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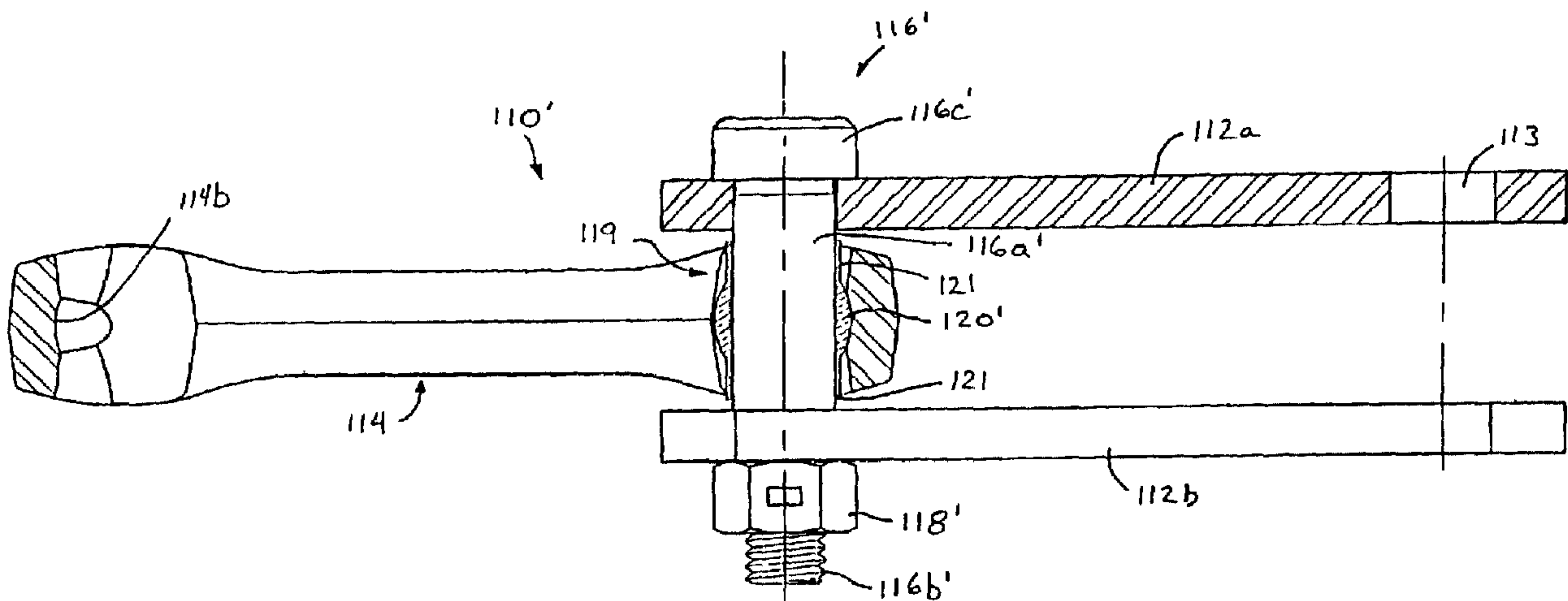
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(54) Title: CONVEYOR CHAIN



(57) **Abrégé/Abstract:**

A section of chain (10) for conveying product along a material handling system includes a pin (16) connecting a center link (14) between two side links (12a, 12b). The pin may have a sleeve portion around and at least partially along the shaft portion of the pin. The sleeve portion may engage a center link of the section of chain and may be movable relative to the shaft portion. The sleeve portion may comprise a polymeric material. The sleeve portion may be molded over an I-pin or the like, or may be slid or pressed onto a stud-ended pin or bolt or the like. The pin may include a generally spherical-shaped ball member (20) or portion on the shaft portion to enhance flexibility of the section of chain. The pin may be dialable or selectably positioned at the side links to control and adjust the wear surface of the pin relative to the chain links.

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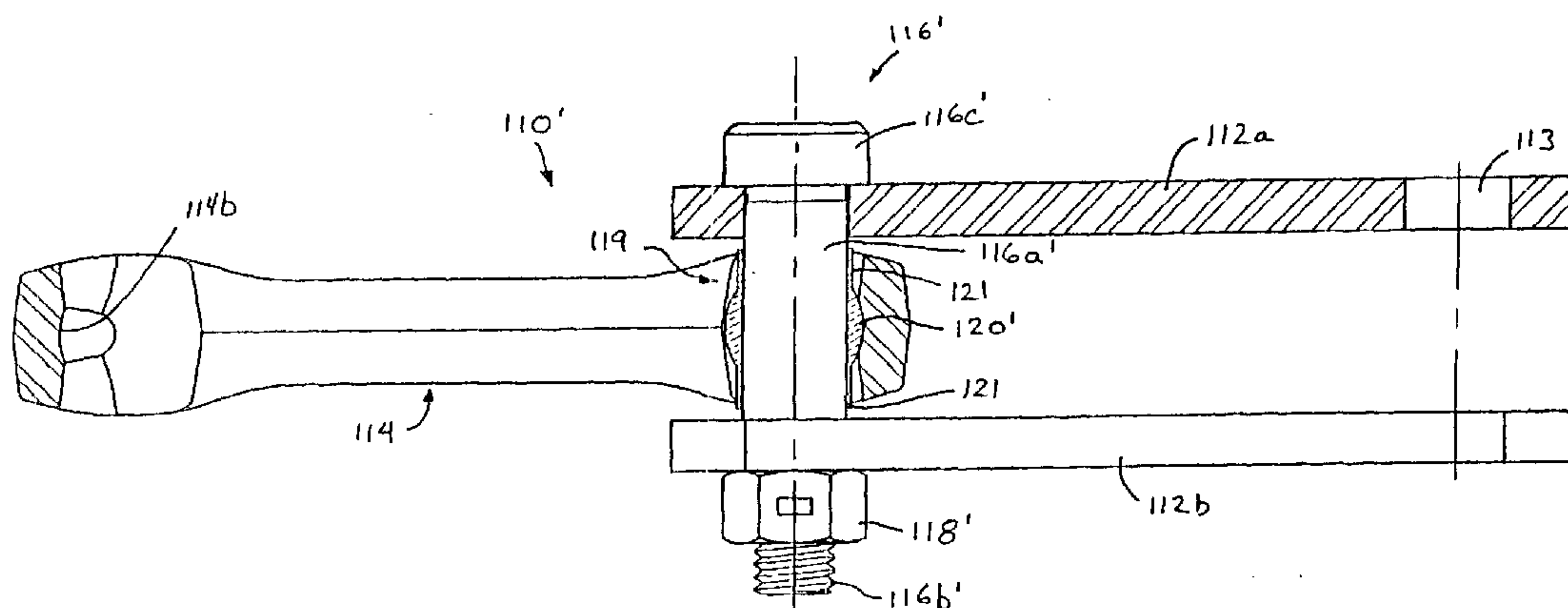
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(54) Title: CONVEYOR CHAIN



(57) Abstract: A section of chain (10) for conveying product along a material handling system includes a pin (16) connecting a center link (14) between two side links (12a, 12b). The pin may have a sleeve portion around and at least partially along the shaft portion of the pin. The sleeve portion may engage a center link of the section of chain and may be movable relative to the shaft portion. The sleeve portion may comprise a polymeric material. The sleeve portion may be molded over an I-pin or the like, or may be slid or pressed onto a stud-ended pin or bolt or the like. The pin may include a generally spherical-shaped ball member (20) or portion on the shaft portion to enhance flexibility of the section of chain. The pin may be dialable or selectably positioned at the side links to control and adjust the wear surface of the pin relative to the chain links.

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CONVEYOR CHAIN

FIELD OF THE INVENTION

The present invention relates generally to chains for a conveying system and, more particularly, to bolted or pinned chains which allow for pivotable movement between a center
5 link and a pair of side links about an axis defined by a bolt or pin extending therethrough.

BACKGROUND OF THE INVENTION

Chains for power transmission or for material handling and processing systems, such as, for example, chains which function to move a plurality of trolleys or the like along a path in a processing plant, warehouse or the like, flex as the chain travels along the desired path.
10 As the chain is routed along the desired path, the chain may flex to make lateral turns and/or upward or downward curves along the path. As the chain flexes, the links of the chain may move relative to the pins or bolts which define the joints of the chain. The relative movement of the links and the pins or bolts leads to wear on the components and may eventually lead to a failure of the links and/or the pins or bolts. The wear may increase if the friction between
15 the pin and the links increases, such as due to a roughened surface (such as by machining grooves, scratches or the like) of one or more of the components and/or a lack of lubrication on the components.

In certain situations, the chains may have to negotiate an incline from horizontal which is approximately 45–60 degrees above or below horizontal. Such sharp inclines may
20 result in binding of the chain links as they flex or bend along the curve. It is known to provide an I-pin chain with a ball formed on the forged I-pin between the side links and the center link of the chain. The I-pin may be forged with a ball or spherical shape at a center portion of the pin, such that the center link may be pivotable about the ball between the side links. The I-pin is fixed relative to the side link and center link of the chain, which is
25 commonly known as "rivetless chain." The ball may provide for additional flexibility in the chain, but still wears against the center chain links as the chain links move relative to the pins.

Therefore, there is a need in the art for an improved chain that overcomes the shortcomings of the prior art.

SUMMARY OF THE INVENTION

The present invention is intended to provide enhanced flexibility of chain links and enhanced relative movement between the chain links and pins, while reducing or controlling wear on the chain or reducing or controlling the effects of wear on the chain components.

5 According to an aspect of the invention, there is provided a pin for connecting chain links together to define a section of chain for conveying product along a conveying system comprises opposite end portions for engaging at least one link of the section of chain, a shaft portion extending between the opposite end portions, and a sleeve portion around and at least partially along the shaft portion. The sleeve portion is arranged on the shaft portion to
10 engage another link of the section of chain when the opposite end portions are engaged with the at least one link of the section of chain. The sleeve portion is movable relative to the shaft portion.

 In one form, the sleeve portion comprises a generally spherical or toroidal-shaped member, which may be generally centrally positioned along the shaft portion. The sleeve
15 portion may include a spacer portion at one or both ends of the toroidal-shaped member. The shaft portion of the pin may be generally cylindrical or may have a spherical or ball or rounded portion formed thereon. In another form, the shaft portion may comprise a generally cylindrical shaft portion and the sleeve portion may comprise a generally cylindrical sleeve portion.

20 The sleeve portion may comprise a polymeric material. The sleeve portion may be molded onto the shaft portion of the pin, which may be an I-pin or bolted pin or the like, or the sleeve portion may be slid or removably positioned over the shaft portion of a bolted pin or stud type pin when a fastener is removed from a fastener end of the pin.

 Optionally, the sleeve portion may comprise a low coefficient of friction coating on
25 an inner surface of the sleeve portion which slidably engages the shaft portion, or the sleeve portion may comprise a low coefficient of friction coating on an outer surface of the sleeve portion which engages the other chain link.

 According to another aspect, there is provided a section of chain for conveying product along a conveying system, said section of chain comprising a first chain link; at least
30 one second chain link having apertures through opposite ends thereof; and a pin for connecting said first and second chain links together to define a section of chain for conveying product along a conveying system, said pin comprising opposite end portions for engaging said second link of the section of chain; a shaft portion extending between said opposite end portions; and a sleeve portion around and at least partially along said shaft

portion, said sleeve portion being arranged on said shaft portion to engage said first link of the section of chain when said opposite end portions are engaged with said second link of the section of chain, said sleeve portion being movable relative to said shaft portion, wherein said pin further comprises one of a bolted pin and a stud, at least one of said opposite end portions comprising a fastener end for receiving a fastener thereon to retain said pin at the chain links of the section of chain.

The aperture may comprise a slotted opening having a narrowed region adjacent to the recessed area and a larger diameter region adjacent to the narrowed region. The head portion of the pin is insertable through the larger diameter region and the shaft portion is slidable through the narrowed region to the recessed area, whereby the mating surface of the head portion engages the recessed area to retain the pin in the aperture.

According to yet another aspect, there is provided a section of chain for conveying product along a conveying system comprises at least two chain links having apertures through opposite ends thereof and a pin connecting the chain links together via insertion of the pin through the apertures in the chain links. The pin comprises opposite end portions and a shaft portion extending between the opposite end portions. The shaft portion defines a wear surface which engages at least one of the chain links. The pin is selectably adjustable between at least three positions relative to the chain links to adjust the wear surface of the shaft portion relative to the chain links. The pin is generally non-rotatable relative to the chain links when in each of the at least three positions.

At least one of the end portions of the pin may define at least three engaging portions for selectably engaging corresponding wall portions of the aperture of at least one of the chain links, whereby the pin is generally non-rotatable relative to the chain links by engagement of at least some of the engaging portions with at least some of the wall portions when in each of the at least three positions.

According to yet another aspect, there is provided a method of adjusting a wear surface of a pin of a section of chain comprises providing a section of chain having at least two chain links having apertures through opposite ends thereof and a pin connecting the chain links together via insertion of the pin through the apertures in the chain links. The pin comprises opposite end portions and a shaft portion extending between the opposite end portions. The shaft portion defines a wear surface which engages at least one of the chain links. The pin is non-rotatably positioned relative to the chain links in one of at least two positions. The pin is generally non-rotatable relative to the chain links when in each of the at least two positions. The pin is selectably rotated to a different one of the at least two

positions relative to the chain links to adjust the wear surface of the shaft portion relative to the chain links.

A degree of wear on the pin may be monitored and the pin may be selectably rotated to limit or control wear on wear surfaces of the pin corresponding to the at least two
5 positions. The pin may be marked to indicate which of the at least two positions have been selected and used.

According to yet another aspect of the present invention, a section of chain for conveying product along a conveying system comprises at least two chain links and a pin for connecting the chain links together. At least one of the chain links defines a pin engaging
10 region. The pin engaging region defines a first wear surface. The pin has opposite end portions and a shaft portion and defines a second wear surface. At least one of the first and second wear surfaces comprises a polymeric material.

At least one of the pin engaging regions and the pin has a low coefficient of friction coating thereon. The coating may comprise a diamond like coating at at least one of the first
15 and second wear surfaces.

According to another aspect of the present invention, a section of chain for conveying product along a conveying system comprises at least two chain links and a pin for connecting the chain links together. At least one of the chain links defines a pin engaging region. The pin engaging region defines a first wear surface. The pin has opposite end portions and a
20 shaft portion and defines a second wear surface. At least one of the first and second wear surfaces comprises a replaceable wear surface, whereby the replaceable wear surface is removable from the at least one chain link and/or removable from the pin to facilitate replacement of the replaceable wear surface.

The replaceable wear surface may comprise a polymeric material. The replaceable
25 wear surface may comprise a sleeve positionable around the shaft portion of the pin. The sleeve may comprise a toroidal-shaped member. The replaceable wear surface may comprise an insert at an end portion of the at least one chain link, whereby the insert engages the shaft portion of the pin. The pin engaging regions and/or the pin may have a low coefficient of friction coating thereon, such as a Teflon coating or a diamond like coating or the like, at the
30 first and/or second wear surfaces.

According to another aspect of the present invention, a pin for connecting and retaining at least two chain links of a section of chain comprises a shaft portion and a replaceable wear surface member removably positioned around at least a portion of the shaft portion. The section of chain is movable to convey product along a conveying system. The

replaceable wear surface is removable from the pin to facilitate replacement of the replaceable wear surface.

The pin may comprise at least one fastening portion at at least one end of the pin for receiving a fastener thereon to retain the pin at the chain links. The replaceable wear surface may be removable and replaceable via sliding the replaceable wear surface off from and onto the shaft portion at the fastening portion when the fastener is removed therefrom. The replaceable wear surface may comprise a toroidal-shaped member or a generally cylindrical member or the like. The replaceable wear surface may comprise a metallic or a polymeric material and may be coated with a low coefficient of friction material or coating.

According to yet another aspect of the present invention, an I-pin for a chain includes opposite head portions and a shaft portion. A unitary polymeric sleeve is movably positioned around at least a portion of the shaft portion of the I-pin. The polymeric sleeve may rotate or slide around the shaft portion.

The sleeve may be molded onto the shaft portion and may be broken free after it has been molded and, thus, may be generally freely rotatable about the shaft portion of the I-pin. The sleeve provides a unitary polymeric component on the shaft portion of the I-pin for engagement with the center link and/or side links of a chain. The sleeve may be molded of a durable polymeric material.

Optionally, the shaft portion of the I-pin may have a smooth exterior surface to facilitate loosening or breaking free of the sleeve about the shaft portion. A low coefficient of friction surface or material may be provided between the sleeve and the pin and/or between the sleeve and the chain links.

In one form, the polymeric sleeve may be molded in a generally spherical shape to provide a generally spherical or toroidal-shaped ball member on the shaft portion of the pin. In another form, the shaft portion of the pin may include a ball member, such that the sleeve is molded over the ball member of the pin. In another form, the sleeve may be a generally cylindrical sleeve molded over a generally cylindrical shaft portion of the pin.

According to another aspect of the present invention, a method for forming an I-pin includes forming a pin having a shaft portion and molding a polymeric sleeve portion onto the shaft portion of the I-pin. The sleeve portion may be a generally cylindrical sleeve portion or may be a generally spherically shaped ball member molded onto the shaft portion of the pin. The sleeve portion may be broken free from the shaft portion of the pin such that the sleeve portion is movably or slidably positioned on the shaft portion.

Therefore, the present invention provides a pin member or pin which may provide enhanced flexibility and wear life to a section of chain. The pin may have a movable sleeve portion around a shaft portion of the pin to reduce friction and wear on the pin and on the chain links of the section of chain. The pin may include a ball member or toroidal-shaped member at or on the shaft portion of the pin, and the center link may include a concave engaging surface for engaging the ball member, to enhance flexibility of the section of chain. The present invention may provide a rotatable pin for rotating within the chain links as the chain travels and bends along the conveying path. Optionally, the chain may be selectably rotated or dialed between two or more generally fixed orientations relative to the chain links to control the wear on the pin and enhance the life cycle of the pin. Optionally, the pin or one or more of the chain links may have a wear surface which comprises a polymeric material, which may reduce friction between the pin and chain links. The polymeric wear surface may be a polymeric insert at an end of the center link or may be a polymeric sleeve portion molded or positioned around the shaft portion of the pin. The pin may comprise an I-pin type of chain pin, a bolted type of chain pin or a double ended stud type of chain pin, and may include a generally spherical-shaped ball member or toroidal-shaped member at the shaft portion to enhance flexibility in the section of chain. Optionally, the pin and/or the chain link may have a replaceable wear surface member to facilitate replacement of the wear surface or surfaces, without having to replace the entire pin and/or chain link.

These and other objects, advantages, purposes and features of the present invention will become apparent upon review of the following specification in conjunction with the drawings.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is an exploded perspective view of a ball and socket bolted chain or section of chain in accordance with the present invention, having a double ended stud type pin;

FIG. 2 is a top plan view of the section of chain of FIG. 1;

FIG. 3 is a side elevation and partial sectional view of the section of chain of FIGS. 1 and 2;

FIG. 4 is an exploded perspective view of another ball and socket bolted section of chain in accordance with the present invention;

FIG. 5 is a top plan view of the section of chain of FIG. 4;

FIG. 6 is a side elevation and partial sectional view of the section of chain of FIGS. 4 and 5;

FIG. 7 is a top plan view of another ball and socket bolted section of chain in accordance with the present invention;

FIG. 8 is a side elevation and partial sectional view of the section of chain of FIG. 7;

FIG. 9 is an exploded perspective view of another ball and socket section of chain in
5 accordance with the present invention;

FIG. 10 is a side elevation and partial sectional view of the ball and socket section of chain of FIG. 9;

FIG. 11 is a perspective view of another ball and socket bolted section of chain in accordance with the present invention, with one of the side links removed;

10 FIG. 12 is a top plan view of the section of chain of FIG. 11;

FIG. 13 is a side elevation and partial sectional view of the section of chain of FIGS. 11 and 12, with both side links shown;

FIG. 14 is a top plan view of another bolted section of chain with the bolted pin being rotatable relative to the chain links;

15 FIG. 15 is a side elevation and partial sectional view of the section of chain of FIG. 14;

FIG. 16 is a perspective view of a ball and socket I-pin section of chain in accordance with the present invention with a pin rotatably received through the chain links, with one of the side links removed to show additional detail of the pin;

20 FIG. 17 is a top plan view of the section of chain of FIG. 16;

FIG. 18 is a side elevation and partial sectional view of the section of chain of FIGS. 16 and 17, with both side links shown;

FIG. 19 is a top plan view of another I-pin section of chain with the I-pin being rotatable relative to the chain links;

25 FIG. 20 is a side elevation and partial sectional view of the section of chain of FIG. 19;

FIG. 21 is a perspective view of another ball and socket I-pin section of chain in accordance with the present invention, with the pin being selectably non-rotatably positioned at the chain links, shown with one of the side links removed to show additional detail of the
30 pin;

FIG. 22 is a top plan view of the section of chain of FIG. 21;

FIG. 23 is a perspective view of another ball and socket I-pin section of chain with the pin being selectably non-rotatably positioned at the chain links, shown with one of the side links removed to show additional detail of the pin;

FIG. 24 is a top plan view of the section of chain of FIG. 23;

FIG. 25 is a side elevation and partial sectional view of the section of chain of FIGS. 23 and 24, with both side links shown;

FIG. 26 is a top plan view of another section of chain similar to the section of chain of FIGS. 23-25, with a pin having a generally cylindrical shaft portion;

FIG. 27 is a side elevation and partial sectional view of the section of chain of FIG. 26;

FIG. 28 is a perspective view of another section of chain similar to the sections of chain of FIGS. 23-27, with a bolted-pin being non-rotatably positioned at the section of chain, shown with one of the side links removed to show additional detail of the bolted pin;

FIG. 29 is a top plan view of the section of chain of FIG. 28;

FIG. 30 is a side elevation and partial sectional view of the section of chain of FIGS. 28 and 29;

FIG. 31 is a top plan view of another section of chain similar to the section of chain of FIGS. 28-30, with a bolted-pin having a generally cylindrical shaft portion;

FIG. 32 is a side elevation and partial sectional view of the section of chain of FIG. 31;

FIG. 33 is an exploded perspective view of a ball and socket section of chain in accordance with the present invention, with a ball member molded around a shaft portion of a pin;

FIG. 34 is a side elevation and partial sectional view of the section of chain of FIG. 33;

FIG. 35 is a side elevation and partial sectional view of another ball and socket section of chain in accordance with the present invention;

FIG. 36 is a side elevation and partial sectional view of another ball and socket section of chain in accordance with the present invention, with a sleeve molded around the shaft portion and ball member of the pin;

FIG. 37 is a side elevation and partial sectional view of a section of chain in accordance with the present invention, with a generally cylindrical sleeve molded over a generally cylindrical shaft portion of the pin;

FIG. 38 is a perspective view of a center link and pin of another section of chain in accordance with the present invention, with a plastic insert at each end of the center link;

FIG. 39 is a side elevation and partial sectional view of another section of chain in accordance with the present invention, with a plastic insert at each end of the center link and a ball member formed on the shaft of the pin; and

FIG. 40 is a side elevation and partial sectional view of another section of chain in accordance with the present invention, with a rounded plastic insert at each end of the center link and a concave or narrowed shaft of the pin.

DESCRIPTION OF THE PREFERRED EMBODIMENTS

Referring now to the drawings and the illustrative embodiments depicted therein, a chain or section of chain 10 for conveying product along a conveying system, such as for material handling or processing systems or the like, includes a pair of side links 12a, 12b and a center link 14 (FIGS. 1-3). The center link 14 is retained between the side links 12a, 12b by a double ended stud or stud type pin 16, which extends through a center region 14a of center link 14 and through an opening 13 in each side link 12a, 12b and is retained therein by a corresponding fastener or nut 18. Stud 16 includes a generally spherical or toroidal-shaped ball member 20 positioned generally at a mid-point or mid-region of a shaft portion 16a of stud 16. Chain 10 includes multiple linkages connected together in a continuous loop about a conveying system, as is known in the art. For ease of description, only one section or set of linkages of the chain is shown and described herein, with the other linkages of the chain being substantially identical.

Stud 16 includes center or shaft portion 16a and a pair of opposite threaded portions or fastener portions or ends 16b. Threaded portions 16b are of a narrower diameter than shaft portion 16a, such that the ends of center portion 16a provide an abutting surface 16c against the fastener 18 as the fastener is tightened onto stud 16 or for abutting against an inward side of the respective side link 12a, 12b, to maintain the spacing between the side links when the chain is assembled. Stud 16 may comprise a metallic material, such as steel, stainless steel or iron or the like, similar to conventional studs or pins of chain for material handling systems and the like.

Ball member 20 may be integrally formed as part of stud 16 or may be press-fit or otherwise secured on shaft portion 16a to retain the position of ball member 20 in the middle region of stud 16. However, ball member 20 may otherwise be loosely fit onto shaft portion 16a of stud 16 and allowed to slide along or rotate around stud 16, without affecting the scope of the present invention. Ball member 20 may comprise a metallic material or a plastic or polymeric material. Center link 14 is a generally oval shaped or elongated ring and includes an inner rounded or concave mating or engaging surface 14b (FIGS. 2 and 3) at each end

thereof for engaging and partially receiving ball member 20 therein to retain center link 14 at ball member 20 as the chain is moved along the conveying path.

Ball member 20 thus allows for pivotal movement of center link 14 via sliding engagement of concave surface 14b along ball member 20. This provides greater flexibility to bolted chain 10 and may allow bolted chain 10 to negotiate sharper vertical curves in the conveying path without binding the links or joints of the chain. The ball member and correspondingly formed concave surface thus may distribute the loads over a generally constant surface area, reducing or substantially eliminating the stress concentration that typically occur when conventional chains articulate through vertical inclines and declines. The ball member and concave surface engagement may also function to distribute the loads between the chain links and stud or pin over a greater surface area than conventional chains. This decreases the wear on the bolt and chain links and may result in less maintenance and a greater life cycle for the chain.

Referring now to FIGS. 4-6, a chain or section of chain 110 includes a pair of side links 112a, 112b and a center link 114. The center link 114 is retained between corresponding ends of the side links 112a, 112b by a bolt or bolted pin 116 extending through corresponding openings 113 and through a center region 114a of center link 114. Bolt 116 includes a shaft portion 116a, a threaded portion or end 116b, which threadedly receives a nut 118 thereon to retain the chain links together, and a head or head portion 116c at an opposite end from threaded end 116b. Bolt 116 further includes a generally spherical or generally toroidal-shaped ball member 120, which may slide onto shaft portion 116a of bolt 116. Similar to center link 14, center link 114 is a generally elongated ring with a rounded or concave surface 114b at each end thereof for engaging and partially receiving ball member 120 therein to maintain center link 114 on ball member 120.

Ball member 120 may be loosely fit along bolt 116 to ease assembly of the chain links and to allow the ball member 120, and thus the center link 114, to move between the side links as the chain links negotiate through the conveying path. Optionally, ball member 120 may snugly receive shaft portion 116a or may be press fit onto shaft portion 116a to retain ball member 120 in place on shaft member 116a of bolt 116. Because ball member 120 defines a wear surface of bolt 116, ball member 120 may be removably positioned on shaft portion 116a to allow the ball member and wear surface to be replaced without replacing the entire bolted pin, thereby reducing maintenance and replacement costs of the chain components.

Optionally, as shown in FIGS. 7 and 8, a chain or section of chain 110' includes side links 112a, 112b and a center link 114. The center link 114 is retained between corresponding ends of the side links 112a, 112b by a bolt or bolted pin 116' extending through corresponding openings 113 and through center region 114a of center link 114. Bolt 116' includes a shaft portion 116a', a threaded portion or end 116b', which threadedly receives a nut 118' thereon to retain the chain links together, and a head or head portion 116c' at an opposite end from threaded end 116b'. Bolt 116' further includes a sleeve portion or a spacer-ball-spacer assembly or member or portion 119 (FIG. 8), which may include a generally spherical or toroidal-shaped ball member or portion 120' and generally cylindrical spacers or sleeves 121 positioned at opposite ends of the ball member 120'. As discussed above, center link 114 may be a generally elongated ring with a rounded or concave surface 114b at each end thereof for partially receiving ball member 120' therein to maintain center link 114 on ball member 120'.

Sleeve portion 119 may slide onto shaft portion 116a' of bolt 116' or may be molded thereon. Sleeve portion 119 may comprise a metallic material, such as steel or the like, and may be treated or coated, such as with a low coefficient of friction material or slick material, such as a Teflon material or a diamond like coating (DLC) or other suitable low friction material or the like, such that sleeve portion 119 provides a highly durable and slick wear surface or engaging surface along shaft portion 116a of bolt or pin 116. Optionally, sleeve portion 119 may comprise a polymeric material and may be slid onto or molded onto shaft portion 116a, which may comprise a metallic material.

Sleeve portion 119 may be loosely fit along bolt 116' to ease the assembly of the chain and to allow the ball member 120' and spacers or sleeves 121, and thus the center link 114, to move between the side links as the chain links negotiate through the conveying path. Sleeve portion 119 may include a hollow spacer ring or sleeve 121 positioned around and along the shaft 116a' at one or both sides of ball member 120' to maintain ball member 120' generally at a center region of the shaft 116a' of pin 116'. The spacer-ball-spacer configuration or assembly may be formed as a unitary member (as shown in FIG. 8) or may be separate components (one or two spacers and a ball member) slid onto the shaft portion 116a', without affecting the scope of the present invention. By making the spacer-ball-spacer configuration a unitary structure, assembly of the bolt and maintenance or replacement of the wear surface/ball member (if necessary) may be substantially simplified.

Such a spacer-ball-spacer configuration assists in maintaining the ball member in the center region of the shaft of the pin, while allowing for generally free rotational movement of

the ball member about the shaft. This helps to keep the ball member in alignment with the recess 114b in center link 114 as the chain bends and moves along its conveying path. The spacer-ball-spacer configuration also provides for low cost replacement of the wear portions of the pin, since the spacer-ball-spacer assembly may be replaced without replacing the entire
5 pin. For example, the nut may be removed and the spacer-ball-spacer structure (preferably a unitary structure) may be slid off of the bolt and replaced, thereby providing a new wear surface (a new ball member) to the bolted pin, without replacing the entire bolted pin. This may provide significant cost savings, especially in applications where the pin may be formed of stainless steel or other expensive materials and, thus, may be costly to replace.

10 Referring now to FIGS. 9 and 10, a chain or section of chain 210 includes a pair of side links 215, 212 and a center link 214. The center link 214 is retained between the side links 215, 212 by a threaded or bolted pin 216 which extends through a center region 214a of center link 214 and through an opening 215a in side link 215 and an opening 213 in side link 212 and is retained therein and therethrough by a corresponding fastener or nut 218.

15 Threaded pin 216 includes a generally spherical or toroidal-shaped ball member 220 positioned generally at a mid-point or mid-region of threaded pin 216. Similar to chain 10, chain 210 includes multiple linkages connected together in a continuous loop about a conveying system, as is known in the art. For ease of description, only one set of linkages of the chain is shown and described herein, with the other linkages being substantially identical.

20 Threaded pin 216 includes a center portion or shaft portion 216a and a threaded portion 216b at one end of the shaft portion 216a and a head or head portion 216c at the other end of the shaft 216a. Threaded portion 216b has a narrower diameter than center portion or shaft 216a, such that the end of shaft 216a provides an abutting surface 216d for abutting against the fastener or nut 218 as the nut is tightened onto threaded pin 216 or for abutting
25 against an inward side of side link 212, to maintain the spacing between the side links when the chain is assembled.

As can be seen in FIGS. 9 and 10, head 216c of threaded pin 216 is a generally rectangular shaped head such that portions of the head 216c extend laterally outward from the end of the shaft 216a in opposite directions. Side link 215 includes a pair of slotted openings
30 215a which may be sized to receive head 216c therethrough when the head is rotated approximately 90 degrees from the orientation shown in FIG. 9, in order to facilitate assembly and disassembly of the chain links. Alternately, side link 215 may include circular apertures or passageways similar to side link 212, because the threaded portion 216b and shaft 216a may be inserted through such an aperture or opening to assemble the section of

chain. Side link 215 includes a recessed region or pocket 215b at each end for head 216c of threaded pin 216 to rest when the section of chain is assembled. The recessed region 215b is defined by a raised end portion 215c and raised inward portions 215d which engage the sides of the head 216c to substantially preclude rotation of the pin relative to the side and center links.

Center link 214 may be substantially similar to center links 14 and 114, discussed above, and may comprise a generally oval shaped or elongated ring defining an opening 214a and having an inner rounded or concave surface 214b at each end of the center link for partially receiving ball member 220 therein to retain center link 214 at ball member 220 as the chain is moved along the conveying path.

Ball member 220 may be slid loosely onto shaft 216a of threaded pin 216 and allowed to slide and rotate along shaft 216a, or may be press fit onto or substantially fixed relative to shaft 216a, without affecting the scope of the present invention. Alternately, the threaded pin and ball member may be forged or otherwise integrally formed as a single member, without affecting the scope of the present invention.

Similar to ball members 20 and 120 of chains 10 and 110, respectively, ball member 220 of chain 210 allows for pivotal movement of center link 214 via sliding engagement of concave surface 214b along ball member 220. This provides greater flexibility to chain 210 and may allow the chain to negotiate sharper vertical curves in the conveying path without binding the links or joints of the chain. Chain 210 thus provides a half rivetless - half bolted pin which may negotiate sharper vertical turns and/or negotiate turns about different axes. The ball member and correspondingly formed concave surface thus may distribute the loads over a generally constant surface area, reducing or substantially eliminating the stress concentration that typically occur when conventional bolted chains articulate through vertical inclines and declines. The ball member and concave surface engagement may also function to distribute the loads between the chain links and bolt over a greater surface area than conventional bolted chains. This decreases the wear on the bolt and chain links and may result in less maintenance and a greater life cycle for the chain.

Although shown as having a generally rectangular shaped head, the threaded pin of the present invention may alternately have a rounded head (not shown in FIGS. 9 and 10) which may rest upon a generally flat or recessed side link, thereby allowing the threaded pin to rotate relative to the side and center links, without affecting the scope of the present invention.

Referring now to FIGS. 11-13, a chain or section of chain 310 includes a pair of side links 315, 312 and a center link 314. The center link 314 is retained between the side links 315, 312 by a threaded pin or bolted pin 316 which extends through a center region 314a of center link 314 and through an opening 315a in side link 315 and an opening 313 in side link 312 and is retained therein and therethrough by a corresponding fastener or nut 318. Threaded pin 316 includes a generally spherical or toroidal-shaped ball member 320 positioned generally at a mid-point or mid-region of a shaft portion 316a of threaded pin 316. Similar to chain 10, chain 310 includes multiple linkages connected together in a continuous loop about a conveying system, as is known in the art. For ease of description, only one section or set of linkages of the chain is shown and described herein, with the other sections or sets of linkages being substantially identical.

Threaded pin 316 includes center portion or shaft portion 316a, a threaded portion or end 316b at one end of the shaft portion 316a, and a head portion 316c at the other end of the shaft portion 316a. Threaded portion 316b has a narrower diameter than shaft portion 316a, such that the end of shaft portion 316a provides an abutting surface 316d for abutting against the fastener or nut 318 as the fastener is tightened onto threaded pin 316 or for abutting against an inward side of side link 312, to maintain the spacing between the side links when the chain is assembled. Head portion 316c may include a hex head or the like to facilitate preventing rotation of the bolt while the fastener 318 is being fastened thereto.

As can be seen in FIGS. 11-13, head 316c of threaded pin 316 is a generally circular-shaped head. Side link 315 includes a pair of slotted openings 315a which include a larger inward opening 315b, which may be sized to receive head 316c therethrough, in order to facilitate assembly and disassembly of the chain links, and a smaller outward opening 315c, which is smaller than the diameter of head 316c to retain head 316c therein. The smaller opening 315c defines a narrowed slotted region adjacent to larger opening 315b, to allow shaft portion 316a to slide along the slotted region to position head portion 316c at smaller outward opening 315c, whereby head portion 316c is retained in the smaller opening 315c, as discussed below. Alternately, side link 315 may include circular apertures or passageways similar to side link 312, because the threaded portion 316b and shaft portion 316a may be inserted through such an aperture or opening to assemble the section of chain.

Side link 315 includes a recessed area or region or pocket 315d at and at least partially around the smaller opening 315c at each end for head portion 316c of threaded pin 316 to rest when the chain is assembled. The lower portion or mating surface 316d (FIG. 13) of head portion 316c is preferably curved or rounded, while the recessed region 315d is

correspondingly formed to provide generally uniform engagement and mating between the mating surface 316d of head portion 316c and recessed region 315d. The correspondingly formed mating surface and recessed region facilitate substantially smooth rotation between the pin 316 and the link 315, such that the bolted pin 316 may freely rotate relative to side links 315, 312 and center link 314 as the chain travels along its conveying path.

Center link 314 may be substantially similar to the center links discussed above, and may comprise a generally oval shaped or elongated ring defining an opening 314a and having an inner rounded or concave surface 314b at each end thereof for partially receiving ball member 320 therein to retain center link 314 at ball member 320 as the chain is moved along the conveying path.

Similar to the ball members and chains discussed above, ball member 320 of chain 310 allows for pivotal movement of center link 314 via sliding engagement of concave surface 314b along ball member 320. This provides greater flexibility to chain 310 and may allow the chain to negotiate sharper vertical curves in the conveying path without binding the links or joints of the chain. Chain 310 thus provides a half rivetless - half bolted pin which may negotiate sharper vertical turns and/or negotiate turns about different axes. The ball member and correspondingly formed concave surface thus may distribute the loads over a generally constant surface area, reducing or substantially eliminating the stress concentration that typically occur when conventional bolted chains articulate through vertical inclines and declines. The ball member and concave surface engagement may also function to distribute the loads between the chain links and bolt over a greater surface area than conventional bolted chains. This decreases the wear on the bolt and chain links and may result in less maintenance and a greater life cycle for the chain.

Ball member 320 may be slid loosely onto shaft portion 316a of threaded pin 316 and allowed to slide and rotate along shaft portion 316a, or may be press fit onto or substantially fixed relative to shaft portion 316a, without affecting the scope of the present invention. Alternately, the threaded pin and ball member may be forged or otherwise integrally formed as a single member, without affecting the scope of the present invention. Optionally, ball member 320 may be loosely slid onto shaft portion 316a, and may include a hollow spacer ring or sleeve (such as shown generally at 321 in FIG. 13) positioned along and around the shaft portion 316a at one or both sides of ball member 320 to maintain ball member 320 generally at a center region of shaft portion 316a of pin 316. The spacer-ball-spacer configuration or assembly may be formed as a unitary member or may be separate components (two spacers and a ball member) slid onto the shaft portion 316a, without

affecting the scope of the present invention. By making the spacer-ball-spacer configuration a unitary structure, assembly of the bolt and maintenance or replacement of the ball member (if necessary) may be substantially simplified.

As discussed above with respect to bolted pin 116', such a spacer-ball-spacer
5 configuration assists in maintaining the ball member in the center region of the shaft of the pin, while allowing for generally free rotational movement of the ball member about the shaft. This helps to keep the ball member in alignment with the recess 314b in center link 314 as the chain bends and moves along its conveying path. The spacer-ball-spacer configuration also provides for low cost replacement of the wear portions of the pin, since the
10 spacer-ball-spacer assembly may be replaced without replacing the entire pin. The nut may be removed and the spacer-ball-spacer structure (preferably a unitary structure) may be slid off of the bolt and replaced, thereby providing a new wear surface (a new ball member) to the bolted pin, without replacing the entire bolted pin. This may provide significant cost savings, especially in applications where the pin may be formed of stainless steel or other expensive
15 materials and, thus, may be costly to replace.

Referring to FIGS. 14 and 15, a chain or section of chain 310' includes a pair of side links 315, 312 and a center link 314'. The center link 314' is retained between the side links 315, 312 by a threaded pin 316' which extends through a center region 314a' of center link 314' and through an opening 315a in side link 315 and an opening 313 in side link 312 and is
20 retained therein and therethrough by a corresponding fastener or nut 318. Threaded pin 316' includes a generally cylindrical shaft portion 316a', while center link 314' includes a generally flat or curved, non-concave surface 314b' for engaging shaft portion 316a'. Chain 310' may be otherwise substantially similar to chain 310, discussed above, such that a detailed discussion will not be repeated herein. Head portion 316c' may include a curved or
25 rounded mating surface 316d', while side link 315 may include a correspondingly formed mating surface 315d at the smaller slotted opening 315c, as discussed above, to facilitate generally smooth rotation of pin 316' relative to the chain links. Optionally, threaded pin 316' may include a generally cylindrical sleeve portion (not shown in FIGS. 14 and 15) formed or positioned around shaft portion 316a', such as discussed above or as discussed below.

30 Additionally, it is further envisioned that a round headed pin may be provided with a rounded head at each end of the pin and a ball member on the shaft portion between the heads or head portions. The head portions may insert through enlarged, generally circular openings in the side links and slide outwardly toward the ends of the side links to a narrowed opening. The narrowed opening substantially precludes the circular heads from moving therethrough,

and thus retains the side links relative to the center links and the pin. The side links may include a recessed area around the narrowed opening to receive the rounded head therein to substantially preclude longitudinal movement of the pin relative to the side link toward the enlarged opening in the side link. The pin is thus allowed to rotate relative to the side links and the center link, while allowing greater flexibility and reduced wear of the chain.

For example, and with reference to FIGS. 16-18, a chain or section of chain 410 includes a pair of side links 412a, 412b and a center link 414. The center link 414 is retained between the side links 412a, 412b by a pin 416 which extends through a center region 414a of center link 414 and through an opening 413 in each side link 412a, 412b. Openings 413 include a larger, inward generally circular opening 413a and a smaller, outer generally circular slot or opening 413b. Each head portion 416c of pin 416 is smaller than the larger opening 413a and larger than the smaller end opening 413b in each side link 412a, 412b and retains pin 416 to side links 412a, 412b and center link 414. Pin 416 includes a generally spherical or toroidal-shaped ball member 420 positioned generally at a mid-point or mid-region of a shaft portion 416b of pin 416. Center link 414 may be substantially similar to the center links discussed above and may comprise a generally oval-shaped or elongated ring having an inner rounded or concave surface 414b at each end thereof for partially receiving ball member 420 therein to retain center link 414 at ball member 420 as the chain is moved along the conveying path. Similar to chain 10, chain 410 includes multiple linkages connected together in a continuous loop about a conveying system, as is known in the art. For ease of description, only one section or set of linkages of the chain is shown and described herein, with the other sections or sets of linkages being substantially identical.

Pin 416 includes center portion or shaft portion 416a and a head portion 416c at each end of the shaft portion 416a. Head portions 416c are generally circular and include an angled or curved underside portion or mating surface 416d (FIG. 18) which extends generally radially outward from each end of the shaft portion 416a such that head portion 416c has a larger diameter than shaft portion 416a.

As can be seen in FIGS. 16 and 17, each side link 412a, 412b includes a pair of slotted openings 413 which may be sized to receive head 416c through the larger inward opening 413a, in order to facilitate assembly and disassembly of the chain. Once inserted through openings 413a, pin 416 and head portion 416c may be moved outwardly until shaft portion 416a is within outer slot or opening 413b. Head portion 416c then may rest at least partially within a curved recessed portion 413c at outer opening 413b, such that pin 416 and side links 412a, 412b and center link 414 are generally secured together. The curved

recessed portion or region 413c is generally correspondingly formed with the mating surface 416d of head portion 416c to facilitate generally uniform engagement and generally smooth rotation between the head portion of the pin and the side links. Because the recessed portion 413c and mating surface 416d of head portion 416c are generally circular and are
5 correspondingly formed, pin 416 is allowed to rotate relative to side links 412a, 412b as the chain moves along its conveying path, in order to substantially evenly distribute wear on the pin 416 and, thus, to extend the life cycle of the pin 416.

Similar to the other ball members discussed above, ball member 420 of chain 410 allows for pivotal movement of center link 414 via sliding engagement of concave surface 414b along ball member 420. This provides greater flexibility to chain 410 and may allow
10 the chain to negotiate sharper vertical curves in the conveying path without binding the links or joints of the chain. Chain 410 thus provides a pin which may negotiate sharper vertical turns and/or negotiate turns about different axes. The ball member and correspondingly formed concave surface thus may distribute the loads over a generally constant surface area,
15 reducing or substantially eliminating the stress concentration that typically occur when conventional chains articulate through vertical inclines and declines. The ball member and concave surface engagement may also function to distribute the loads between the chain links and pin over a greater surface area than conventional chains. This decreases the wear on the bolt and chain links and may result in less maintenance and a greater life cycle for the chain.

Referring now to FIGS. 19 and 20, a chain or section of chain 410' includes a pair of side links 412a, 412b and a center link 414'. The center link 414' is retained between the side links 412a, 412b by a rotatable pin 416' which extends through a center region 414a' of center link 414' and through an opening 413 in each side link 412a, 412b. Pin 416' includes a generally cylindrical shaft portion 416a', while center link 414' includes a generally flat or
25 non-concave surface 414b' for engaging shaft portion 416a'. Both head portions 416c' of pin 416' comprise a generally rounded and generally circular mating surface 416d', while side links 412a, 412b include a correspondingly formed recessed area or region or engaging surface 413c for engaging mating surfaces 416d' to facilitate generally smooth rotation between pin 416' and side links 412a, 412b. Chain 410' may be otherwise substantially
30 similar to chain 410, discussed above, such that a detailed discussion will not be repeated herein. Optionally, pin 416' may include a generally cylindrical sleeve portion (not shown in FIGS. 19 and 20) formed or positioned around shaft portion 416a', such as discussed above or as discussed below.

It is further envisioned that a head portion of a pin of a section of chain may be non-circular shaped and the recessed portion of at least one of the side links may be correspondingly non-circular shaped, such that the head portion, and thus the pin, may be non-rotatably secured relative to the side link. For example, the head portion and a recessed region of a side link may be formed with two or more sides to provide for non-rotational engagement between the head and the recessed region, while allowing the head and bolt or pin to be manually rotated to adjust or change the wear surface engagement of the shaft and ball member with the center link. For example, the pin or bolt portion (shaft and head) of a bolted pin may be ratcheted or rotated sixty degrees or ninety degrees or some other amount (depending on the number of sides of the head and the recessed region) to provide a new wear surface of the shaft and ball member (the portion that engages the center link and wears as the links bend and turn relative to one another). The adjustable or dialable pin design of the present invention thus may provide for a significant increase in the life cycle of the pin, because the ball member (or spacer-ball-spacer assembly) may be replaced as needed, and/or the pin may be selectably rotated to provide a new wear surface as needed, which may substantially extend the overall life of the pin or bolted pin.

Each wear surface may span or cover approximately sixty or ninety or one hundred twenty degrees or the like about the pin, whereby the pin may be rotated that amount as necessary to provide six or four or three different wear surfaces or wear surface portions about the pin and, thus, to facilitate control of the amount of wear or amount of time of wear on each wear surface portion. This may be especially useful in applications where the chain is an exact pitch design (the chain length is specified with a small tolerance range), and excessive wear in one or more of the pins may affect the pitch, which may cause the chain to bind or chains to bind (if two such chains are running alongside one another with something connected or cradled between them) or may otherwise adversely affect the chain or conveyor. The pins of such a chain may then be dialed or rotated to provide a new wear surface for each pin, thereby effectively resetting the pins to their original tolerances and thus resetting the chain to its initial specified length. For example, each pin may be dialed to a second position or a third position or any other position to move a fresh wear surface to the wear position. In some cases, only a first pin or set of pins may need to be reset to provide an appropriate adjustment. Optionally, the degree of wear on the pin or shaft portion (or wear surface) may be monitored or determined, and the pin may be selectably rotated to limit wear on the wear surface of the pin. It is further envisioned that each station or lobe on the head portion of the

pin may be marked or numbered to indicate which portion or portions of the wear surface have already been selected or used.

For example, and with reference to FIGS. 21 and 22, a head portion 516c of a pin 516 and openings 513b of side links 512 may be non-circular, such that pin 516 is non-rotatably positioned at and through side links 512 of a chain or section of chain 510. For example, the head portion 516c may be lobed or non-circular shaped with three or more sides or portions 516e (such as the three curved and generally equal-sized sides shown in FIGS. 21 and 22), with the recessed area 513c correspondingly shaped to receive the head portion, such that the mating surfaces of the head portion 516c within the correspondingly formed recessed area or impression 513c in the side link 512 substantially precludes rotation of the pin 516 relative to the side links 512. For example, one or more of the side portions of the head portion may engage a corresponding one or more of the sidewalls of the side link at the aperture to substantially preclude rotation of the pin relative to the side links. The wear surface of the shaft portion 516a and/or ball member 520 thus may be generally fixed relative to the side links 512 by substantially fixing head 516c within recess 513c, such that only a portion of the shaft and/or ball member will contact and wear against the concave surface 514b of the center link 514 as the chain travels along its conveying path.

The pin may be generally non-rotatable relative to the chain links when in each of the positions (such as in each of the three positions in the illustrated embodiment of FIGS. 21 and 22, or such as in each of the six positions in the illustrated embodiment of FIGS. 23-25, discussed below, or such as in any other number of positions suitable for such an arrangement), and may be selectably adjustable between the positions to adjust the wear surface of the shaft portion of the pin relative to the chain links, thereby providing selective engagement of a portion or portions of the wear surface with the chain links. Such an arrangement allows for controlling the wear and life cycle of the pin by wearing a particular wear surface of the pin at a time, and allows for controlled or manual or selective rotation from one wear surface of the pin to the next wear surface of the pin. The pin and side link arrangement of the present invention thus provides a dialable pin, which may be manually and selectably dialed or rotated to provide a new wear surface against the side link after the first wear surface has been sufficiently worn. Such a configuration facilitates control of the amount of time each wear surface or wear surface portion of the pin is in use.

Optionally, and with reference to FIGS. 23-25, a head portion 516c' of a pin 516' and openings 513b' of side links 512' may be non-circular, such that pin 516' is non-rotatably positioned at and through side links 512' of a chain or section of chain 510'. As shown in

FIGS. 23-25, the head portion 516c' may be generally hex-shaped, with the recessed area 513c' being correspondingly shaped to receive the head portion, such that the mating of the head portion 516c' within the correspondingly formed recessed area or impression 513c' in the side link 512' substantially precludes rotation of the pin 516' relative to the side links 512' in six positions. The wear surface of the shaft portion 516a' and/or ball member 520' may be generally fixed relative to the side links 512' by substantially fixing head 516c' within recess 513c', such that only a portion of the shaft and/or ball member will contact and wear against the concave surface 514b of the center link 514 as the chain travels along its conveying path. The wear surface may then be adjusted or dialed to a new surface by rotating the pin 516' to a next orientation relative to the recesses 513c' in the side links 512', as discussed above.

Optionally, and with reference to FIGS. 26 and 27, a chain or section of chain 510" may include a pin 516", which may comprise shaped or lobed or hex-shaped head portions 516c" at opposite ends of a generally cylindrical shaft portion 516b". The center link 514" includes a generally flat or curved, non-concave surface 514b" for engaging the shaft portion 516b". Pin 516" and side links 512' may be otherwise substantially similar to pin 516' discussed above, such that a detailed discussion will not be repeated herein.

Optionally, and with reference to FIGS. 28-30, a chain or section of chain 610 may include a threaded pin or bolt 616, which may comprise a shaft portion 616a and a shaped or lobed or hex-shaped head portion 616c at one end and a threaded portion 616b at an opposite end for receiving a nut or threaded fastener or the like 618 (FIG. 30). Head portion 616c may be received in a correspondingly formed or shaped recess 613c in one side link 612a, while a generally cylindrical shaft portion 616a and/or threaded end portion 616b may extend through an opening 613 in the other side link 612b of chain 610. The center link 614 includes a concave inner surface 614b for engaging a ball member 620 positioned along shaft portion 616a of pin 616. Ball member 620 may be formed as part of shaft portion 616a or may be slid onto or formed or molded onto shaft portion 616a, and pin 616 may include a generally cylindrical sleeve portion over and along shaft portion 616a, such as any of the types of sleeves discussed above or below. Chain 610 may be otherwise substantially similar to chain 510, discussed above, such that a detailed discussion will not be repeated herein.

Optionally, and with reference to FIGS. 31 and 32, a chain or section of chain 610' may include a threaded pin or bolt 616', which may comprise a shaft portion 616a' and a non-circular shaped or lobed or hex-shaped head portion 616c' at one end and a threaded portion 616b' at an opposite end for receiving a nut or threaded fastener or the like 618. Head portion 616c' may be received in a correspondingly formed or shaped recess 613c in one side link

612a, while a generally cylindrical shaft portion 616a' and/or threaded end portion 616b' may extend through an opening 613 in the other side link 612b of chain 610'. The center link 614' may include a generally flat or non-concave inner surface 614b' for engaging shaft portion 616a' of pin 616'. Optionally, pin 616' may include a generally cylindrical sleeve portion
5 over and along shaft portion 616a', such as the types of sleeve portions discussed above or below. Chain 610' may be otherwise substantially similar to chain 610, discussed above, such that a detailed discussion will not be repeated herein.

The dialable pin configuration of the present invention thus may provide significantly greater life cycles for such chains and pins, since the pins do not have to be replaced when
10 one or more of the pins wears a sufficient amount. Such a pin-head and side link design may be implemented with a pin with a ball member or the like or a pin with a generally cylindrical shaft portion and no ball member, without affecting the scope of the present invention. Also, such a dialable pin concept is suitable for use on a double headed pin or a single headed or bolted pin, with the head or heads of the pin being lobed or non-circular shaped or formed
15 and engaged with a correspondingly formed recess in one of the side links, while a threaded end of a bolted pin may extend through a circular opening in the other side link and may be secured therein by a nut or other fastener, without affecting the scope of the present invention.

The ball members and/or the concave surfaces of the center links of the present
20 invention may comprise a metallic material, or may comprise a nylon or plastic or polymeric material, without affecting the scope of the present invention. The selected material is preferably a highly durable material which may minimize wear of the ball and/or the concave surface when the chain is in use and moving through various curves while under load.

Therefore, the present invention provides a bolted or pinned or half-bolted chain
25 which has improved flexibility to ease negotiation of the chain links through sharp vertical changes in the chain path. The ball members of the present invention allow for pivotable movement between the center link and side links as the chain negotiates through the conveying path. More particularly, the ball member allows the center link to pivot about a longitudinal axis of the bolt, stud or pin in a conventional manner, while also allowing the
30 center link to pivot about the ball member in other directions as well, such as pivoting upward or downward relative to the side links. The ball member and bolt or stud combination of the present invention thus allows the bolted chain to negotiate inclines along the conveying path without binding or excessive wear occurring at the chain joints. Also, the ball and socket type connection of the present invention allows the chain to flex about both

axes, which further may allow the chain to twist or corkscrew over a sufficient length of track. Because the ball member may be loosely fit onto the bolt or stud, the ball member of the present invention provides for an easy assembly process of the bolt or stud and also facilitates easy disassembly or disconnection of the chain links for service or maintenance of the bolted chain. The present invention thus provides for a chain with much greater flexibility which is easy to manufacture and assemble.

Referring now to FIGS. 33 and 34, a chain or section of chain 710 includes a pair of side links 712a, 712b and a center link 714. The center link 714 is retained between the side links 712a, 712b by a pin 716, such as an I-pin or other type of pin, bolt, stud or the like, which extends through a center region or opening 714a of center link 714 and through an opening 713a in each side link 712a, 712b. Pin 716 is retained in and through side links 712a, 712b by opposite head portions 716a of pin 716 engaging a recessed region 713b of a respective side link 712a, 712b. Pin 716 includes a generally spherical or toroidal-shaped ball member 720 which is molded around a shaft portion 716b of pin 716. Chain 710 includes multiple linkages connected together in a continuous loop about a conveying system, as is known in the art. For ease of description, only one set of linkages of the chain is shown and described herein, with the other linkages being substantially identical.

Shaft portion 716b of pin 716 may comprise a metallic material, and may provide a substantially smooth exterior surface. The shaft portion 716b may be coated or treated, or may have a material deposited thereon, in a manner to provide a low coefficient of friction surface of the shaft portion. For example, the shaft portion may be treated with a Teflon type material or a hard, carbon or diamond like material, or any other durable and low coefficient of friction material or slick material, without affecting the scope of the present invention. The low coefficient of friction or slick surface facilitates the breaking free of the molded sleeve or ball 720 (as discussed below) and further facilitates rotation or movement of the sleeve or ball 720 relative to the shaft of the pin as the chain negotiates through its conveying path. Although shown as an I-pin type fastener or connector, the side links 712a, 712b and center 714 may be retained together via any other type of pin or bolt or stud for connecting the links of a chain, without affecting the scope of the present invention.

Each head portion 716a of pin 716 may comprise a generally rectangular shaped head such that portions of the head 716a extend laterally outward from the end of the shaft portion 716b in opposite directions. Each side link 712a, 712b includes a pair of slotted openings 713a which may be sized to receive head 716a therethrough when the head is rotated approximately 90 degrees from the orientation shown in FIGS. 33 and 34, in order to

facilitate assembly and disassembly of the chain links. Each side link 712a, 712b includes a recessed region or pocket 713b at each end for head 716a of pin 716 to rest when the chain is assembled. The recessed region 713b is defined by a raised end portion 713c and raised inward portions 713d which engage the sides of the head 716a to substantially preclude
5 rotation of the pin 716 relative to side links 712a, 712b and center link 714.

Center link 714 may comprise a generally oval shaped or elongated ring defining inner opening 714a. Center link 714 includes an inner rounded or concave surface 714b (FIGS. 33 and 34) at each end thereof for engaging and partially receiving ball member 720 therein to retain center link 714 at ball member 720 as the chain is moved along the
10 conveying path. Optionally, concave surface 714b may be coated or treated with a low coefficient of friction material, similar to shaft portion 716b of pin 716 discussed above, to reduce the friction between ball member 720 and center link 714.

Ball member 720 may comprise a plastic or polymeric member which is molded onto the substantially smooth metallic shaft portion 716b of pin 716. Ball member 720 is molded
15 around the shaft portion 716b after pin 716 is formed and is then allowed to cool and harden around the shaft portion. Because shaft portion 716b has a substantially smooth, and preferably slick, outer surface, ball member 720 may be twisted, slid, or otherwise moved relative to shaft portion 716b to break ball member 720 free from shaft member 716b. Ball member 720 may then be freely rotatable and movable around and along shaft portion 716b
20 to allow for relative movement between ball member 720 and pin 716. Ball member 716 thus provides a unitarily formed ball which is movable relative to pin 716 and center link 714. Ball member 716 may include one or more spacers or sleeve portions extending from one or both ends of the ball member, such as discussed above, to assist in retaining the ball member in place along the shaft portion.

25 Ball member 720 may be formed by injection molding a polymeric material around the shaft portion 716b of pin 716 or may be formed via any other molding process or method, without affecting the scope of the present invention. Ball member 720 may be formed of a durable, hard polymeric material, such as a nylon material or filled nylon material, a PBT material, or an engineering plastic or the like. Clearly, however, other materials may be used
30 for ball member 720, without affecting the scope of the present invention.

Optionally, ball member 720 may provide an exterior surface which has a durable and low coefficient of friction to also enhance the relative movement between ball member 720 and center link 714. Optionally, ball member 720 may be coated with a slick or low coefficient of friction material, similar to shaft portion 716b discussed above, to further

reduce the friction between the components, without affecting the scope of the present invention.

Ball member 720 thus allows for pivotal movement of center link 714 via sliding engagement between ball member 720 and shaft portion 716b of pin 716 and/or sliding
5 engagement between concave surface 714b and ball member 720. This provides greater flexibility to chain 710 and may allow chain 710 to negotiate sharper vertical curves in the conveying path without binding the links or joints of the chain. The ball member and correspondingly formed concave surface thus may distribute the loads over a generally constant surface area, reducing or substantially eliminating the stress concentration that
10 typically occur when conventional chains articulate through vertical inclines and declines. The ball member and concave surface engagement may also function to distribute the loads between the chain links and pin over a greater surface area than conventional chains. This decreases the wear on the pins and chain links and may result in less maintenance and a greater life cycle for the chain. Also, because chain 710 may provide a low coefficient of
15 friction surface at shaft 716b of pin 716 and/or at center link 714, ball member 720 may provide for a reduced amount of friction between the center link 714 and pin 716, which may further reduce wear on the components and may result in even less maintenance and an even greater life cycle of the chain.

Referring now to FIG. 35, a chain or section of chain 710' includes side links 712a,
20 712b and a center link 714, which are retained together by a pin 716'. Side links 712a, 712b and center link 714 may be substantially similar to the side links and center link discussed above with respect to chain 710, such that a detailed discussion of these components will not be repeated herein.

Pin 716' includes head portions 716a', similar to pin 716, and a shaft portion 716b'
25 extending between the opposite head portions 716a'. Shaft portion 716b' comprises a generally cylindrical shaft portion and includes a notched or narrowed section 716c' at a central region thereof. Notched section 716c' of pin 716' may extend circumferentially around shaft portion 716b', so as to define a narrowed diameter central region of shaft portion 716b'. However, notched section 716c' may optionally provide separate notched portions or
30 indents or grooves in a portion of the central region of the shaft portion, without affecting the scope of the present invention.

Similar to pin 716, discussed above, shaft portion 716b' and notched section 716c' of pin 716' may provide substantially smooth, cylindrical exterior surfaces. Shaft portion 716b' may be coated or treated in a manner to provide a low coefficient of friction surface of the

shaft portion, such as described above with respect to pin 716 of chain 710. The low coefficient of friction or slick surface facilitates the breaking free of a molded sleeve or ball 720' (discussed below) and further facilitates rotation or movement of the sleeve or ball 720' relative to the shaft of the pin as the chain negotiates through its conveying path. Although
5 shown as an I-pin type fastener or connector, the links 712a, 712b and 714 may be retained together via any other type of pin or bolt or stud, such as formed using the principles described above, for connecting the links of a chain, without affecting the scope of the present invention.

Pin 716' includes a generally spherical or toroidal-shaped ball member 720' around
10 shaft portion 716b' of pin 716'. Ball member 720' may comprise a plastic or polymeric member which is molded onto the substantially smooth shaft portion 716b' of pin 716', such that an inner portion of ball member 720' is molded within notched section 716c' of pin 716'. Ball member 720' is molded around the shaft portion 716b' after pin 716' is formed and is then allowed to cool and harden around the shaft portion. Because shaft portion 716b' and
15 notched section 716c' may have a substantially smooth and cylindrical outer surface, and which may be treated with a slick material or coating, ball member 720' may be twisted or otherwise moved relative to shaft portion 716b' to break ball member 720' free from shaft member 716b'. Ball member 720' may then be freely rotatable around and notched section 716c' of shaft portion 716b' to allow for relative movement between ball member 720' and pin
20 716'. Notched section 716c' functions to retain ball member 720' in place and limits or substantially precludes longitudinal movement of ball member 720' along shaft portion 716b' of pin 716'.

Similar to ball member 720, ball member 720' may be formed by injection molding a polymeric material around the shaft portion 716b' of pin 716' or may be formed via any other
25 molding process or method, without affecting the scope of the present invention. Optionally, ball member 720' may provide an exterior surface which is durable and which may have a low coefficient of friction to enhance the relative movement between ball member 720' and center link 714. Optionally, ball member 720' may be coated with a slick or low coefficient of friction material, such as discussed above with respect to ball member 720, to further
30 reduce the friction between the components. As discussed above, concave surface 714b of center link 714 may also or otherwise be coated or treated to provide a surface having a lower coefficient of friction for engagement with ball member 720'.

Referring now to FIG. 36, a chain or section of chain 810 includes side links 712a, 712b and a center link 714, which are retained together by a pin 816. Side links 712a, 712b

and center link 714 may be substantially similar to the side links and center link discussed above with respect to chain 710, such that a detailed discussion of these components will not be repeated herein.

Pin 816 includes opposite head portions 816a, similar to pin 716, and a shaft portion 816b extending between the opposite head portions 816a. Shaft portion 816 comprises a generally cylindrical shaft portion and includes a spherical or toroidal-shaped ball member 820 integrally formed at a generally central region thereof. Similar to pin 716, discussed above, shaft portion 816b and ball member 820 of pin 816 may provide a substantially smooth exterior surface. Shaft portion 816b and ball member 820 may be coated or treated in a manner to provide a low coefficient of friction surface, such as described above with respect to pin 716 of chain 710. The low coefficient of friction or slick surface facilitates the breaking free of a molded sleeve 817 (discussed below) from the shaft portion and ball member of pin 816 and further facilitates rotation or movement of the sleeve relative to the shaft and ball of the pin as the chain negotiates through its conveying path. Although shown as an I-pin type fastener or connector, the links 712a, 712b and 714 may be retained together via any other type of pin or bolt or stud for connecting the links of a chain, such as formed using the principles discussed above, without affecting the scope of the present invention.

Pin 816 includes a polymeric or plastic sleeve portion 817 molded around shaft portion 816b and ball member 820. The slick surface of the shaft portion 816b and ball member 820 allows sleeve portion 817 to be broken free from shaft portion 816b after sleeve portion 817 is molded thereon, and allows for rotation of sleeve portion 817 about shaft portion 816b and ball member 820. Optionally, sleeve portion 817 may provide a highly durable and slick or low friction surface for engagement of sleeve 817 with the concave surface 714b of center link 714. As can be seen in FIG. 36, sleeve portion 817 may extend along shaft portion 816b to head portions 816a, such that sleeve portion 817 also provides a highly durable and optionally slick or low friction surface for engagement between shaft portion 816b and the end walls 713e of openings 713a of side links 712a, 712b.

Similar to ball members 720 and 720', sleeve portion 817 may be molded over shaft portion 816b after pin 816 has been formed. Because shaft portion 816b and ball member 820 of pin 816 provide a substantially smooth surface and may have a treated slick surface at which sleeve portion 817 is molded, the sleeve portion may be twisted or otherwise moved relative to pin 816 to break the molded sleeve portion away from shaft portion 816b and ball member 820, such that sleeve portion 817 may be generally freely rotatable about shaft portion 816b and ball member 820. When chain 810 is assembled, sleeve portion 817 may

thus be movable relative to side links 712a, 712b, center link 714 and pin 816 as the chain flexes or bends through its conveying path.

Similar to ball member 720, sleeve portion 817 may be formed by injection molding a polymeric material around the shaft portion 816b and ball member 820 of pin 816 or may be
5 formed via any other molding process or method, without affecting the scope of the present invention. Optionally, sleeve portion 817 may provide an exterior surface (or may be treated with a material or coating) which has a low coefficient of friction to enhance the relative movement between sleeve portion 817 and center link 714, such as discussed above with respect to ball member 720. As discussed above, concave surface 714b of center link 714
10 may also or otherwise be coated or treated to provide a surface having a lower coefficient of friction for engagement with sleeve portion 817.

Referring now to FIG. 37, a chain or section of chain 810' includes side links 712a, 712b and a center link 714', which are retained together by a pin 816'. Side links 712a, 712b and center link 714' may be substantially similar to the side links and center link discussed
15 above with respect to chain 710, such that a detailed discussion of these components will not be repeated herein. However, center link 714' of chain 810' may have a generally straight or slightly curved pin engaging surface 714b' at its ends, and does not include a concave pin engaging surface.

Pin 816' includes opposite head portions 816a', similar to pin 716, and a shaft portion
20 816b' extending between the opposite head portions 816a'. Shaft portion 816' comprises a generally cylindrical shaft and preferably provides a substantially smooth exterior surface. Shaft portion 816b' may be coated or treated in a manner to provide a low coefficient of friction surface of the shaft portion, such as described above with respect to pin 716 of chain 710. For example, the shaft portion may be treated with a Teflon type material or a hard,
25 diamond like material or any other low coefficient of friction material or slick material, without affecting the scope of the present invention, to substantially reduce the friction between the components. The low coefficient of friction or slick surface facilitates the breaking free of a molded sleeve 817' and further facilitates rotation or movement of the sleeve relative to the shaft of the pin as the chain negotiates through its conveying path.
30 Although shown as an I-pin type fastener or connector, the links 712a, 712b and 714' may be retained together via any other type of pin or bolt or stud for connecting the links of a chain, such as formed using the principles discussed above, without affecting the scope of the present invention.

Pin 816' includes a polymeric or plastic sleeve portion 817' molded around shaft portion 816b'. Optionally, sleeve portion 817' may provide a highly durable and slick or low friction surface for engagement of sleeve portion 817' with the surface or end 714b' of center link 714'. As can be seen in FIG. 37, sleeve portion 817' may extend along shaft portion 816b' to head portions 816a', such that sleeve portion 817' may also provide a highly durable (and optionally a slick or low friction surface) for engagement of shaft portion 816b' with the end walls 713e of openings 713a of side links 712a, 712b. As discussed above, the end of center link 714' may also or otherwise be coated or treated to provide a surface having a lower coefficient of friction for engagement between sleeve portion 817' and center link 714'.

Similar to ball members 720 and 720' and sleeve portion 817, sleeve portion 817' is molded over shaft portion 816b' after pin 816' has been formed. Because shaft portion 816b' of pin 816' provides a substantially smooth and slick surface at which sleeve portion 817' is molded, the sleeve portion may be twisted or otherwise moved relative to pin 816' to break the sleeve portion away from shaft portion 816b', such that sleeve portion 817' may be generally freely rotatable about shaft portion 816b'. When chain 810' is assembled, sleeve portion 817' is movable relative to side links 712a, 712b, center link 714' and pin 816' as the chain flexes or bends through its conveying path. Similar to ball member 720, sleeve portion 817' may be formed by injection molding a polymeric material around the shaft portion 816b' of pin 816' or may be formed via any other molding process or method, without affecting the scope of the present invention.

As shown in FIG. 38, a chain or section of chain 910 may include a pin 916 and side links (not shown) and a center link 914. Center link 914 includes a plastic or polymeric insert or member 915 at each end for engagement with the shaft portion 916b of pin 916. The plastic insert 915 may be coated or treated with or may otherwise provide a durable and slick or low friction material surface for engagement between plastic insert 915 of center link 914 and pin 916. Insert 915 may be molded at an end 914b of center link 914, such as via an injection molding process or the like. However, insert 915 may be molded to center link 914 via other molding means, or may be snapped or removably mounted or affixed to center link 914, without affecting the scope of the present invention.

Pin 916 includes opposite head portions 916a and shaft portion 916b extending between the opposite head portions 916a. Shaft portion 916 may comprise a generally cylindrical shaft and may provide a substantially smooth exterior surface. Similar to pin 716, shaft portion 916 may be coated or treated in a manner to provide a slick or low coefficient of

friction surface. However, other pins, bolts, studs or the like may be implemented with chain 910, without affecting the scope of the present invention.

Each end 914b of center link 914 provides a recessed area 914c, such that insert 915 may be molded or retained generally within recessed area 914c and may be substantially
5 fixed or non-movable within recessed area 914c. In the illustrated embodiment, recessed area 914c provides a pair of raised end stops 914d at opposite ends of a generally octagonal, curved surface 914e. The octagonal surface 914e and end stops 914d function to substantially preclude movement of insert 915 relative to center link 914 after insert 915 has been molded thereto. Although shown as an octagonal shaped surface, clearly, other shapes
10 may be provided at recessed area 914c to limit or substantially preclude movement of insert 915 relative to center link 914.

Insert 915 may provide a durable and low coefficient of friction surface 915a for engagement with shaft portion 916b of pin 916. The low coefficient of friction surface may provide enhanced relative movement between pin 916 and center link 914, which may reduce
15 wear on the center link and pin as the chain negotiates through the conveying path. Insert 915 may be made from a durable polymeric or plastic material, such as a nylon material, such as a filled nylon material, a PBT material, or an engineering plastic or the like. Optionally, the insert 915 and/or the shaft portion 916b of pin 916 may be treated or coated with a material which provides a low coefficient of friction to enhance relative movement between
20 the pin and the center link and to reduce the wear on the components to increase the life cycle of the chain.

Although shown as being implemented with a cylindrical shaft pin, it is envisioned that an insert in accordance with the present invention may include a concave surface (not shown in FIG. 38), such as a surface similar to concave surface 714b of center link 714,
25 discussed above, for engagement with a ball member or other sleeve or the like (also not shown in FIG. 38) on the shaft portion of the pin, such as a ball member or sleeves of the types discussed above or the like. The insert may be removably mounted or affixed to the center link to facilitate removal and replacement of the insert to provide a center link with a replaceable wear surface.

30 Referring now to FIG. 39, a chain or section of chain 910' includes a pin 916', side links 912' and a center link 914'. Center link 914' includes a plastic or polymeric insert or member 915' at each end for engagement with the shaft portion 916b' of pin 916'. The plastic insert 915' may be coated or treated with or may otherwise provide a durable and slick or low friction material surface for engagement between center link 914' and pin 916'. Insert 915'

may be molded at an end 914b' of center link 914', such as via an injection molding process or the like. However, insert 915' may be molded to center link 914' via other molding means, without affecting the scope of the present invention. Optionally, insert 915' may be a replaceable insert which may be snapped or otherwise affixed to the end 914b' of the center link 914'. Insert 915' may include a concave engagement surface or socket 915a' for generally uniformly engaging an outer surface of a ball portion or member 920' of pin 916', discussed below.

Pin 916' includes opposite head portions 916a' and shaft portion 916b' extending between the opposite head portions 916a'. Shaft portion 916' comprises a generally cylindrical shaft and includes generally spherical ball member or portion 920' at a generally central portion of shaft portion 916b'. Ball member 920' may be formed as part of the shaft of pin 916' or may be slid or molded or formed onto pin 916', such as utilizing the principles discussed above or the like. The outer surface of shaft portion 916b' and ball portion 920' preferably provides a substantially smooth exterior surface for engagement with insert 915' at center link 914'. Similar to pin 716, shaft portion 916b' and ball portion 920' of pin 916' may be coated or treated in a manner to provide a slick or low coefficient of friction surface. However, other pins, bolts, studs or the like may be implemented with chain 910', without affecting the scope of the present invention.

Similar to center link 914, discussed above, each end 914b' of center link 914' may provide a recessed area (shown generally at 914c'), such that insert 915' may be molded generally within the recessed area and may be substantially fixed or non-movable within the recessed area. Recessed area 914c' may be generally similar to recessed area 914c discussed above, such that a detailed discussion of the recessed area will not be repeated herein. Recessed area 914c' of center link 914' may include a concave mating surface or socket, which generally corresponds to the curvature of socket or surface 915a' of insert 915'. The insert 915' may then be molded to have a generally uniform thickness at the socket area. However, the center link 914' may have a generally flat or otherwise formed mating surface for the insert 915', without affecting the scope of the present invention.

Insert 915' may provide a durable and low coefficient of friction surface 915a' for engagement with ball member or portion 920' of pin 916'. The low coefficient of friction may provide enhanced relative movement between pin 916' and center link 914', which may reduce wear on the center link and pin as the chain negotiates through the conveying path. The corresponding shapes or curvature of the surface 915a' and ball member 920' provides for enhanced flexibility of chain 910', with reduced wear on the ends of center link 914'

and/or on the shaft of pin 916'. Insert 915' may be made from a durable polymeric or plastic material, such as a nylon material, such as a filled nylon material, a PBT material, or an engineering plastic or the like. The ball and socket arrangement minimizes any tension and shear stresses on the insert 915', such that insert 915' may be exposed primarily to
5 compression loads, which are less damaging to the plastic insert. Optionally, the insert 915' and/or the ball portion 920' of pin 916' may be treated or coated with a material which provides a low coefficient of friction to enhance relative movement between the pin and the center link and to reduce the wear on the components to increase the life cycle of the chain.

Optionally, as shown in FIG. 40, a chain or section of chain 910" may include a pin
10 916", side links 912" and a center link 914". Center link 914" includes a plastic or polymeric insert or member 915" at each end for engagement with the shaft portion 916b" of pin 916". The plastic insert 915" may be coated or treated with or may otherwise provide a durable and slick or low friction material surface for engagement between center link 914" and pin 916". Insert 915" may be molded at an end 914b" of center link 914", such as via an injection
15 molding process or the like. However, insert 915" may be molded to center link 914" via other molding means, without affecting the scope of the present invention. Insert 915" may include a convex engagement surface 915a" for generally uniformly engaging a narrowed, outer surface of shaft portion 916b" of pin 916", as discussed below.

Pin 916" includes opposite head portions 916a" and shaft portion 916b" extending
20 between the opposite head portions 916a". Shaft portion 916" comprises a generally cylindrical shaft and includes a narrowed portion 916c" at a generally central portion of shaft portion 916b". Narrowed portion 916c" provides an inwardly curved surface for generally uniform engagement with the outwardly curved surface 915a" of insert 915". The outer curved surface 916c" of shaft portion 916b" may provide a substantially smooth exterior
25 surface for engagement with convex surface 915a" of insert 915" at center link 914". Similar to pin 716, shaft portion 916" may be coated or treated in a manner to provide a slick or low coefficient of friction surface. However, other pins, bolts, studs or the like may be implemented with chain 910", without affecting the scope of the present invention.

Center link 914" may be substantially similar to center links 914, 914', discussed
30 above, such that a detailed discussion of the center link 914" will not be repeated herein. Suffice it to say that insert 915" may be molded generally within a recessed area at each end of center link 914" and may be substantially fixed or non-movable within the recessed area. Optionally, the insert may be snapped or otherwise affixed to or removably mounted to the center link and may provide a removable and replaceable wear surface for the center link.

Similar to inserts 915 and 915', insert 915" may provide a durable and low coefficient of friction surface 915a" for engagement with the narrowed curved portion 916c" of shaft 916b" of pin 916". The low coefficient of friction provides enhances relative movement between pin 916" and center link 914", which may reduce wear on the center link and pin as the chain negotiates through the conveying path. The corresponding shapes or curvature of the outwardly curved surface 915a" of insert 915" and the inwardly curved surface 916c" of pin 916" provides for enhanced flexibility of chain 910", with reduced wear on the ends of center link 914" and/or on the shaft of pin 916". Insert 915" may be made from a durable polymeric or plastic material, such as a nylon material, such as a filled nylon material, a PBT material, or an engineering plastic or the like. Optionally, the insert 915" and/or the curved surface 916c" of pin 916" may be treated or coated with a material which provides a low coefficient of friction surface or coating to enhance relative movement between the pin and the center link and to reduce the wear on the components to increase the life cycle of the chain.

Although each of the embodiments of chain sections discussed above with respect to FIGS. 33-40 includes a pin with oval or rectangular head portions, it is further envisioned that a round-headed pin may be provided (such as with any of the chains of the present invention discussed above) with a rounded head at each end of the pin and a ball member and/or sleeve or the like on the shaft portion between the heads. The head portions may then insert through enlarged, generally circular openings in the side links and may slide outwardly toward the ends of the side links to a narrowed opening. The narrowed opening substantially precludes the circular heads from moving therethrough, and thus retains the side links relative to the center links and the pin. The side links include a recessed area around the narrowed opening to receive the rounded head therein to substantially preclude movement of the pin relative to the side links toward the enlarged openings in the side links. The pin is thus allowed to rotate relative to the side links and the center link, while allowing greater flexibility and/or reduced wear of the chain.

Although shown as being joined by a pin in FIGS. 33-40, it is further envisioned that the linkages of the chains of the present invention may otherwise be joined by other pins, bolts, studs, half-bolt / half-pin type pins or fasteners, or any other fastening or joining means for retaining the side links and the center links together. The fastening or joining means may be formed using the principles discussed above.

It is envisioned that any of the pin and chain link embodiments discussed above may have wear surface components comprising any suitable material. For example, the wear

surface components may comprise steel or other metallic material, bronze or other softer metallic material, a polymeric material, such as engineering plastics or other strong and durable polymeric material, a ceramic material, a sintered material, a Teflon material, a fluorocarbon material or the like. It is also envisioned that any of the pin and chain link
5 embodiments discussed above may include a low coefficient of friction or slick coating at or on the wear surfaces of the pins and/or chain links. The coating thus may provide enhanced relative movement or rotation or sliding between the wear surfaces to enhance performance of the section of chain and to increase the life cycle of the section of chain. The coating may comprise a Teflon type coating or a diamond like coating or the like or any other low
10 coefficient of friction or slick coating or the like suitable for such an application.

The present invention thus may provide a diamond like coating (DLC) over a metallic or plastic (or other material) wear surface of the pin or chain link. The wear surface component may define a substantially smooth wear surface to enhance sliding or moving engagement with a corresponding wear surface of the section of chain. The metallic wear
15 surface component or components may be formed to be substantially smooth via any known means, such as via tumbling, vibrating or the like of the component, such as tumbling with ceramic chips or steel chips or the like, while the plastic wear surface component or components may be molded to the desired form and/or tumbled or vibrated to achieve the desired surface smoothness, without affecting the scope of the present invention.

The diamond like coating on the wear surface or surfaces provides enhanced
20 performance and an increased life cycle to the section of chain. Because such a diamond like coating may be sensitive to point loading and shock in some applications, the wear surface components of the present invention may comprise a ball member and a correspondingly formed socket to reduce or substantially eliminate such point loading conditions. Such ball
25 and socket arrangements reduce such shock or impact or point loading concerns by spreading out the contact area over a wider area.

Therefore, the present invention provides a sleeve and/or a ball member and a pin or bolt or stud which allows for pivotal movement of the center link via sliding engagement of the center link along the ball member and via sliding engagement of the sleeve or ball
30 member around the shaft of the pin. This provides greater flexibility to the chain and may allow the chain to negotiate sharper vertical curves in the conveying path without binding the links or joints of the chain. The ball member and correspondingly formed concave surface thus may distribute the loads over a generally constant surface area, reducing or substantially eliminating the stress concentration that typically occur when conventional chains articulate

through vertical inclines and declines. The ball member and concave surface engagement may also function to distribute the loads between the chain links and pin or bolt or stud over a greater surface area than conventional chains. This decreases the wear on the pin and chain links and may result in less maintenance and a greater life cycle for the chain.

5 The ball member, and/or the sleeve over the ball member or shaft of the pin, and/or the insert at the center link may comprise a high strength, durable polymeric or plastic material. The selected material may comprise a highly durable material which may minimize wear of the ball member and/or the concave surface when the chain is in use and moving through various curves while under load. The material may be selected or treated or coated
10 to also provide a low coefficient of friction surface to enhance movability of the components relative to one another and to reduce wear on the components. Optionally, one or more of the contacting surfaces may be treated or coated with a material which may provide a low coefficient of friction surface between the contacting components to further reduce wear on the chain. The ball member, sleeve, insert or other wear surface component may optionally
15 comprise a metallic material, such as steel or the like or bronze or other softer metallic or the like, and may be coated or treated to enhance the low friction engagement between respective wear surfaces. Because the ball member, pin and/or center link of the present invention may include a low friction surface, the present invention provides for a reduced amount of friction between the center link and the pin and/or between the side links and the pin, which further
20 reduces wear on the components and may result in less maintenance and a greater life cycle of the chain. It is envisioned that the low friction surfaces of the present invention may substantially reduce or obviate the need for lubrication of the joints of the chain during use. The present invention thus may provide a non-lubricated chain with plastic components.

 The present invention also provide for a unitary plastic component formed or molded
25 onto a pin to enhance the performance of the chain. The molded ball member or sleeve may be broken free from the pin to allow relative movement between the pin and the ball member or sleeve while also allowing relative movement between the ball or sleeve and the chain links. The molded ball or sleeve is thus a unitarily formed, durable member which provides an engagement component between the metal components of the chain. Optionally, a metal
30 or plastic ball or sleeve may be slid or pressed onto a stud or bolt type pin and may be removable therefrom.

 The present invention thus provides a chain which has improved flexibility to ease negotiation of the chain links through sharp vertical changes in the chain path. The ball member or the narrowed shaft portion of the present invention allows for pivotable

movement between the center link and side links as the chain negotiates through the conveying path. More particularly, the ball member or narrowed shaft portion allows the center link to pivot about a longitudinal axis of the pin in a conventional manner, while also allowing the center link to pivot relative to the pin, such as about the ball member or within
5 the curved shaft portion, in other directions as well, such as pivoting upward or downward relative to the side links. The ball member/curved engagement surface and pin combination of the present invention thus allows the chain to negotiate inclines along the conveying path without binding or excessive wear occurring at the chain joints. Also, the ball and socket type connection of the present invention allows the chain to flex about both axes, which
10 further may allow the chain to twist or corkscrew over a sufficient length of track. The present invention thus provides for a chain with much greater flexibility and greater life cycles which is easy to manufacture and assemble.

Changes and modifications in the specifically described embodiments may be carried out without departing from the principles of the present invention, which is intended to be
15 limited only by the scope of the appended claims, as interpreted in accordance with the principles of patent law.

What is claimed is:

1. A section of chain for conveying product along a conveying system, said section of chain comprising:
 - a first chain link;
 - at least one second chain link having apertures through opposite ends thereof; and
 - a pin for connecting said first and second chain links together to define a section of chain for conveying product along a conveying system, said pin comprising:
 - opposite end portions for engaging said second link of the section of chain;
 - a shaft portion extending between said opposite end portions; and
 - a sleeve portion around and at least partially along said shaft portion, said sleeve portion being arranged on said shaft portion to engage said first link of the section of chain when said opposite end portions are engaged with said second link of the section of chain, said sleeve portion being movable relative to said shaft portion, wherein said pin further comprises one of a bolted pin and a stud, at least one of said opposite end portions comprising a fastener end for receiving a fastener thereon to retain said pin at the chain links of the section of chain.
2. The section of chain of claim 1, wherein said sleeve portion is removably positioned over said shaft portion when said fastener is removed from said fastener end.
3. The section of chain of claim 1, wherein said sleeve portion comprises a toroidal-shaped member.
4. The section of chain of claim 3, wherein said toroidal-shaped member is generally centrally positioned along said shaft portion.
5. The section of chain of claim 4, wherein said sleeve portion comprises opposite spacer portions at opposite ends of said toroidal-shaped member.
6. The section of chain of claim 5, wherein said toroidal-shaped member and said opposite spacer portions comprise a unitary sleeve portion.

7. The section of chain of claim 1, wherein said sleeve portion comprises a replaceable wear surface component and is removable from said pin to facilitate replacement of said sleeve portion.
8. The section of chain of claim 1, wherein said sleeve portion comprises a low coefficient of friction coating on an outer surface of said sleeve portion which engages the other chain link.
9. The section of chain of claim 1, wherein at least one of an inner surface of said sleeve portion and an outer surface of said shaft member comprises a low coefficient of friction coating thereon.
10. The section of chain of claim 9, wherein said sleeve portion comprises a metallic material.
11. The section of chain of claim 1, wherein said opposite end portions and said shaft portion of said pin comprise a metallic material and said sleeve portion comprises a polymeric material.

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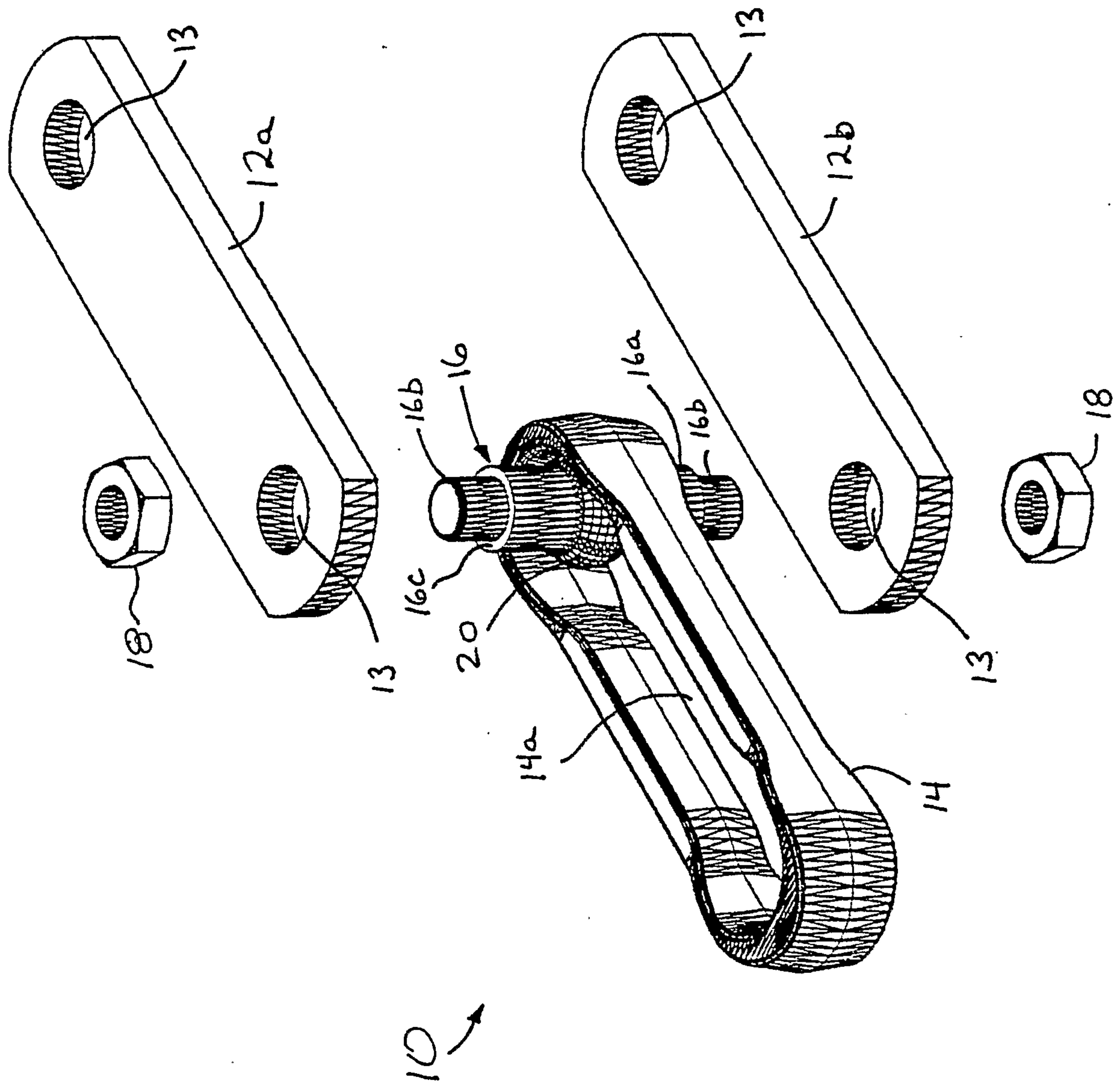
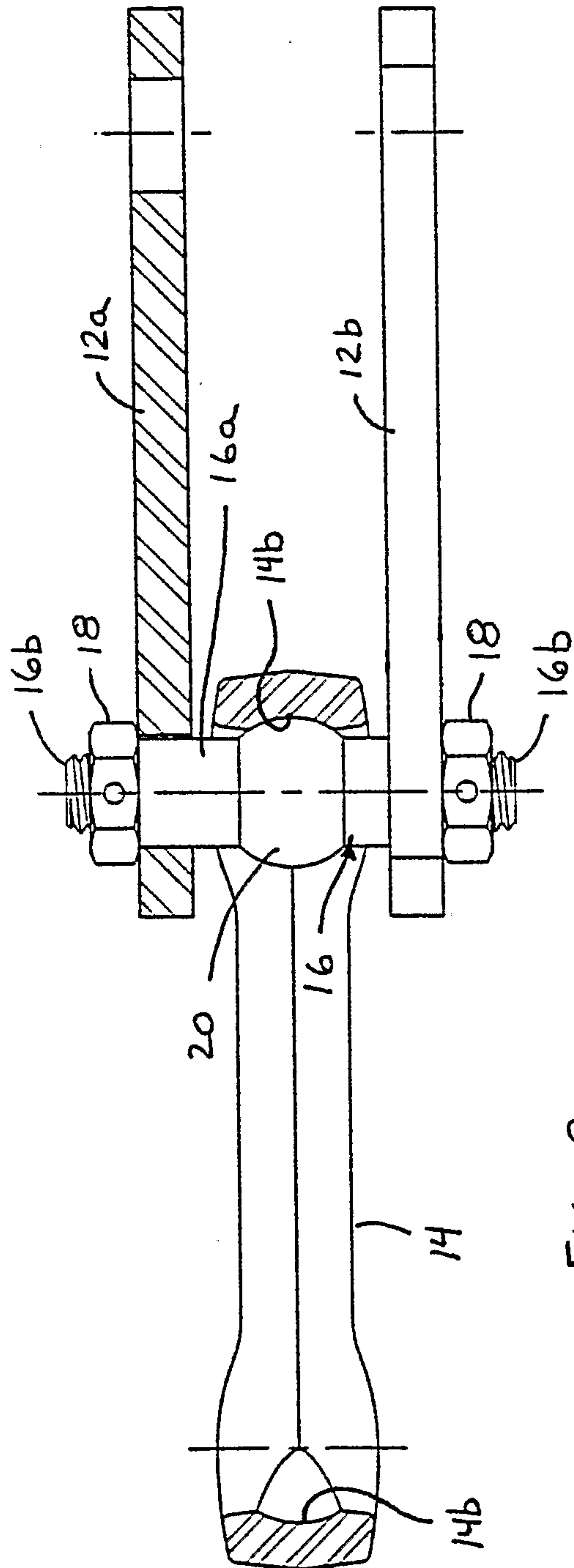
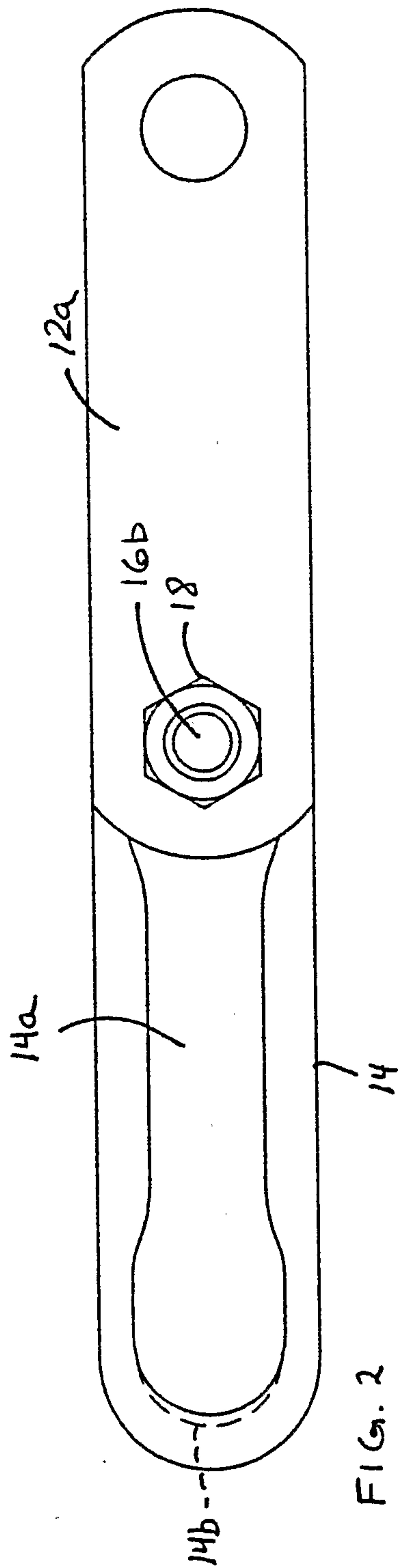


FIG. 1

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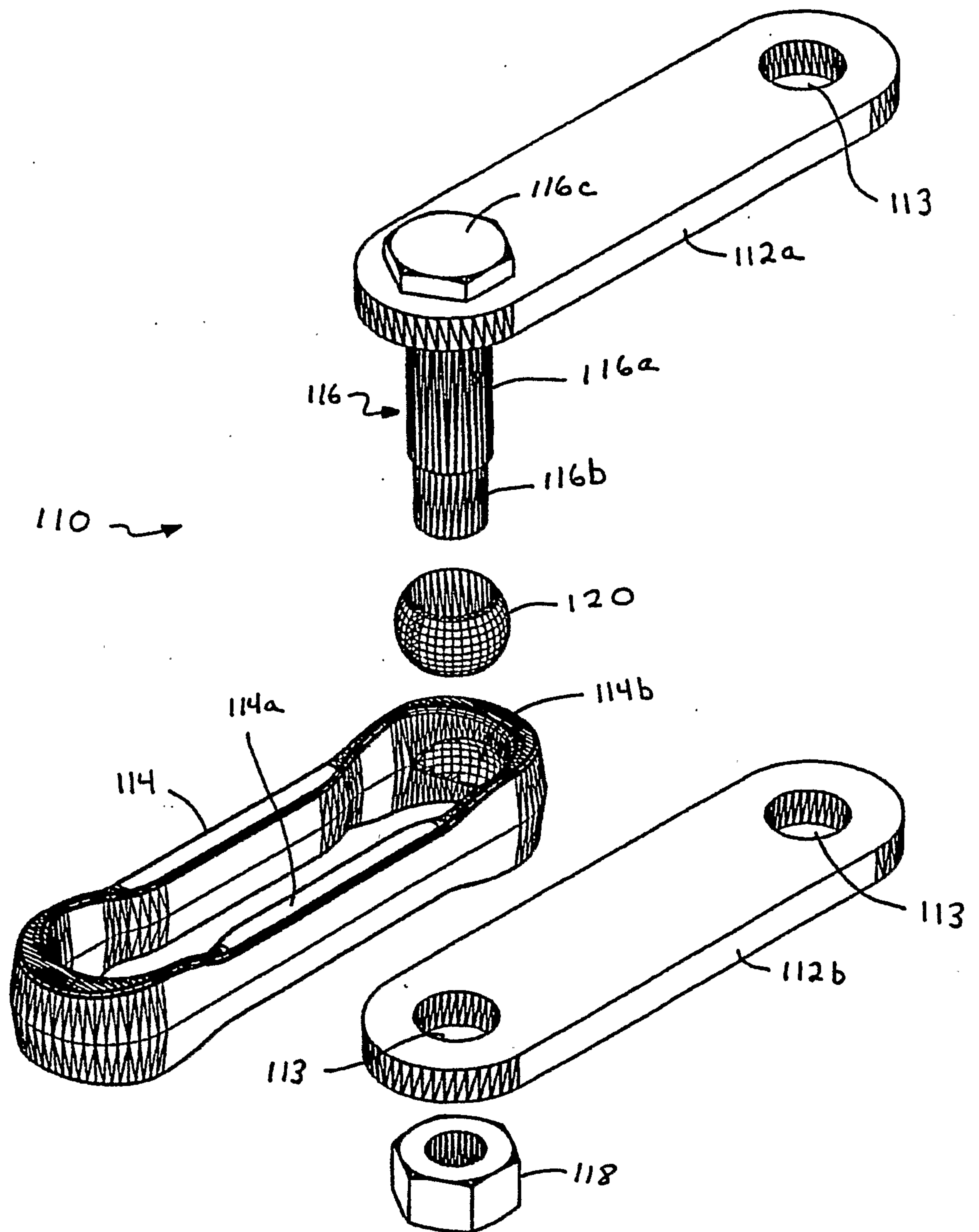
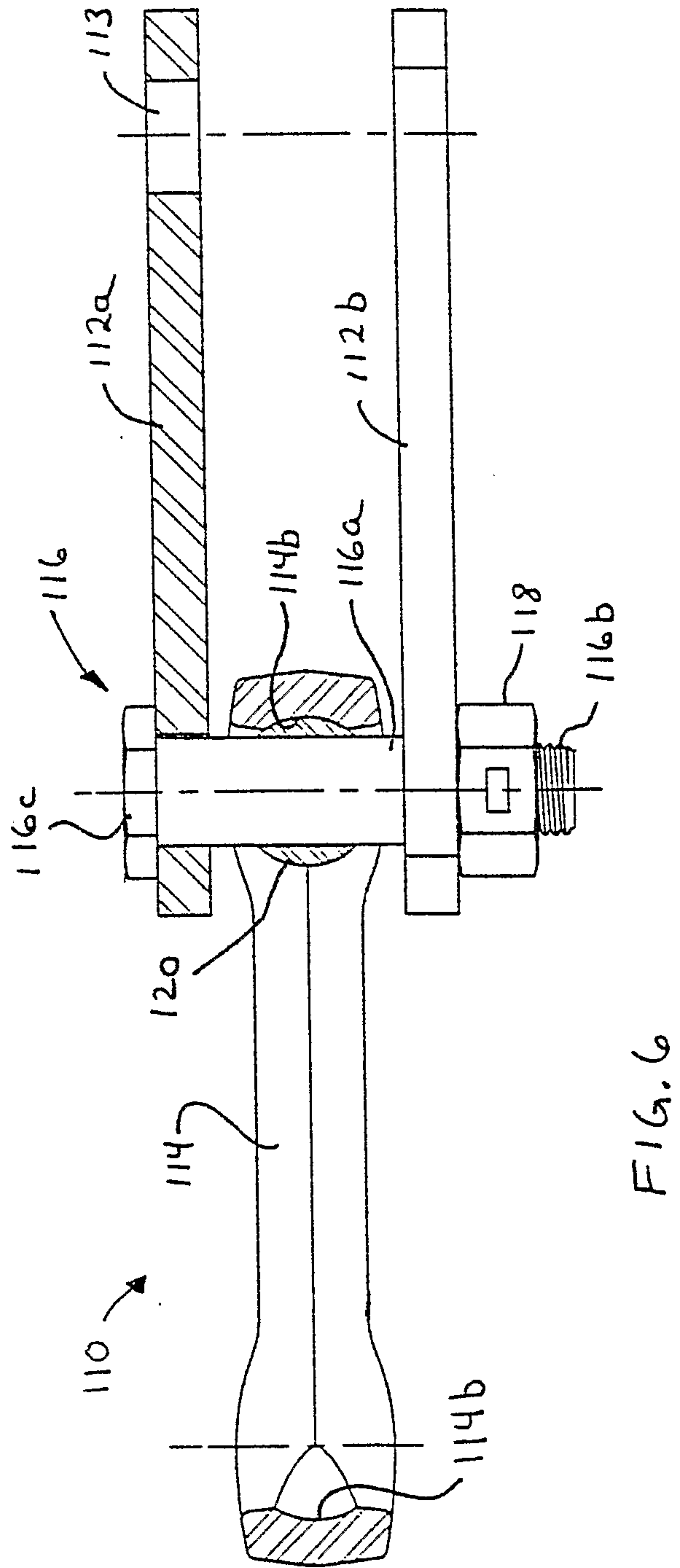
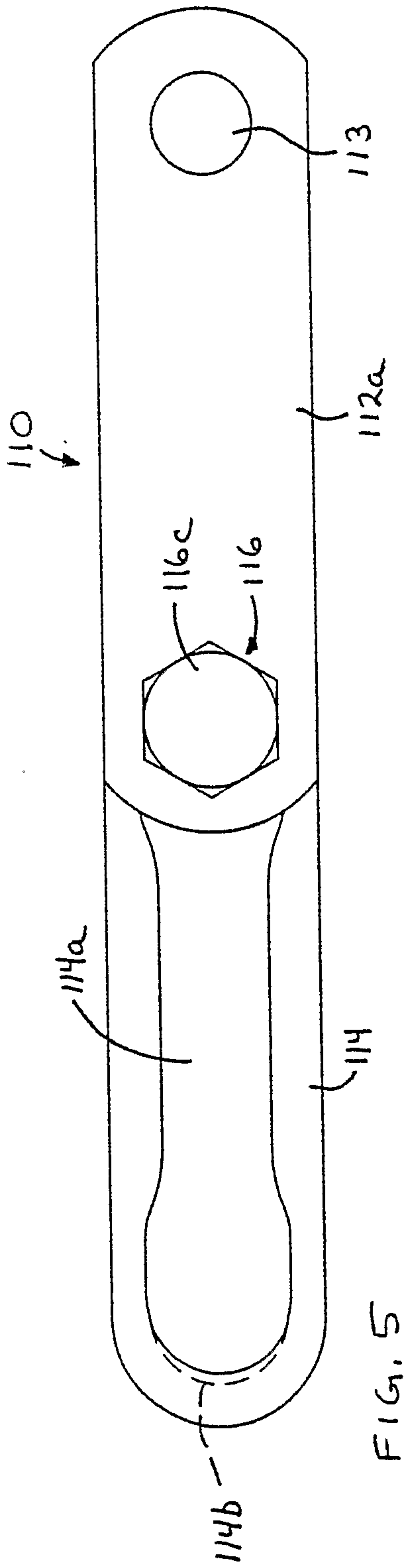
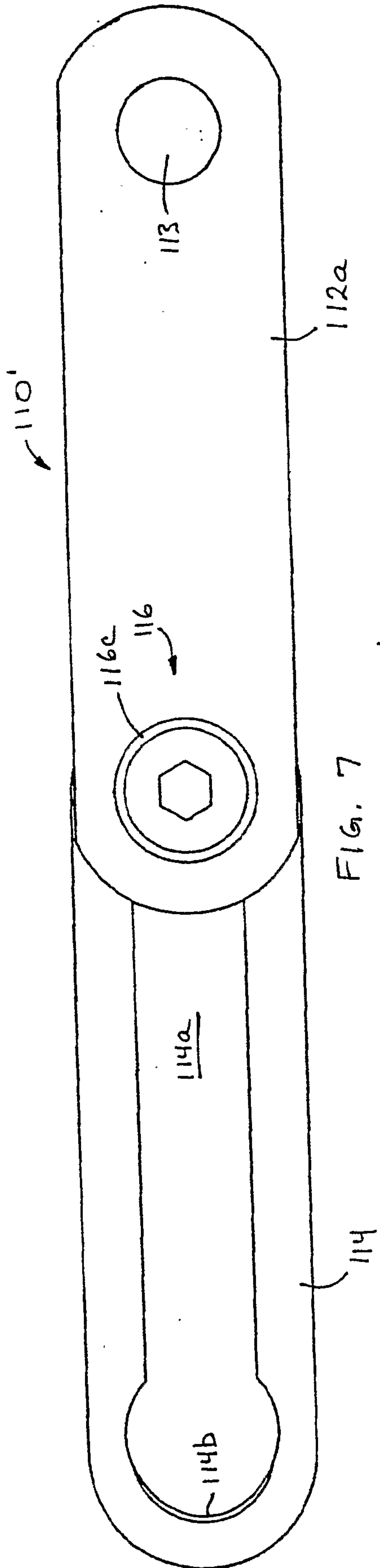


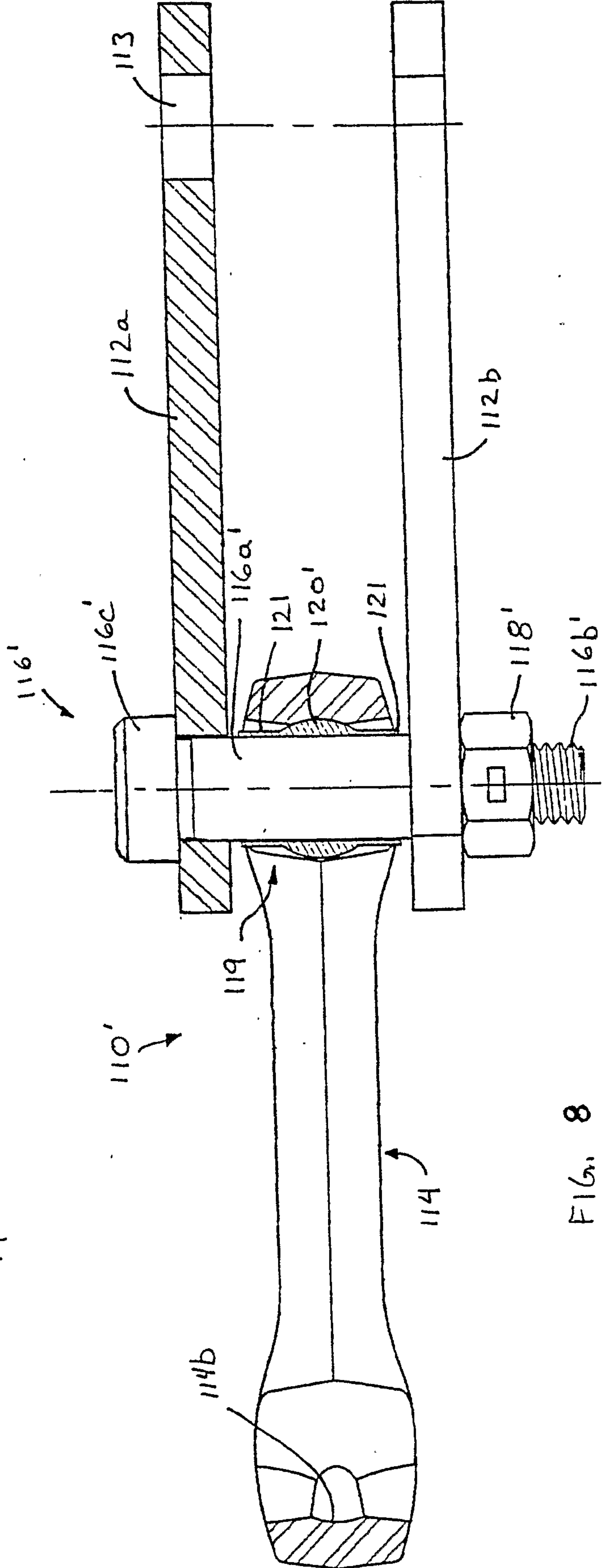
FIG. 4

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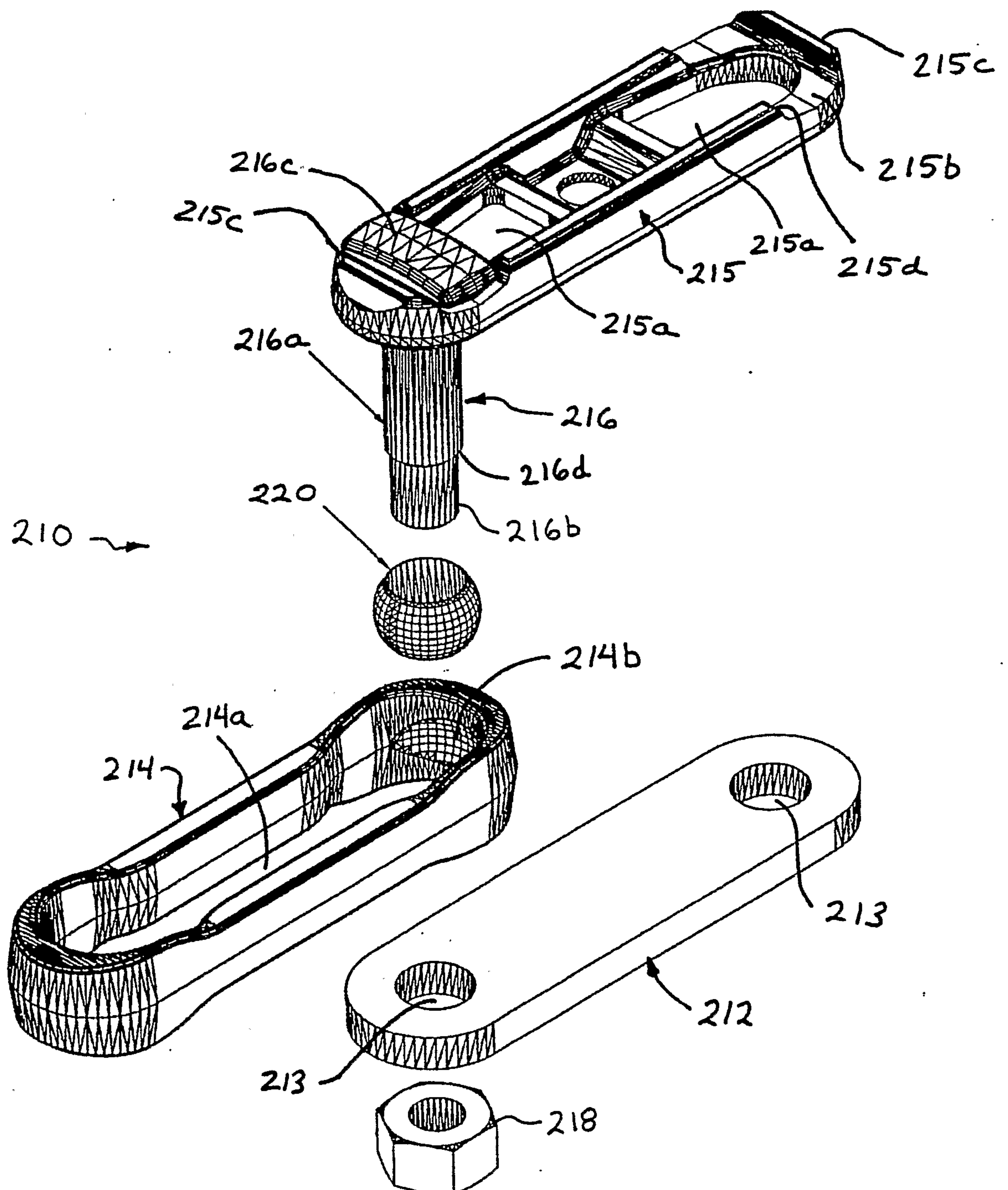


FIG. 9

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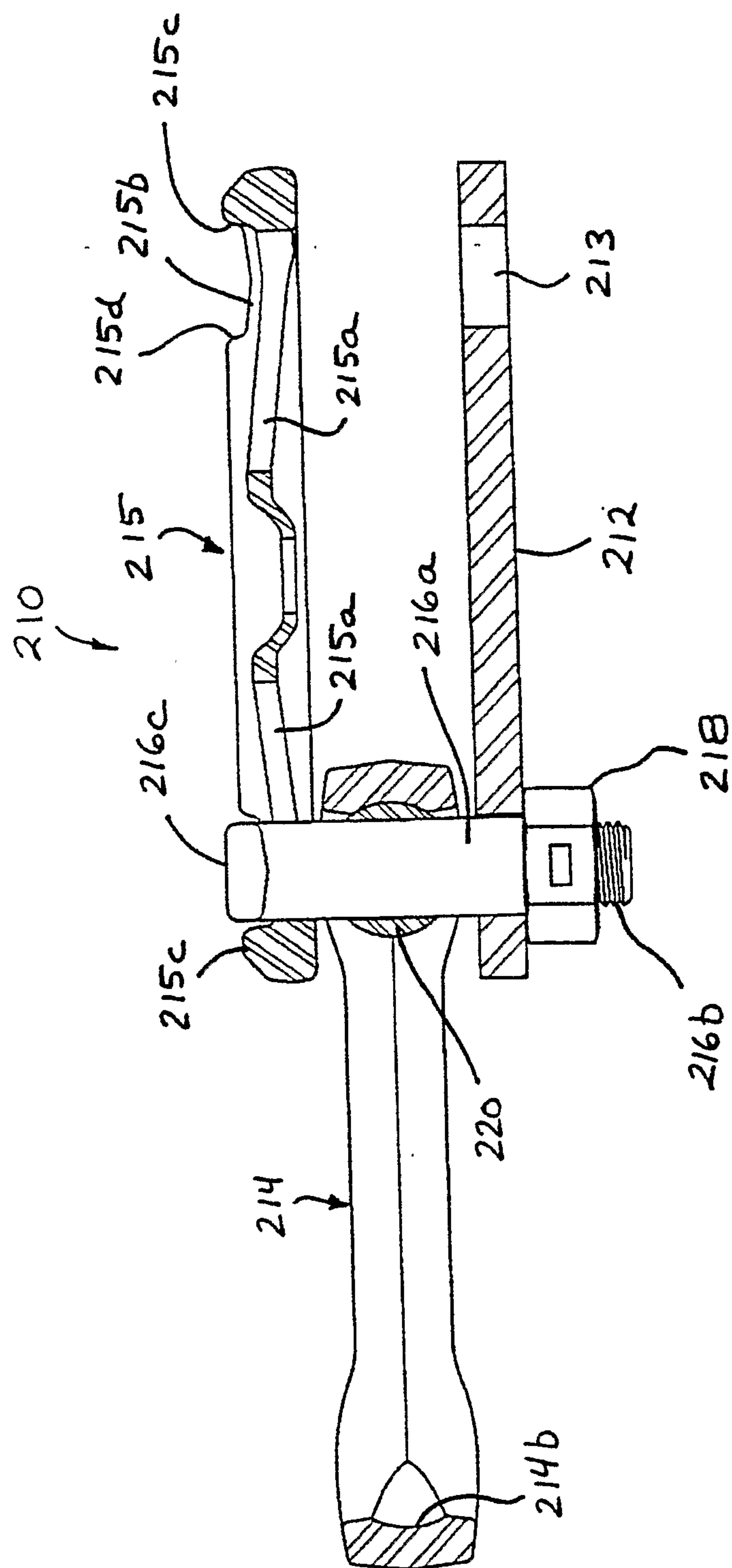
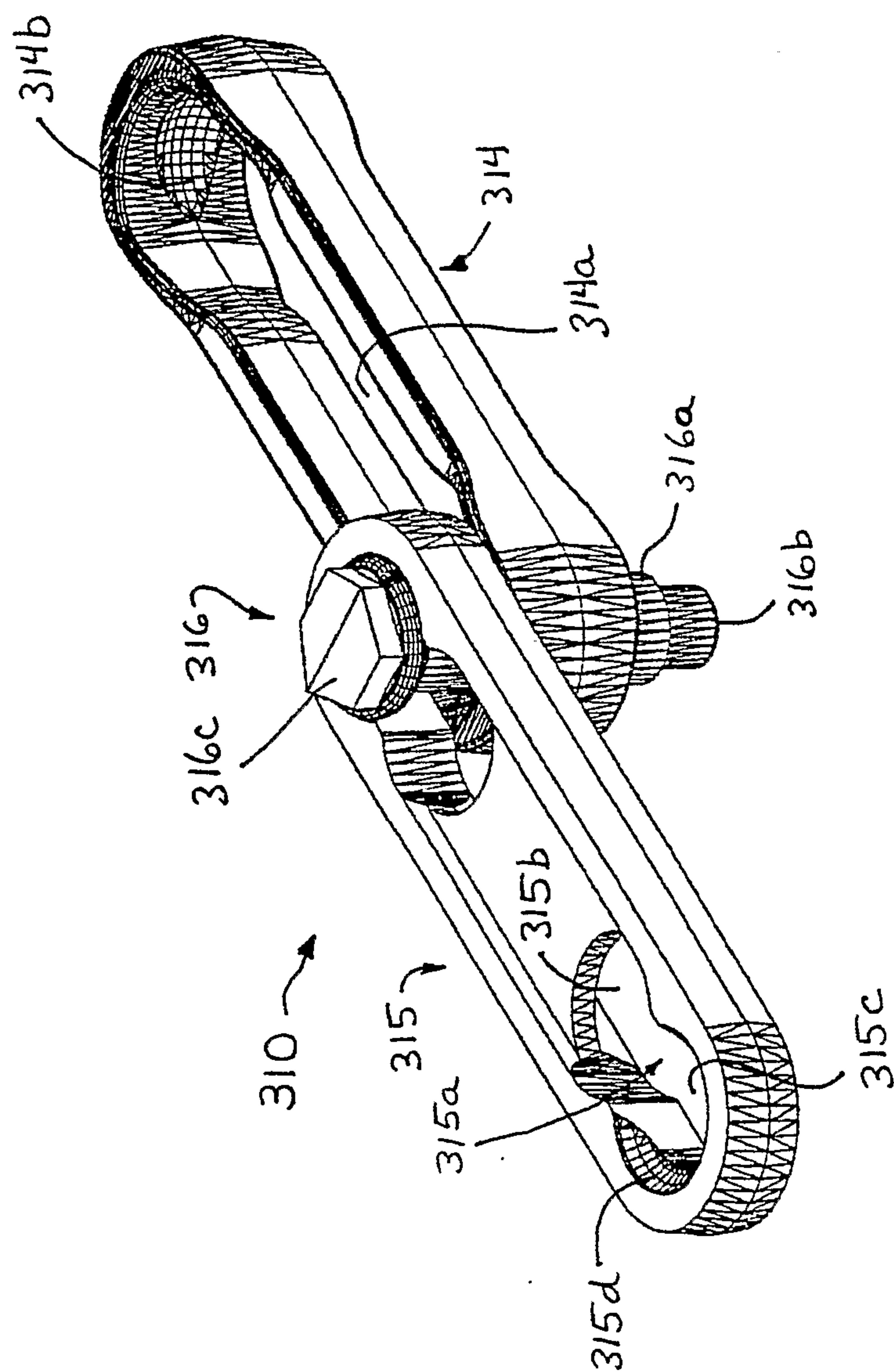


Fig. 10

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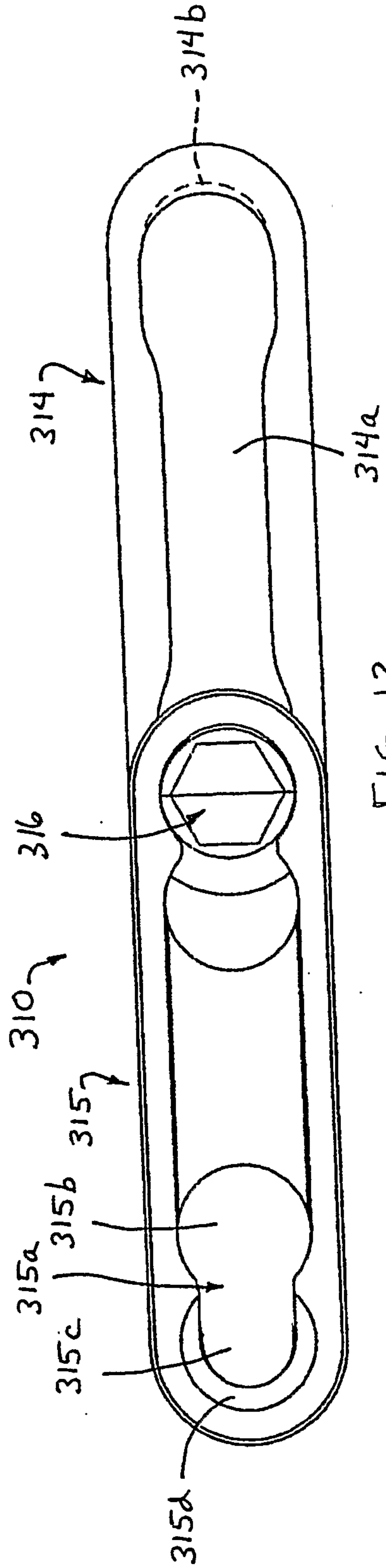


FIG. 12

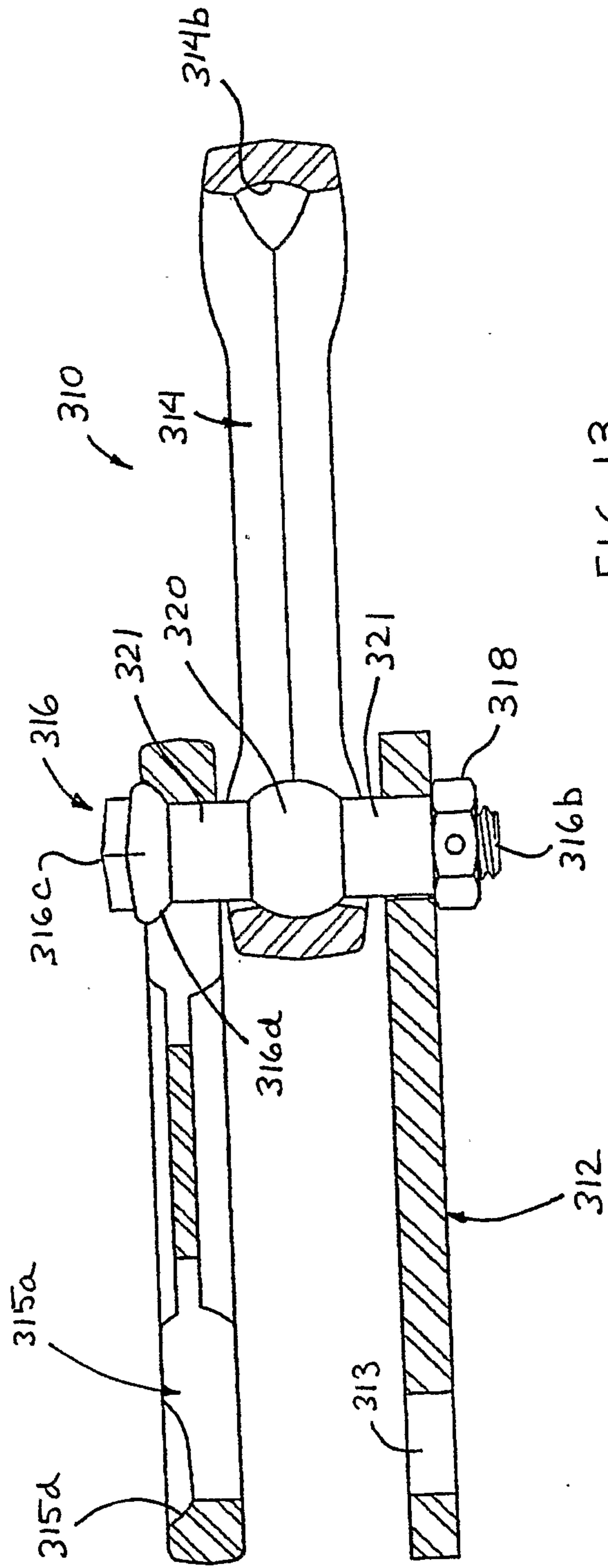
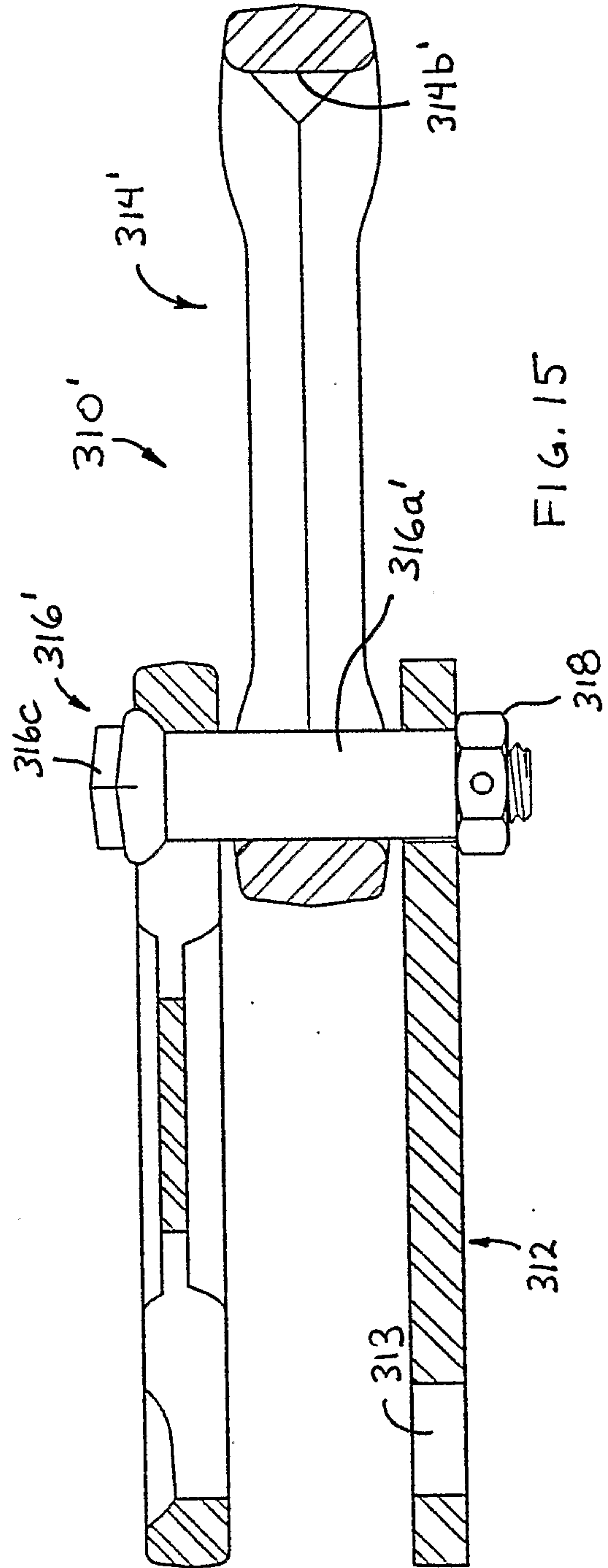
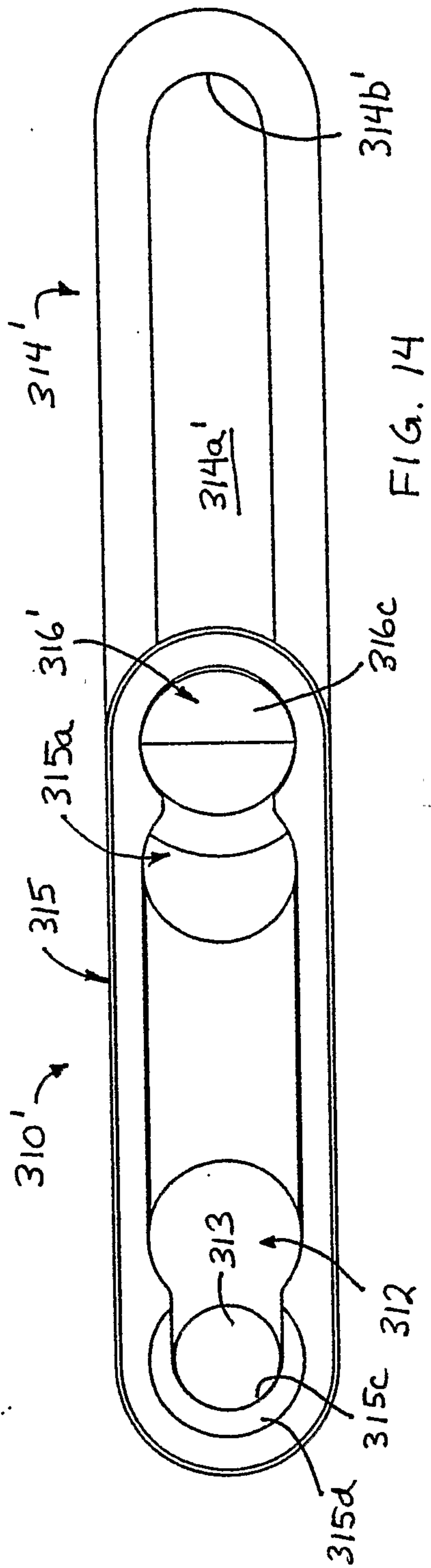
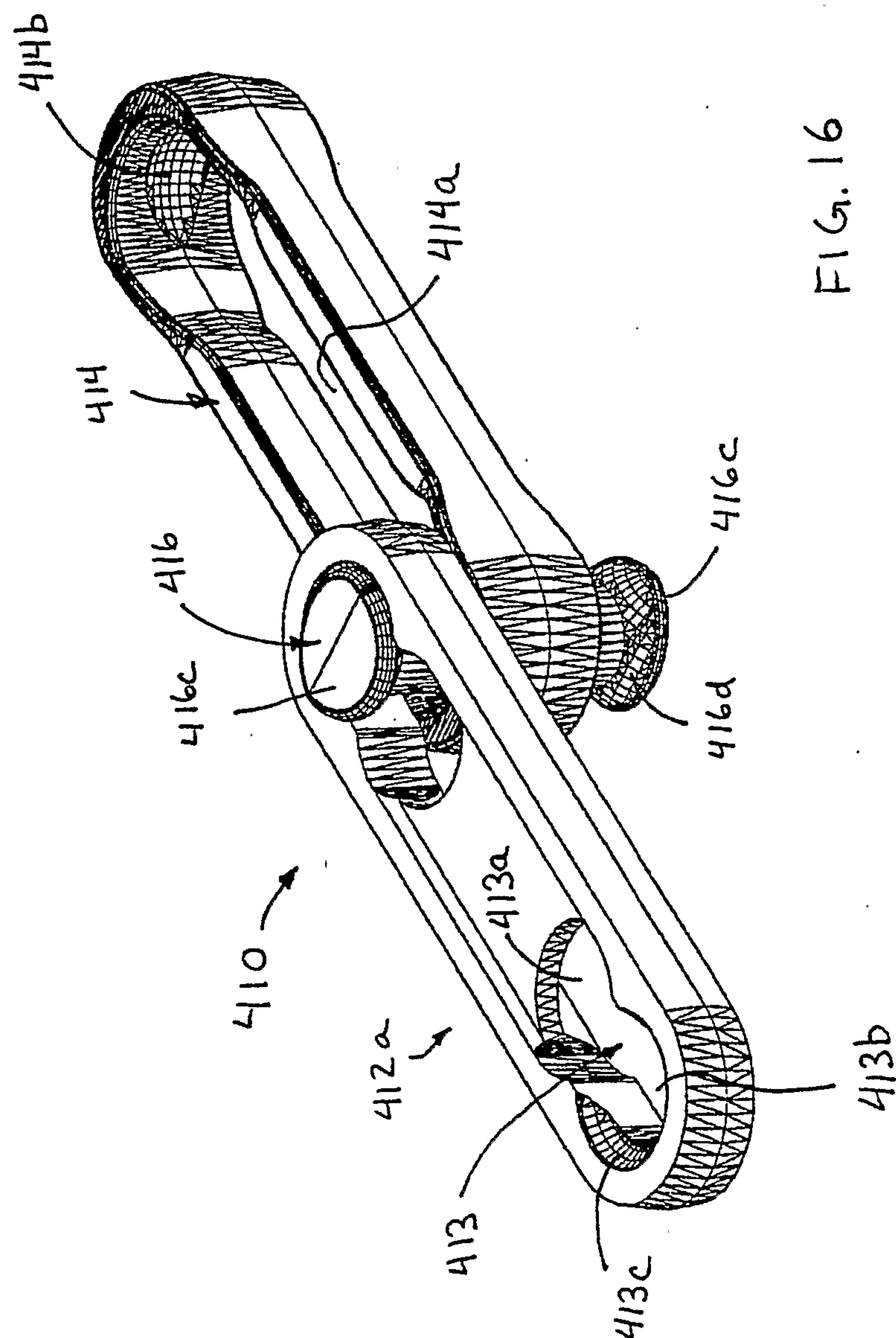


FIG. 13

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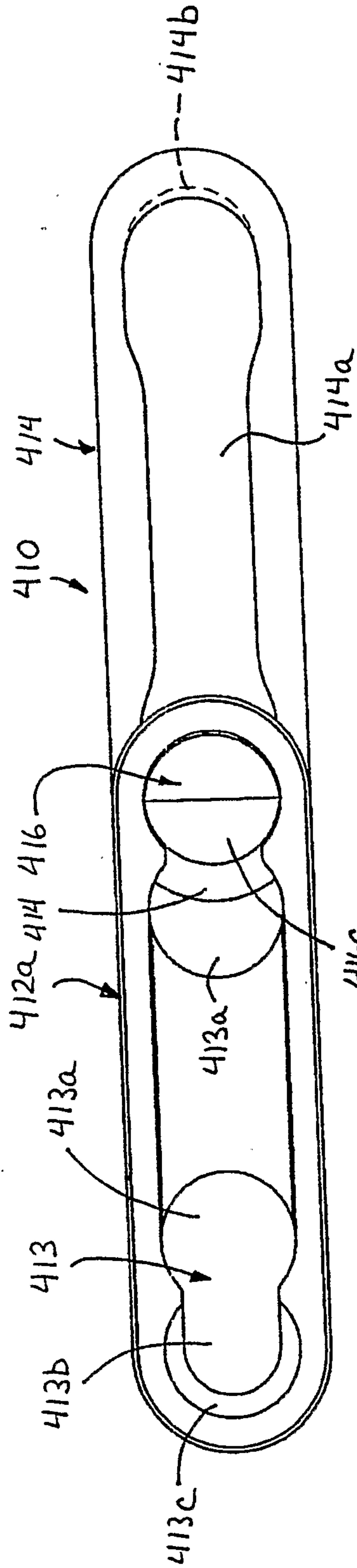
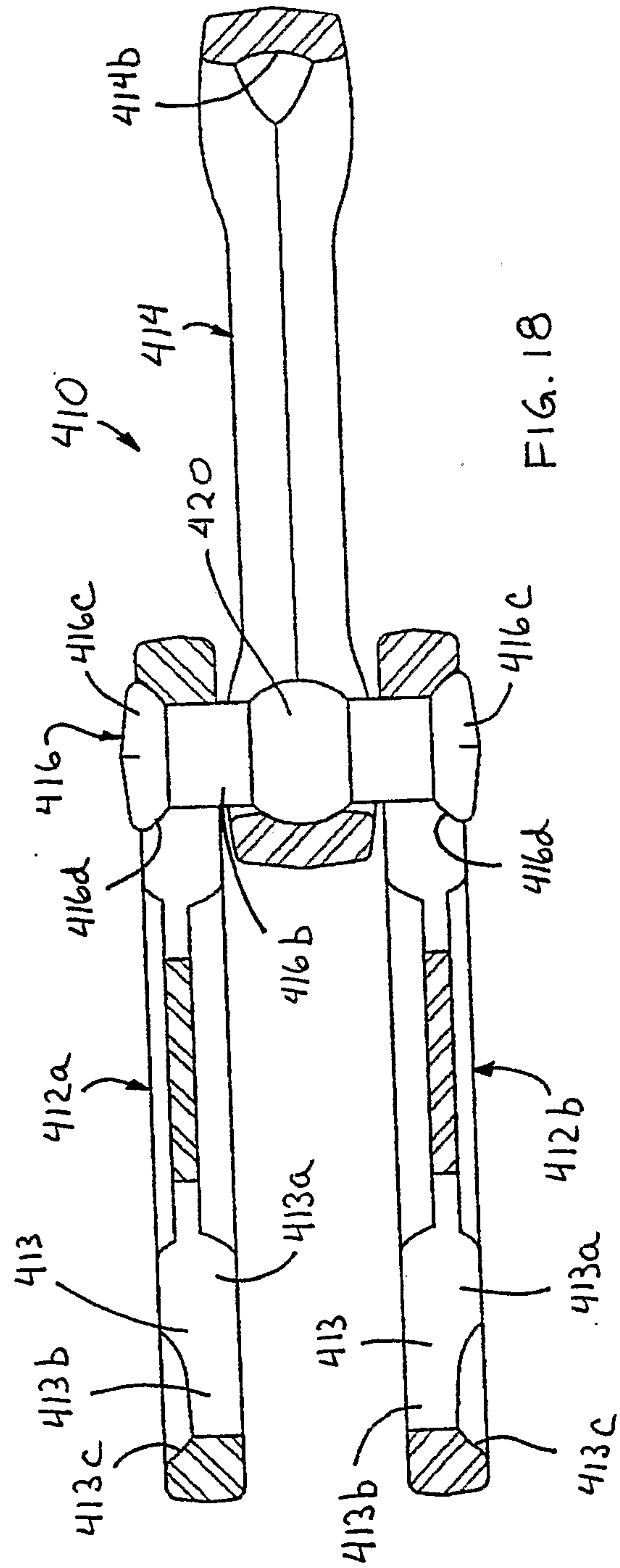
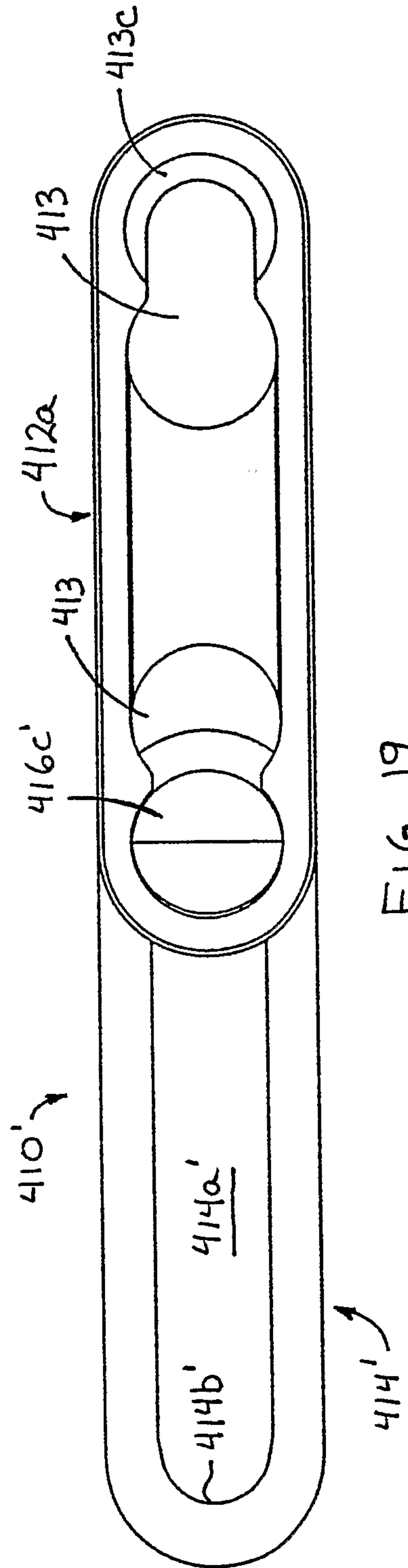
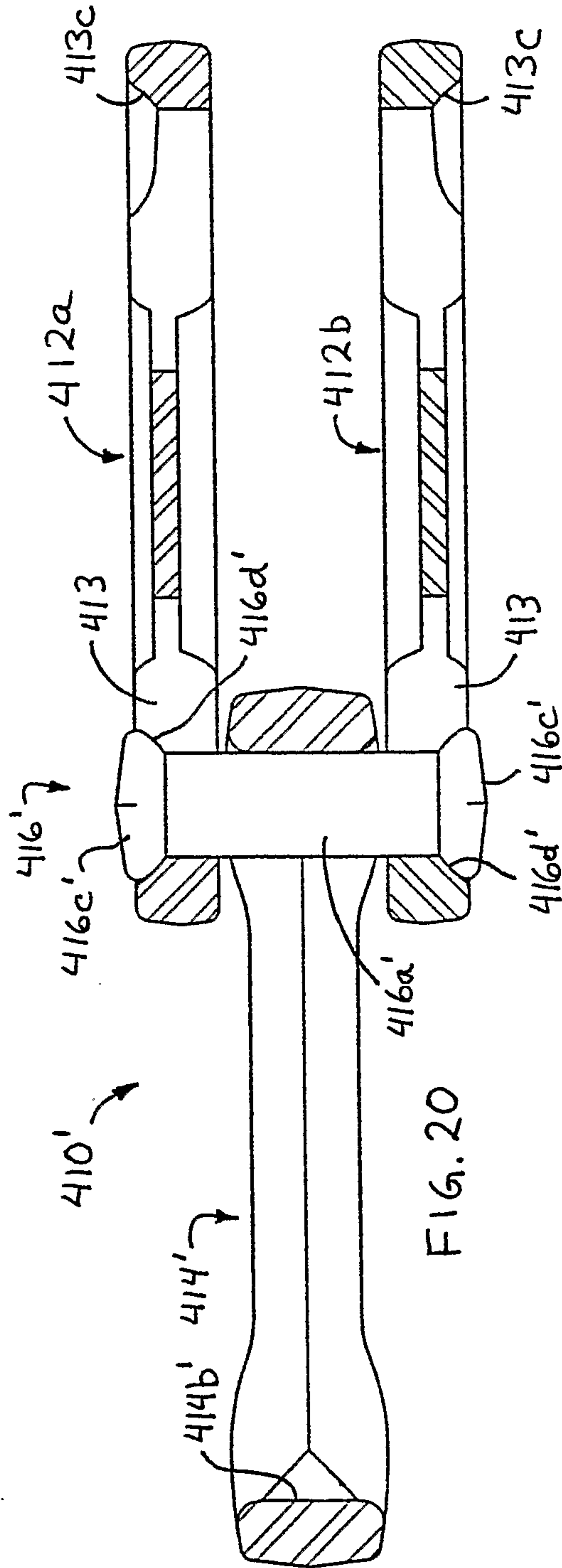


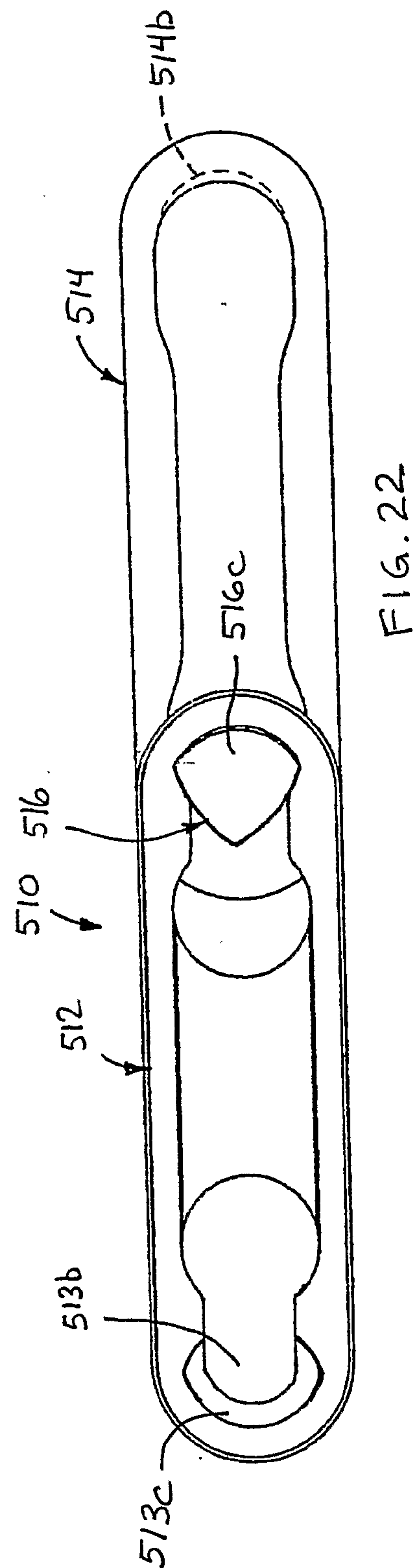
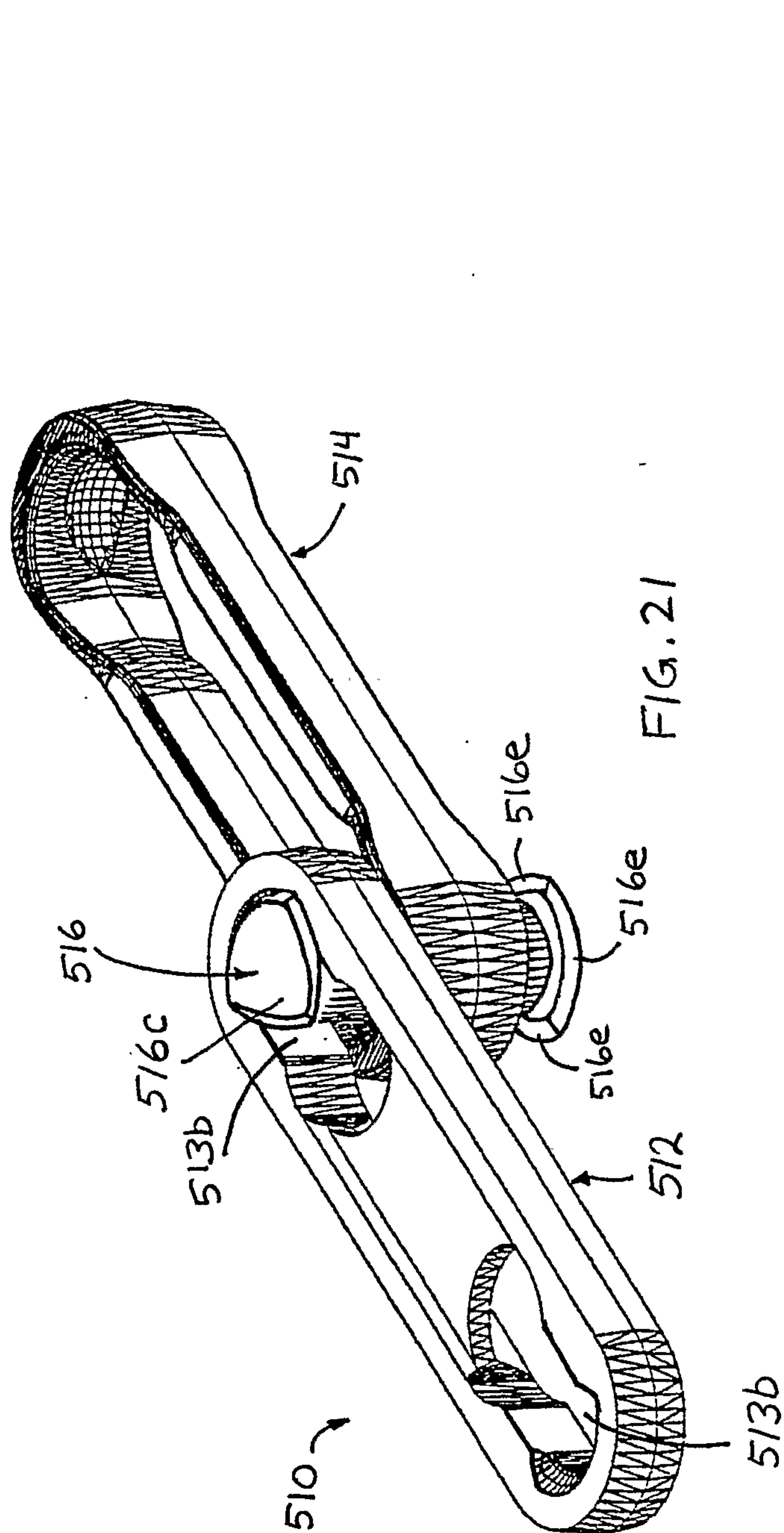
FIG. 17



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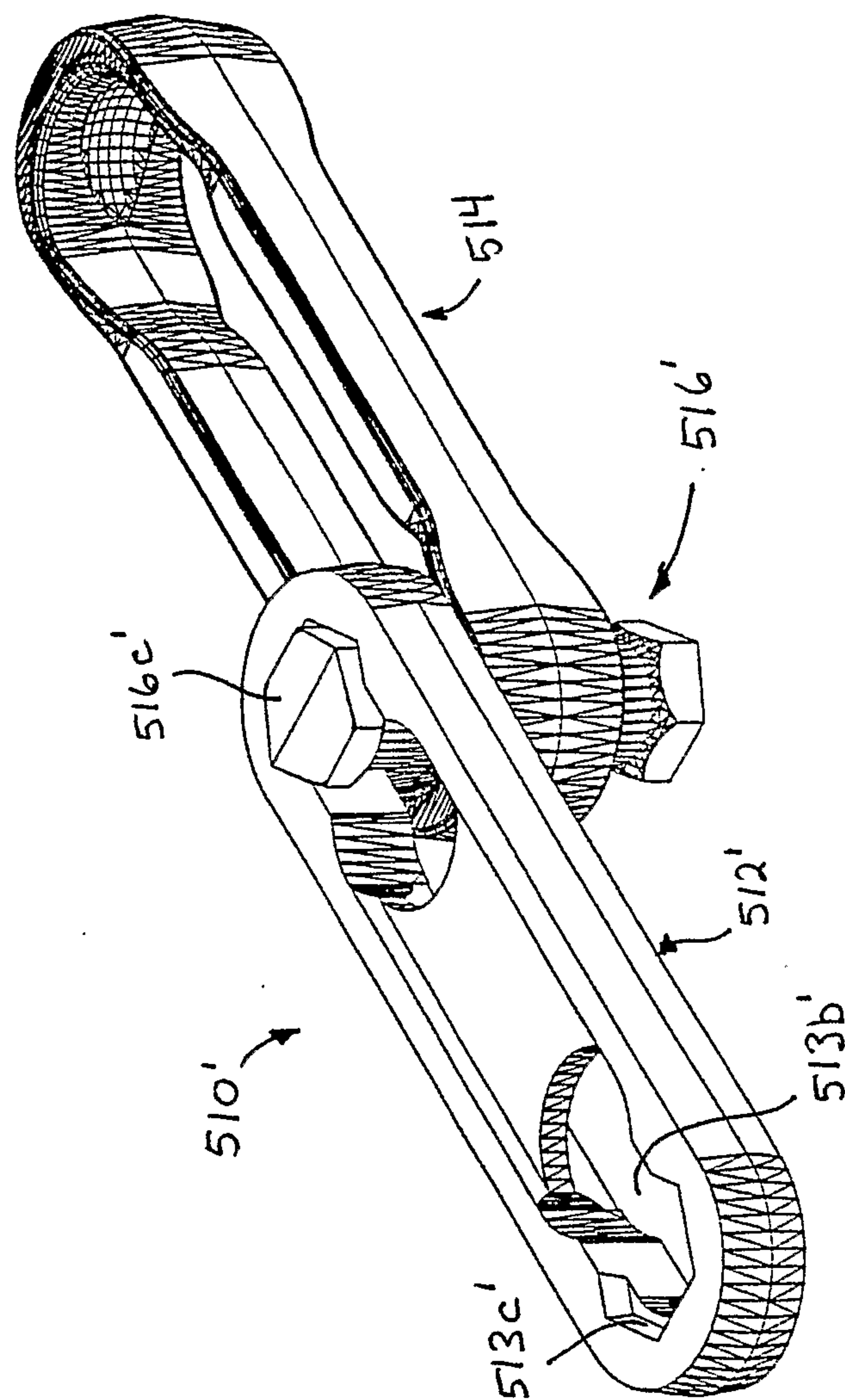


FIG. 23

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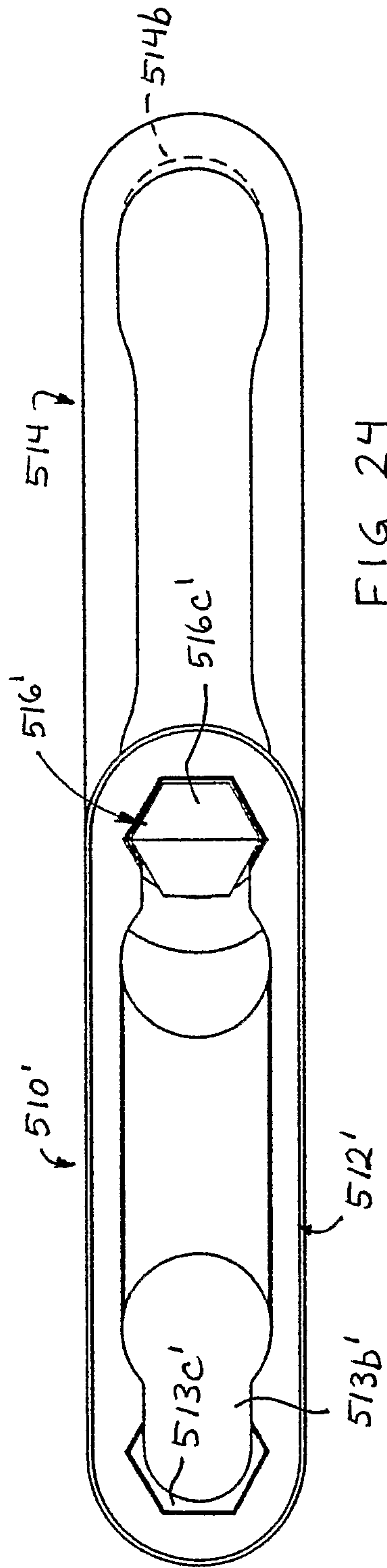


FIG. 24

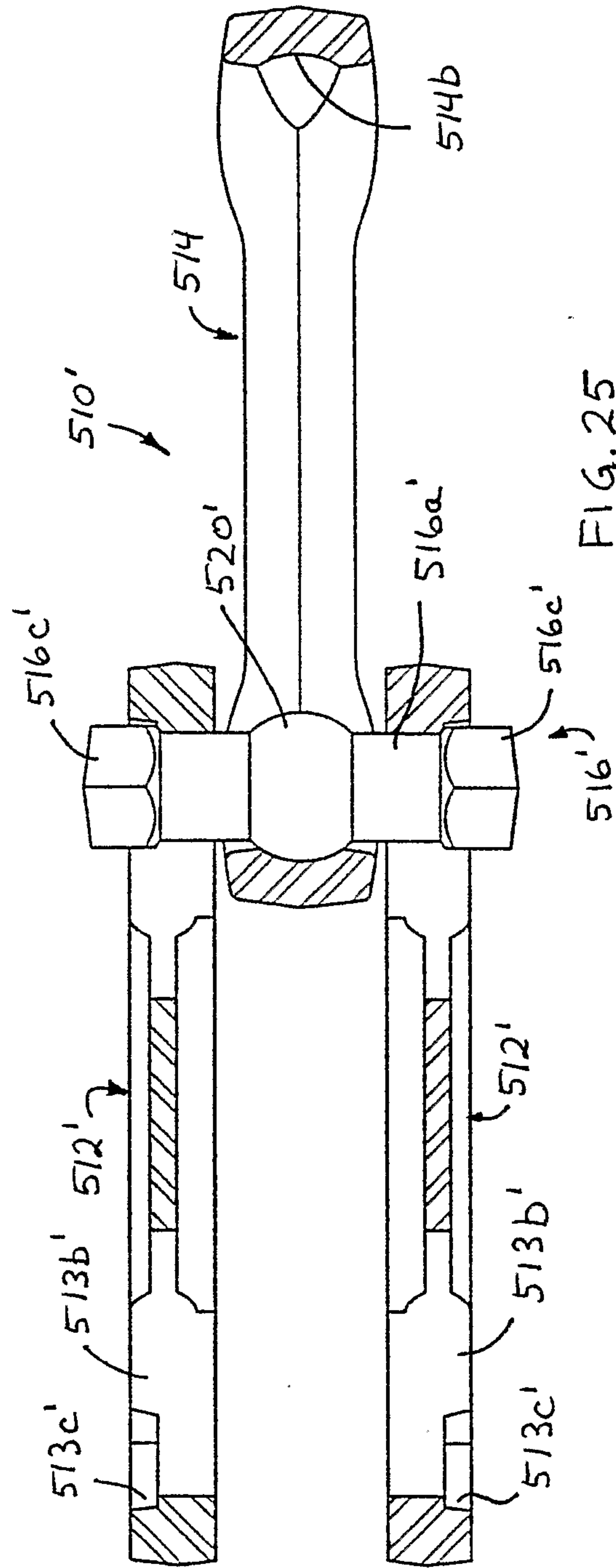
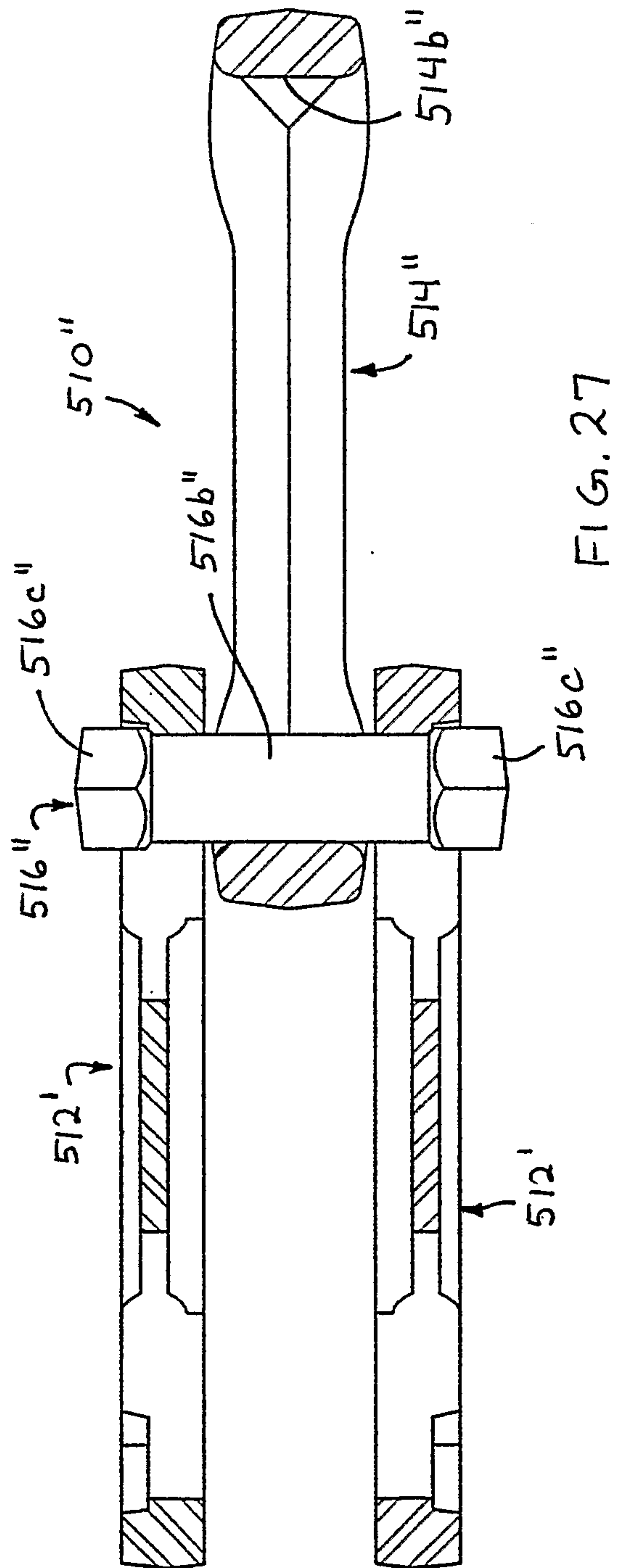
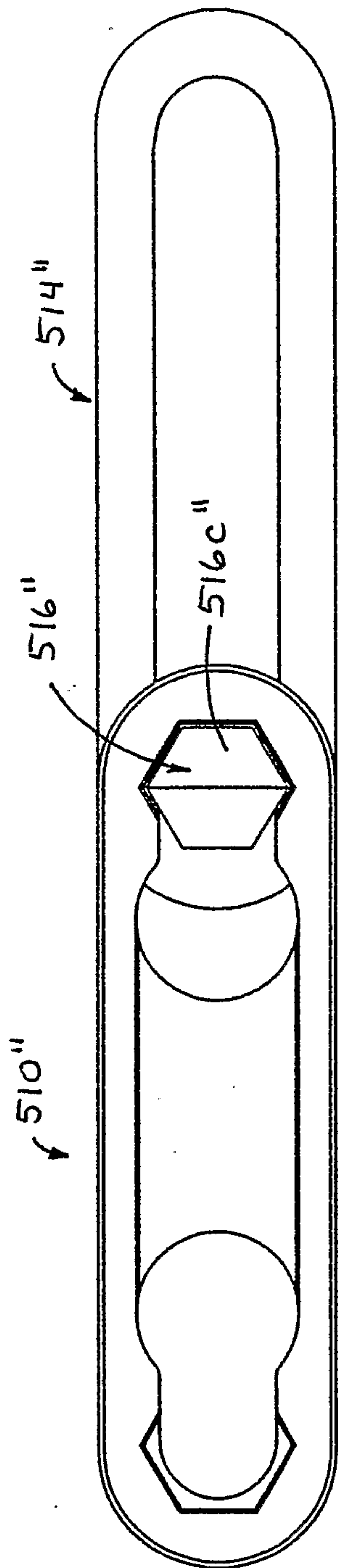
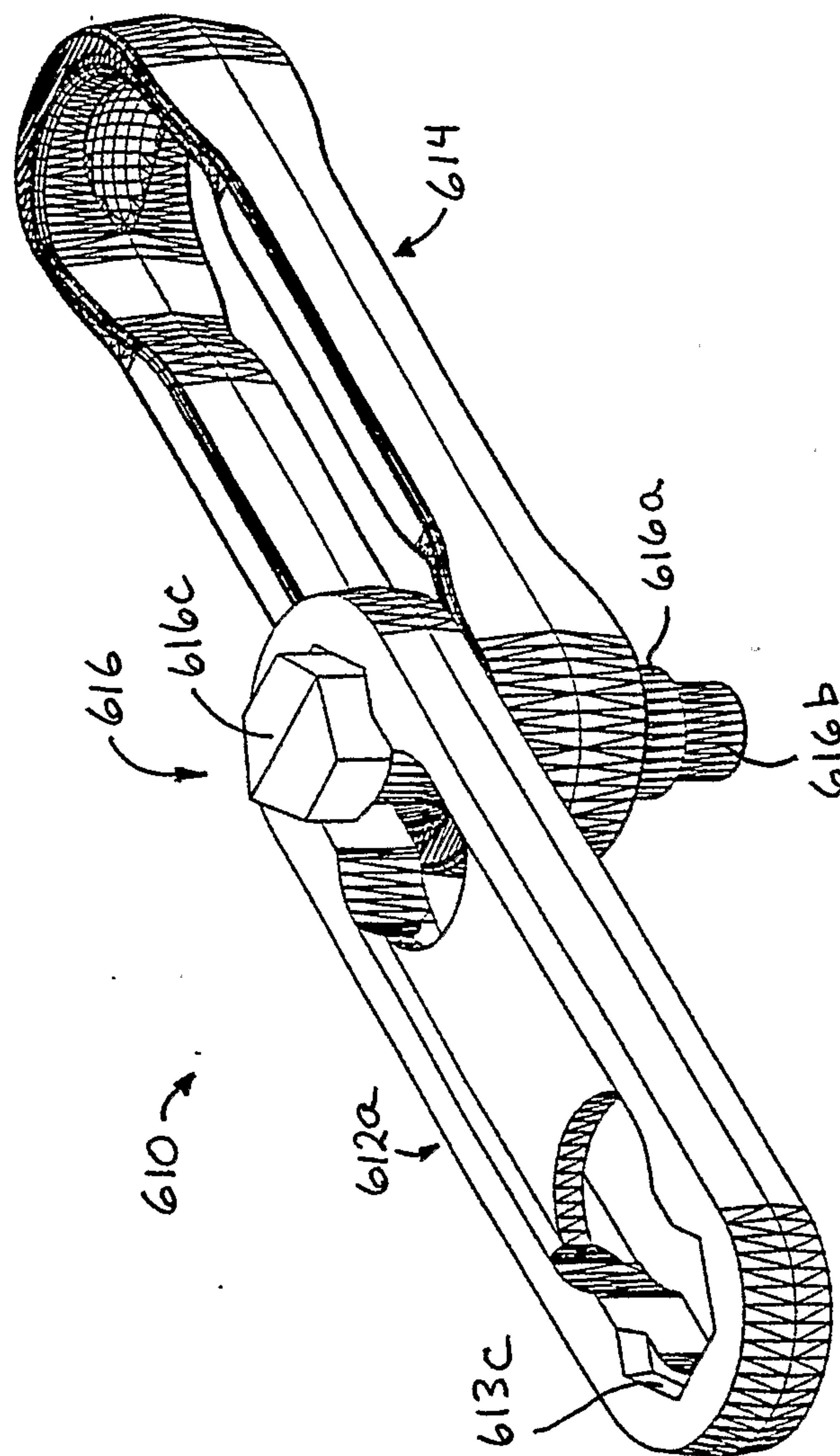


FIG. 25

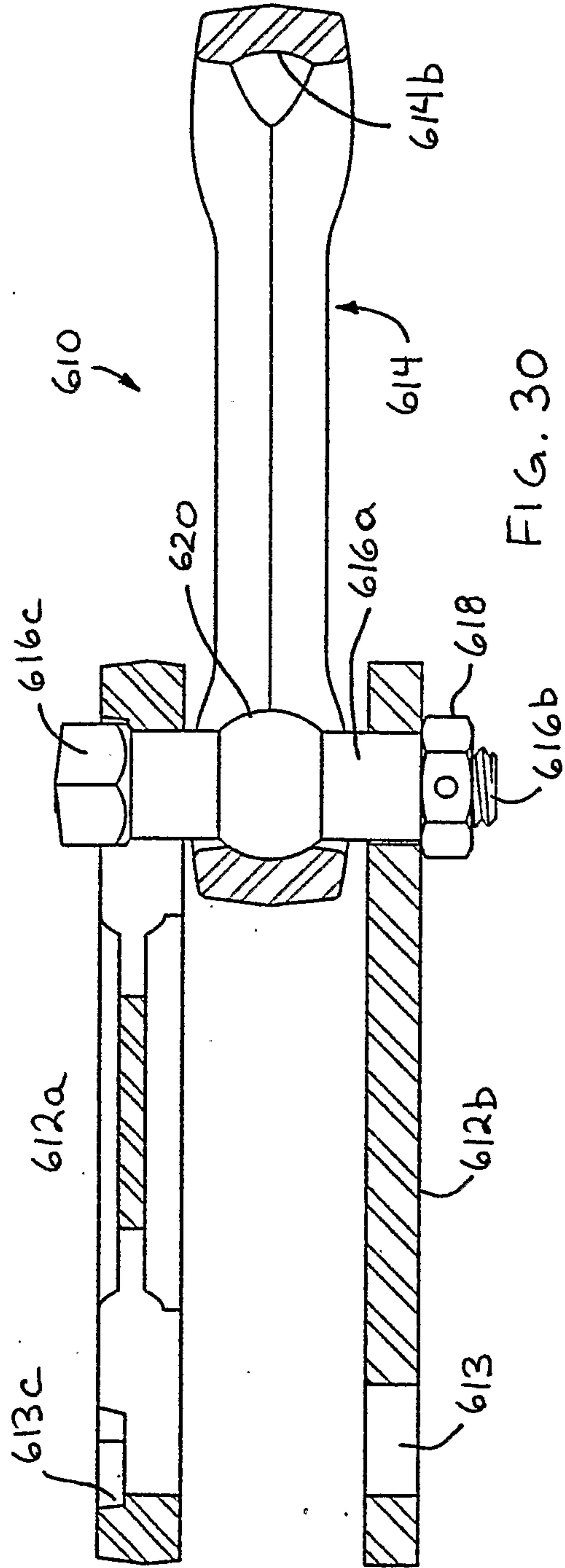
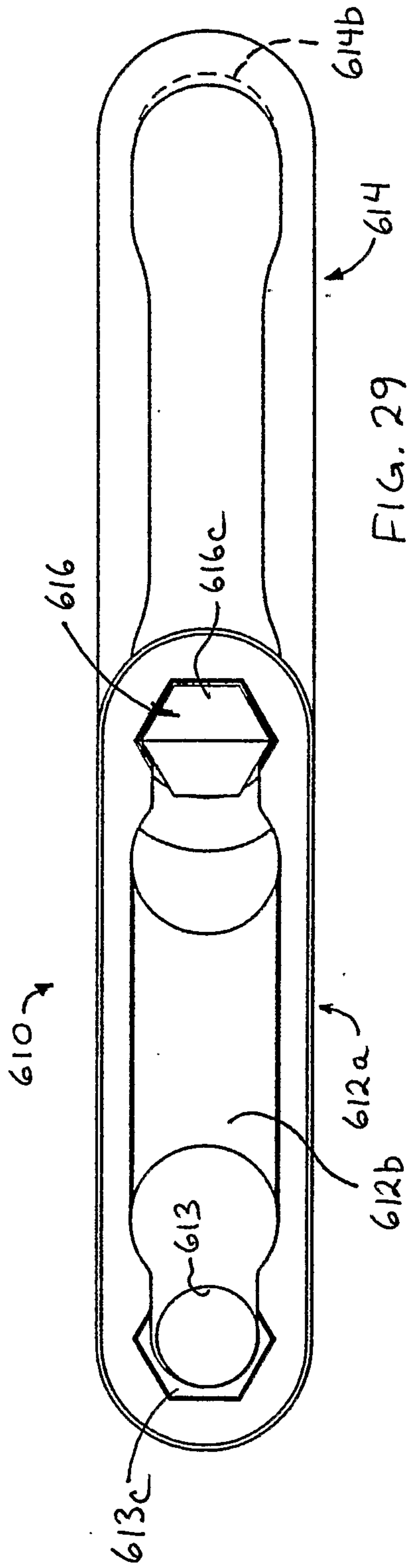
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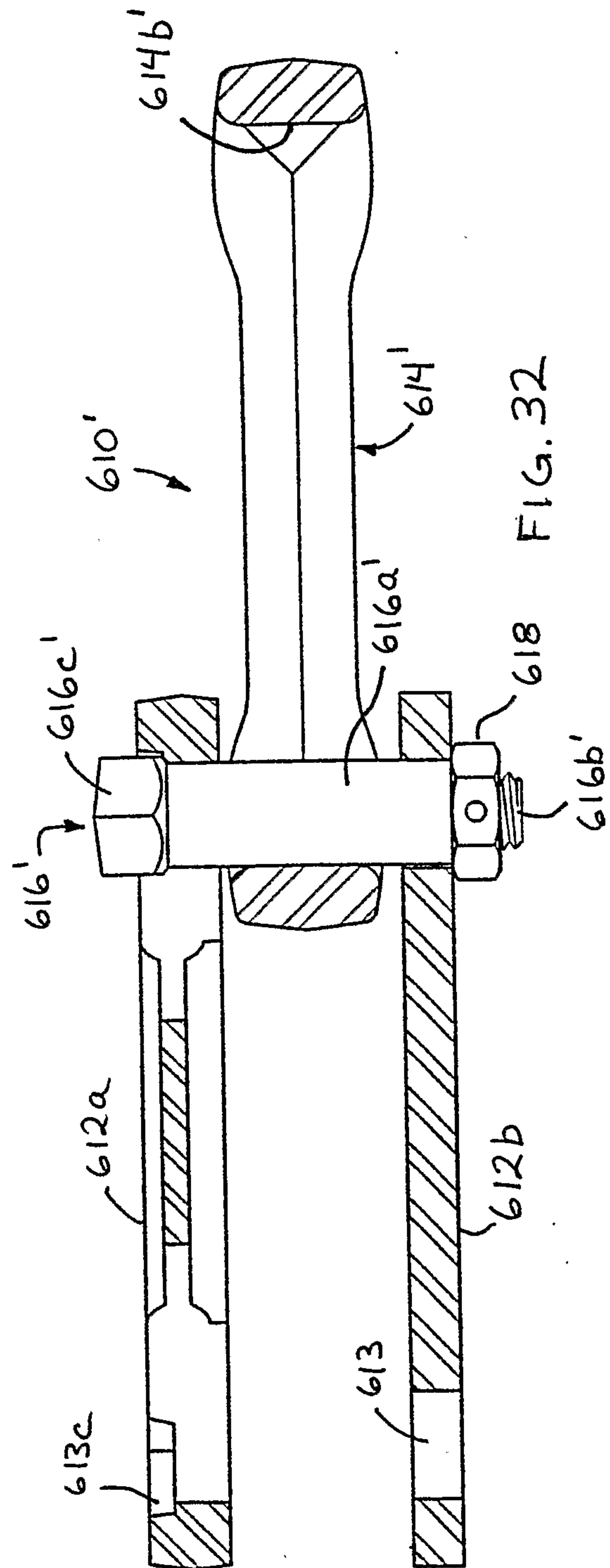
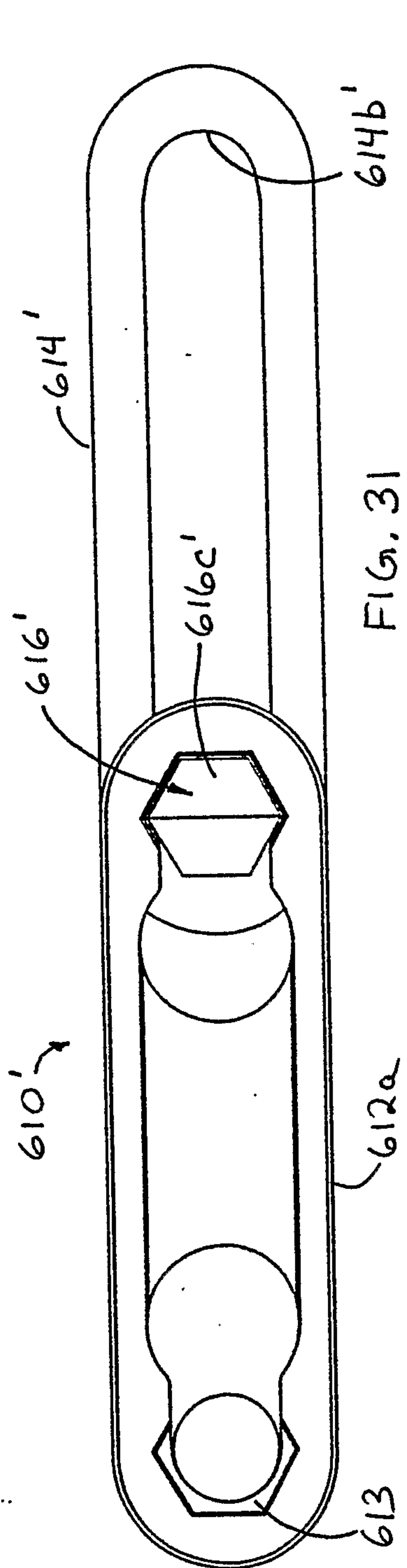
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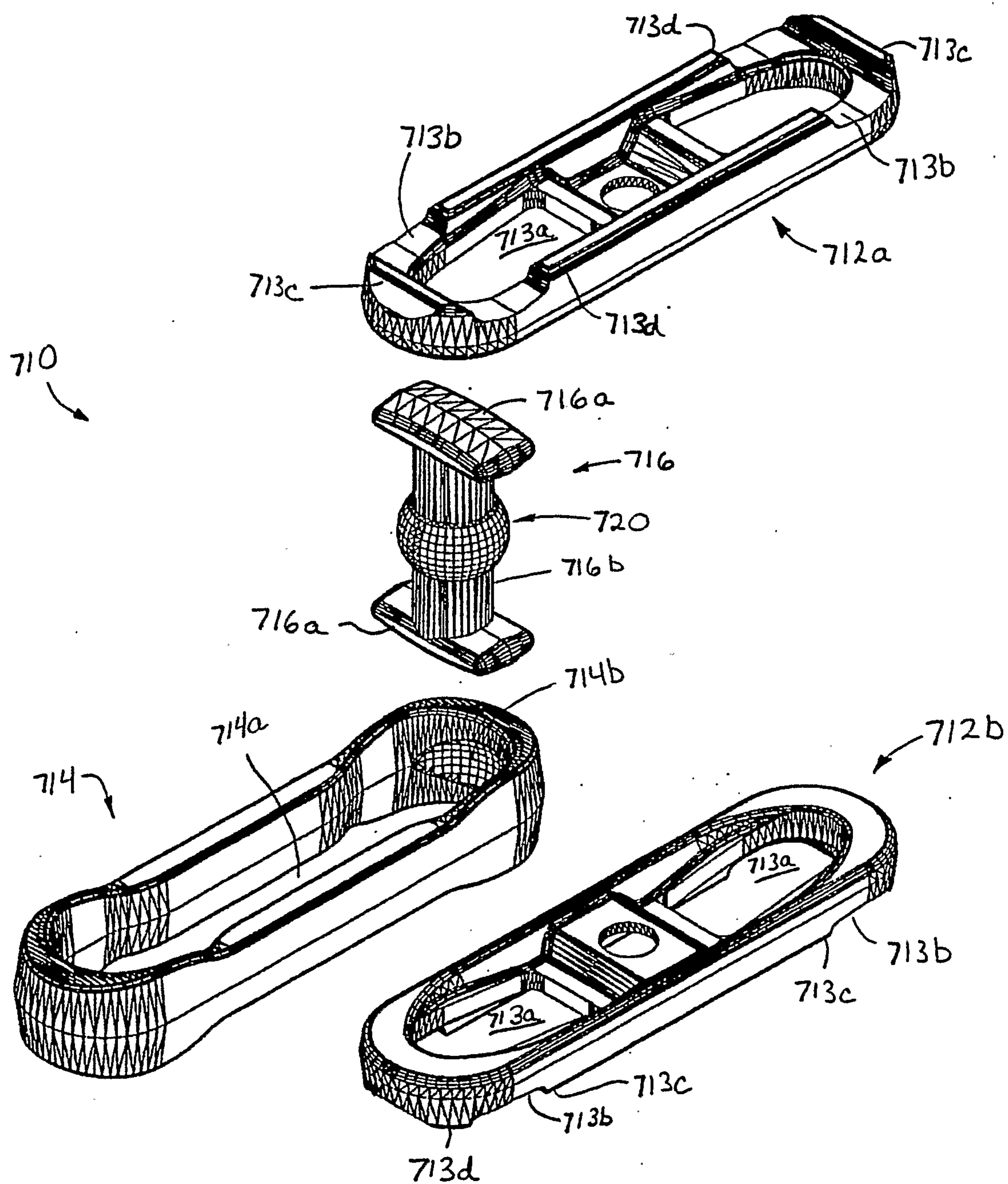


FIG. 33

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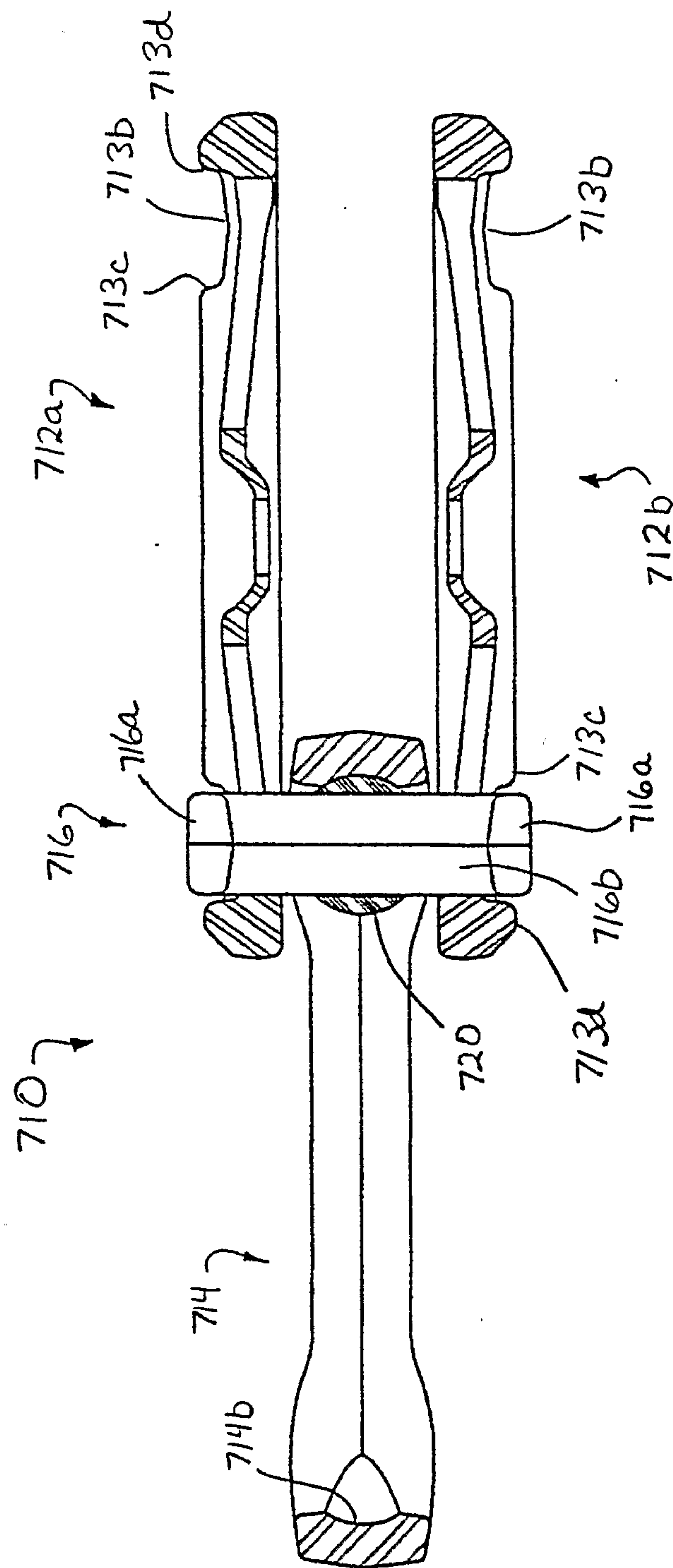


FIG. 34

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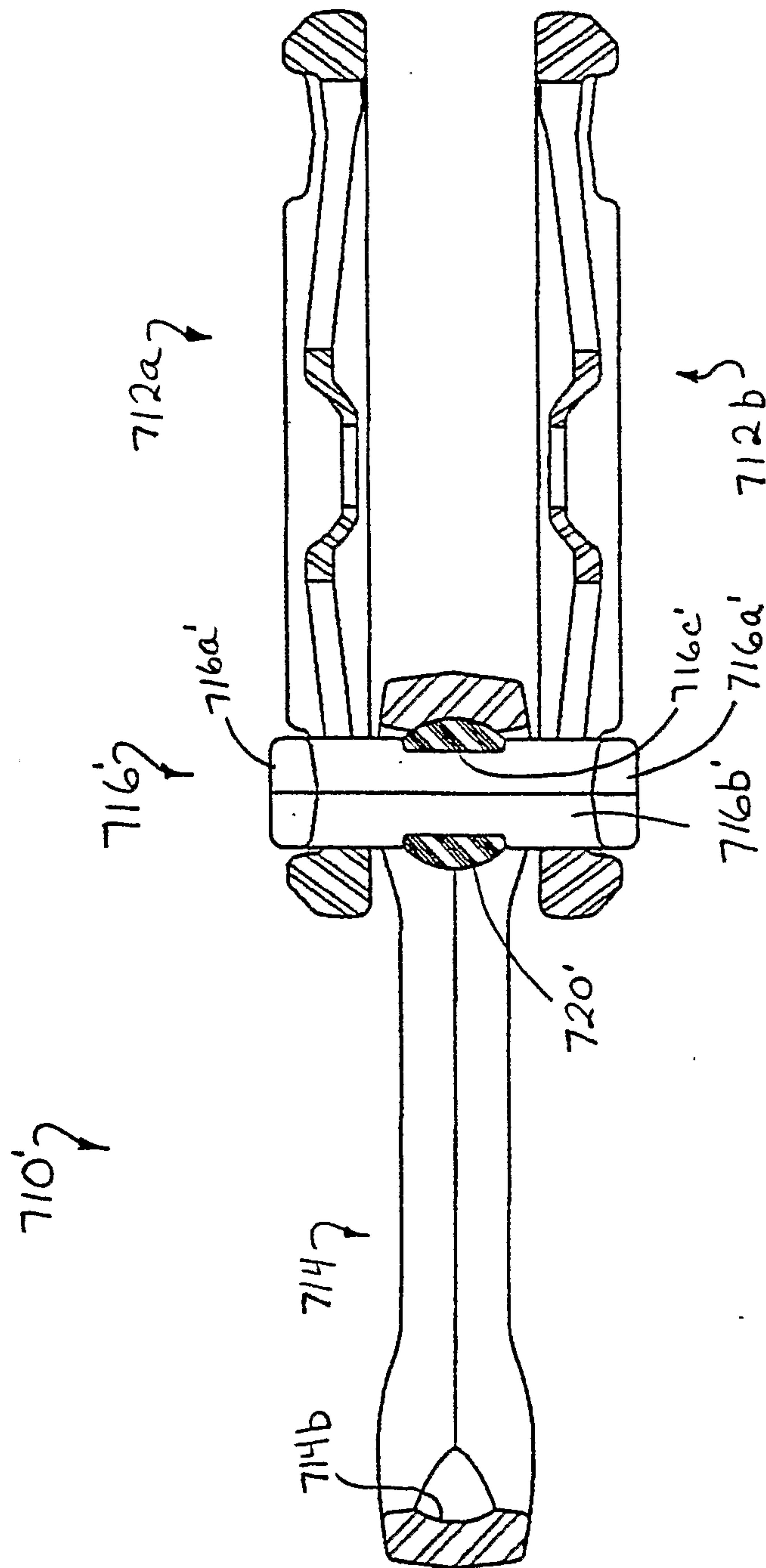


FIG. 35

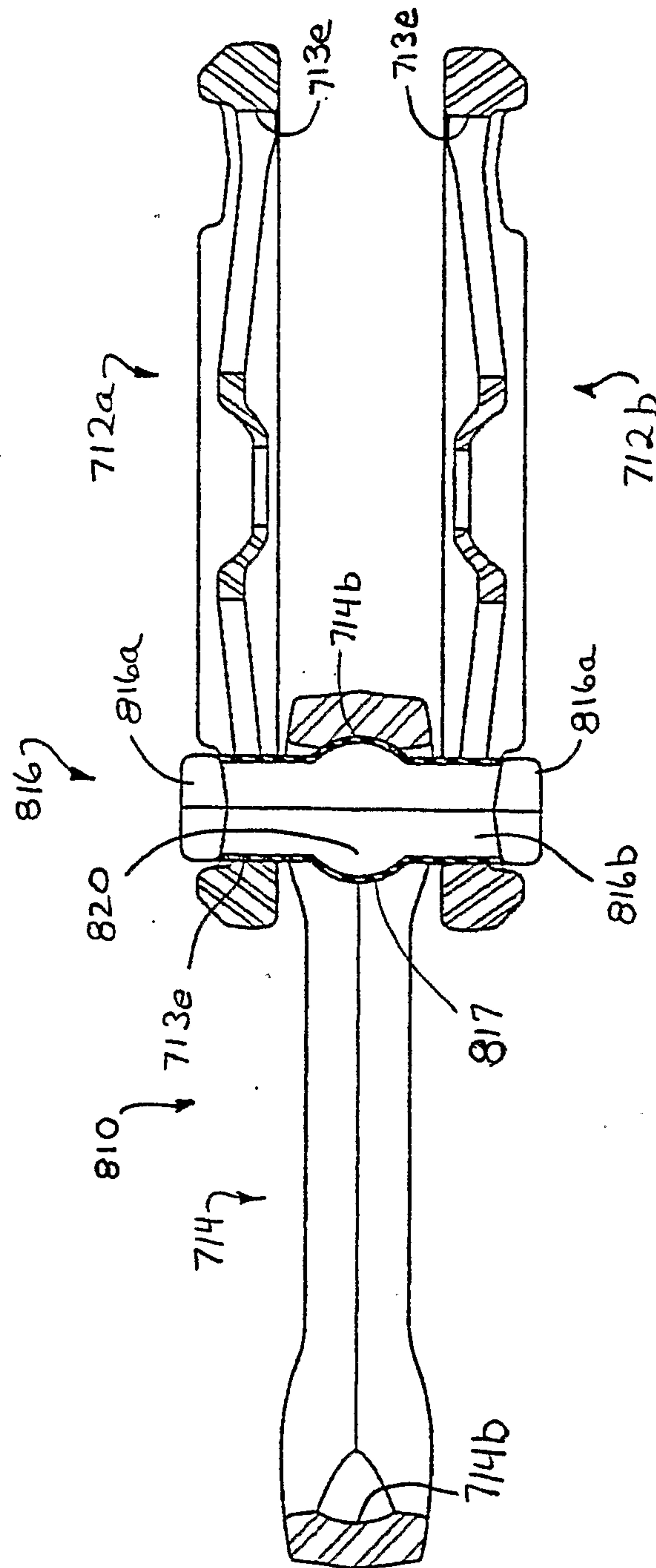


FIG. 36

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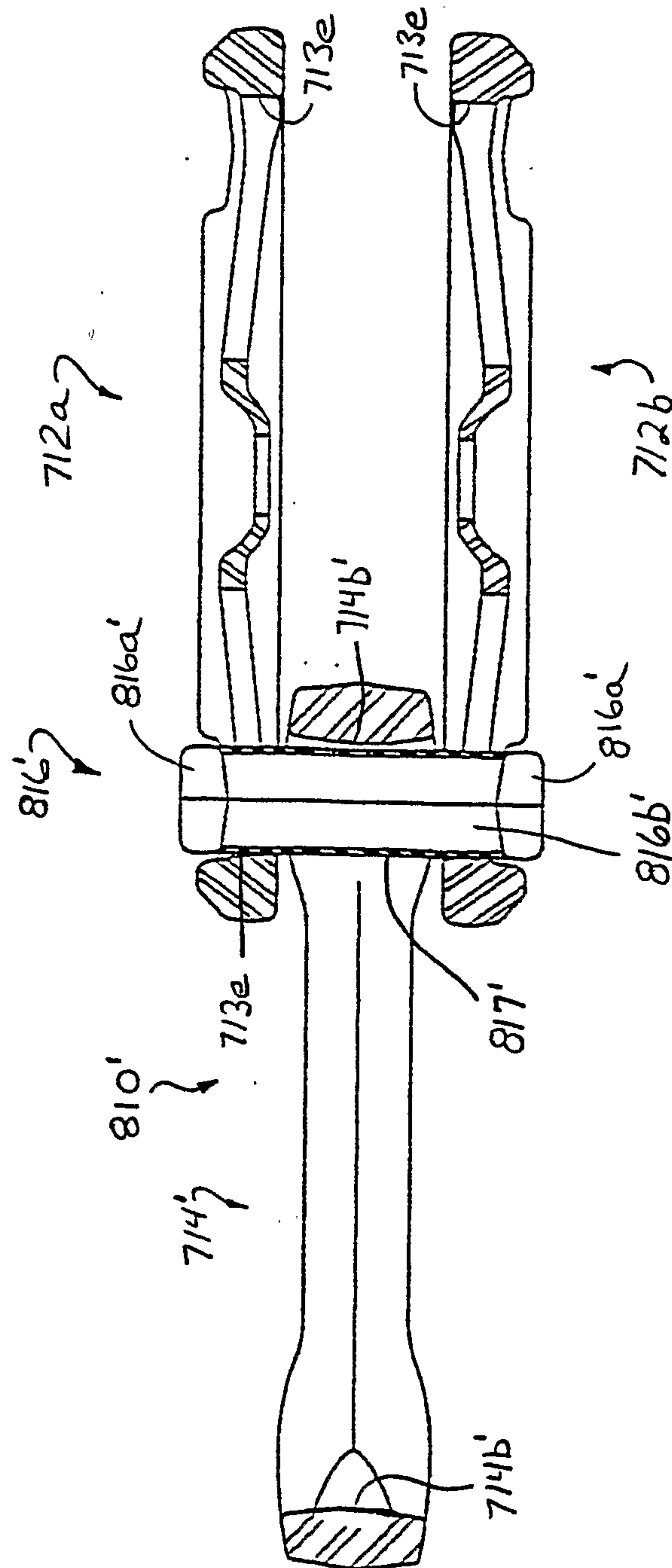


FIG. 37

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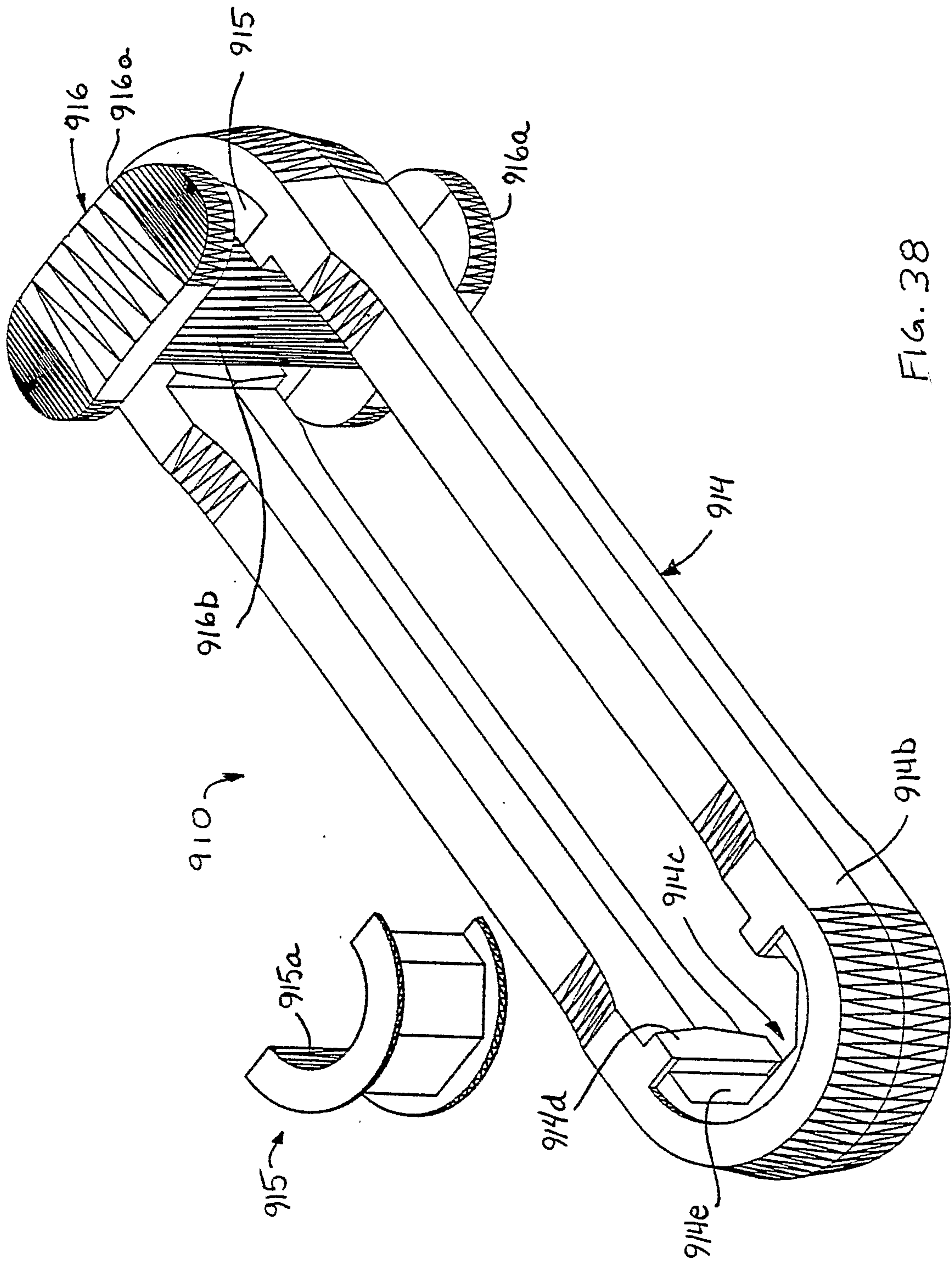


FIG. 38

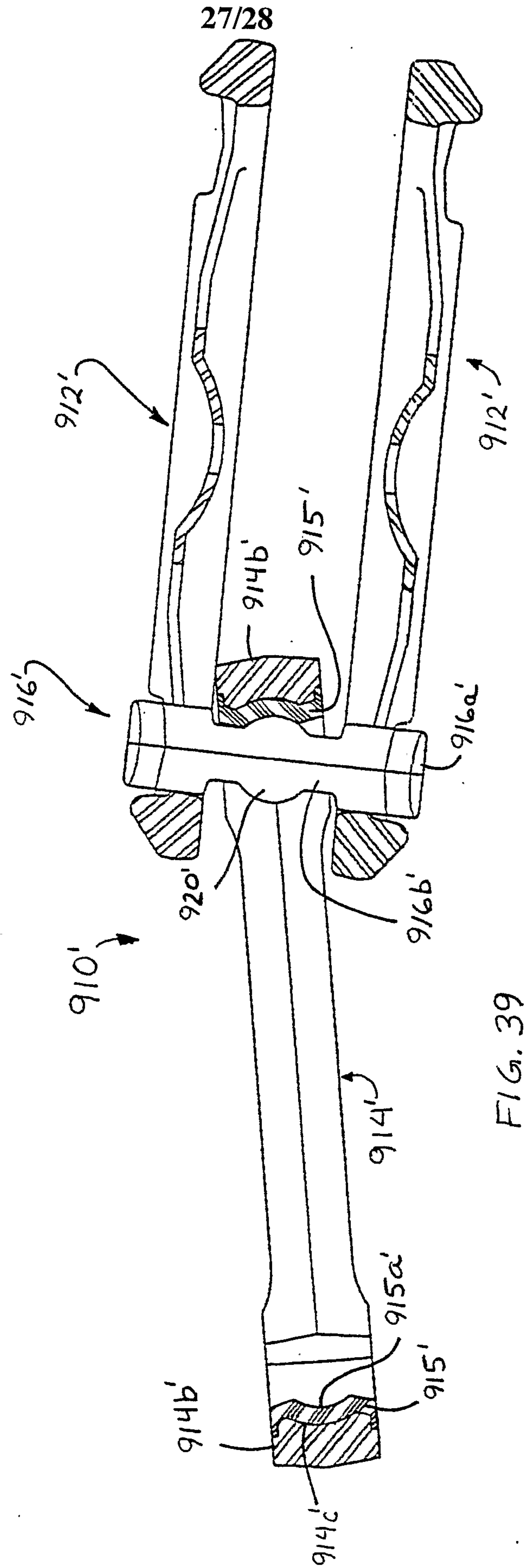


FIG. 39

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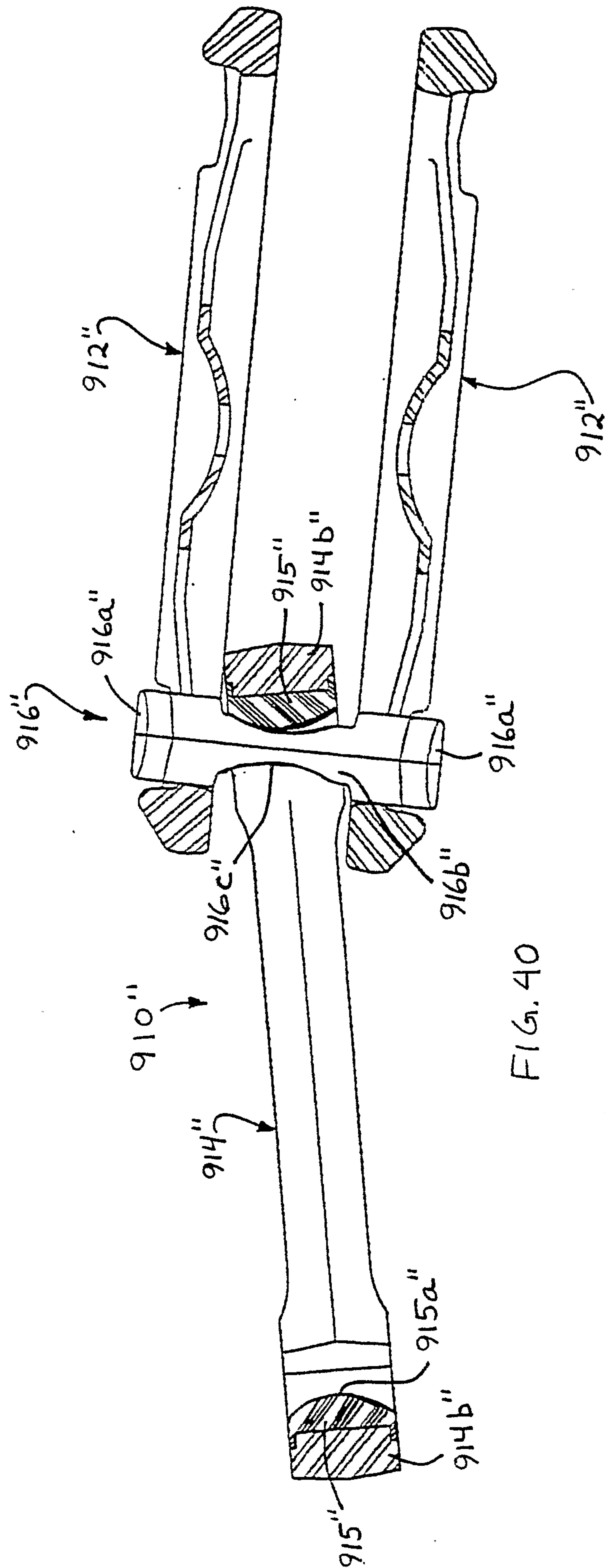


FIG. 40

