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DESCRIPTION

Cross-Reference to Related Applications

[0001] This application is a continuation-in-part of US Application Serial No, 13/542,222, filed on July 5, 2012, which is a continuation-in-part of US Application Serial No. 12/463,060 filed on May 8, 2009. This application also claims the benefit of US Application Serial No, 61/601,876, filed on February 22, 2012.

Technical Field

[0002] The present disclosure relates to an automated system to sense the operating condition of a combustion system and to make preset adjustments to achieve desired operation of the turbine thru out an optimized load range.

Background

[0003] Lean premixed combustion systems have been deployed on land based gas turbine engines to reduce emissions, such as NO_x and CO. These systems have been successful and, in some cases, produce emission levels that are at the lower limits of measurement capabilities, approximately 1 to 3 parts per million (ppm) of NO_x and CO. Although these systems are a great benefit from a standpoint of emission production, the operational envelope of the systems is substantially reduced when compared to more conventional combustion systems. As a consequence, the control of fuel conditions, distribution and injection into the combustion zones has become a critical operating parameter and requires frequent adjustment, when ambient atmospheric conditions, such as temperature, humidity and pressure, change. The re-adjustment of the combustion fuel conditions, distribution and injection is termed tuning.

[0004] Controlled operation of a combustion system generally employs a manual setting of the operational control elements of a combustor to yield an average operational condition. These settings may be input through a controller, which as used herein shall refer to any device used to control the operation of a system. Examples include a Distributed Control System (DCS), a fuel turbine controller, a programmable logical controller (PLC), a stand-alone computer with communication to another controller and/or directly to a system,

[0005] These settings are satisfactory at the time of the setup, but conditions may change and cause unacceptable operation in a matter of hours or days. Tuning issues are any situation whereby any operational parameters of a system are in excess of acceptable limits. Examples include emissions excursion outside of allowable limits, combustor dynamics excursion outside

of allowable limits, or any other tuning event requiring adjustment of a turbine's operational control elements, Other approaches use a formula to predict emissions based on gas turbine's operating settings and select a set point for fuel distribution and/or overall machine fuel/air ratio, without modifying other control elements, such as fuel gas temperature, Still other approaches employ a remote connection to the site by tuning experts, that will periodically readjust the tune, from the remote location, These approaches do not allow for continuous timely variation, do not comprehensively take advantage of actual dynamics and emission data or do not modify fuel distribution, fuel temperature and/or other turbine control elements,

[0006] Another variable that impacts the lean premixed combustion system is fuel composition, Sufficient variation in fuel composition will cause a change in the heat release of the lean premixed combustion system. Such change may lead to emissions excursions, unstable combustion processes, or even blow out of the combustion system,

[0007] In recent years, over-capacity of power generation, even that using F-class firing temperature gas turbines, has resulted in much of the installed gas turbine fleet running in a cyclic mode versus baseload operation, This means that many gas turbine operators are forced to shut their equipment down overnight, when power prices are so low that the losses incurred running overnight far outweigh the costs of starting the equipment every morning, This operation process has an impact on the maintenance of the equipment as each stop / start cycle causes a resulting load cycle on the equipment.

[0008] To combat this situation, gas turbine operators are investigating ways of running their equipment overnight while incurring the smallest economic loss possible. One viable solution is to lower the minimum load a gas turbine can achieve while still maintaining acceptable emissions levels, This method of operation is commonly referred to as "Turndown."

[0009] "Turndown" has been used within the power generation industry for many years, As such, nothing directly related to this mode of operation is included as part of this patent, What is novel is the approach used by the ECOMAX™ tuning controller to tune the combustion system while in turndown, as well as the method incorporated within ECOMAX™ to mitigate detrimental effects on the combined-cycle heat recovery steam generator (HRSG) caused by low steam flows and high gas turbine exhaust temperatures, The system is also applicable to simple. cycle operation; however, most simple cycle systems are applied to peak power generation and have a desirable shut - down process in the operating plan.

[0010] Often, as gas turbine loads are reduced, HRSG steam flows reduce while gas turbine exhaust temperatures rise. This combination, in conjunction with inadequate intra-stage attemperation flow capacity, often results in excessively high HRSG outlet steam temperatures (steam turbine inlet steam temperatures). In many cases these steam temperatures approach material limitations and can lead to pre-mature component failure. On the other extreme, steam conditioning / attemperation systems with adequate condensate flows can provide enough condensate to keep the superheat outlet temperature within specifications at the point of entrance into a steam turbine; however, localized over-attemperation can occur. This over-

attenuation often leads to condensate impacting directly on steam piping downstream of the attenuator, causing excessive thermal fatigue in the piping sections immediately downstream of desuperheaters / attenuators.

[0011] To date efforts have focused on manually (if at all) modifying a gas turbine's fuel-to-air (f/a) ratio to keep the HRSG design constraints satisfied, However, factors such as ambient temperature changes, turbine component degradation, etc., can necessitate periodic manipulation of the gas turbine's f/a ratio, at low loads, to ensure acceptable HRSG inlet conditions, Automated manipulation of the f/a ratio of a gas turbine utilizing real-time HRSG operational information, as well as real-time gas turbine operational information, provides an efficient means to maximize HRSG component life.

[0012] It is understood that manipulation of a gas turbine's fuel-air ratio will directly affect the engine's "tune", and as such any approach to accomplish this must be accompanied by another automated turbine control scheme to "re-tune" the turbine as-needed.

[0013] Mis-operation of the combustion system manifests itself in augmented pressure pulsations or increasing of combustion dynamics, Pulsations can have sufficient force to destroy the combustion system and dramatically reduce the life of combustion hardware. Additionally, improper tuning of the combustion system can lead to emission excursions and violate emission permits. Therefore, a means to maintain the stability of the lean premixed combustion systems, on a regular or periodic basis, within the proper operating envelope, is of great value and interest to the industry, Additionally, a system that operates by utilizing near real-time data, taken from the turbine and HRSG sensors, would have significant value to coordinate modulation of fuel composition, fuel distribution, fuel gas inlet temperature, and/or overall machine f/a ratio (HRSG inlet temperature and airflow).

[0014] US Patent Publication No. 2011/0056181 is generally directed to control systems and methods for controlling a load point of a gas turbine engine. The control system includes a controller that receives a temperature signal and a pressure signal associated with exhaust gases from the gas turbine engine.

[0015] US Patent Publication No. 2010/0286890 is generally directed to a system for tuning the operation of a gas turbine based on measuring operational parameters of the turbine and directing adjustment of the operational controls for various operational elements of the turbine.

[0016] US Patent No. 5,319,936 is generally directed to combustor systems and turbine systems for stabilizing a premixed flame having first and second premixed burners having annular injection slots are concentrically adjacent and the injection slots are substantially aligned with each other in a longitudinal direction of the burners.

[0017] US Patent Publication No. 2008/0289314 is generally directed to a fuel delivery system for a gas turbine engine includes one or more subsets of combustors, and at least two fuel manifolds including a fuel manifold coupled to each combustor subset and configured to deliver

fuel to only the subset of combustors during a predetermined gas turbine engine operating mode.

[0018] US Patent No. 4,333,310 is generally directed to a combined cycle electric power plant includes gas and steam turbines and a steam generator for recovering the heat in the exhaust gases exited from the gas turbine and for using the recovered heat to produce and supply steam to the steam turbine.

[0019] US Patent Publication No. 2007/0163267 is generally directed to a combustor for a gas turbine, the combustor including a burner system and a fuel supply system. A sensing system measures pressure pulsation values and/or emission values for each burner group. US Patent Publication No. 2006/0040225 is generally directed to a method for operating a furnace with a multi-burner system for generating hot gas, particularly a gas turbine, preferably of a power plant, comprising a combustion chamber with at least one burner. EP Patent Publication No. 1909032 is generally directed to a method of reducing emissions in a gas turbine engine including generating a swirling flow with a first oxygen source and a first fuel within a combustor, combusting the first fuel using the first oxygen source to produce combustion products within the combustor, generating a swirling flow with a second oxygen source and a second fuel and mixing with the combustion products, combusting the second fuel using the second oxygen source to produce second combustion products within the combustor, and generating a swirling flow with a third oxygen source and a third fuel and mixing with the second combustion products.

[0020] US Patent Publication No. 2011/0265487 is generally directed to tuning processes implemented by an auto-tune controller for measuring and adjusting the combustion dynamics and the emission composition of a gas turbine (GT) engine via a tuning process. US Patent Publication No. 2009/0126367 is generally directed to a gas turbine combustion system for a turbine engine that may include a plurality of combustor baskets positioned circumferentially about a longitudinal axis of the turbine engine, at least one firing nozzle extending into each of the plurality of combustor baskets and a fuel delivery system in communication with each of the at least one firing nozzles and in communication with at least one fuel source.

[0021] US Patent Publication No. 2010/0077970 is generally directed to systems and methods for controlling exhaust steam temperatures from a finishing superheater. The system may include a controller which includes control logic for predicting an exhaust temperature of steam from the finishing superheater using model-based predictive techniques.

Summary

[0022] Provided herein is a system according to claim 1, a system according to claim 12, a method according to claim 14 and methods according to claims 16 or 19 for tuning the operation of a turbine and optimizing the mechanical life of a heat recovery steam generator, Provided therewith is a turbine controller, sensor means for sensing operational parameters,

control means for adjusting operational control elements, The controller is adapted to tune the operation of the gas turbine in accordance preprogrammed steps in response to operational priorities selected by a user, The operational priorities preferably comprise optimal heat recovery steam generator life,

[0023] The present disclosure provides a controller and method for optimizing the fuel-air ratio of a gas-turbine combustor toward mitigating the detrimental effects of the turbine's exhaust conditions on the expected life of a Heat Recovery Steam Generator (HRSG) system, especially during low-load conditions, The gas turbine consumption system is of the type having sensor means for measuring operational parameters of the turbine and control means for controlling various operational elements of the turbine, The operational parameters of the turbine which are received by the controller include combustor dynamics, turbine exhaust temperature (overall fuel/air ratio), turbine exhaust emissions, and various heat recovery steam generator (HRSG) steam conditions, The operational control elements may include the fuel gas blend ratio (ratio of non-pipeline quality fuel gas to pipeline quality fuel gas), fuel distribution within the combustion system, fuel temperature and turbine exhaust temperature, The turbine/power plant system can also include a distributed control system (DCS) communicating with the sensor means and the control means, The tuning controller is normally connected to the turbine system through the DCS (although the tuning controller can connect directly to the gas turbine controller),

[0024] The tuning controller operates by receiving data from the sensor means, Operational priorities for the turbine are set within the controller and are typically selected from optimum NOx emissions, optimum power output, optimum combustor dynamics, optimum fuel gas blend ratio, and / or optimum HRSG life, The data received from the turbine sensors is compared to stored operational standards within the controller, The selected operational standards are based on the set operational priorities. A determination is made as to whether the turbine operation conforms to the operational standards, In addition, upon the data being out of conformance, a determination is made of the dominant tuning criteria again based on the preset operational priorities, Once the logical determinations are made, the tuning controller communicates with the operational control means through the DCS to perform a selected adjustment in an operational parameter of the turbine, The selected operational adjustment is based on the dominant tuning criteria and has a preset fixed incremental value and defined value range. Each incremental change is input over a set period of time, which is sufficient for the turbine to gain operational stability, Once the time period passes, operational data is again received from the turbine sensor means to determine if an additional incremental change is desired. Generally speaking, upon completing the adjustments within the defined range, a further operational parameter adjustment is selected, again based on the dominant tuning criteria, and a further fixed incremental adjustment is made within a defined range and over a set period of time, The tuning process continues by the controller receiving operational data to determine if the operation is conforming to the operational standards or whether an additional incremental adjustment is required. The operational parameters being adjusted by the tuning controller are preferably the fuel/air ratio within the gas turbine, the combustor fuel distribution split within the nozzles of the combustor, the fuel gas inlet temperature, and/or the fuel gas

blend ratio.

[0025] It is understood that the tuning controller, when Optimum HRSG life is selected by the plant operator, will first evaluate what changes (if any) need to be made to the gas turbine f/a ratio to mitigate potential HRSG mechanical concerns and make these necessary changes. Subsequent to this optimization process, the tuning controller will tune the gas turbine, if needed, using the standard parameters of fuel splits, fuel gas temperature, and/or fuel gas composition (Note: gas turbine f/a ratio is not an option),

[0026] In a further aspect of the disclosure, the system performs a method for determination of the dominant gas turbine combustion system tuning scenario through the use of Boolean hierarchical logic and multiple levels of control settings,

[0027] In another aspect of the disclosure, the method performed relates to automated control of the gas turbine inlet fuel temperature through automated modification of the fuel gas temperature control set point within a Distributed Control System (DCS).

[0028] In a still further aspect of the disclosure, a method for automated control of a gas turbine inlet fuel temperature is defined by automated modification of the fuel gas temperature control set point within the fuel gas temperature controller,

[0029] In another aspect of the disclosure a method for communicating turbine control signals to a gas turbine controller is accomplished through the use of an existing gas turbine communication link with an external control device, such as, for example a MODBUS Serial or Ethernet communication protocol port existing on the turbine controller for communication with the Distributed Control System (DCS).

[0030] In a still further aspect of the disclosure a method for modification of a gas turbine combustion system is defined by a series of auto tuning settings via a user interface display, which utilizes Boolean-logic toggle switches to select user-desired optimization criteria. The method is preferably defined by optimization criteria based on Optimum Combustion Dynamics, Optimum NO_x Emissions, Optimum Power, Optimum Heat Rate, Optimum CO Emissions, Optimum Heat Recovery Steam Generator (HRSG) Life, Optimum Gas Turbine Fuel Blend Ratio or Optimal Gas Turbine Turndown Capability whereby toggling of this switch changes the magnitude of the combustor dynamics control setting(s).

Brief Description of Drawings

[0031] For the purpose of illustrating the invention disclosed herein, the drawings show forms that are presently preferred. It should be understood that the invention is not limited to the precise arrangements and instrumentalities shown in the drawings of the present disclosure,

Fig. 1 shows an exemplary embodiment of a schematic representation of an operational plant

communication system encompassing the gas turbine engine system, incorporating a gas turbine tuning controller as well as communication with various elements of the HRSG via plant DCS.

Fig. 2 shows an exemplary embodiment of a functional flow chart for the operation of a tuning controller according to the present disclosure.

Fig. 3 shows an exemplary embodiment of a user interface display for selecting the optimization mode within the present disclosure,

Fig. 4 shows an exemplary schematic of the inter-relationship of various optimization mode settings,

Fig. 5 shows an exemplary overview schematic of the process steps utilized to determine the alarm signals triggered according to the present disclosure,

Fig. 6 shows an exemplary process overview of the steps to determine allowable turbine tuning parameters,

Fig. 7 shows a further detailed exemplary process according to the steps shown in Fig. 6.

Fig. 8 provides a further detailed exemplary schematic of the steps the present disclosure utilizes to determine the dominant tuning concern,

Fig. 9 shows a first example schematic of the determination of the system's dominant tuning concern, given various alarm inputs into the present disclosure,

Fig. 10 shows a second example schematic of the determination of the system's dominant tuning concern, given various alarm inputs into the present disclosure.

Fig. 11 shows a third example schematic of the determination of the system's dominant tuning concern, given various alarm inputs into the present disclosure,

Fig. 12 shows a fourth example schematic of the determination of the system's dominant tuning concern, given various alarm inputs into the present disclosure,

Fig. 13 shows a first operational example of operational tuning of a gas turbine engine system as contemplated by the present disclosure,

Fig. 14 shows a second operational example of operational tuning of a gas turbine engine system as contemplated by the present disclosure,

Fig. 15 shows a third operational example of operational tuning of a gas turbine engine system as contemplated by the present disclosure,

Fig. 16 shows a fourth operational example of operational tuning of a gas turbine engine system as contemplated by the present disclosure.

Detailed Description

[0032] The present disclosure generally relates to systems and methods for tuning the operation of combustion turbines, In the depicted embodiments, the systems and methods relate to automatic tuning of combustion turbines, such as those used for power generation, Persons of ordinary skill in the art will appreciate that the teachings herein can be readily adapted to other types of combustion turbines, Accordingly, the terms used herein are not intended to be limiting of the embodiments of the present disclosure, Instead, it will be understood that the embodiments of the present disclosure relate generally to the field of combustion turbines, and in particular for systems, methods and computer readable media for tuning of combustion turbines,

[0033] Fig. 1 is a communication diagram for a gas turbine engine (not shown), within which a tuning controller 10 of the present disclosure operates, A distributed control system (DCS) 20 serves as the main communication hub, As an alternative, a plant using the gas turbine controller as a DCS, may also have the tuning controller 10 communicate directly to the gas turbine controller 30, As a further alternative, the tuning controller 10 can communicate directly with the gas turbine controller 30, irrespective if the gas turbine controller 30 is functioning as a DCS. Most of the turbine control is performed through the DCS 20, A turbine controller 30 communicates directly with the gas turbine and with the DCS 20, In the present disclosure, information relevant to turbine operation, e.g., turbine dynamics, turbine exhaust emissions, etc. are directed through the DCS 20 to the tuning controller 10. The tuning controller 10 is contemplated to be a stand-alone PC used to run as a programmable logical controller (PLC), The tuning controller 10 is preferably a separate computer from the turbine controller 30 and does not normally communicate directly with the turbine controller 30, except through the DCS 20, However; as mentioned above, the tuning controller 10 can be configured to directly communicate with the gas turbine controller 30,

[0034] Referring now to Figs. 1 and 2, the tuning controller 10 is contemplated to be a stand-alone PC used to run as a programmable logical controller (PLC). The tuning controller 10 is preferably a separate computer from the turbine controller 30 that is in constant communication from with the turbine controller 30, The signals from the tuning controller 10 may also be transferred to the turbine controller 30 or other controls within the system by the use of an external control device, such as a MODBUS Serial or Ethernet communication protocol port existing on or added to the system,

[0035] The relevant operational data is received from sensor means associated with the turbine, For example, the turbine exhaust emission reading is taken from stack emissions by a continuous emissions monitoring system (CEMS) 40, which is connected to the DCS, Combustion dynamics is sensed using a dynamic pressure sensing probe located within the combustion region of the turbine combustor, As shown, a continuous dynamics monitoring system (CDMS) 50 is provided and communicates with the DCS. The CDMS 50 preferably

uses either direct mounted or wave guide connected pressure or light sensing probes to measure the combustion dynamics. Another relevant operational parameter is the fuel gas temperature, Again, this temperature information is directed to the tuning controller 10 through the DCS 20 from the fuel heating controller 60. Since part of the tuning operation may include adjustment of the fuel temperature, there may be a two-way communication between the tuning controller 10 and the fuel heating unit 60. The DCS 20 communicates with a fuel blend ratio controller 70 to adjust the ratio of pipeline quality gas to non-pipeline quality gas (for subsequent consumption within the gas turbine). There exists in direct communication between the fuel blend ratio controller 70 and the tuning controller 10 via the DCS 20. Last, as part of this disclosure, certain key operating parameters of the HRSG 80 are sent to the tuning controller 30 via the DCS 20, If the tuning controller 10 determines that various parameters of the HRSG 80 are outside of allowable ranges, changes to the gas turbine f/a ratio are sent from the tuning controller 10 through the DCS 20 to the gas turbine controller 30.

[0036] Relevant operational data from the turbine and HRSG is collected several times per minute. This data collection allows for near real-time system tuning. Most relevant turbine and HRSG operational data is collected by the tuning controller 10 in near real-time. However, the turbine exhaust emissions sensor means is typically received by the tuning controller 10 with a 2 to 8 minute time lag from current operating conditions, This time lag necessitates the need for the tuning controller 10 to receive and buffer relevant information, for a similar time lag, before making operational tuning adjustments, This tuning controller 10 tuning adjustment time lag assures that all of the operational (including exhaust emissions) data is representative of a stable turbine operation before and after any adjustments have been made. Once the data is deemed stable, the tuning controller 10 determines whether there is a need for adjustment of tuning parameters. If no adjustment is necessary, the tuning controller 10 maintains the current tuning and waits to receive the next data set. If changes are desired, tuning commences. First, HRSG operational data are compared against HRSG component mechanical limits, If any HRSG mechanical limits (or margin against such limits) are violated, the tuning controller 10 will alter the gas turbine f/a ratio through the DCS 20 to the turbine controller 30, Subsequently, if there is sufficient margin in the key operational characteristics of the gas turbine (namely exhaust emissions and combustor dynamics), the tuning controller 10 can send a command (if applicable) to the fuel gas ratio controller 70 (through the DCS 20) to increase the ratio of non-pipeline quality gas to pipeline quality gas,

[0037] All determinations of the need for turbine tuning are performed within the tuning controller 10, The tuning operation is started based on an "alarm" created by receipt of operational data outside of preset operational criteria. In order for the tuning operation to be initiated, the alarm - and thus the data anomaly - must continue for a predetermined period of time.

[0038] One example of a tuning adjustment is the variation of the fuel nozzle pressure ratio to adjust combustion dynamics, With the requirement of higher firing temperatures to achieve greater flame temperatures and efficiency, turbine combustors must release more energy in a given combustor volume, Better exhaust emissions are often achieved by increasing the mixing

rate of fuel and air upstream of the combustion reaction zone, The increased mixing rate is often achieved by increasing the pressure drop at the fuel nozzle discharge, As the mixing rate increases in combustors, the turbulence generated by combustion often leads to noise within the combustor and may lead to the generation of acoustic waves. Typically, acoustic waves are caused when the sound waves of the combustion flames are coupled with the acoustic characteristics of the combustor volume or the fuel system itself,

[0039] Acoustic waves may affect the internal pressure in the chamber, Where pressure near a fuel nozzle rises, the rate of fuel flowing through the nozzle and the accompanying pressure drop decreases, Alternatively, a decrease in pressure near the nozzle will cause an increase in fuel flow, In cases where a low fuel nozzle pressure drop allows fuel flow oscillation, a combustor may experience amplified pressure oscillations, To combat the pressure oscillations within the combustor, combustion dynamics are monitored and the fuel air ratio and fuel nozzle pressure ratio may be modified to reduce or eliminate unwanted variations in combustor pressure, thereby curing an alarm situation or bringing the combustion system back to an acceptable level of combustion dynamics,

[0040] As shown in Fig. 2, the data received from the sensing means for the HRSG operational parameters (80), combustor dynamics (50), turbine exhaust emissions (40), and other relevant turbine operating parameters (30) are directed through the DCS (20) to the tuning controller (10). These input values are then compared to standard or target operational data for the turbine, The stored operational standards are based, at least in part, on the operational priority settings for the turbine, These priority settings are defined on the main user interface 12 of the tuning controller 10 and are shown graphically in Fig. 3, Based on the priority settings, a series of adjustments are made to the operation of the turbine by the turbine controller 10 connected through the DCS 20. The adjustments are directed to the control means, including the fuel heating unit 60 (Fig. 1), fuel blend ratio controller 70, and various other operational elements 90 of the turbine (Fig, 2).

[0041] The interface display 12 shown in Fig. 3 is comprised of switches (each having an On/Off indication), These switches allow the user to specify the desired tuning priorities for the operation of the turbine, The switched operational priorities include optimum NOx emissions 14, optimum power 16, optimum combustor dynamics 17, optimum fuel blend ratio 18, and optimum HRSG life 19, Each of these switches is set by the user to adjust the preferred operation of the turbine, Within the tuning controller are functions that operate within the priorities set by the switches, Preferably, if both the optimum NOx emissions switch 12 and the optimum power switch 14 are set to "On", the controller 10 will run in the optimum NOx mode, not optimum power. Thus, to run in optimum power mode, the optimum NOx emissions switch 12 must be "Off". Optimum dynamics 17 can be selected at any time. It is explicitly noted that other User-Interface Toggle Switches (not shown) may be used, including parameters such as Optimum Heat Rate, Optimum CO emissions, Optimum Heat Recovery Steam Generator (HRSG) Life, Optimum Gas Turbine Fuel Blend Ratio, Optimal Gas Turbine Turndown Capability, etc.

[0042] Fig. 4 shows a graphical representation of the interrelationship of the interface display switches. As shown, switching one parameter "On" will alter the alarm limits to a different level than their "Off" level. In the example shown in Fig. 4, the alarm limits are shown with both Optimum NOx and optimum power in the "On" position and in the "Off" position. These points on the graph are then modified by the selection of optimum dynamics (represented throughout by the symbol δ) in either the "On" or "Off" position. The points shown on the graph of Fig. 4 represent an exemplary set of limits for dynamics, based on the user's selected operational priorities,

[0043] Returning to Fig. 2, there is shown a representation of the logical flow of the determinations and calculations made within the tuning controller 10. The tuning controller 10 receives the actual operating parameters of the turbine through the turbine controller 30, combustor dynamics through the CDMS 50, turbine exhaust emissions through the CEMS 40, and relevant HRSG operating parameters 80. This sensor data is directed to the tuning controller 10 through the DCS 20. The received sensor data is compared to stored operational standards to determine if the turbine operation is conforming to the desired settings. The operational standards are based on the preset operational priorities of the turbine, defined by the switches 14, 16, 17, 18, and 19 on the main user interface display 12 of the tuning controller 10 (Fig. 3).

[0044] Based on the preset operational priorities, a hard-coded hierarchical Boolean-logic approach determines the dominant tuning criteria based on operational priorities. From this logical selection, the tuning controller 10 implements a fixed incremental adjustment value for changing an operational parameter of the turbine within a maximum range of adjustment (e.g., high and low values). The tuning changes are made in a consistent, pre-determined direction over a pre-determined increment of time and are dependent on the dominant tuning criteria at present. It is contemplated that no formulaic or functional calculations are made to determine the magnitude of tuning adjustments; rather, the incremental adjustments, the direction of the adjustments, the time span between adjustments, and the maximum range for the adjustments for each parameter and for each tuning criteria are stored in the tuning controller 10.

[0045] As shown in Fig. 2, when Optimum HRSG Life 19 is not selected by the operator, the tuning controller 10 determines whether the emissions are in compliance 100 and whether the combustor dynamics are at acceptable levels 102. If both are in compliance with the set operational standards, the tuning controller 10 waits for the next data set from the CEMS 40 or the CDMS 50, and for other turbine operational data 90. If both are in compliance with the set operational standards and possess sufficient operational margin, and Optimum Fuel Blend Ratio 18 is selected, the tuning controller 10 will send a command to the fuel blend ratio controller 70 to increase the ratio of non-pipeline quality gas to pipeline quality gas. If the received data is non-conforming with the operational standards, i.e. above or below alarm levels, as is the case with step 104 of Fig. 2, the tuning operation moves to the next tuning step of first determining the dominant tuning concern 106. The logical adjustment of turbine operation is defined by the dominant tuning criteria 106, which is based, at least in part, on the preset operational priorities set within the user interface 12, as will be discussed below with

respect to Fig. 8.

[0046] If Optimum HRSG Life 19 is selected by the operator, the first decision the tuning controller makes is an assessment of the margin of pertinent HRSG parameters (including but not limited to high pressure superheat outlet steam temperature, hot reheat outlet steam temperature, high pressure superheat steam desuperheater margin against saturation (degrees Fahrenheit of temperature immediately downstream of attemperator compared to saturation temperature), hot reheat steam desuperheater margin against saturation) against design limits. These temperature margins are compared against allowable margins as defined by the user. If the actual temperature margins are less than the allowable margins, the tuning controller 10 will automatically adjust the turbine controller's f/a ratio 122. In this particular case, the tuning controller 10 has first adjusted the gas turbine's f/a ratio for an external reason (HRSG component life). This change can adversely affect the gas turbine's current state-of-tune. Therefore, the normal gas turbine tuning scheme is performed by the tuning controller 10; however, changes to the turbine's f/a ratio are not allowed. The remaining gas turbine tuning scheme is defined below,

[0047] In a preferred operation, the tuning controller 10 will first attempt to change the turbine combustor fuel splits 108. The fuel split determines the distribution of the fuel flow to the fuel nozzles in each combustor. It should be noted that while the current embodiment indicates the presence of two adjustable fuel circuits, this approach can be utilized for one, two or more fuel circuits. If these adjustments do not resolve the tuning issue and do not place the operational data back into conformance with the operational standards, a further adjustment is performed. In certain situations or if the efficacy of fuel split changes on resolving high combustor dynamics is low, the next incremental adjustment is a change of the fuel gas temperature set point. In this adjustment step, the tuning controller 10 sends a modified fuel gas inlet temperature signal to the DCS 20, which is directed to the fuel heating unit 60.

[0048] Referring again to Fig. 2, if modification of the combustor fuel splits and/or fuel gas inlet temperature does not resolve the tuning issue 110, the tuning controller 10 will then alter the overall fuel/air ratio 112. This approach makes changes to the turbine thermal cycle utilizing fixed incremental changes over pre-determined amounts of time. The step is intended to adjust the exhaust temperature (up or down) by adjusting the air to fuel ratio in accordance with predetermined, standard control curves for the turbine operation, which are maintained within the memory of the tuning controller 10. If changes made to the gas turbine's overall fuel/air ratio do not resolve the tuning issue 114 or if Optimum HRSG Life 19 is enabled and HRSG mechanical concerns exist, the tuning controller 10 will adjust the fuel blend ratio 116.

[0049] In the present disclosure, it is contemplated that all control changes directed by the tuning controller 10 are fed back to the turbine system (30, 90), fuel gas temperature controller 60, and fuel blend ratio controller 70 through the DCS 20. However, the tuning controller 10 can be configured to communicate directly with the turbine controller 30. These changes are implemented directly within the various controller means within the system or through the turbine controller 30. When the operational data is returned to the desired operational

standards, the tuning settings are held in place by the tuning controller pending an alarm resulting from non-conforming data received from the sensor means through the DCS.

[0050] The adjustments sent from the tuning controller 10 to the turbine controller 30 or the associated controller (60, 70) means are preferably fixed in magnitude, Thus, the adjustments are not recalculated with new data or optimized to a target, The adjustments are part of an "open loop", Once started, the adjustments move incrementally to the preset maximum or maximum within a specified range, unless an interim adjustment places the operation data into conformance with the operational standards, Under most circumstances, when the full incremental range for one operational parameter is completed, the tuning controller moves on to the next operational parameter, which is defined by the preset operational priorities, The specific order of operational control elements is not fixed, and can be determined by operational priorities, The logic of the tuning controller drives the operational control element adjustment based on a "look-up" table stored within the memory of the tuning controller and preset operational priorities.

[0051] The tuning controller preferably addresses one operational parameter at a time. For example, the dominant tuning criteria dictates the first adjustment to be made. In the preferred example discussed above, the fuel distribution/split parameter is first adjusted. As indicated in Fig. 2, the fuel split of fuel circuit 1 is first addressed, followed by the split for fuel circuit 2, Again, this method can be applied to any combustion system with one or more adjustable fuel circuits, The fuel gas inlet temperature adjustment generally follows the fuel split adjustments when needed, Within each step, there is an incremental adjustment, followed by a time lag to permit the adjusted turbine operation to stabilize. After the time lag, if the current operational data analyzed by the tuning controller indicates that turbine operation still remains outside of the operational standards, the next incremental adjustment within the step is made. This pattern repeats for each step, Under most circumstances, only when one adjustment step is completed does the tuning controller move onto the next operational parameter, It should be noted that there exists an over-riding loop whereby the tuning controller 10 will directly increase the non-pipeline quality gas blend ratio (through the fuel blend ratio controller 70) if key turbine operational characteristics possess ample operational margin (against alarm conditions) 118. The control methodology of this over-riding control loop is identical to that mentioned above for fuel splits and turbine f/a ratio - a change is made in a pre-defined direction, a pre-defined amount, in a pre-defined amount of time.

[0052] The tuning controller preferably controls combustion operation to maintain proper tuning in variable conditions of ambient temperature, humidity and pressure, all of which vary over time and have a significant effect on turbine operation, The tuning controller may also maintain the tuning of the turbine during variation in fuel composition. Variation in fuel composition may cause a change in the heat release, which can lead to unacceptable emissions, unstable combustion, or even blow out, The tuning controller will adjust the fuel composition entering the turbine indirectly through changes in the fuel blend ratio 116.

[0053] Another aspect regarding fuel splits within the combustor deals directly with combustion

systems having a series of outer nozzles (of the same type, controlled by an outer fuel split affecting circumferential distribution of fuel within the outer fuel nozzles) in combination with a center nozzle (of same or different type compared to outer nozzles, controlled by an inner / center fuel split). Within this framework, the center nozzle can either operate with a "rich" or "lean" fuel-to-air ratio, as compared to the f/a of the outer fuel nozzles. Most combustion tuning keeps the combustion system either on a "lean center nozzle" or a "rich center nozzle" mode of operation. In some circumstances, better flame stability can be achieved with a "rich center nozzle" fuel split profile when compared to a "lean center nozzle"; however, this normally results in higher NO_x emissions. Therefore, a hybrid fuel schedule is of particular interest, whereby the combustion system utilizes a "lean center nozzle" fuel split schedule at higher load conditions (where flame stability is less of a concern but NO_x emissions are more of a concern) transitioning to a "rich center nozzle" fuel split schedule at lower load and turndown conditions (where flame stability is more of a concern and NO_x is less of a concern). The system of the present disclosure determines which fuel split schedule is employed (rich or lean center nozzle) at each operating point (allowing for the use of a hybrid fuel split schedule), and adjusts the fuel split schedule (Fuel Circuit 1 Split and Fuel Circuit 2 Split) in the proper direction. Again, changes made are of fixed magnitude in fixed time intervals.

[0054] Another point regarding fuel splits deals directly with combustion systems having a series of outer nozzles (of same or different type, controlled by an outer fuel split affecting circumferential distribution of fuel within the outer fuel nozzles) with no center nozzle. Within this framework, a subset of these outer nozzles (referred to generally as minor circuit 1) can either operate with a "rich" or "lean" fuel-to-air ratio, as compared to the f/a of the remaining outer fuel nozzles (referred to as major circuit 1). Most combustion tuning keeps the combustion system either on a "lean minor circuit 1" or a "rich minor circuit 1" circumferential fuel split mode of operation. In some circumstances, better flame stability can be achieved with a "lean minor circuit 1" fuel split profile when compared to a "rich minor circuit 1" fuel split profile; however, this can result in higher NO_x emissions. Therefore, a hybrid fuel schedule is of particular interest, whereby the combustion system may utilize a "rich minor circuit 1" fuel split schedule at higher load conditions (where flame stability is less of a concern but NO_x emissions are more of a concern) transitioning to a "lean minor circuit 1" fuel split schedule at lower load and turndown conditions (where flame stability / CO is more of a concern and NO_x is less of a concern). The system of the present disclosure determines which fuel split schedule is employed (rich or lean minor circuit 1) at each operating point (allowing for the use of a hybrid fuel split schedule), and adjusts the fuel split schedule (Fuel Circuit 1 Split and Fuel Circuit 2 Split, if applicable) in the proper direction. Again, changes made are of fixed magnitude in fixed time intervals.

[0055] One further aspect regarding fuel splits deals directly with combustion systems having one or more annular rings of fuel nozzles (of same or different type, controlled by an circumferential fuel split affecting circumferential distribution of fuel within each ring of fuel nozzles), whereby a second family of fuel splits may be available (if more than one annular ring of fuel nozzles exists) which adjusts the relative (radial) amount of fuel to each of the radially concentric fuel rings (ring 1, ring 2, etc.), Within this framework, a subset of each ring's fuel

nozzles (referred to as minor circuit ring 1, minor circuit ring 2, etc.) can either operate with a "rich" or "lean" fuel-to-air ratio, as compared to the f/a of the remaining ring's fuel nozzles (referred to as major circuit ring 1, major circuit ring 2, etc.). Most combustion tuning keeps the combustion system either on a "lean minor circuit ring 1" or a "rich minor circuit ring 1" (and similar approaches for rings 2, 3, etc.) circumferential fuel split mode of operation. In some circumstances, better flame stability can be achieved with, using ring 1 as an example, a "lean minor circuit ring 1" fuel split profile when compared to a "rich minor circuit ring 1" fuel split profile; however, this can result in higher NO_x emissions. Therefore, a hybrid fuel schedule is of particular interest, whereby the combustion system may utilize a "rich minor circuit ring 1" fuel split schedule at higher load conditions (where flame stability is less of a concern but NO_x emissions are more of a concern) transitioning to a "lean minor circuit ring 1" fuel split schedule at lower load and turndown conditions (where flame stability / CO is more of a concern and NO_x is less of a concern). The system of the present disclosure determines which fuel split schedule is employed for each ring, if applicable (rich or lean minor circuit ring 1, rich or lean minor circuit ring 2, etc.) at each operating point (allowing for the use of a hybrid fuel split schedule), and adjusts the fuel split schedule (Fuel Circuit 1 Split and Fuel Circuit 2 Split, if applicable) in the proper direction. Again, changes made are of fixed magnitude in fixed time intervals.

[0056] Fig. 5 provides a schematic that details the framework for determining the dominant tuning concern 106, as included in Fig. 2. Future steps will be described below with respect to Fig. 8. First, relevant emissions parameters 120 and combustor dynamics 122 are received by the tuning controller 10 from the CEMS 40 and CDMS 50, as detailed above. The relevant emissions parameters 120 and combustor dynamics 122 are then compared to allowable tuning limits 124 that are also provided to the tuning controller 10. The allowable tuning limits are in the form of preset ranges that may be adjusted using the tuning interface 12 of Fig. 3 and determined according to the logic set forth below with respect to Figs. 6 and 7. The output of this comparison is a series of "True" alarms 126 of various tuning concerns, where an alarm condition is indicated if the sensed operational data 120, 122 is above or below a given alarm range set forth in the tuning limits 124.

[0057] Alarm conditions may have more than one level or tier. For example, there may be varying degrees of severity of an alarm, such as: high "H"; high-high "HH"; high-high-high "HHH" and low "L"; low-low "L"; low-low-low "LLL". The "True" logical alarms 126 are subsequently ranked according to their level of importance (e.g. high - high "HH" alarms are more important than high "H" alarms, etc.) in step 130. If more than one tuning concern shares the same level, the tuning concerns will then be ranked according to the user preferences as set forth below with respect to Fig. 8. If only one "True" alarm emerges, this will be selected and used as the dominant tuning concern 106 to initiate the tuning process as set forth in Fig. 2. However, the results of the process of Fig. 5, namely the ranked "True" alarms 130, will be processed through user determined criteria, as shown in Fig. 8, before a dominant tuning concern 106 is confirmed,

[0058] In Fig. 6, a flow chart is provided to explain how the allowable tuning limits 124 are

determined, Once determined, the tuning limits 124 are compared to the operational data 120, 122 as set forth above and shown in Fig. 5, First, the User Interface Toggle Switches 14, 16, 17 corresponding to those in the interface display 12 of Fig. 3, are compared against each other, utilizing an internal hierarchy to allow passage of the alarm constraints relative to the most significant toggle switch, Thus, depending on which switches are in the "On" position, different tuning limits will be included in the allowable tuning limits 124, Each of Optimum NOx, Optimum Power and Optimum Dynamics has a collection of preset limits (denoted by the numerals 134, 136 and 138 in Fig, 6), depending on whether the corresponding toggle switch 14, 16, 17 is in the "On" or "Off" position, There is also an internal set of default limits 140 to be used when none of the toggle switches are in the "On" position,

[0059] The internal hierarchy will determine which tuning limits shall take precedence in the event that competing toggle switches 14, 16 or 17 are in the "On" position. In the present example, the hierarchy ranks Optimum NOx 14 above Optimum Power 16, Optimum Dynamics 17 may be selected at any time and will simply alter the tuning limits of the other selections given, such as is shown in Fig, 4, If Optimum NOx 14 and Optimum Power 16 are both in the "On" position, the tuning limits for Optimum NOx 134 will be used, Additionally, the tuning limits for Optimum Dynamics 138 are utilized if this toggle switch 17 is activated, If no User Interface Toggle Switches 14, 16, 17 are active, default tuning limits 140 are provided as the allowable tuning limits 124, All of the tuning limits 134, 136, 138 and 140 that may be used to construct the allowable tuning limits for the tuning controller 10 may be developed by the end user and programmers and are then preferably hard coded into the tuning controller 10 for a given application, The methodology outlined in Fig, 6 is meant to provide an exemplary framework for incorporation of a number of different User Interface Toggle Switches, such as those options set forth above with respect to Fig. 3 including Optimum HRSG Life 19, whereby only a subset are specifically outlined in this disclosure,

[0060] Fig, 7 shows a specific example of the flow chart of Fig. 6 given for the determination of a subset of the system's allowable tuning limits. In this example, the tuning limits for High NOx, High HighNOx, High Class 1 δP 's, High Class 2 δP 's will be determined based on preset tuning limits and the user's preferences, The various exemplary tuning limits are provided for Optimum NOx 134, Optimum Power 136, Optimum Dynamics 138, and No Optimal Settings 140 are given corresponding numerical values (shown respectively in blocks 152, 154, 156 and 158). The corresponding numerical values given for each criterion vary, such that the allowable limits 124 will be different depending on which toggle switches 14, 16 or 17 are selected. By way of example, the Optimum NOx 134, 152 and Optimum Power 136, 154 give limits for NOx, but also provide limits for Dynamics that are to be used in the event that Optimum Dynamics 138, 156 is not selected. However, in the event that the Optimum Dynamics toggle 17 is selected, the Class 1 δP 's and Class 2 δP 's values provided therefore 156 shall be used instead of the values listed with respect to Optimum NOx 134, 152 and Optimum Power 136, 154.

[0061] In this particular example, the toggle switches for Optimum NOx 14 and Optimum Dynamics 17 are selected, with the switch for Optimum Power 16 left in the "Off" position. Thus,

the values from Optimum NOx for High NOx and High HighNOx 152 are provided, Also, because Optimum Dynamics 17 is also selected, the Dynamics values for High Class 1 δP 's and High Class 2 δP 's 138, 156 replace those δP 's values provided with respect to Optimum NOx 134, 152. As a result, the allowable tuning limits 124 are provided as shown in block 160. These allowable tuning limits 124 correspond to those used in Fig, 5, as described above, to determine whether information from the CEMS 40 and CDMS 50 is in an alarm state or operating normally,

[0062] Fig. 8, shows a further schematic for the process of incorporating a user's priorities and the "True" alarm conditions received for determining the dominant tuning concern 106. It is this tuning concern 106 which dictates the turbine operational changes the turbine controller 10 performs, as shown in Fig. 2.

[0063] First, a determination is made of all potential dominant tuning issues 142. These include, but are not limited to: combustor blowout, CO emissions, NOx emissions, Class 1 combustor dynamics (Class 1 δP 's), Class 2 combustor dynamics (Class 2 δP 's), and HRSG mechanical life, The list of potential dominant tuning issues 142 is determined by the user and programmer and may be based on a number of factors or operational criteria. By way of example, Class 1 and Class 2 combustor dynamics δP 's refer to combustion dynamics occurring over specific ranges of acoustic frequencies, whereby the range of frequencies is different between Classes 1 and 2, Indeed, many combustion systems can possess different acoustic resonant frequencies corresponding to Class 1 and Class 2, and variations in these 2 dynamics classes may be mitigated utilizing different turbine operational parameter changes for each different turbine and / or combustor arrangement. It should also be noted that certain combustion systems may have none, 1, 2, or greater than 2 different "classes" (frequency ranges) of combustor dynamics which can be tuned, This disclosure utilizes a system whereby 2 different combustor dynamics classes are mentioned, However, it is fully intended that this disclosure can be broadly applied to any number of distinct dynamics frequency classes (from 0 to greater than 2).

[0064] After determination of the potential dominant tuning issues 142, these issues are ranked in order of significance 144 according to the end user's needs as well as the detrimental effects that each tuning concern can have on the environment and / or turbine performance, The relative importance of each potential dominant tuning concern can be different with each end user, and for each combustor arrangement, For example, some combustion systems will demonstrate an extreme sensitivity to combustor dynamics, such that normal daily operational parameter variations can cause a normally benign dynamics tuning concern to become catastrophic in a very short amount of time, In this case, one or both of the dominant dynamics tuning concerns (Class 1 and Class 2) may be elevated to Priority 1 (Most Important). By way of example in Fig. 7, combustor blowout is listed as the most important Dominant Tuning Concern 144. This ranking is used to determine the dominant tuning concern in the event that there are multiple alarms with equal levels of severity, This ranking of Dominant Tuning Concerns 144, from most to least important, provides the overall framework where the specific Boolean Logic Hierarchy 148 is created, For example, assuming Class 1

and Class 2 δP 's obey generally monotonic behavior relative to perturbations in system operational parameters, a High-High "HH" Class 2 δP 's alarm may be more significant than High "H" Class 1 δP 's alarm. Additionally, in the example given in Fig. 8 for the Boolean Logic Hierarchy 148, High "H" NOx emissions is more significant than High "H" Class 2 dynamics, This means that if both High "H" NOx and High "H" Class 2 dynamics are both "in alarm" (Logic = True), in the absence of other alarms being "True", the autotuning system will tune for High "H" NOx because it is the dominant tuning concern, Finally, it can be seen that Blowout is ranked above NOx Emissions and both are ranked above Class 1 δP 's. Thus, if there were high "H" alarms returned for all three categories, Blowout would be the dominant tuning concern, followed by NOx Emissions and then Class 1 δP 's. This Boolean Logic Hierarchy 148 will be what is compared to the "True" alarms 130 returned by comparing the allowable tuning limits 124 to the operational data 120, 122 as set forth above with respect to Fig. 5.

[0065] All "True" tuning alarms 130 are provided as ranked by severity (e.g. HHH above HH, etc.), The "True" tuning alarms 130 are then compared with the hard-coded Boolean Logic Hierarchy 148, in step 150 to determine which tuning will become the "True" Dominant Tuning Concern 106. This one "True" Dominant Tuning Concern 106 is now passed into the remainder of the autotuning algorithm, as detailed in Fig. 2, as the Dominant Tuning Concern 106 to be mitigated by operational changes,

[0066] Thus the tuning controller 10 may be configured to optimize the mechanical life of a Heat Recovery Steam Generator (HRSG) through the manipulation of the gas turbine exhaust temperature (turbine fuel air (f/a) ratio) when key HRSG operational characteristics indicate insufficient design margin against over-temperature (at steam outlet conditions) and / or over-attenuation (at intra-stage desuperheater), The HRSG will have sensors for measuring operational parameters of the boiler, and the turbine having sensor means for measuring operational parameters of the turbine, as discussed above, The HRSG operational parameters include high pressure and/or hot reheat steam outlet temperatures and/or high pressure and/or hot reheat intra-stage desuperheater outlet temperatures and pressures, The turbine operational parameters include combustor dynamics and turbine exhaust emissions. Using the logic applied above with respect to alarm levels, the turbine controller will adjust various operational control elements, such as fuel distribution and / or fuel-to-air (f/a) ratio, as needed,

[0067] The control system for optimizing HRSG life is set forth above in Fig. 1 and (optionally) relies on the turbine controller 10 communicating through the DCS20 to the sensor means and the control means listed above to control the operational control elements of the turbine. In order to maximize HRSG life in light of other potential operational priorities, a user will select operational priorities for HRSG and / or other turbine operation, selected from the group comprising optimum NOx emissions, optimum power output, optimum combustor dynamics, optimum HRSG life, and / or optimum fuel blend ratio (ratio of non-pipeline quality gas to pipeline quality gas). The below examples will be provided for instances where the Optimum HRSG Life 19 is selected in the control panel shown in Fig. 3, such that Optimum HRSG Life is an operating priority, potentially in addition to other selected priorities,

[0068] During operation, the turbine controller will receive operational data from the gas turbine sensor means and the HRSG sensor means. The operational data will be compared to stored operational standards, based on the selected operational priorities. Using this comparison, the turbine controller will determine if both the HRSG and gas turbine operation conform to the operational standards,

[0069] To the extent that either the HRSG or gas turbine operational parameters are not within allowable limits, the tuning controller 10 will determine the dominant tuning criteria for non-conforming operation of the HRSG and / or the gas turbine, based on the preset operational priorities. With the dominant tuning criteria determined, the turbine controller 10 will communicate with the selected operational control elements to perform a selected adjustment in the operational control element of the gas turbine. The operational control element may be combustor fuel distribution split within the nozzles of the combustor, fuel gas inlet temperature, fuel/air ratio within the turbine, and / or gas fuel blend ratio (fuel composition). The adjustment to the operational control element will be based on the dominant tuning criteria and have a fixed incremental value and defined range, each incremental change input over a set period of time sufficient for the turbine to gain operational stability,

[0070] The sensing process will be repeated in open loop fashion, such that the turbine controller will subsequently receive further data regarding the operational parameters from the HRSG and gas turbine sensor means upon passage of a set period of time to determine if an additional incremental change is desired. If additional tuning is required, further incremental adjustments will be made to the operational control element within a defined range. To the extent that the range of available adjustments to a particular control element are exhausted, the tuning controller 10 will select a further operational control element adjustment based on the dominant tuning criteria, the further selected adjustment having a fixed incremental value and defined range, with each incremental adjustment made over a set period of time sufficient for the turbine to gain operational stability. The sensing and adjustment (if needed) process will be continued during the operation of the turbine and HRSG,

[0071] In one embodiment, the system can be programmed with stored operational data, such that the mechanical life of an HRSG is optimized in the tuning process by first adjusting the fuel-to-air ratio of the gas turbine in increments to change the HRSG hot gas inlet conditions to provide sufficient design margin in key HRSG operational parameters (i.e. lowering or raising the temperature of the hot gas inlet). The tuning can then continue as may be required as a result of these changes made to the turbine's f/a ratio. For example, the HRSG may be provided with sensor means for measuring operational parameters of the associated boiler, including high pressure and/or hot reheat outlet steam temperatures as well as high pressure and/or hot reheat intra-stage desuperheater outlet temperatures and pressures. The gas turbine having will also have sensor means for measuring operational parameters of the turbine, including stack emissions and combustion dynamics from the turbine and control means for various operational elements of the turbine, including fuel distribution and / or fuel temperature and / or fuel blend ratio and / or fuel-to-air ratio. Optionally, the tuning controller 10, various sensor means and control means may either be connected directly or via a

distributed control system (DCS). The control system may also be provided with means for setting operational priorities for turbine operation, selected from the group comprising optimum NOx emissions, optimum power output, optimum combustor dynamics, optimum fuel blend ratio, and / or optimum HRSG life, such as that shown in Fig. 3 discussed above. Using this tuning system, operational priorities are selected, operational parameters are sensed and tuning occurs using the methods discussed herein with respect to optimizing HRSG life and turbine operation, provided that in this instance, f/a ratio is the predetermined first operational control element to be adjusted to optimize HRSG life, while other operational control elements may be adjusted in order to keep the combustion turbine within the allowable limits for each operational parameter.

[0072] A method of optimizing the mechanical life of an HRSG through tuning the operation of a gas turbine, is now disclosed using the systems described herein. The method first includes establishing a communication link between the turbine controller 10 and (optionally) the DCS 20 and receiving data from the HRSG and / or gas turbine sensor means regarding the status of various operational parameters of the HRSG and the turbine, The operational parameter values are then compared to set of standard data to determine if adjustment to operational control elements is required in order to bring the operation of the turbine or HRSG into allowable limits, If tuning is needed, the tuning controller will communicate with the selected operational control elements to perform a defined incremental adjustment of the selected control element. The system then receives, at the tuning controller, via the sensor means and DCS operational parameter data regarding the operation of both the HRSG and the turbine from the sensor means and determines if the adjustment conforms turbine operation to a set standard or if a further incremental adjustment is desired,

[0073] The sensed data from the HRSG may include steam outlet temperatures and / or steam superheater intra-stage attemperator over-saturation conditions, The operational control element that is adjusted to modify the values of these sensed parameters may be the fuel-to-air ratio of the turbine. Once the HRSG values are within allowable limits, if further tuning is required to bring the operation of the turbine within its allowable limits, This will be done according to the tuning methods described above, preferably without further modification to the f/a ratio, such that the operational control elements of fuel gas temperature, fuel splits or fuel blend ratio.

[0074] The tuning of the system may be adapted for method for tuning a premixed combustion system wherein there exists two distinct modes of operation, The turbine being tuned (not shown) may have an outer ring of identical fuel nozzles utilizing an outer nozzle fuel split to modulate the circumferential fuel distribution within these outer nozzles, an inner fuel nozzle which utilizes an inner nozzle fuel split to adjust the fuel-to-air ratio of the inner to outer nozzles, The outer an inner nozzles discussed herein are known to those skilled in the art and not specifically recounted herein, The two distinct modes of operation comprise a "lean" inner nozzle mode whereby the f/a ratio of the inner nozzle is less than the f/a ratio of the outer fuel nozzles, and a "rich" inner nozzle whereby the f/a ratio of the inner nozzle is greater than the f/a ratio of the outer fuel nozzles. The method for tuning a system having these distinct modes

comprises selecting, at the tuning controller 10 of a hybrid fuel split schedule for varying modes based on the turbine load, The "lean" center nozzle fuel split schedule will be at higher load conditions and the "rich" fuel split schedule will be used at lower load and turndown conditions, where the turbine is being operated at the lowest level possible in order to maintain operation of the HRSG.

[0075] The method may include the steps disclosed above along with making, at the tuning controller 10, a determination of whether the current mode of operation is utilizing either a "lean" or "rich" inner nozzle mode of operation, and selecting the direction of adjustment for Fuel Circuit Split 1 and / or Fuel Circuit Split 2 when a tuning issue exists, depending on which mode of operation is being utilized at the current operating conditions and preset operational parameters stored within the turbine controller,

[0076] All of the methods provided above can be conducted using Boolean-logic toggle switches, such as those shown in Fig, 3, to select user-desired optimization criteria, One of the optimization criteria is Optimum HRSG Life, whereby toggling of this switch to a "1" ("TRUE") allows the tuning controller to improve HRSG mechanical operating margins, such as steam outlet temperature and / or steam superheater intra-stage attemperator saturation temperature margin, through changes in the HRSG inlet conditions via modifications to the gas turbine fuel-to-air ratio, These changes may be made using the methods and systems disclosed above.

[0077] A method for tuning a premixed combustion system is also provided whereby there exists an outer ring of fuel nozzles utilizing an outer nozzle fuel split to modulate the circumferential fuel distribution within these outer nozzles, utilizing two modes of operation: a "lean minor circuit 1" subset of the outer nozzles whereby the f/a ratio of this outer fuel nozzle subset is less than the f/a ratio of the remaining outer fuel nozzles, and a "rich minor circuit 1" inner nozzle whereby the f/a ratio of this outer fuel nozzle subset is greater than the f/a ratio of the remaining outer fuel nozzles, The method includes usage of a hybrid fuel split schedule, with a "rich minor circuit 1" fuel split schedule at higher load conditions, and usage to a "lean minor circuit 1" fuel split schedule at lower load and turndown conditions, The method may also include varying other operational control elements, as described herein, in order to bring the operation of the turbine or HRSG into allowable limits,

[0078] The method may also include usage of a hybrid fuel split schedule, with a "lean minor circuit 1" fuel split schedule at higher load conditions, and usage to a "rich minor circuit 1" fuel split schedule at lower load and turndown conditions, Further, the method may include making a determination, at the tuning controller 10, if the current mode of operation is utilizing either a "lean minor circuit 1" or "rich minor circuit 1" mode of operation, and adjusting, using operational control elements for fuel splits, the Fuel Circuit Split 1 and / or Fuel Circuit Split 2 in the proper direction, when a tuning issue exists, depending on which mode of operation is being utilized at the current operating conditions,

[0079] A method is also provided for tuning a premixed combustion system, using similar systems and steps as described above, whereby there exists one or more annular rings of fuel

nozzles (of same or different type, controlled by an circumferential fuel split affecting circumferential distribution of fuel within each fuel nozzle ring), In the current system, it is contemplated that a second family of fuel splits may be available (if more than one annular ring of fuel nozzles exists), which adjusts the relative (radial) amount of fuel to each of the radially concentric fuel nozzle rings (ring 1, ring 2, etc.), utilizing two modes of operation: a "lean minor circuit ring 1" subset of the ring 1 fuel nozzles whereby the f/a ratio of this ring 1 fuel nozzle subset is less than the f/a ratio of the remaining fuel nozzles of ring 1, and a "rich minor circuit ring 1" subset of the ring 1 fuel nozzles whereby the f/a ratio of this outer fuel nozzle subset is greater than the f/a ratio of the remaining fuel nozzles of ring 1, The tuning method comprises , use of a hybrid fuel split schedule, with a "lean minor circuit ring 1" fuel split schedule at higher load conditions, usage of a "rich minor circuit ring 1" fuel split schedule at lower load and turndown conditions, and similar usage of "rich" and "lean" fuel split schedules, one at high loads and the other at lower load / turndown conditions, for each of the remaining fuel nozzle rings of the combustion system. Each of the fuel schedules may be pre-programmed into the tuning controller 10 and selected based on sensed operational parameters of the system,

[0080] The method may also be modified, depending on operational priorities and user input to the tuning controller 10, to include usage of a hybrid fuel split schedule, with a "rich minor circuit ring 1" fuel split schedule at higher load conditions, usage to a "lean minor circuit ring 1" fuel split schedule at lower load and turndown conditions, and similar usage of "rich" and "lean" fuel split schedules, one at high loads and the other at lower load / turndown conditions, for each of the remaining fuel nozzle rings of the combustion system,

[0081] A method is also provided for tuning a premixed combustion system, such as that disclosed above, whereby there exists one or more annular rings of fuel nozzles (of same or different type, controlled by an circumferential fuel split affecting circumferential distribution of fuel within each fuel nozzle ring), whereby a second family of fuel splits may be available (if more than one annular ring of fuel nozzles exists) which adjusts the relative (radial) amount of fuel to each of the radially concentric fuel nozzle rings (ring 1, ring 2, etc.), utilizing two modes of operation: a "lean minor circuit ring 1" subset of the ring 1 fuel nozzles whereby the f/a ratio of this ring 1 fuel nozzle subset is less than the f/a ratio of the remaining fuel nozzles of ring 1, and a "rich minor circuit ring 1" subset of the ring 1 fuel nozzles whereby the f/a ratio of this outer fuel nozzle subset is greater than the f/a ratio of the remaining fuel nozzles of ring 1, The method first comprises the step of determining if the current mode of operation is utilizing either a "lean minor circuit ring 1" or "rich minor circuit ring 1" mode of operation, making a similar determination of the current mode of operation, either "rich" or "lean" minor fuel circuit operation, for each of the remaining fuel nozzle rings of the combustion system, Once these determinations are made, the method comprises the step of adjusting, via the tuning controller 10 and selected operational control element, Fuel Circuit Split 1 and/or Fuel Circuit Split 2 in the proper direction, when a tuning issue exists. The direction of adjustment is determined based on which mode of operation is being utilized at the current operating conditions.

[0082] Figs, 9-12 provide exemplary visual representations of the autotuning system interface depicting how the Boolean Logic Hierarchy works in practice, Fig. 9 shows the alarms returned

in connection with the example set forth above with respect to Fig. 8, Namely, alarms are returned for Class 2 δP 's at the levels of H 162, HH 164 and HHH 166, In addition, alarms for NOx 168 and Class 1 δP 's 170 are returned at the H level. Since more extreme levels trump conflicts of different alarms at the same level, the HHH Class 2 δP 's is the priority and therefore the dominant tuning concern 172,

[0083] Figs. 10-12 show various further examples of the dominant tuning concern for different "True" alarm levels under the user defined hierarchy 144 of Fig. 8. Fig. 10 shows high pressure steam at maximum operating temperature and the high pressure steam desuperheater at saturation conditions (placing water into the steam pipe), with no other alarms active, Thus, HRSG Mechanical Life Optimization is the dominant tuning concern, Fig. 11 shows a Class 2 δP 's at an H level, with NOx at both an H and HH condition, thus making High NOx as the dominant tuning concern, Finally, Fig. 12 shows both Class 1 δP 's and Class 2 δP 's at the H level. Referring to the user ranking of dominant tuning issues 144 in Fig. 8, Class 1 δP 's is ranked as a priority above Class 2 δP 's and thus, although the severity of the alarms is equal, Class 1 δP 's becomes the dominant tuning concern,

[0084] In Figs. 13 -16, there is shown various examples of the operational results of a tuning operation of a tuning controller of the present disclosure based on operational data from a running turbine system, In Fig. 13, the dominant tuning concern is high Class 2 δP 's, and a change in the combustor fuel split E1 is made in reaction to a high Class 2 δP 's alarm generated when the combustor dynamics moves outside of the set operational priorities for optimum dynamics. The actual combustor dynamics data received by the turbine controller 10 from, for example, the CDMS 50 is designated as 200 in the graph, The moving average for the combustor dynamics is identified in the graph as 202. When the combustor dynamics exceed the dynamics alarm limit value 204 for a set period of time TA an alarm goes off within the tuning controller, This alarm causes the first event E1 and a resulting incremental adjustment in the combustor fuel split tuning parameter 206, As illustrated, the incremental increase in the fuel split causes a corresponding drop in the combustor dynamics 200, with the average combustor dynamics 202 dropping below the dynamics alarm limit 204. As time continues, the tuning is held by the tuning controller and the average combustor dynamics 202 maintains its operational position below the dynamics limit 204, Thus, no further adjustments necessary or alarms issued,

[0085] In Fig. 14, the tuning criteria is High NOx emissions, As NOx emissions data 210 is received from the tuning controller, an alarm is generated after the passage of time TA, The alarm is caused by the NOx emissions 210 exceeding the operational standard or tuning limit 212. The alarm activates a first event E1 resulting in an incremental increase in the fuel split 214. After a period of time TB from the first event E1, the NOx alarm is still activated due to the NOx emissions 210 exceeding the preset tuning limit 212, This continued alarm after time TB causes a second event E2 and a second incremental increase in the fuel split value 214. This second increase is equal in magnitude to the first incremental increase, The second event E2 causes the NOx emissions level 210 to drop below the preset limit 212 within the review time period and halts the alarm, As the NOx emissions 210 remains below the limit 212, the fuel

split 214 tuning is held and the operation of the turbine continues with the defined operational parameters,

[0086] In Fig. 15, the tuning criteria is Blowout, with the alarm created by a low NOx reading received by tuning controller, As shown, the NOx tuning limit 220 is defined. Upon passage of the set time period TA from receiving NOx level data 222, the alarm is generated and a first event E1 occurs, At the first event E1, the fuel split level 224 is incrementally adjusted downward, After a set passage of time TB from event E1 additional NOx emissions data 222 is received and compared to the preset alarm level 220, Because the NOx is still below the alarm level 220, a second event E2 occurs resulting in a further incremental reduction in the fuel split value 224. A further passage of time TC from event E2 occurs and additional data is received. Again, the NOx data 222 is low, maintaining the alarm and resulting in a further event E3. At event E3, the fuel split value 224 is again reduced by the same incremental amount, This third incremental adjustment results in the NOx emissions 222 rising above the preset limit 220 and results in removal of the alarm, The fuel split 224 tuning value set after event E3 is held in place by the tuning controller 10.

[0087] In Fig, 16, the tuning criteria is again Blowout, whereby the NOx emissions data 230 received by the tuning controller 10 is again tracking along the lower emissions limit 232. At the first tuning event E1, the fuel split value 234 is incrementally dropped to result in a corresponding increase in the NOx emissions 230 over the lower limit 232, After this first incremental adjustment, the NOx emissions for a period of time holds above the limit 232 and then begins to again fall. At the second tuning event E2, the fuel split value 234 is again adjusted by the designated fixed incremental value. This second adjustment then places the fuel split value 234 at its defined minimum within the preset range of allowable values (determined as a hard coded limit within the tuning controller 10), Because this limit is reached, the tuning operation moves to the next operational parameter, which is normally the second fuel circuit adjustment, In the example provided, this second circuit value (not shown) is already at its set maximum/minimum and is therefore not adjusted, Thus, the tuning operation moves on to the next operational parameter, load control curves 236. As shown, at event E2 an incremental adjustment is made in the load control curve value 236, The increase in the load control curve value 236 results in a corresponding increase in the NOx emission 230 to a value above the minimum 232 and removes the alarm, Upon removal of the alarm, the tuning settings are held and no further adjustments are made, The tuning controller 10 then proceeds to receive data from the sensor means, through the DCS, and continues to make comparisons with the set operational standards (including the minimum NOx emissions limit EL).

[0088] The present invention has been described and illustrated with respect to a number of exemplary embodiments thereof, It should be understood by those skilled in the art from the foregoing that various other changes, omissions and additions may be made therein, without departing from the scope of the present invention, with the scope of the present invention being described by the foregoing claims,

REFERENCES CITED IN THE DESCRIPTION

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P A T E N T K R A V

1. Afstemningssystem til at optimere den mekaniske levetid af en varmegenvindingsdampgenerator, hvor varmegenvindingsdampgeneratoren drives i forbindelse med en forbrændingsturbin, k e n d e t e g n e t ved, at afstemningssystemet omfatter:

- 5 en afstemningsstyringsindretning (10);
- sensormiddel til at registrere driftsparametre af en turbine og en varmegenvindingsdampgenerator; og
- styringsmiddel til at styre diverse driftsstyringselementer,
- hvor afstemningsstyringsindretningen (10) er indrettet til at afstemme driften af gas-
- 10 turbinen i overensstemmelse med de følgende trin:
 - at specificere, af en bruger, driftsprioriteter for mindst én af varmegenvindingsdampgenerator- eller turbinedriften, hvor at specificere driftsprioriteterne bevirker en ændring af lagrede driftsstandarde for varmegenvindingsdampgeneratoren og turbinen,
 - 15 at modtage driftsparameterdata for driften af turbinen og varmegenvindingsdampgeneratoren fra sensormidlet, hvor driftsparameterdataet omfatter varmegenvindingsdampgeneratorparametre valgt fra gruppen bestående af udgangstemperatur af højtryksdamp, udgangstemperatur af varm genopvarmningsdamp, udgangstemperatur af højtryks-intratin-deoverheder og udgangstemperatur af varm genop-
 - 20 varmnings-intratin-deoverheder,
 - at sammenligne driftsdataet med de lagrede driftsstandarde baseret på de specificerede driftsprioriteter for at bestemme om turbinedriften er i overensstemmelse med driftsstandarde og om varmegenvindingsgeneratorparametrene er i overensstemmelse med tilladte marginer vedrørende overtemperatur ved dampud-
 - 25 gangsbetingelser og overtemperering ved intratin-deoverhederen,
 - at bestemme det dominerende afstemningskriterie (106) for afvigende drift af varmegenvindingsdampgeneratoren og turbinen, baseret på de forudindstillede driftsprioriteter,
 - at kommunikere med styringsmidlet for at udføre en valgt justering i et første
 - 30 driftsstyringselement af gasturbinen,
 - at modtage driftsdata fra sensormidlet når en indstillet tidsperiode er passeret, for at bestemme om en yderligere inkrementel ændring er ønsket, og at foretage yderligere justeringer af det valgte første driftsstyringselement,
 - at vælge en yderligere driftsparameterjustering, forskellig fra det første valgte
 - 35 driftsstyringselement, og
 - at modtage driftsdata fra sensormidlet når en indstillet tidsperiode er passeret, for at bestemme om en yderligere inkrementel ændring er ønsket.

2. System ifølge krav 1, yderligere omfattende et distribueret styringssystem, hvorved afstemningsstyringsindretningen (10) kommunikerer med sensormidlet og kontrolmid-

let ved hjælp af det distribuerede styringssystem.

3. System ifølge krav 1, hvor driftsprioriteterne er valgt fra gruppen bestående af optimale NO_x-emissioner, optimalt effektoutput, optimal forbrændingskammerdynamik, optimal varmegenvindingsdampgeneratorlevetid og optimalt brændstofblandingsforhold.

5 4. System ifølge krav 3, hvor det optimale brændstofblandingsforhold omfatter forholdet mellem gas af ikke-rørledningskvalitet og gas af rørledningskvalitet.

5. System ifølge krav 1, hvor den valgte driftsstyringselementjustering er baseret på det dominerende afstemningskriterie (106), hvor den valgte justering har en fast inkrementel værdi og et defineret område, hver inkrementelle ændring indføres over en fast
10 tidsperiode som er tilstrækkelig til at turbinen kan opnå driftsstabilitet.

6. System ifølge krav 1, hvor justeringerne i driftsstyringselementerne af gasturbinen er valgt fra gruppen omfattende forbrændingskammerbrændstofdistributionsfordeling indeni dyserne af forbrændingskammeret, brændstofgasindgangstemperatur, brændstof/luftforhold indeni turbinen og gasbrændstofblandingsforhold (brændstofsammensætning).

15 7. System ifølge krav 1, hvor driftsprioriteterne omfatter optimal varmegenvindingsdampgeneratorlevetid og hvor justering af brændstof-til-luft-forholdet i turbinen vælges som det første driftsstyringselement til at blive justeret når den valgte driftsprioritet er optimal varmegenvindingsdampgeneratorlevetid.

8. System ifølge krav 1, hvor afstemningsstyringsindretningen (10) er indrettet til at
20 optimere den mekaniske levetid af varmegenvindingsdampgeneratoren ved først at justere brændstof-til-luft-forholdsdriftsstyringselementet før der foretages eventuelle afstemningsjusteringer baseret på de øvrige driftsprioriteter.

9. System ifølge krav 8, hvor, efter justering af brændstof-til-luftforholdsdriftsstyringselementet, afstemmer afstemningsstyringsindretningen (10) derefter
25 yderligere styringselementer i respons på ændringer i driftsparametrene af turbinen som opstår som et resultat af justering af brændstof-til-luft-forholdsdriftsstyringselementet.

10. System ifølge krav 1, hvor driftsdataet er valgt fra gruppen yderligere bestående af udgangstryk af højtryks-intratin-deoverheder og udgangstryk af varm genopvarmningsintratin-deoverheder.

30 11. System ifølge krav 1, hvor turbinen har driftsparametre valgt fra gruppen bestående af turbineudstødningsemissioner og forbrændingsdynamik.

12. System til at styre driften af en forbrændingsturbin, hvor forbrændingsturbinen drives i forbindelse med en varmegenvindingsdampgenerator, k e n d e t e g n e t ved, at systemet omfatter:

35 en afstemningsstyringsindretning (10) til at lagre data relateret til driften af turbinen og varmegenvindingsdampgeneratoren, at specificere driftsprioriteter for mindst én af varmegenvindingsdampgenerator- eller turbinedriften, hvor at specificere driftsprioriteterne bevirker en ændring af lagrede driftsstandarde for varmegenvindingsdampgeneratoren og turbinen, at modtage driftsdata relateret til driften af turbinen og varmegenvindings-

dampgeneratoren, hvor driftsparameterdataet omfatter varmegenvindingsdampgeneratorparametre valgt fra gruppen bestående af udgangstemperatur af højtryksdamp, udgangstemperatur af varm genopvarmningsdamp, udgangstemperatur af højtryks-intratrindeoverheder og udgangstemperatur af varm genopvarmnings-intratrindeoverheder, og at
 5 tilvejebringe instruktionssignaler relateret til driften af turbinen og varmegenvindingsdampgeneratoren;

sensormiddel til at registrere data relateret til driften af turbinen og varmegenvindingsdampgeneratoren og at kommunikere den registrerede data til afstemningsstyringsindretningen (10), hvor afstemningsstyringsindretningen sammenligner driftsdataet med
 10 de lagrede driftsstandarter baseret på de specificerede driftsprioriteter for at bestemme om turbinedrift er i overensstemmelse med driftsstandarterne og om varmegenvindingsgeneratorparametrene er i overensstemmelse med tilladte marginer vedrørende overtemperatur ved dampudgangsbetingelser og overtemperering ved intratrindeoverhederen; og

styringsmiddel til at styre driftsstyringselementer af turbinen i respons på et instruktionssignal tilvejebragt af afstemningsstyringsindretningen (10).
 15

13. System ifølge krav 12, hvor driftsprioriteterne er valgt fra gruppen bestående af optimale NOx-emissioner, optimalt effektoutput, optimal forbrændingskammerdynamik, optimalt brændstofblandingsforhold og optimal varmegenvindingsdampgeneratorlevetid.

14. Fremgangsmåde til at optimere den mekaniske levetid af en varmegenvindingsdampgenerator gennem afstemning af driften af en gasturbine, k e n d e t e g n e t ved, at fremgangsmåden omfatter:
 20

at tilvejebringe et sensormiddel til at registrere driftsparametre af varmegenvindingsdampgeneratoren;

at tilvejebringe et styringsmiddel til at styre driftsstyringselementer af turbinen, hvor
 25 styringsmidlet justerer valgte styringselementer i respons på styringssignaler fra en afstemningsstyringsindretning (10);

at tilvejebringe afstemningsstyringsindretningen (10) til at specificere driftsprioriteter for mindst én af varmegenvindingsdampgenerator- eller turbinedriften, hvor at specificere driftsprioriteterne bevirker en ændring af lagrede driftsstandarter for varmegenvindingsdampgeneratoren og turbinen;
 30

at modtage data vedrørende driftsparametrene fra sensormidlet og at sende styringssignaler til styringsmidlet, hvor driftsparametrene omfatter varmegenvindingsdampgeneratorparametre valgt fra gruppen bestående af udgangstemperatur af højtryksdamp, udgangstemperatur af varm genopvarmningsdamp, udgangstemperatur af højtryksintratrindeoverheder og udgangstemperatur af varm genopvarmnings-intratrindeoverheder;
 35

at etablere en kommunikationsforbindelse mellem afstemningsstyringsindretningen (10), styringsmidlet og sensormidlet;

at registrere data vedrørende driftsparametrene af varmegenvindingsdampgenerato-

ren og turbinen og at transmittere de registrerede data til afstemningsstyringsindretningen (10);

at modtage de registrerede driftsparameterdata ved afstemningsstyringsindretningen (10) og at sammenligne de registrerede driftsparameterdata med de lagrede driftsstandar-
 5 der som er lagret i afstemningsstyringsindretningen (10), for at bestemme om turbinedrift er i overensstemmelse med driftsstandarderne og om varmegenvindingsgeneratorparamet-
 rene er i overensstemmelse med tilladte marginer vedrørende overtemperatur ved damp-
 udgangsbetingelser og overtemperering ved intratrin-deoverhederen og om justering af et
 driftsstyringselement er nødvendigt for at forbedre driftsparametre af turbinen eller var-
 10 megenvindingsdampgeneratoren, hvor de forudindstillede driftsparameterniveauer er base-
 ret på driftsprioriteter af turbinen;

at kommunikere, fra afstemningsindretningen (10) til styringsmidlet, styringssignaler til at udføre en defineret inkrementel justering af et første valgt driftsstyringselement,

efter en forudindstillet tidsperiode fra justeringen af det første valgte driftsstyrings-
 15 element, at registrere data vedrørende driftsparametre og at transmittere de registrerede data til afstemningsindretningen (10) og at sammenligne de registrerede driftsdata med de forudindstillet driftsparameterniveauer for at bestemme om yderligere justeringer i driftsstyringsmidlet er nødvendige,

at bestemme om det første driftsstyringselement er i stand til at modtage yderligere
 20 justeringer og at kommunikere, fra afstemningsstyringsindretningen (10) til styringsmidlet, styringssignaler til at udføre en defineret inkrementel justering af et andet valgt driftsstyringselement.

15. Fremgangsmåde ifølge krav 14, hvor det første valgte styringselement som skal justeres, er brændstof-til-luft-forholdet af turbinen, hvor justeringen til brændstof-til-luft-
 25 forholdet gøres i respons på at driftsparametre af varmegenvindingsgeneratoren er uden for tilladte grænser når de sammenlignes med lagrede driftsstandarder.

16. Fremgangsmåde til at afstemme et forblandet forbrændingssystem af en forbrændingsturbine som drives i forbindelse med en varmegenvindingsdampgenerator, hvor turbinen omfatter mindst ét forbrændingskammer som har en ydre ring af identiske
 30 brændstofdyser og en indre brændstofdyse som kan anvende en indre dysebrændstoffordeling til at justere brændstof-til-luft-forholdet af den indre til de yder dyser, k e n d e t e g n e t ved, at fremgangsmåden omfatter:

at tilvejebringe en turbinestyringsindretning (30) som er programmeret til at drive det forblandede forbrændingssystem i mindst to særskilte driftstilstande, hvor de særskilte
 35 tilstande omfatter en tynd indre dysetilstand hvor brændstof-til-luft-forholdet af den indre dyse er mindre end brændstof-til-luft-forholdet af de ydre dyser, og en rig indre dysetilstand hvor brændstof-til-luft-forholdet af den indre dyse er større end brændstof-til-luft-forholdet af de ydre dyser,

at registrere driftsbetingelserne af turbinen, hvor de registrerede betingelser indbe-

fatter højbelastning, lavbelastning og dellast; og

at drive det forblandede forbrændingssystem ved anvendelse af både den tynde indre dysetilstand og den rige indre dysetilstand ifølge driftsbetingelsen af turbinen; hvor ændring af driftstilstanden af turbinen fra tynd indre dysetilstand til rig indre dysetilstand

5 udføres ved:

at specificere driftsbetingelser for turbinedriften ud fra et område af parametre, hvor at specificere driftsprioriteterne bevirker en ændring af lagrede driftsstandarder for turbinen; og

at justere brændstoffordelingen af turbinen i respons på en afvigelse i driftsbetingelserne fra de lagrede driftsstandarder.

10

17. Fremgangsmåde ifølge krav 16, hvor det forblandede forbrændingssystem drives i en tynd indre dysetilstand under højbelastningsbetingelser og det forblandede forbrændingssystem drives i en rig indre dysetilstand under lavbelastnings- og dellastsbetingelser.

18. Fremgangsmåde ifølge krav 16, hvor det forblandede forbrændingssystem drives i en rig indre dysetilstand under højbelastningsbetingelser og det forblandede forbrændingssystem drives i en tynd indre dysetilstand under lavbelastnings- og dellastsbetingelser.

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19. Fremgangsmåde til at afstemme et forblandet forbrændingssystem af en forbrændingsturbin som drives i forbindelse med en varmegenvindingsdampgenerator, hvor turbinen omfatter mindst ét forbrændingskammer som har en ring af ydre brændstofdyser hvor brændstoffordelingen mellem valgte delmængder af de ydre brændstofdyser kan justeres individuelt, og en indre brændstofdyse, k e n d e t e g n e t ved, at fremgangsmåden omfatter:

20

at tilvejebringe en turbinestyringsindretning (30) som er programmeret til at drive det forblandede forbrændingssystem i mindst to særskilte driftstilstande, hvor den første driftstilstand omfatter en tynd mindre kreds, hvor brændstof-til-luft-forholdet af den ydre brændstofdysedelmængde er mindre end brændstof-til-luft-forholdet af de øvrige ydre brændstofdyser, og den andendriftstilstand omfatter en rig mindre kreds, hvor brændstof-til-luft-forholdet af den ydre brændstofdysedelmængde er større end brændstof-til-luft-forholdet af de øvrige ydre brændstofdyser;

25

30

at registrere driftsbetingelserne af turbinen, hvor de registrerede betingelser indbefatter højbelastning, lavbelastning og dellast; og

at drive det forblandede forbrændingssystem ved anvendelse af både den tynde indre dysetilstand og den rige indre dysetilstand ifølge driftsbetingelsen af turbinen; hvor ændring driftstilstanden af turbinen fra tynd indre dysetilstand til rig indre dysetilstand

35 udføres ved:

at specificere driftsbetingelser for turbinedriften ud fra et område af parametre, hvor at specificere driftsprioriteterne bevirker en ændring af lagrede driftsstandarder for turbinen; og

at justere brændstoffordelingen af turbinen i respons på en afvigelse i driftsbetingelserne fra de lagrede driftsstandarder.

20. Fremgangsmåde ifølge krav 19, hvor det forblandede forbrændingssystem drives i en tynd mindre kredstilstand under højbelastningsbetingelser og det forblandede forbrændingssystem drives i en rig mindre kredstilstand under lavbelastnings- og dellastsbetingelser.

21. Fremgangsmåde ifølge krav 19, hvor det forblandede forbrændingssystem drives i en rig mindre kredstilstand under højbelastningsbetingelser og det forblandede forbrændingssystem drives i en tynd mindre kredstilstand under lavbelastnings- og dellastbetingelser.

22. Fremgangsmåde ifølge krav 19, yderligere omfattende trinnene at bestemme om den aktuelle driftstilstand er en tynd mindre kreds eller en rig mindre kreds og justere den valgte delmængde af ydre brændstoffdyser i den rette retning når et afstemningsproblem findes, afhængig af hvilken driftstilstand anvendes ved de aktuelle driftsbetingelser.

23. Fremgangsmåde ifølge krav 19, hvor turbinen omfatter en flerhed af ringformede ringe af brændstoffdyser, hvor brændstoffordelingen til hver ring kan justeres baseret på driftsbetingelsen af en turbine, således at brændstof-til-luft-forholdet af en første ringformet ring af brændstoffdyser og en anden ringformet ring af brændstoffdyser kan justeres uafhængigt.

24. Fremgangsmåde ifølge krav 23, hvor den første ringformede ring af brændstoffdyser selektivt kan drives ved et højere brændstof-til-luft-forhold end den anden ringformede ring af brændstoffdyser.

DRAWINGS

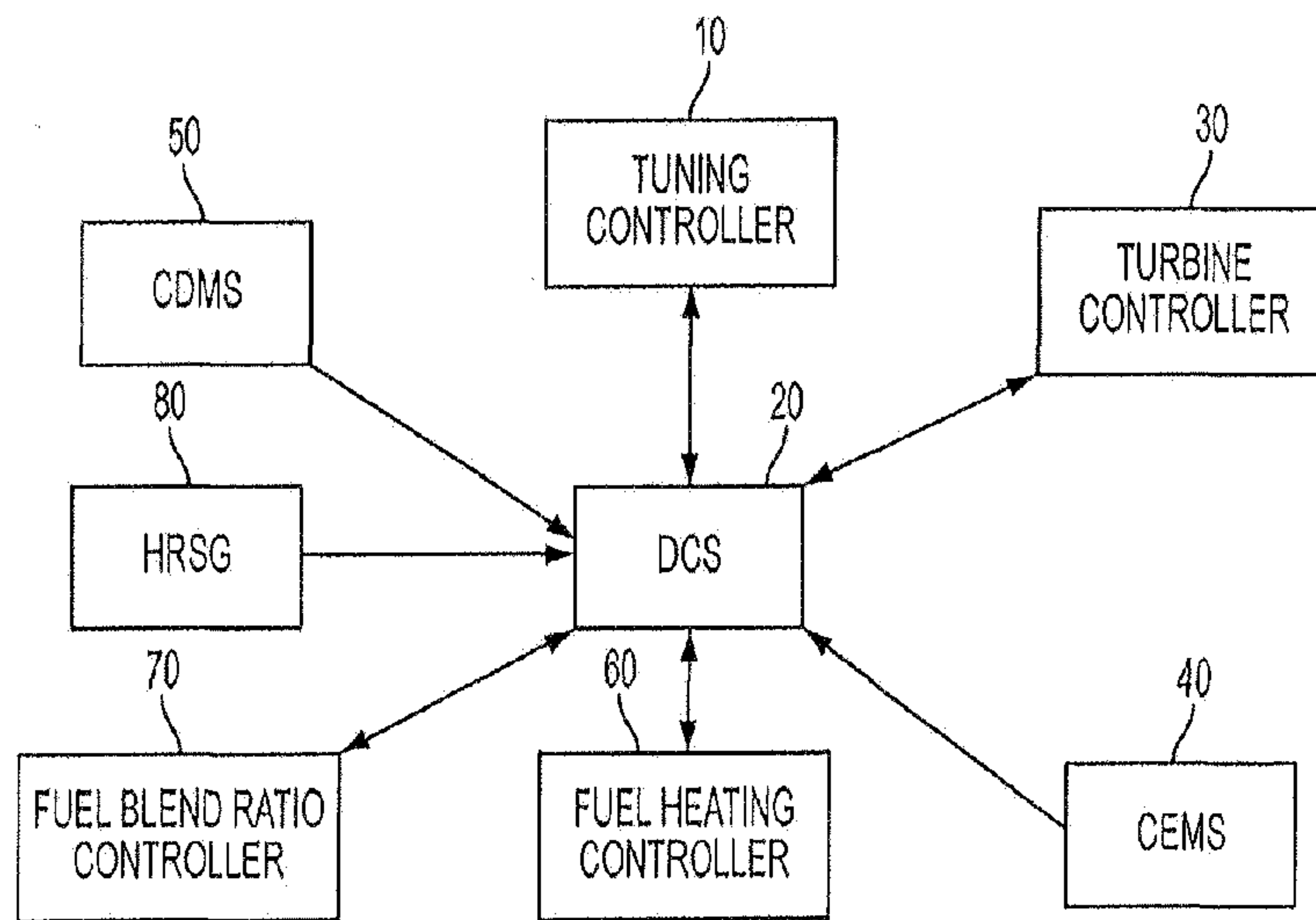
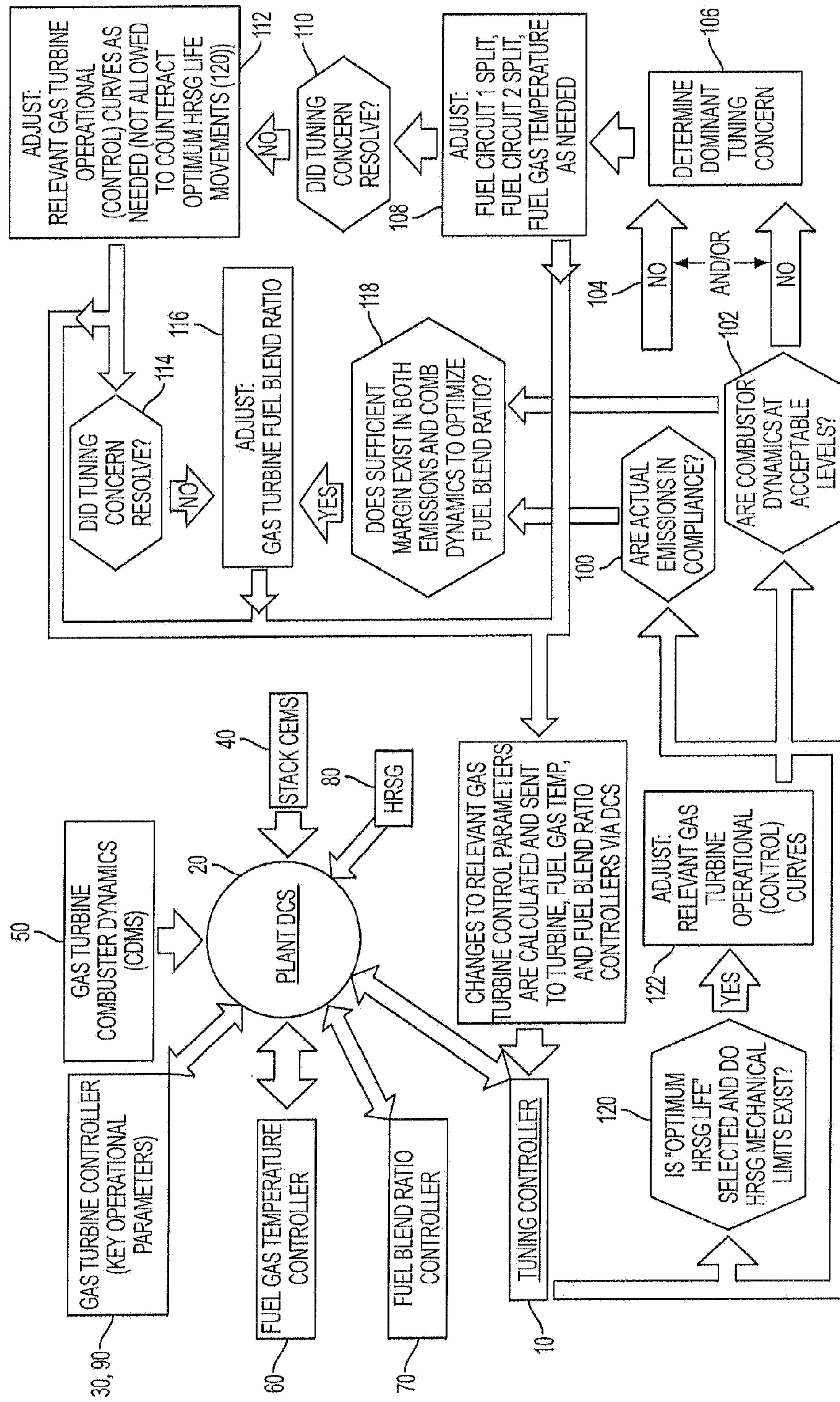


FIG. 1



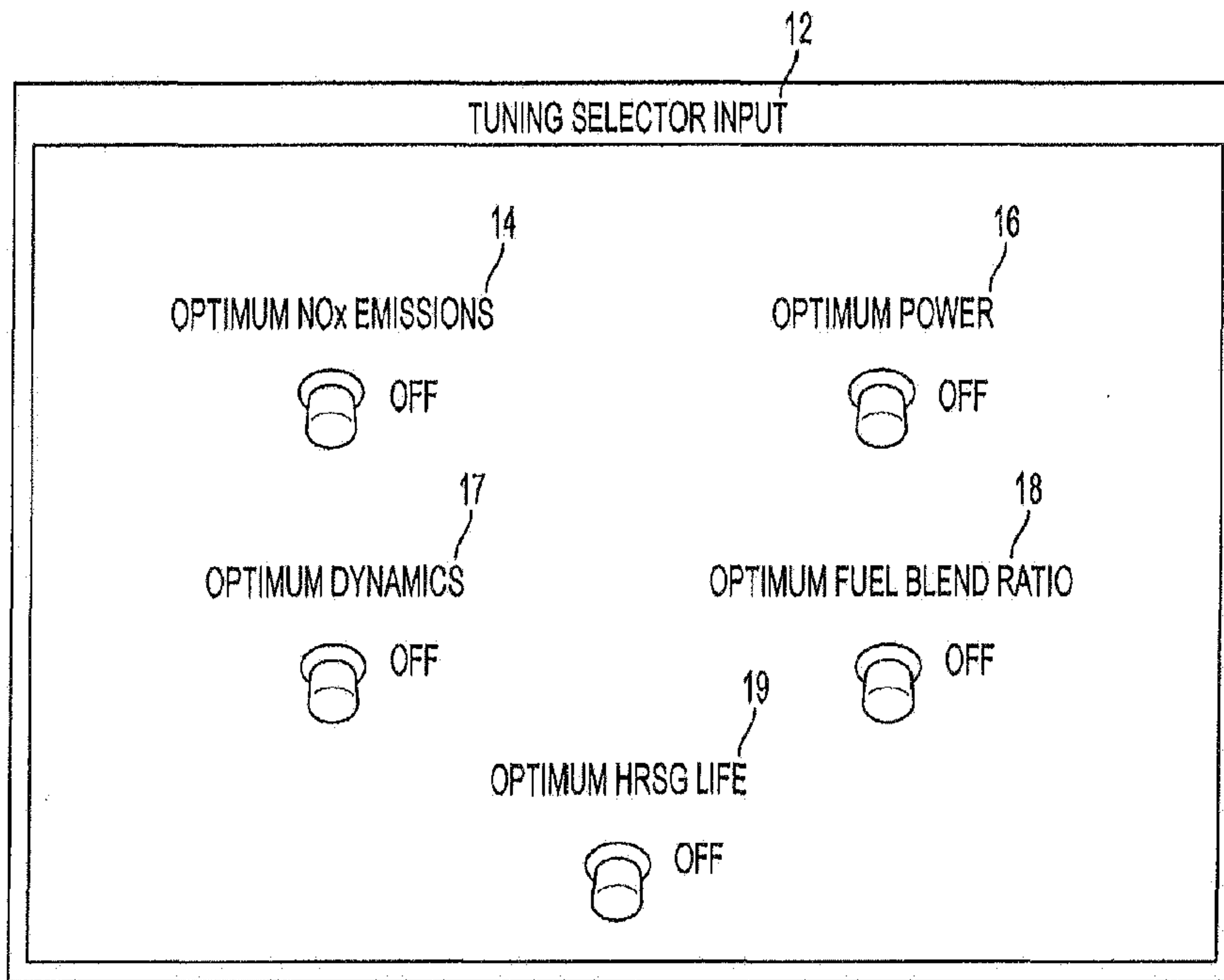


FIG. 3

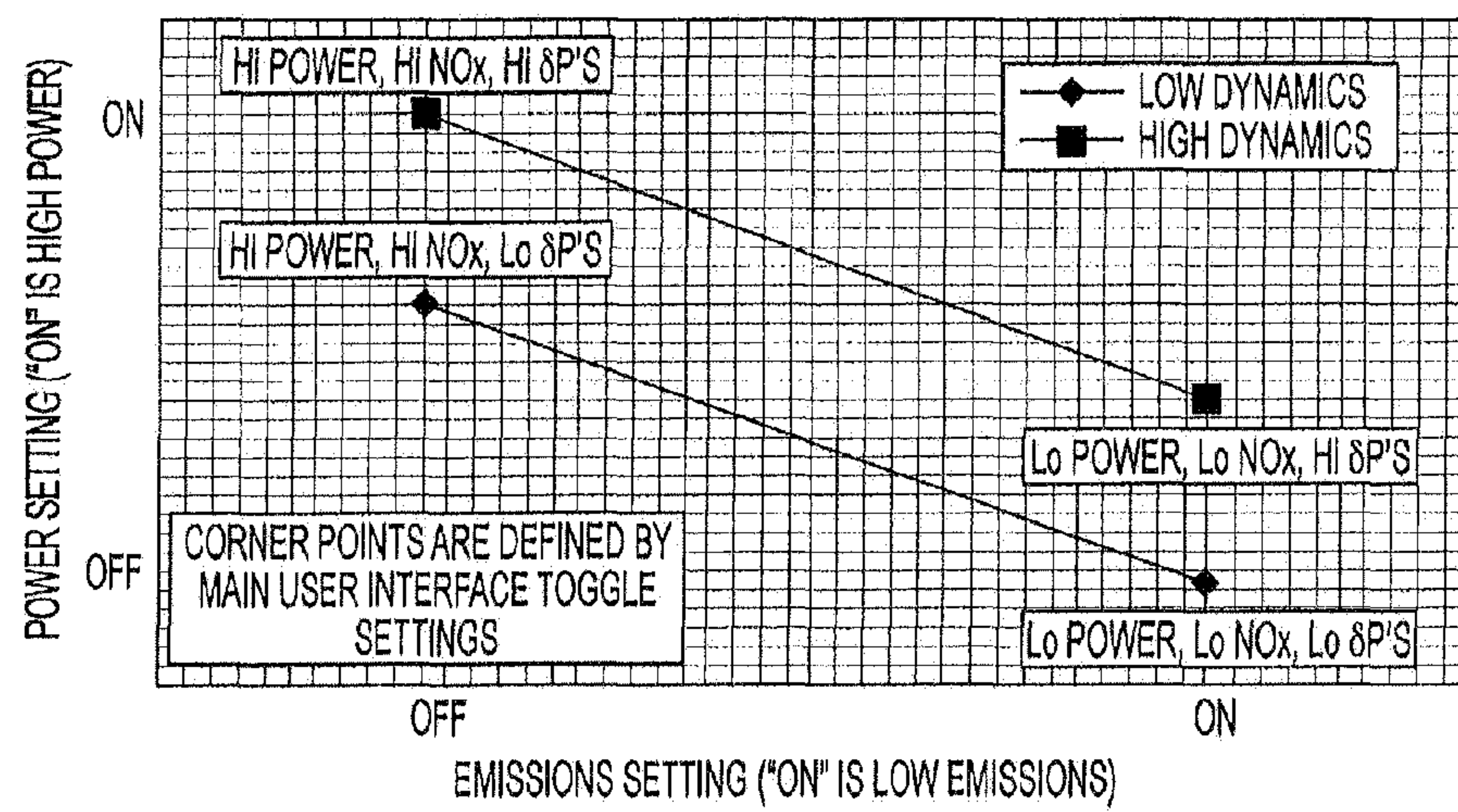


FIG. 4

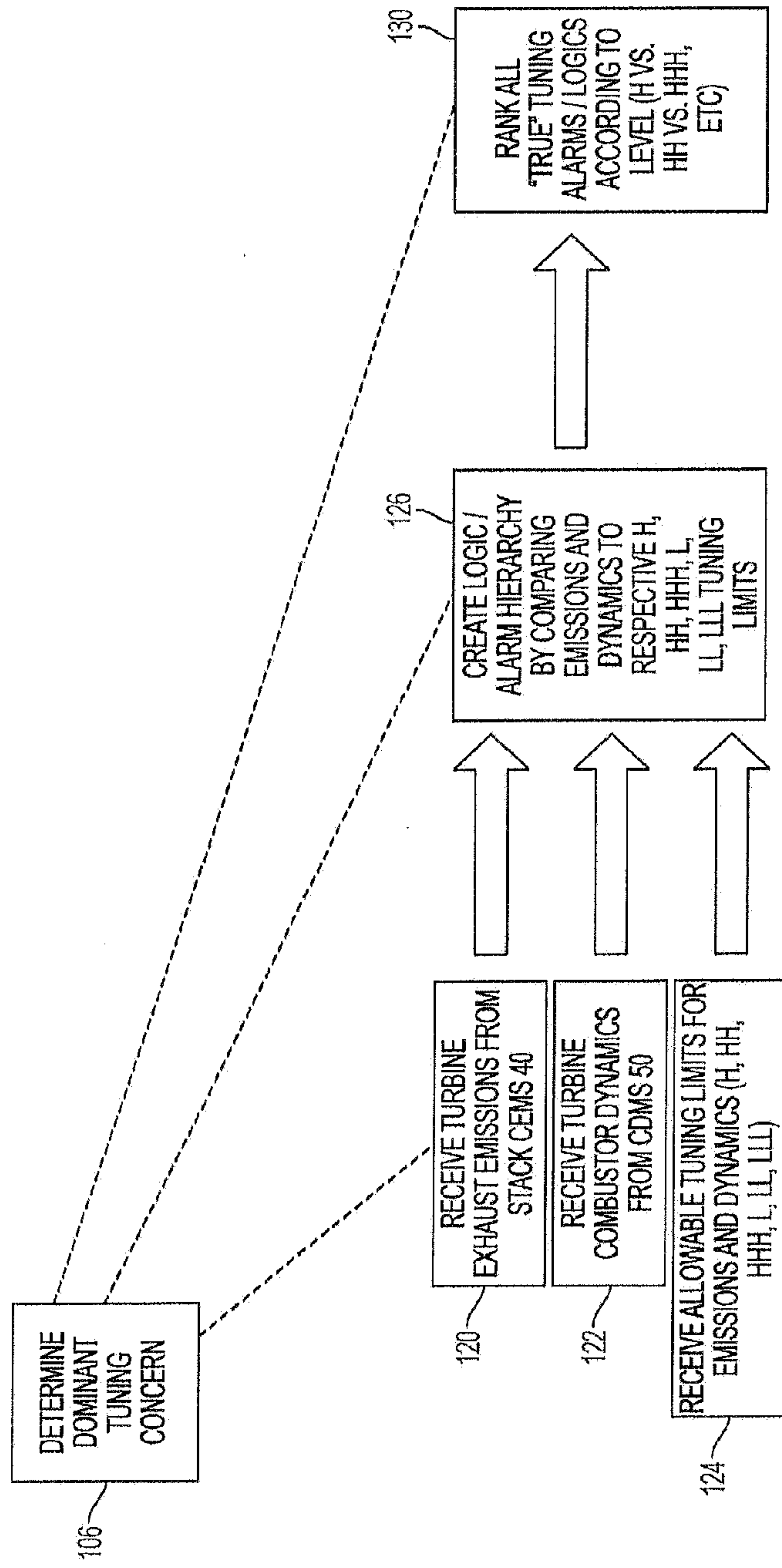


FIG. 5

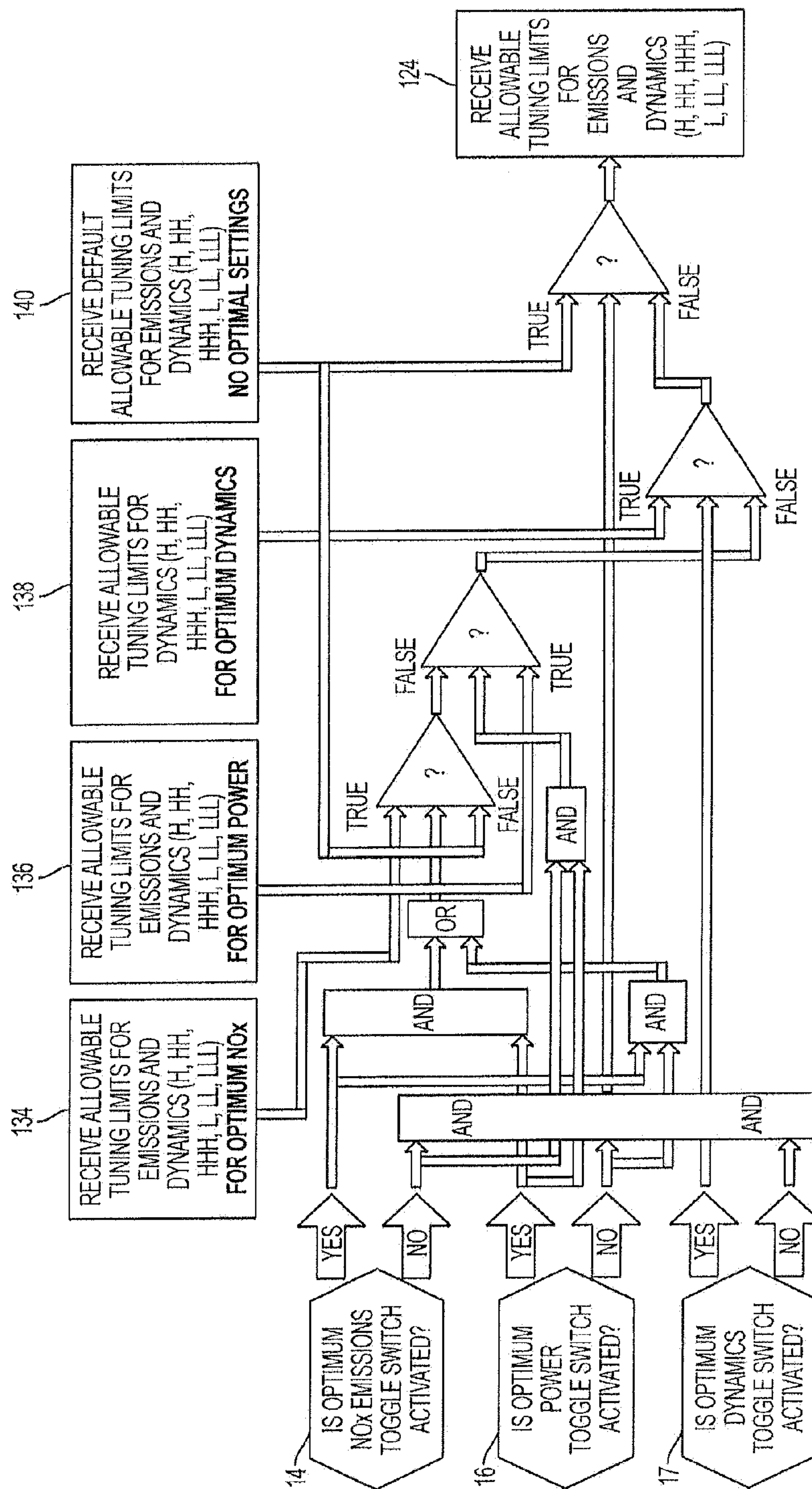


FIG. 6

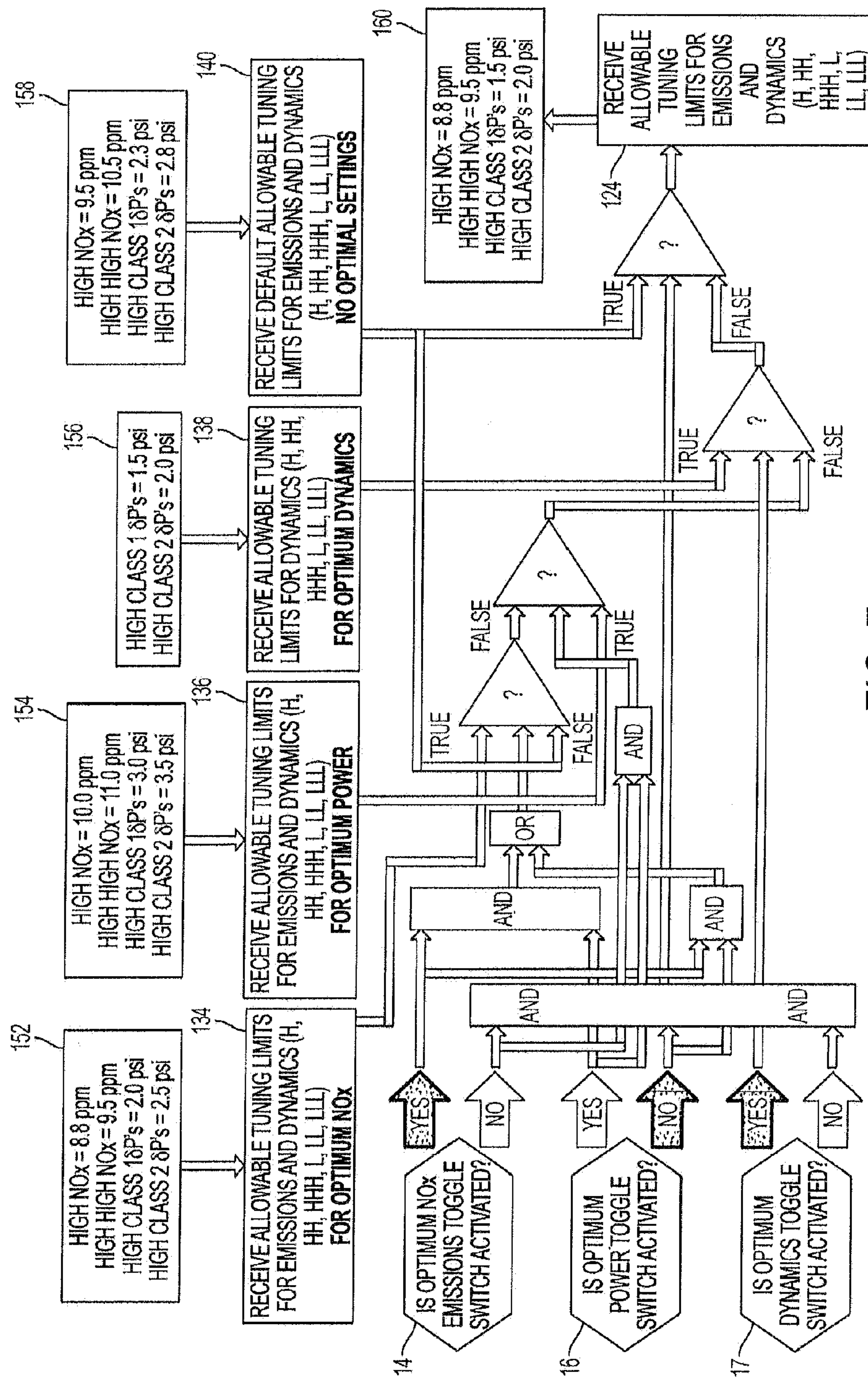


FIG. 7

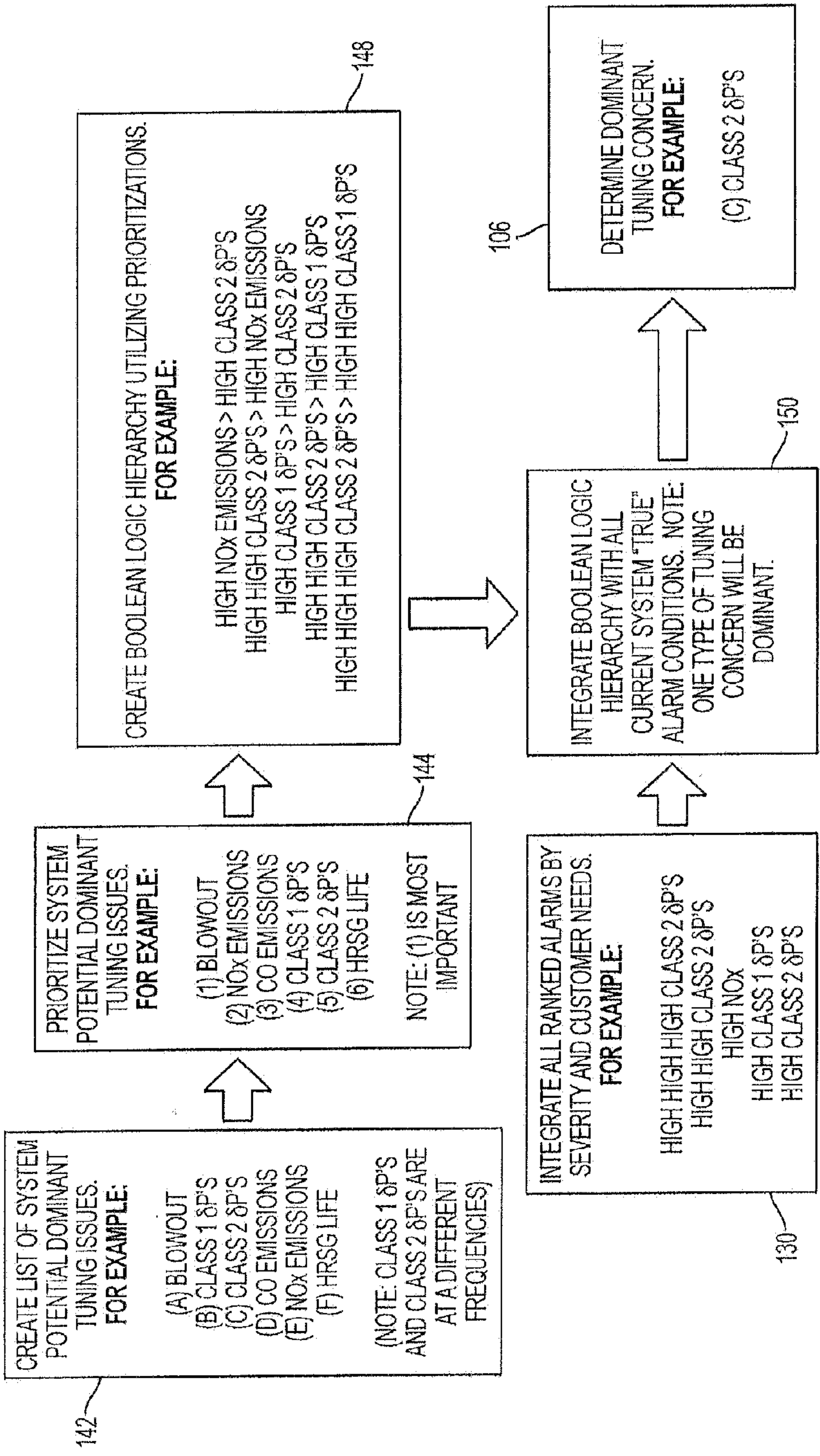


FIG. 8

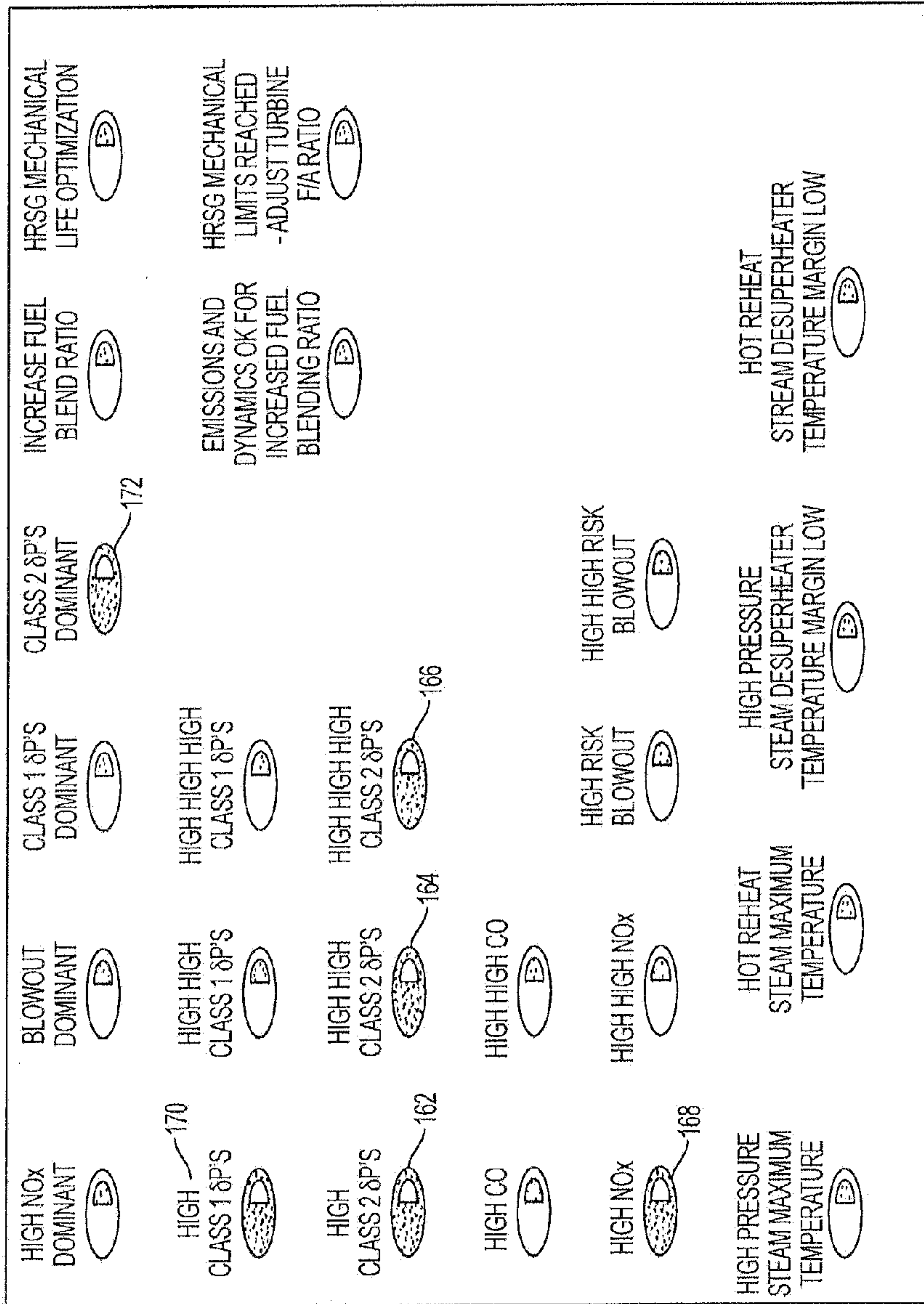


FIG. 9

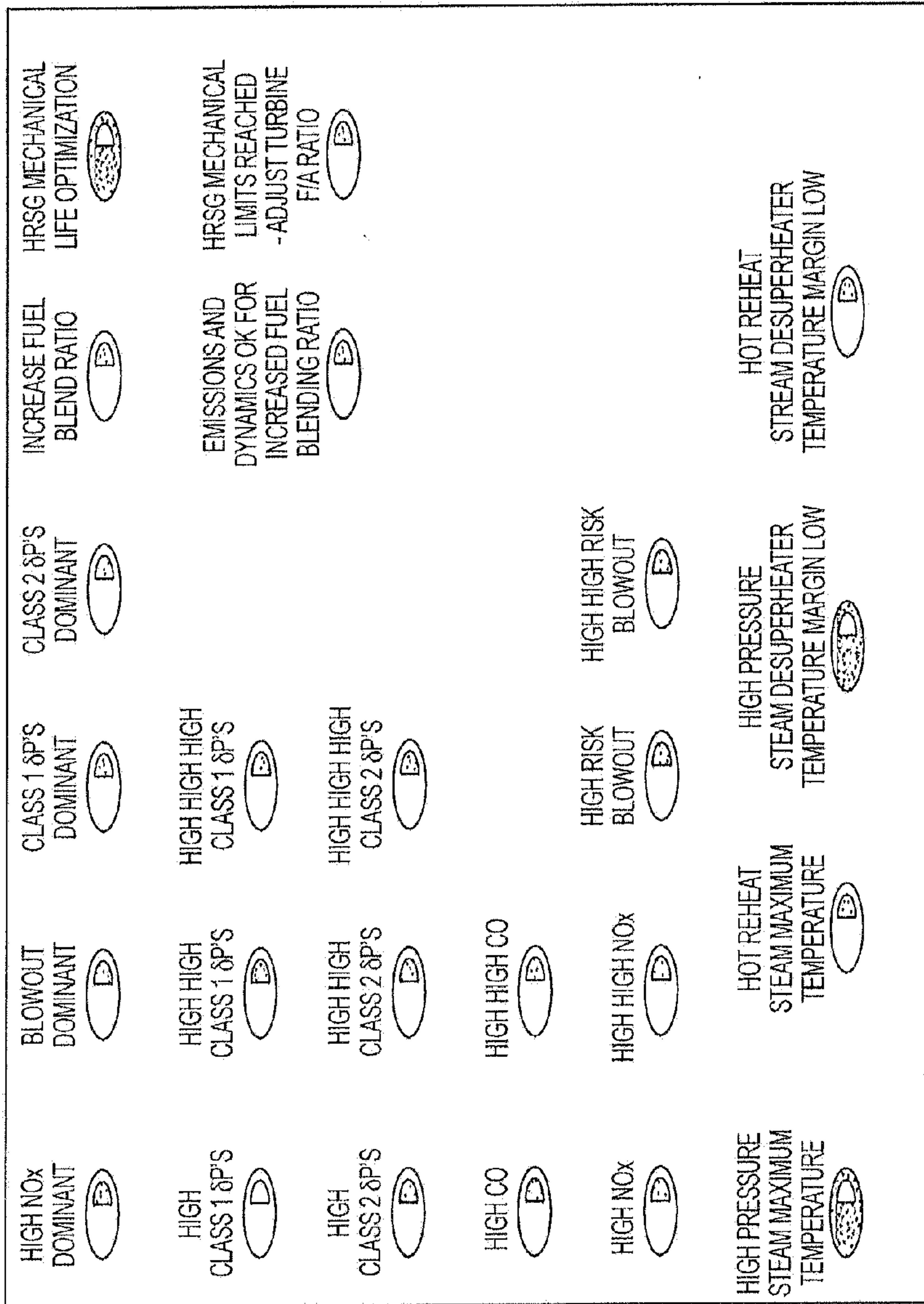


FIG. 10

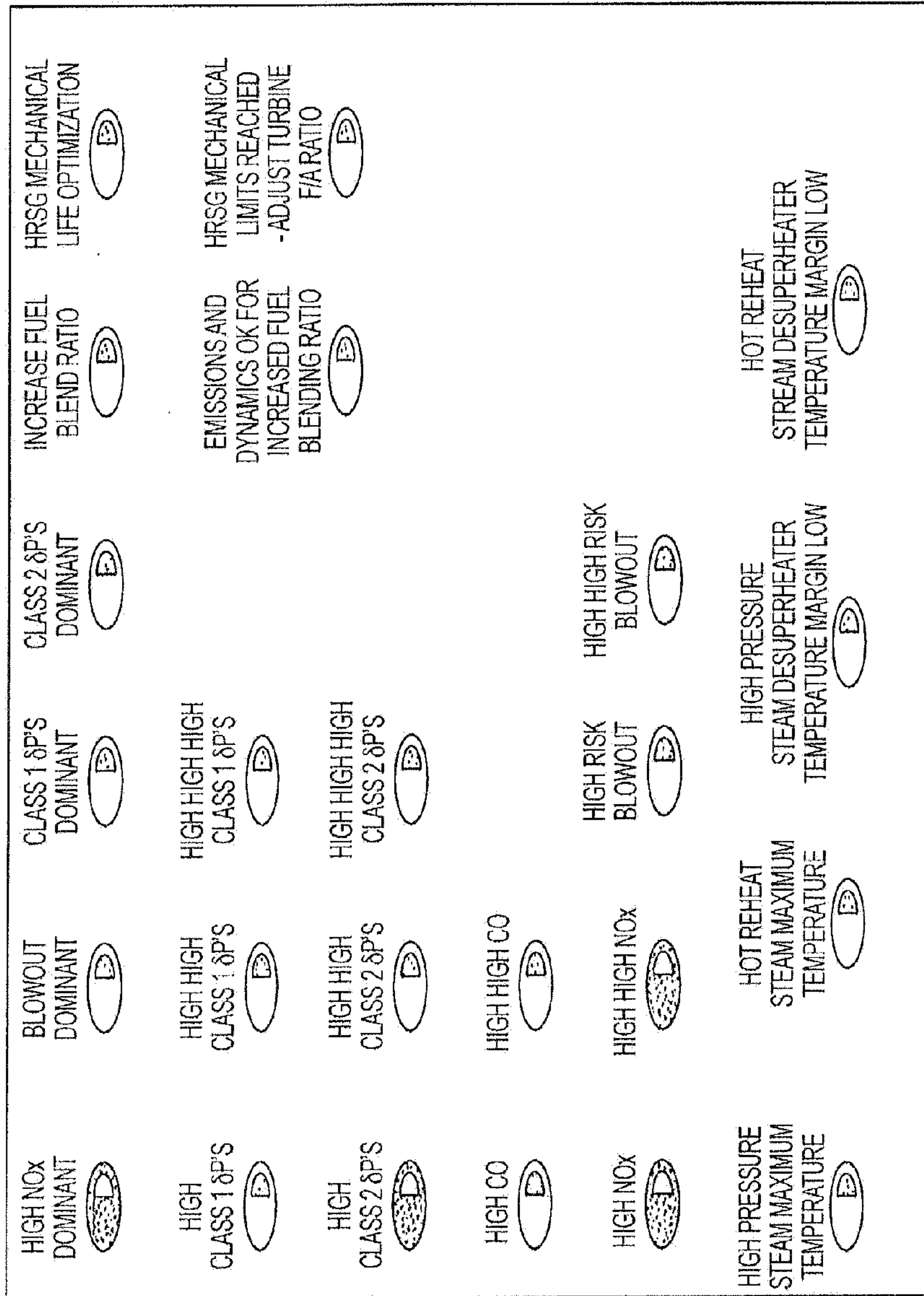


FIG. 11

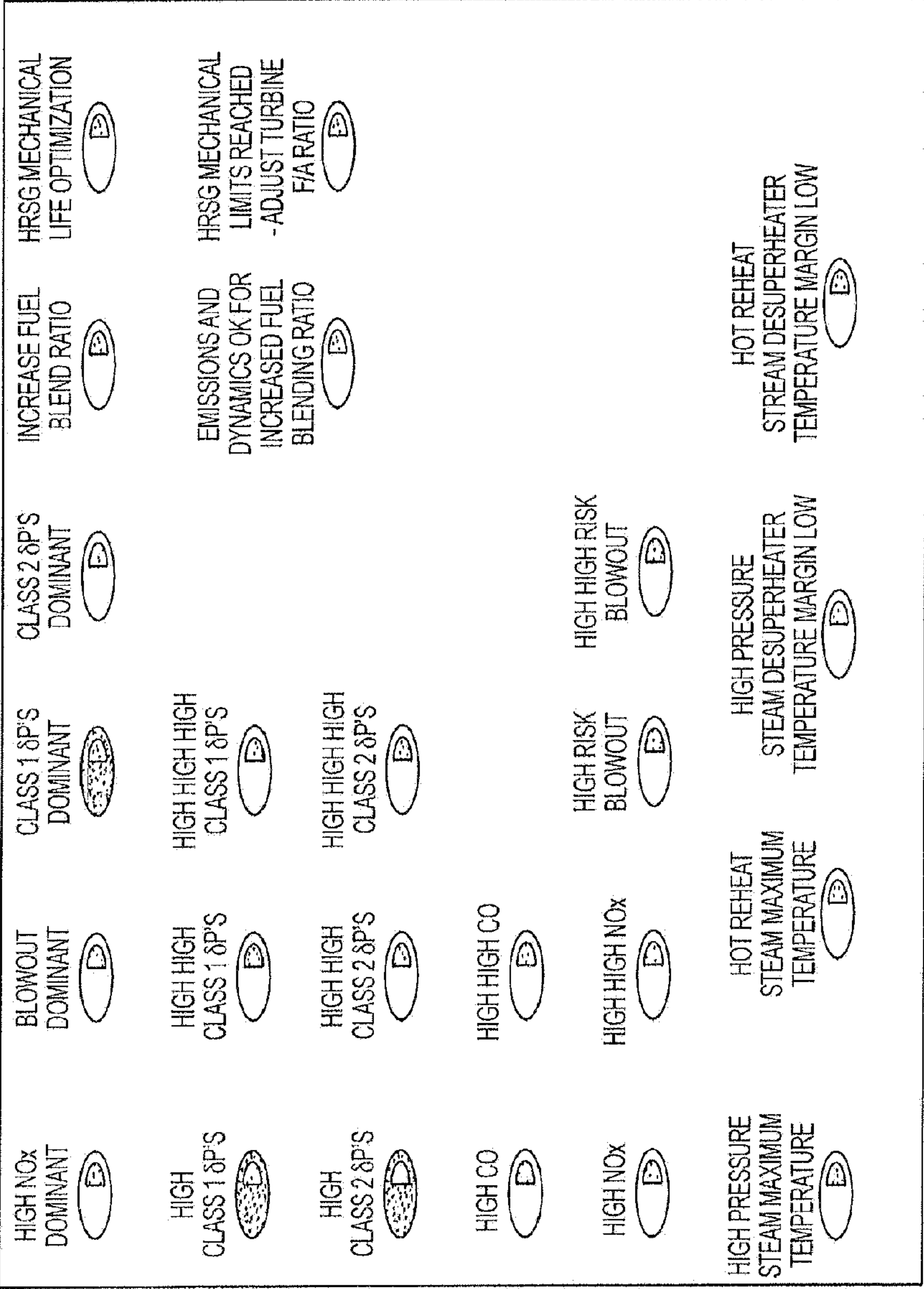


FIG. 12

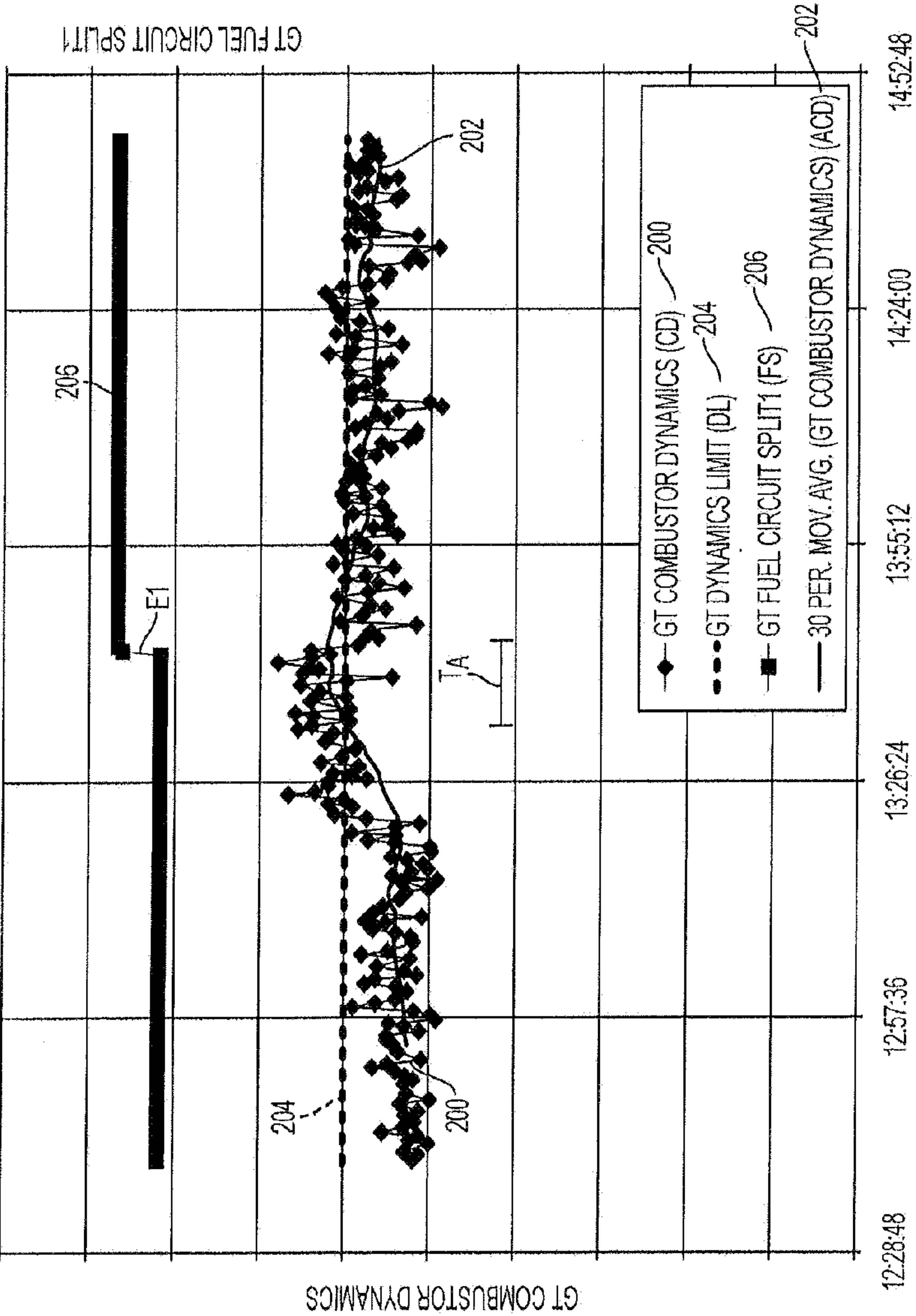


FIG. 13

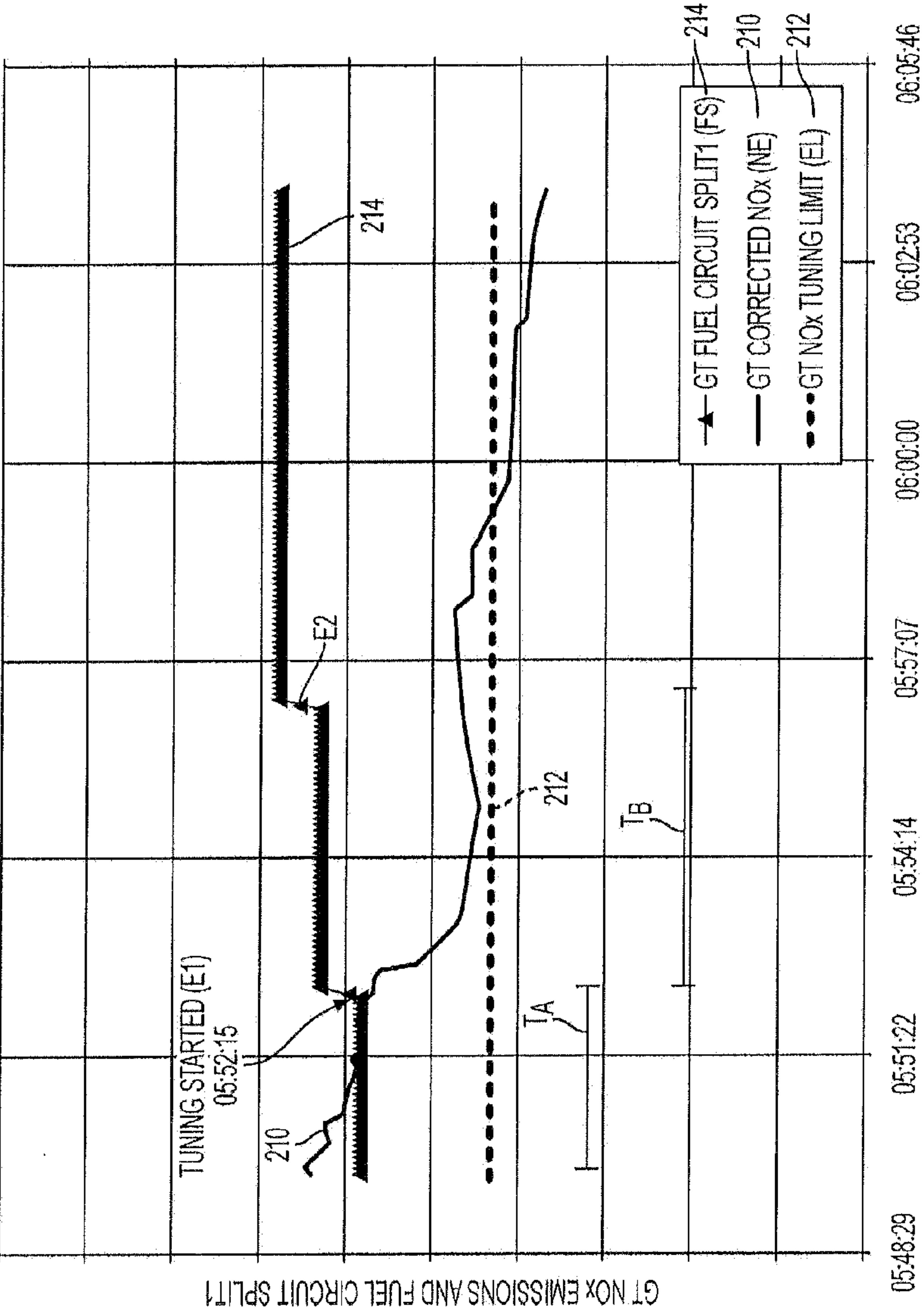


FIG. 14

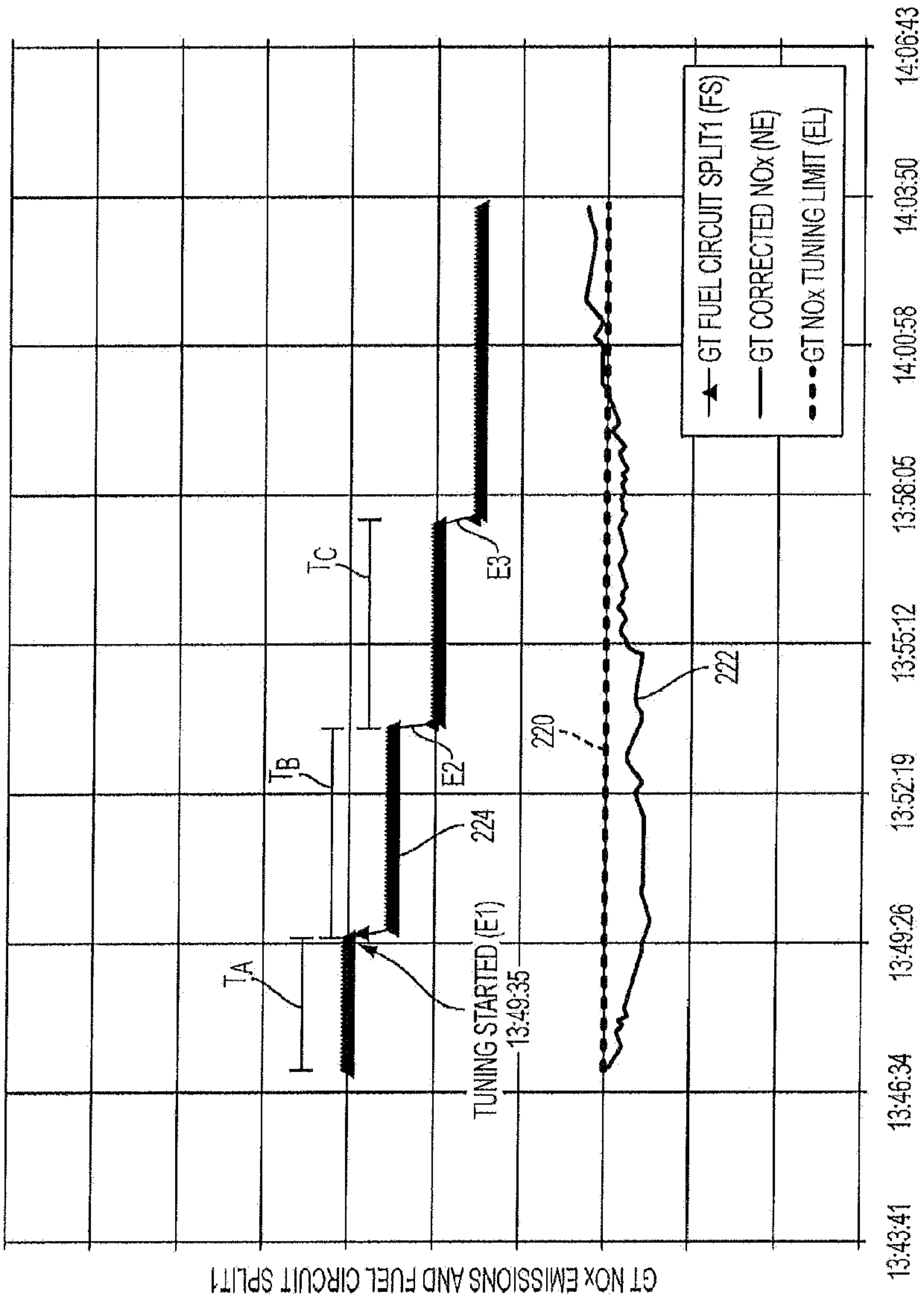


FIG. 15

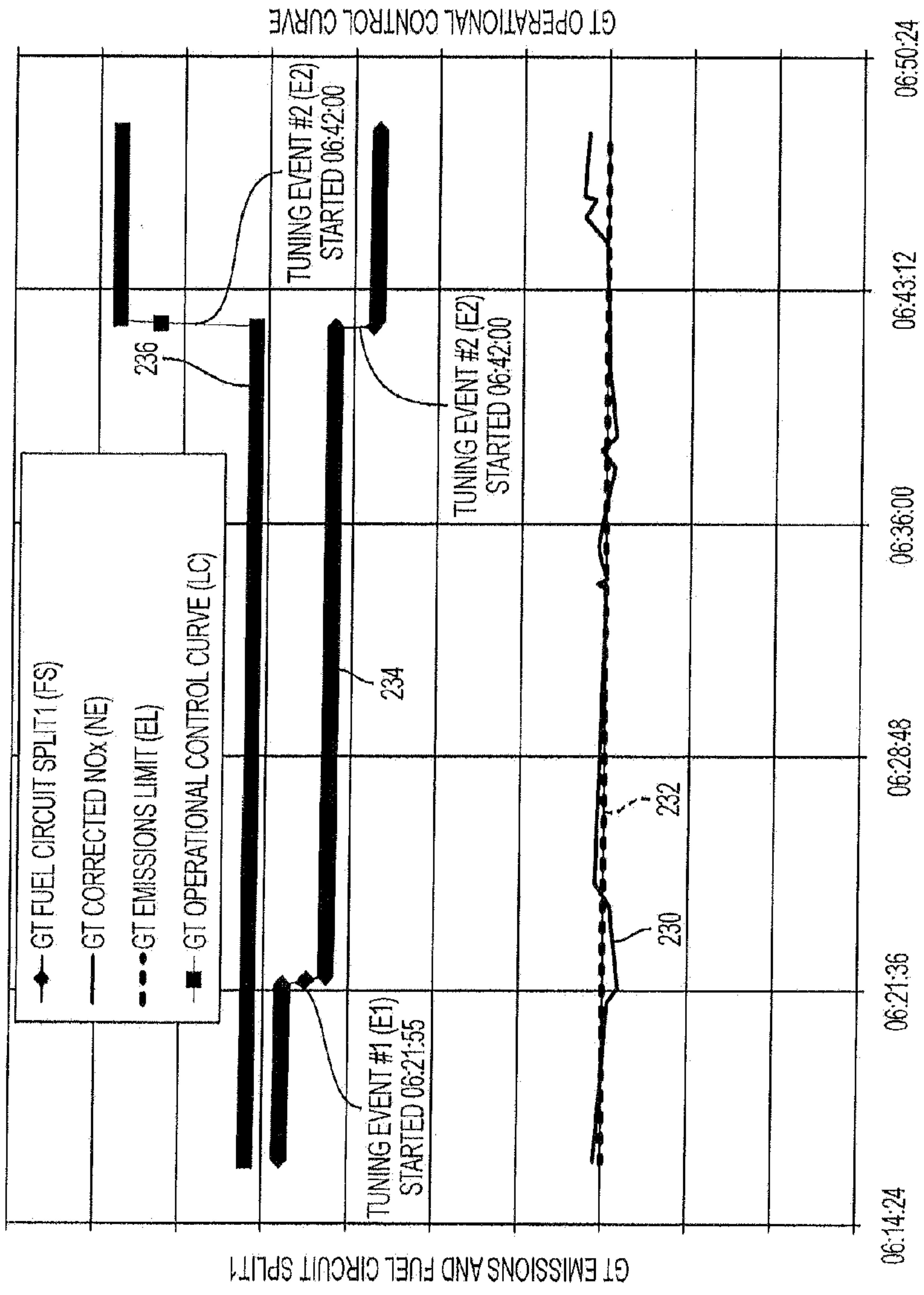


FIG. 16