April 30, 1935. E. D. GIBSON 1,999,992

DUPLICATE BILL STAMPER AND CUTTER

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Fig. 1.

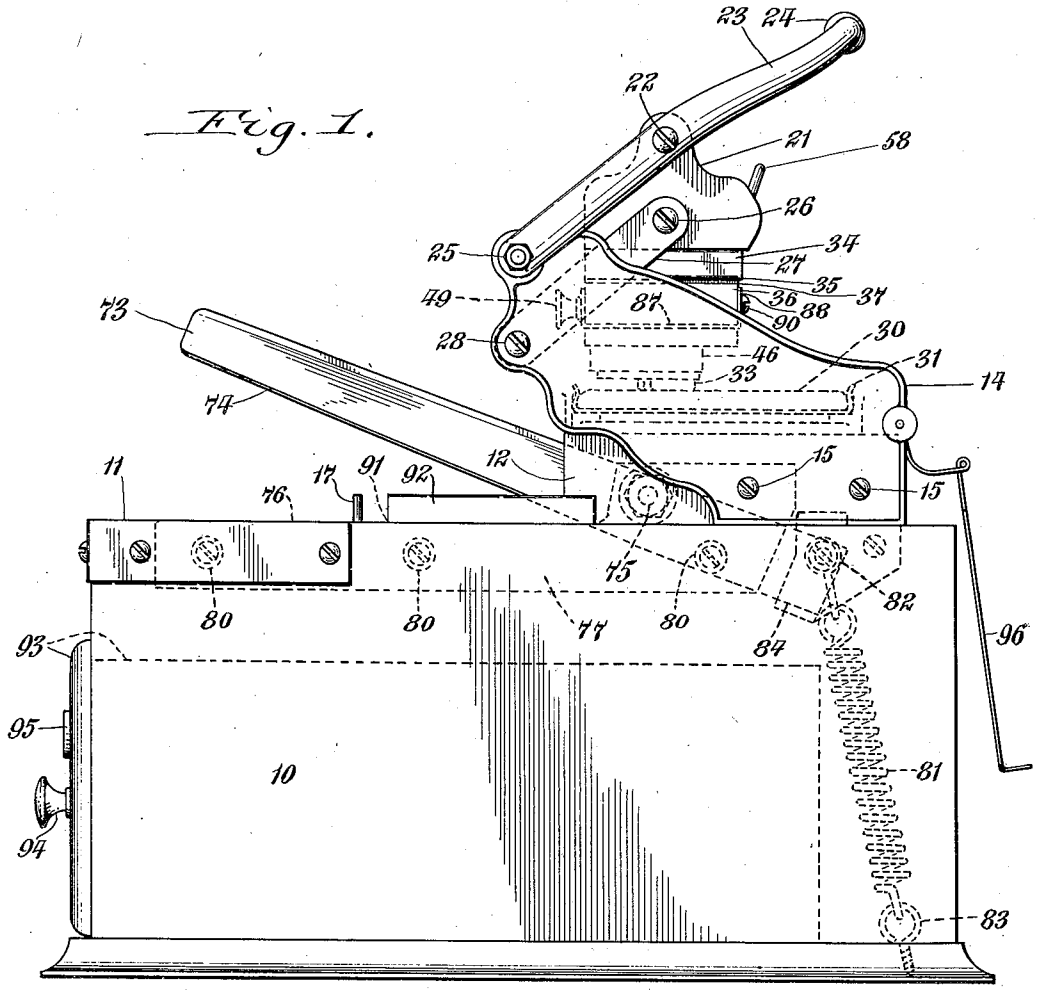


Fig. 6.

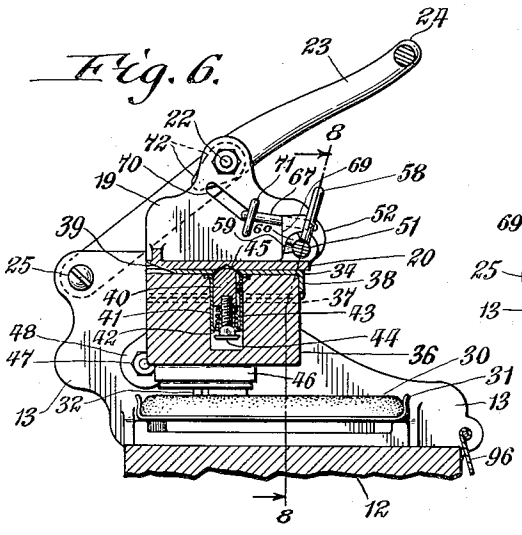
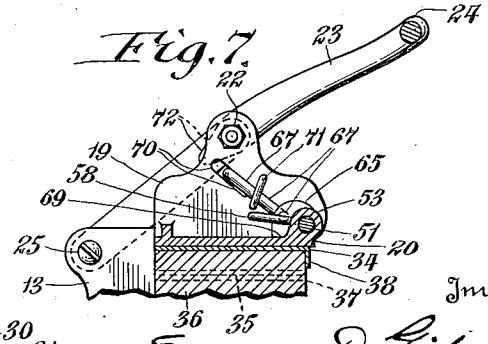


Fig. 7.



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DUPLICATE BILL STAMPER AND CUTTER

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Fig. 2.

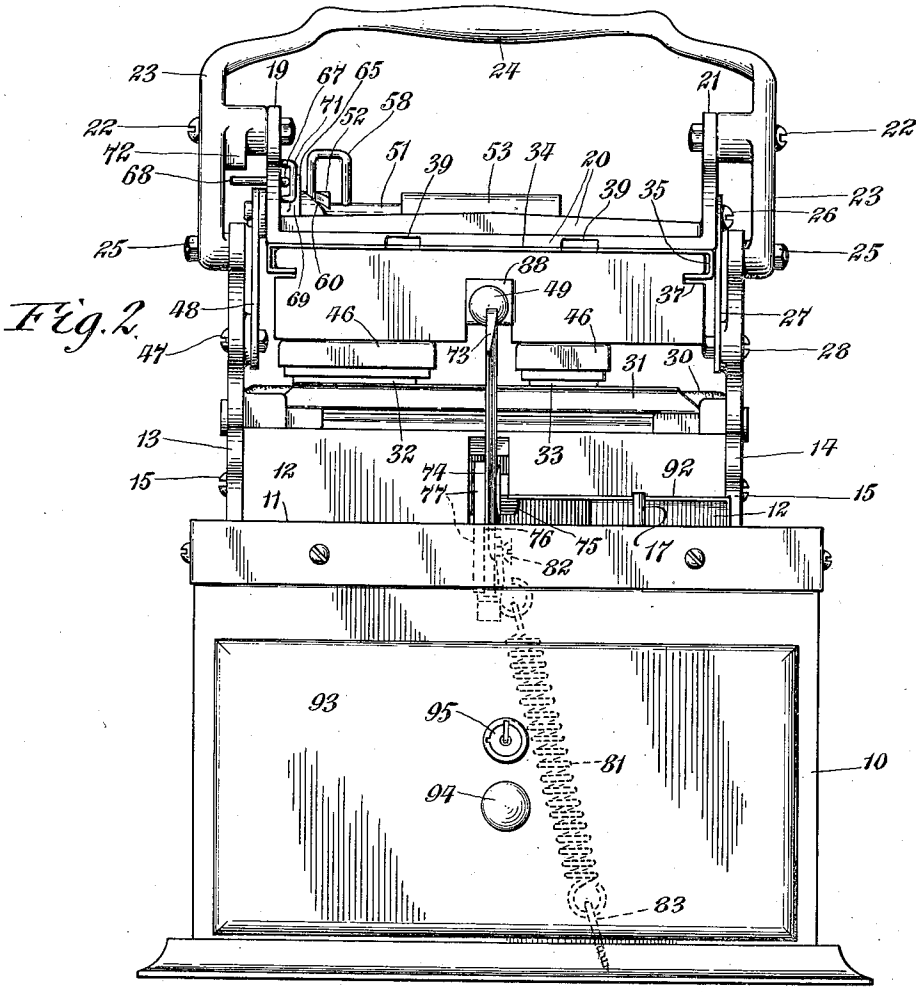
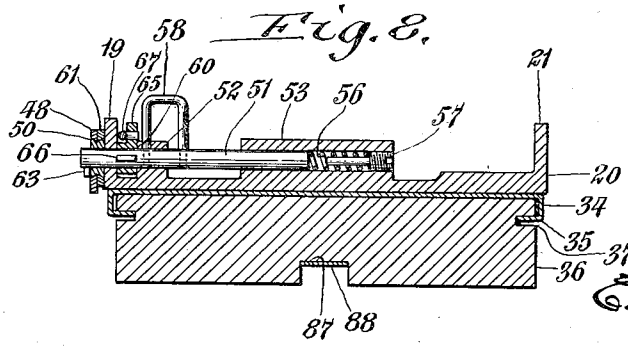


Fig. 8.



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DUPLICATE BILL STAMPER AND CUTTER

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Fig. 3.

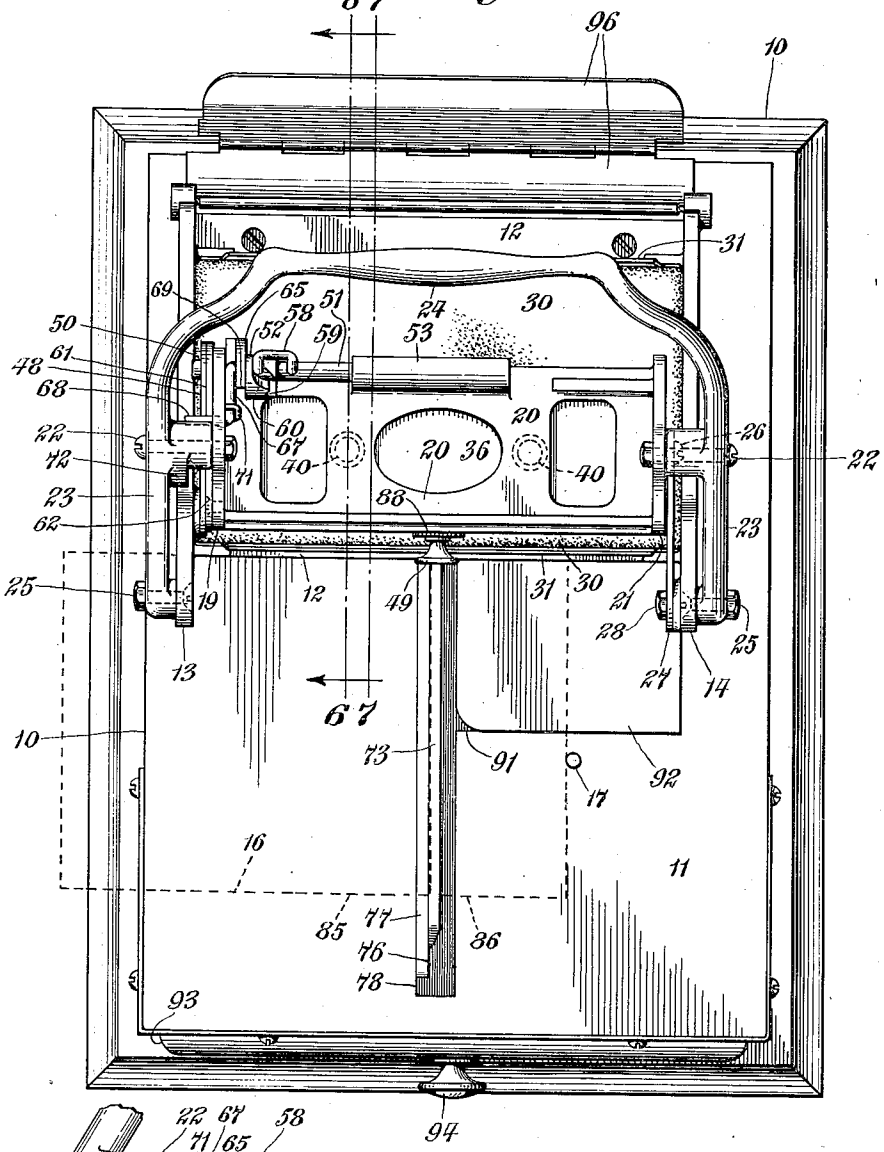
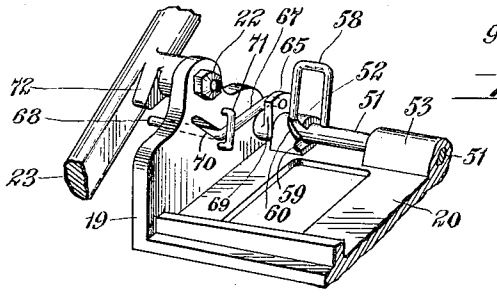


Fig. 9.



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DUPLICATE BILL STAMPER AND CUTTER

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Fig. 4.

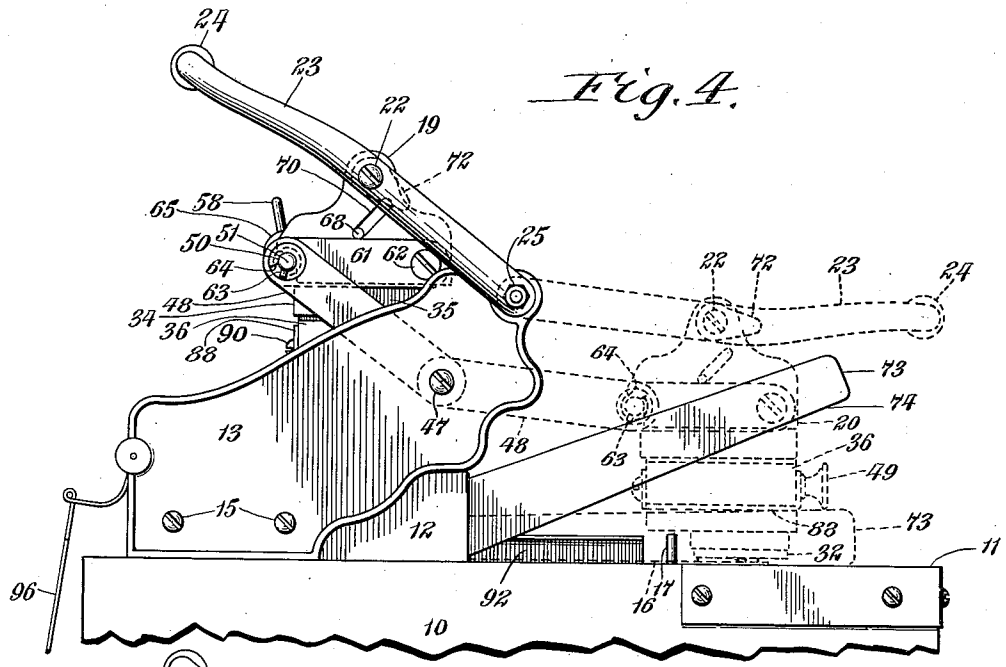
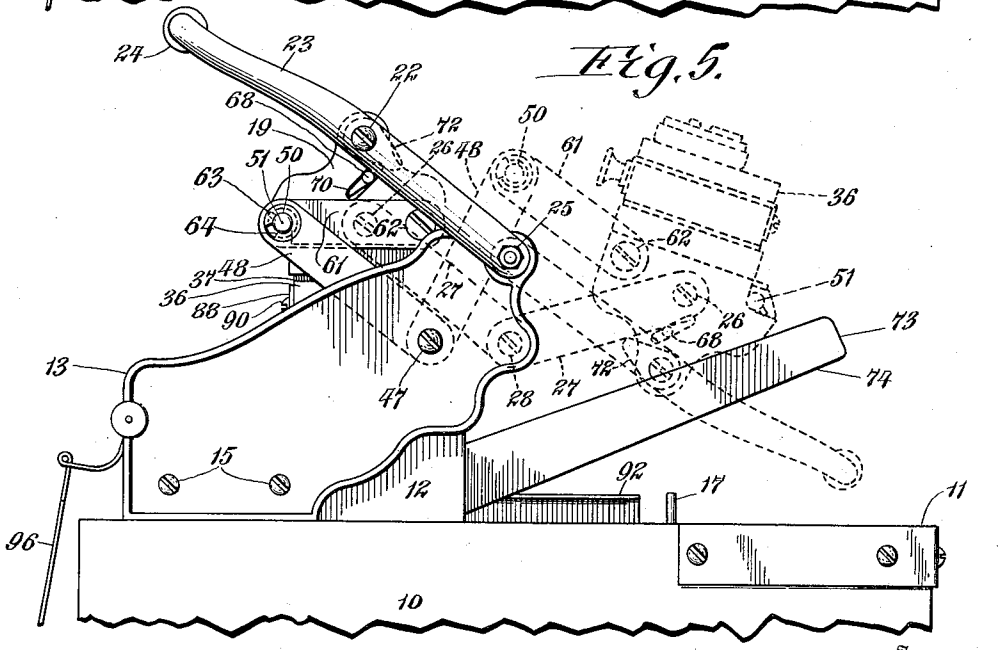


Fig. 5.



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Patented Apr. 30, 1935 1,999,992

UNITED STATES PATENT OFFICE

1,999,992

DUPLICATE BILL STAMPER AND CUTTER

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Application February 21, 1930, Serial No. 430,168

14 Claims. (Cl. 101—301)

This invention relates to a device for simultaneously printing or stamping, independently, the body and the stub of a duplicate bill, commonly known as a "coupon" and simultaneously cutting off the stamped stub and collecting the same in a drawer for purposes of filing.

One object of this invention is to provide a rapid and convenient means of turning the printing block face up, when desired, so as to permit of inspecting, cleaning, repairing or altering the type of the same.

A further object of the invention is to provide a convenient means of removing the printing block which carries its companion pair of type faces and replacing the same by a printing block carrying another pair of type faces.

Another object of the invention is to provide a cutting knife which will continuously operate sharply and smoothly even when the parts have become considerably worn through use.

In the accompanying drawings:

Fig. 1 is a right side elevation of the invention shown in position ready for the insertion of a duplicate bill.

Fig. 2 is a front elevation thereof.

Fig. 3 is a top plan thereof.

Fig. 4 is a fragmentary left-side elevation of the invention, showing the printing-cutting position in dotted lines.

Fig. 5 is a similar, fragmentary, left-side elevation, but with the actuating parts thrown into different position, whereby, as the printing head is moved forwardly into the dotted line position, the printing block is turned face up.

Fig. 6 is a fragmentary, vertical, transverse section of the invention taken on line 6—6, Fig. 3.

Fig. 7 is a similar section, indicated by line 7—7, Fig. 3 but with the parts set (as in Fig. 5) for reversal of the printing block.

Fig. 8 is a vertical, longitudinal section through the printing head and block, taken on line 8—8, Fig. 6.

Fig. 9 is a fragmentary perspective of the parts controlling the movement of the printing head.

In the following description, similar characters of reference indicate like parts in the several figures of the drawings.

When debit statements or bills are sent out at the beginning of each month by public utilities corporations, it is customary to print these bills upon a single, stiff, mailable card or a single, flexible sheet with the important data written thereon in duplicate, i. e., on the main body of the card and also on the stub thereof, which two parts in combination constitute what may be denominated a duplicate bill. When this duplicate bill is returned from the consumer with a payment by an appropriate check or money order to the corporation, each of the two parts of said bill are stamped suitably with, for instance, the words "received" etc. This stamping, in effect, changes the duplicate bill into a duplicate receipt, the stub thereof being cut off and retained by the corporation, while the main body of the duplicate receipt is sent out to the consumer as a notification or acknowledgement to him that his bill has been paid and received.

The present invention effects this duplicate stamping and cutting in a more expeditious and satisfactory manner than heretofore, particularly as relates to the operation of either changing the whole printing block carrying the duplicate type face, or of turning said printing block face up so as to permit of conveniently inspecting, cleaning, repairing or altering its type face.

The main frame of this improved duplicate bill stamper and cutter comprises a rectangular, box-like structure 10 provided with a flat upper deck or table 11. Upon the rear part of said table 11 is suitably secured a spreader or base block 12 to the lateral opposite ends of which are secured vertical, longitudinal side frame plates 13 and 14 by means of screws 15 or otherwise. The duplicate bill 6 (indicated by dotted lines in Fig. 3) is laid by the cashier upon said table 11 of the main frame and is pressed rearwardly against the base block 12 and to the right against a suitably located stop pin 17.

The numeral 20 represents a printing head, which extends transversely across the device and has a length somewhat less than the distance between the side frame plates 13 and 14 and is thereby longitudinally receivable between the same. The opposite ends of said printing head 20 extend longitudinally and upwardly above the main portion of said head to form symmetrical end flanges 19 and 21. To said flanges are coaxially pivoted at 22 the two integral arms of a U-shaped, operating lever 23. The central, horizontal, connecting portion 24 of said operating lever is preferably of such suitable form as to be comfortably grasped by the hand of the cashier operating the device.

The lower extremities of the two arms of said operating lever 23 are pivoted coaxially at 25 to the main frame of the device. This pair of pivot centers 25 together with the pair of pivot centers 22 aforesaid constitute a set or pair of primary pivot centers. In operation, the cashier grasps

the middle portion 24 of the U-shaped, operating lever 23 and moves it about its pair of pivots 25 as a fulcrum and thereby carries the printing head 20 along with it.

The right end (see Fig. 1) of said printing head 20 is pivoted at 26 to a main, parallel-motion link 27, the opposite end of which is pivoted at 28 to the main frame. The pivotal centers of the opposite ends of this main link 27 constitute a secondary set of pivot centers. The distance between said centers 26 and 28 which constitute this secondary set of pivot centers is identical with the distance between the centers 22 and 25 of the primary set of pivot centers. The fixed coaxial centers 25 and the main link fixed center 28 all are located in the main frame and lie in one stationary plane which is preferably a vertical, transverse plane as shown. Because of the relationship existing between the pivot distances, it follows that, as the operating lever 23 is actuated, the moving pivots 22 and 26 will be in one common moving plane which will be at all times parallel to said fixed plane providing, however, that the lever 23 is not moved forwardly past the position in which all four of said pivots 25, 28 and 22, 26 lie in one common plane, i. e., into a position corresponding to the dead center position of the combination of primary and secondary pivot centers. In other words, a parallel motion of the printing head 20 is obtained when the operating lever 23 is moved back and forth, so long as the pivots 22 and 26 do not move in front of a transverse vertical plane passing through the pivots 25 and 28.

Arranged in the frame between the vertical end frame plates 13 and 14 of the main frame is a suitable, horizontal, inked pad 30 resting in a container or metal trough 31, as best shown in Figs. 3 and 6. This pad is constructed in the usual manner of a suitable absorbent material and is periodically supplied with ink, and is used to ink the lower printing surfaces of a pair of type faces, viz., the body type face 32 and the stub type face 33 whenever the operating lever is moved to its rearmost position. Said pair of type faces is caused to move downwardly upon said inked pad 30 in a constantly parallel direction relatively thereto, due to the primary and secondary sets of parallel-motion linkage aforesaid, inasmuch as said linkage is connected with and thereby controls the movement of the printing head 20 to which the said type faces 32 and 33 are secured in the following manner:

As best shown in Fig. 8, the printing head 20 has attached thereto, upon its normally lower face, a printing head plate 34 which may either be integral therewith or spot welded, as shown, or otherwise secured thereto. Said plate has its opposite, lateral ends turned first vertically down and then horizontally inward to form a pair of longitudinal guideways 35. Slidable upon said guideways is a printing block 36 which is provided with suitable longitudinal grooves 37 upon its flat end faces which are adapted to engage with the said guideways 35 of the printing head plate 34. Lengthwise movement of the printing block 36 relatively to the printing head 20 is limited by a transverse stop flange 38 (see Fig. 6) which is secured to and depends from the lower rear part of the printing head plate 34. Movement away from or out of engagement with said stop flange 38 is resiliently restrained by means of a pair of spring plungers or resilient detents 40 which are slidably arranged within metal cups 41. Between the bottom wall 42 of each of said cups 41 and the bottom face of its companion detent 40 is disposed a compression spring 43 which tends to move said detent outwardly (normally upwardly), the extreme outward movement being limited by an adjustable limiting screw 44. The nose of each resilient detent 40 is conically pointed and is adapted to be received within or engaged with a detaining notch 45 formed in the lower face of the printing head 20. The engagement of the noses of said resilient detents 40 with their companion detaining notches 45 operate to resiliently restrain longitudinal movement of the printing block 36 relatively to the printing head 20. Said resilient detents also form a resilient power connection between the handle 23 and the coupon being stamped and cut. In other words, the printing face 33 is resiliently and softly pressed down on the coupon, thus preventing the spattering of ink and jar and noise. It will also be noted (see Figs. 3 and 6) that said detents 40 are located inwardly of the outer edges of the printing block 36 which the latter is in its operative position. This provides a construction whereby the printing face 33 is compelled to bear down evenly upon the surface of the coupon, even though the thickness of the same is irregular or uneven (as for instance if it carries a government revenue stamp), and even though the table 11 becomes warped or otherwise misaligned. To initially depress the pair of said resilent detents 40 whenever the printing block 36 in which they are set, is pushed backwards on the guideways 35, the normally lower front portion of the printing head is bevelled at 39 so as to operate in the capacity of a wedge and to push inwardly each of said detents flush with the adjacent flat face of said printing head 20. Further pushing back of said printing block 36 brings the same into contact with the aforesaid stop flange 38, whereupon each of said detents 40 is again free to move outwardly under the influence of its spring 43 and into resilient engagement with the detaining notch 45 aforementioned.

The type faces 32 and 33 may be identical with or different from each other as may be desired for the particular type of bills being handled. Customarily, each type face is cemented into a shallow metallic pan 46 which may be secured in any suitable way to the normally lower, flat face of the printing block 36. When it is desired to have a different cashier operate the device with his own individual printing block, or when, for any other reason, it becomes desirable to use a different set of type face, the whole printing block 36 is removed from the printing head 20 and is replaced by a different printing block having secured thereto a different set of type face. This replacement of the printing block 36 is facilitated by the employment of a convenient knob or handle 49 which is suitably secured to the front face of said block and is adapted to be grasped by the fingers of the person making the exchange of printing blocks. When either or both of the type faces become obsolete or excessively worn, the same may be removed from its printing block and replaced.

Pivoted at 47 to the left side frame plate 13 is a floatable, parallel-motion link 48 whose upper end is floatably pivoted at 50 to the printing head upon a retractive shaft 51 slidably and rotatably arranged horizontally and transversely in the end flange 19 and in the journals 52 and 53 in the printing head 20. The upper floating center 50 of said floatable link 48 and the lower

permanent center 47 thereof constitute a tertiary set of parallel-motion pivot centers, the distance between which is the same as the distance between the primary and secondary sets of pivot centers 22, 25 and 26, 28. Any one pair of these parallel-motion pivot sets is theoretically capable of causing the printing head 20 to move in parallelism with the main frame of the device. Where such a movement is positively desired for an extreme movement of the operating lever 23 in either direction, then if either the secondary or the tertiary pivot set is used in conjunction with the primary pivot set, such a positive movement is not obtained, inasmuch as they would have a dead-center position, as previously described. This means in practice that as the operating lever is moved forwardly, there would be no guarantee in such case as to whether the printing head would continue past the dead center position in parallel motion, as per dotted lines in Fig. 4, or whether it will turn over as in dotted lines in Fig. 5.

When it is desired to bring the type faces 32 and 33 forward and down upon the duplicate bill 16 so as to stamp the same (as illustrated in Fig. 4) it is necessary to retain the parallel motion relationship for all positions of the operating lever 23. This is accomplished by the tertiary set of pivot centers 50, 47 which in such case co-operate with the primary and secondary pivot sets and are arranged to assume the determining control of the printing head 20 when said primary and secondary pivot centers are positioned in their dead center position, which position occurs when all of said primary and secondary pivots lie in one single plane. Of course, said tertiary set of pivot centers 50, 47 also has a dead center position with respect to the primary pivot centers 22, 25, this position occurring when all of said primary and tertiary pivot centers lie in one common plane. However, this dead center plane is not identical with the dead center plane of the primary and secondary sets of pivot centers, and hence no actual or physical dead-center position occurs if all three sets of parallel-motion centers are simultaneously in operation. Thus under these conditions the printing head 20 may be kept at all times parallel to the table 11 of the main frame, irrespective of the position of the operating handle 23. This means in practice that the type faces 32 and 33 are pressed positively and evenly upon the ink pad 30 at the rear end of the stroke of the operating lever 23, and are also pressed positively and evenly upon the duplicate bill 16 at the front end of the stroke of said lever.

With the type faces 32 and 33 lying below and on the underside of the printing head, it would be inconvenient to inspect, clean, repair or alter said type faces. In the present device, this difficulty is avoided by providing an arrangement whereby the printing block 36 is rendered capable of being turned completely upside down and thus the type face exposed face up. This action is obtained as follows:

The floatable parallel-motion link 48 may, when desired, be caused to "float", and thereby rendered totally inoperative, by moving inwardly and longitudinally the aforementioned horizontal, transverse, retractive shaft 51 which (see Fig. 8) is journaled in bearings 52 and 53 and also in the adjacent end flange 19 of the printing head 20. Said shaft is pressed resiliently outwardly by a coaxial, compression spring 56 which bears against the inner end of said retractive shaft 51 and is held in place by an externally threaded, spring plug 57 screwed into the said printing head and bearing against the outer end of said compression spring. Longitudinal movement of said retractive shaft is obtained by manually throwing forward a bifurcated, actuating arm 58 which is suitably secured to said retractive shaft and normally projects upwardly therefrom. As best shown in Figs. 3 and 9, the outer face of this actuating arm rides against a helical cam face 60 whose axis is concentric with said retractive shaft 51 and is suitably formed in the bearing 52 so that, as said actuating arm 58 is moved forwardly, it rides forwardly and inwardly upon said cam face 60 and thereby rotates said retractive shaft and moves the same inwardly and thus out of engagement with the floating center 50 of the floatable, parallel-motion link 48. Said retractive shaft 51 is automatically retained in its retracted position by means of a shallow groove 59 (see Fig. 9) which is formed at the innermost end of the cam face 60 and into which the outer face of the actuating arm 58 is resiliently forced by aforesaid compression spring 56.

Pivoted to said floatable link 48 upon the aforesaid floating pivot center 50 thereof is a distancing link 61 whose forward end is permanently pivoted at 62 to the left-end flange 19 of the printing head 20. Whether or not the retractive shaft 51 passes through and thereby fixes the position of the floating pivot center 50 of the links 48 and 61 relatively to the printing head, said floating center 50 will nevertheless be in accurate alignment with the retractive shaft 51 whenever said printing head 20 is at the rear part of its stroke. This is due to the fact that the primary and secondary sets of parallel-motion linkage are, in this position, considerably off their dead center position. For this reason (and for other reasons which will appear more clearly hereafter) it is desirable that movement of the retractive shaft 51 be only permitted when the printing head is in its extreme rearward position. For this purpose, the outer end of said retractive shaft 51 is provided with a lateral, outwardly projecting retaining finger 63 which (in the positions of Figs. 1–4, 6, 8 and 9) is positioned outside of the outer faces of said floatable and distancing links 48 and 61. With the printing block 20 in its rearward position, as shown in these figures, said retaining finger 63 is enabled to move inwardly through the said floatable and distancing links 48 and 61 by the provision of an arcuate clearance notch 64 which is preferably, though not necessarily, of helical or spiral form to provide a suitable clearance for said finger 63 as the retractive shaft 51 is moved longitudinally inwardly and, at the same time, is rotated, due to the action of the bifurcated actuating arm 58 riding against the aforesaid helical cam face 60.

Riding upon the retractive shaft 51 and located between the end flange 19 and the outer bearing 52 of the printing head 20, is a crank collar 65 which is connected to said retractive shaft by means of a feather key 66 so as to be able to slide thereon and yet be constrained to rotate therewith. Pivoted to the upper part of said crank collar 65 is a connecting rod 67 which is bent sharply horizontally and outwardly to form a controlling finger 68 which passes through a straight, inclined guide slot 70 formed in the end flange 19 of the printing head 20. Said connecting rod 67 (see Fig. 2) lies against the inner face of said end flange 19 and is thereby pre-

vented against lateral outward movement while lateral inward movement is restrained by a bridge 71 made of a U-shaped rod which straddles said connecting rod and has its opposite ends suitably secured to the aforesaid end flange 19. The operating lever 23 is provided with an integral tilting cam 72 which is formed on the inner face of the left arm thereof and projects downwardly from the left pivot 22. The operation of this cam 72 with the aforesaid controlling finger 68 is as follows:

With the printing block in its rearmost position, if the actuating arm 58 is moved forwardly, the crank collar 65 is also rotated, and this throws the connecting rod 67 forward. Inasmuch as the guide slot 70 is inclined, this forward movement of the connecting rod 67 causes a forward and upward movement of the controlling finger 68 along the path caused by said slot 70, arriving finally at the past dead center position shown in Figs. 5 and 7 where the thrust against the end of said finger 68 is carried by the square side 69 of the collar 65. In the meantime and simultaneously therewith, the floatable link 48 and the distancing link 61, although still pivotally connected together at their common floating center 50, have said common floating center disconnected from the printing head 20 due to the inward movement of the retractive shaft 51. If now the operating lever 23 is moved forward, the printing head 20 will initially be only under the control of the primary and secondary linkages having their pivots at 22, 25 and 26, 28 respectively, but in this initial movement, the printing head 20 is under very definite control due to the fact that the stability of said printing head is determined by the distance between the two parallel planes intersecting said primary and second pivot sets, and this distance is, initially, relatively large. This distance, however, diminishes as we approach the dead-center position, where all the primary and secondary pivot sets lie in one single plane. However, just before said printing head 20 arrives at this dead-center position, the aforesaid controlling finger 68 has come into contact with the lower rear face of the tilting cam 72. Thereupon a further movement forward of the operating lever 23 causes said printing head to be definitely forced to move into a certain one of its two, past-dead-center positions, in this case into the position in which the upper pivot 26 of the main link 27 is caused to lag behind the upper pivot 22 of the main lever. This causes the printing head 20 to be turned over or reversed into the position shown by dotted lines in Fig. 5, thereby turning the type faces 32 and 33 face up. This renders convenient the usual daily operation of changing the date which is printed by said type faces, and also the inspection, cleaning, repair or complete replacement of said type faces. When the desired operation upon said type faces has been completed, the operating lever 23 is thrown back to its extreme rearward position, and the actuating arm 58 of the retractive shaft 51 is pushed rearwardly to its initial position. This causes the printing head 20 to thereafter move parallel to the table 11 of the frame of the device as per full and dotted lines in Fig. 4. Thereafter until the actuating arm 58 is again moved, every back stroke of the operating lever 23 will cause the type faces to drop evenly upon the inked pad 30 and pick up a thin even film of ink which, when said operating lever is moved forward, will be pressed upon the two sections of the duplicate bill 16.

A movable knife blade 73 (see Fig. 1), having its left side face arranged flat and perpendicular and its right face bevelled to form a suitable lower cutting or shearing edge 74, is pivoted at 75 to the main frame. This movable knife blade is adapted to shear or cut the duplicate bill 16 between the lower sharpened edge 74 of said knife blade and the upper sharpened edge 76 of a stationary knife blade or longitudinal cutting bar 77. The latter is arranged against the vertical left face or wall of a slot 78 formed in the table 11 of the main frame and is suitably secured to said frame by a number of screws 80 which pass horizontally and transversely through said cutting bar 77 and into the adjacent portion of the main frame. The knife blade 73 is resiliently held in its elevated position by means of a tension spring 81 whose upper end is pivotally connected to said knife blade at 82 and whose lower end is suitably secured to the main frame by an eye screw 83 or otherwise. The upward movement of said knife blade is limited by a stop 84 which is suitably secured to the main frame and is adapted to bear against the lower rear face of said knife blade when the latter is in its extreme elevated position. It will be noticed that said spring 81 is arranged at an oblique angle. This arrangement provides not only for the resilient raising of the knife blade but also for resiliently forcing the same sidewise into clean cutting contact with the cutting bar 77 at all positions of said knife blade. Thus the knife cuts clean and sharp even when the parts are considerably worn after prolonged use.

The cashier lays the duplicate bill 16 on the table 11 of the main frame and pushes it back against the base block 12 and also to the right against the stop pin 17 so that the division line between the body 85 of said duplicate bill and the stub 86 thereof is directly under the left or flat face of the knife blade 73. Thereupon the operating lever 23 is moved forwardly, the actuating arm 58 being in this case in its rearmost position. The central part of the normally lower face of the printing block 36 is provided with a depressing face 87 which is preferably lined with a metal depressing strip 88 whose front and rear ends are bent up against the vertical front and rear adjacent faces of said printing block 36, the rear end being secured by a suitable wood screw 90 and its front end by the screw shank of the knob 49 aforementioned. As the operating lever 23 is moved forwardly, said depressing strip 88 comes into contact with the upper flat edge of the knife blade 73, and a continued forward movement of said operating lever causes said knife blade to be depressed to the dotted line position of Fig. 4. This action operates to move the lower or sharpened edge 74 of said knife blade 73 below the upper or sharpened edge 76 of the cutting bar 77, thereby severing the stub 86 of the duplicate bill 16 from the main body 85 thereof. Prior to this severing of said stub 86, the central and rear part of said stub, constituting the bulk of its weight, has been positioned over a rectangular aperture 91, which is preferably covered over by a raised, stationary roof 92 which is preferably constructed of sheet metal and is suitably secured to the main frame. The severed stub 86 falls of its own unbalanced weight down through said aperture 91 and drops into a drawer 93 which is adapted to collect the accumulated stubs for purposes of filing. This drawer is periodically opened by means of a convenient knob or drawer pull 94 and emptied, and then slid back in place and se-

cured by a suitable flush lock 95 to prevent unauthorized removal of the accumulating stubs.

Simultaneously with this cutting action (caused by the depressing of the knife blade 73) two ink-wetted type faces 32 and 33 have come into contact respectively with the body 85 and the stub 86 of the duplicate bill 16. The stub, type face 33, upon its downward movement, moves out beyond or clears the raised roof 92 and holds the stub 86 down upon the table 11 until the operating lever 23 is raised. Thereupon said stub is free to fall down through aforesaid aperture 91 into the drawer 93 beneath as aforesaid. When the operator is through using the device for the day, he moves the operating lever 23 forward and throws up and forward the double-hinged, metal dust cover 96 so as to protect the inked pad 30 from dust accumulation and ink evaporation.

The operation of the device is as follows:

If the operator, in starting to use the device, desires to use his individual printing block or any other "special" printing block in place of the printing block then in place, he grasps the printing block handle 94, slides out the not required printing block 36 and replaces it with the desired printing block carrying thereon another pair of type faces 32 and 33. If, however, he only desires to inspect, clean, repair or alter the date, etc. of the existing printing block, he merely throws the actuating arm 58 forwardly and also moves the operating lever 23 forwardly, whereupon the said printing block 36 is turned upside down as per dotted lines of Fig. 5 with the type faces 32 and 33 face up. When the desired operation upon said type faces has been completed, he throws the dust cover 96 back out of the way and thereby uncovers the inked pad 30. He then moves the operating lever 23 rearwardly and then throws back the actuating arm 58. Then he locates in proper position the first duplicate bill 16 by placing the same back against the base block 12 and to the right against the stop pin 17, and pulls the operating lever 23 forward to the dotted line position of Fig. 4. This causes the main body 85 and the stub 86 of the duplicate bill 16 to be independently stamped by type faces 32 and 33 respectively. Simultaneously the knife blade 73 has been depressed, and this results in the cutting off of said stub 86, which, when the operating lever 23 is moved back, falls down through the aperture 91 and into the drawer 93 which is periodically emptied of stamped stubs.

I claim as my invention:

1. A duplicate bill stamper and cutter comprising a frame; a printing head carrying a type face; an operating lever pivoted to said frame and to said printing head upon a primary set of pivot centers; a main parallel-motion link also pivoted to said frame and to said printing head but upon a secondary set of pivot centers, so that said lever and said main link have a dead center position; a floatable parallel-motion link likewise pivoted to said frame and printing head but upon a tertiary set of pivot centers, one of which is a floating center; and a distancing link permanently pivoted to the floating center of said floatable link; and optional means for uncoupling the common floating center of said floating and said distancing link.

2. A duplicate bill stamper and cutter comprising a frame; a printing head carrying a type face; an operating lever pivoted to said frame and to said printing head upon a primary set of pivot centers; a main parallel-motion link also pivoted to said frame and to said printing head but upon a secondary set of pivot centers, so that said lever and said main link have a dead center position; and a controlling finger which may be optionally moved so as to cause a definite movement of said printing head on one side of said dead center position when said printing head is moved.

3. A duplicate bill stamper and cutter comprising a frame; a printing head carrying a type face; an operating lever pivoted to said frame and to said printing head upon a primary set of pivot centers; a main parallel-motion link also pivoted to said frame and to said printing head but upon a secondary set of pivot centers, so that said lever and said main link have a dead center position; a floatable, parallel-motion link likewise pivoted to said frame and said printing head but upon a tertiary set of pivot centers, one of which is a floating center; a distancing link permanently pivoted at its opposite ends, one end of which is upon said floating center of said floatable link; a controlling finger which may be optionally moved so as to cause a definite movement of said printing block on one side of said dead center position when said printing head is actuated; and means for optionally moving said controlling finger into either operative or inoperative position and simultaneously either uncoupling or coupling the common floating center of said floatable link and said distancing link.

4. A duplicate bill stamper and cutter comprising a frame; a printing head carrying a type face; an operating lever pivoted to said frame and to said printing head upon a primary set of pivot centers; a main parallel-motion link also pivoted to said frame and to said printing head but upon a secondary set of pivot centers so that said lever and said main link have a dead center position; a floatable parallel-motion link likewise pivoted to said frame and to said printing head, but upon a tertiary set of pivot centers, one of which is permanently on said frame; a distancing link pivoted at its opposite ends to said floatable link and to said printing head; and optional means for preventing movement of said distancing link.

5. A duplicate bill stamper and cutter comprising a frame; a printing head carrying a type face; an operating lever pivoted to said frame and to said printing head upon a primary set of pivot centers; a main parallel-motion link also pivoted to said frame and to said printing head but upon a secondary set of pivot centers so that said lever and said main link have a dead center position; a floatable parallel-motion link likewise pivoted to said frame and to said printing head, but upon a tertiary set of pivot centers, one of which is permanently on said frame; a distancing link pivoted at its opposite ends to said floatable link and to said printing head; and optional means for simultaneously preventing movement of said distancing link and for controlling which of two movements the printing block will make when it arrives near its dead center position.

6. A duplicate bill stamper and cutter comprising a frame; a printing head carrying a type face; an operating lever pivoted to said frame and to said printing head upon a primary set of pivot centers; a main parallel-motion link also pivoted to said frame and to said printing head but upon a secondary set of pivot centers, so that said lever and said main link have a dead center position; and a singly actuated means for controlling the movement of said printing block on both sides of its dead center position so as to cause either a

translatory movement of said printing head or a reversal thereof.

7. A duplicate bill stamper and cutter comprising a frame; a printing head carrying a type face; an operating lever pivoted to said frame and to said printing head upon a primary set of pivot centers; a main parallel-motion link also pivoted to said frame and to said printing head but upon a secondary set of pivot centers, so that said lever and said main link have a dead center position; a floatable parallel-motion link likewise pivoted to said frame and to said printing head but upon a tertiary set of pivot centers, one of which is permanently on said frame and the other of which is floatable on said printing head and is provided with a clearance notch; a distancing link pivoted at one end to said printing head and at its other end to said floating center of said floatable link; a retractive shaft mounted on said printing head and adapted to fix the floating pivot center of said floatable link and said distancing link; and a retaining finger secured to said retractive shaft and adapted to optionally either prevent a movement of said shaft or to pass through said clearance notch and to allow a movement of said shaft when the said floatable and distancing links are in one certain relationship only.

8. A duplicate bill stamper and cutter comprising a frame; a printing head; an operating lever having a cam and pivoted to said frame and said head upon a primary set of pivot centers; a main parallel-motion link also pivoted to said frame and said head but upon a secondary set of pivot centers so as to create a dead center position; a distancing link and a floatable link floatably pivoted together at companion ends and pivoted at their other ends to said head and said frame respectively; a retractive shaft mounted on said head and adapted to fix said common floating center of said floatable and distancing links; a crank collar connected with said retractive shaft; and a controlling finger connected with said crank collar and adapted to bear against said cam on said operating lever and thereby control the past-dead-center movement of the printing head.

9. A duplicate bill stamper and cutter comprising a printing head; a printing block slidably arranged thereunder; a type face arranged upon said printing block; and resilient means for preventing sliding disengagement of said printing block from said printing head, the force of said resilient means being exerted in a direction perpendicular to said type face.

10. A duplicate bill stamper and cutter comprising a printing head having a guideway; a printing block formed to slide upon said guideway; a type face upon said printing block; and means for resiliently preventing sliding disengagement of said printing block from said printing head, the force of said resilient means being exerted in a direction perpendicular to said type face.

11. A duplicate bill stamper and cutter comprising a printing head having a laterally projecting, longitudinal guideway and having a stop; a printing block adapted to slide upon said guideway and be restrained against lateral movement thereby, and to be limited by said stop; a type face upon said printing block; and a resilient detent to prevent sliding disengagement of said printing block, the force of said resilient detent being exerted in a direction perpendicular to said type face.

12. A duplicate bill stamper and cutter comprising a printing head having a detaining notch and an entering cam face; laterally inturned flanges arranged at the opposite longitudinal edges of said head; a printing block, a type face thereon, and a resilient detent arranged in said printing block and adapted to initially be depressed by the entering cam face of said printing head and thereafter to resiliently engage with said detaining notch.

13. A duplicate bill stamper and cutter comprising a frame; a knife blade pivoted on said frame; a printing head movably arranged on said frame and having an entering cam face and a detaining notch; laterally inturned flanges forming a guideway arranged at the opposite longitudinal edges of said printing head; a stop flange arranged at the rear end of said printing head; a printing block adapted to slidably engage with said guideway and to bear against said stop and provided with a depressing face adapted to bear upon said knife blade; and a resilient detent secured to said printing block and adapted to resiliently engage with said detaining notch.

14. A duplicate bill stamper and cutter comprising a printing head having a guideway; a printing block carrying a printing face and adapted to slide in said guideway; and a resilient detent adapted to prevent disengagement of said block from said guideway and exerting its resilient force in a direction perpendicular to said printing face and located inwardly of the outer edges of the printing block when the latter is in its operative position.

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