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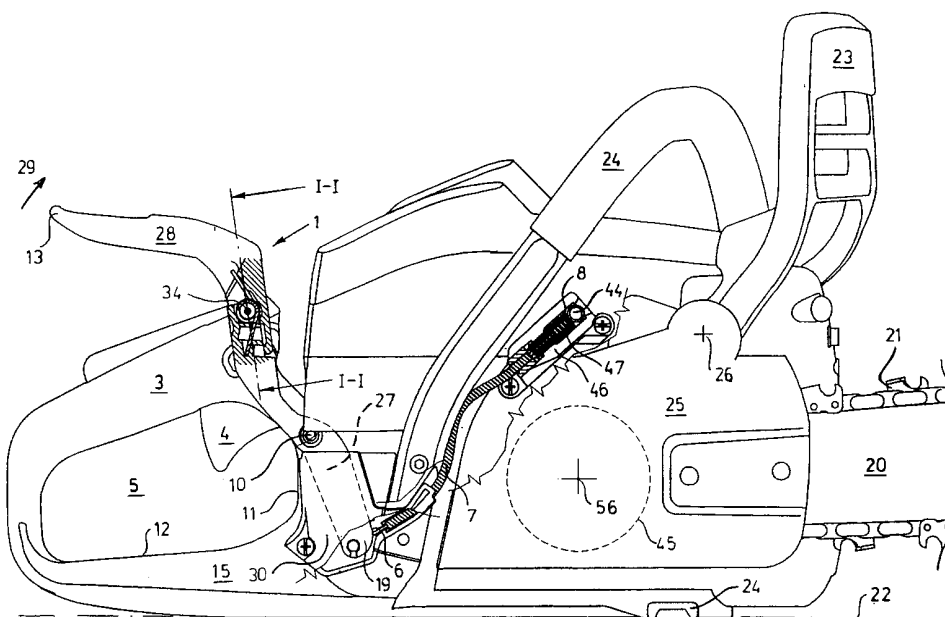
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[Continued on next page]

(54) Title: TRIGGER DEVICE FOR CHAIN BRAKE



(57) Abstract: A brake triggering device with a pivotably mounted trigger arm (1) located in connection to a handheld working tool's, mainly a chain saw's (2), rear handle (3), with a throttle control (4) and a handle opening (5) located below the handle (3), so that the trigger arm (1), when the saw changes inclination in relation to the operator's forearm, e.g. at the event of kickback or fall, is actuated by the operator's hand or arm, whereby the trigger arm (1) via a transfer mechanism actuates a brake (9) so that this stops the movement of the saw chain. The pivot (10) of the trigger arm is located in connection to a front (11) or a bottom (12) side of the handle opening (5).



WO 01/39942 A1



For two-letter codes and other abbreviations, refer to the "Guidance Notes on Codes and Abbreviations" appearing at the beginning of each regular issue of the PCT Gazette.

TRIGGER DEVICE FOR CHAIN BRAKE

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Technical field

The subject invention refers to a brake triggering device with a trigger arm located in connection to a handheld working tool's, mainly a chain saw's, rear handle with a throttle control and a handle opening located below the handle, so that the trigger arm, when the saw changes inclination in relation to the operator's forearm, e.g. at the event of kickback or fall, is actuated by the operator's hand or arm, whereby the trigger arm via a transfer mechanism actuates a brake to stop the movement of the saw chain.

Background of the invention

When using a chain saw there is a risk of several kinds of severe accidents. There might be so called kickback accidents, i.e. the saw bar with its rotating saw chain is swung up towards the saw operator. There might be fall accidents, when the user perhaps slips and falls over the saw with its rotating saw chain. In order to reduce the risk of damages at the event of kickback the chain saw is normally equipped with a chain brake which is actuated by a so called kickback guard. This is mounted in front of the chain saw's front handle, a so called handle frame. At the event of kickback the user's forehead often hits the kickback guard, which then actuates the chain brake. The chain brake can also be actuated due to the inertia of the kickback guard when the rapid kickback occurs. However, there might be accidents due to slow kickbacks, or when the user falls over the saw, where it might be a great risk that the above-mentioned trigger principles would not function.

Swedish patent 441992 describes a trigger device for a kickback guard that is so designed that it can be actuated by the rear hand. A trigger handle is located above the rear hand and is entirely mechanically connected to the chain saw's ordinary kickback guard located in front of the front handle frame. The

trigger handle is thus connected to the engine unit that is not equipped with vibration damping, while the rear handle as well as the handle frame are isolated from vibrations. This is a clear disadvantage. A redesigned version has been produced and marketed, where a trigger handle is mounted to the rear handle and
5 connected via a wire to a trigger mechanism for the chain brake. The trigger arm is designed as a lever journalled a few centimetres above the saw's throttle control. This location of the journal results in that the angle of the operator's arm will not correspond particularly well with the angle of the trigger arm so that the trigger arm will chafe against the arm, both at easy contact or at strong contact,
10 i.e. at an actual release. Furthermore, the trigger wire in this design would have to be drawn totally unprotected at the outside of the saw over a great deal of its length. Hereby there is a risk that it might be damaged.

Purpose of the invention

15 The purpose of the subject invention is to substantially reduce the above-mentioned problems in a trigger device with a trigger arm located in connection to a portable working tool's rear handle.

Summary of the invention

20 The above-mentioned purpose is achieved in a device in accordance with the invention having the characteristics appearing from the appended claims.

The trigger device in accordance with the invention is thus essentially characterized in that the pivot of the trigger arm is located in connection to a front or a bottom side of the handle opening. Hereby the trigger arm will get a
25 more favourable geometry of motion more corresponding to the motion of the arm at the event of kickback etc. Comparing with the known solution such a change could appear as evident. However, it means that a great number of changes have to be made on the tool to enable this location of the pivot of the trigger arm. The handle part must be adapted to make place for the trigger arm,
30 which in turn must be given a complicated design to be able to cooperate with the handle part without intruding on the necessary space in the handle opening. The location of the pivot of the trigger arm also results in that the trigger mechanism

can be given a much more protected location, e.g. the transfer wire can be drawn entirely protected at the same time as it can be given a considerably shorter and straighter drawing than in the earlier known design.

The invention is mainly intended to be used for chain saws run by internal
5 combustion engines or electric engines. However, it could also be used for other portable working tools with a rear handle, mainly tools of the cutting type. One example is a chain saw provided with a saw wire or similar in stead of a saw chain. Another example is a cutting machine, at least if it is provided with a relatively small and light cutting disc. For, a large cutting disc has a very large
10 moment of inertia. When its rotational speed has to be stopped rapidly by a triggered brake this would result in a very strong reactive moment that tends to twist the tool downwards. Then there is a great risk that the operator would cut his foot or bone before the disc has stopped.

Further characteristics and advantages of the invention will be apparent
15 from the detailed description of preferred embodiments and with the support of the drawing figures.

Brief description of the drawing

The invention will be described in closer detail in the following by way of
20 various embodiments with reference to the accompanying drawings, in which the same numbers in the different figures state one another's corresponding parts.

Figure 1 shows a side view of a chain saw equipped with the trigger device in accordance with the invention.

Figure 2 shows a partial cross-section along the line II-II in figure 1
25 greatly enlarged. Hereby is illustrated how a lateral guide between the trigger arm and the rear handle is arranged. The rear handle is shown only partly in a cross-sectional view.

Figure 3 illustrates an imagined partial cross-section corresponding to that
in figure 3 but for a first alternative embodiment of the lateral guide between the
30 trigger arm and the rear handle. The enlargement is approximately half as large as in figure 2. Only the outlines of the arm and the handle are shown.

Figure 4 illustrates an imagined partial cross-section corresponding to that in figure 3 but for a second alternative embodiment of the lateral guide between the trigger arm and the rear handle.

Figure 5 shows from the side a clutch housing with a strap brake and a mechanism for actuating this. The clutch housing is shown from the opposite side and in a larger scale, compared to figure 1, i.e. figure 5 shows the backside of the clutch housing with components.

Description of preferred embodiments

10 In the partly schematic figure 1 numeral reference 2 designates a chainsaw or a power saw with a saw bar 20 and a saw chain 21. In the following forwards is meant as the direction of the saw bar 20, while downwards also means downwards in the figure. For example, the saw could be placed on the ground designated by numeral reference 22. The saw has an ordinary handle frame 24 with a kickback guard 23 located in front of it. It has a rear handle 3 with a throttle control 4 and a handle opening 5 located below the handle 3. A clutch housing 25 is provided with a brake device, which will be actuated by the kickback guard 23 when this turns forwards. This is pivotably mounted at the pivot point 26. All this is conventional and will therefore not be described in
15 closer detail. What is characteristic are the trigger arm 1 and the transfer mechanism 6, 7, 8, which actuates a brake 9, so that this stops the movement of the saw chain if the trigger arm 1 above the handle 3 is affected to create a rotation in the direction of the arrow 29 around the pivot 10 of the trigger arm. This pivot is located in front of the handle opening 5. Hereby a favourable
20 geometry of motion is created where the protruding outer end 13 of the arm is moving forwards and upwards, as shown by the arrow 29. The rear handle 3 is arranged in handle unit 15, which extends under the saw body and to which the front handle frame 24 is attached, partly to the underside of the saw at the one end of the handle frame, and partly at the other end of the handle frame. This
25 unit, thus containing both the front handle and the rear handle, is then anti-vibration mounted to the other part of the saw. Normally also the fuel tank is integrated in this anti-vibration mounted handle unit. Below its pivot 10 the
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trigger arm 1 is concealed by a cover 30. This is cut up at its underside in order to clearly illustrate the arm's other outer end 19, which is connected to a part of the transfer mechanism, e.g. a wire 6. The wire 6 extends in its casing 7. The wire with its casing is located inside the handle frame 24, which in the drawing is shown in a cut up mode in order to illustrate the wire in its casing. The handle frame 24 is thus secured to the handle unit 15, which comprises both the front handle and the rear handle.

Consequently, in this case the trigger arm 1 is embodied as a double-armed lever. Furthermore it comprises a bottom part 27 and a top part 28. These are pivotally connected to each other at the pivot point 34 and spring-loaded against a stop in the position shown in the figure. This becomes evident from a cross-section adjacent to the pivot. This means thus that under load in the direction of the arrow 29 the parts 27 and 28 will be butt joint to each other. If on the other hand there is a load in the opposite direction the part 28 will rotate in relation to the part 27 under spring-load. This is a feature that is used to prevent the trigger arm 1 from breaking into pieces in case the saw might get jammed or similar. However, this feature is not absolutely necessary. The figure shows a very favourable embodiment of the trigger arm where the pivot is located in front of the handle opening but in connection to a front side of the handle opening. To place it even more far ahead from the handle opening would not be any advantage, and to place it at the very handle opening would result in that the trigger arm would intrude upon this somewhat, which is disadvantageous, but still quite possible if the size of the handle opening should be adapted. Obviously the pivot could as well be displaced either upwards or downwards with a maintained function of the double-armed lever. However, it is also possible to place the pivot in connection to a bottom side 12 of the handle opening. In case the trigger arm should be embodied as a single-armed lever, then the illustrated location of the attachment of the wire could in fact be maintained. The moving direction of the wire and the lever-ratio would of course be affected by where the pivot is located, but it is evident that it can be located in connection to either a front side or a bottom side of the handle opening. And, of course also any intermediate location between both these. The other end 19 of the

lever is preferably provided with apertures for wire-attachment in the conventional way, such as a bicycle hand brake for example. As becomes apparent from figure 1 the lower part of the wire is located entirely protected behind the cover 30 and the handle frame 24. It has a very short and straight run.

5 Over its entire length it is protected by lying either under or deeper than the cover 30, the handle frame 24 and the clutch housing 25. Obviously, this is a major advantage.

As becomes apparent from figure 1 the trigger arm 1 has a large total length. In the tough environments these products are used there is a great risk that
10 the arm 1 might be under load in a lot of different directions also sideways. This would result in very heavy strains on the trigger arm 1 and its attachment at the pivot 10. Therefore the arm is preferably so embodied that it is laterally guided at a certain part of the trigger arm 1 located between its pivot 10 and its protruding outer end 13 above the rear handle. In the figures 2-4 are shown some examples
15 of such guides, which can be arranged in many different ways. For practical reasons it is preferable that the trigger arm 1 is located essentially on one side of the rear handle 3 and runs down on one side of the handle unit 15. In the shown examples this is on the left side seen from the front and backwards in the direction of the saw. A location on the right side is of course also conceivable.

20 Figure 2 shows thus a greatly enlarged partial cross-sectional view along the line II-II according to figure 1. The trigger arm 1 is provided with a protruding part 18, which is arranged so that it creates a groove 17 between itself and the other part of the trigger arm. A part 16, protruding from the handle, is shaped as a hook 16 or an L-profile and penetrates down in the groove 17.
25 Thereby the lateral guide is created. The hook 16 is created in that there is an aperture between the end point 50 of the hook and the top edge 51 of the lateral wall 52 of the handle 3. Since this aperture normally points downwards there is little risk that dirt particles might penetrate into the handle through the aperture. However, obviously the parts could also be arranged inversely so that the
30 aperture points upwards. The protruding part 18 is thus protruding up into this aperture, which has an adapted length perpendicularly to the plane of the paper, so that the protruding part 18, and consequently the trigger arm, can move a

desirable distance in the corresponding direction. For, at least on one side of this aperture the protruding part from the handle 3, that is the hook 16, will preferably end up with a side wall or end wall.

Hereby there will be at least one end wall that connects the hook 16 with
5 the side 52 of the handle 3. The lower part of the end wall extends between the points 50 and 51 according to the dash-dotted line. This strengthens the hook 16 substantially at the same time as it preferably serves as an abutment 14 for the protruding part 18 of the trigger arm 1. The end wall is thus located above the plane of the paper. The abutment 14 is thus integrated with the lateral guide and
10 both are arranged on the handle 3. The abutment could also be arranged somewhere else on the handle unit 15, e.g. so that it cooperates with the trigger arm at a part located between its pivot and its other outer end 19. However, preferably the abutment is arranged on the tool so that it cooperates with the trigger arm at a part located between its pivot 10 and its outer end 13 above the
15 rear handle, so that, when the operator's hand or arm violently actuates the trigger arm, this will be pressed against the abutment 14, resulting in less strains on the lever and the transfer mechanism 6, 7, 8. The part of the trigger arm that is provided with this lateral guide can be either the bottom part 27, as shown in the figure, or the top part 28. This implies also the lateral guides according to the
20 figures 3 and 4.

Figure 3 illustrates a trigger arm 1 with a protruding part 53, which is so adapted that it will create such a wide groove 54 between itself and the other part of the trigger arm 1 that the handle 3 goes into the groove 54. The trigger arm is thus sitting partly astraddle of the handle 3, which is illustrated by a continuous-
25 line. However, the arm could also be drawn downwards, as shown by dash-dotted lines, so that it is situated completely astraddle of the handle 3. The downward extension of the part 53 could thus also be journalled in the handle unit 15. Thereby a considerably wider pivotal width for the trigger arm 1 will be created. The part 53 could also be made very thin since the leftwards forces give rise to
30 tensions in the part 53, while any forces to the right will be taken up in that the arm 1 presses against the handle 3. A wider pivotal width would thereby result in

a somewhat different lateral guide. Preferably the handle 3 also has a protruding abutment 14.

In figure 4 the abutment 14 is embodied as a protruding part 54, which has an outer part 55 creating a groove between itself and the very handle 3. The width of the groove is so adapted that the trigger arm can be displaced inside it. Thereby the trigger arm is laterally guided.

The lateral guides illustrated in the figure 2-4 are all arranged at a part of the trigger arm that is located between its pivot 10 and its protruding outer end 13 above the rear handle. These guides are useful for a trigger arm both embodied as a double-armed lever and as a single-armed lever.

For a trigger arm embodied as a double-armed lever the lateral guide can be carried out both at the upper part of the trigger arm, as described above, or in the corresponding way at the lower part located between its pivot 10 and its other outer end 19. The trigger arm can also be laterally guided by resting against the handle unit 15 at an upper part when under load towards the handle unit, or, under load in the opposite direction, resting against the handle unit at a lower part located between the pivot 10 of the trigger arm and its other outer end 19. The trigger arm is thus in all cases laterally guided at least at one part of the trigger arm located between its pivot point and its respective outer end 13, 19.

So far we have examined the conditions valid for the bearing and guiding of the trigger arm 1. In the following will be described in short how the movement of the trigger arm according to the arrow 29 actuates the brake 9 to stop the movement of the saw chain. As mentioned above the wire 6 is attached to the outer end 19 of the trigger arm. This wire extends inside its casing 7 up to a holder 46 mounted to the tool's engine body. The holder 46 has a travelling trolley 8, which is movable in the longitudinal direction of the holder and spring-loaded in the direction away from the trigger arm 1. The wire 6 is mounted to this travelling trolley 8. This means that when the trigger arm 1 rotates in the direction according to the arrow 29 the wire 6 will pull the trolley 8 against the spring-load from push-back spring 47. The travelling trolley 8 has an aperture 44. Into this aperture a pin will penetrate when mounting of the clutch housing. The pin is a part of a component of the brake 9. The brake 9 acts against the exterior

periphery of a clutch drum 45, which rotates around crankshaft centre 56, as becomes apparent from figure 1.

Figure 5 thus shows the clutch housing 25 from its rear side so that the brake 9 becomes apparent. The crankshaft centre 56 becomes evident from figure 5 and figure 1. The clutch drum 45 rotates around this centre. It has a cylindrical exterior surface, which on the inside is affected by the engine's centrifugal clutch, and on the outside is affected by a brake band 35 integrated with the brake 9, which is completely mounted into the clutch housing 25. The brake 9 has a brake band 35, which extends overbearingly one turn round the clutch drum 45. Its one end is attached to the clutch housing while its other end is attached to a compressed pressure spring 36. The pressure spring is held compressed by a toggle-joint mechanism 37 and 38. The part 38 has wing-shaped rejects 39. These are affected by the kickback guard so that the part 38 is rotated in the counter-clockwise direction when the kickback guard is pushed forwards towards the saw bar. Hereby the toggle-joint mechanism is no longer able to keep the pressure spring 36 compressed and thereby the pressure spring will pull the one end of the brake band 35 so that the brake band will tighten against the on the inside rotating clutch drum. This is entirely conventional and will therefore not be described in further detail. However, on the other hand a pivotable arm 40 is pivoted to the pivot point 41 in the clutch housing. Its one end 42 is pressed onto the part 38 of the toggle-joint mechanism. The other end of the pivotable arm is provided with a pin 43. When mounting the clutch housing this pin will penetrate into an aperture 44 embodied in the trolley 8, which will be pulled by the triggering wire 6 in the holder 46 against the action from the push-back spring 47. Since the pin 43 fits in the aperture 44 the clutch housing does not have to be permanently connected to the triggering wire 6, which simplifies dismantling of the saw in most cases. Consequently, when the triggering wire 6 pulls the pin 43 the other end of the pivotable arm 40 will press on the part 38 of the toggle-joint mechanism so that this will be angled out and the spring will be released in order to tighten the brake. After release the trigger device will be reset to its origin position in that the kickback guard 23 is pulled against the handle frame 24

resulting in that the part 38 is rotated in the clockwise direction, according to figure 5.

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CLAIMS

1. A brake triggering device with a pivotably mounted trigger arm (1) located in connection to a handheld working tool's, mainly a chain saw's (2), rear handle (3) with a throttle control (4) and a handle opening (5) located below the handle (3), so that the trigger arm (1), when the saw changes inclination in relation to the operator's forearm, e.g. at kickback or fall, is actuated by the operator's hand or arm, whereby the trigger arm (1) via a transfer mechanism (6, 7, 8, 40) actuates a brake (9) so that this stops the movement of the saw chain, characterized in that the pivot (10) of the trigger arm is located in connection to a front (11) or a bottom (12) side of the handle opening (5).

2. A brake triggering device according to claim 1, characterized in that the pivot (10) of the trigger arm is located in front of the handle opening (5).

3. A brake triggering device according to claim 1 or 2, characterized in that the trigger arm and/or the tool's handle unit (15) is so arranged that the trigger arm is laterally guided at least at a part of the trigger arm (1) located between its pivot and its respective outer end (13, 19).

4. A brake triggering device according to claim 3, characterized in that the lateral guide is arranged at a part of the trigger arm (1) located between its pivot (10) and its protruding outer end (13) above the rear handle.

5. A brake triggering device according to any one of the preceding claims, characterized in that an abutment (14) is arranged on the tool so that it cooperates with the trigger arm at a part of the trigger arm located between its pivot (10) and its respective outer end (13, 19), so that when the operator's hand or arm violently actuates the trigger arm this will be pressed against the abutment (14), resulting in reduced load on the transfer mechanism (6, 7, 8).

6. A brake triggering device according to any one of the preceding claims, characterized in that the trigger arm is located on one side of the rear handle (3) and the therewith integrated handle unit (15) in front of the handle opening (5).

7. A brake triggering device in accordance to any one of the claims 4-6,
c h a r a c t e r i z e d in that the lateral guide is arranged in form of one from the
trigger arm protruding part (18) arranged so that it will create a groove (17)
between itself and the other part of the trigger arm, and one from the handle
5 protruding part (16) shaped as a hook (16) or an L-profile, which protrudes down
into the groove (17) and runs in it.

8. A brake triggering device according to any one of the preceding claims,
c h a r a c t e r i z e d in that the lever is embodied as a double-armed trigger
arm, where its protruding outer end (13) is located above the handle and its other
10 outer end (19) is connected to a part of the transfer mechanism, e.g. a wire 6.

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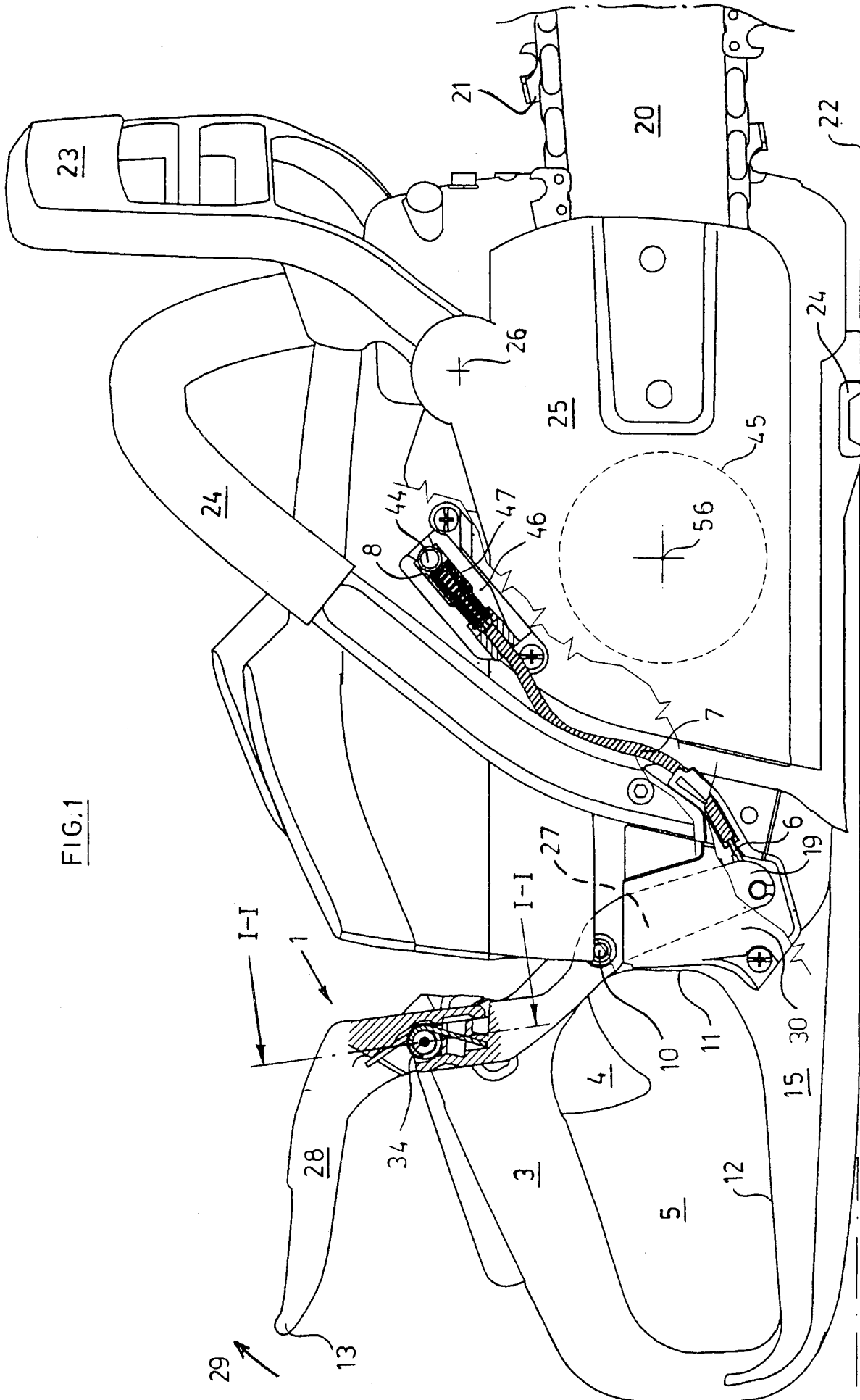


FIG. 1

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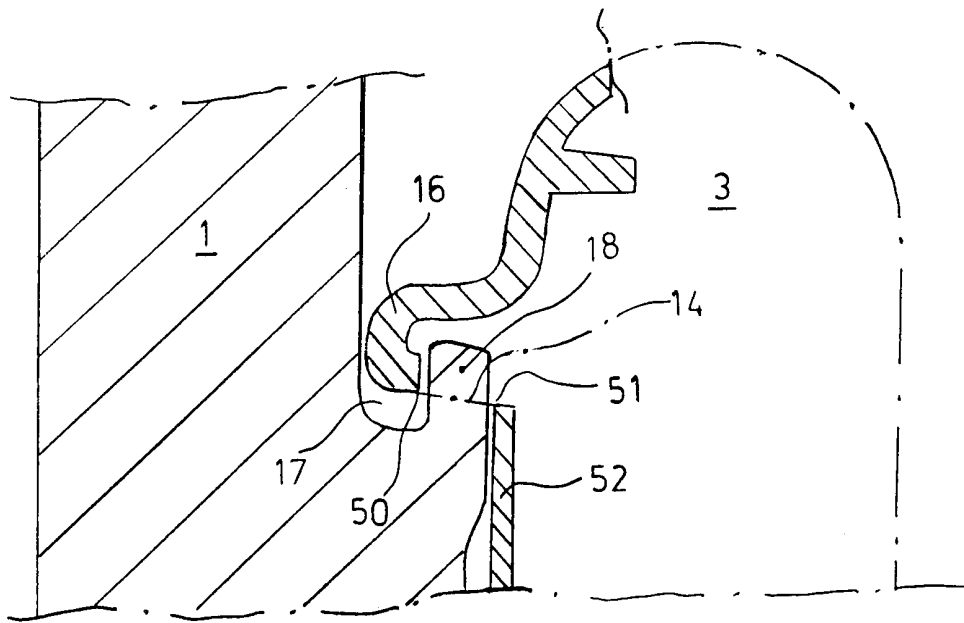
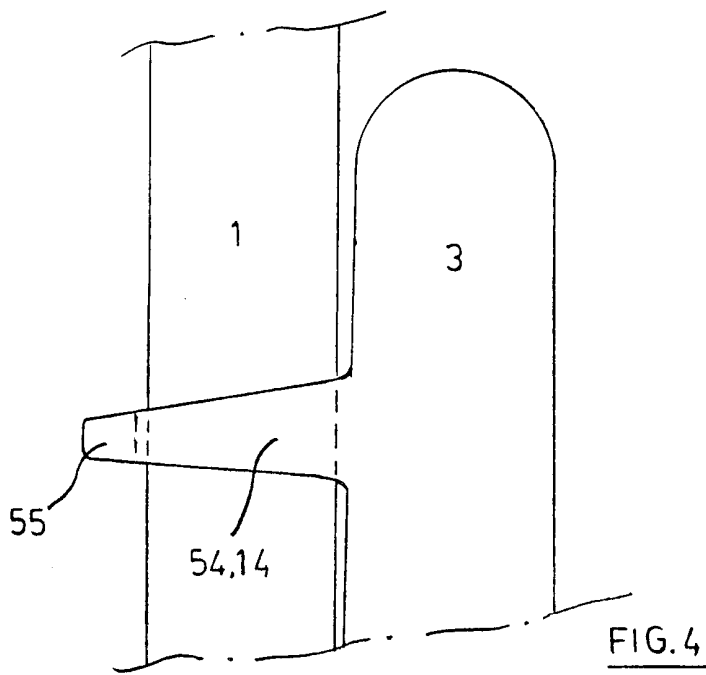
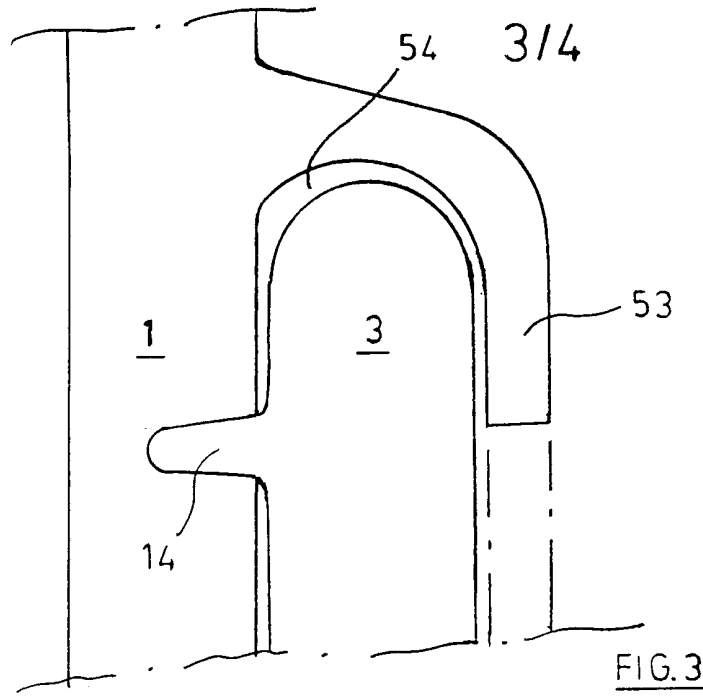
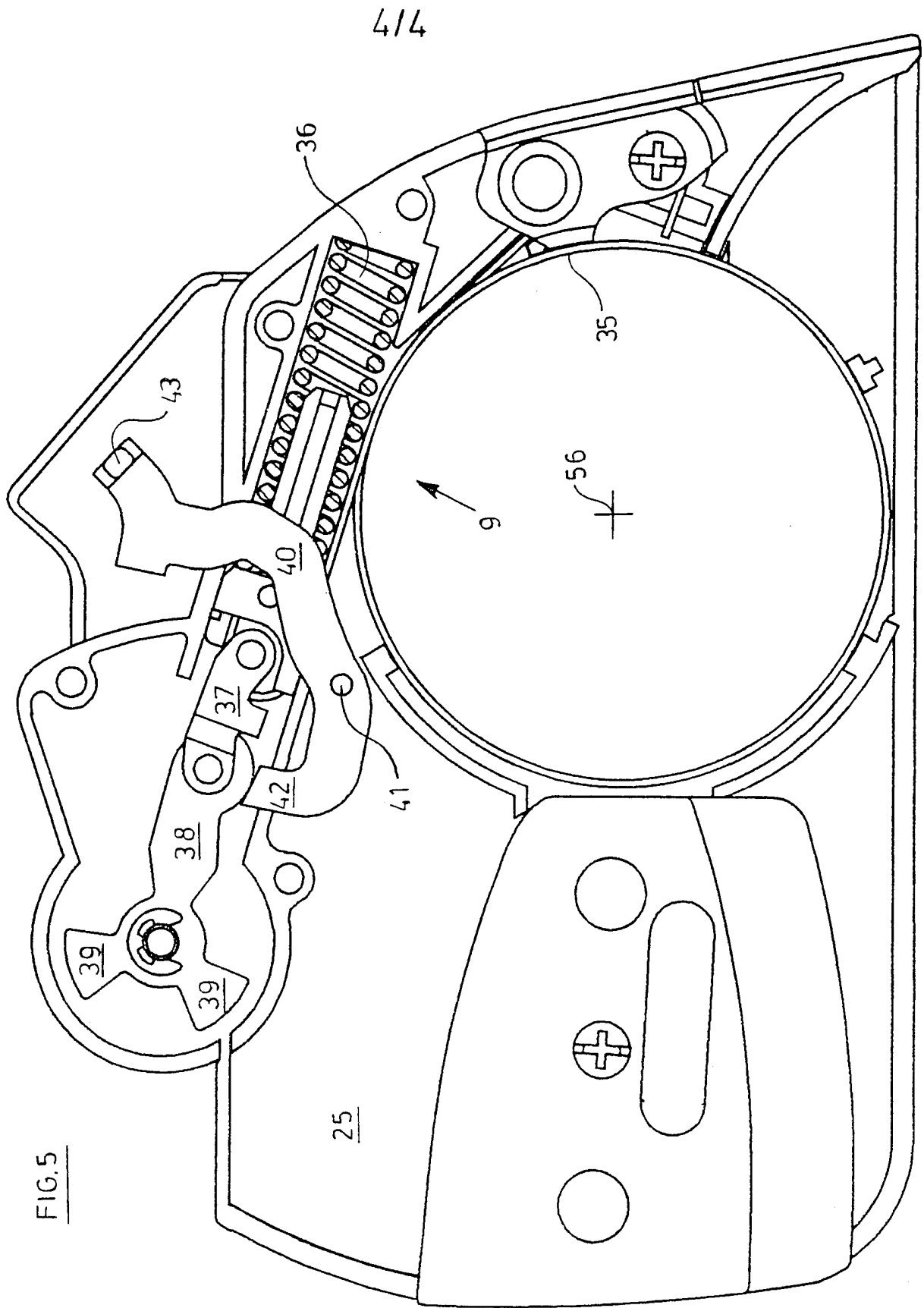


FIG. 2





INTERNATIONAL SEARCH REPORT

International application No.

PCT/SE 00/02276

A. CLASSIFICATION OF SUBJECT MATTER

IPC7: B27B 19/00

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC7: B27B, B27G

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

SE,DK,FI,NO classes as above

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	WO 8604294 A1 (WALLMARK, KENNETH), 31 July 1986 (31.07.86), abstract --	1-8
A	SKOGEN, Volume 1, 1990, BENGT-AKE ALRIKSSON, "Förbättrat V-skydd passar flera sågmodeller" page 40 - page 41 --	1-8
A	JORD & SKOG, Volume 12, 1987, JOHAN LÖFGREN, "Äntligen ett fungerande kastskydd" page 32 - page 33 --	1-8
A	EP 0945234 A2 (ROBERT BOSCH GMBH), 29 Sept 1999 (29.09.99), abstract --	1-8

 Further documents are listed in the continuation of Box C.
 See patent family annex.

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"&" document member of the same patent family

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INTERNATIONAL SEARCH REPORT

Information on patent family members

05/02/01

International application No.

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