

[54] **ONE-PIECE ROLLER-IMPULSE MEMBER FOR TIMEPIECE ESCAPEMENT**
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[22] Filed: Feb. 26, 1971
[21] Appl. No.: 119,265
[52] U.S. Cl.58/122
[51] Int. Cl.G04b 15/08
[58] Field of Search.....58/116 R, 121 R, 122, 140

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[57] **ABSTRACT**
A one-piece safety roller and impulse member for a timepiece escapement control is disclosed. The combination element is a low coefficient of friction engineering plastic and serves the control purpose of two separate elements heretofore used in the art of timepiece lever escapements to which this invention pertains.

2 Claims, 5 Drawing Figures

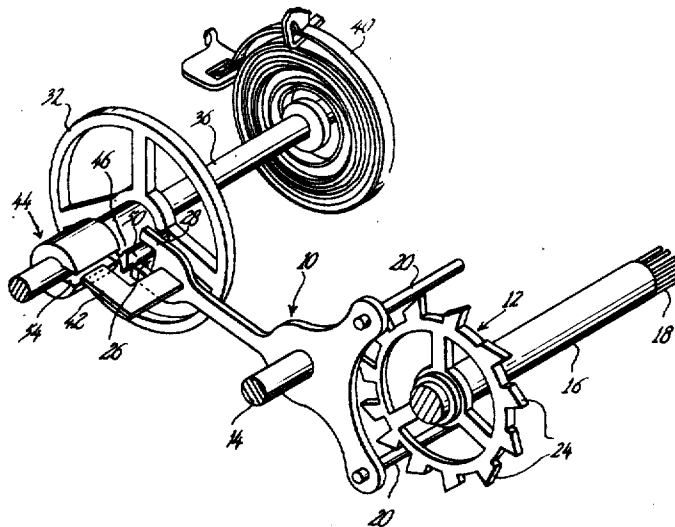


FIG. 1

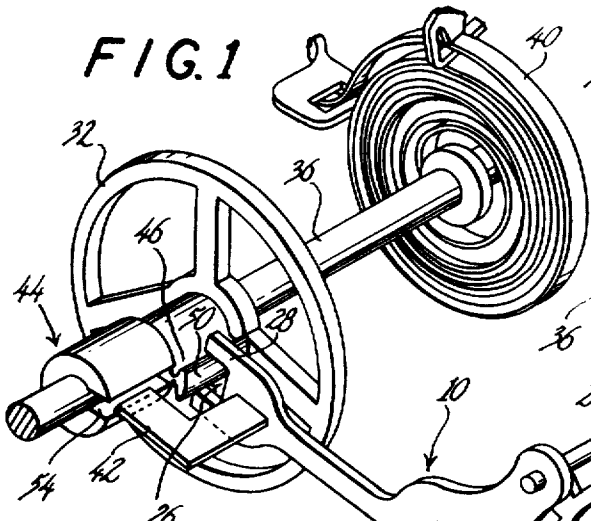


FIG. 2

PRIOR ART

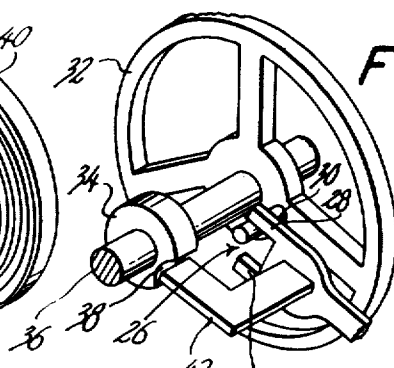


FIG. 3

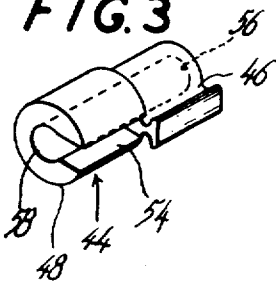


FIG. 5

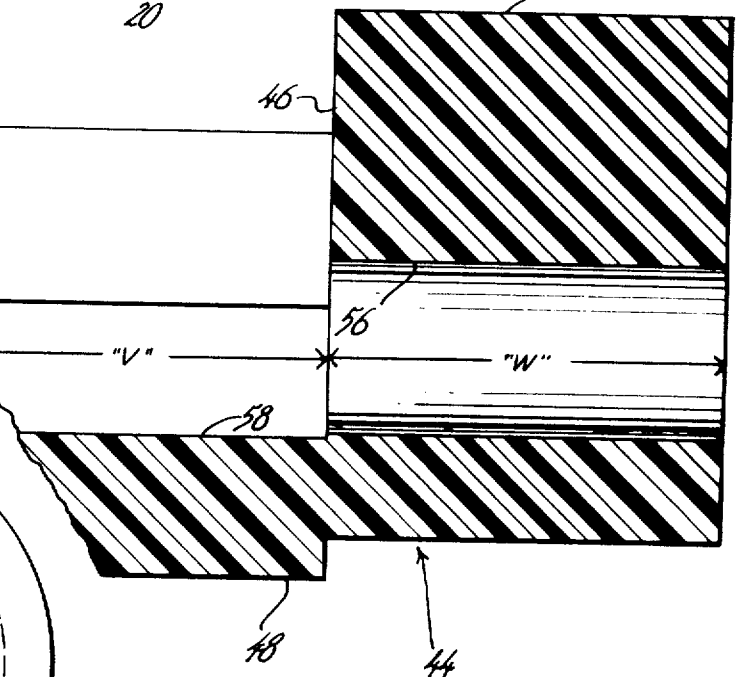
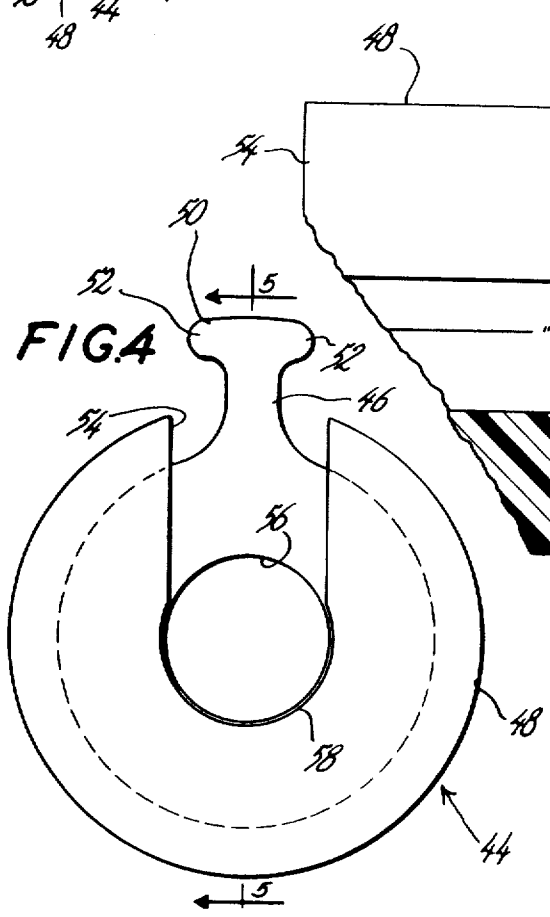


FIG. 4



ONE-PIECE ROLLER-IMPULSE MEMBER FOR TIMEPIECE ESCAPEMENT

BACKGROUND OF THE INVENTION

Lever escapements are predominately used in springwound clocks and watches. Common use is a toothed escape wheel connected through a gear train. The escape wheel is powered from the well known mainspring or other power source. An escapement lever pivoted about a point between its ends has at one end a pair of pallet pins and at its other end a fork. The pins are separated by a distance between the teeth of the escape wheel according to a well known formula. A balance wheel, biased by the usual hairspring, is arranged for oscillation on a balance staff.

As shown in the prior art (FIG. 2) an impulse pin of hardened steel (or jewel) is fixed to the balance wheel. A safety roller of brass or the like is fixed on the balance wheel shaft and is laterally separated from the impulse pin. The safety roller cooperates with the well known guard pin which is fixed in the escapement lever adjacent the forked end thereof. The indented or crescent portion of the safety roller allows the guard pin to pass freely when the escapement lever is unlocked, during oscillations of the balance wheel.

More specifically, the escape wheel, driven by the mainspring of the timepiece through its gear train, forces the escapement lever to pivot. Consequently, the forked end of the escapement lever acts on the impulse pin, causing the balance wheel to rotate and increase tension of the hairspring. The tensioned hairspring then causes rotation of the balance wheel in the opposite direction. The escapement lever being held in its banking position due to the draft action between the pallet and escape wheel tooth, is positioned for entry of the impulse pin as the balance rotates. Due to the inertia of the balance wheel and the energy stored in the hairspring, the impulse pin enters the fork slot of the lever during its return, rotating said lever and unlocking the lever pallet from the escape wheel tooth. When the unlocking action is completed, the escape wheel, driven by the mainspring and gear train, rotates, forcing the lever pallet outward thereby again pivoting the lever. The opposite side of the fork slot on the pivoting lever then strikes the opposite side of the impulse pin, causing the balance wheel to rotate and increase tension of the hairspring in the opposite direction. The opposite lever pallet at this time, locks another escape wheel tooth and the lever is now positioned for entry of the impulse pin in the opposite direction. As long as power is transmitted to the escape wheel, the sequence described continues to cycle.

The type escapement described is the commonly known detached lever escapement, where the balance is free of the influence of the escapement and power from the gear train, except during unlocking and impulse. The lever is positioned for repeated entry and re-entry of the impulse member into the forked end of the escapement lever. If the timepiece and consequently the movement is subjected to an external shock, the escapement lever may be jolted away from its banking position at a crucial time during balance wheel oscillation, and cause lock-up of the movement. To insure entry of the impulse pin into the lever fork, the motion of the lever away from the banking during shocks is limited by the interaction of the guard pin and safety roll. The lock remains between pallet and escape wheel and is commonly referred to as guard pin safety lock.

In light of the above, the principal object of the present invention is to provide an improved timepiece escapement mechanism that features an extremely efficient escapement control element in the form of a combination safety roll and impulse member that is simple in construction, very inexpensive to manufacture, is easy to install, requires no adjustment, is very durable in use and has very low coefficient of friction. In prior and current escapements of the type disclosed herein, the impulse pin must necessarily be lubricated to reduce the coefficient of friction during unlocking. However, lubrication

is detrimental during impulse because of the adhesion between the fork members and impulse pin. In keeping with the principal object of this invention, the one piece safety roll and impulse member is made of a low coefficient of friction plastic, requiring no lubrication, while smoothly effecting the unlocking sequence of the escapement cycle thereby eliminating the problem of adhesion.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is an enlarged schematic perspective view of the escapement mechanism of a timepiece showing the novel combination safety-roller — impulse member of the present invention;

FIG. 2 is similar to FIG. 1 but shows the safety roller and impulse pin arrangement so well known in the prior art;

FIG. 3 is a perspective view of the combination safety roller — impulse member of FIG. 1;

FIG. 4 is greatly enlarged front elevational view thereof; and
FIG. 5 is a sectional view taken along lines 5 — 5 of FIG. 4.

DESCRIPTION OF A PREFERRED EMBODIMENT

Referring now to FIG. 1 of the drawing, there is shown an escapement comprising an escape lever 10 and an escape wheel 12 which form the basic portion of a clock or watch escapement. The escape lever is mounted on an escape lever shaft 14 which is journaled in the base plates (not shown) of the movement. The escape wheel 12 is mounted on an escape wheel shaft 16, one end of which is also journaled in the base plates. The toothed end of shaft 16 is driven by the main spring of the movement by means of the gear train, the operation and characteristics of which are very well known and therefor not disclosed herein. The escape lever 10 carries a pair of pallet pins 20 which alternately engage the teeth 24 of the escape wheel 12 and effect the impulsing and locking operations that result in a unidirectional rotation of the escape wheel 12. If the escape wheel 12 is not periodically locked, the main spring will rotate the gear train at full speed until the main spring is completely unwound. The escape wheel is prevented from such free rotation by the action of the escape lever 10 which includes a fork-like portion 26 at one end thereof.

In the prior art, shown in FIG. 2, the opening formed by the times of the fork, is adapted to fit over an impulse pin 30, which is usually of hardened steel or a jewel and is fixed adjacent the hub of a balance wheel 32 in a well-known manner. Roller 34, commonly known as the safety roller and customarily of brass, is secured fast to a balance wheel staff 36 by pinning or the like. An indented crescent 38 of the safety roller allows the roller to pass under the urge of a hairspring 40 when the escape lever 10 is unlocked. The movement of the lever by the impulse pin is made at the fork; the impulse pin enters the fork opening and unlocking and impulse are effected. As one tooth slides off the impulse plane of one of the pallet pins, another tooth is stopped by the other pallet pin. During the stopping of the escape wheel, the balance wheel 32 continues to rotate; impulse pin 30 is then completely free of the fork-like portion 26. During the circular motion of the balance, the hairspring 40 has been "wound up" until it has stopped the balance and reversed its motion. The impulse pin 30 then returns to the fork opening, strikes a fork tine and pivots lever 10 whereupon the escape wheel 12 is unlocked. The sliding of a tooth 24 on the other pallet so begins again and the cycle continues. Thus the motion of the balance wheel 32 is maintained by the locking and unlocking action to generate impulses. Guard pin 42 is fixed at the forked end of escape lever 10 and aligns with crescent 38 during the unlocking and impulse functions of the escape lever. During the remainder of the oscillatory excursion of the balance wheel the guard pin 42 serves to prevent unlocking of the escapement lever 10 and thereby guards against malfunction of the escapement mechanism.

According to the present invention, a simple one-piece roller and impulse member is generally designated 44 and is provided with an impulse portion 46 and a safety roller portion 48. The composite unit 44 is preferably of a durable plastic such as DELRIN, for one example, and is formed as shown in the drawings. The impulse portion 46 is provided with a fork engaging extension 50 having rounded edges 52 which contribute to the smooth operation of the escapement by rolling in and out of contact with the tine 28. Due to the character of the plastic material, application of lubrication between parts is unnecessary.

To insure that the all-important concentricity of the safety roller portion 48 is preserved, and also to facilitate assembly, the safety roller portion is provided with a complete opening 54 or poising slot for accomodating the guard end of guard pin 42 in distinction to the notch 38 of the prior art FIG. 2. The poising slot 54 is aligned with the fork-engaging extension 50, as best seen in FIG. 4.

It is important to note that a bore 56 in impulse portion 46 is of lesser diameter than the semi-circular base 58 of the opening 54 in safety roller portion 48. Reference is made to the zone designated "W" which is of lesser diameter than the zone designated "V" in FIG. 5.

Accordingly, in assembling the composite one piece roller and impulse member, the bore 56 is entered on the balance staff and the composite unit is positioned by force-fitting, completely independent of the balance wheel. Assembly is greatly facilitated by obviating independent and critical alignment of impulse pin and safety roller which was heretofore the practice.

From the foregoing description it may be readily ascertained that the invention may be readily and reliably reproduced and assembled at a very high production rate and with attendant low manufacturing costs.

What is claimed is:

1. In a timepiece having a driven escape wheel with evenly spaced teeth about the periphery thereof, a two-ended escape lever pivoted between its ends, the escape lever having pallets at one end engaging the teeth of the escape wheel and a fork at its other end, a guard pin also at the other end of the escape lever adjacent the fork, a balance staff impulsed in a first direction by the fork of the escape lever through the driven teeth of the escape wheel, a hairspring connected to the balance staff to drive same in a direction opposite the impulsed first direction following an impulse imparted by the fork of the escape lever, the improvement comprising

a composite impulse member and safety roller formed of durable plastic and force-fitted on the balance staff in operative relation to the fork and guard pin of the escape lever to timingly control the drive of the escape wheel, said impulse member having an impulse portion integral therewith and said safety roller having a slotted portion therein, the impulse and slotted portions being formed in the same axial plane to operatively cooperate with the fork and guard pin, respectively, of the escape lever, and means integral with said impulse member and safety roller for maintaining the concentricity of the safety roller during the force-fitting of the composite impulse member and safety roller on the balance staff.

2. The composite impulse member and safety roller of claim 4 wherein said means for maintaining the concentricity of said safety roller comprises, the impulse member having bore therethrough and the safety roller slotted portion having a substantially semi-circular base portion on the same axis of the bore as the impulse member, the bore in the impulse member being of smaller diameter than the semi-circular bore in the safety roller, whereby the concentricity of the safety roller is preserved during force fitting of the composite impulse member and safety roller on the balance staff.

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