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(54) Title: PIVOT FOOT FOR DEACTIVATING ROCKER ARM

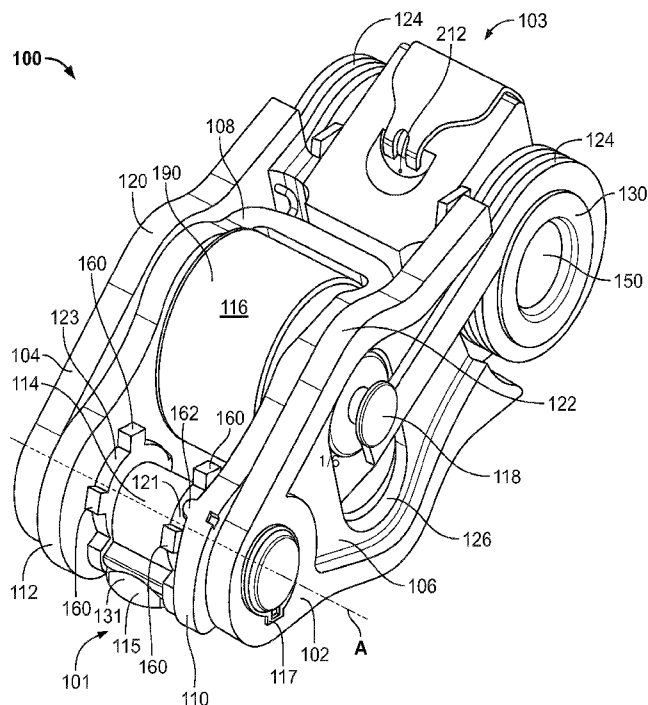


FIGURE 1

(57) Abstract: A dual body rocker arm comprising an outer arm, an inner arm, a substantially cylindrical pivot axle, a pivot foot and a first and second pivot foot retaining ring. The inner arm is disposed between side arms of the outer arm. The pivot axle is mounted to both the inner and outer arm. The pivot foot has a cylindrical axle interfacing surface and a valve tip interface. Retaining rings mount the pivot foot to the axle, allowing partial rotation of the pivot foot about the axle. Interfering protrusions on the retaining rings limit the range of motion of the pivot foot. A pivot foot having at least one interfering block is configured to restrict motion of the pivot foot by interfering with at least one interfering tab.

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## **PIVOT FOOT FOR DEACTIVATING ROCKER ARM**

### **FIELD OF THE INVENTION**

[0001] This application is directed to pivot feet of deactivating rocker arms for internal combustion engines.

### **BACKGROUND**

[0002] Many internal combustion engines utilize rocker arms to transfer rotational motion of cams to linear motion appropriate for opening and closing engine valves. Integration of rocker arms with the rest of the valve train, for example the valve tip, is necessary, and features that assist in such integration are advantageous.

### **SUMMARY**

[0003] A dual body rocker arm comprises an outer arm, an inner arm, a substantially cylindrical pivot axle, a pivot foot and a first and second pivot foot retaining ring. The outer arm and inner arm each have a first and second side arm. The inner arm is disposed between side arms of the outer arm. A pivot axle is mounted in apertures at one end of both the inner and outer arm. The pivot foot has a cylindrical axle interfacing surface and a valve tip interface. Retaining rings mount the pivot foot to the axle, allowing partial rotation of the pivot foot about the axle. Interfering protrusions on the retaining rings interfere with interfering projections located on one or both of the inner arms, limiting the range of motion of the pivot foot.

[0004] In another embodiment, a dual body rocker arm comprises an outer arm, an inner arm having at least one interfering tab, a substantially cylindrical pivot axle and a pivot foot having at least one interfering block configured to restrict motion of the pivot foot by interfering with at least one interfering tab.

### **BRIEF DESCRIPTION OF THE DRAWINGS**

[0005] It will be appreciated that the illustrated boundaries of elements in the drawings represent only one example of the boundaries. One of ordinary skill in the art will appreciate that a single element may be designed as multiple elements or that multiple elements may be

[0006] designed as a single element. An element shown as an internal feature may be implemented as an external feature and vice versa.

[0007] Further, in the accompanying drawings and description that follow, like parts are indicated throughout the drawings and description with the same reference numerals, respectively. The figures may not be drawn to scale and the proportions of certain parts have been exaggerated for convenience of illustration.

[0008] **Figure 1** illustrates a perspective view of an exemplary rocker arm **100** incorporating pivot foot **115**.

[0009] **Figure 2** illustrates an exploded view of exemplary rocker arm **100** incorporating pivot foot **115** shown in **Figure 1**.

[0010] **Figure 3** illustrates a side view of deactivating rocker arm **100** in relation to a cam **300**, lash adjuster **340** and valve stem **350**.

[0011] **Figure 4** illustrates a front view of deactivating rocker arm **100** in relation to a cam **300**, lash adjuster **340** and valve stem **350**.

[0012] **Figure 5** illustrates an exploded view of pivot foot assembly **500**.

[0013] **Figure 6** illustrates a side view of pivot foot **115**.

[0014] **Figure 7** illustrates a perspective view of an alternative embodiment of rocker arm **100** having a pivot foot **700**.

[0015] **Figure 8** is a perspective view of pivot foot **700**.

#### DETAILED DESCRIPTION

[0016] Certain terminology will be used in the following description for convenience in describing the figures and will not be limiting. The terms “upward,” “downward,” and other directional terms used herein will be understood to have their normal meanings and will refer to those directions as the figures are normally viewed.

[0017] **Figures 1** and **2** illustrate a perspective view and exploded view, respectively, of an exemplary deactivating rocker arm **100**. Deactivating rocker arm **100** is shown by way of example only and it will be appreciated that the configuration of deactivating rocker arm **100** that is the subject of this disclosure is not limited to the configuration of deactivating rocker arm **100** illustrated in the figures shown herein.

[0018] As shown in **Figures 1** and **2**, deactivating rocker arm **100** includes an outer arm **102** having a first outer side arm **104** and a second outer side arm **106**. An inner arm **108** is disposed between first outer side arm **104** and second outer side arm **106**. Inner arm **108** has a first inner side arm **110** and a second inner side arm **112**. Inner arm **108** and outer arm **102** are both mounted to a substantially cylindrical pivot axle **114**, located adjacent first end **101** of rocker arm **100**, which restricts motion of inner arm **108** relative to outer arm **102** to rotational motion about a longitudinal axis **A** of pivot axle **114**. When in its deactivated state, deactivating rocker arm **100** exhibits a rotational movement of inner arm **108** relative to outer arm **102**. Pivot axle **114** is secured in place relative to inner arm **108** and outer arm **102** by axle clips **117** inserted into recesses **119** in pivot axle **114**. Axle clips **117** are biased to fit securely to the contour of recesses **119** and prevent motion of axle **114** along longitudinal axis **A**. In the illustrated embodiment, pivot axle **114** is mounted in inner pivot axle apertures **220** and outer pivot axle apertures **230** adjacent the first end **101** of rocker arm **100**. Spacing between recesses **119** is chosen so that when axle clips **117** are placed in recesses **119**, axle clips **117** also contact outer arm **102** and thereby prevent motion of pivot axle **114** with respect to outer arm **102** and inner arm **108** along axis **A**. As used herein, “radially” denotes a direction of a line intersecting **A** at a point and lying in a plane perpendicular to **A**. The term “radially outward” defines the radial direction pointing away from axis **A**. Conversely, “radially inward” defines the radial direction pointing toward axis **A**.

[0019] When rocker arm **100** in the deactivated state, inner arm **108** pivots downwardly relative to outer arm **102** when lifting portion **324** of cam **300** (as shown in **Figure 3**) comes into contact with roller **116** of bearing **190**, thereby pressing it downward. Axle slots **126** allow for downward movement of bearing axle **118**, and therefore of inner arm **108** and bearing **190**. As cam **300** continues to rotate, lifting portion **324** of cam **300** rotates away from roller **116** of bearing **190**, allowing bearing **190** to move upwardly as bearing axle **118** is

biased upwardly by bearing axle springs **124**. The illustrated bearing axle springs **124** are torsion springs secured to mounts **150** located on outer arm **102** by spring retainers **130**. Bearing axle springs **124** are secured adjacent second end **103** of rocker arm **100** and have spring arms that come into contact with bearing axle **118**. As bearing axle **118** and spring arm move downward, bearing axle **118** slides along spring arm. Knob **262** extends from the end of bearing axle **118** and creates a slot **264** in which spring arm sits.

[0020] With reference to **Figures 1** and **2**, pivot foot **115** is mounted on pivot axle **114** between first **110** and second **112** inner side arms. First and second clips **121**, **123** retain pivot foot **115** against pivot axle **114**. In the illustrated embodiment, clips **121**, **123** are contoured rings that surround pivot axle **114**. In other embodiments, clips **121**, **123** need not completely surround pivot axle **114**. As shown below in **Figure 6** below, pivot foot **115** has a concave cylindrical axle interfacing surface **125** extending between first and second inner side arms **110**, **112**. A valve tip interface **131** is disposed on the opposite side of pivot foot **115**, and it features valve tip interfacing surface **135** and second clip mounting portions **127**, **129**. Clips **121**, **123** overlap first and second clip mounting portions **127**, **129** of pivot foot **115**. First clip mounting portion **127** is disposed between valve tip interface surface **135** and first inner side arm **110**, while second clip mounting portion **129** is disposed between valve tip interface surface **135** and second inner side arm **112**.

[0021] Clips **121**, **123** have interfering protrusions **160** extending radially outward relative to longitudinal axis **A** of axle **114**. Upon rotation of clips **121**, **123** about axle **114** in either direction, an interfering protrusion **160** will come into contact with an interfering projection **162** extending from one of the first and second inner side arms **110**, **112** to the other of first and second inner side arms **110**, **112**. In the illustrated case, projection **162** is on first inner side arm **110**. Once in contact, interfering protrusion **160** and interfering projection **162** prevent any further rotation of clips **121**, **123** in the selected direction about axis **A** relative to inner body **108**. Clips **121**, **123**, however, may then be rotated in the opposite direction until contact is made between a different interfering protrusion **160** and interfering projection **162**.

[0022] The mechanism for selectively deactivating rocker arm **100**, which in the illustrated embodiment is found near second end **103** of rocker arm **100**, is shown in **Figure 2** as

comprising latch **202**, latch spring **204**, spring retainer **206** and clip **208**. Latch **202** is configured to be mounted inside outer arm **102**. Latch spring **204** is placed inside latch **202** and secured in place by latch spring retainer **206** and clip **208**. Once installed, latch spring **204** biases latch **202** toward first end **101** of rocker arm **100**, allowing latch **202**, and in particular engaging portion **210** to engage inner arm **108**, thereby preventing inner arm **108** from moving with respect to outer arm **102**. When latch **202** is engaged with inner arm **108** in this way, rocker arm **100** is in the activated state, and will transfer motion from cam **300** to valve stem **350**. To deactivate rocker arm **100**, oil pressure sufficient to counteract the biasing force of latch spring **204** may be applied, for example, through port **212**, which is configured to permit oil pressure to be applied to surface of latch **202**. Oil pressure is applied, latch **202** retracts from engagement with inner arm **108**, allowing inner arm **108** to rotate about pivot axle **114**. In both activated and deactivated states, linear portion **250** of orientation clip **214** engages latch **202** at flat surface **218**. Orientation clip **214** is mounted in clip apertures **216**, thereby maintaining horizontal orientation of linear portion **250** relative to rocker arm **100**.

[0023] **Figures 3** and **4** illustrate a side view and front view, respectively, of rocker arm **100** in relation to cam **300** having lift lobe **320** with base circle **322** and lifting portion **324**, and two circular “no-lift” or safety lobes **310** positioned above first and second safety lobe contacting surfaces **120**, **122**. Circular safety lobes **310** are concentric with base circle **322** of lift lobe **320**, and have a smaller diameter than the diameter of base circle **322**. It should be noted that the diameter of two safety lobes **310** need not be identical, need not be circular, and may have a diameter equal to or larger than the diameter of base circle **322**. In such a scenario, first and second safety lobe contacting surfaces **120**, **122** should be appropriately located such that they are spaced from safety lobes **310** under normal engine operation, but also come into contact with safety lobes **310** under certain abnormal engine conditions. As is clear from **Figures 3** and **4**, first and second safety lobe contacting surfaces **120**, **122**, when used in combination with circular safety lobes **310**, do not transfer rotational motion of cam **300** to rocker arm **100**. Alternatively, safety lobes **310** may be replaced with conventional lobes configured to transmit rotational motion of cam **300** to rocker arm **100** through surfaces **120**, **122**.

[0024] **Figures 3 and 4** illustrate roller **116** in contact with lift lobe **320**. A lash adjuster **340** engages rocker arm **100** adjacent its second end **103**, and applies upward force to rocker arm **100**, and in particular outer rocker arm **102**, while mitigating against valve lash. Valve stem **350** engages pivot foot **115** adjacent first end **101** of rocker arm **100**. In the activated state, rocker arm **100** periodically pushes valve stem **350** downward, which serves to open the corresponding valve (not shown). In the deactivated state, inner arm **108** undergoes lost motion movement relative to outer arm **102** about pivot axle **114**. As shown in **Figures 1 and 2**, bearing axle **118** may be mounted in bearing axle apertures **260** of inner arm **108**. In such a configuration, axle slots **126** of outer arm **102** accept bearing axle **118** and allow for lost motion movement of bearing axle **118** and by extension inner arm **108** when rocker arm **100** is in a deactivated state. “Lost motion” movement can be considered movement of rocker arm **100** in response to cam **300** that does not transmit rotating motion of cam **300** to the valve. In the illustrated embodiments, lost motion is exhibited by pivotal motion of inner arm **108** relative to outer arm **102** about pivot axle **114**.

[0025] As shown in **Figure 4**, clips **121, 123** have valve tip guides **164** that assist in maintaining proper alignment between valve tip **351** and pivot foot **115**. Valve tip guides **164** extend radially outward from axis **A** farther than valve tip interfacing surface **135** of pivot foot **115**, thereby extending along opposite sides of valve tip **351** as seen in **Figure 4**. In the embodiment shown in **Figure 4**, valve tip guides **164** prevent valve tip **351** from moving from side to side along **A** relative to pivot foot **115**, which could result in pivot foot **115** becoming disengaged with valve tip **351**.

[0026] **Figure 5** illustrates an exploded view of pivot foot assembly **500** comprising pivot foot **115** and clips **121, 123**. When assembled, clips **121, 123** overlap first and second clip mounting portions **127, 129** of pivot foot **115**, securing pivot foot **115** to axle **114**. When rocker arm **100** is assembled, interfering protrusions **160** extending radially outward relative to longitudinal axis **A** of axle **114**, as shown in **Figures 1, 2, 4 and 5**. Rotation of clips **121, 123** and pivot foot **115** about axle **114** in either available direction will result in interfering contact between interfering protrusion **160** and interfering projection **162**, at which point further rotation in that direction is prevented. The amount of rotation pivot foot **115** and clips **121, 123** are permitted depends on the space between interfering protrusions **160**. Between

interfering protrusions **160** shown in **Figure 5** are projection gaps **140** that do not extend as far radially outward from axis **A** as interfering protrusions **160**, thereby ensuring projection gaps **140** do not interfere with an interfering projection **162**. When rocker arm **100** is assembled, interfering projection **162** is disposed adjacent a projection gap **140** and between interfering protrusions **160**.

[0027] Prior to contact with valve stem **350**, assembled pivot foot **115** and clips **121**, **123** are rotatably secured to pivot axle **114**, and their rotational motion limited as stated above by interference between interfering protrusions **160** and interfering projection **162**. The limited range of rotational movement ensures pivot foot **115** is in a suitable location for assembly of rocker arm **100** into the engine (not shown), and in particular for assembly with valve tip **351**. When engaged by valve stem **350**, rotational movement of pivot foot **115** and clips **121**, **123** is further limited. During normal engine operation, valve tip **351** may become momentarily disengaged with pivot foot **115**, such as when upward momentum of the rocker arm **100** causes the rocker arm **100** to continue movement upward after the valve (not shown) is closed and the valve stem **350** and valve tip **351** have ceased moving upward. At such a point in time, the valve tip **351** may become disengaged with the valve tip interfacing surface **135**. When the valve tip **351** and valve tip interfacing surface **135** become disengaged in this way, valve tip guides **164** assist in preventing excessive side to side movement of the valve tip **351** along direction **A**. Preventing such side to side movement, along with the restricted movement of pivot foot **115** due to clips **121**, **123** and interfering projection **162**, allows valve tip **351** to readily reengage pivot foot **115**.

[0028] **Figure 6** illustrates a side view of pivot foot **115**. As shown in **Figure 6**, pivot foot **115** has a concave cylindrical axle interfacing surface **125**. Valve tip interfacing surface **135** is illustrated as a flat surface, but may take other forms, such as a curved or indented surface.

[0029] **Figure 7** illustrates a partial perspective view of a rocker arm **100** having an alternative embodiment of pivot foot **700**. The illustrated pivot foot **700** does not require clips **121**, **123**. Instead, pivot foot **700** has interfering blocks **132** that restrict movement of pivot foot **700** about axle **114** due to interfering tabs **134** extending from each of first and second inner side arms **110**, **112** toward the opposing inner side arm. In this arrangement, pivot foot

**700** is in a “loose-trapped” fit between the tabs **134** and axle **114**. When interfering blocks **132** contact interfering tabs **134**, movement of pivot foot **700** is restricted. Likewise, when the axle interfacing surface **125** contacts axle **114**, the movement of the pivot foot **700** is limited to rotational movement about the axle **114**, and then only to the extent the rotational motion is not prohibited by interference by interfering blocks **132** coming in contact with interfering tabs **134**.

[0030] **Figure 8** illustrates pivot foot **700** found in the embodiment shown in **Figure 7**. Interfering blocks **132** are integral with pivot foot **700** and extend radially outward from axis **A** toward interfering tabs **134**. Interfering tabs **134** are visible in **Figure 9**, which shows a perspective view of inner body **108**. When the alternative embodiment of rocker arm **100** is assembled, motion of pivot foot **700** will be restricted by interfering contact between axle interfacing surface **125** and axle **114**, and between interfering tabs **134** and interfering blocks **132**. This interfering contact prevents not only undesirable rotational motion about **A**, but also restricts pivot foot to remain approximately between axle **114** and interfering tabs **134** until assembly of rocker arm **100** with the engine (not shown).

[0031] When rocker arm **100** is assembled into position within an internal combustion engine (not shown), the restricted amount movement allowed pivot foot **700** as described herein ensures it will be in position to engage valve tip **351** without the need to manually adjust the position of pivot foot **700**. When engaged by valve stem **350**, movement of pivot foot **700** is further restricted, and axle interfacing surface **125** will contact axle **114**. As stated previously, during normal engine operation, valve tip **351** may become momentarily disengaged from pivot foot **700**. In the illustrated embodiment, interfering tabs **134** assist in preventing excessive side to side movement along direction **A**. This allows valve tip **351** to readily reengage pivot foot **700**.

[0032] For the purposes of this disclosure and unless otherwise specified, “a” or “an” means “one or more.” To the extent that the term “includes” or “including” is used in the specification or the claims, it is intended to be inclusive in a manner similar to the term “comprising” as that term is interpreted when employed as a transitional word in a claim. Furthermore, to the extent that the term “or” is employed (e.g., A or B) it is intended to mean

“A or B or both.” When the applicants intend to indicate “only A or B but not both” then the term “only A or B but not both” will be employed. Thus, use of the term “or” herein is the inclusive, and not the exclusive use. See, Bryan A. Garner, A Dictionary of Modern Legal Usage 624 (2d. Ed. 1995). Also, to the extent that the terms “in” or “into” are used in the specification or the claims, it is intended to additionally mean “on” or “onto.” Furthermore, to the extent the term “connect” is used in the specification or claims, it is intended to mean not only “directly connected to,” but also “indirectly connected to” such as connected through another component or multiple components. As used herein, “about” will be understood by persons of ordinary skill in the art and will vary to some extent depending upon the context in which it is used. If there are uses of the term which are not clear to persons of ordinary skill in the art, given the context in which it is used, “about” will mean up to plus or minus 10% of the particular term. From about X to Y is intended to mean from about X to about Y, where X and Y are the specified values.

[0033] While the present disclosure illustrates various embodiments, and while these embodiments have been described in some detail, it is not the intention of the applicant to restrict or in any way limit the scope of the claimed invention to such detail. Additional advantages and modifications will readily appear to those skilled in the art. Therefore, the invention, in its broader aspects, is not limited to the specific details and illustrative examples shown and described. Accordingly, departures may be made from such details without departing from the spirit or scope of the applicant's claimed invention. Moreover, the foregoing embodiments are illustrative, and no single feature or element is essential to all possible combinations that may be claimed in this or a later application.

CLAIMS

## 1. A dual body rocker arm comprising:

a first end, a second end, an outer arm, an inner arm, a substantially cylindrical pivot axle, a pivot foot, and a first and second pivot foot retaining ring;

the outer arm having a first and second outer side arm extending from the first end to the second end, and outer pivot axle apertures disposed adjacent the first end and configured for mounting the pivot axle;

the inner arm disposed between the first and second outer side arms and having a first and second inner side arm, the first and second inner side arms having inner pivot axle apertures disposed adjacent the first end and configured for mounting the pivot axle;

the pivot axle disposed in the inner pivot axle apertures and the outer pivot axle apertures;

the pivot foot having a cylindrical axle interfacing surface extending between the first and second inner side arms, a valve tip interface disposed between the first and second inner side arms, and first and second retaining ring mounting portions, the first retaining ring mounting portion disposed between the valve tip interface and the first inner side arm, and the second retaining ring mounting portion disposed between the valve tip interface and the second inner side arm; and,

the first and second retaining rings at least partially surrounding the pivot axle and partially rotatably mounting the pivot foot to the pivot axle.

## 2. The dual body rocker arm of claim 1, wherein at least one of the first and second retaining rings has at least one interfering protrusion.

3. The dual body rocker arm of claim 2, wherein the at least one of the first and second retaining rings has a first and second interfering protrusion separated by a projection gap configured to permit movement of the at least one of the first and second retaining rings relative to an interfering projection disposed on the inner arm.
4. The dual body rocker arm of claim 1, wherein at least one of the first and second inner side arms has an interfering projection extending from at least one of the first inner side arm toward the second inner side arm, or the second inner side arm toward the first inner side arm.
5. The dual body rocker arm of claim 4, wherein the interfering projection is disposed adjacent a projection gap of at least one of the retaining rings and is configured to limit rotational movement of the retaining ring about the pivot axle.
6. The dual body rocker arm of claim 1, wherein the retaining rings comprise guides extending further from a longitudinal axis of the pivot axle than a valve tip interfacing surface of the pivot foot.
7. The dual body rocker arm of claim 1, further comprising:
  - a first axle clip recess disposed adjacent a first end of the pivot axle, a second axle clip recess disposed adjacent a second end of the pivot axle; and,
  - an axle clip disposed within each of the first and second axle clip recesses.
8. The dual body rocker arm of claim 1, wherein at least one of the first and second retaining rings has a first and second interfering protrusion separated by a projection gap configured to permit movement of the at least one of the first and second retaining rings relative to an interfering projection extending from at least one of the first inner side arm toward the second inner side arm, or the second inner side arm toward the first inner side arm, and wherein the interfering projection is disposed adjacent the projection gap of the at least one of the retaining rings and is configured to limit rotational movement of the retaining ring about the pivot axle.

9. A dual body rocker arm comprising:

a first end, a second end, an outer arm, an inner arm, a substantially cylindrical pivot axle, and a pivot foot;

the outer arm having a first and second outer side arm extending from the first end to the second end, and outer pivot axle apertures disposed adjacent the first end and configured for mounting the pivot axle;

the inner arm disposed between the first and second outer side arms and having a first and second inner side arm, the first and second inner side arms having inner pivot axle apertures disposed adjacent the first end and configured for mounting the pivot axle, the first inner side arm having an interfering tab adjacent the first end and extending toward the second inner side arm, the second inner side arm having an interfering tab adjacent the first end and extending toward the first inner side arm;

the pivot axle disposed in the inner pivot axle apertures and the outer pivot axle apertures;

the pivot foot having a concave cylindrical axle interfacing surface extending between the first and second inner side arms, a valve tip interface disposed between the first and second inner side arms, and at least one interfering block, the at least one interfering block disposed between the valve tip interface and one of the first inner side arm and second inner side arm, the at least one interfering block configured to interfere with at least one of the interfering tabs.

10. The dual body rocker arm of claim 9 further comprising:  
first and second interfering blocks, the first interfering block disposed between the valve tip interface and the first inner side arm, the second interfering block disposed between the valve tip interface and the second inner side arm.
11. The dual body rocker arm of claim 9 wherein the at least one interfering block is configured to dispose the pivot foot approximately between the pivot axle and interfering tabs.
12. The dual body rocker arm of claim 9 wherein the interfering tabs are configured to restrict movement of a valve tip relative to the pivot foot.
13. The dual body rocker arm of claim 9, further comprising:  
a first axle clip recess disposed adjacent a first end of the pivot axle, a second axle clip recess disposed adjacent a second end of the pivot axle;  
an axle clip disposed within each of the first and second axle clip recesses.
14. A dual body rocker arm comprising:  
a first end, a second end, an outer arm, an inner arm, a substantially cylindrical pivot axle, and a pivot foot having an axle interfacing surface and a valve tip interface;  
the outer arm having a first and second outer side arm extending from the first end to the second end, and outer pivot axle apertures disposed adjacent the first end and configured for mounting the pivot axle;  
the inner arm disposed between the first and second outer side arms and having a first and second inner side arm, the first and second inner side arms having inner pivot axle apertures disposed adjacent the first end and configured for mounting the pivot axle;

the pivot axle disposed in the inner pivot axle apertures and the outer pivot axle apertures; and,

a means for retaining the pivot foot in a position suitable for engagement by a valve tip.

15. The dual body rocker arm of claim 14, wherein the means for retaining comprises:

at least one interfering projection extending from at least one of the first inner side arm toward the second inner side arm, or the second inner side arm toward the first inner side arm.

16. The dual body rocker arm of claim 14, wherein the means for retaining comprises at least one retaining rings at least partially surrounding the pivot axle and partially rotatably mounting the pivot foot to the pivot axle.

17. The dual body rocker arm of claim 16 wherein the means for retaining comprises at least one interfering protrusion.

18. The dual body rocker arm of claim 14 wherein the means for retaining comprises at least one interfering tab extending from the first inner side arm toward the second inner side arm or the second inner side arm toward the first inner side arm.

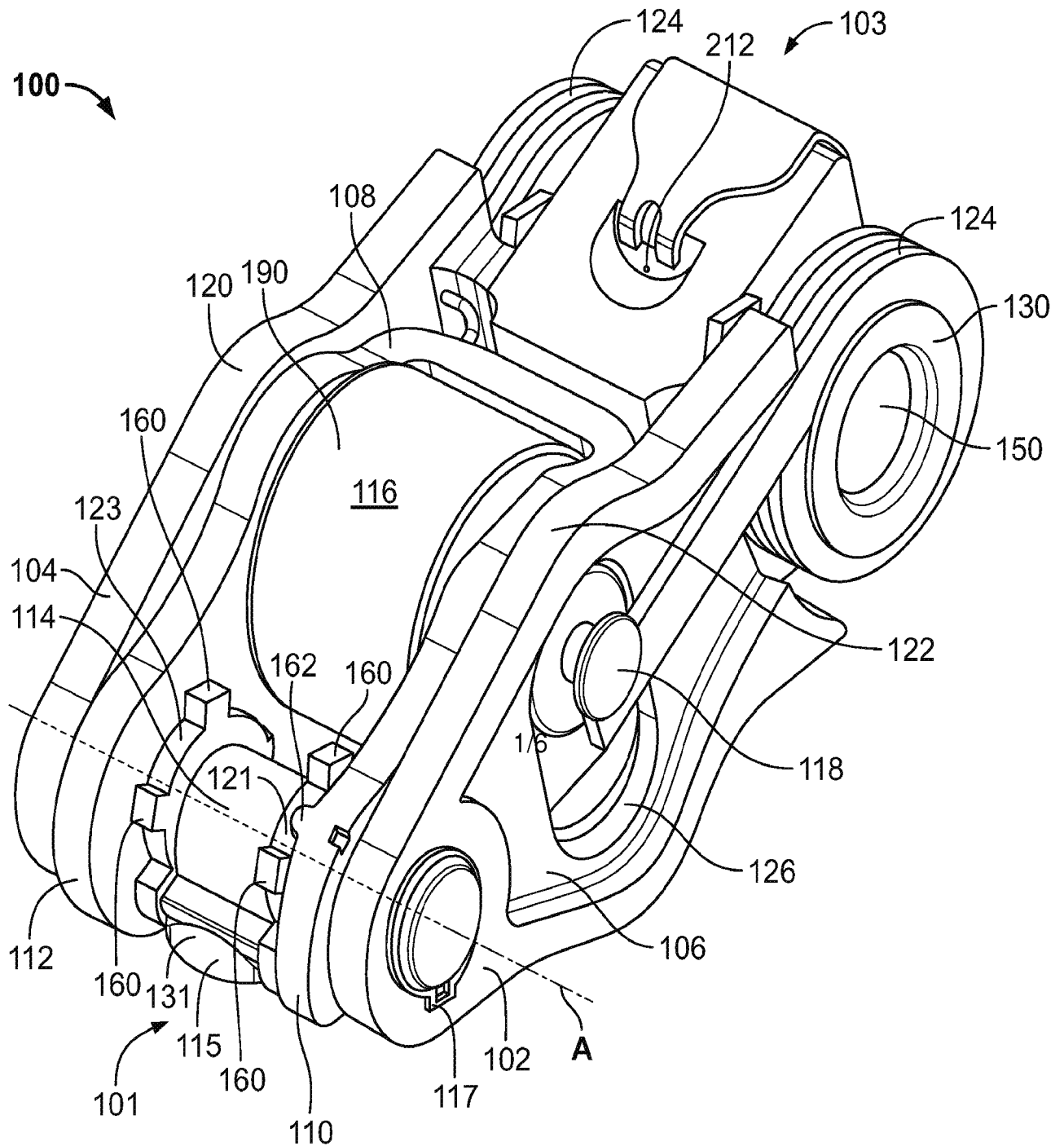
19. The dual body rocker arm of claim 14 wherein:

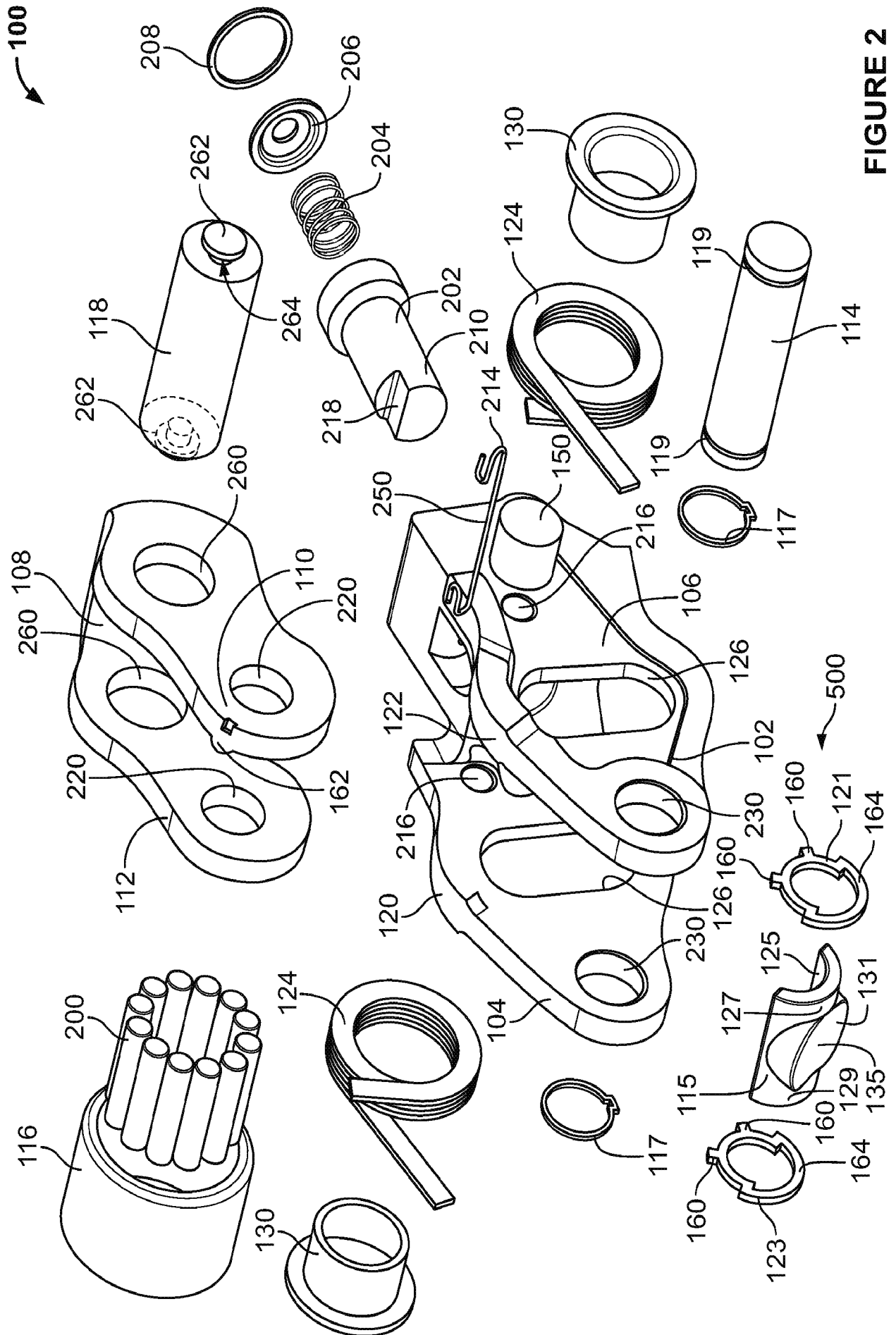
the pivot foot comprises a cylindrical axle interfacing surface extending between the first and second inner side arms, a valve tip interface disposed between the first and second inner side arms; and,

wherein the means for retaining comprises at least one interfering block, the at least one interfering block disposed between the valve tip interface and one of the first inner side arm and second inner side arm.

20. The dual body rocker arm of claim 14, comprising:

- a first axle clip recess disposed adjacent a first end of the pivot axle, a second axle clip recess disposed adjacent a second end of the pivot axle; and,
- an axle clip disposed within each of the first and second axle clip recesses.

**FIGURE 1**



## FIGURE 2

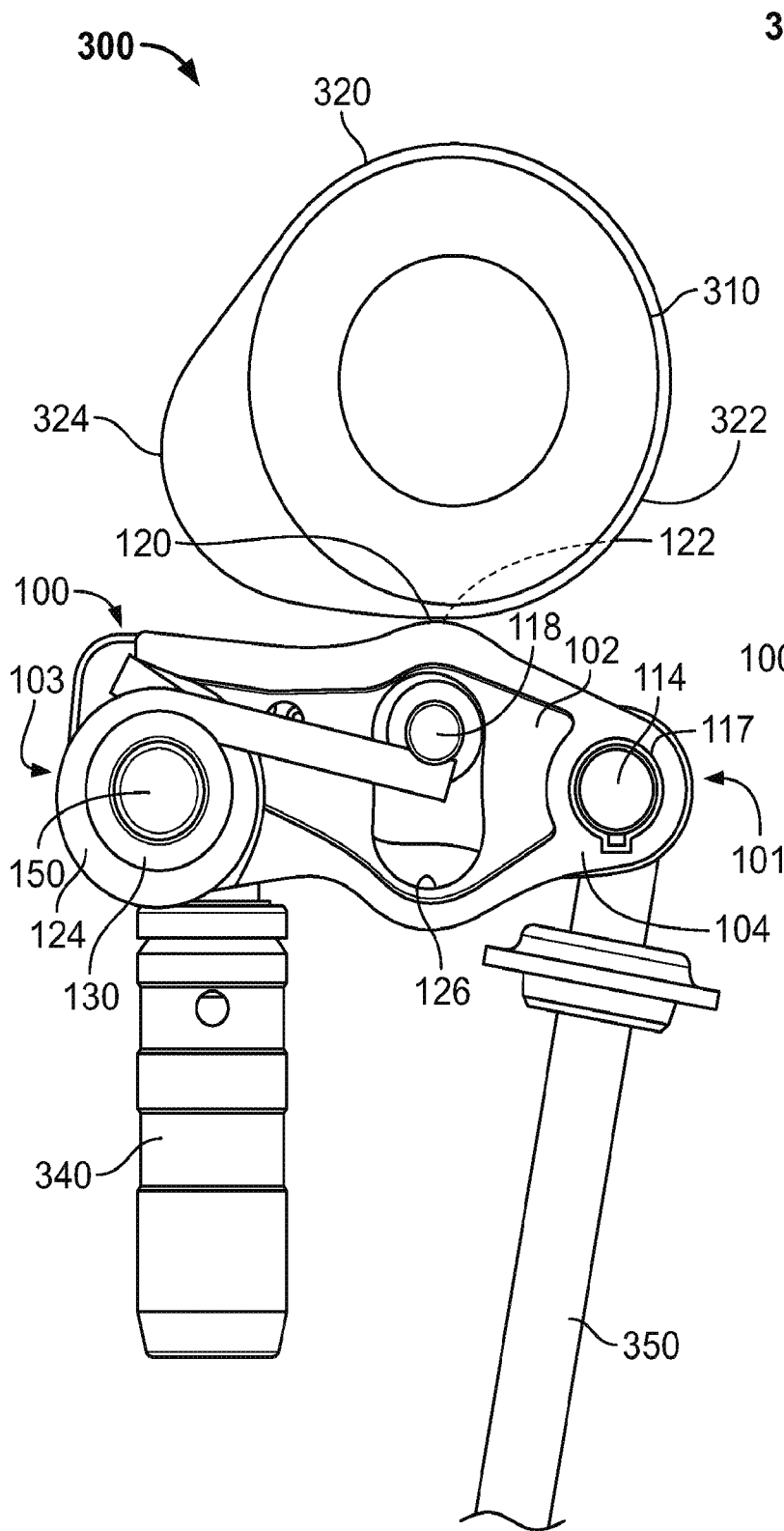


FIGURE 3

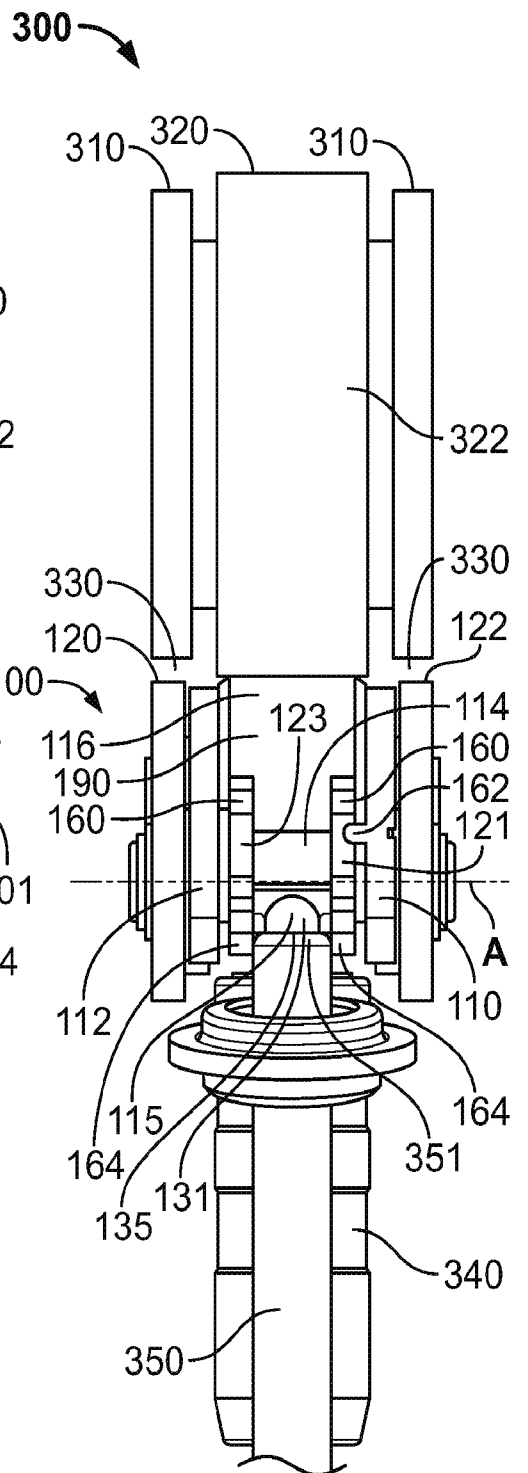


FIGURE 4

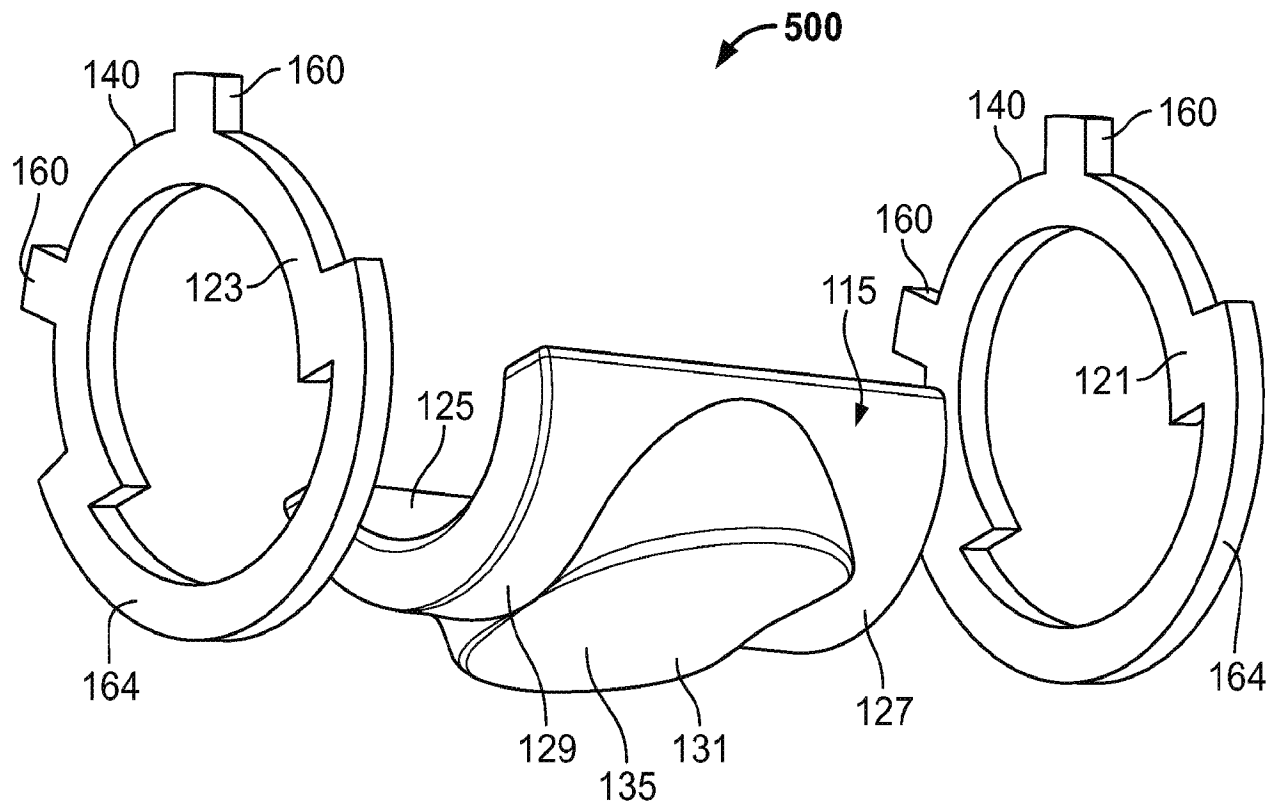


FIGURE 5

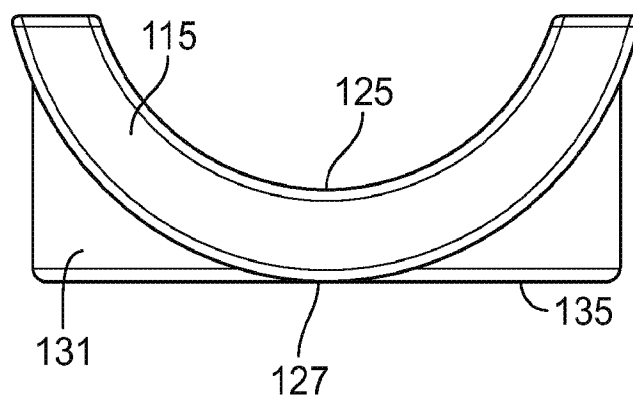
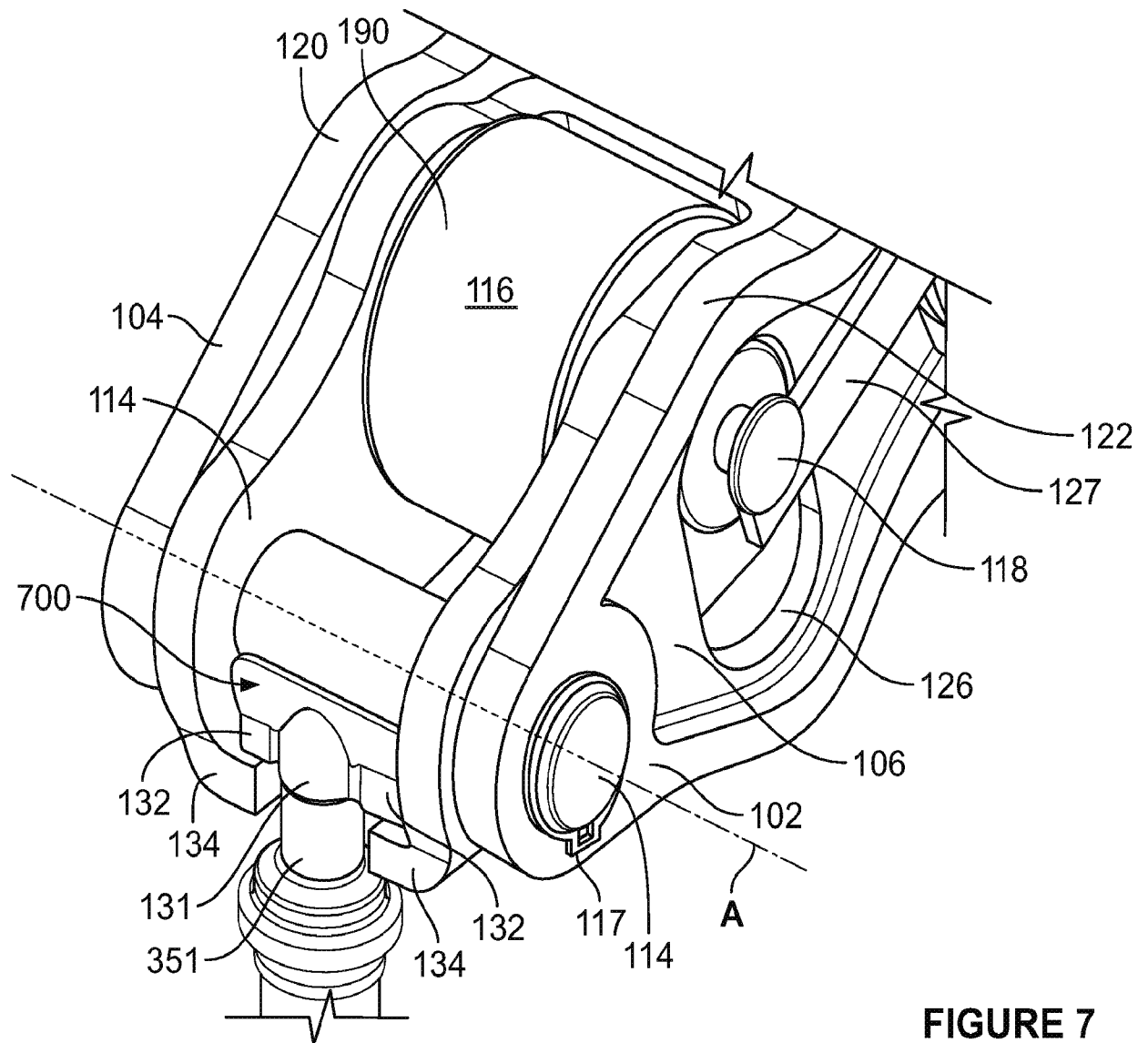
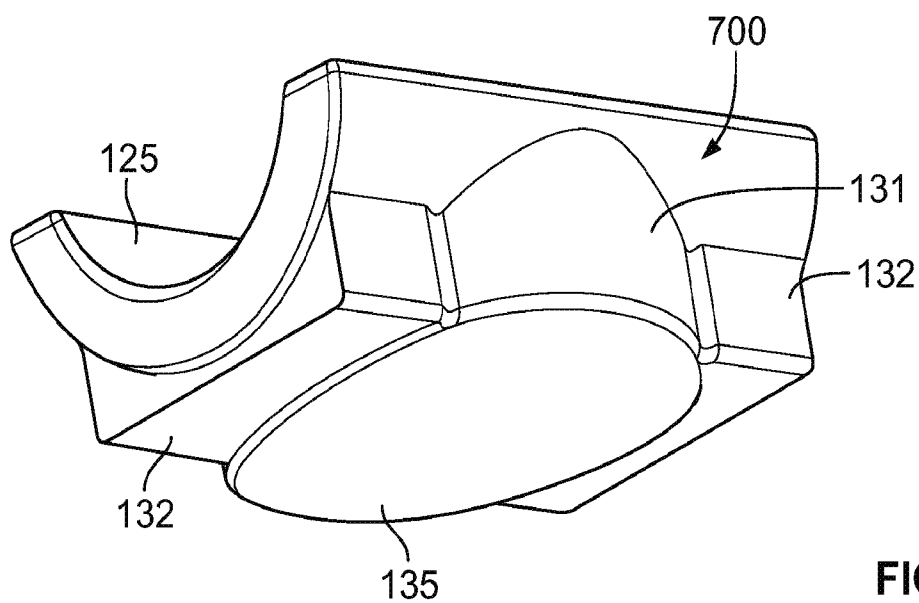


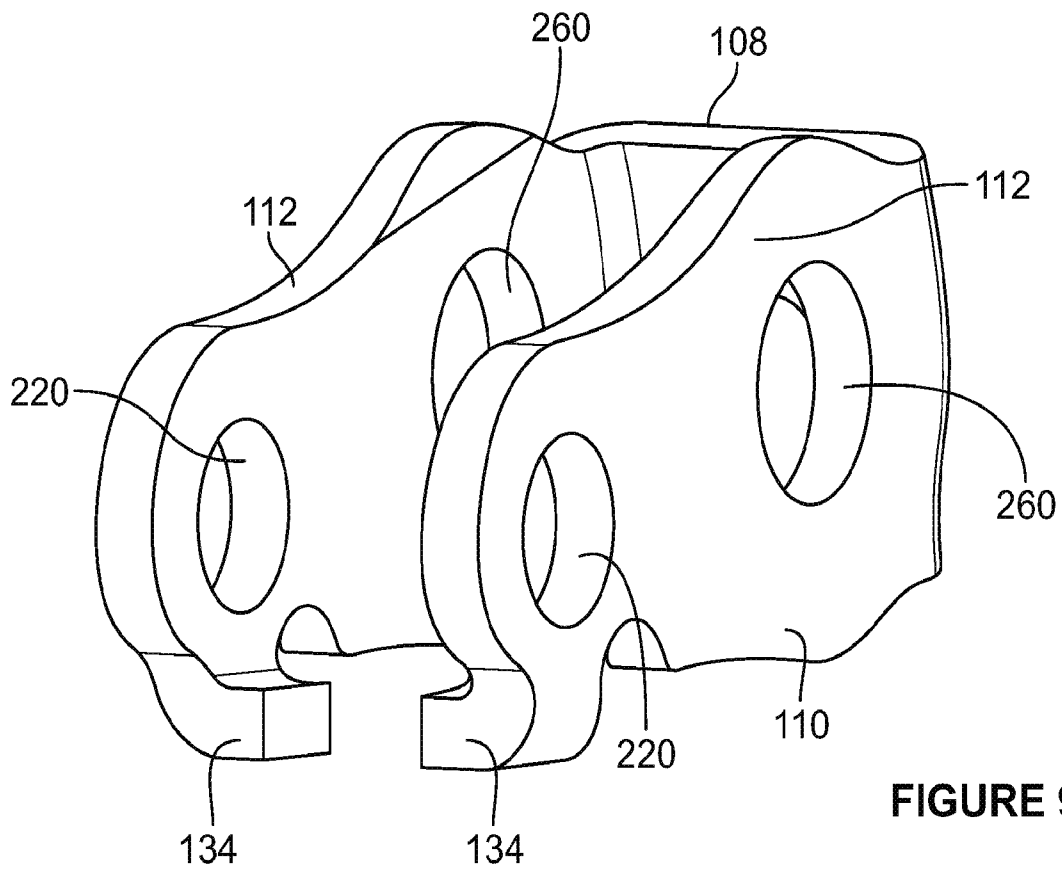
FIGURE 6



**FIGURE 7**



**FIGURE 8**



**FIGURE 9**

## INTERNATIONAL SEARCH REPORT

International application No  
PCT/US2012/034494

## A. CLASSIFICATION OF SUBJECT MATTER

INV. F01L1/18 F01L1/46 F01L13/00  
ADD.

According to International Patent Classification (IPC) or to both national classification and IPC

## B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)  
F01L

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

EPO-Internal, WPI Data

## C. DOCUMENTS CONSIDERED TO BE RELEVANT

| Category* | Citation of document, with indication, where appropriate, of the relevant passages                            | Relevant to claim No. |
|-----------|---------------------------------------------------------------------------------------------------------------|-----------------------|
| X         | DE 10 2009 041367 A1 (SCHAEFFLER TECHNOLOGIES GMBH [DE])<br>24 March 2011 (2011-03-24)<br>the whole document  | 14-18                 |
| A         | -----<br>the whole document                                                                                   | 1,9,19,<br>20         |
| A         | DE 10 2007 025182 A1 (SCHAEFFLER KG [DE])<br>4 December 2008 (2008-12-04)<br>the whole document               | 1,9,14                |
| X         | -----<br>US 2001/023672 A1 (CECUR MAJO [US])<br>27 September 2001 (2001-09-27)<br>the whole document<br>----- | 14                    |



Further documents are listed in the continuation of Box C.



See patent family annex.

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Date of the actual completion of the international search

25 June 2012

Date of mailing of the international search report

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Authorized officer

Klinger, Thierry

# INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

PCT/US2012/034494

| Patent document<br>cited in search report | Publication<br>date | Patent family<br>member(s) | Publication<br>date |
|-------------------------------------------|---------------------|----------------------------|---------------------|
| DE 102009041367 A1                        | 24-03-2011          | NONE                       |                     |
| DE 102007025182 A1                        | 04-12-2008          | NONE                       |                     |
| US 2001023672 A1                          | 27-09-2001          | NONE                       |                     |