18 comprises a plurality of turbine blades
25 28, and a hub or rotor 30 to which the blades 18 are connected.

The blade section 18 and the speed changing drive section 20 can be
grouped as the primary power generating portion while the auxiliary power
section 24 (as well as the auxiliary power or back up power components,

including those that are shown in other embodiments) can be considered as being in a secondary power generating portion.

The primary and secondary power generating portions together function in a manner to enable the generator 22 to provide firm power.

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B. Summary Of Some Prior Art Components.

With the overall description of this first embodiment being presented, attention is now directed to Fig. 2, which, as indicated above, is an exploded drawing of a number of components which in themselves exist in the prior art and are commonly used in present day wind turbines. In Fig. 2 only two of the three blades 28 are shown and the rotor 30 is not shown. There is a low speed shaft 32, which (as shown in Fig. 3) connects to a speed changing drive section 34 which is shown somewhat schematically, and (as its name implies) provided a power output at a higher RPM than that of the shaft 32.

15 This drive section 34 is commonly in the form of a gear section. In general, the rotational speed of the low speed shaft 32 would be between about 30 to 60 rotations per minute, and the gear section 34 is in turn connected to the generator 22 to cause it to rotate at a speed between about 1,200 to 3,600 RPM. This would typically be a rotational speed required by a large number of present day generators to produce electricity. The gear section 34 connects to a shaft 38 which is located in the generator 22.

20 There is provided an anemometer 40 which measures the wind speed, and also a wind vane 42 to ascertain wind direction. Both the wind speed and the wind direction data are transmitted to a controller 44. The controller 44, as its name implies, performs various control functions. For example, it controls a yaw drive 46 and its associated motor 48 to keep the blade section 18 facing into the wind as the wind direction changes, starts and stops the wind turbine, etc.. There is also provided a disc brake 49 for the low speed shaft 32, and in the prior

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art this can be applied mechanically, electrically, or hydraulically to stop the rotation of the rotary components in emergencies.

All, or most all, of the components which are shown in Fig. 2 are, or may be, present in the embodiments of the present invention. However, for
5 convenience of illustration (e.g., to avoid cluttering up the drawings), these are not shown in the following drawings (Figs. 3 - 11) which illustrate the nine embodiments of the present invention.

C. Description Of The First Embodiment Of The Present Invention.

10 All (or many) of the components of this first embodiment are shown in at least one of Figs. 1, 2 and 3. Reference is now made to Fig. 3. It will be noted that a number of the components which appear in Fig. 3 also appear in either Figs. 1 or 2. For clarification, those components in Fig. 3 which already appear in either or both of Figs. 1 and 2, are given like numerical designations, with an
15 "a" suffix distinguishing those particular components. Then the components which appear in Fig. 3 and which do not appear in either Figs. 1 or 2 will be given new numerical designations.

To proceed now with a description of the first embodiment of Fig. 3, as in Fig. 1, there are the blades 28a which are attached to the hub 30a. The hub 30a
20 in turn connects to and drives the low speed shaft 32a. The low speed shaft 32a in turn drives the speed changing drive section 20a which then provides a high rotational speed power output to the generator 22a.

The components of this first embodiment described in the paragraph immediately above, are already found in Fig. 1 or 2. In Fig. 3 there is also shown
25 an auxiliary drive section 24a capable of providing a drive output to the generator 22a.

For convenience of description, in describing the location of the components in Fig. 3, the hub 30a shall be considered to be a front or forward location, and the location of the auxiliary drive unit 24a shall be considered as

having a rear location. Also, the axis of rotation of the hub 30a, the blades 28a, and also of the low speed shaft 32a, and any other components which rotate on the same axis, shall be designated the "power generating axis of rotation 67".

To return now to the description of this first embodiment, the auxiliary
5 drive unit 24a provides a rotating drive output to a torque converter 66. The torque converter 66 in turn has a drive connection to an overrunning drive member 68 (which can be simply an overrunning drive clutch) that in turn connects to the rear end of the high speed shaft 38a of the generator 22a. Then the forward end of the shaft 38a of the generator 22a connects to a forward
10 overrunning drive member 69 that connects to the drive output of the speed changing drive section 34a. The torque converter 66 located between the auxiliary drive unit 24a and the generator 22a may or may not be required and depends on the design speed of the generator 22a and the auxiliary drive unit 24a. If the operating speed of the auxiliary drive unit 24a is a close match to the
15 generator 22a operating speed, the overrunning clutch 68 would provide an adequate method of coupling the generator 22a to the auxiliary drive unit 24a.

There are many types of conventional drives that could function as the auxiliary drive unit 24a. For example, this could include an internal combustion engine, external combustion engine, steam turbine, steam engine, or hybrid
20 drive. The most common types of drives would include, but not be limited to, gasoline engines, diesel engines, natural gas engines, gas turbine engines, steam turbines, steam engines, sterling engines, gas expanders, or hydraulic or electric motors with a nearby source of power or hydraulic energy. Sources of energy for the auxiliary drive unit 24a could include gasoline, diesel, jet fuel,
25 heavy oil, natural gas, propane, hydrogen, ethanol, coal, wood, or any other energy source suitable for the auxiliary drive or its cooperating equipment.

D. Operation Of The First Embodiment.

To describe now the operating features of this first embodiment, let us review three different situations, namely:

- i. the wind is at a sufficient velocity so that it is able to generate sufficient power to produce the desired power output of the generator 22a;
- ii. the wind velocity is not sufficient to drive the blade section at all, and the auxiliary drive section 24a is activated to generate the needed electric power; and
- iii. the wind velocity is such that it is able to rotate the blade section 18a to generate only an electrical power output which is below the desired output of the generator 22a, and to obtain the desired level of the total electrical power output, it is necessary to operate the auxiliary drive section 24a.

In the first situation (where there the wind power is at a sufficiently high level), the blade section 18a is rotated to drive the blades sections 18a at full power output or near full power output. More specifically, the blade section 18a is rotating with sufficient power output so that the speed augmenting drive section 20a is acting through an overrunning clutch 69 to drive the generator 22a at a sufficient power output so that sufficient electrical power is developed. The overrunning drive section 68, which connects to the shaft 38a of the generator 22a, simply overruns its connection to the auxiliary drive section 24a, thus, the auxiliary drive section 24a remains stationary.

Let us now take the second situation where there is either no wind or such a small velocity of the wind that the blade section 18a is put in a position where it is stationary or simply not rotating. In this situation, the auxiliary drive section 24a is activated manually or automatically so that its rotational output is directed through the torque converter 66, which in turn acts through the overrunning drive 68, which is caused to rotate in a direction so that it drives the generator 22a.

At the same time, the speed changing drive section 20a remains stationary, and since the connection between the drive section 20a and the generator 22a is the overrunning drive member 69, the generator 22a is able to operate to rotate in a manner so that it has no drive connection with the drive
5 section 20a and is driven totally by the auxiliary power unit 24a.

Let us now consider the third situation, which is that the wind generated power is great enough to achieve a useful lower power output level, but is not great enough to meet the desired power output. In this instance, the auxiliary drive section 24a would be utilized to cause rotation of its torque converter 66 to
10 act through its drive member 68 and provide power to the rear end of the shaft 38a of the generator 22a.

At the same time, the pitch of the blades 28a could be set at an angle of attack to optimize the power output that is developed by the use of both power sources. The effect of this is that the shaft 38a of the generator 22a would be
15 driven at both its front and rear end portions, so that there would be sufficient power to generate the desired electrical power output.

Also, in this third operating mode, the two overrunning drive members (drive clutches) 68 and 69 are operating in their engaged position, so that these are providing rotational forces to the generator 22a at a sufficiently high power
20 output.

E. Applications Of The First Embodiment.

Let us now turn our attention to some of the possible applications of the system of the first embodiment of the invention (i.e., the various ways it might be
25 used). As indicated earlier in this text, one of the drawbacks of a wind turbine is that it produces power intermittently. Thus, this puts wind power in the category of "non-firm energy producers". However, by combining the wind power turbine in the combination of this first embodiment, this now becomes a source of firm power that could supply energy to a power grid on a continuous basis.

Another situation of possible use is where there is a municipality which needs a reliable source of electricity. With the system of the first embodiment, the system could be engineered so that the auxiliary power source by itself could generate an adequate level of electric power. In that situation, the auxiliary power source would be able to operate as the sole power source in that time interval when the wind turbine power source would be idle. Then as the wind energy was available, the system could be operated in the mode mentioned above as Mode 1 or Mode 2 where the electric power output would be entirely from the wind turbine or as Mode 3, a dual drive mode, where the combined operation of both the wind turbine and the auxiliary power section are utilized to drive the generator 22a.

From the above comments, it becomes apparent that only the one generator 22a would be needed in each of the three modes. There are various expenses incurred in providing electric power through a generator, such as the cost of switchgear, transformers, etc.. With this arrangement of this embodiment, that extra expense is alleviated by utilizing the same generator for:

- i) the "only wind power mode";
- ii) the "sole auxiliary power mode"; and
- iii) the "combined wind power/auxiliary power mode".

It is to be understood that all of the components (or a large number of the components) that are shown in Fig. 2 could be utilized also in each of the several embodiments of the present invention.

i) Generator Types

To comment generally on the generator 22a, wind turbines are supplied with several different types of generators, including induction generators, double fed induction generators (for speed control), variable slip induction generators (for limited changes in speed), synchronous generators (directly and indirectly connected), and DC generators (typically small wind turbines). Most wind turbines in service are standard

induction generators which are constant speed machines. Variable speed generators, with the exception of DC generators, can be held at a fairly constant speed with the control system. This is a plus for the operation of the auxiliary drive in that the additional energy input to the generator does not change the generator speed appreciably. Additional torque input to the generator simply causes more power output from the generator. The DC generator is not considered an ideal candidate for the auxiliary drive as too much torque from the auxiliary drive could speed up the wind turbine to the point where the wind would not contribute to energy production.

ii) Auxiliary Drive Considerations

To comment generally about the different possibilities of the auxiliary drive 24a, it could be coupled directly to the generator via a torque converter or overrunning clutch or it can be connected through a gearbox, again using a torque converter or overrunning clutch. In most cases, an overrunning clutch will be sufficient; however, if there is a need to run the engine at constant speed and vary the output shaft speed to the generator, a torque converter can be used. If the wind turbine is at rest (zero speed) and the operator wishes to run the generator, he can start the auxiliary drive 24a. Because the generator is at rest, the overrunning clutch will engage the generator as soon as the auxiliary drive commences startup. The generator rotor will rotate along with the auxiliary drive shaft during startup and will continue to rotate at the same speed as the auxiliary drive at all times.

To connect the generator to the grid, the auxiliary drive must speed the generator rotor up to a speed that matches the generator rotating magnetic field. At that point the generator breaker can be closed to connect the generator to the grid. Any additional power input from the auxiliary drive to the generator will cause power to flow out of the

generator to the grid. An alternate method of starting up the generator would be to use the soft start feature supplied with most large scale wind turbines to connect them to the grid. In this case the wind must be used to rotate the propeller, gear, and generator to get it close to the normal
5 operating speed before closing the breaker. In some cases the soft start feature can be used to start the generator from dead stop. In this case, the generator acts as a motor until it gets up to speed at which time the wind energy input causes power to flow out from the generator.

If the auxiliary drive had a torque converter, the operator could start
10 the auxiliary drive and run it up to operating speed before engaging the torque converter to spin up the generator. With the torque converter the engine speed could be changed and the output shaft speed from the torque converter could be held at a constant speed or, conversely, the engine speed could be kept constant and the output speed could be
15 varied along with the generator speed.

The generator can be driven from the wind turbine end, the auxiliary drive end, or both ends at the same time. The generator will not know the difference. It only knows that torque is being applied to its rotor to generate electricity. It would be possible to use the auxiliary drive to
20 reduce the impact of wind gusts on the wind turbine. This could be done by applying a certain amount of power from the auxiliary drive which is over and above the power being supplied to the generator from the wind. In this case the wind turbine would not be supplying full rated power to the generator. When a wind gust hits the wind turbine and increases the
25 generator output and causes high loads on the gearbox, the auxiliary drive would receive a governor signal to reduce its power output so the generator and gear do not experience damaging load increases. Wind turbine manufacturers are constantly working on improvements to minimize the damaging effects of wind gusts and would welcome new

solutions to the problem. The current methods to control the effect of wind gusts are associated with the electrical control systems and generators. Variable slip generators are used to help solve the wind gust problem by allowing the generator to temporarily speed up (increased generator slip) to allow the additional wind energy to be converted into kinetic energy and not forcing the energy through the generator. It would be like installing a clutch between the wind turbine propeller shaft and gearbox to allow the clutch to slip during wind gusts to avoid damage to the gears.

iii) Structural Considerations

An additional benefit of the auxiliary drive arrangement is to change the center of gravity of the nacelle. The auxiliary drive acts as a counterweight on the opposite end of the nacelle as the propeller, hub, shaft and gearbox. Due to the extreme weight of those components, the wind turbine nacelle must be positioned to keep its center of gravity above the center of the tower. This means the propeller is positioned quite close to the tower which causes the propeller blades to bend every time they pass by the wind turbine support tower. The wind shadow and flexing of propeller blades has caused fatigue failures of blades in the past. The weight of the auxiliary drive on the opposite end of the nacelle would allow the nacelle to be repositioned so that the propeller blades are farther away from the tower and less susceptible to flexing and fatigue failures.

F. The Second Embodiment Of The Invention.

Reference will now be made to Figs. 4, 4A and 4B which are different configurations of this second embodiment.

The second embodiment is similar to the first embodiment except that some of the auxiliary drive components are placed in different relative positions, and an auxiliary drive speed changing section is added in Figs. 4A and 4B to allow the installation of two auxiliary drives. An example of the foregoing would

be the installation of a natural gas engine as an auxiliary drive and a steam turbine as the second auxiliary drive. The second auxiliary drive would be part of an energy recovery system that would recover waste heat from the first auxiliary drive and convert the waste heat into steam. The steam would then be used as
5 an energy input to the second auxiliary drive. Another example of a use for a second auxiliary drive would be a solar/wind hybrid wind turbine shown in Fig. 7 where steam generated in the solar collector is routed to the second auxiliary drive (steam turbine) to provide additional power to the generator.

Components of this second embodiment which are the same as, or similar
10 to, components shown in Figs. 1, 2 and 3, will be given like designations with a "b" suffix distinguishing those of the second embodiment, and the newly mentioned components are given new numerical designations. Further, to distinguish between the three different versions, in the version of Fig. 4, a suffix of "b-1" will distinguish those of the version of Fig. 4, a suffix of "b-2" will
15 distinguish those of the second version of 4A, and the suffix of "b-3" will distinguish those of the third version of Fig. 4B.

All three of these versions of the second embodiment have the following components, the blades 28b, the hub 30b, the low speed shaft 32b, the speed changing section 20b, the generator section 22b, and the auxiliary drive section
20 24b. In Fig. 4, all of these components are arranged in substantially the same way as corresponding components in Fig. 3, except that there are two auxiliary drive sections 24b-1 and 25b-1. Other specific features are the same, such as having the torque converter and overrunning clutches located in substantially the same manner as in the first embodiment of Fig. 3.

25 The first version of the second embodiment of Fig. 4 differs from the first embodiment in that in addition to the auxiliary drive 24b-1 there is a second auxiliary drive 25b-1 which connects to the speed changing drive section 20b-1. Power from the auxiliary drive 25b-1 is transmitted through the speed changing drive section 20b-1 to the generator 22b-1 for additional power output. In Fig. 4

an overrunning clutch 27b-1 must be installed to de-couple the wind turbine shaft 32b-1 from the speed changing drive 20b-1 when there is insufficient wind to rotate wind turbine shaft 32b-1.

5 In Fig. 4A the second version of Fig. 4, we have the same components of the wind turbine blades 28b-2, the hub 30b-2, the low speed shaft 32b-2, the speed changing section 20b-2, the generator section 22b-2, and the auxiliary power section 24b-2. Fig. 4A differs from Fig. 4 in that there is provided a second speed changing section 26b-2 which has an operative connection to the auxiliary drive section 24b-2. Then there is a second auxiliary drive section 25b-10 2 which also has an operative connection through the second speed changing section 26b-2. The second auxiliary drive section 25b-2 provides additional power to generator 22b-2 as available from energy recovery systems or energy generation systems other than wind, which are part of the hybrid wind turbine system.

15 Fig. 4B has substantially the same components as in Fig. 4A, except that in addition to the auxiliary drive section 24b-3 transmitting power through the second speed changing section 26b-3, the second auxiliary drive section 25b-3 is located on the same side of the second speed changing section 26b-3. In other respects, it functions the same way as the second version of Fig. 4A.

20

G. A Third Embodiment Of The Invention.

A third embodiment of the present invention will now be described with reference to Figs. 5 and 5A. Components of this third embodiment which are the same as, or similar to, components of the earlier embodiments will be given like numerical designations, with a "c" suffix distinguishing those of the third
25 embodiment.

In this third embodiment, the basic system as shown in Fig. 3 is used, so that the main components and their functions of this third embodiment are substantially the same as in the third embodiment as they are in the first

embodiment. However, the added feature is that the auxiliary engine drive is combined with two stages of an organic rankine cycle heat recovery system to increase the overall efficiency of the engine drive.

In this embodiment the two stages of heat recovery 50c and 51c are
5 located in the wind turbine support tower 11c.

With this system, the heat recovery process captures waste heat from the auxiliary engine 24c exhaust and the auxiliary engine 24c coolant. Also, the waste heat is converted into useful electricity using a separate turbine and generator which is part of the heat recovery system located in the tower 11c.

10 In Fig. 5A the hot exhaust from auxiliary drive engine 24c flows to an organic rankine cycle boiler 52c to vaporize the organic working fluid. The cooled exhaust then flows to an emission control unit 53c before being discharged to atmosphere. The rankine cycle involves a boiler feed pump 54c which pumps the organic working fluid to the boiler 52c for vaporization. The
15 vapor then flows to the expansion turbine 55c which is coupled to a generator 56c. Power from the generator 56c is connected to the wind turbine electrical switchgear. The vapor then flows out of the expansion turbine to the air cooled condenser 57c where it is condensed back into a liquid. The liquid working fluid then flows back to the boiler feed pump 54c for recirculation.

20 In this embodiment the auxiliary drive engine coolant is routed from auxiliary drive engine 24c to an organic rankine cycle boiler 58c to vaporize the organic working fluid. The cooled engine coolant is then pumped back to engine 24c using coolant circulation pump 59c. The rankine cycle involves a boiler feed pump 60c which pumps the organic working fluid to the boiler 58c for
25 vaporization. The vapor then flows to the expansion turbine 61c which is coupled to a generator 62c. Power from the generator 62c is connected to the wind turbine electrical switchgear. The vapor then flows out of the expansion turbine 61c to the air cooled condenser 62c where it is condensed back into a liquid.

The liquid working fluid then flows back to the boiler feed pump 60c for recirculation.

With the conversion of waste energy into additional electricity, the auxiliary drive 24c-1 is a very efficient source of additional power for the hybrid wind turbine.

H. A Fourth Embodiment Of The Invention.

Fig. 6 shows a fourth embodiment of the present invention. Components of this fourth embodiment which are the same as, or similar to, components of the earlier embodiment will be given like numerical designations, with a "d" suffix distinguishing those of the fourth embodiment. This fourth embodiment has the same basic operating components as shown in the first embodiment, except that in this fourth embodiment, the auxiliary drive section 24d is steam powered. Further, the steam that is generated to supply the power is generated by a boiler that is located in the support tower 11d. The fuel can be solid fuel, liquid fuel, gaseous fuel, or other fuels.

As shown in Fig. 6, there is the support structure 13d mounted to the tower 11d, the blade section 18d, a speed changing drive section 20d, and a generator 22d. There are also the two overrunning drive members 68d and 69d on opposite sides of the generator 22d.

There is a solids fuel hopper 90, which directs the solid fuel 92 into a furnace area 94, where there is a forced draft generated by the fan 96. Further, there is a liquid and/or natural gas burner 98, a steam drum 100, a mud drum 102, a boiler flue gas discharge 104, and a bag house 106. There is a steam conduit 108 leading to a steam drive turbine 110. The steam drive turbine 110 is positioned to supply power to the generator 22d. The steam exhaust from the steam turbine 110 flows along a conduit 112 to an air cooled surface condenser 114 and is cooled by a fan 116. The condensate then flows to the feed water pump 105 and back to the boiler steam drum 102.

I. The Fifth Embodiment Of The Invention.

A fifth embodiment of the present invention will now be described with reference to Fig. 7. Components of this fifth embodiment which are the same as, or similar to, components of any of the earlier embodiments will be given like numerical designations, with an "e" suffix distinguishing those of this fifth embodiment.

In this fifth embodiment, there is a solar thermal power source in addition to the wind turbine power and also the auxiliary power section. In this case, there would be three sources of power to drive the generator, namely:

- i) wind;
- ii) solar generated power; and
- iii) the auxiliary drive section which, as indicated previously in this text, could be fueled by a wide variety of energy sources, such as an engine driven by diesel fuel, natural gas, ethanol, etc..

The wind and solar energy inputs would produce non-firm energy that cannot be depended upon as a constant source of power. However, the auxiliary drive 24e (engine or turbine) would be the ultimate backup for firm power generation. Thus, with these three options offered with the wind turbine, the customer could purchase a basic wind turbine, a wind turbine with a solar thermal energy drive, a wind turbine with an engine or turbine (steam, gas turbine, etc.) auxiliary drive, or a wind turbine with both a solar thermal energy drive and an engine or turbine drive. Thus, different sources of energy input to the wind turbine are not mutually exclusive and can cooperate to maximize the output of the wind turbine. With that background information having been given, Fig. 7 shows the basic components that are shown in the first embodiment of Fig. 3, and the components discussed above with reference to Fig. 7.

Thus, there is the source of firm power in the form of an auxiliary drive engine 24e or other power source (see Fig. 7A).

In this embodiment, in Fig. 7 there is the tower 11e which supports the rotatably mounted support structure 13e, the speed changing drive section 20e, and the generator 22e. Fig. 7A shows the engine auxiliary drive 24e-1 and the auxiliary drive 25e-1 in the form of a steam turbine. There is a condenser 152e
5 which directs the condensate from the boiler feed pump 148e-1.

To provide the solar energy, Fig. 7A shows there is a plurality of heliostats 130e which reflect the sun rays in a converging pattern to a solar absorber 132e that is mounted in the tower 11e. Fig. 7A shows there is a boiler feed pump 134e which pumps water or other liquid up through the solar absorber 132e to a steam
10 drum 136e-1 so that the steam can be separated from the steam and water mixture generated in the solar absorber 132e-1. The steam or other gaseous drive medium then travels upwardly to a steam turbine 125e-1. The steam turbine auxiliary drive 25e-1 provides a rotary power output to the generator 22e-1 in combination with the engine auxiliary drive power output 24e-1, or through
15 another operative connection to the generator 22e-1.

In operation, either or both of the non firm power sources (i.e., the wind power source and the solar power source) are utilized to provide the energy output to rotate the generator 22e-1. In the event that either or both of the wind power and solar power are absent because of the surrounding weather
20 environment, and are producing no usable power, or only a smaller output of power, then the auxiliary power source 24e-1 can be used to supplement the power input to an adequate level. However, if the solar power source and/or the wind power source are adequate, then the auxiliary power section 24e-1 will not be required.

25

J. The Sixth Embodiment Of The Invention.

A sixth embodiment of the present invention will now be described with reference to Fig. 8. Components of this sixth embodiment which are the same as, or similar to, components of any of the earlier embodiments will be given like

numerical designations with an "f" suffix distinguishing those of this sixth embodiment.

In this sixth embodiment, there is an addition of a steam rankine cycle heat recovery system to recover heat from the engine auxiliary drive exhaust. To
5 describe this sixth embodiment, reference is made to Fig. 8.

Hot engine exhaust leaving the auxiliary drive exhaust flows to a heat recovery steam generator 144f where the heat in the exhaust generates steam. The cooled exhaust then flows to the emission control unit 146f for treatment before it is discharged to atmosphere.

10 A boiler feed water pump 148f pumps water to the heat recovery steam generator 144f to raise steam. The steam and water mixture flows to a steam drum 149f, which is part of the heat recovery steam generator 144f, to allow the steam to separate from the mixture and flow to a steam turbine auxiliary drive
15 25f. This steam turbine 25f converts the steam energy into mechanical work by turning the turbine wheel and driving the auxiliary speed changing drive section 26f and the generator 22f through overrunning clutches 68f and 69f.

After giving up a portion of its energy to the steam turbine 25f, the steam flows to an air cooled condenser 152f where it is condensed back into water. The steam condensate then flows through a vacuum deaerator 154f for oxygen
20 removal before flowing to the boiler feed water pump 148f which pumps the feed water back to the heat recovery steam generator 144f to generate more steam.

The addition of the heat recovery system to the engine auxiliary drive increases the overall thermal efficiency of the engine auxiliary drive. Several types of steam drivers can be used to drive the generator. An example of an
25 alternate type of steam drive would be a rotary screw steam drive machine.

K. A Seventh Embodiment Of The Invention.

A seventh embodiment of the present invention will now be described with reference to Fig. 9. Components of this seventh embodiment which are the

same as, or similar to, components of earlier embodiments will be given like numerical designations, with a "g" suffix distinguishing those of this seventh embodiment.

This seventh embodiment comprises a solar thermal energy system which
5 combines the benefits of wind power with solar power using the same turbine structure.

In this embodiment, the entire solar thermal energy system is separated from the wind turbine power generation system. The solar thermal system uses an organic ranking cycle heat recovery system to convert solar energy into
10 electricity. Fig. 9 shows the process flow for the solar thermal system. The system components can be located in the support tower or the nacelle substructure of the ninth embodiment.

As shown in Fig. 9 there is a solar energy input to a solar absorber 190g which provides heat to a high temperature heat transfer fluid which is pumped
15 through the absorber 190g using circulating pump 189g. The heat transfer fluid then passes through a heat exchanger 192g where it vaporizes the organic rankine cycle working fluid. The cooled heat transfer fluid then flows back to the circulation pump 189g where it is pumped back to the solar absorber 190g. The vaporized organic fluid flows out of the heat exchanger 192g and into the
20 expander turbine 193g. The expander turbine 193g is coupled to a generator 194g which produces electric power. The vaporized working fluid, usually propane or butane, passes through the expander turbine to a condenser 195g where it is condensed back to a liquid. The liquid then flows to a pump 196g which pumps the working fluid back to the exchanger 192g for conversion back
25 into a vapor. An expansion tank 197g is provided to allow for the expansion of the heat transfer fluid in the solar thermal system.

In addition to the cost savings of combining the wind and solar energy systems in one structure, the solar addition to the wind turbine has the added

benefit of providing additional power output during the daylight hours when it is needed most.

L. An Eighth Embodiment Of The Invention.

5 An eighth embodiment of the present invention will now be described with reference to Fig. 10. Components of this eighth embodiment which are the same as, or similar to, components of earlier embodiments will be given like numerical designations, with an "h" suffix distinguishing those of this eighth embodiment.

10 This eighth embodiment comprises a solar thermal energy system and an engine system with heat recovery which combines the benefits of wind power, solar power, and engine power using the same wind turbine support structure.

Fig. 10 shows the process flow for the combined solar thermal system and engine system with heat recovery. In the engine plus heat recovery system, hot engine exhaust leaving the engine 24h flows to a heat recovery steam generator 144h where the heat in the exhaust generates steam. The cooled exhaust then flows to the emission control unit 146h for treatment before it is discharged to the atmosphere. A boiler feed water pump 148h pumps water to the heat recovery steam generator 144h to raise steam. The steam and water mixture flows to a steam drum 149h, which is part of the heat recovery steam generator 144h, to allow the steam to separate from the mixture and flow to the steam turbine auxiliary drive 25h. The steam turbine 25h converts the steam energy into mechanical work by turning the turbine wheel and driving the gear 26h and generator 22h through overrunning clutches. After giving up a portion of its energy to the steam turbine 25h, the steam flows to an air cooled condenser 152h where it is condensed back into water. The steam condensate then flows through a vacuum deaerator 154h for oxygen removal before flowing to the boiler feed water pump 148h which pumps the feed water back to the heat recovery steam generator 144h to generate more steam.

In the solar thermal system, the solar energy input to a solar absorber 132h is converted into steam which drives a steam turbine 25h which is coupled to the wind turbine main generator 22h. The steam then exits the steam turbine 25h and flows to an air cooled condenser 152h where the steam is condensed back into water. The water then flows through a vacuum deaerator 154h to remove oxygen and then to the feed water circulating pump 148h where it is pumped back to the solar absorber to generate more steam.

M. A Ninth Embodiment Of The Invention.

This ninth embodiment of the present invention will now be described with reference to Fig. 11. Some of the components in this ninth embodiment which are substantially the same as, or similar to, corresponding components of earlier embodiments will be given like numerical designations, with an "i" distinguishing those of this ninth embodiment. Accordingly, there are the propeller blades 28i along with the hub 30i. There is also the speed changing power section 20i, the generator 22i, and auxiliary speed changing section 26i, an auxiliary drive section 24i, and a second auxiliary drive power 25i, which in this instance is in the form of a steam driven turbine.

This ninth embodiment differs from the earlier embodiments in that the support structure (i.e., the nacelle 13i) has a nacelle substructure 14i to provide additional working areas for various purposes, such as to house heat recovery equipment associated with, for example, an auxiliary steam turbine drive.

The existing technology utilizes space in the wind turbine support tower and nacelle to house all equipment necessary to operate a wind turbine. At times it can be a challenge to install all equipment in the allowable space in a cost efficient manner and there is very little room for any extra equipment. Because the nacelle rotates to keep the wind turbine blades facing the wind, any equipment located in the support tower which must cooperate with equipment in the nacelle must address the problem of rotation. This means the design must

incorporate flexible joints, cables, hoses and other interconnections that allow the necessary rotation. By installing a nacelle substructure below the nacelle and on the downwind side of the support tower, it is possible to provide a large amount of space to mount equipment which rotates with the nacelle. Thus, the problem of interfacing equipment that does not rotate with equipment that does rotate is eliminated.

Another advantage of nacelle substructures is that it can be shop fabricated and lifted by crane to attach to the underside of the nacelle. Because the nacelle substructure is designed with a width that is no wider than the support tower, there are no detrimental effects to efficient air flow across the tower which would have a negative impact on the wind turbine output. To the contrary, the shape of the nacelle substructure enclosure will act like a tail behind the tower to assist in yaw control.

This ninth embodiment can be advantageous to any of the options described in earlier embodiments, including a standard wind turbine without any of these options. The substructure could be used with a standard wind turbine to house the electrical gear or other equipment located in the tower to achieve a cost savings during manufacturing. Due to the extremely tall support towers, it is possible to design the height of the substructure such that it extends down the tower as required to house all equipment intended to be located in the tower.

Although an auxiliary engine drive 24i and steam turbine drive 25i are shown coupled to the auxiliary gear 26i, various other configurations shown in other options are equally suited to cooperate with the nacelle substructure. For example, in Fig. 11, there is shown in the upper part of the substructure a heat recovery steam generator or organic rankine cycle heat recovery equipment, generally designated 260i. Then below this there are air cooled condensers indicated at 262i. Below the nacelle substructure 14i there is a support structure 272i which provides support for the substructure, and which could also provide support for at least part of the nacelle 13i itself. This support structure 272i

comprises a pair of circumferential rings 274i which are connected to the tower 11i, and there are roller bearings 268i which are rotatably mounted for circular movement on the rings 274i around the tower 11i. Then the support structure 272i of the substructure, such as indicated at 278i, is supported by these bearing
 5 rings.

The nacelle substructure has the same width as the tower. Thus, the substructure can be extended further down the tower to accommodate additional equipment. There are various options which include the following:

- i) an engine only configuration in the nacelle;
- 10 ii) /HRSG/Steam Turbine/Air Cooled Condenser;
- iii) engine/orc heat recovery/air cooled condenser;
- iv) solar steam generator/steam turbine/air cooled condenser;
- v) engine/HRSG/solar steam generator/steam turbine/air cooled condenser;
- 15 vi) solar thermal heat absorber (heat transfer fluid)/orc heat recovery/air cooled condenser;
- vii) engine/solar thermal heat absorber (heat transfer fluid)/orc heat recovery/air cooler.

The nacelle substructure 141i is attached to the underside of the nacelle
 20 such that it rotates with the nacelle. Various pieces of equipment 260i can be located within the substructure on various levels. Examples are heat recovery equipment 260i, air cooled condensers 262i and cooling fans 264i. As indicated above, the structural supports 274i for the nacelle substructure are supported by the tower using metal support rings 270i enable the nacelle with roll around the
 25 support rings 270i when the nacelle rotates to face the wind. Solar thermal absorbers 280i are located on the support tower itself.

Obviously, the vertical dimension of the nacelle substructure could vary substantially. In the representation of the sub-nacelle in Fig. 1, its depth dimension (indicated at "b" in Fig. 11), is about 40% of the horizontal length

dimension (indicated at "a" in Fig. 11), extending from the forward working end of the nacelle to the rear working end. Obviously, this vertical dimension "b" could be increased or decreased substantially, depending upon various factors. For example, this 40% dimension could be decreased down to about 30%, 20%,
5 15%, or 10%, or even as low as about 5%. Also, it could be greatly increased to values of, for example, 50%, 75%, 100%, 150%, 200%^, 250%, 300%, 400%, 500%, or even possibly higher.

The nacelle substructure is an elegantly simple method of providing large amounts of space for equipment which rotates along with the nacelle and thereby
10 eliminating the problem of interfacing rotating and non-rotating equipment. The additional weight will also be a counterweight to the wind turbine blades and will allow them to be located further from the tower, thus, reducing the blade flex when the blades pass by the tower.

To summarize at least some of the features of the present invention, the
15 embodiments of the present invention provide the following advantages:

- i) The auxiliary drive system will allow a wind turbine to generate firm power rather than non-firm energy.
- ii) The hybrid wind turbine which incorporates the solar thermal heat recovery system into the wind turbine allows the same generator,
20 switchgear, support tower, real estate, and transmission lines to be used by both the wind turbine and solar thermal power generator.
- iii) The nacelle sub-structure provides additional space to install equipment that must move with the equipment in the nacelle such as heat recovery steam generators, air coolers, organic rankine cycle heat
25 recovery systems and electrical gear.
- iv) The nacelle sub-structure enclosure will act as a tail fin on the wind turbine to assist with yaw control.

- v) The nacelle sub-structure module can be constructed in a shop with ideal working conditions thus improving worker productivity and reducing construction costs.
- 5 vi) The equipment located in the nacelle sub-structure can be installed in an upright position and remain in an upright position throughout the construction process. Equipment located in the support tower must be turned on its side at some point during the shop fabrication, shipment, or construction process.
- 10 vii) The energy conversion efficiency in BTU/KWH of the hybrid wind turbine which uses wind, and/or solar and/or thermal energy inputs in one combined system is very efficient when compared with the heat rate in BTU/KWH of a thermal energy conversion system alone due to the non-thermal energy inputs from the wind and solar systems.
- 15 viii) All components of the hybrid wind turbine can be procured and constructed using commercially available equipment and commercially available engineering and construction practices.
- ix) The nacelle sub-structure provides an alternate escape route for operations personnel in the event of a fire in the nacelle.

While the present invention is illustrated by description of several
20 embodiments and while the illustrative embodiments are described in detail, it is not the intention of the applicants to restrict or in any way limit the scope of the appended claims to such detail. Additional advantages and modifications within the scope of the appended claims will readily appear to those skilled in the art. The invention in its broader aspects is therefore not limited to the specific details,
25 representative apparatus and methods, and illustrative examples shown and described. Accordingly, departures may be made from such details without departing from the spirit or scope of applicants' general concept.

THEREFORE I CLAIM

1. A wind turbine assembly capable of providing a total firm power output, said assembly comprising;
 - 5 a. a combined wind turbine support structure comprising a tower and a power generating support structure that is rotatably mounted to the tower;
 - b. a generator section positioned in or at the power generating support structure;
 - c. a wind power section mounted to the power generating support structure and comprising at least a rotary mounted blade section to provide a non-firm wind power output;
 - 10 d. an auxiliary power section which is to provide an auxiliary firm power output for said generator section ;
 - e. said assembly being provided with power and/or other energy transfer connections to accomplish rotary power, transfer and/or other energy transfer between one or more of said generator section, wind power section, and said auxiliary power section;
 - 15 f. said wind turbine assembly and being arranged to be able to provide firm power at a desired power level to the generator section by operating in at least three different modes of operation, namely;
 - 20 i. an "only wind power mode" for a situation where there is sufficient wind velocity so that the blade section alone is able to generate sufficient non-firm power, and the auxiliary power section can remain dormant;
 - ii. an "only auxiliary power mode" where the wind velocity is sufficiently low so that it is not able to drive the blade section, or at least so slowly as to be little value, and the auxiliary power section can be activated to
- 25

deliver sufficient firm power to the generator section to maintain the power output that is needed; and

- 5 iii. a “combined wind power/auxiliary power mode” where the velocity of the wind is such that the rotation of the blade section is able to provide only enough power to meet only partially the level of power needed, and the auxiliary power section is operated at a level to deliver to the generator section sufficient power to meet the level of power needed.
- 10 2. The assembly as recited in claim 1, wherein there is a primary drive section comprising said wind power section and a power transfer section to transmit rotary power from the wind power section to the generator section.
3. The assembly as recited in claim 1, wherein said primary drive section comprises a speed changing drive section to increase the rotational speed transmitted to the generator section.
- 15 4. The assembly as recited in claim 2, wherein said generator section comprises a generator having first and second generator drive input connections, with the first generator drive input connection having a first operative connection to the primary drive section and the second generator drive input connection has a second operative connection to the auxiliary power section.
- 20 5. The assembly as recited in claim 4, wherein there is a first overrunning drive connection between said primary power section and the generator drive input section and there is a second overrunning drive connection which is located between the generator section and the auxiliary drive section whereby said generator is able to be driven from the auxiliary power section without being in a drive mode with said primary drive section, and the generator could be
- 25 driven by the auxiliary power section while not being in a drive mode with the primary drive section.

6. The assembly as recited in claim 1, wherein said auxiliary power section comprises a torque converter to provide a variable speed power input to said generator section.
7. The assembly as recited in claim 2, wherein said primary drive connection
5 also receives a power input from at least one power source from said auxiliary power section to transmit power from said at least one auxiliary power source to said generator section.
8. The assembly as recited in claim 2, wherein said auxiliary power section
10 comprises one or more of an internal combustion engine, an external combustion engine, a steam turbine, a steam engine, a hybrid drive, a gasoline engine, a diesel engine, a natural gas engine, a gas turbine engine, a sterling engine, a gas expander, a hydraulic motor, and an electric motor, and a power source for the auxiliary power section could be one or more of
15 hydraulic energy, gasoline, diesel fuel, jet fuel, heavy oil, natural gas, propane, hydrogen, ethanol, coal, wood, and any other energy source suitable for the auxiliary power section.
9. The assembly as recited in claim 1, wherein said auxiliary power section
20 comprises a fuel burning engine having an engine exhaust and a liquid engine coolant which are directed to heat recovery sections to extract waste energy and convert the waste energy to useful power.
10. The assembly as recited in claim 9, wherein the heat recovery from the exhaust and the engine coolant is accomplished in rankine cycle sections with cooled exhaust being discharged and the coolant being returned to the engine.
- 25 11. The assembly as recited as claim 1, further comprising a steam generating section located in or adjacent to said tower and a steam turbine comprising at

least in part the auxiliary power section, with the steam directed to the steam turbine to provide a rotating power output to said generator section.

12. The assembly as recited in claim 1, comprising a solar absorbent section, a pump and conduit section being arranged to direct a liquid medium through the solar absorbent section to heat the liquid medium to provide a gaseous medium portion, and a liquid medium portion, and direct the liquid medium portion back to the solar absorber to absorb further solar heat and continue to provide the liquid form of the liquid and gaseous form of the liquid medium, said assembly further comprising a gaseous liquid turbine to function as at least a part of a non-firm auxiliary power section, whereby there are three sources of power to drive the generator, namely;
- iv. i. wind;
 - v. ii. solar generated power; and
 - vi. iii. a portion of the auxiliary drive section which would be fueled by a firm source of energy.
13. The assembly as recited in claim 12, wherein said liquid medium comprises water.
14. The assembly as recited in claim 1, wherein said power generating support structure has a vertical axis of rotation, a front to rear front support structure portion located between the vertical axis of rotation and the location of the blade section of the assembly and a rear support structure portion which is positioned between the vertical axis of rotation and an end portion of the support structure that is located, with respect to the axis of rotation, at a location diametrically opposite from the forward end section, said assembly comprising a substructure which is located beneath the rear support structure portion so as to be vertically aligned with said rear support structure portion and connected to said support structure so as to remain positioned below the rear support structure portion as the support structure rotates to various

orientations so that the blade section is facing the wind, said substructure having a containing region where various equipment can be stored.

5 15. The assembly as recited in claim 14, wherein said substructure has sidewall portions relative to a front to rear alignment axis, with said sidewalls having a sidewall location which is no wider than a width location of the tower so as to remain shielded from the air stream flowing rearwardly and around the tower.

10 16. The assembly as recited in claim 14, wherein said substructure has a vertical dimension of the containing region which is at least as great as about twenty percent of a horizontal length dimension of a containing working region of the substructure.

17. The assembly as recited in claim 14, wherein said substructure is supported at least in part by a support structure which is mounted to the tower so as to be able to rotate around the tower with the power generating support structure.

15 18. A method of providing and operating a wind turbine assembly to provide a firm power output, said assembly comprising;

20 a. providing a combined wind turbine support structure comprising a tower and a power generating support structure that is rotatably mounted to the tower;

b. positioning a generator section to be positioned in or at the power generating support structure;

25 c. mounting a wind power section with a blade section to the power generating support structure and positioning these in a wind location to provide a non-firm wind power output to the generator section;

d. operating an auxiliary power section to provide an auxiliary firm power output for said generator section ;

- e. continuing to operate said wind turbine assembly in selected modes to provide firm power at a desired power level to the generator in at least three different modes of operation, namely;
- 5 i. an “only wind power mode” for a situation where there is sufficient wind velocity so that the blade section alone is able to generate sufficient non-firm power, and the auxiliary power section can remain dormant;
 - 10 ii. an “only auxiliary power mode” where the wind velocity is sufficiently low so that it is not able to drive the blade section, or at least so slowly as to be little value, and the auxiliary power section can be activated to deliver sufficient firm power to the generator section to maintain the power output that is needed; and
 - 15 iii. a “combined wind power/auxiliary power mode” where the velocity of the wind is such that the rotation of the blade section is able to provide only enough power to meet only partially the level of power needed, and the auxiliary power section is operated at a level to deliver to the generator section sufficient power to meet the level of power needed.
19. The method as recited in claim 18, further comprising providing a solar absorbent section and directing a liquid mediums through the solar absorbent section to heat the liquid medium to provide a liquid medium portion and a
- 20 gaseous liquid medium portion and directing the gaseous medium to drive a power turbine to provide power to the generator, and to also provide a firm power source to said generator section so that there are three sources of power to drive the generator, namely;
- 25 i. wind;
 - ii. solar generated power; and
 - iii. a portion of the auxiliary drive section which would be fueled by a firm source of energy.

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20. The method as recited in claim 18, further comprising providing a steam generating section and directing steam to an auxiliary power section to drive a steam turbine to provide power for the generator.

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FIG. 1

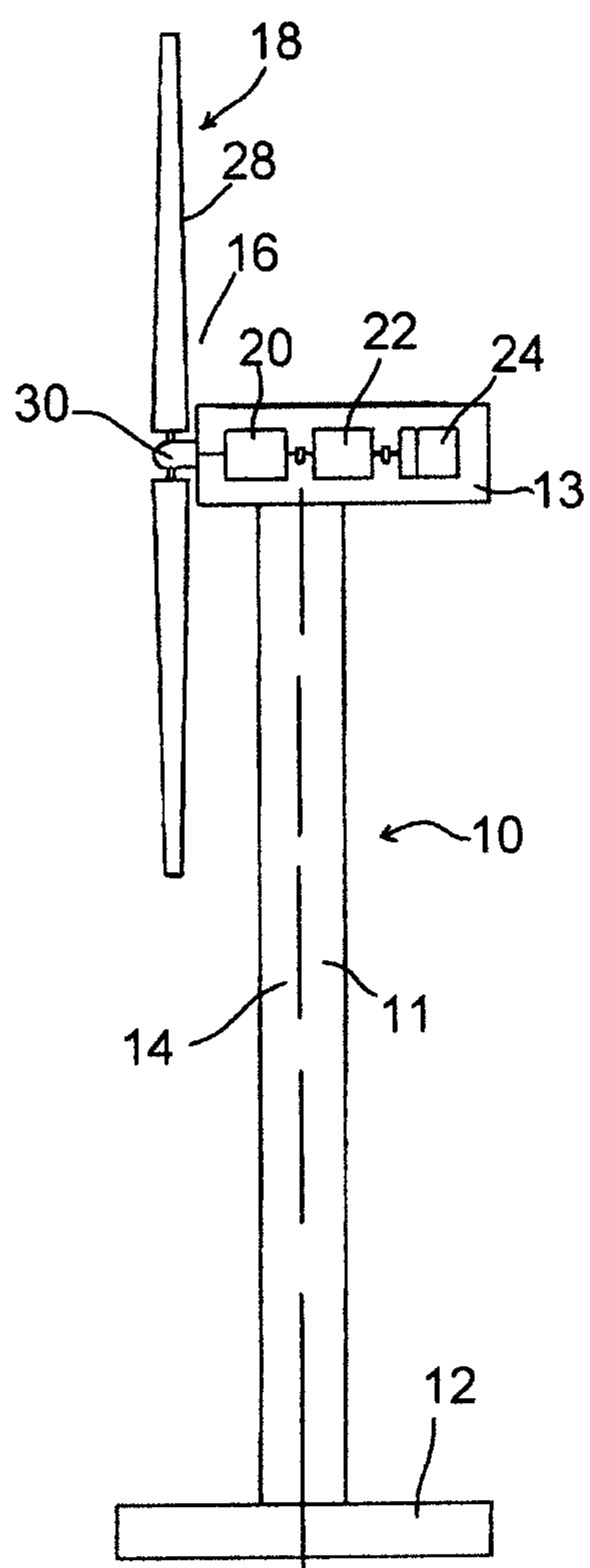
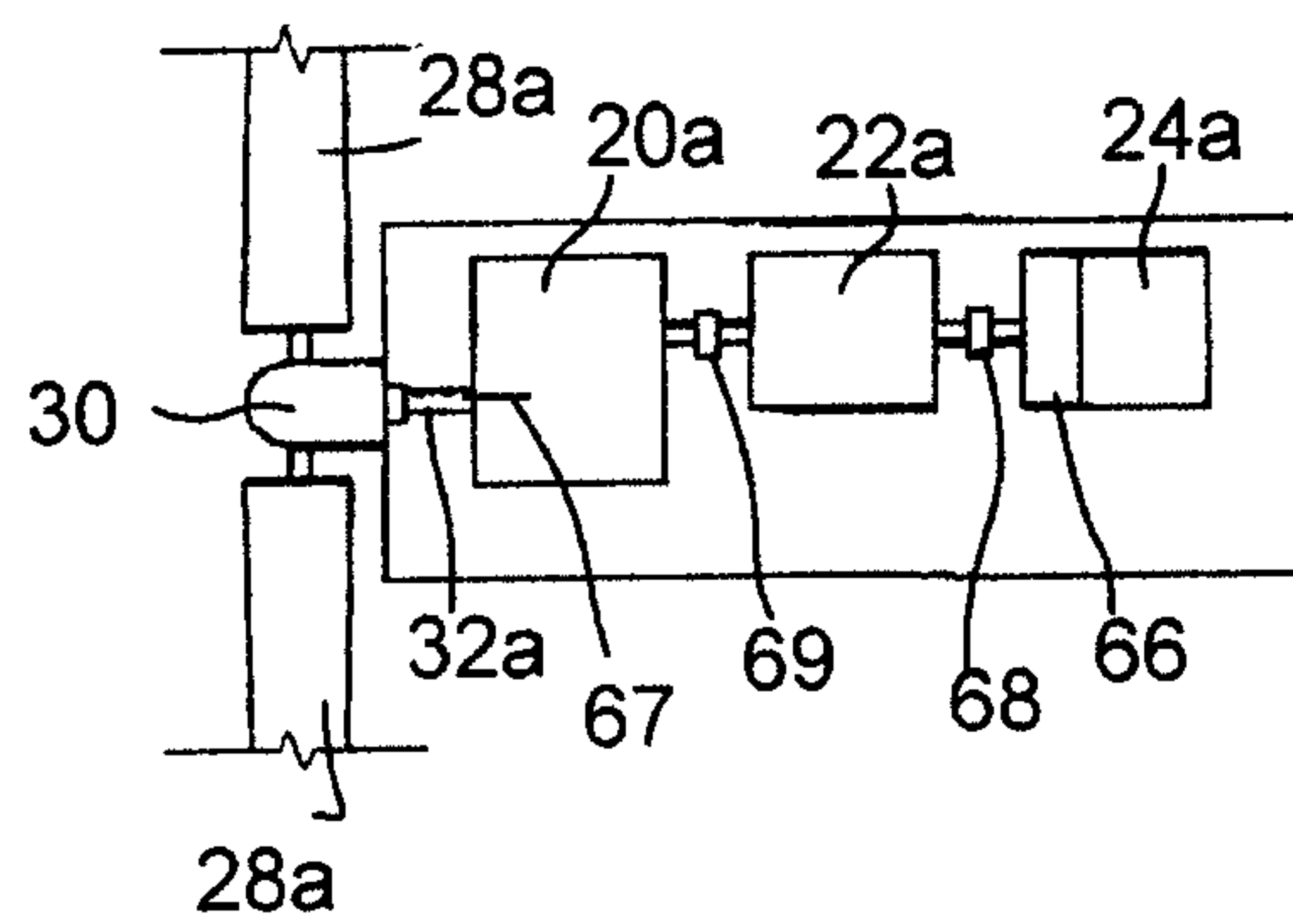
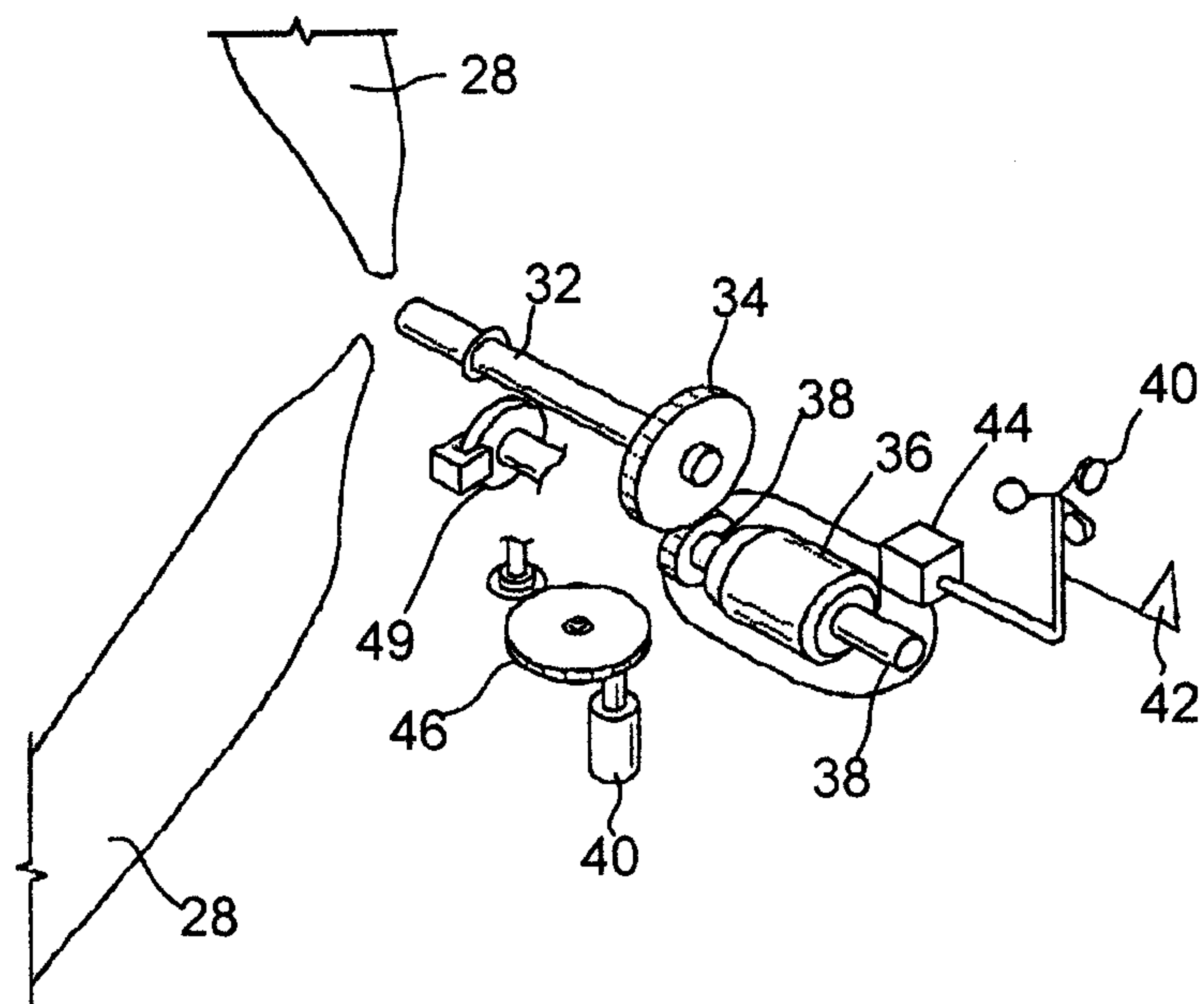


FIG. 3



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FIG. 2



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FIG. 4

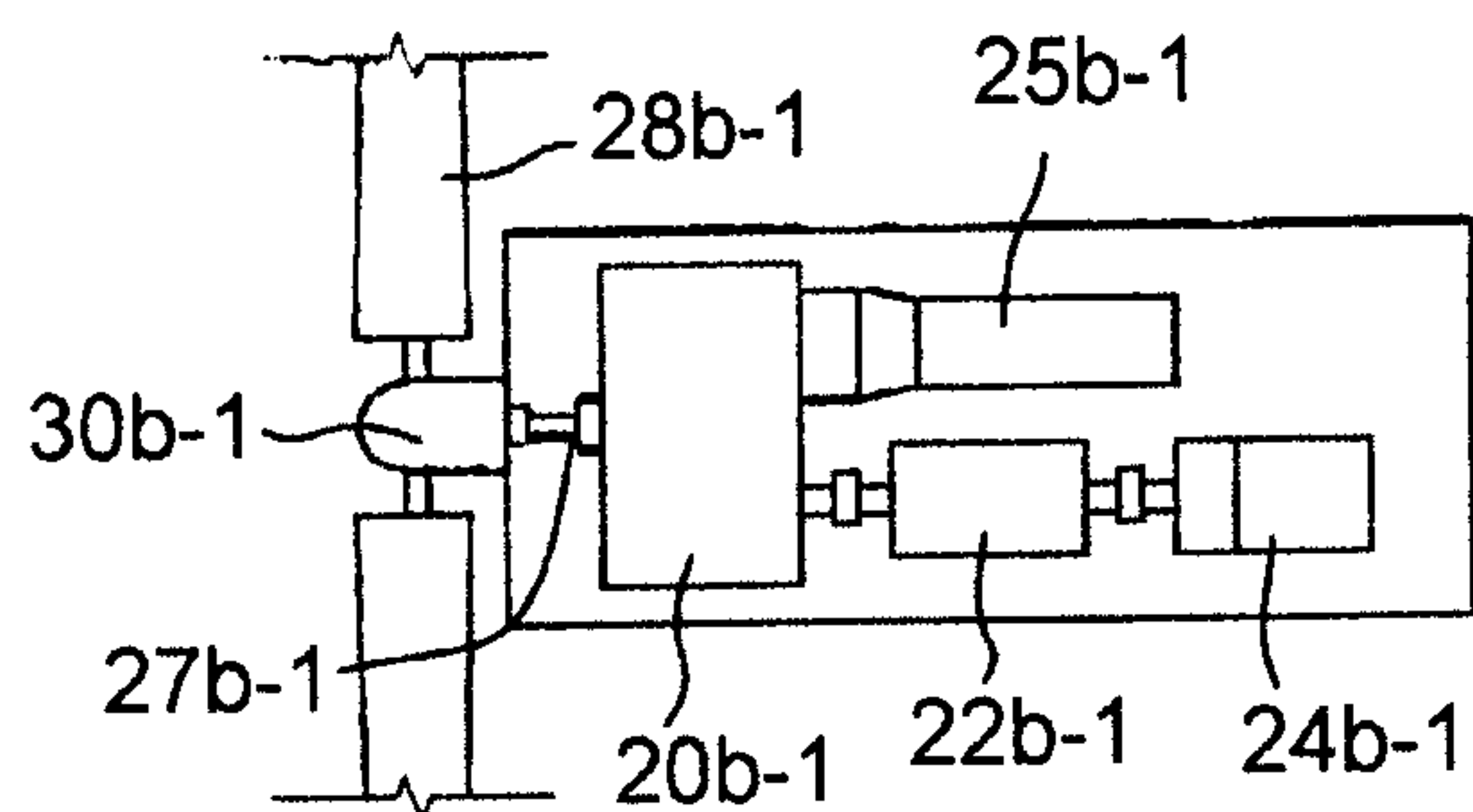


FIG. 4A

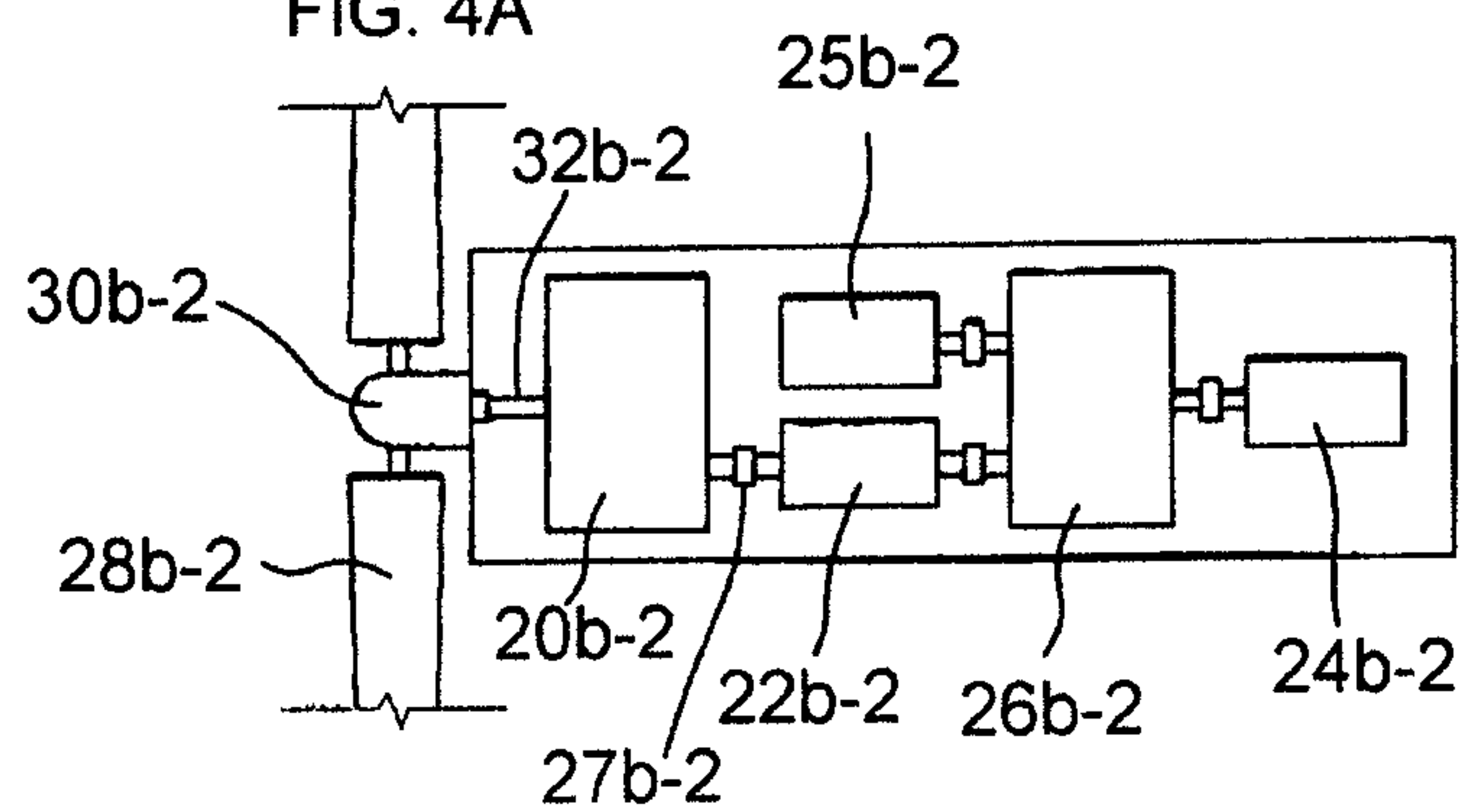
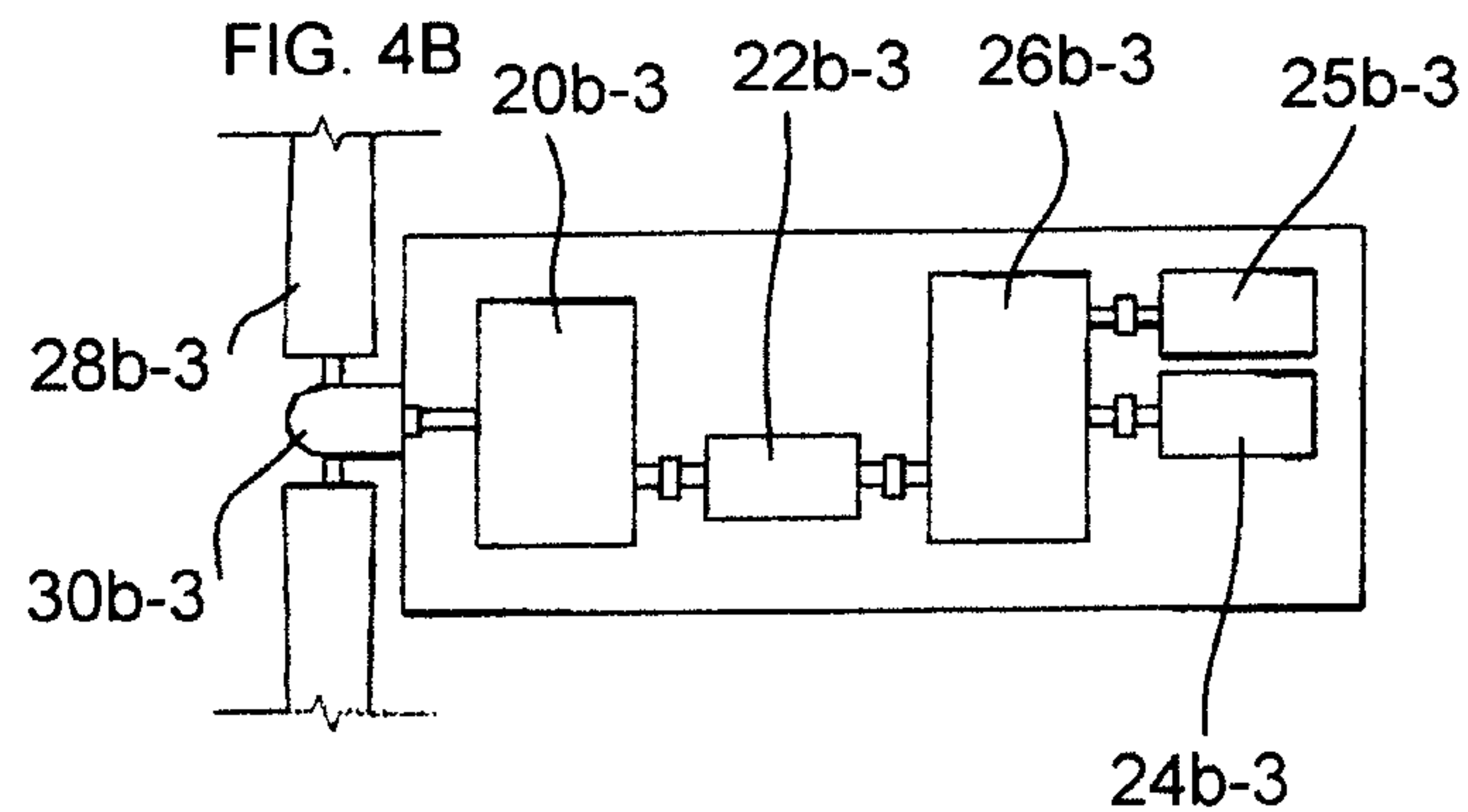


FIG. 4B



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FIG. 5

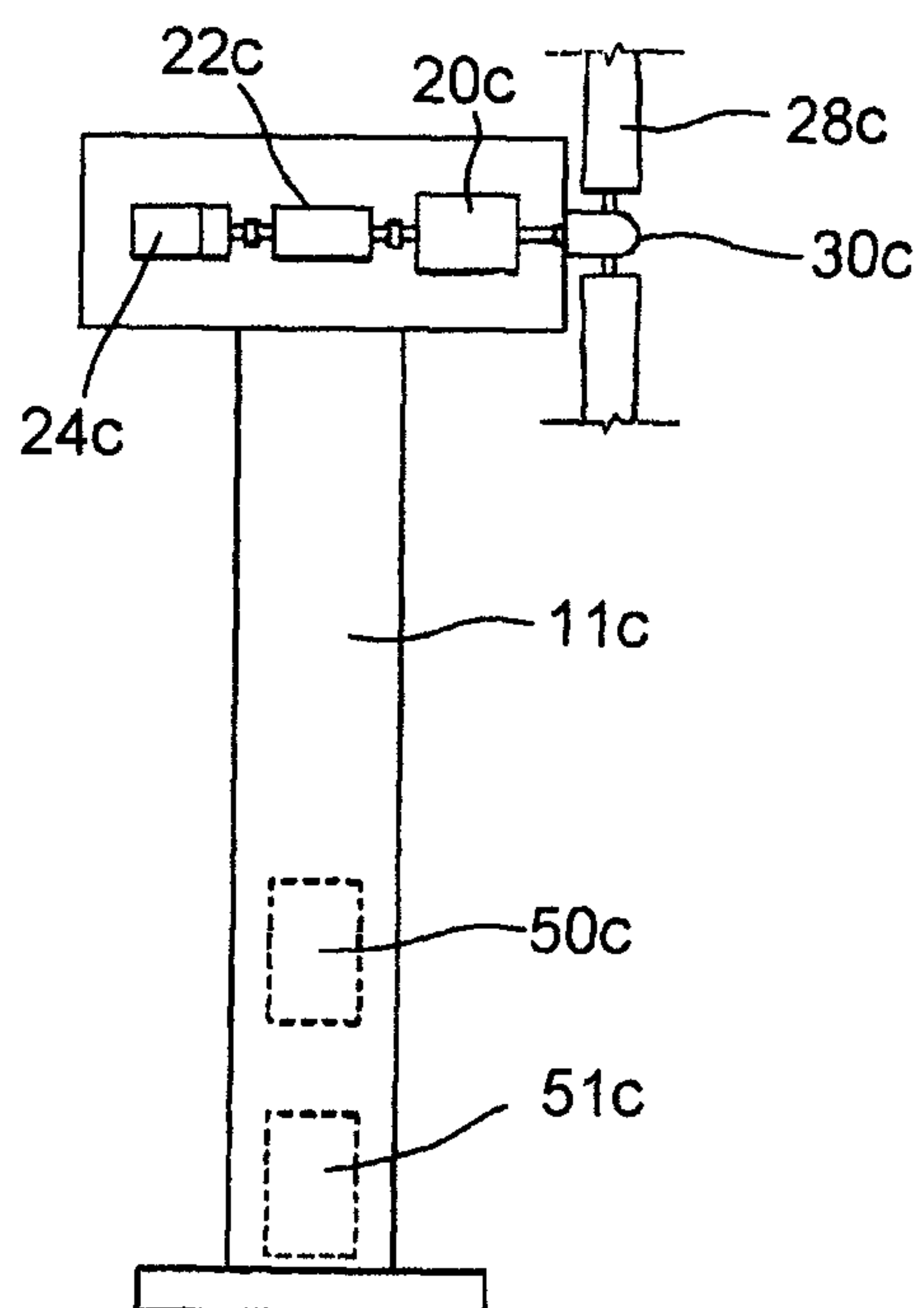
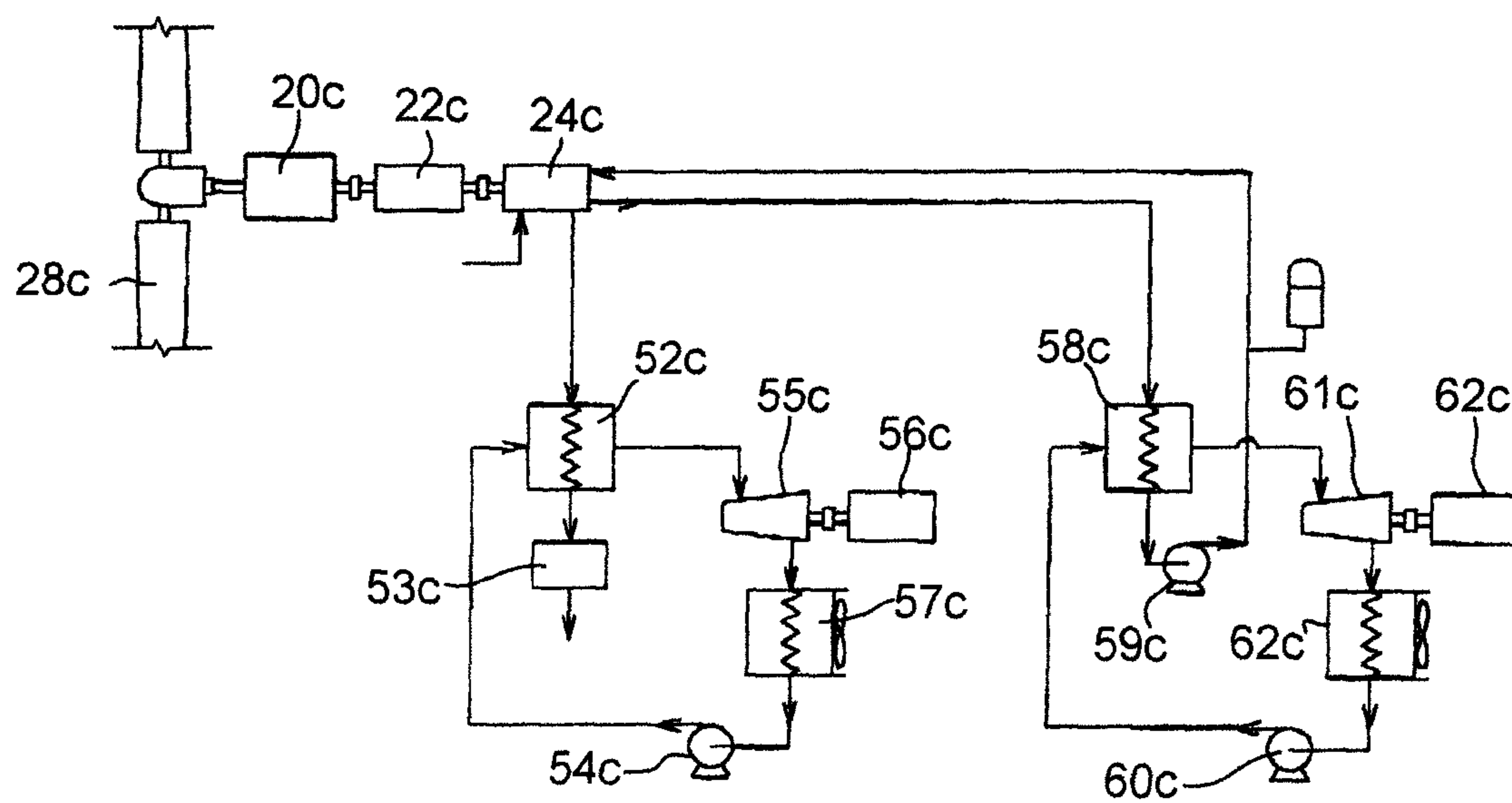
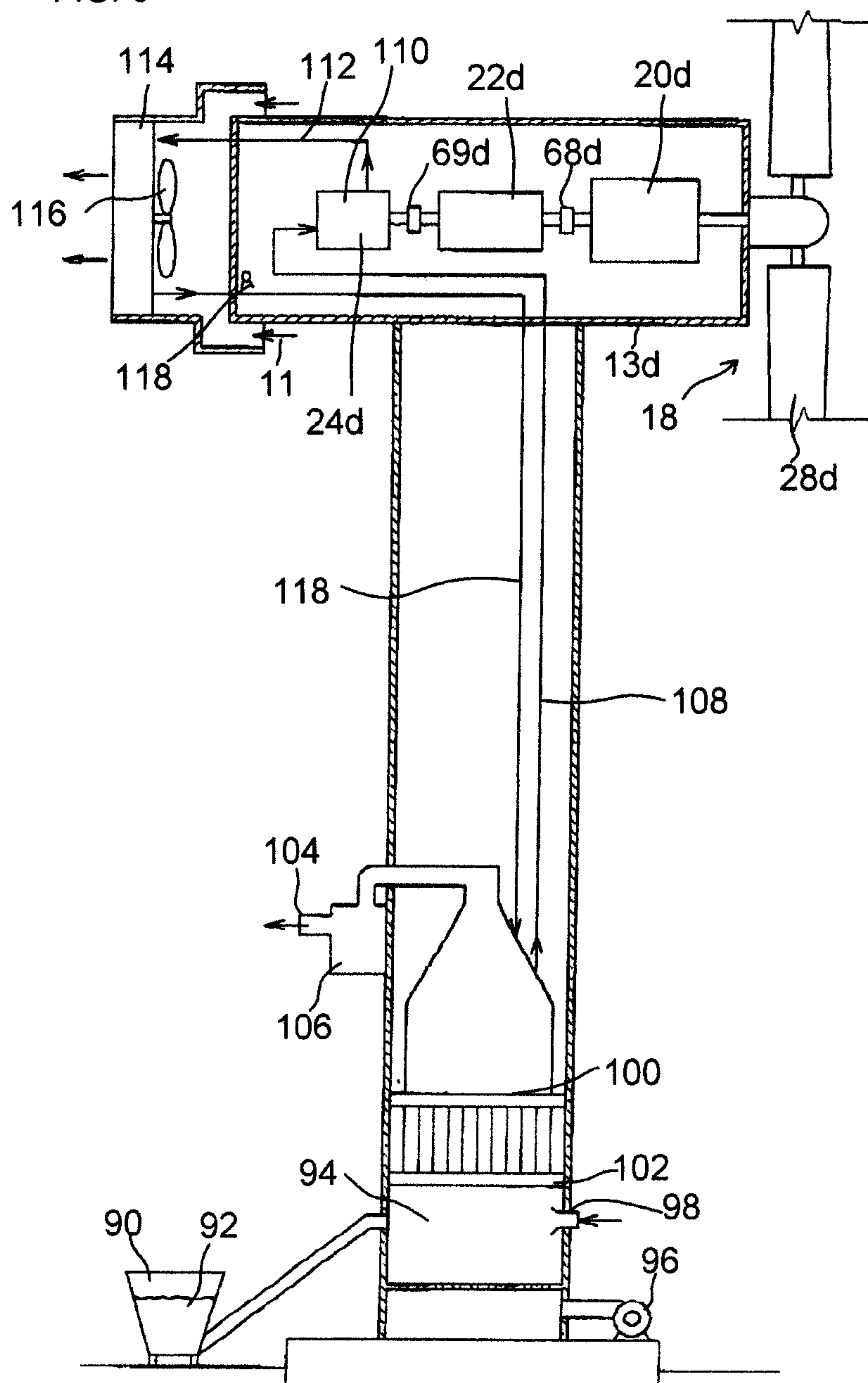


FIG. 5A



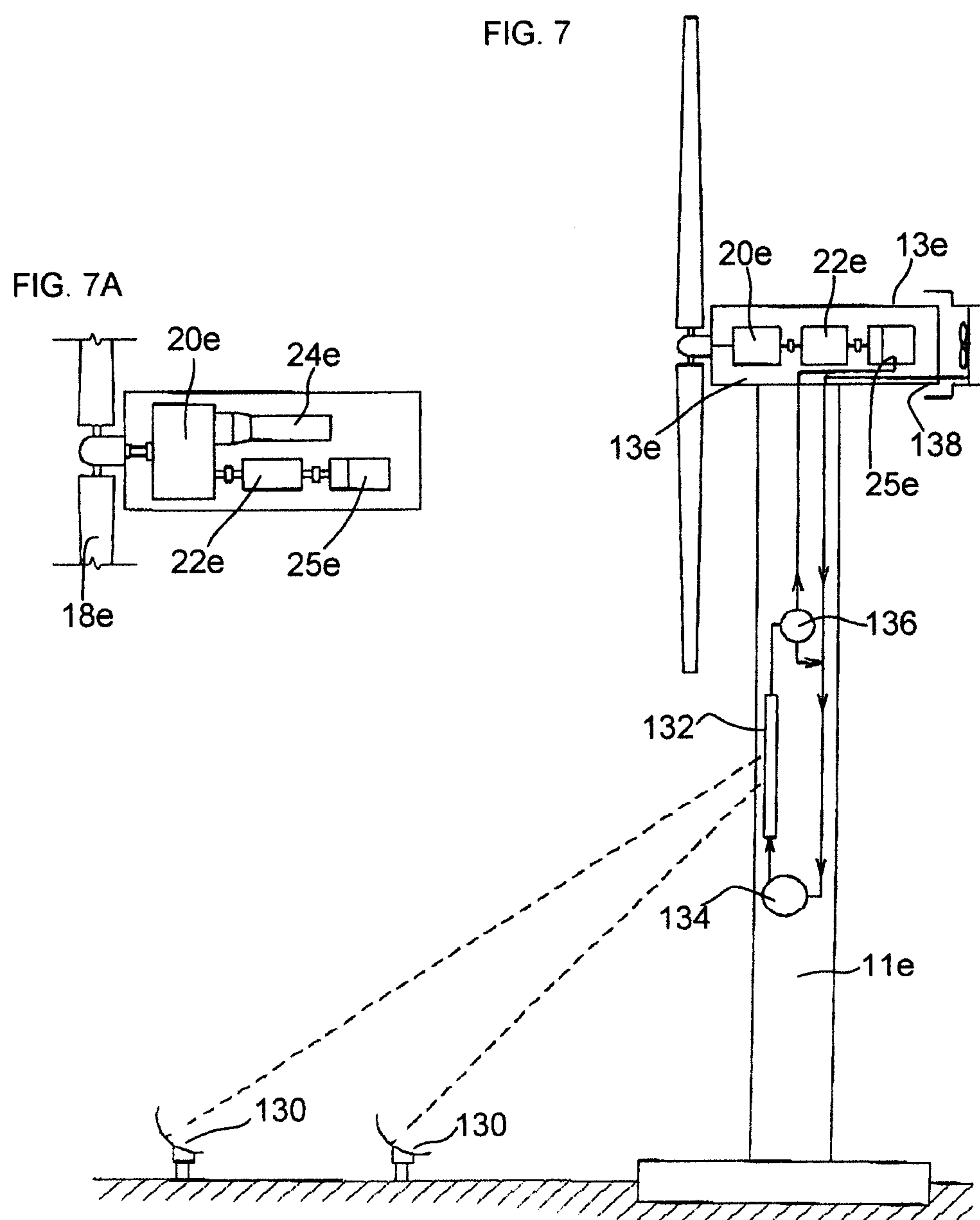
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FIG. 6



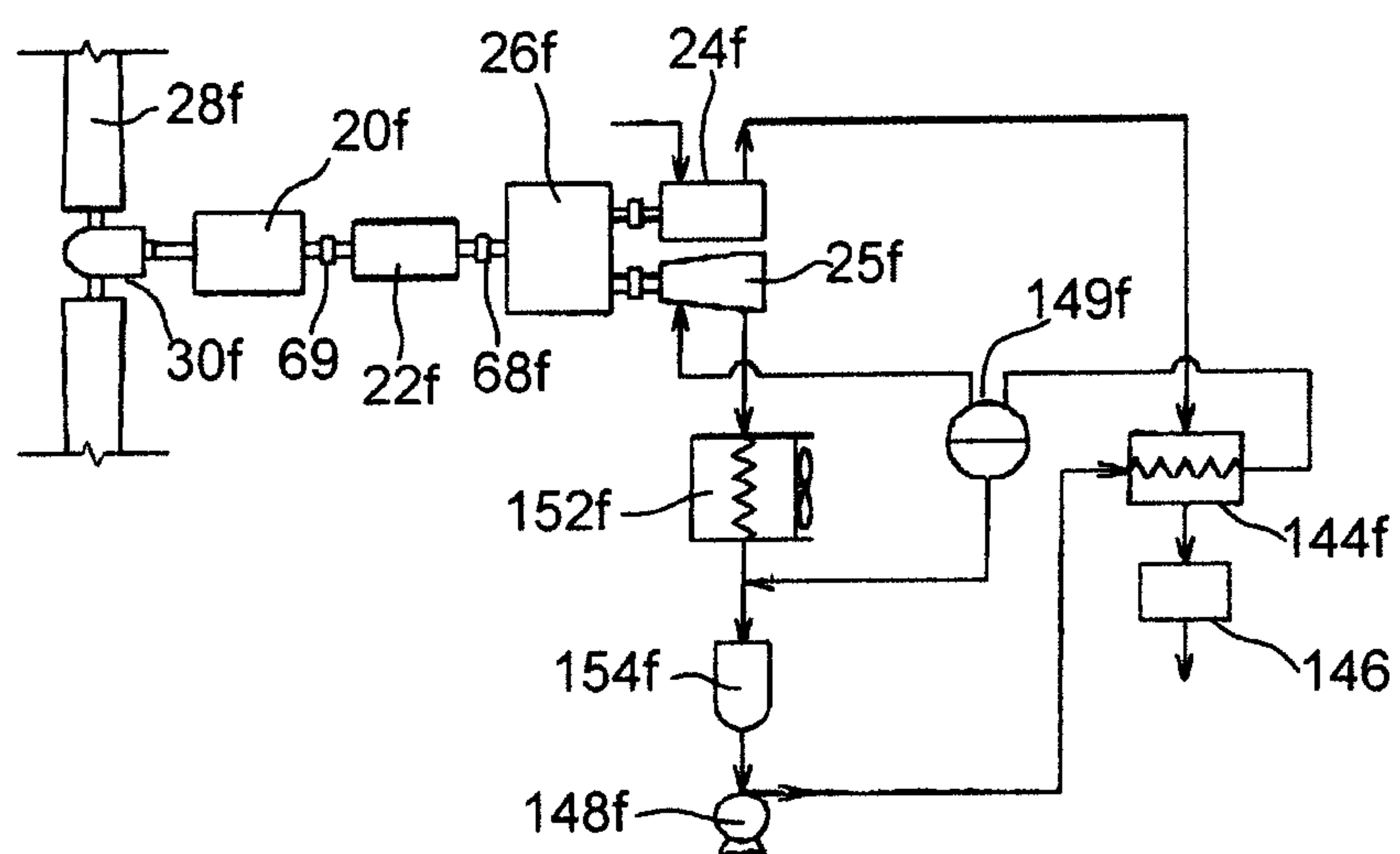
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FIG. 7



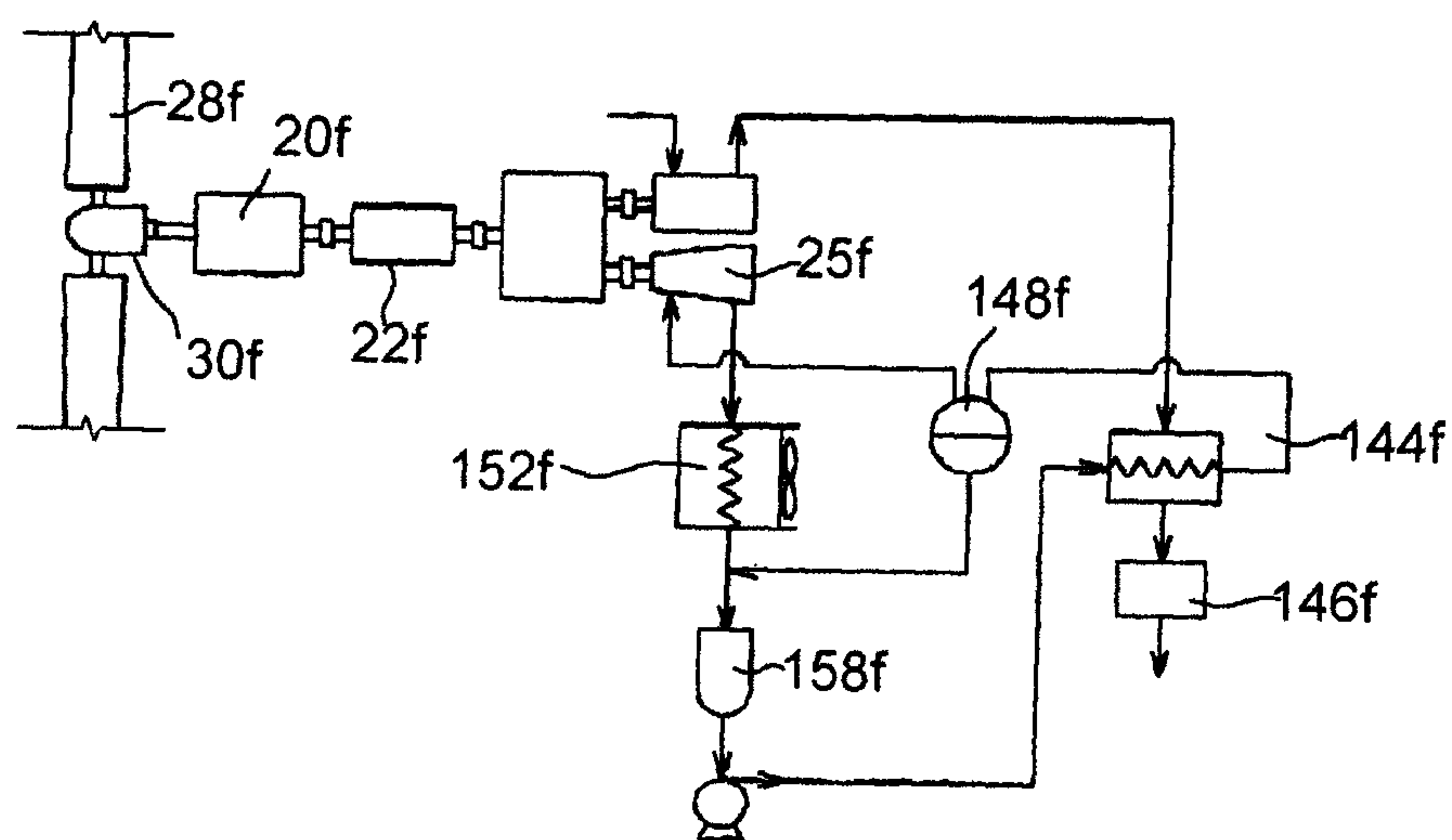
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FIG. 8



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FIG. 9



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FIG. 10

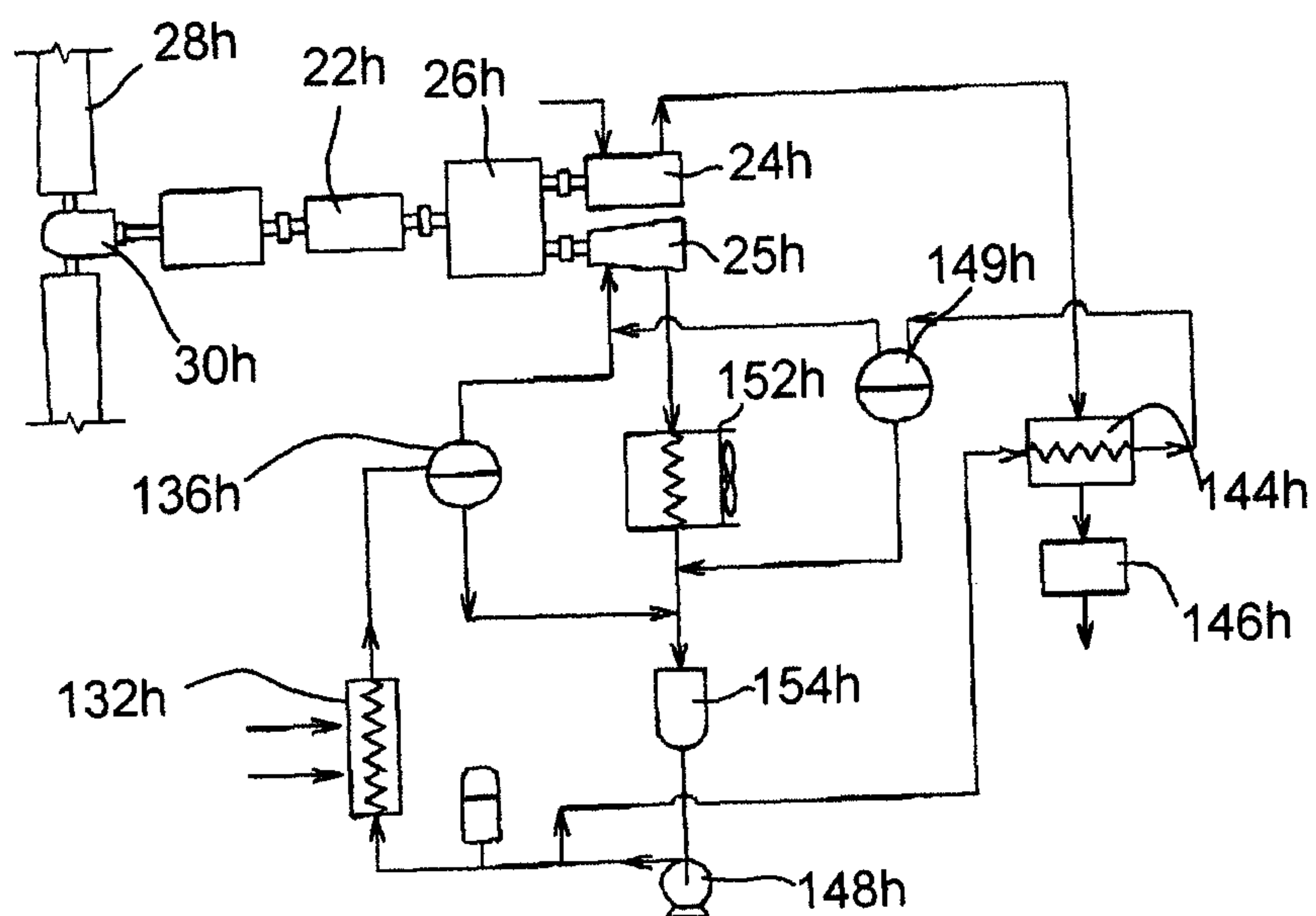
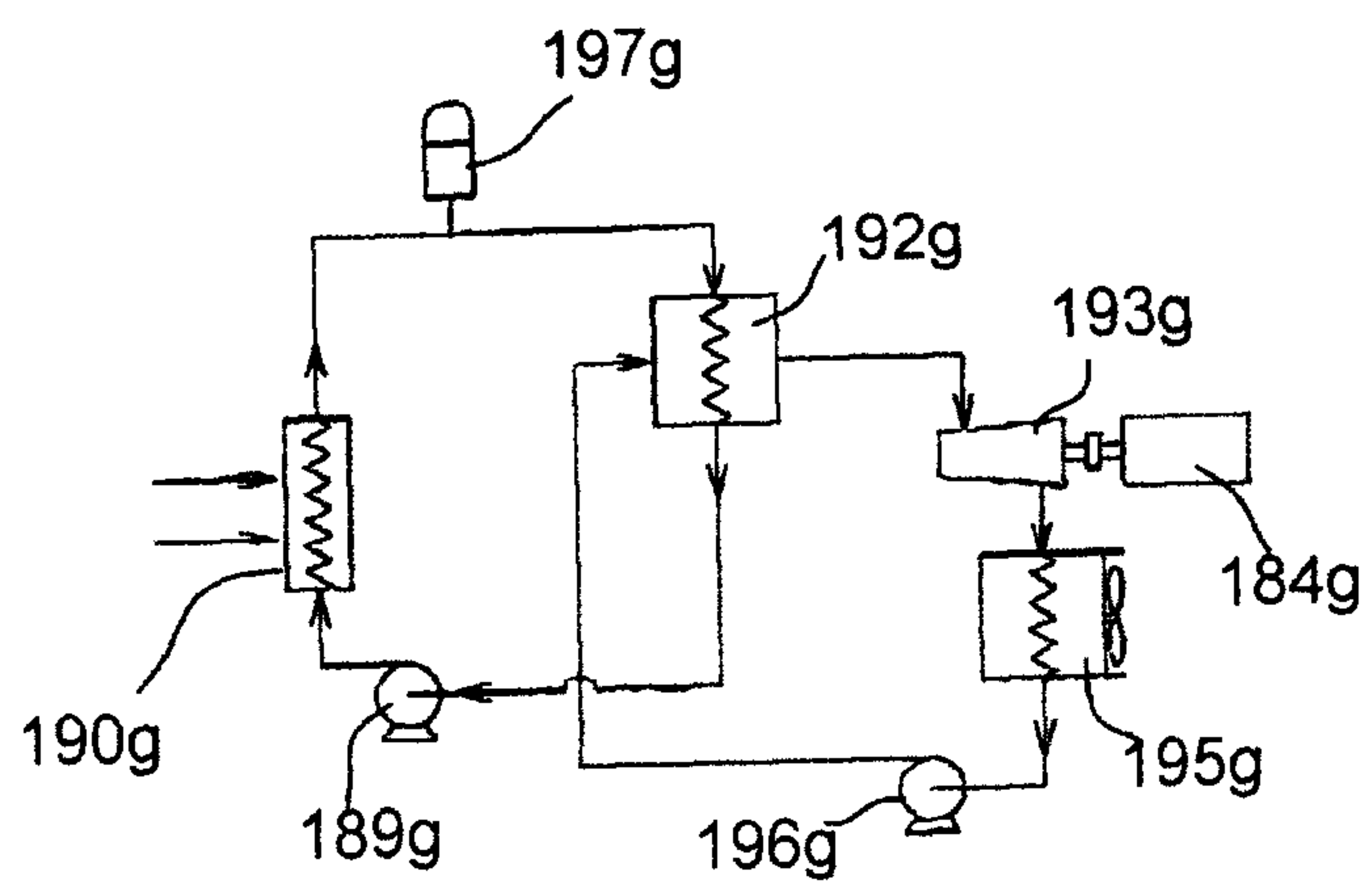


FIG. 9



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FIG. 11

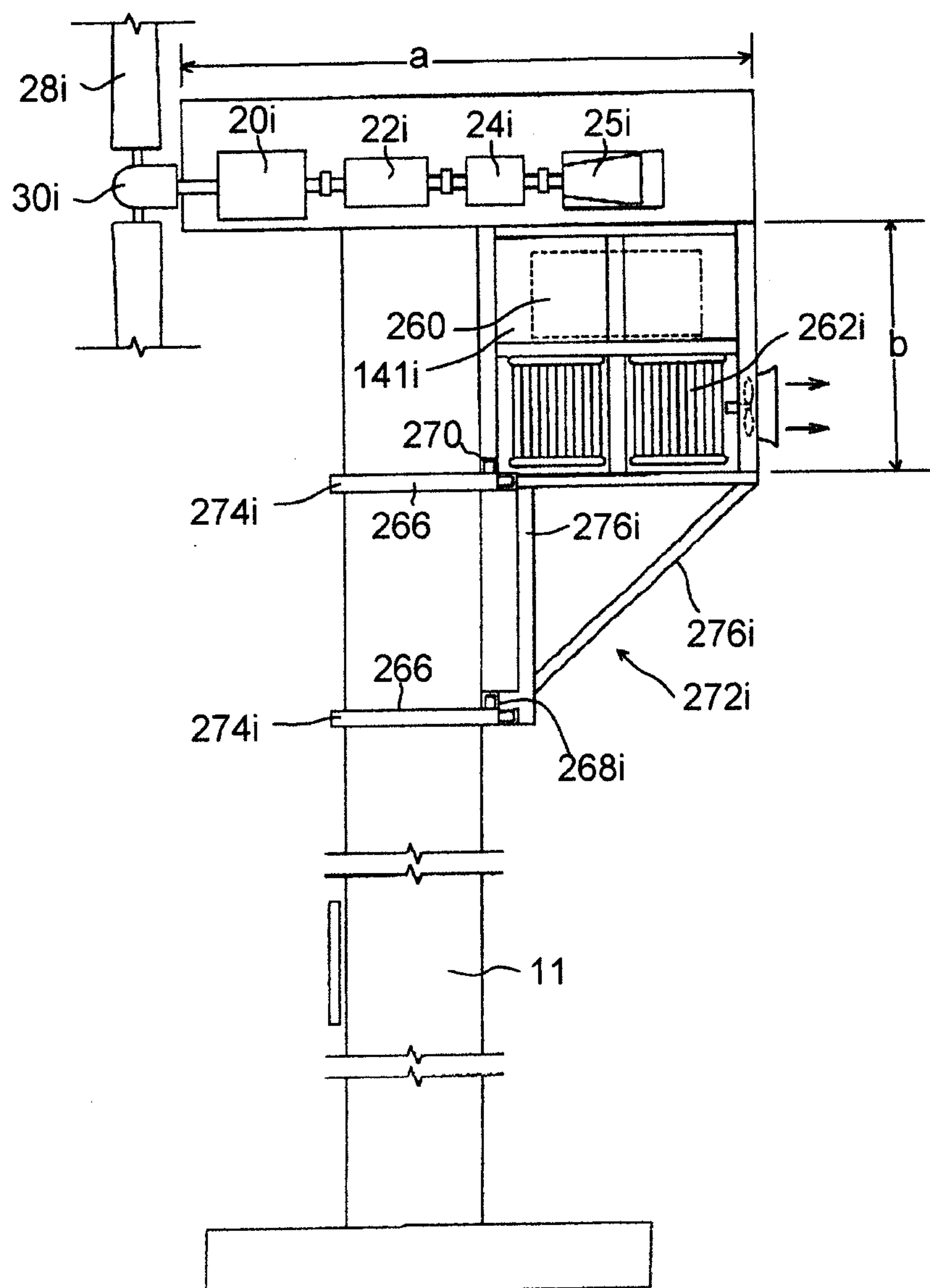


FIG. 1

