

(19) World Intellectual Property Organization
International Bureau



(43) International Publication Date
10 August 2006 (10.08.2006)

PCT

(10) International Publication Number
WO 2006/084268 A1

(51) International Patent Classification:
F04D 13/02 (2006.01) **F04D 29/20** (2006.01)

(21) International Application Number:
PCT/US2006/004250

(22) International Filing Date: 3 February 2006 (03.02.2006)

(25) Filing Language: English

(26) Publication Language: English

(30) Priority Data:
60/650,645 4 February 2005 (04.02.2005) US
11/098,336 4 April 2005 (04.04.2005) US

(71) Applicant (for all designated States except US): **SUNDYNE CORPORATION** [US/US]; 14845 W. 64th Avenue, Arvada, CO 80007 (US).

(72) Inventors: **EDWARDS, Stanley, W.**; 12560 W. 60th Avenue, Arvada, CO 80004 (US). **MCGILVREY, Loren, G.**; 2847 W. Deer Creek Trail, Highlands Ranch, CO 80129 (US). **SWENSON, Sheldon**; 10232 W. 77th Place, Arvada, CO 80005 (US).

(74) Agent: **STEPHENSON, Gregory, R.**; Hamilton Sundstrand Corporation, One Hamilton Road, MS 1-1-BC18, Windsor Locks, CO 06090 (US).

(81) Designated States (unless otherwise indicated, for every kind of national protection available): AE, AG, AL, AM, AT, AU, AZ, BA, BB, BG, BR, BW, BY, BZ, CA, CH, CN, CO, CR, CU, CZ, DE, DK, DM, DZ, EC, EE, EG, ES, FI, GB, GD, GE, GH, GM, HR, HU, ID, IL, IN, IS, JP, KE, KG, KM, KN, KP, KR, KZ, LC, LK, LR, LS, LT, LU, LV, LY, MA, MD, MG, MK, MN, MW, MX, MZ, NA, NG, NI, NO, NZ, OM, PG, PH, PL, PT, RO, RU, SC, SD, SE, SG, SK, SL, SM, SY, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, YU, ZA, ZM, ZW.

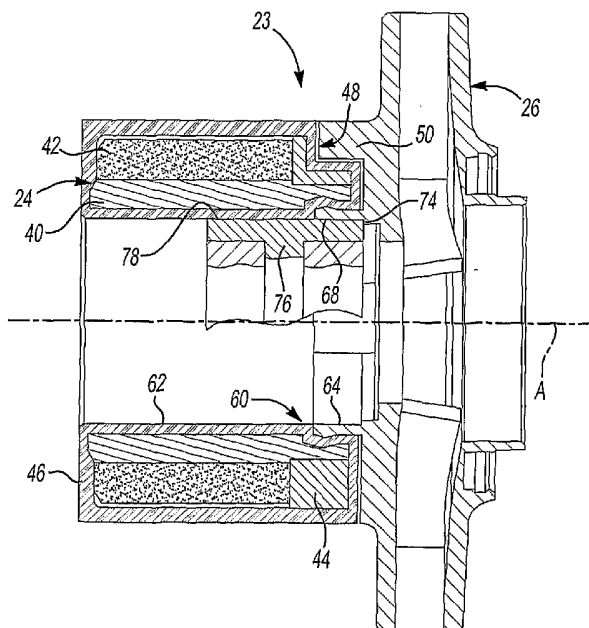
(84) Designated States (unless otherwise indicated, for every kind of regional protection available): ARIPO (BW, GH, GM, KE, LS, MW, MZ, NA, SD, SL, SZ, TZ, UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, MD, RU, TJ, TM), European (AT, BE, BG, CH, CY, CZ, DE, DK, EE, ES, FI, FR, GB, GR, HU, IE, IS, IT, LT, LU, LV, MC, NL, PL, PT, RO, SE, SI, SK, TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW, ML, MR, NE, SN, TD, TG).

Published:

- with international search report
- before the expiration of the time limit for amending the claims and to be republished in the event of receipt of amendments

[Continued on next page]

(54) Title: TWO PIECE SEPARABLE IMPELLER AND INNER DRIVE FOR PUMP



(57) Abstract: A separable impeller and inner drive are constructed from different plastics. The inner drive includes a metallic drive ring that receives drive lugs extending from the impeller. A bushing directly supports the inner drive and impeller and maintains extensions from the impeller in engagement with an annular groove in the inner drive.



WO 2006/084268 A1



For two-letter codes and other abbreviations, refer to the "Guidance Notes on Codes and Abbreviations" appearing at the beginning of each regular issue of the PCT Gazette.

DESCRIPTION

TWO PIECE SEPARABLE IMPELLER AND INNER DRIVE FOR PUMP

This application claims priority to U.S. Provisional Patent Application No. 60/650,645, filed February 4, 2005.

5 TECHNICAL FIELD

This invention relates to a magnetically driven chemical pump having a two piece, separable impeller and inner drive.

Magnetic drive centrifugal pumps include a wet portion, which contains the process fluid that is being pumped, and a dry portion having a drive, which provides
10 power to the pumped fluid. The dry portion is exposed only to the atmosphere surrounding the pump. In one typical magnetic drive design, an inner and outer drive are separated by a plastic containment shell, which prevents the pumped fluid from escaping to the environment. The outer drive, which is usually driven by an electric motor, is located in the dry portion and magnetically drives the inner drive in
15 the wet portion that is attached to a pump impeller. Since magnetic drive pumps are seal-less, they are often selected to pump very acidic or caustic process fluids, such as hydrochloric acid, nitric acid, and sodium hypochlorite.

The inner drive, which includes magnets, and impeller are typically integrally formed with one another. A plastic coating surrounds the magnets
20 preventing the magnets from corroding and the pump from failing. Typically, the impeller is constructed from a fiber reinforced plastic to provide strength, which dictates that the plastic encapsulating the magnets be formed from the same material. However, the reinforcing fibers permit the process fluid to wick into the area with the magnets thereby permitting corrosion. Accordingly, it is desirable to use a non-
25 reinforced plastic to encapsulate the magnets.

Inner drive assemblies have been proposed that have an impeller that is separable from the inner drive. In one example arrangement, a pentagonal extension from the impeller is received in a corresponding shaped aperture in the inner drive to permit the transfer of torque from the inner drive to the impeller. The coupling

between the impeller and the inner drive typically causes cold flowing of the plastic, which undesirably distorts the plastic coating.

The separable impeller and inner drive have been secured by various locking features. In one example, multiple pins are used to retain the impeller and inner drive. In another arrangement, flexible prongs are received by the inner drive. A bushing directly supports the locking feature provided by the impeller, but does not directly support the inner drive. Instead, the inner drive is supported by the impeller requiring the tolerances between the inner drive and impeller interface to be tightly maintained to provide desired alignment between the bushing and inner drive.

What is needed is an improved two piece, separable impeller and inner drive that addresses the problems described above.

DISCLOSURE OF INVENTION

The present invention includes an inner drive assembly for a magnetic pump. The inner drive assembly includes an inner drive having magnets. The inner drive is rotatable about an axis and includes an inner drive inner surface. An impeller is secured to the inner drive by a locking feature that extends axially from the impeller. The impeller has an impeller inner surface provided by the locking feature. A bushing engages the inner surfaces and directly supports the inner drive and impeller.

The inner drive includes an outer surface and a drive pocket extending to the outer surface. A drive lug extends from the impeller and is received in the drive pocket for transmitting torque from the inner drive to the impeller. The arrangement of the drive pocket relative to the outer surface is less likely to trap the process fluid, which is desirable during service of the pump.

The inner drive includes a non-reinforced plastic for encapsulating the magnets. The impeller is constructed from a fiber reinforced plastic. A metal drive ring, which defines the drive pockets, is mounted on a metal yoke that supports the magnets. The drive ring is metallic and transfers torque to the impeller without deforming the non-reinforced plastic on the inner drive. The yoke is radially spaced from the locking feature to provide rigidity in the area of the locking feature to better maintain engagement between the inner drive and impeller.

Accordingly, the present invention provides an improved two piece, separable impeller and inner drive.

These and other features of the present invention can be best understood from the specification and drawings, the following of which is a brief description.

5 BRIEF DESCRIPTION OF DRAWINGS

Figure 1 is a cross-sectional view of a magnetically driven sealless centrifugal pump.

Figure 2 is a cross-sectional view of the inventive inner drive assembly having a separable inner drive and impeller.

10 Figure 3A is a perspective view of the inner drive.

Figure 3B is a perspective view of the impeller.

Figure 4 is a perspective view of a drive ring used in the inventive inner drive assembly.

Figure 5 is an elevational view of the inner drive assembly.

15 Figure 6 is an enlarged cross-sectional view of the inner drive assembly indicated at circle 6 in Figure 5.

BEST MODE FOR CARRYING OUT THE INVENTION

A magnetically driven sealless centrifugal pump assembly 10 is shown in Figure 1. The assembly 10 includes a motor 12 driving a pump 14. Specifically, the
20 motor 12 rotationally drives an outer drive 18 with a drive shaft 16. The outer drive 18 is supported within a housing 20 that defines a dry portion 22.

An inner drive assembly 23 includes an inner drive 24 and an impeller 26. The inner drive assembly 23 is mounted on a stationary shaft 28 and rotatable about an axis A. The inner drive assembly 23 is arranged within a containment shell 30
25 and a casing 32 that provide a wet portion 34. The wet portion 34 contains a process fluid that is pumped by the impeller 26 from an inlet 36 to an outlet 38. The inner drive 24 is rotationally driven in response to rotation of the outer drive 18, as is well known in the art.

Referring to Figure 2, the inner drive 24 includes a yoke 40 supporting
30 multiple magnets 42 arranged circumferentially about the yoke 40. A spacer or

drive ring 44 is mounted on the yoke 40 in an interference fit adjacent to the magnets 42. The yoke 40 is typically magnetic and the drive ring 44 is typically constructed from a non-magnetic metallic material.

5 The inner drive 24 is encapsulated in a non-reinforced plastic coating 46 to protect the magnet 42 and other inner drive components from the process fluid. Since the impeller 26 is separable from the inner drive 24, the impeller 26 may be constructed from a fiber reinforced plastic to provide structural rigidity to the impeller 26. The inner drive 24 and impeller 26 include faces 67 and 69 adjacent to one another. (See Figure 3).

10 Referring to Figures 2 and 3A and B, the drive ring 44 includes circumferentially spaced drive ring pockets 48. The drive ring pockets 48 receive drive lugs 50 axially extending from the impeller 26. More specifically, the drive ring 44 provides cavities 47 that define the drive ring pockets 48, which is best shown in Figure 4.

15 Preferably, the drive ring pockets 48 extend to an outer surface 52 of the inner drive 24, which prevents process fluid from becoming trapped within the drive ring pockets 48. Trapped process fluid, which is typically very corrosive, can pose a danger to technicians servicing the pump assembly 10.

20 Referring to Figures 5 and 6, an end 54 of the drive lugs 50 preferably extends adjacent to and in close proximity with the plastic coating 46 in the drive ring pockets 48, which prevents excess fluid from collecting within the drive ring pocket 48. The drive lugs 50 include spaced apart sides 56 that are in close proximity to lateral sides 58 provided by the drive ring 44. Preferably, the sides 56 and 58 are parallel to a radius R extending from the axis A to ensure efficient torque transmission from the inner drive 24 to the impeller 26 and minimize deformation of the coating 46.

25 The inner drive 24 includes an inner drive inner surface 62, and the impeller 26 includes an impeller inner surface 64. Complimentary locking feature 60 interlock the inner drive 24 and impeller 26. Specifically, multiple extensions 68 axially extending from the impeller 26 cooperate with an annular groove 66, or individual groove segments or pockets, spaced inwardly from the inner drive inner surface 62. A protuberance 70 on the extensions 68 extend radially outwardly and

30

are received by the annular groove 66. The impeller inner surface 64 is provided by the extension 68. The inner surfaces 62 and 64 are generally cylindrical in shape and are aligned with one another so that a common line may extend along the inner surfaces 62 and 64. The inner drive includes a key 72 that rotationally locates a
5 bushing 76. Alternatively, an interference fit can be used between the bushing 76 and inner drive 24. The bushing 76 supports both the inner drive 24 and impeller 26 by engaging the inner surfaces 62 and 64 with an outer surface 78 of the bushing 76. As a result, the inner drive 24 and impeller 26 need not be aligned relative to one another, which would require tight tolerance, but are instead aligned and supported
10 directly by the bushing 76.

The bushing 76 is axially located against a shoulder 74 on the impeller 26. The bushing 76 maintains the extension 68 radially and maintains engagement with the annular groove 66.

Although a preferred embodiment of this invention has been disclosed, a
15 worker of ordinary skill in this art would recognize that certain modifications would come within the scope of this invention. For that reason, the following claims should be studied to determine the true scope and content of this invention.

CLAIMS

1. An inner drive assembly for a magnetic pump comprising:
an inner drive having a magnet and rotatable about an axis, the inner drive including an inner drive inner surface;
an impeller removably secured to the inner drive by a locking feature that
5 extends axially from the impeller, the impeller having an impeller inner surface provided by the locking feature; and
a bushing in engagement with the inner surfaces, the bushing supporting the inner drive and the impeller.
2. The inner drive assembly according to claim 1, wherein the inner drive is encapsulated in an unbroken, non-reinforced plastic.
3. The inner drive assembly according to claim 1, wherein the inner surfaces are arcuate.
4. The inner drive assembly according to claim 3, wherein the inner surfaces are aligned with one another to include a common axially ending line lying along the inner surfaces.
5. The inner drive assembly according to claim 4, wherein the inner surfaces provide a generally cylindrical surface mating with an outer surface of the bushing.
6. The inner drive assembly according to claim 1, wherein the inner drive includes a groove spaced radially outwardly from the inner drive inner surface, the locking feature provided by an extension having a protuberance that is received in and complementary to the groove.

7. An inner drive assembly for a magnetic pump comprising:
an inner drive rotatable about an axis, the inner drive having a outer surface
and a drive pocket ending to the outer surface; and
an impeller including an axially extending drive lug removably received in
5 the drive pocket for transmitting torque from the inner drive to the impeller.
8. The inner drive assembly according to claim 7, wherein the drive pocket and
drive lug include mating sides generally parallel with a radius extending through the
axis.
9. The inner drive assembly according to claim 7, wherein the drive lug extends
generally to the outer surface.
10. The inner drive assembly according to claim 7, wherein the inner drive
includes a metallic drive ring defining the drive pocket, and a plastic coating
arranged between the drive ring and drive lug.
11. The inner drive assembly according to claim 10, wherein the inner drive
includes a yoke supporting magnets and the drive ring mounted on the yoke
proximate to the magnets.
12. The inner drive assembly according to claim 11, wherein a locking feature
extends axially from the impeller and is received by a complementary surface on the
inner drive, yoke spaced radially from the locking feature, and a plastic coating
covering the yoke providing the complementary surface.
13. The inner drive assembly according to claim 7, wherein an axial end of the
drive lug extends to a position adjacent to the inner drive.

14. An inner drive assembly for a magnetic pump comprising:
an inner drive including a magnet encapsulated by a non-reinforced plastic;
and
an impeller removably coupled to the inner drive by a locking feature, the impeller constructed of a fiber reinforced plastic.
15. The inner drive assembly according to claim 14, wherein the inner drive is rotatable about an axis and includes a yoke supporting the magnet, the yoke spaced radially from the locking feature with the non-reinforced plastic arranged between the yoke and locking feature.
16. The inner drive assembly according to claim 15, wherein the locking feature extends axially from the impeller.
17. The inner drive assembly according to claim 15, wherein a drive ring is mounted on the yoke, the drive ring defining a drive pocket receiving a drive lug that extends axially from the impeller, the non-reinforced plastic arranged between the drive ring and drive lug.
18. The inner drive assembly according to claim 17, wherein the drive ring and drive lug include mating sides generally parallel with a radius extending through the axis.

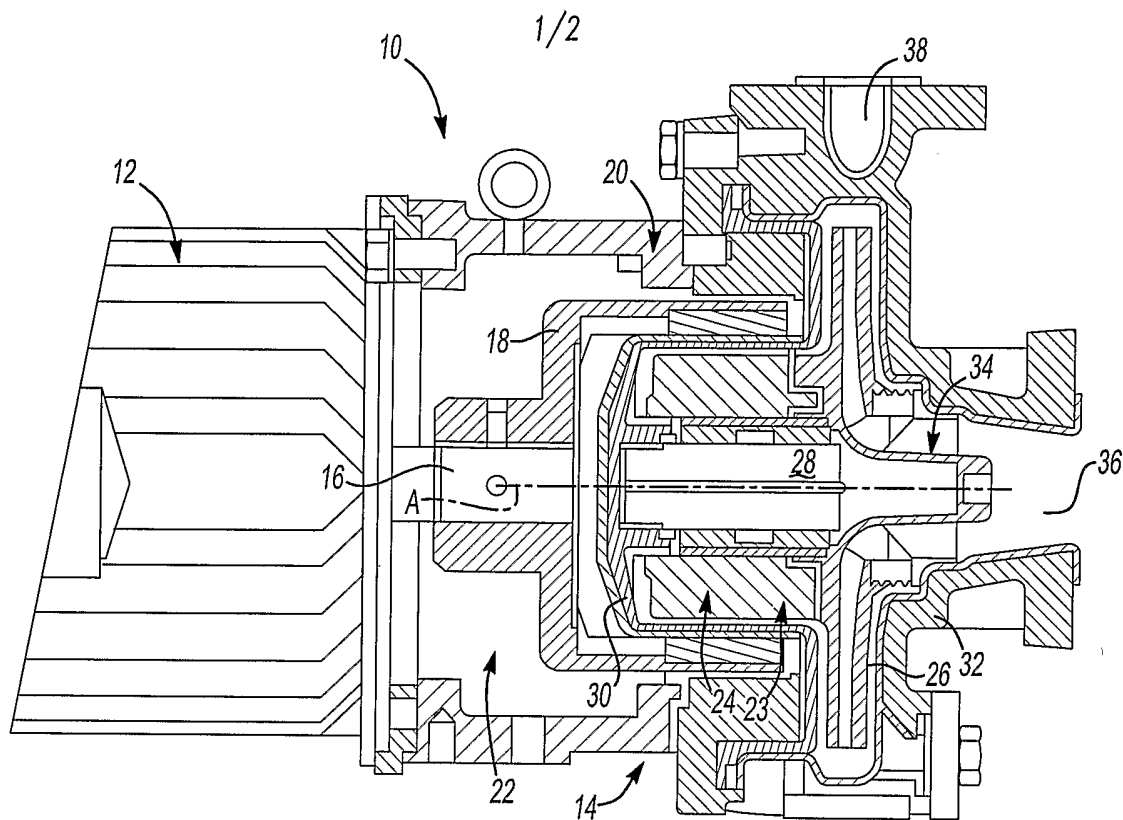


Fig-1

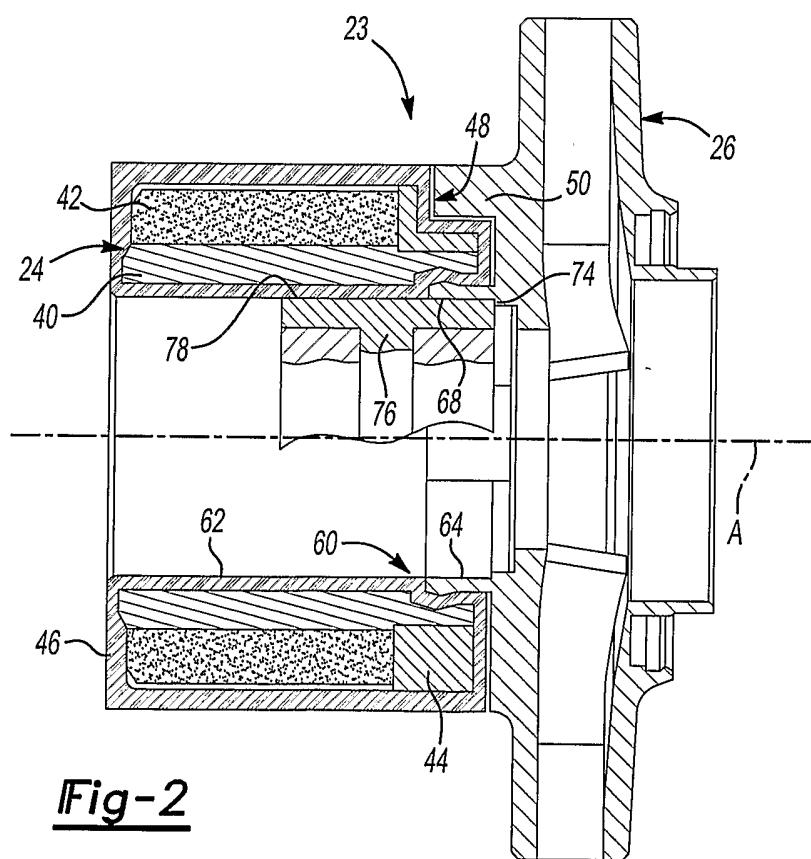
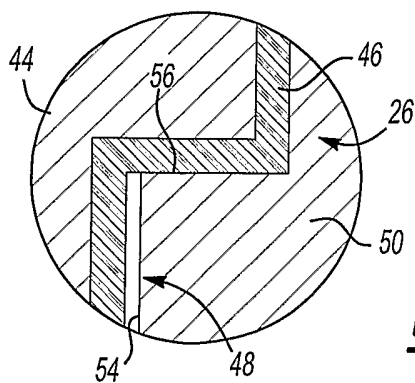
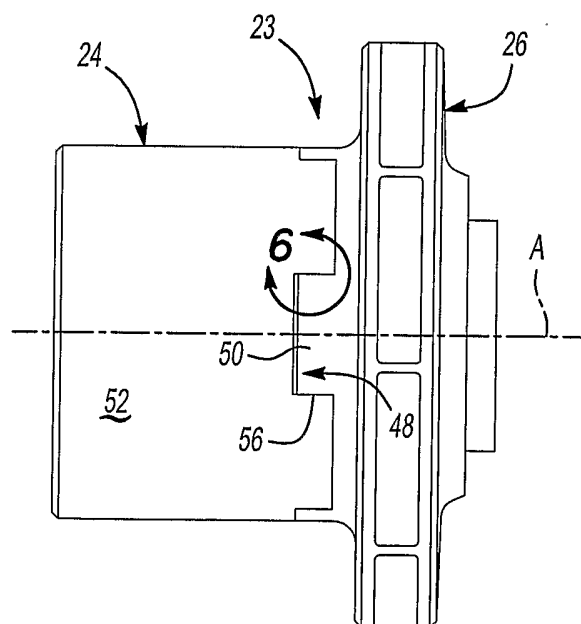
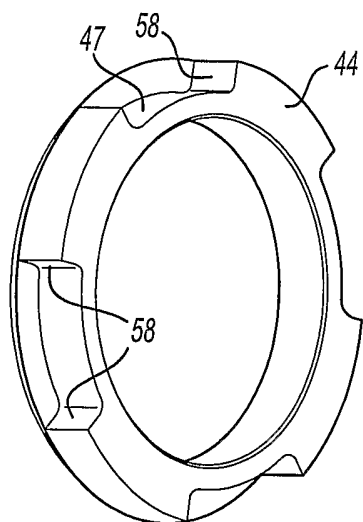
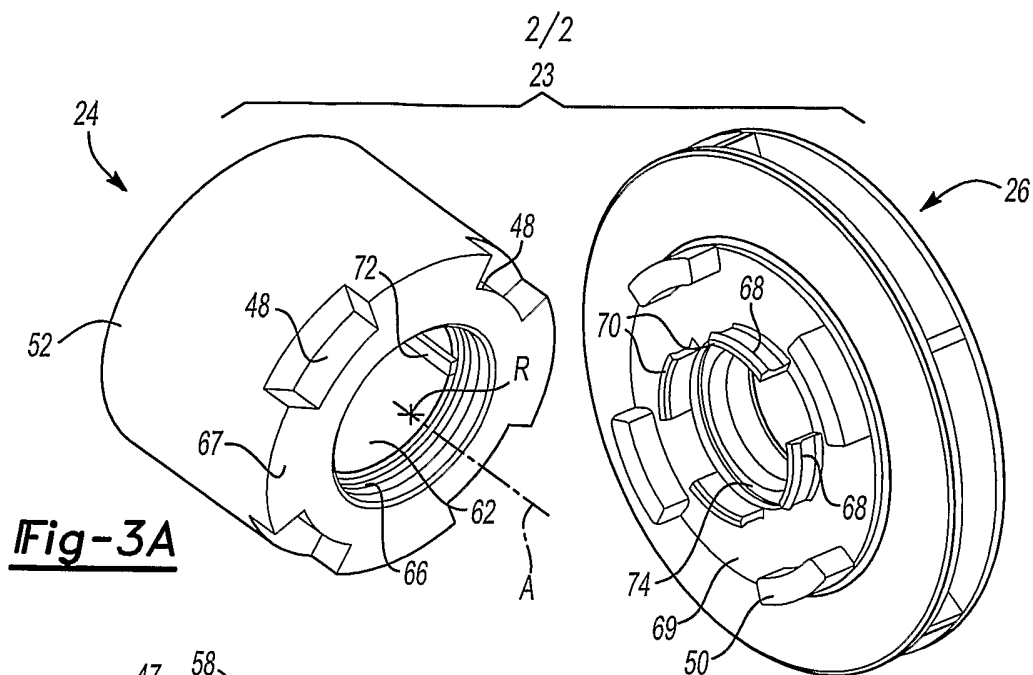


Fig-2



INTERNATIONAL SEARCH REPORT

International application No
PCT/US2006/004250

A. CLASSIFICATION OF SUBJECT MATTER

INV. F04D13/02 F04D29/20

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)
F04D

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal, WPI Data, PAJ

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	EP 1 340 917 A (IWAKI CO., LTD) 3 September 2003 (2003-09-03) the whole document	1-10, 13-15
X	US 2002/054820 A1 (FUKAMACHI MASATOSHI ET AL) 9 May 2002 (2002-05-09) the whole document	1, 7, 14
X	DE 40 15 519 A1 (SWF AUTO-ELECTRIC GMBH, 7120 BIETIGHEIM-BISSINGEN, DE) 21 November 1991 (1991-11-21) the whole document	1, 7, 14
A	US 4 013 384 A (OIKAWA ET AL) 22 March 1977 (1977-03-22) the whole document	1, 7, 14
	----- -/--	

☒ Further documents are listed in the continuation of Box C.

☒ See patent family annex.

* Special categories of cited documents:

"A" document defining the general state of the art which is not considered to be of particular relevance

"E" earlier document but published on or after the international filing date

"L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)

"O" document referring to an oral disclosure, use, exhibition or other means

"P" document published prior to the international filing date but later than the priority date claimed

"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention

"X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone

"Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art.

"&" document member of the same patent family

Date of the actual completion of the international search

23 June 2006

Date of mailing of the international search report

03/07/2006

Name and mailing address of the ISA/
European Patent Office, P.B. 5818 Patentlaan 2
NL - 2280 HV Rijswijk
Tel. (+31-70) 340-2040, Tx. 31 651 epo nl,
Fax: (+31-70) 340-3016

Authorized officer

Giorgini, G

INTERNATIONAL SEARCH REPORT

International application No
PCT/US2006/004250

C(Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	US 5 380 112 A (SCHICKTANZ ET AL) 10 January 1995 (1995-01-10) the whole document -----	1,7,14
A	US 5 269 664 A (BUSE ET AL) 14 December 1993 (1993-12-14) the whole document -----	1,7,14

INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

PCT/US2006/004250

Patent document cited in search report		Publication date	Patent family member(s)	Publication date
EP 1340917	A	03-09-2003	WO 02099283 A1 US 2004009079 A1	12-12-2002 15-01-2004
US 2002054820	A1	09-05-2002	CA 2360401 A1 CN 1353251 A ES 2212699 A1 IT T020011020 A1 JP 2002138985 A TW 503299 B	06-05-2002 12-06-2002 16-07-2004 28-04-2003 17-05-2002 21-09-2002
DE 4015519	A1	21-11-1991	NONE	
US 4013384	A	22-03-1977	DE 2532262 A1 FR 2278957 A1 GB 1496035 A	29-01-1976 13-02-1976 21-12-1977
US 5380112	A	10-01-1995	AT 135089 T DE 9204349 U1 DK 563437 T3 EP 0563437 A2	15-03-1996 12-11-1992 20-05-1996 06-10-1993
US 5269664	A	14-12-1993	DE 4331560 A1 GB 2270717 A JP 3514492 B2 JP 6200894 A	17-03-1994 23-03-1994 31-03-2004 19-07-1994