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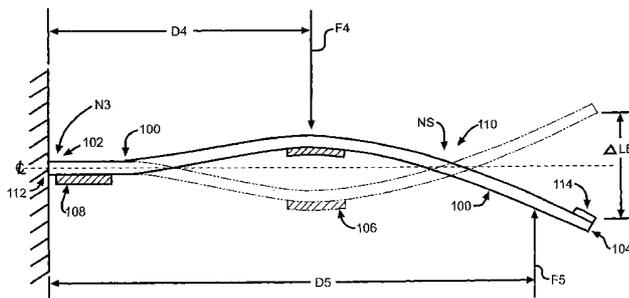
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(54) Title: SENSOR FOR NON-CONTACTING ELECTROSTATIC DETECTOR



(57) Abstract: A sensor for an electrostatic detector comprising an elongated vibratory element supported at a mechanical node at one end in the manner of a cantilever beam, a sensitive electrode on the vibratory element near the other end and adapted to be disposed toward an electrical charge, field or potential being measured, and a driver transducer on the vibratory element for vibrating the element in a direction to vary the capacitive coupling between the electrode and the electrical charge, field or potential being measured. The driver transducer is at a location along the beam and is operated at a vibratory frequency such' that a virtual mechanical node appears along the beam. This, in turn, prevents vibration of the mechanical node so that the degree of stiffness of the mechanical node does not affect the operating frequency and/or the displacement at the free end of the cantilever. The variation of the vibration amplitude of the beam and - thus the detector's motion amplitude at the end of the beam is made independent of both the mass of the sensor/probe and/or the mass of the structure, such as a clamping/holding fixturing, used to place the sensor/probe into its measuring position relative to the surface or area under test. The electrical performance is thus stabilized and is made independent of the particular clamping/holding structure used. An amplifier and the sensitive electrode are combined in an assembly on the vibratory element thereby providing a low impedance amplifier output which in turn allows a narrow spacing around the vibratory element to prevent entry of contaminants. The vibratory element can be provided with a structure which allows measurements with various spatial resolution characteristics. The magnitude of vibration of the vibratory element is sensed substantially at the mechanical mounting node and utilized by a circuit to control the amplitude of vibration.



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SENSOR FOR NON-CONTACTING  
ELECTROSTATIC DETECTOR

Background Of The Invention

The invention relates to the electrical measurement art, and more particularly to a new improved sensor or probe for use as a non-contacting electrostatic field, electrostatic voltage, or electrostatic charge detecting  
5 device.

Non-contacting electrostatic detectors are employed in those applications where electrical charge transferred to the measuring device through physical contact with the surface, circuit, or device being  
10 measured/monitored would modify or destroy the data. Such measurements include, for example, monitoring the voltage level of a photoconductor used in a copy machine or other electrophotographic processes, monitoring of electrostatic charge build-up on a dielectric web  
15 process line such as is used in textile or plastic film manufacturing processes, or monitoring the area around semiconductor manufacturing/handling processes where charge accumulation presents a source of product contamination or destruction through electrostatic  
20 discharge (arcing) events.

Heretofore, electrostatic sensors/probes of the type required to produce high accuracy, fast speed, low noise measurements, and yet be of physically small size, have been too expensive to manufacture to allow their  
25 use in higher quantity lower cost processes and/or apparatus such as low end laser printers, copy machine, or multiple point electrostatic charge build-up monitoring systems.

30

Summary of the Invention

It is, therefore, a primary object of this invention to provide a new and improved sensor/probe for

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electrostatic measurements which satisfies the foregoing requirements yet is of low cost to manufacture.

It is a further object of this invention to provide a sensor/probe having electrical performance independent  
5 of both the mass of the sensor/probe and/or the mass of the clamping/holding structure used to place the sensor/probe into its measurement position relative to the surface or area under test.

It is a further object of this invention to provide  
10 such a sensor/probe which minimizes or more easily eliminates the production of noise and offset signals thereby avoiding the need for providing relatively large spacing between electrical elements of the sensor/probe amplifier circuitry and sensor/probe housing elements  
15 thus reducing the ability of contamination to enter into the inside of the sensor/probe structure.

It is a further object of this invention to provide a sensor/probe which can be easily configured to provide various spatial resolution characteristics.

It is a further object of this invention to provide  
20 a sensor/probe having an electrical speed of response which is substantially increased so as to accurately measure higher frequency electrostatic data.

It is a further object of this invention to provide  
25 such a sensor/probe which is small in size, simple in construction, economical to manufacture and reliable in operation.

The present invention provides a sensor for an electrostatic detector comprising an elongated vibratory  
30 element supported at a mechanical node at one end in the manner of a cantilever beam, a sensitive electrode on the vibratory element near the other end and adapted to be disposed toward an electrical charge, field or potential being measured, and driver means on the

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vibratory element for vibrating the element in a direction to vary the capacitive coupling between the electrode and the electrical field or potential being measured. The driver means is at a location along the beam and is operated at a vibratory frequency such that a virtual mechanical node appears along the beam. This, in turn, prevents vibration of the mechanical node so that the degree of stiffness of the mechanical node does not affect the operating frequency and/or the displacement at the free end of the cantilever. The variation of the vibration amplitude of the beam and thus the detector's motion amplitude at the end of the beam is made independent of both the mass of the sensor/probe and/or the mass of the structure, such as a clamping/holding fixturing, used to place the sensor/probe into its measuring position relative to the surface or area under test. The electrical performance is thus stabilized and is made independent of the particular clamping/holding structure used. An amplifier and the sensitive electrode are combined in an assembly on the vibratory element thereby providing a low impedance amplifier output which in turn allows a narrow spacing around the vibratory element to prevent entry of contaminants. The vibratory element can be provided with a structure which allows measurements with various spatial resolution characteristics.

The foregoing and additional advantages and characterizing features of the present invention will become clearly apparent upon a reading of the ensuing detailed description together with the included drawing wherein:

Brief Description Of The Drawing Figures

Fig. 1 is a perspective view with parts removed of a prior art sensor;

5 Figs. 2A and 2B are diagrammatic views illustrating operation of the cantilever beam in the sensor of Fig. 1;

Fig. 3 is a diagrammatic view of a tuning fork system for use in a sensor;

10 Figs. 4A, 4B and 4C are graphs illustrating vibrational modes of a cantilever beam of the type shown in Figs. 2A and 2B;

Fig. 5 is a diagrammatic view of a sensor employing a cantilever beam according to the present invention;

15 Fig. 6 is a diagrammatic view further illustrating the present invention;

Fig. 7 is a schematic block diagram of a circuit for use with the sensor of the present invention;

20 Fig. 8 is a diagrammatic view of a sensor according to another embodiment of the present invention wherein the amplifier and sensitive electrode are combined into a single assembly; and

25 Fig. 9 is a diagrammatic view of a sensor according to another embodiment of the present invention which allows measurements with various spatial resolution characteristics.

Detailed Description Of The Illustrated Embodiment

30 Referring first to Fig. 1, there is shown a prior art sensor generally designated 10 wherein a vibrating element 40, in the form of a single cantilever, has an end connected by either attachment or integration into the baseplate 14 while the free end provides a mounting

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surface for an amplifier 90 and a sensitive electrode  
50. For a more detailed description of prior art sensor  
10, reference may be made to United States Patent No.  
4,763,078 issued August 9, 1988 entitled "Sensor For  
5 Electrostatic Voltmeter", the disclosure of which is  
hereby incorporated by reference.

As conventionally known, a simple cantilever beam  
in vibration produces unbalanced mechanical torque at  
the point of attachment or connection to the mechanical  
10 node structure, or, in the case of the sensor of Fig. 1,  
a baseplate structure. This is in contrast to a double  
cantilever beam structure, such as a tuning fork, where  
the mechanical forces produced by the motion of each of  
the tines of the tuning fork can be made equal and  
15 opposite thus producing no net mechanical torque at the  
common connection (fulcrum) of the fork. Therefore, a  
tuning fork operates at a stable amplitude and frequency  
as a function of mass (or mounting stiffness) present at  
the fulcrum whereas a single cantilever beam of the type  
20 shown in Fig. 1 is unstable in frequency and amplitude  
of vibration as the mechanical mass or mechanical  
stiffness at the baseplate attachment point varies.  
This unstable effect is particularly troublesome as an  
attempt is made to reduce the physical size of the  
25 baseplate structure in an effort to reduce the  
sensor/probe size and/or weight.

In particular, as the size of the probe is reduced  
significantly from the illustrative size shown and  
described in Patent 4,763,078, the amplitude of  
30 vibration of the free end of the cantilever beam can  
become unstable and dependent upon the fixturing used to  
position the sensor/probe relative to the surface/area  
under test. In addition, due to the unbalanced nature  
of a single cantilever beam causing the probe/sensor

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structure as well as the mounting fixture to be involved in the mechanical vibration system of the cantilever, undesired audible vibrations are produced and radiated outside of the sensor/probe structure to cause annoying audible noise. These mechanically undesirable effects could be eliminated by using an inherently balanced tuning fork in place of the single cantilever beam. However, manufacturing costs when using the double cantilever, i.e., tuning forks, would be prohibitive because the tuning fork cannot be easily fabricated as an integrated part of the baseplate 14 by inexpensive chemical milling processes or the like.

Another limitation on the operation of the probe/sensor shown in Fig. 1 can arise from the electrical connection made between sensitive electrode 50 and amplifier 90 via wire connection 52. Sensitive electrode 50 is operated at a fairly high impedance level (in the order of megohms) to be an effective electrostatic quantity detector. Wire connection 52, connected directly to sensitive electrode 50, is also at the same high impedance level as electrode 50. Even though connection wire 52 can be kept short, in the order of 2mm, it still must wrap around the end of cantilever 40 to be connected to amplifier 90. As the wire 52 is moving synchronously with the cantilever 40, it is vibrating relative to the end of the opening 36 in baseplate 14. To prevent the generation of extraneous electrical noise and signals onto wire 52 due to either contact potential differences in the metallic materials used by baseplate 14 and wire 52 or charge retaining dust, toner or other contaminants on either wire 52 or baseplate 14, a relatively large spacing between wire 52 and baseplate 14 must be accommodated. A larger spacing will decrease the amount of coupling between the

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contamination noise sources and wire 52, thus tending to decrease the undesirable noise effects of the contamination. However, it is also desirable to keep all of the openings between the cantilever 40 and  
5 baseplate 14 at a minimum spacing to prevent or reduce the amount of contamination which can enter into the inside of the sensor/probe structure.

Fig. 2A illustrates the operation of the single cantilever of the device of Fig. 1 described in patent  
10 4,763,078. As shown, the cantilever beam 40A is supported at one end at a mechanical node (N) while the other end is free to vibrate. Fig. 2A shows the cantilever beam 40A in the unenergized or rest position. A line 70 drawn through the beam to the mounting node  
15 (N) defines this rest position. Upon application of power to the oscillator circuitry including the drive and feedback piezoelectric ceramic chips (not shown), the cantilever beam 40A is made to oscillate at or around its fundamental resonate frequency (approximately  
20 700hz for the dimensions shown in the illustrative device of patent 4,763,078).

Fig. 2B illustrates the limits of mechanical motion produced by the beam 40B to produce vibratory motion  $\Delta L$  at the free end of the beam. It should be noted that  
25 any torque generated on the mechanical mounting node N by the motion of the cantilever beam is not compensated by any equal and opposite torque which would be generated, for example, by the tuning fork system 74 as shown in Fig. 3. In Fig. 3 torque T2 is exactly  
30 compensated by torque T3. T3 has the same magnitude but opposite direction because the tine 76 associated with T3 is always moving in a 180° phase related motion to the tine 78 associated with T2, as conventionally known.

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Therefore node N1 experiences equal and opposite torque to produce no net torque.

In the case of the system of Fig. 2 where the single cantilever operation is uncompensated and node (N) is not massive and therefore not mechanically "stiff", node (N) will be vibrated and will enter into the mechanical equation describing the motion of the cantilever as to operating frequency and motion  $\Delta L$  at the free end of the cantilever. As the degree of stiffness of the node (N) is also dependent upon the mass of the sensor/probe holding fixturing, the amplitude of oscillation of the free end and thus electrode 50 will also depend upon the fixturing stiffness and therefore the particular application which the sensor/probe is placed. Any unstableness of the amplitude of electrode 50 will adversely affect the accuracy and stability of the electrostatic quantity sensed by the sensor/probe particularly when an attempt is made to reduce the size and/or weight of the sensor/probe structure thus resulting in a less massive or less stiffness of the node (N).

The foregoing is further illustrated by considering the dynamics of thin cantilevered beams. Any vibrating beam will exhibit a set of discrete, resonant modes, the shapes and frequencies of which depend on the nature of the imposed boundary conditions. In Figs 4A, 4B and 4C curves 80, 82 and 84 depict the shapes of the first three vibrational modes ( $n = 1, 2, 3$ , respectively) for a cantilevered beam with one clamped end and one free end. These modes are infinite in number, though only the first few are likely to be of any practical significance in a device of the type described herein. The  $n = 1$  mode is very similar in appearance to the static displacement of a beam subjected to a normal load

at the end. The  $n = 2$  mode has a single inflection point and zero-crossing and the  $n = 3$  mode has two inflections and zero-crossings. Successively higher modes feature additional inflection points and zero-crossings.

The resonant frequencies of the modes have prescribed ratios with respect to each other. For a beam with ideal clamping - which imposes the conditions of zero motion and zero slope at the fixed end - the ratios are:

$$f_1:f_2:f_3:\dots = 1 : 6.2 : 17.5 : \dots$$

Thus, if the first resonance is at  $f_1 = 400$  Hz, then  $f_2 = 2.48$  kHz,  $f_3 = 7.0$  kHz, and so forth. In a metallic beam, the mechanical losses are low, so that the quality factors of the modes are high, that is,  $Q_1, Q_2, Q_3 > 50$ . Thus, the resonances may be considered to be well-isolated.

Ideal clamping is impossible to approach in any practical, that is, manufacturable probe of the type shown in Fig. 1, but it is also unnecessary. The resonant modes for the design shown in Fig. 1 are found to have shapes very similar to those of the "ideally clamped" case shown in Figs. 4A, 4B and 4C; however, the ratio of the resonant frequencies is somewhat altered:

$$f_1:f_2:f_3:\dots = 1 : 8 : 22:\dots$$

Another observation is that, as long as the piece is securely welded to its housing, the vibrational losses of these well-isolated resonances remain low, that is,  $Q > 50$ .

The differing shapes for the modes suggest that one should be able to excite selectively and optimally a given mode by proper placement of the piezoelectric actuator. Likewise, the position of a piezoelectric element might be optimized in order to detect the

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presence of a given mode with the maximum achievable sensitivity. Piezoelectric transducers convert AC electrical excitation into mechanical deformation. When one of these elements is rigidly attached to a beam, the voltage-induced changes in the lateral dimensions of the element - along the surface of the beam - induce a shear that causes the beam to deflect. Thus, as determined in accordance with the present invention, the best location for exciting (and also for sensing) any mode is at the point where the curvature of the deflected beam is greatest. This simple hypothesis about optimal transducer placement has been experimentally verified for the  $n = 1, 2,$  and  $3$  modes of a cantilevered beam.

Fig. 5 illustrates operation of the cantilever beam 100 of the sensor of the instant invention. As shown, cantilever beam (100) is supported at a mechanical node (N3), also designated 102, while the other 104 is free to vibrate. The oscillator circuitry as well as the placement of the drive 106 and feedback 108 piezoelectric ceramic chips along the cantilever beam 100 are different from the oscillator and chip placement in the sensor of patent 4,763,078 as will be explained herein. In the instant invention, when power is applied to the oscillator circuitry and piezo chips 106, 108, the cantilever beam 100 vibrates at a frequency which causes a virtual mechanical node labeled (NS), also designated 110, to appear along the cantilever beam. Due to the appearance of the virtual mechanical node (NS) when operating the cantilever 100 at a frequency above its fundamental frequency, a substantially compensating torque can be produced at the mechanical mounting node (N3) so that substantially no net torque is applied at node (N3).

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As shown in Fig. 5, the vibration of cantilever beam 100, as defined by the limits of vibration illustrated by the solid and dotted positions, produces a vibration of  $\Delta LB$  at the end 104 of the beam. As the virtual node (NS) remains stationary over the oscillatory vibration cycle, the portion of the beam between nodes (N3) and (NS) is, at all times, moving in the opposite direction as the section of the beam between node (NS) and the free end 104 of the beam. The production of the substantially compensating torque is shown graphically in Fig. 5. In Fig. 5 torque  $T_4$  produced by force  $F_4$  applied at distance  $D_4$  from node N3 is substantially compensated by torque  $T_5$  produced by  $F_5$  applied at distance  $D_5$  from node N3. Distance  $D_4$  is measured from node N3 to the center of mass of beam 100, and distance  $D_5$  is measured from node N3 to the reflected center of mass located between node (NS) and end 104.

When the substantially compensating torque  $T_5$  is produced, substantially no net torque is applied to node N3. With substantially no net torque applied at node N3, node N3 cannot be vibrated and therefore its degree of stiffness does not significantly affect the operating frequency of the beam 100 and/or the displacement  $\Delta LB$  at the end 104 of the cantilever 100. Thus the single cantilever beam 100 can be made to act as if its mounting node is torque compensated, similar to a tuning fork, when the cantilever is operated at a frequency which produces a virtual mechanical node, or multiple virtual mechanical nodes, at the proper position along the cantilever beam. With its mounting node torque substantially compensated, the variation of the vibration amplitude  $\Delta LB$  and thus the detector's motion amplitude at the end of the beam is made independent of

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both the mass of the sensor/probe and/or the mass of the clamping/holding fixturing used to place the sensor/probe into its measuring position relative to the surface or area under test. The electrical performance is thus stabilized and is made independent of the particular clamping/holding device used, which is an objective of this invention.

Considering application of the second ( $n = 2$ ) vibrational mode of the cantilevered beam 100 with the drive transducer 106 located so that it overlaps the peak near the middle, the mechanical response of the beam 100 will be maximized because, when deflected, its curvature is the greatest at this point. The feedback piezoelectric transducer 108 employed to sense the motion and to provide a signal to close the feedback loop, is located fairly close to the clamp 112. Placed at another location where the curvature is large, this element is found to sense an entirely adequate signal. The capacitive electrode 114, which is the heart of the non-contacting ESV instrument, is attached very close to the end 104 of the beam 100 where the amplitude of motion is greatest.

Using higher-order modes in a cantilevered beam ameliorates the serious problem relating to vibration isolation between the beam and its mounting. To understand how this improvement is effected, one may compare the shapes of the vibrational modes of the first and second modes in Figs. 4A and 4B. For the  $n = 1$  mode, the beam spends one-half of each cycle entirely above and the other half entirely below the equilibrium position. As a result, a maximum of torque is communicated to the housing by the beam. Because no practical housing or mount can be sufficiently massive to prevent it, this torque always leads to sympathetic

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vibration of the mount, and other components to which the probe is attached. On the other hand, for the  $n = 2$  mode, the deflection always bisects the equilibrium position, thereby providing some degree of balance  
5 between the outer and inner portions of the beam. The degree of self-balance increases as mode number  $n$  rises. Whether or not, due to other practical limiting factors, modes beyond the second ( $n = 2$ ) can be exploited has not yet been demonstrated. Still, from the standpoint of  
10 vibration isolation, the higher modes are better.

Another advantage of using higher-order modes is that a higher operational frequency for the probe improves the robustness of the support electronics and also increases the achievable sampling rate when the  
15 probe is used to measure electrostatic surface potential in critical applications such as xerographic copier/printers. This latter feature will offer a significant enhancement to xerographic process controllability of possibly great importance for the new  
20 class of low-cost, color copier/printers (three primaries + black) now on the horizon. If present-day black and white copy speeds are to be approached in color machines, higher speed -- and improved spatial resolution of photoreceptor charge -- will be essential.  
25 Improved spatial resolution is crucial to achieving the more restrictive registration requirements of color technology.

Consider the following example to illustrate the improvement. The second mode of a one cm cantilevered  
30 beam in a non-contacting electrostatic voltmeter (ESV) shifts the operating frequency from about 400 Hz to about 2400 Hz. This six-fold increase considerably improves the effective ESV bandwidth. Another advantage is that the mechanical loss of the second mode is

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smaller than the first, that is,  $Q_2 > Q_1$ . The higher  $Q$  of the second mode is valuable in two respects. First, it improves the electrical-to-mechanical efficiency of the piezoelectric drive transducer, which translates to stronger vibrational motion for given AC excitation. Second, higher  $Q$  improves the robustness of the feedback control circuitry by improved filtering of noise signals from the output of the sensing piezoelectric element.

The foregoing is illustrated further by considering Fig. 6 which illustrates the first and second modes for a cantilevered beam for an electrostatic voltmeter according to the present invention. Beam 100' is provided with drive and feedback piezo transducers 106' and 108', respectively, and capacitive sensor 114'.

Fig. 6 also shows the displacement profiles 120 and 122 for the first ( $n=1$ ) and second ( $n=2$ ) modes. The Euler-Bernoulli continuum beam model gives a simple expression for the natural frequency of the first mode:

$$f_1 = \frac{1}{2\pi} \frac{(1.875)^2 t}{L^2} \sqrt{\frac{E}{\rho}}$$

where  $L$  and  $t$  are the length and width of the beam, and  $E$  and  $\rho$  are Young's modulus and the mass density of the material, respectively.

For typical ESV probe dimensions, the  $n=1$  mode provides achievable resonant frequency about half the value needed to provide sufficient sampling bandwidth for color copiers. In addition, the resonance peak of assembled ESV probe units exhibits unacceptably high sensitivity to the method of mounting in a copier. The foregoing equation shows that, once the beam material has been chosen, the resonant frequency can be changed

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only by varying length L or thickness t. Increasing the frequency this way reduces the amplitude, which in turn lowers the voltage-detecting sensitivity of the probe.

Exploiting the second (n=2) mode instead of the first alleviates both these problems, and with minimal  
5 redesign of beam or housing. The resonant frequency of the second mode, shown in Fig. 6, is:

$$f_2 = \frac{1}{2\pi} \frac{(4.694)^2 t}{L^2} \sqrt{\frac{E}{\rho}}$$

10 From both equations, it follows that  $f_2/f_1 \approx 6.3$ , thereby solving the first problem. Furthermore, we find experimentally that the sensitivity to mounting of the n=2 resonance peak is much lower than for the n=1 mode, presumably because the motion of the second mode tends  
15 to be self-balancing. By comparison, the vibration of the first resonance virtually vanishes for some mounting arrangements, whereas the amplitude of the second resonance drops only 10% to 30%. It is very important to note that efficient excitation and detection of the  
20 n=2 mode require placement of piezotransducer elements near maxima of the curvature. Fig. 6 shows optimized placements for the drive 106' and motion-sensing 108' elements. Likewise, the capacitative (ESV) electrode 114' must be located either near the end of the beam (as  
25 shown in Fig. 6), or at the location of the local maximum displacement closer to the clamped end.

An amplifier 130 is provided on beam 100 and functions in a manner similar to that of amplifier 100 disclosed in the above-referenced patent 4,763,078. A  
30 circuit 140 for operating the sensor of Figs. 5 and 6 is shown diagrammatically in Fig. 7. Circuit 140 has a

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portion 142 operatively connected to amplifier 130 for providing operating voltages for the amplifier and for pre-processing signals received from amplifier 130 and transmitting such signals to the signal processing/  
5 monitoring portion of the electrostatic voltmeter in a known manner. Circuit 140 also has a drive or oscillator portion 144 for providing drive signals to operate the drive transducer 106 to vibrate the beam. The feedback transducer 108 is connected in controlling  
10 relation to portion 144. The frequency of vibration of beam 100 is sensed by transducer 108 and the signal indicative thereof is utilized by circuit portion 144 to control the frequency of the drive signal generated thereby and thus control the vibration frequency of beam  
15 100. The various electrical connections can be made in a manner similar to that disclosed in patent 4,763,078.

To facilitate the proper frequency of operation in order to produce virtual node NS designated 110 in Fig. 5, the oscillator circuitry 144 together with the piezo  
20 electric ceramic chips 106, 108 operate as a closed loop frequency selective positive feedback system which maintains electrical oscillation at the particular frequency which is synchronous with the mechanical frequency where the virtual node appears. This  
25 frequency is related to the fundamental mechanical oscillation frequency of the single cantilever 100 by a factor of approximately six times. Therefore a cantilever of the dimensions illustrated in patent 4,763,078 would operate at a frequency of approximately  
30 4 kilohertz when operated according to the principle of this invention, instead of approximately 700 Hz when operating in the fundamental node. This results in the detector operating at the 4 kilohertz frequency thus increasing the speed at which the detector can sense and

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follow the electrostatic data being measured, thus accomplishing another objective of this invention.

The sensor of the present invention operates in the following manner. The sensor housing is positioned with  
5 electrode 114 facing a surface, i.e. a test surface, bearing the electrostatic field or potential to be measured. With the electrode sensitive surface exposed to the test surface there is capacitive coupling between electrode 114 and the test surface. Oscillator 144  
10 applies an a.c. voltage through to piezoelectric element 106 causing it to vibrate in a direction generally perpendicular to the plane of plate from which beam 100 extends and at a frequency substantially equal to the oscillator output frequency. The cantilevered beam 100  
15 transmits the vibrations of piezoelectric element 106 to electrode 114. Electrode 114 is vibrated in a direction generally perpendicular to the plane thereof and at a frequency determined by the oscillator output frequency and the mechanical characteristics of element 100 as  
20 previously described. As electrode 114 is vibrated it moves toward and away from the test surface thereby varying the capacitive coupling between the electrode sensitive surface and the test surface.

As element 100 vibrates, the motion thereof  
25 generates a voltage in piezoelectric element 108 attached to element 100 which is fed back electrically to the circuit 140 to complete a feedback loop to sustain the oscillation. In particular, the signal fed back representative of the actual vibration or  
30 oscillation is compared to a reference signal representative of desired oscillation or vibration, e.g. the to control the output of the oscillator. This is done in a manner readily apparent to those skilled in

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the art, so that any further detailed description is believed to be unnecessary.

If the potential of the detector electrode 114, which is the system feedback potential, is different from the potential on the test surface, an a.c. signal is induced on the surface of electrode 114. This induced a.c. signal is applied to the input of preamplifier 130. The amplitude and phase of the induced signal, relative to the drive signal obtained from the oscillator, is dependent on the magnitude and polarity of the difference potential, respectively, of the two surfaces. The output of amplifier 130 is applied to the electrostatic voltmeter and then applied to the input of a demodulator (not shown). The synchronous demodulator, using the drive signal obtained from the oscillator as a reference phase, functions to remove the amplitude and phase information from the induced a.c. signal. A line frequency derived oscillator signal can be used to insure that the modulation of any ambient induced signals onto the detector electrode 114 cannot appear as noise after the demodulation process. The demodulated signal, which is a d.c. voltage, then can be amplified in a high gain amplifier, and the output of that amplifier is applied to the input of a suitable device, such as a meter, for continual display and monitoring of the test surface potential.

Fig. 8 illustrates an embodiment of the present invention which reduces the ability of contaminants to pass through the spacing between the vibrating element and adjacent structure thereby reducing the ability of such contaminants to enter into the internal structure of the sensor/probe. The sensor/probe according to the present invention typically can include the structure

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wherein, the vibratory element supported at one end from a plate has side and end edges in closely spaced relation to the edges of an opening in the plate in which the vibratory element is located. For example, as  
5 shown in Fig. 1 of the above-referenced patent 4,763,078 a relatively narrow spacing exists between side edges 42, 44 of vibratory element 40 and edges 32, 34 of the opening in the plate and, similarly, between edge 46 of element 40 and edge 36 of the opening. Making this  
10 spacing as narrow as possible reduces the likelihood of contamination entering the internal structure of the sensor/probe.

As shown in Fig. 8, the capacitive sensor 114' or detector surface, amplifier 130' and a protective  
15 element 150 (if used) are fabricated into a single low profile assembly 152 mounted on beam 100" near the free end thereof. Protective element 150 can be in the form of a layer overlying the detector surface 114" and of a material which protects surface 114" but does not  
20 interfere with the capacitive coupling of surface 114" to the field or potential being detected. The use of this integrated assembly 152 facilitates the placement of amplifier 130' on the same surface of the cantilever beam 100" as detector surface 114' rather than on the  
25 opposite surface as disclosed in patent 4,763,078. This integrated package assembly 152 eliminates connecting wire designated 52 in patent 4,763,078 to connect the detector element to the amplifier due to the connection being integrated into the assembly. This eliminates the  
30 routing of wire 52 shown in patent 4,763,078, a high impedance level connection, past the electrical noise producing contamination found in the area at the end of the opening in which the cantilever vibrates. Instead, in the arrangement of Fig. 8, a connecting wire 156 is

- 20 -

connected to the output 158 of amplifier 130', which is a relatively low output impedance afforded by the amplifier 130' and is routed around the end of the cantilever 100", and eventually connects to a circuit  
5 like that designated 140 in Fig, 7 or to the sensor/probe output cable assembly. Lead 160 is soldered or otherwise connected to beam 100" to provide a common return for amplifier 130'. The ability of the contamination and/or contact potential between the  
10 vibrating cantilever 100" and the end of the opening in the plate to produce electrical noise in connection 156 is dramatically reduced due to the reduced impedance level of connection circuitry 156 over the connection circuitry heretofore employed. This reduced noise  
15 effect will allow the spacing between the wire 156 and the end of the opening to be reduced without the disadvantage of increasing the noise signals as found in the prior art. Using a reduced spacing will reduce the ability of contamination to enter into the internal  
20 structure of the sensor/probe, thus accomplishing another objective of this invention.

Fig. 9 illustrates an embodiment of the present invention which allows measurements with various spatial resolution characteristics. Beam 100'" has driver 106"  
25 and feedback 108" transducers thereon along with an amplifier/sensing electrode assembly 152' thereon similar to assembly 152 of Fig. 8. Fig. 9 shows the sensor/probe of this invention with added element SR, also designated 170, a spatial resolution-determining  
30 element. Aperture A, also designated 172, located in element SR in registry with detector element or assembly 152' can be made in various shapes and opening sizes/areas to provide a variety of spatial resolution characteristics to the measurement of electrostatic

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data, particularly electrostatic voltage or charge data associated with a measured surface to which the sensor/probe is coupled. The use of element SR carried by beam 100'' thus accomplishes yet another objective of this invention by providing a configuration which allows measurements with various spatial resolution characteristics. However, the sensor/probe of the present invention may be operated without element SR if wide spatial resolution characteristics are desired.

10 As previously described, the oscillator circuitry 144 and placement of the piezo electric chips 106, 108 or beam 100 can be advantageous designed to cause the cantilever to operate reliably and with stability at that particular frequency which causes the virtual node NS, also designated 110 in Fig. 5, to appear along the cantilever to produce balanced torque at the cantilever mounting node N3, also designated 102 in Fig. 5. In its simplest form the oscillator circuitry has a frequency selective characteristic which provides high positive feedback gain at the particular mechanical resonate or near resonate frequency of the cantilever beam where the stable virtual node NS is generated. The mechanical resonance of the cantilever beam is a function of the dimensions of the beam, the material of the beam, and the mass and compliance of the various elements added to the beam such as the piezo chips and detector/electrode/amplifier assembly. As these mechanical frequency determining factors are stable with time and temperature while the frequency selective characteristics of the oscillator circuitry can also be made quite stable, long term stability of the operation of the cantilever can be anticipated using ordinary low cost components. In particular, use of the feedback piezo electric ceramic chip 108 as an oscillator loop

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feedback element into the high gain frequency selective oscillator circuitry 144 whose output feeds electrical energy into the drive piezo electric ceramic chip 106 completes an oscillatory loop. As this loop includes  
5 the mechanical resonate characteristics of the cantilever beam onto which the feedback and drive piezo ceramics are placed, stable operation of the cantilever beam can be maintained.

The feedback piezo ceramic chip may be placed at  
10 other positions along the cantilever as dictated by other constraints. Although the technique described herein results in the advantageous use of a single cantilever beam in an electrostatic sensor/probe, the technique described can be used advantageously in other  
15 disciplines as well.

It is therefore apparent that the present invention accomplishes its intended objectives. While embodiments of the present invention have been described in detail, that is done for the purpose of illustration, not  
20 limitation.

In the Claims:

1. A sensor for an electrostatic detector such as an electrostatic voltmeter comprising:
- 5 (a) an elongated vibratory element having two ends;
- (b) means for supporting said vibratory element at one of said ends as a cantilever beam;
- (c) a sensitive electrode on said vibrating  
10 element near the other of said ends and disposed toward an electrical charge, field or potential being detected having capacitive coupling to said sensitive electrode;
- (d) a drive transducer attached to said vibratory  
15 element at a predetermined location thereon for vibrating said element at a predetermined frequency in a direction to vary the capacitive coupling between said sensitive electrode and the electrical charge, field or  
20 potential being detected;
- (e) an oscillator operatively connected to said drive transducer for operating said transducer to vibrate said element; and
- (f) an amplifier combined with said sensitive  
25 electrode in an assembly on said vibrating element, said amplifier deriving an electrical signal from said sensitive electrode containing information about the electrical charge, field or potential being detected.
- 30
2. The sensor according to claim 1, wherein said means for supporting said vibratory element defines a mechanical mounting node at said end and wherein the location of said drive transducer and the frequency of

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vibration of said element are selected to vibrate said element at a frequency above the fundamental vibration frequency of the cantilever beam so that a substantially compensating torque is produced at the mechanical  
5 mounting node of the beam so that substantially no net torque is applied to said mounting node so that the degree of stiffness of the mounting node does not affect the operating frequency of the beam or the displacement of the free end of the beam.

10

3. The sensor according to claim 1, wherein said drive transducer is at a location on the cantilever beam where the curvature is greatest for the selected mode of vibration of the beam.

15

4. The sensor according to claim 2, wherein the selected mode of vibration of the beam is the second mode.

20

5. The sensor according to claim 1, wherein said amplifier is mounted on said vibrating element and said sensitive electrode is on said amplifier.

25

6. The sensor according to claim 1, further including a feedback transducer on said vibratory element for sensing the vibration frequency of said element.

30

7. The sensor according to claim 6, further including a control connected to said feedback transducer and connected to said oscillator for controlling said oscillator to vibrate said element at the predetermined frequency.

8. The sensor according to claim 6, wherein said drive transducer and said feedback transducer are located on one surface of said vibratory element and said assembly of amplifier and sensitive electrode assembly is located  
5 on the opposite surface of said vibratory element.

9. The sensor according to claim 5, further including a protective element overlaying said sensitive electrode and of a material which allows capacitive coupling  
10 between said sensitive electrode and the electrical charge, field or potential being detected.

10. The sensor according to claim 1, further including a low impedance connection leading from the output of  
15 said amplifier to a location of use for the derived signal containing information about the electrical charge, field or potential being detected.

11. A sensor for an electrostatic detector such as an  
20 electrostatic voltmeter comprising:

- (a) an elongated vibratory element having two ends;
- (b) means for supporting said vibratory element at one of said ends as a cantilever beam;
- 25 (c) a sensitive electrode on said vibratory element near the other of said ends and disposed toward an electrical charge, field or potential being detected having capacitive coupling to said sensitive electrode;
- 30 (d) a drive transducer attached to said vibratory element at a predetermined location thereon for vibrating said element at a predetermined frequency in a direction to vary the capacitive coupling between said sensitive

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electrode and the electrical charge, field or potential being detected;

- 5 (e) an oscillator operatively connected to said drive transducer for operating said transducer to vibrate said element;
- (f) means for deriving an electrical signal from said sensitive electrode containing information about the electrical charge, field or potential being detected; and
- 10 (g) said vibrating element including a spatial resolution determining structure including an aperture in registry with said sensitive electrode of a size and shape selected to provide desired spatial resolution
- 15 characteristics to the detection of the electrical charge, field or potential.

12. The sensor according to claim 11, wherein said means for deriving an electrical signal comprises an amplifier.

20

13. The sensor according to claim 11, further including a feedback transducer on said vibrating element for sensing the vibration of said element.

25

14. The sensor according to claim 13, further including a control connected to said feedback transducer and connected to said oscillator for controlling said oscillator to operate said drive transducer.

30

15. The sensor according to claim 11, wherein said means for supporting said vibrating element defines a mechanical mounting node at said end and wherein the location of said drive transducer and the frequency of

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vibration of said element are selected to vibrate said element at a frequency above the fundamental vibration frequency of the cantilever beam so that a substantially compensating torque is produced at the mechanical mounting node of the beam so that substantially no net torque is applied to said mounting node so that the degree of stiffness of the mounting node does not affect the operating frequency of the beam or the displacement of the free end of the beam.

10

16. A sensor for an electrostatic detector such as an electrostatic voltmeter comprising:

- (a) an elongated vibratory element having two ends;
- 15 (b) means for supporting said vibratory element at one of said ends as a cantilever beam and defining a mechanical mounting node at said end;
- (c) a sensitive electrode on said vibratory element near the other of said ends and disposed toward an electrical charge, field or potential being detected having capacitive coupling to said sensitive electrode;
- 20 (d) a drive transducer attached to said vibratory element at a location thereon between said mechanical mounting node and said other end for vibrating said element at an amplitude and frequency in a direction to vary the capacitive coupling between said sensitive electrode and the electrical charge, field or potential being detected;
- 25 (e) an oscillator operatively connected to said drive transducer for operating said transducer to vibrate said element;
- 30

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- (f) a feedback transducer attached to said vibrating element substantially at said mechanical mounting node to sense the amplitude of vibration of said element;
- 5 (g) a driver/oscillator connected to said feedback transducer and in controlling relation to said drive transducer for controlling the amplitude of vibration of said vibratory element; and
- 10 (h) means for deriving an electrical signal from said sensitive electrode containing information about the electrical charge, field or potential being detected.

17. The sensor according to claim 16, wherein the location of said drive transducer and the frequency of vibration of said element are selected to vibrate said element at a frequency above the fundamental vibration frequency of the cantilever beam so that a substantially compensating torque is produced at the mechanical mounting node of the beam so that substantially no net torque is applied to said mounting node so that the degree of stiffness of the mounting node does not affect the operating frequency of the beam or the displacement of the free end of the beam.

25 18. A method for operating a sensor for an electrostatic detector such as an electrostatic voltmeter wherein the sensor comprises an elongated vibratory element supported at a mechanical mounting node as a cantilever beam and having a sensing electrode on the free end thereof and wherein the element is vibrated to vary the capacitive coupling between the sensitive electrode and an electrical charge, field or potential being detected, said method comprising:

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- 5 (a) applying a vibratory force to said element at  
a location thereon and at a frequency to  
vibrate the element at a frequency above the  
fundamental vibration frequency of the  
cantilever beam so that a substantially  
compensatory torque is produced at the  
mounting node of the element so that  
substantially no net torque is applied at the  
mounting node so that the degree of stiffness  
10 of the mounting node does not affect the  
operating frequency of the beam or the  
displacement of the free end of the beam;
- 15 (b) sensing the magnitude of vibration of the  
element substantially at the mechanical  
mounting node;
- (c) utilizing the sensed vibration amplitude to  
control the amplitude of the vibration force  
applied to the element; and
- 20 (d) deriving an electrical signal from the  
sensitive electrode containing information  
about the electrical charge, field or  
potential being detected.
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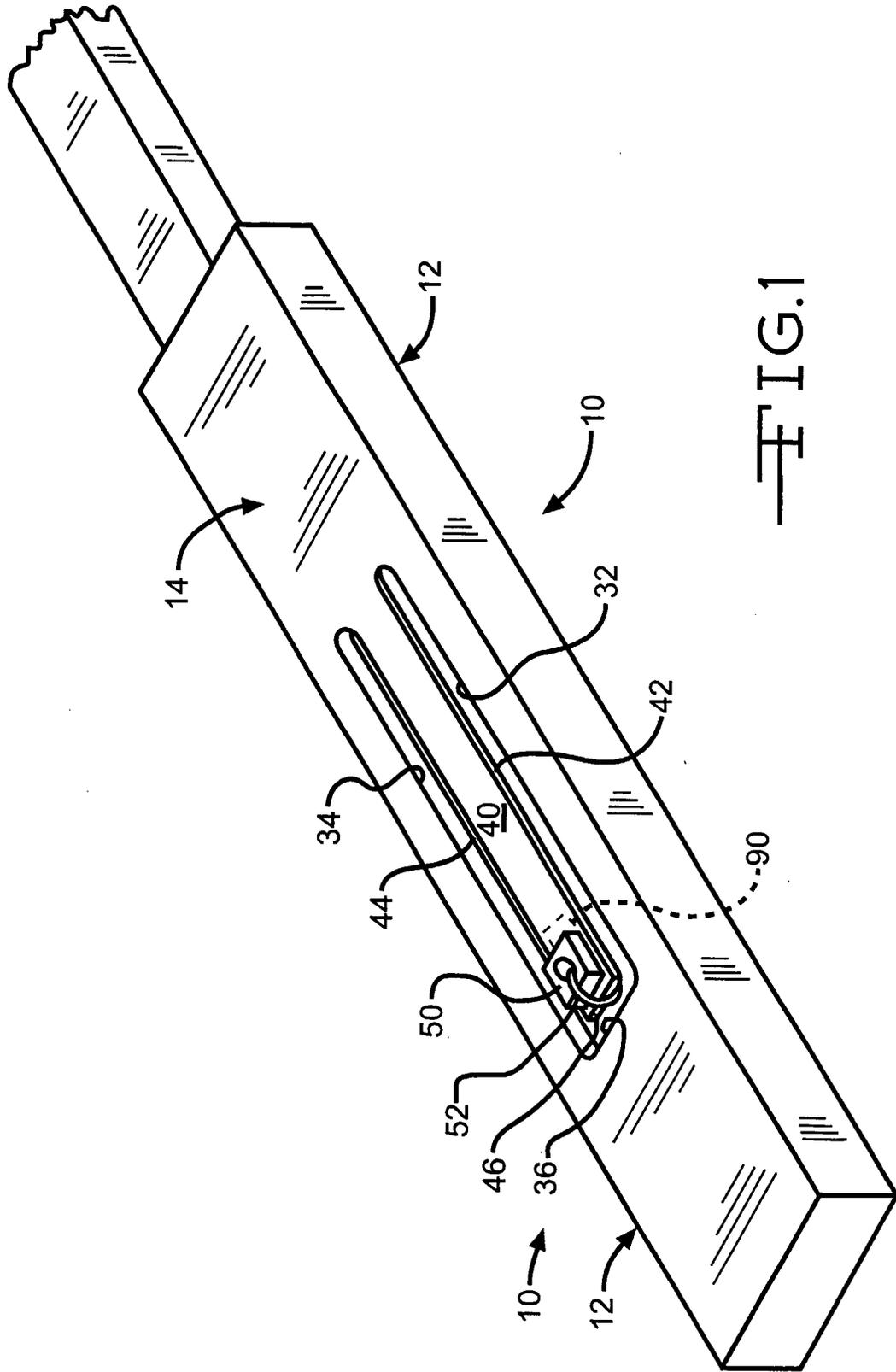


FIG. 1

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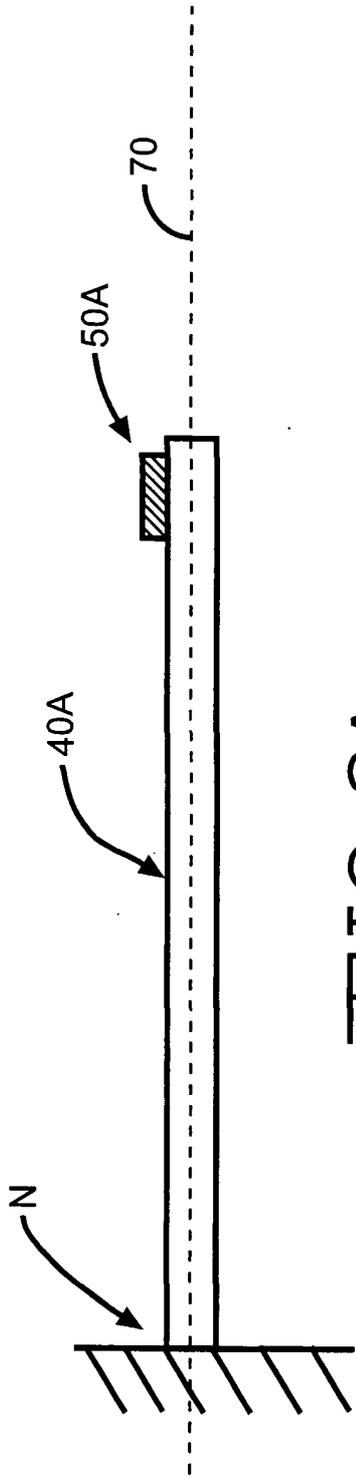


FIG. 2A

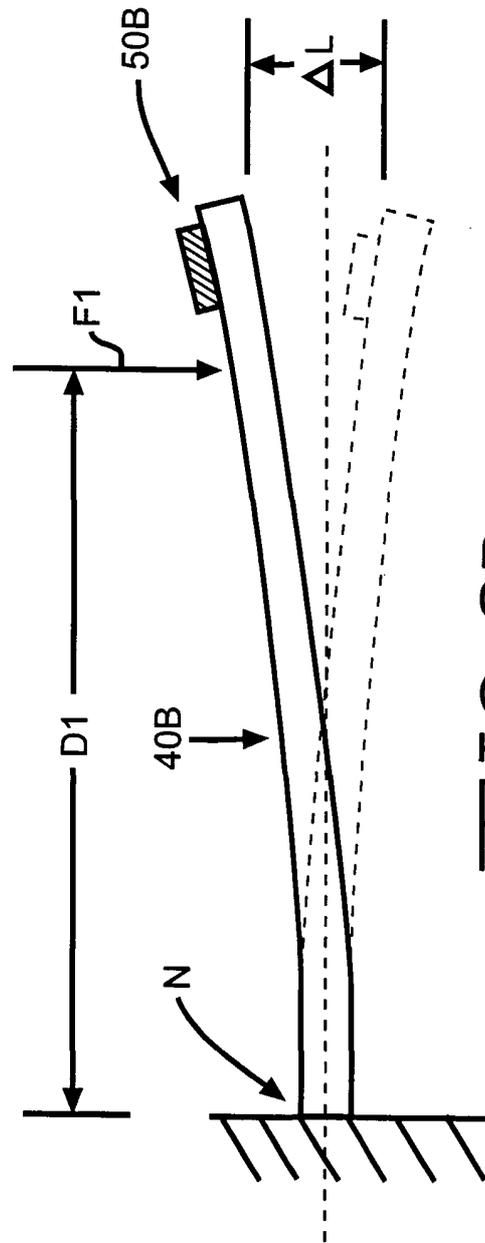


FIG. 2B

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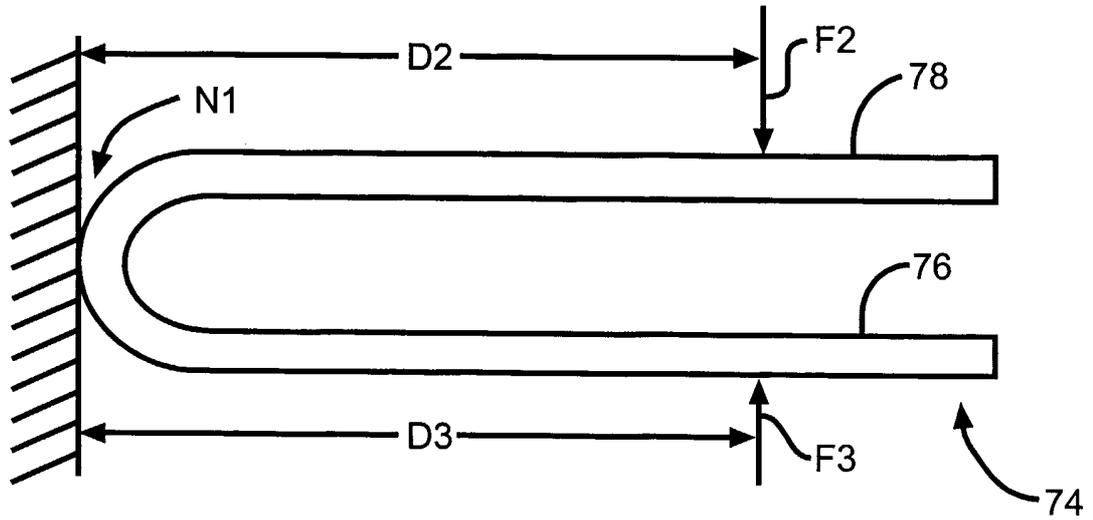


FIG. 3

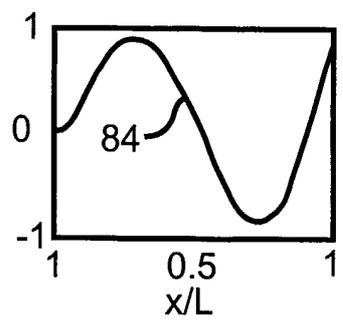
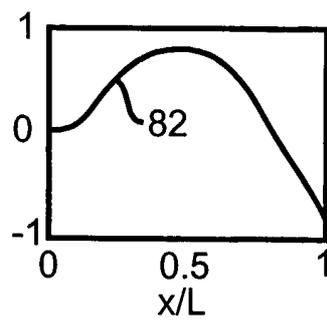
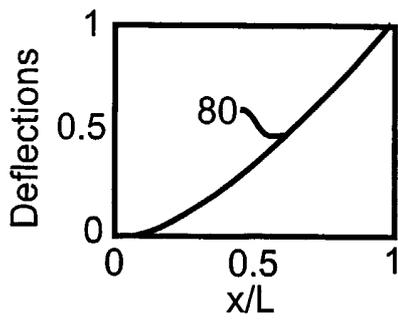


FIG. 4A FIG. 4B FIG. 4C

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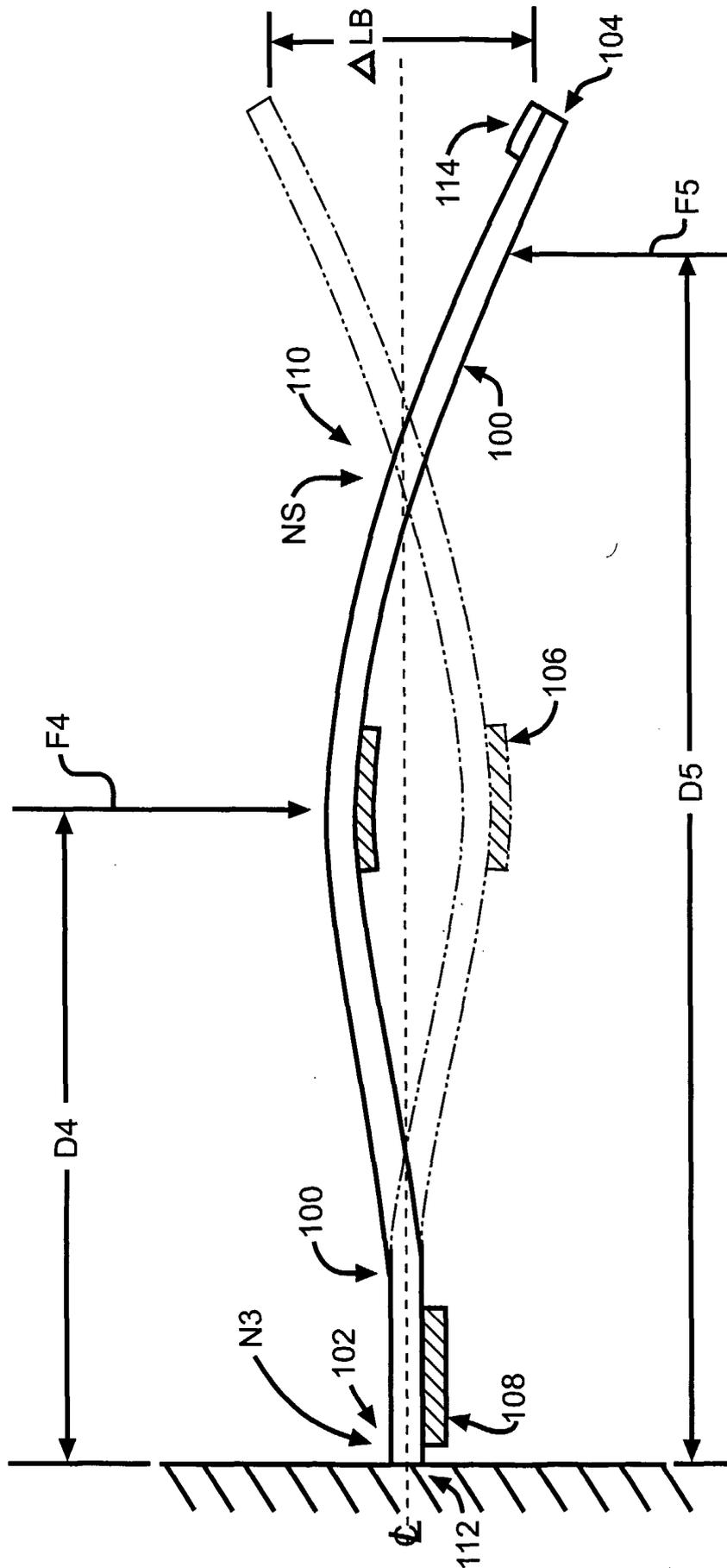


FIG.5

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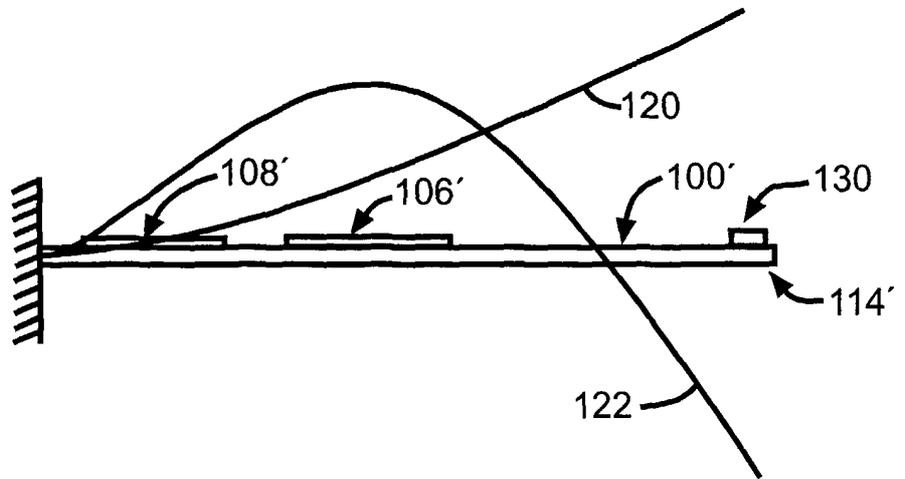


FIG. 6

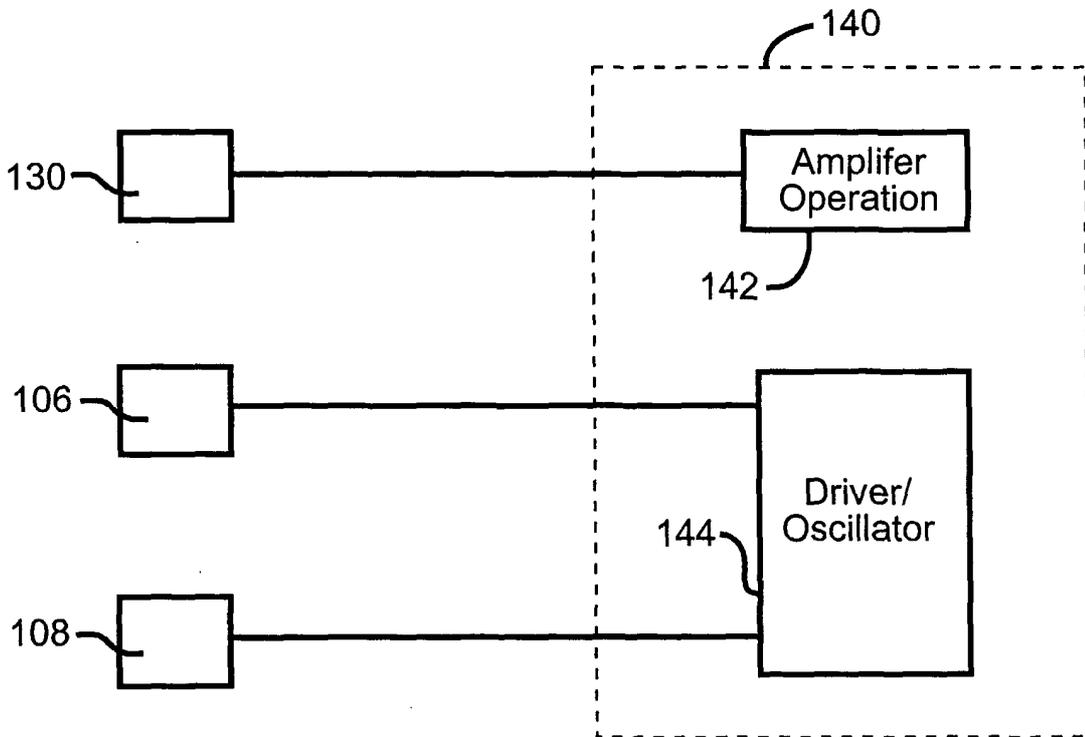


FIG. 7

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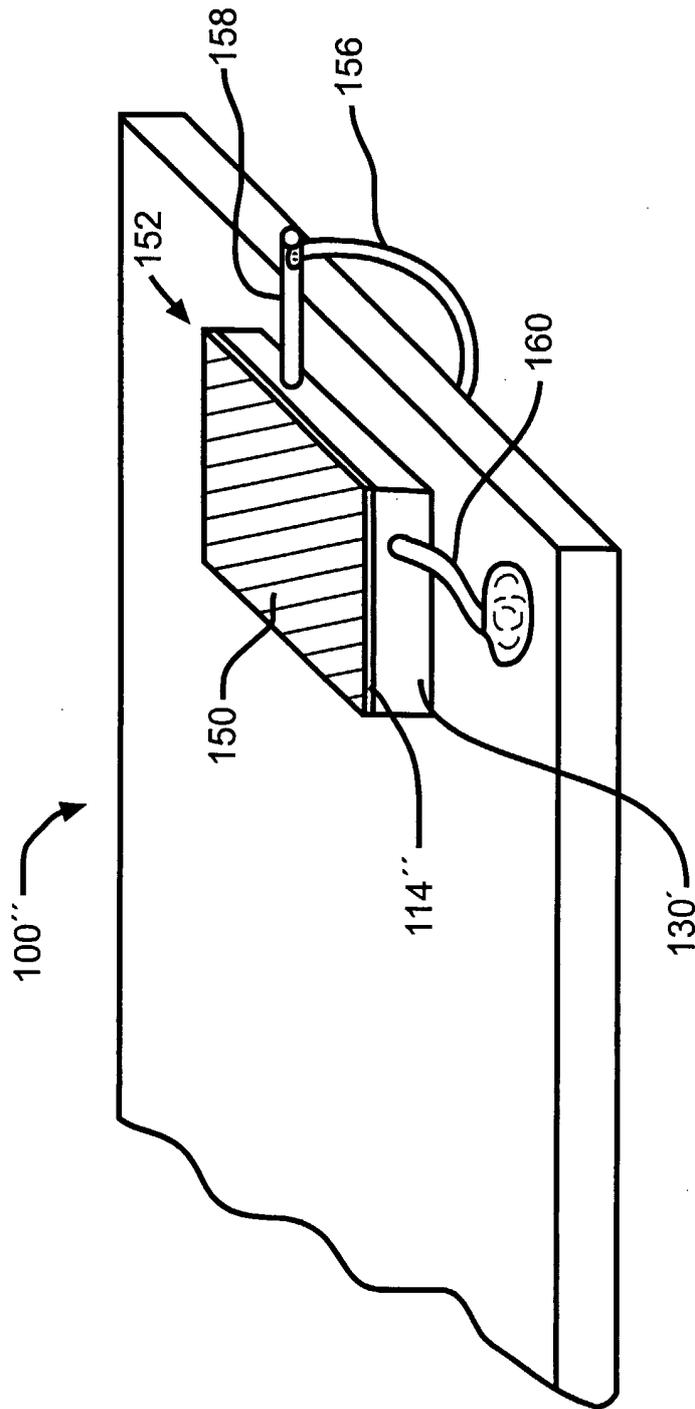


FIG. 8

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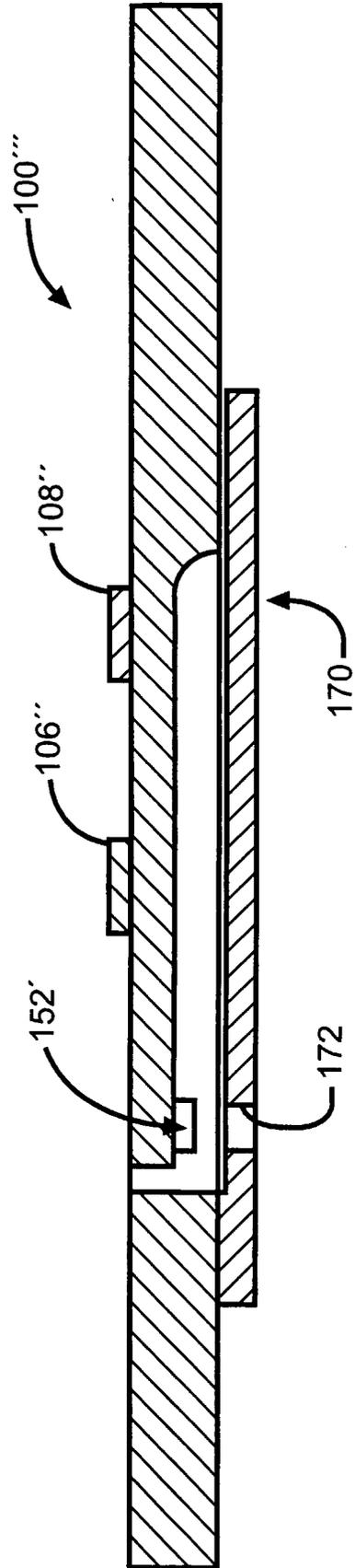


FIG. 9

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