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(19) **United States**(12) **Patent Application Publication****Yoon et al.**(10) **Pub. No.: US 2010/0154455 A1**(43) **Pub. Date: Jun. 24, 2010**(54) **DEHUMIDIFYING COOLING DEVICE FOR DISTRICT HEATING**(76) Inventors: **Seok-Mann Yoon**, Incheon (KR);
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Gwacheon City, Gyeonggi-Do 427-600 (KR)(21) Appl. No.: **11/990,087**(22) PCT Filed: **Mar. 8, 2007**(86) PCT No.: **PCT/KR2007/001148**

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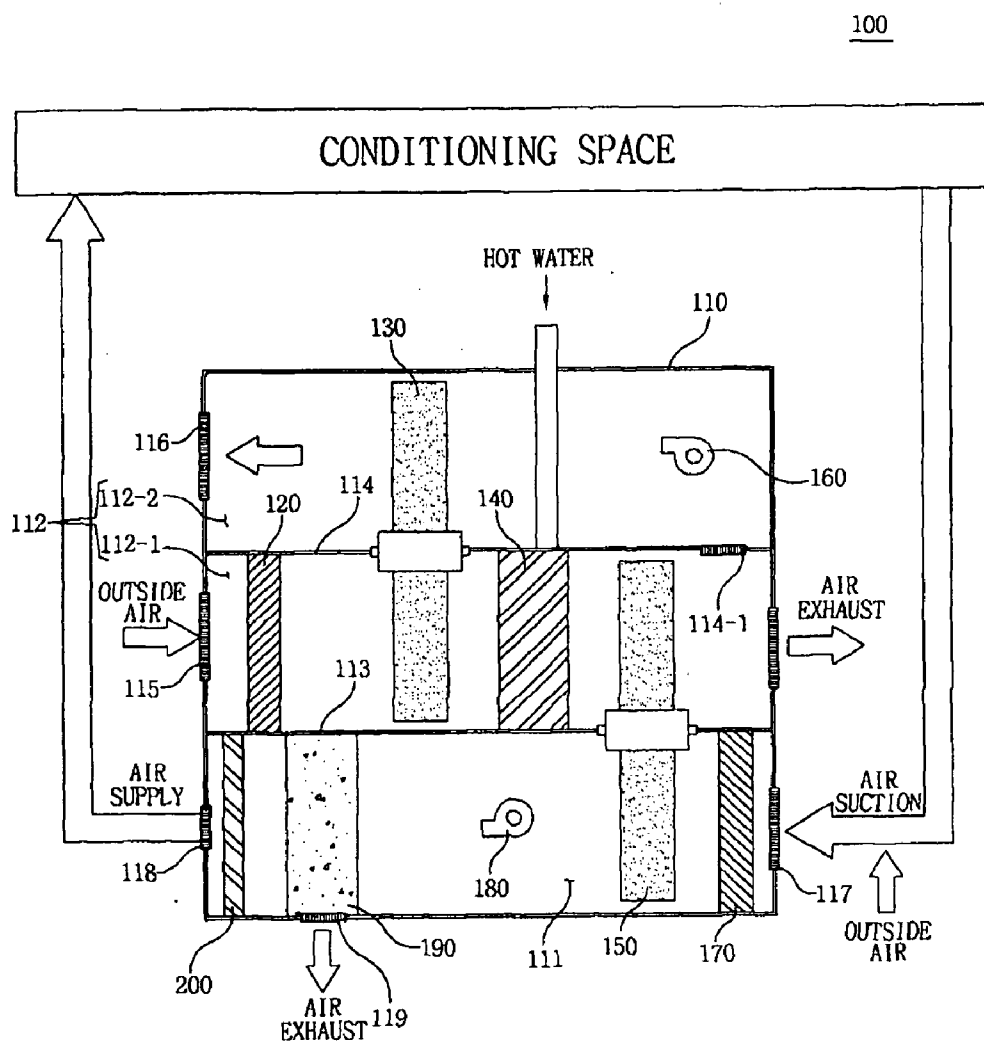
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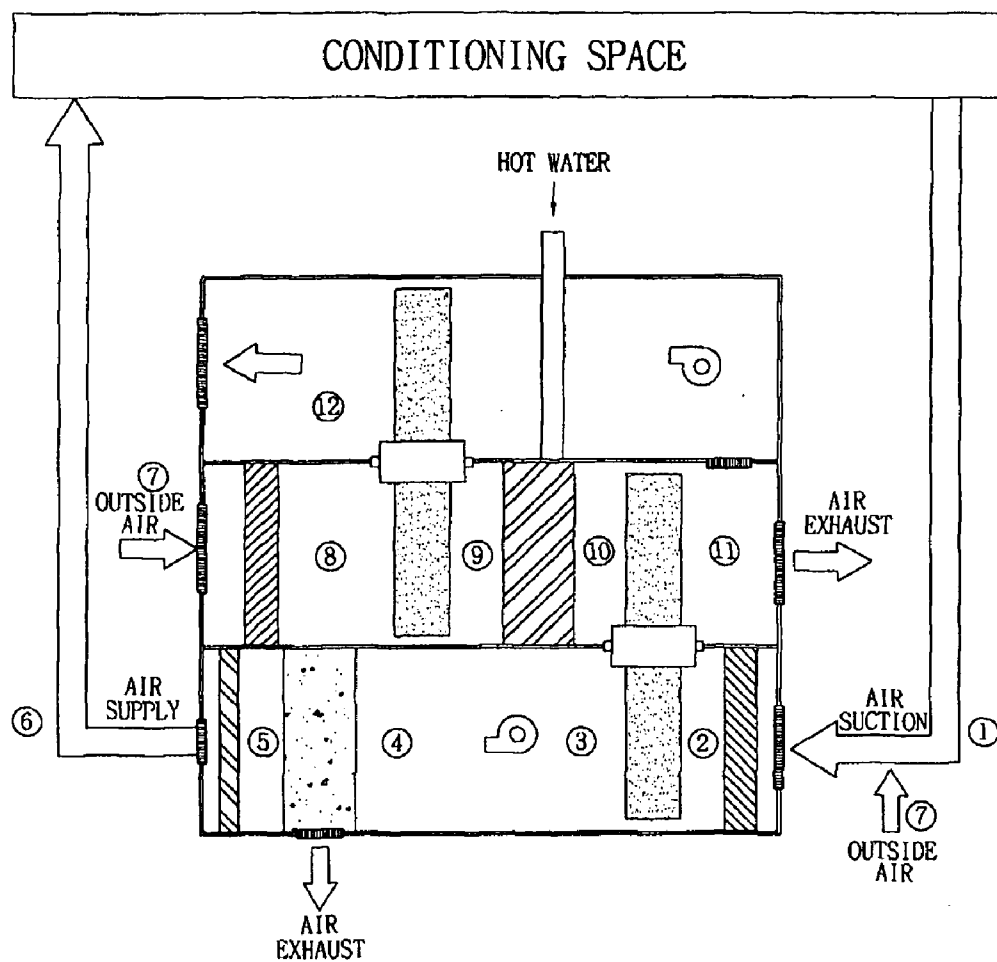
Publication Classification(51) **Int. Cl.****F25B 15/00** (2006.01)**F23L 15/02** (2006.01)**F28D 15/00** (2006.01)(52) **U.S. Cl. 62/271; 165/8; 165/7; 165/104.34**(57) **ABSTRACT**

Disclosed is a dehumidifying cooling device for district heating which comprises; a case having a first partition to divide the interior of the case into a wet channel and a dry channel and a second partition to divided the wet channel into a first wet channel and a second wet channel, a sensible heat exchanger to heat exchange the outside air in the first wet channel with the outside air in the second wet channel, a heating coil for raising the temperature of the outside air in the second wet channel, a rotatable de humidifying wheel for adsorbing and removing moisture contained in the circulated air within the dry channel, and a regenerative-evaporative cooler for cooling the circulated air in the dry channel. With this configuration, the device can carry out an air cooling operation by use of hot water supplied by district heating systems and gas or oil boilers installed in individual households, thereby achieving a reduced device size via the implementation of the cooling operation under the atmospheric pressure state and reducing manufacturing costs by virtue of a simplified system configuration.

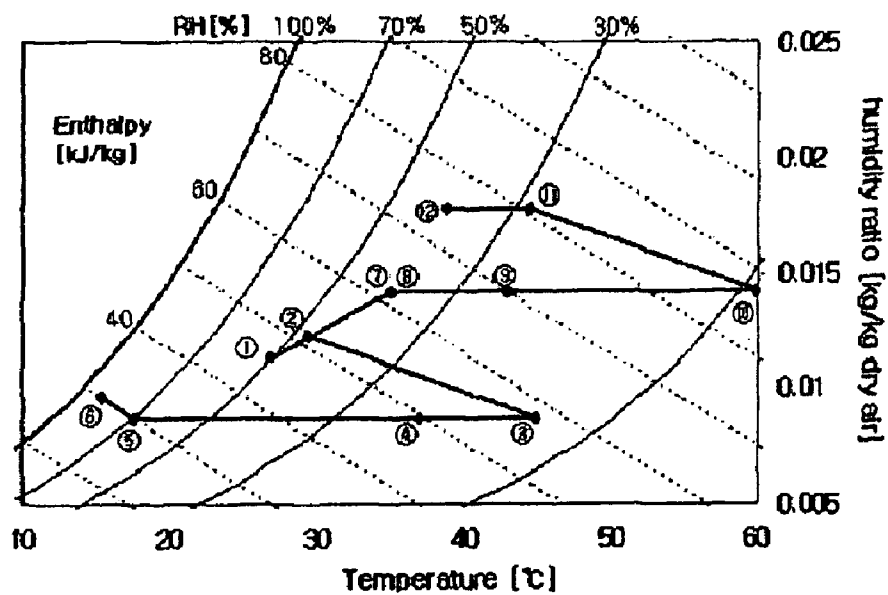
【Figure 1】



【Figure 2】



【Figure 3】



DEHUMIDIFYING COOLING DEVICE FOR DISTRICT HEATING

TECHNICAL FIELD

[0001] The present invention relates to a dehumidifying cooling device for district heating, and more particularly, to a dehumidifying cooling device for district heating which can carry out an air cooling operation by use of hot water supplied by large-scale or small-scale district heating systems and gas or oil boilers installed in individual households.

BACKGROUND ART

[0002] There is spreading a prospect that the recent high oil price situation is not a temporary problem, but is continuously maintained and fixed. Thereby, main energy consuming countries of the world increasingly take a great effort for securing stable energy resources. With the effectuation of the Tokyo protocol dealing with a reduction in the discharge of greenhouse gas for the sake of preventing global warming, it will be expected that an international pressure about the use limit of fossil energy, the criterion of energy efficiency, etc. will be strengthened.

[0003] According to the published energy report, the amount of energy consumed in domestic and business fields of Korean in 2003 is approximately 55 millions TOE a year, and occupies 25.2% of the total national consumption energy. This rate also corresponds to 41.9% on the basis of electricity. For the past four years, the energy consumption of domestic and business fields shows an average annual rate of increase of 5.3%, whereas the consumption of electricity shows an average annual rate of increase of 12%. Accordingly, it will be appreciated that the consumption of electricity particularly has a rapid increase. Estimating on the basis of a variation in the monthly energy consumption of residential buildings and sample survey results about non-residential buildings as the subject of energy controlment, it is analyzed that 50% of the energy consumption of residential buildings and 47% of the energy consumption of business buildings are used for air conditioning. In conclusion, of the energy consumption of buildings, energy required for air conditioning occupies 13% of the national total energy consumption of Korea.

[0004] Accordingly, to guarantee the efficient use of energy and the continuous development of energy industry while observing related international agreements, it is necessary to improve the use efficiency of energy for air conditioning in domestic and business fields. From this viewpoint, there is put in force a so-called collective-energy industry in which thermal energy and electricity, generated from energy generation facilities concentrated on a specific place for improving the use efficiency of energy in domestic and business fields, are supplied collectively to a plurality of users in residential and business areas. It is reported that the collective-energy industry utilizes waste heat occurred during power generation as a heating source for space heating and hot water heating, thereby achieving not only a reduction of energy by approximately 20 to 30% by virtue of an improved use efficiency of energy, but also an improvement of air environment by approximately 30 to 40% by virtue of a reduction usage of fuel and intensive environmental managements. The collective-energy industry is evaluated as an effective industry capable of dealing with related international environmental restrictions including a climatic change convention, etc. In the affirmative evaluation's debt, in Korea, approximately 1.2

million families share in the benefits of district heating in 2003, and in particular, 85% of supplied energy is generated by combined heat and power generation. Korea has a plan to expand the propagation range of district heating to 2 million families by 2010.

[0005] In the combined heat and power generation, namely, cogeneration, the generation ratio of electricity to heat is fixed at 3:5. Therefore, it is important to keep the ratio of electricity to heat at an appropriate level for maximizing the effect of the collective-energy industry. In Korea, the above mentioned generation ratio can be fulfilled in winter, but the summer of Korea has an increased electricity load for air cooling and substantially no heat load. As a result, the operation rate of distinct heating in summer decreases to less than 10%, and this causes a deterioration in the economical efficiency of cogeneration. Actually, no generation results reported between June and September in 2003.

[0006] To improve the operation rate of collective-energy generation facilities for sufficiently utilizing the effects of the industry, excavating the demand of heat in summer is necessary, and in particular, development and propagation of a technology for supplying cooling energy using distinct heating facilities is necessary.

[0007] In one example of the above described cooling energy supply technology, an absorption chiller is installed to a receptor, such as a large-scale building, etc. such that the chiller performs a central cooling operation using energy delivered from distinct heating facilities.

[0008] The absorption chiller is designed to chill water flowing in a pipe by use of heat generated during the evaporation of a liquid-phase refrigerant and condense the evaporated gas-phase refrigerant for the reuse thereof.

DISCLOSURE

Technical Problem

[0009] However, despite of various researches and developments for improving the performance thereof, the absorption chiller has a limit in the improvement of performance due to a low temperature of a heating source. In addition, the absorption chiller has an uneconomical high water return temperature because it cannot use water having a temperature of 80° C. or less, and suffers from a little difference between the temperature of supplied water and the temperature of water to be returned.

[0010] When the absorption chiller is installed to an apartment, etc. occupying the most part of district heating, so as to carry out a central cooling operation, there is a problem in that cold water pipes have to be additionally installed regardless of hot water supply pipes.

[0011] Therefore, the present invention has been made in view of the above problems, and it is an object of the present invention to provide a dehumidifying cooling device for district heating which can carry out an air cooling operation by use of hot water supplied by large-scale or small-scale district heating systems and gas or oil boilers installed in individual households, thereby achieving a reduced device size via the implementation of the cooling operation under the atmospheric pressure state and reducing manufacturing costs by virtue of a simplified system configuration.

Technical Solution

[0012] In accordance with an aspect of the present invention, the above and other objects can be accomplished by the

provision of a dehumidifying cooling device for district heating comprising: a case having a first partition to divide the interior of the case into a wet channel and a dry channel and a second partition to divided the wet channel into a first wet channel, and a second wet channel, the first wet channel being provided at one end thereof with an outside air suction hole for introducing outside air into the first wet channel, the second wet channel being provided at one end thereof with an exhaust hole for discharging the outside air, the second partition being perforated with a flow hole for transferring the outside air from the first wet channel into the second wet channel, the dry channel being provided, at one end thereof, with a circulated air suction hole for introducing circulated air from a conditioning space into the dry channel and, at the other end thereof, with an air supply hole for supplying cooling air into the conditioning space; a sensible heat exchanger configured to rotate about the second partition and serving to heat exchange the outside air, introduced into the first wet channel through the outside air suction hole, with the outside air to be discharged from the second wet channel; a heating coil installed in the second wet channel at a position between a rear end of the sensible heat exchanger and the flow hole and serving to raise the temperature of the outside air passing through the second wet channel by use of heat of hot water introduced into the heating coil; a dehumidifying wheel configured to rotate about the first partition at a position between a rear end of the heating coil and the flow hole and serving to adsorb and remove moisture contained in the circulated air within the dry channel, the dehumidifying wheel being regenerated by evaporating the adsorbed moisture to thereby supply the evaporated moisture into the high-temperature outside air in the first wet channel; and a regenerative-evaporative cooler installed in the dry channel at a position between the circulated air supply hole and the dehumidifying wheel and serving to cool the circulated air in the dry channel, which was dehumidified to high-temperature dry air by the dehumidifying wheel and subsequently, heat exchanged and cooled, the cooled circulated air being delivered to the air supply hole of the case.

[0013] Preferably, the device further comprises a direct-evaporative cooler installed in the dry channel at a position in front of the regenerative-evaporative cooler, the direct-evaporative cooler serving to carry out a secondary cooling operation of the circulated air discharged from the regenerative-evaporative cooler.

[0014] Preferably, the device further comprises a first filter installed in the first wet channel at a position between the outside air suction hole and the sensible heat exchanger and serving to remove impurities contained in the outside air.

[0015] Preferably, the device further comprises an exhaust blower installed in the second wet channel at a position between the sensible heat exchanger and the flow hole and serving to forcibly discharge the outside air from the second wet channel through the exhaust hole.

[0016] Preferably, the device further comprises: a second filter installed between the circulated air suction hole and the dehumidifying wheel and serving to remove impurities contained in the circulated air; and an air supply blower installed between the dehumidifying wheel and the regenerative-evaporative cooler and serving to forcibly discharge the cooled circulated air from the dry channel through the air supply hole, wherein the second filter and the air supply blower are installed in the dry channel.

[0017] Preferably, the case further has a cooler exhaust hole provided at the dry channel for discharging high-temperature air generated while the regenerative-evaporative cooler carries out a secondary cooling operation.

[0018] Preferably, the amount of the high-temperature air to be discharged through the cooler exhaust hole is 30% of the total circulated air.

[0019] Preferably, the hot water to be introduced into the heating coil is delivered from any one selected from among a cogeneration plant, a heating boiler, a micro-turbine, a small gas engine, a small gas turbine, a gas boiler, and an oil boiler.

[0020] Preferably, the circulated air suctioned through the circulated air suction hole is mixed with the outside air by a predetermined mixing ratio of 7:3.

ADVANTAGEOUS EFFECTS

[0021] According to a dehumidifying cooling device for district heating of the present invention having the above described configuration, it is possible to carry out an air cooling operation by use of hot water supplied by large-scale or small-scale district heating systems and gas or oil boilers installed in individual households. Accordingly, the present invention has the effect of achieving a reduced device size via the implementation of the cooling operation under the atmospheric pressure state, and reducing manufacturing costs by virtue of a simplified system configuration.

DESCRIPTION OF DRAWINGS

[0022] The above and other objects, features and other advantages of the present invention will be more clearly understood from the following detailed description taken in conjunction with the accompanying drawings, in which:

[0023] FIG. 1 is a view illustrating the configuration of a dehumidifying cooling device for district heating according to the present invention;

[0024] FIG. 2 is a view illustrating the flow of air in the dehumidifying cooling device for district heating according to the present invention; and

[0025] FIG. 3 is a graph illustrating the temperature distribution of humid air used in the dehumidifying cooling device for district heating according to the present invention.

BEST MODE

[0026] Now, the configuration of a dehumidifying cooling device for district heating according to the present invention will be described in detail with reference to the accompanying drawings.

[0027] In the following description of the present invention, a detailed description of known functions and configurations incorporated herein will be omitted when it may make the subject matter of the present invention rather unclear. Also, the terms used in the following description are terms defined taking into consideration the functions obtained in accordance with the present invention. The definitions of these terms should be determined based on the whole content of this specification because they may be changed in accordance with the option of a user or operator or a usual practice.

[0028] FIG. 1 is a view illustrating the configuration of a dehumidifying cooling device for district heating according to the present invention.

[0029] Referring to FIG. 1, the dehumidifying cooling device 100 according to the present invention comprises a case 110, a first filter 120, a sensible heat exchanger 130, a

heating coil **140**, a dehumidifying wheel **150**, an exhaust blower **160**, a second filter **170**, an air supply blower **180**, a regenerative-evaporative cooler **190**, and a direct-evaporative cooler **200**.

[0030] The case **110** is made of a metallic material and has a rectangular box shape. The case **110** is installed with a first partition **113** to divide the interior of the case **110** into a wet channel **112** and a dry channel **111**. The case **110** is further installed with a second partition **114** to divide the wet channel **112** into a first wet channel **112-1** and a second wet channel **112-2**. The case **110** has an outside air suction hole **115** provided at one end of the first wet channel **112-1** for introducing outside air into the first wet channel **112-1** and an exhaust hole **116** provided at one end of the second wet channel **112-2** for discharging the outside air. The second partition **114** is perforated with a flow hole **114-1** for transferring the outside air from the first wet channel **112-1** into the second wet channel **112-2**. The case **110** also has a circulated air suction hole **117** provided at one end of the dry channel **111** for introducing circulated air from a conditioning space CS into the dry channel **111** and an air supply hole **118** provided at the other end of the dry channel **111** for supplying cooling air into the conditioning space CS. The dry channel **111** of the case **110** is further provided with a cooler exhaust hole **119** for discharging high-temperature air generated while the regenerative-evaporative cooler **190** carries out a secondary cooling operation that will be described hereinafter. The circulated air, introduced into the case **110** through the circulated air suction hole **117**, is mixed with the outside air at a ratio of 7:3, to keep the interior of the case **110** in the atmospheric pressure state.

[0031] The first filter **120** is located in the first wet channel **112-1** of the case **110** at a position between the outside air suction hole **115** and the sensible heat exchanger **130**. The first filter **120** is used to remove impurities contained in the suctioned outside air. Preferably, the first filter **120** is an antibacterial filter, and is easily separable from the case **110**.

[0032] The sensible heat exchanger **130** has a rotating shaft **131** installed in the same direction as the second partition **114** and takes the form of a disc to rotate about the rotating shaft **131** inside the first and second wet channels **112-1** and **112-2** of the case **110**. The sensible heat exchanger **130** is used to heat exchange the outside air introduced into the first wet channel **112-1** through the outside air suction hole **115** with the outside air to be discharged from the second wet channel **112-2** through the exhaust hole **116**. The sensible heat exchanger **130** takes the form of a honeycomb-patterned disc fabricated by processing a thin plate, such as an aluminum plate, etc. suitable for heat exchange. There are provided an additional motor and belt (not shown) for rotation of the sensible heat exchanger **130**.

[0033] The heating coil **140** is located in the first wet channel **112-1** of the case **110** at a position between a rear end of the sensible heat exchanger **130** and the flow hole **114-1**. The heating coil **140** is used to raise the temperature of the outside air passing through the first wet channel **112-1** by use of heat of hot water introduced thereinto. The hot water introduced into the heating coil **140** is delivered from any one selected from among a cogeneration plant, a heating boiler, a micro-turbine, a small gas engine, a small gas turbine, a gas boiler, and an oil boiler, and has a temperature within a range of 60 to 120° C.

[0034] The dehumidifying wheel **150** has a rotating shaft **151** installed in the same direction as the first partition **113**,

and takes the form of a disc to rotate about the rotating shaft **151** inside the first wet channel **112-1** and the dry channel **111** of the case **110**. The dehumidifying wheel **150** is located behind the heating coil **140** and serves to adsorb and remove moisture contained in the circulated air within the dry channel **111**. The dehumidifying wheel **150** is regenerated by evaporating the adsorbed moisture to thereby supply the moisture into the high-temperature outside air within the first wet channel **112-1**. The dehumidifying wheel **150** takes the form of a honeycomb-patterned disc containing an adsorbent, such as silica gel, zeolite, or the like, for adsorbing the moisture contained in the circulated air in a dry adsorption manner. There are provided an additional motor and belt (not shown) for rotation of the dehumidifying wheel **150**.

[0035] The exhaust blower **160** is installed in the second wet channel **112-2** of the case **110** at a position between the sensible heat exchanger **130** and the flow hole **114-1**, and used to forcibly discharge the outside air from the second wet channel **112-2** through the exhaust hole **116**.

[0036] The second filter **170** is installed in the dry channel **111** of the case **110** at a position between the circulated air suction hole **117** and the dehumidifying wheel **150** and used to remove impurities and bad smell contained in the circulated air. Preferably, the second filter **170** is an antibacterial filter, and is easily separable from the case **110**.

[0037] The air supply blower **180** is installed in the dry channel **111** of the case **110** at a position between the dehumidifying wheel **150** and the sensible heat exchanger **130** and used to forcibly discharge the circulated air from the dry channel **111** through the circulated air supply hole **118**.

[0038] The regenerative-evaporative cooler **190** is installed in the dry channel **111** at a position between the circulated air supply hole **118** and the dehumidifying wheel **150**. If the circulated air introduced into the dry channel **111** is dehumidified by the dehumidifying wheel **150** so as to be changed to high-temperature dry air and subsequently, heat exchanged and cooled, the regenerative-evaporative cooler **190** further cools the circulated air. The cooled circulated air is delivered to the air supply hole **118** of the case **110**, whereas the high-temperature air generated during cooling is delivered to the cooler exhaust hole **119**. Here, the amount of the high-temperature air to be discharged through the cooler exhaust hole **119** is 30% of the total circulated air. The interior of the regenerative-evaporative cooler **190** is divided into a dry channel and a wet channel. If a part of the air, passing through the dry channel, is delivered into the wet channel, the air is cooled as water is evaporated by the high-temperature surface of the wet channel, thereby acting to absorb heat from the remaining higher temperature air passing through the dry channel. Thereby, the air passing through the dry channel can be cooled to a dew-point temperature to the maximum extent without an increase of humidity. The configuration of the regenerative-evaporative cooler is disclosed in Korea Patent Registration No. 0409265 and thus, a detailed description thereof will be omitted herein.

[0039] The direct-evaporative cooler **200** is installed in the dry channel **111** of the case **110** at a position in front of the regenerative-evaporative cooler **190**. The direct-evaporative cooler **200** serves to carry out a secondary cooling operation of the circulated air from the regenerative-evaporative cooler **190**, so as to supply the resulting air into the conditioning space CS through the air supply hole **118** of the case **110**.

[0040] Hereinafter, the operation and effects of the dehumidifying cooling device for district heating according to the present invention will be described in detail with reference to FIGS. 1 to 3.

[0041] FIG. 2 is a view illustrating the flow of air in the dehumidifying cooling device for district heating according to the present invention, and FIG. 3 is a graph illustrating the temperature distribution of humid air used in the dehumidifying cooling device.

[0042] Explaining first a dehumidifying cooling operation carried out in the dry channel 111, circulated air from the conditioning space CS, which is mixed with high-temperature and high-humidity outside air, is introduced into case 110 through the circulated air suction hole 117 under the operation of the air supply blower 180. After passing through the second filter 170, the introduced circulated air sequentially passes through the dehumidifying wheel 150 such that the moisture contained in the circulated air is removed by the adsorbent.

[0043] The dehumidified circulated air is heated by adsorptive heat generated from the surface of the dehumidifying wheel 150. The resulting high-temperature and low-humidity circulated air passes through the regenerative-evaporative cooler 190.

[0044] If the circulated air is introduced into the regenerative-evaporative cooler 190, 70% of the circulated air is cooled while passing through the regenerative-evaporative cooler 190, and 30% of the circulated air is discharged to the outside through the cooler exhaust hole 119.

[0045] The circulated air, having passed through the regenerative-evaporative cooler 190, is secondarily cooled while passing through the direct-evaporative cooler 200, thereby being supplied into the conditioning space CS through the air supply hole 118 of the case 110.

[0046] Next, explaining a heat-exchange operation carried out in the wet channel, high-temperature and high-humidity outside air is introduced into the first wet channel 112-1 through the outside air suction hole 115 and passes through the first filter 120 under the operation of the exhaust blower 160. Then, the filtered outside air is heat exchanged with the high-temperature and high-humidity outside air in the second wet channel 112-2 while passing through the sensible heat exchanger 130. Thereby, the outside air with the raised temperature passes through the heating coil 140.

[0047] While passing through the heating coil 140, the temperature of the outside air is further raised by hot water supplied into the heating coil 140. Thereby, the outside air to be delivered into the dehumidifying wheel 150 has a significantly raised temperature.

[0048] Then, while passing through the dehumidifying wheel 150 rotating in a state of adsorbing moisture therein, the outside air forcibly evaporates moisture thereof. Thereafter, the dehumidified outside air is moved into the second wet channel 112-2 through the flow hole 114-1. Through the above described process, the surface of the dehumidifying wheel 150 is returned to an original dried state thereof, thereby recovering a dehumidifying ability thereof.

[0049] The high-temperature and high-humidity outside air, moved into the second wet channel 112-2, is heat exchanged with the outside air in the first wet channel 112-1 while passing through the sensible heat exchanger 130, and discharged to the outside through the exhaust hole 116.

[0050] Referring to FIGS. 2 and 3, in the dehumidifying cooling device for district heating according to the present

invention, if circulated air ① is introduced into the case, the circulated air ① is mixed with outside air ⑦, to produce mixed air ② having a raised temperature and absolute humidity. While passing through the dehumidifying wheel, the mixed air ② is changed to higher-temperature and lower absolute-humidity air ③.

[0051] Then, the air ③ is heat exchanged to produce air ④ having a rapid drop only in temperature. In sequence, while passing through the regenerative-evaporative cooler, the temperature of the air ④ is changed into air ⑤ having a slightly lowered temperature and slightly raised absolute humidity.

[0052] Meanwhile, when the introduced outside air ⑦ passes through the first filter, the filtered air ⑧ has the same temperature and absolute humidity as those of the air ⑦. The air ⑧ is heat exchanged with the outside air ⑪ in the first wet channel while passing through the sensible heat exchanger. The resulting heat exchanged air ⑨ is slightly raised in temperature, but keeps the same absolute humidity as that of the air ⑧. Then, the air ⑨ is raised only in temperature while passing through the heating coil, resulting in high-temperature air ⑩.

[0053] The air ⑩ is dropped in temperature, but raised in absolute humidity in the course of passing through the dehumidifying wheel, thereby being changed to low-temperature and high-humidity air ⑪. Then, while passing through the sensible heat exchanger in the second wet channel, the air ⑪ is heat exchanged with the outside air ⑧ in the first wet channel such that the heat exchanged air ⑫, which is slightly dropped in temperature but keeps the same absolute humidity as the air ⑪, is discharged through the exhaust hole.

[0054] In conclusion, in the dehumidifying cooling device for district heating according to the present invention, air to be supplied into a conditioning indoor space is subjected to the transfer of heat and moisture via a direct contact with the dehumidifying cooling device. This has the effect of achieving excellent transfer efficiency and producing and supplying cooling air with a low-temperature heating source of 60° C. Further, differently from conventional absorptive devices, the dehumidifying cooling device is operable in the atmospheric pressure state and has a simplified configuration, resulting in a considerable reduction of manufacturing costs.

INDUSTRIAL APPLICABILITY

[0055] As apparent from the above description, a dehumidifying and cooling device according to the present invention can be installed to residential and business buildings, etc. using hot water delivered by district heating facilities, so as to utilize the hot water as a source for cooling a room.

[0056] Although the preferred embodiments of the present invention have been disclosed for illustrative purposes, those skilled in the art will appreciate that various modifications, additions and substitutions are possible, without departing from the scope and spirit of the invention as disclosed in the accompanying claims.

1. A dehumidifying cooling device for district heating comprising:

a case having a first partition to divide the interior of the case into a wet channel and a dry channel and a second partition to divide the wet channel into a first wet channel and a second wet channel, the first wet channel being provided at one end thereof with an outside air suction

hole for introducing outside air into the first wet channel, the second wet channel being provided at one end thereof with an exhaust hole for discharging the outside air, the second partition being perforated with a flow hole for transferring the outside air from the first wet channel into the second wet channel, the dry channel being provided, at one end thereof, with a circulated air suction hole for introducing circulated air from a conditioning space into the dry channel and, at the other end thereof, with an air supply hole for supplying cooling air into the conditioning space,

- a sensible heat exchanger configured to rotate about the second partition and serving to heat exchange the outside air, introduced into the first wet channel through the outside air suction hole, with the outside air to be discharged from the second wet channel;
 - a heating coil installed in the second wet channel at a position between a rear end of the sensible heat exchanger and the flow hole and serving to raise the temperature of the outside air passing through the second wet channel by use of heat of hot water introduced into the heating coil;
 - a dehumidifying wheel configured to rotate about the first partition at a position between a rear end of the heating coil and the flow hole and serving to adsorb and remove moisture contained in the circulated air within the dry channel, the dehumidifying wheel being regenerated by evaporating the adsorbed moisture to thereby supply the evaporated moisture into the high-temperature outside air in the first wet channel; and
 - a regenerative-evaporative cooler installed in the dry channel at a position between the circulated air supply hole and the dehumidifying wheel and serving to cool the circulated air in the dry channel, which was dehumidified into high-temperature dry air by the dehumidifying wheel and subsequently, heat exchanged and cooled, the cooled circulated air being delivered to the air supply hole of the case.
2. The device according to claim 1, further comprising:
- a direct-evaporative cooler installed in the dry channel at a position in front of the regenerative-evaporative cooler, the direct-evaporative cooler serving to carry out a sec-

ondary cooling operation of the circulated air discharged from the regenerative-evaporative cooler.

- 3. The device according to claim 1, further comprising: a first filter installed in the first wet channel at a position between the outside air suction hole and the sensible heat exchanger and serving to remove impurities contained in the outside air.
- 4. The device according to claim 1, further comprising: an exhaust blower installed in the second wet channel at a position between the sensible heat exchanger and the flow hole and serving to forcibly discharge the outside air from the second wet channel through the exhaust hole.
- 5. The device according to claim 1, further comprising: a second filter installed between the circulated air suction hole and the dehumidifying wheel and serving to remove impurities contained in the circulated air; and an air supply blower installed between the dehumidifying wheel and the regenerative-evaporative cooler and serving to forcibly discharge the cooled circulated air from the dry channel through the air supply hole, wherein the second filter and the air supply blower are installed the dry channel.
- 6. The device according to claim 1, wherein the case further has a cooler exhaust hole provided at the dry channel for discharging high-temperature air generated while the regenerative-evaporative cooler carried out a secondary cooling operation.
- 7. The device according to claim 6, wherein the amount of the high-temperature air to be discharged through the cooler exhaust hole is 30% of the total circulated air.
- 8. The device according to claim 1, wherein the hot water to be introduced into the heating coil is delivered from any one selected from among a cogeneration plant, a heating boiler, a micro-turbine, a small gas engine, a small gas turbine, a gas boiler, and an oil boiler.
- 9. The device according to claim 1, wherein the circulated air suctioned through the circulated air suction hole is mixed with the outside air by a predetermined mixing ratio.
- 10. The device according to claim 1, wherein the predetermined mixing ratio of the circulated air to the outside air is 7:3.

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