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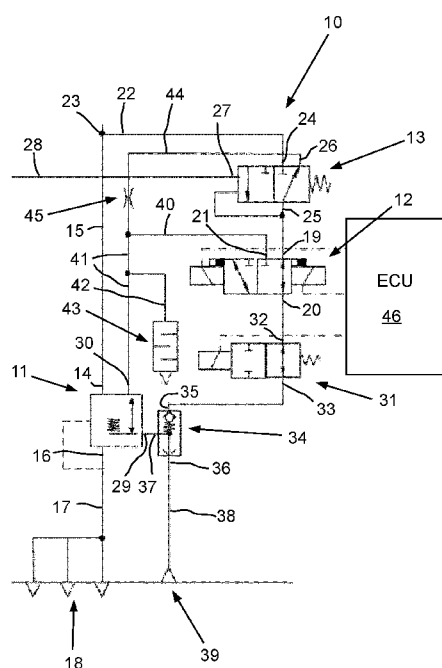


Fig. 3

(57) Abstract: The invention relates to an electro-pneumatic hand brake system (10) for a vehicle with spring-loaded brakes, comprising a pneumatic relay valve (11), a bistable solenoid valve (12) for controlling the relay valve (11) and a control unit (46) for controlling the bistable solenoid valve (12), wherein an outlet of the relay valve (11) is connected to ports (18) for the spring-loaded brakes, and wherein an inlet of the bistable solenoid valve (12) is provided for connecting to a line (22) carrying ventilation pressure, the electro-pneumatic hand brake system (10) further comprising a self-retaining switching valve with a control input that is provided before the inlet of the bistable solenoid valve (12), wherein • a) the self-retaining switching valve (13) can be moved against the force of a restoring element from a blocking position into a flow-through position when there is sufficient pressure at its control input, and • b) an outlet of the self-retaining switching valve (13) is connected to the inlet of the bistable solenoid valve (12), and • c) the control input of the self-retaining switching valve (13) is also connected to a signal line via which a signal pressure can be fed to the control input as a control pressure at least for a short time, and • d) the self-retaining switching valve (13) in its flow-through position connects its outlet with its inlet and • e) the self-retaining switching valve (13) in its blocking position connects its outlet (25) with a bleed outlet.

Electro-pneumatic hand brake system

Description

The invention relates to an electro-pneumatic hand brake system pursuant to the preamble of claim 1. In particular, it is an electro-pneumatic hand brake system comprising a pneumatic relay valve, a bistable solenoid valve for controlling the relay valve and a control unit for controlling the bistable solenoid valve, wherein an outlet of the relay valve is connected to ports for the spring-loaded brakes, and wherein an inlet of the bistable solenoid valve is provided for connecting to a line carrying ventilation pressure.

Modern commercial vehicles have a pneumatic brake installation with an electronic brake system. The pneumatic brake installation includes spring-loaded brakes used as locking brakes. These are also known as parking brakes. The parking brakes act by spring force and can be released by the ventilation of spring brake cylinders or locked by bleeding them.

The spring brake cylinders can be combined with service brake cylinders so that the spring-loaded brakes and service brakes act on the same brake pistons. Appropriate constructional measures can be taken to avoid mechanical overloading of the brake pistons by the addition of braking forces from the service brakes and the spring brakes. Should the service brakes be applied while the parking brakes are engaged, the spring brake cylinders are ventilated at the same time to avoid the addition of braking forces.

Valves for regulating the brake pressure are controlled electronically via the electronic brake system. For safety reasons, pneumatic control of the valves regulating the brake pressure is also provided.

The parking brakes are also controlled electronically. By actuating a solenoid valve, the ventilation or bleeding of the spring brake cylinders is regulated. For safety reasons, the solenoid valve must always assume a unique switching position, which must be maintained in the event of a power supply failure. The solenoid valve is therefore realized as a bistable solenoid valve.

The bistable solenoid valve can be part of a subsystem, designated here as the electro-pneumatic hand brake system. The electrical part of the electro-pneumatic hand brake system can be a subsystem of the electronic brake system. The electro-pneumatic hand brake system may be provided with a dedicated electronic control unit. However, the latter can also be integrated in a brake control unit or another control unit.

The pneumatic part of the electro-pneumatic hand brake system of a motor vehicle is typically connected to a brake circuit III, while brake circuits I and II comprise the service brakes. Brake circuit III usually also supplies a trailer with supply pressure.

Brake circuit I and II, on one hand, and brake circuit III, on the other, are connected to each other via a so-called bleed back function. If the pressure drops in one of the brake circuits I and II, the pressure in brake circuit III also drops, so that the spring brake cylinders are automatically bled and the spring-loaded brakes take effect. By taking advantage of the bleed back function, the parking brakes can be activated by the operator of the vehicle applying the service brakes several times, thus releasing pressure in the brake circuits I and II, in particular when the engine is switched off or the generation of compressed air is stopped.

If the previously described case occurs during travel, the bistable solenoid valve is in its travel position. This means that even after the vehicle has come to a standstill, the spring brake cylinders are ventilated and the parking brakes are released as soon as the pressure in brake circuit III is sufficiently high again. If the vehicle is standing on an inclined surface, it may start to roll. Depending on the surroundings, this can lead to dangerous situations, for example under the following conditions:

- The electrical system continues to fail or is malfunctioning. The operator keeps the engine running so that compressed air continues to be delivered and the pressure in the brake circuits increases again.
- The operator had turned off the engine before reducing the pressure in brake circuit III by applying the service brakes and actuating the bleed back function. After some time, the operator restarts the engine in order to activate the ventilation in the cab. The pressure in the brake circuits increases again.
- The operator leaves the cab with the engine running in order to check the electro-pneumatic hand brake.
- The operator leaves the vehicle. A mechanic starts the engine to heat the cab.

In all of these situations the vehicle may start to move because the parking brakes are ventilated again. This leads to unintentional vehicle movement. The situations described above are an incomplete list of examples. Other situations with a similar hazard potential are possible. In all cases, the cause is the still open position of the bistable solenoid valve in the electro-pneumatic hand brake system.

The object of the present invention is to create a system by means of which the dangerous situations described above can be avoided.

To achieve this object, the electro-pneumatic hand brake system has the features of claim 1. In particular, a self-retaining switching valve provided with a control input is provided before the inlet of the bistable solenoid valve, wherein

- a) the self-retaining switching valve can be moved against the force of a restoring element from a blocking position into an flow-through position when there is sufficient pressure at its control input, and
- b) an outlet of the self-retaining switching valve is connected to the inlet of the bistable solenoid valve, and
- c) the control input of the self-retaining switching valve is also connected to a signal line via which a signal pressure can be fed to the control input as a control pressure at least for a short time, and

- d) the self-retaining switching valve in its flow-through position connects its outlet with its inlet, and
- e) the self-retaining switching valve in its blocking position connects its outlet with a bleed outlet.

Accordingly, the self-retaining switching valve is arranged upstream of the bistable solenoid valve. In this context, "self-retaining" means that in the flow-through position and with sufficient pressure at the inlet and/or outlet, the switching valve retains its flow-through position regardless of the pressure at the control input. After the switching valve has assumed its blocking position, sufficient pressure must be present at the control input in order to move the switching valve back into the flow-through position. Once the flow-through position has been attained, the pressure at the control input can decrease again. The switching valve is then independent of the pressure at the control input. The self-retaining switching valve thus prevents ventilation of the spring brake cylinder, even when the bistable solenoid valve is still in the flow-through position.

The control input of the self-retaining switching valve can be controlled in different ways. For example, a brake system line carrying a supply pressure can be connected to the control input of the self-retaining switching valve via a line. This line is switched through by a solenoid valve provided that electrical energy is available and/or the solenoid valve is controlled by a control unit. The activation of the solenoid valve via the control unit can be triggered by auxiliary conditions implemented in the software of the control unit or selectively by the operator.

In a further aspect of the invention, the self-retaining switching valve can assume its blocking position as soon as the pressure at its outlet falls below a limit value. Here the self-retaining function of the switching valve is linked to the pressure at the outlet of the switching valve. In particular, the limit value is 1.2 to 2.5 bar, preferably 1.5 bar.

In a further aspect of the invention, the bleed outlet of the self-retaining switching valve can be connected to a bleed outlet of the bistable solenoid valve via a first

bleed line and a second bleed line connected thereto. As a result, no dedicated bleed outlet is required. The two ventilation lines can also be different sections of a single bleed line.

In a further aspect of the invention, the first bleed line can be provided with a throttle. This prevents sudden bleeding when changing from the flow-through position to the blocking position of the self-retaining switching valve. In this way, feedback can also be avoided.

In a further aspect of the invention, the first bleed line and the second bleed line can be connected via a third bleed line to a bleed outlet of the relay valve. In this way, a common bleed system can be utilized.

In a further aspect of the invention, the third bleed line or the second bleed line can be connected to a bleed. This applies in particular if the first bleed line is equipped with a throttle.

In a further aspect of the invention, the input of the self-retaining switching valve can be connected to an input of the relay valve. This allows both inputs to be connected to a common line carrying a ventilation pressure. The ventilation pressure preferably corresponds to a pressure in brake circuit III, which is also designated as the parking brake circuit, or to a supply pressure in the vehicle braking system.

In a further aspect of the invention, the self-retaining switching valve can be a 3/2-way valve. Such valves are well-known, widely used and economical.

In a further aspect of the invention, the bistable solenoid valve can be a 3/2-way valve. This type of valve is also widely used and economical.

In a further aspect of the invention, an outlet of the bistable solenoid valve can be connected to a control input of the relay valve.

In a further aspect of the invention, an electro-magnetic switching valve can be switched between the output of the bistable solenoid valve and the control input of the relay valve, it being possible for said switching valve to switch from a flow-through position to a blocking position. The switching valve is preferably a 2/2-way valve. With the electronic switching valve it is possible to modulate the pressure applied to the control input of the relay valve. In this case, the bistable solenoid valve is only indirectly connected to the control input of the relay valve, i.e. by the intermediate connection of the electromagnetic switching valve.

In a further aspect of the invention, the electromagnetic switching valve without current supply can be in the flow-through position. The switching valve is monostable and, in the case of a power failure for example, assumes the flow-through position.

In a further aspect of the invention, a shuttle valve can be connected between the outlet of the bistable solenoid valve and the control input of the relay valve, wherein

- a) an outlet of the shuttle valve is connected to the control input of the relay valve, and
- b) a first inlet of the shuttle valve is directly or indirectly connected to the outlet of the bistable solenoid valve, and
- c) a second inlet of the shuttle valve is connected to a pressure inlet carrying a service brake pressure.

At this point the shuttle valve acts as an overload protection for service brakes, which are also equipped with spring-loaded brake cylinders. Via the shuttle valve, the spring-loaded brake cylinders are increasingly ventilated with increasing service brake pressure, so that a mechanical overload of brake pistons and/or other brake parts is avoided.

In a further aspect of the invention, the bleed outlet of the self-retaining switching valve can bleed directly into the atmosphere. In addition, a throttle can be provided in the self-retaining switching valve in the region of the bleed outlet.

The subject matter of the invention is also an electronic braking system pursuant to claim 15, namely for a vehicle with spring-applied brakes, with an electro-pneumatic hand brake system according to any one of the claims 1 to 14. The electro-pneumatic hand brake system according to the invention is preferably used in combination with the electronic brake system.

Finally, the subject matter of the invention is also a vehicle with a pneumatic braking installation, spring-loaded brakes and an electronic brake system pursuant to claim 15.

Further features of the invention can be found in the remaining description and in the claims. Advantageous exemplary embodiments of the invention are explained in the following with the help of drawings, where:

Fig. 1 shows a circuit diagram of an electro-pneumatic hand brake system, in the driving position,

Fig. 2 shows the circuit diagram of Fig. 1, but in the parking position,

Fig. 3 shows a slightly supplemented circuit diagram, as shown in Fig. 1, in a malfunction position following a driving position.

Fig. 1 shows a circuit diagram of an electro-pneumatic hand brake system 10 for a vehicle (not shown) with spring-loaded brakes. The hand brake system 10 is preferably a subsystem of an electronic brake system for a pneumatic brake installation. The vehicle is preferably a motor vehicle.

Important components of the hand brake system 10 are a pneumatic relay valve 11, a bistable solenoid valve 12 and a self-retaining pneumatic switching valve 13.

The relay valve 11 is connected with an inlet 14 to a line 15 carrying ventilation pressure. From an outlet 16 of the relay valve 11 a line 17 leads to ports 18 for spring brake cylinders (not shown). With relay valve 11 switched through, the spring brake cylinders are ventilated or at least can be ventilated.

The bistable solenoid valve 12 is preferably a 3/2-way valve and is electromagnetically switchable between the flow-through position shown in Figs. 1 and 3 and the blocking position shown in Fig. 2. In the flow-through position, inlet 19 and outlet 20 are connected. In the blocking position the outlet 20 is connected to a bleed outlet 21.

A second line 22 carrying ventilation pressure is indirectly connected to the inlet 19. The lines 15 and 22 carrying ventilation pressure can be connected to one another via a junction 23. The ventilation pressure preferably corresponds to the supply pressure in a brake circuit provided for the hand brake system 10. This is also known as the parking brake circuit. Lines 15, 22 are connected to ports (not shown) of the parking brake circuit.

The self-retaining, pneumatic switching valve 13 has inlet 24, outlet 25, bleed outlet 26 and control input 27 and is preferably realized as a 3/2-way valve. In the flow-through position, the switching valve 13 connects the line 22 with the inlet 19 of the bistable solenoid valve 12. In the blocking position of switching valve 13, the outlet 25 is switched to the bleed outlet 26, see Fig. 3. As long as there is a pressure above a limit value at outlet 25, the switching valve 13 remains in the flow-through position against the force of a restoring element and is self-retaining there. The limit value is approximately 1.2 to 2.5 bar, preferably 1.5 bar.

The control input 27 is connected to a line 28, which can be supplied with control pressure in a manner not shown, at least for a short time, and thus has the function of a signal line. As control pressure, for example, reserve pressure can be fed into line 28 at a suitable point via a switchable valve (not shown). The valve can be switched manually, pneumatically or electromagnetically, preferably either by an operator or automatically under defined conditions.

The pressure applied to the outlet 20 of the solenoid valve 12 feeds a control input 29 of the relay valve 11. As long as solenoid valve 12 and switching valve 13 are each in the flow-through position and ventilation pressure is present in each of the

lines 15, 22, relay valve 11 is switched through, so that the spring brake cylinder is ventilated via the ports 18.

If there is no pressure at control input 29, the outlet 16 of the relay valve is connected to its bleed outlet 30. The spring brake cylinders are then bled via ports 18 and the relay valve 11, thus ensuring that the parking brakes are activated.

To modulate the pressure between bistable solenoid valve 12 and relay valve 11, an electro-magnetic switching valve 31 is provided, which is preferably a 2/2-way valve and is in the flow-through position without current supply, as shown in all three figures. With the electro-magnetic switching valve 31, the ventilation of the spring brake cylinders can be controlled in steps. In the through-flow position shown, inlet 32 and outlet 33 of the switching valve 31 are connected to each other.

Located here between the electromagnetic switching valve 31 and the relay valve 11 is a shuttle valve 34, also known as an OR valve, which has two inlets 35, 36 and one outlet 37. The inlet 35 is connected to the outlet 33 of the switching valve 31, outlet 37 to the control input 29 of the relay valve 11. The other inlet 36 is connected via a line 38 to a port 39, which is supplied with service brake pressure. When a service brake is actuated, service brake pressure enters the hand brake system 10 via port 39. The higher of the pressures applied to inlets 35, 36 is fed in each case to the control input 29 of the relay valve 11 via shuttle valve 34. This allows the spring brake cylinders to be ventilated depending on the actuation of the service brakes, which prevents mechanical overloading of the brakes.

In the driving position shown in Fig. 1, all four valves 11, 12, 13, 31 are in the flow-through position. The spring-loaded brake cylinders are ventilated with pressure via ports 18.

In the parking position pursuant to Fig. 2, the bistable solenoid valve 12 is in the blocking position, while switching valve 13 and switching valve 31 are preferably in the flow-through position. Due to the blocking position of solenoid valve 12, the pressure at the outlet 20 is reduced via the bleed outlet 21. For this purpose, the

bleed outlet 21 is connected to a bleed 43 via a bleed line 40 and further bleed lines 41, 42. Bleed line 41 extends up to the bleed outlet 30 of the relay valve 11, so that bleed 43 is also connected to bleed outlet 30.

The bleed outlet 26 of the self-retaining switching valve 13 is connected to bleed line 40 and/or bleed line 41 via a bleed line 44. This also creates a connection from the bleed outlet 26 of the self-retaining switching valve 13 to the bleed 43.

Here the bleed line 44 is provided with a throttle 45. Accordingly, when the outlet 25 is bled via bleed outlet 26 in the blocking position of switching valve 13, the pressure in the bleed line 44 adjacent to the switching valve 13 is higher than in bleed line 41.

In another embodiment (not shown), the bleed outlet 26 opens to the atmosphere. Line 44 is not present. A throttle can be integrated in the switching valve 13 so that the pressure at outlet 25 is not reduced too much when bleeding.

In Fig. 3 a control unit (ECU) 46 is additionally shown. This is either provided specifically for the hand brake system 10 or is a component of a brake control unit (not shown) or another electronic control unit in the associated vehicle. Here the control unit 46 is used to control the functions of the bistable solenoid valve 12 and the electromagnetic switching valve 31. The switching status according to Fig. 3 proceeds from the following situation:

The hand brake system 10 is in the driving position, as shown in Fig. 1. Solenoid valve 12 and switching valves 13, 31 are in the flow-through position. The operator has been informed of a malfunction of the electronic braking system or the electronic braking system has failed completely or partially. The parking brakes can no longer be actively applied due to the failure. For safety reasons, in such a situation the operator should use the service brakes to bring the vehicle to a standstill and then apply the service brakes several times to reduce the pressure in the service brake circuits so that a bleed back function also reduces the pressure in the parking brake circuit. This will bleed the spring brake cylinders.

There is also a pressure drop in the region of outlet 25 of the self-retaining switching valve 13. This causes the switching valve 13 to move from its flow-through position to the blocking position as shown in Fig. 3. There is also no pressure at control input 27, since pressure is only fed in line 28 upon manual actuation or automatic actuation dependent on specific conditions. This means that the hand brake system cannot automatically ventilate the spring brake cylinders and thus release the parking brakes as soon as sufficient supply pressure is available again. Instead, a pressure impulse at control input 27 via line 28 is first required to ventilate the parking brake cylinders again.

In the hand brake system 10, the self-retaining switching valve 13 has the function of a safety device in case the situation described above arises. An unintentional release of the parking brakes by restarting the engine with a corresponding build-up of the supply pressure is avoided. The parking brakes remain active until control input 27 is activated with pressure via line 28, in particular by deliberate manual intervention or automatically under defined conditions.

Control unit 46 is connected to a hand brake lever (not shown) for the operator. When the hand brake lever is actuated, the control unit 46 receives the command to activate or deactivate the parking brakes.

List of reference numbers (part of the description)

10 electro-pneumatic hand brake system	33 self-retaining switching valve outlet
11 relay valve	34 shuttle valve
12 bistable solenoid valve	35 shuttle valve inlet
13 self-retaining switching valve	36 shuttle valve inlet
14 relay valve inlet	37 shuttle valve outlet
15 line	38 line
16 relay valve outlet	39 port
17 line	40 bleed line
18 ports	41 bleed line
19 solenoid valve inlet	42 bleed line
20 solenoid valve outlet	43 bleed
21 solenoid valve bleed outlet	44 bleed line
22 line	45 throttle
23 junction	46 control unit
24 self-retaining switching valve inlet	
25 self-retaining switching valve outlet	
26 self-retaining switching valve bleed outlet	
27 self-retaining switching valve control input	
28 line	
29 relay valve control input	
30 relay valve bleed outlet	
31 electromagnetic switching valve	
32 self-retaining switching valve inlet	

Claims

1. An electro-pneumatic hand brake system (10) for a vehicle with spring-loaded brakes,
comprising a pneumatic relay valve (11), a bistable solenoid valve (12) for controlling the relay valve (11) and a control unit (46) for controlling the bistable solenoid valve (12),
wherein an outlet of the relay valve (11) is connected to ports (18) for the spring-loaded brakes,
and wherein an inlet (19) of the bistable solenoid valve (12) is provided for connecting to a line (22) carrying ventilation pressure,
characterized by a self-retaining switching valve (13) with a control input (27) being provided before the inlet (19) of the bistable solenoid valve (12), wherein
- a) the self-retaining switching valve (13) can be moved against the force of a restoring element from a blocking position into a flow-through position when there is sufficient pressure at its control input (27), and
 - b) an outlet of the self-retaining switching valve (13) is connected to the inlet (19) of the bistable solenoid valve (12), and
 - c) the control input (27) of the self-retaining switching valve (13) is also connected to a signal line (28) via which a signal pressure can be fed to the control input (27) as a control pressure at least for a short time, and
 - d) the self-retaining switching valve (13) in its flow-through position connects its outlet (25) with its inlet (24), and
 - e) the self-retaining switching valve (13) in its blocking position connects its outlet (25) with a bleed outlet (26).
2. The system as claimed in claim 1, characterized in that the self-retaining switching valve (13) assumes its blocking position as soon as the pressure at its outlet (25) falls below a limit value.
3. The system as claimed in claim 1 or 2, characterized in that the bleed outlet (26) of the self-retaining switching valve (13) is connected to a bleed outlet (21) of the

bistable solenoid valve (12) via a first bleed line (44) and a second bleed line (40) connected thereto.

4. The system as claimed in claim 3, characterized in that the first bleed line (44) is provided with a throttle (45).

5. The system as claimed in claim 3 or 4, characterized in that the first bleed line (44) and the second bleed line (40) are connected via a third bleed line (41) to a bleed outlet (30) of the relay valve (11).

6. The system as claimed in claim 5, characterized in that the third bleed line (41) or the second bleed line (40) are connected to a bleed (43).

7. The system as claimed in any one of the claims 1 to 6, characterized in that the input (24) of the self-retaining switching valve (13) is connected to an input (14) of the relay valve (11).

8. The system as claimed in any one of the claims 1 to 7, characterized in that the self-retaining switching valve (13) is a 3/2-way shuttle valve.

9. The system as claimed in any one of the claims 1 to 8, characterized in that the bistable solenoid valve (12) is a 3/2-way valve.

10. The system as claimed in any one of the claims 1 to 9, characterized in that an outlet (20) of the bistable solenoid valve (12) is connected to a control input (29) of the relay valve (11).

11. The system as claimed in claim 10, characterized in that an electro-magnetic switching valve (31) is connected between the output (20) of the bistable solenoid valve (12) and the control input (29) of the relay valve (11), it being possible for said switching valve (31) to switch from a flow-through position to a blocking position.

12. The system as claimed in claim 11, characterized in that the electromagnetic switching valve (31) without current supply is in the flow-through position.

13. The system as claimed in any one of the claims 10 to 12, characterized in that a shuttle valve (34) is connected between the outlet (20) of the bistable solenoid valve (12) and the control input (29) of the relay valve (11), wherein

- a) an outlet (37) of the shuttle valve (34) is connected to the control input (29) of the relay valve (11), and
- b) a first inlet (35) of the shuttle valve (34) is directly or indirectly connected to the outlet (20) of the bistable solenoid valve (12), and
- c) a second inlet (36) of the shuttle valve (34) is connected to a pressure inlet (39) carrying a service brake pressure.

14. The system as claimed in any one of the claims 1 to 13, characterized in that the bleed outlet (26) of the self-retaining switching valve (13) bleeds directly into the atmosphere.

15. An electronic braking system for a vehicle with spring-applied brakes, with an electro-pneumatic hand brake system (10) as claimed in any one of the claims 1-14.

16. Vehicle with a pneumatic brake installation, spring-loaded brakes and an electronic brake system as claimed in claim 15.

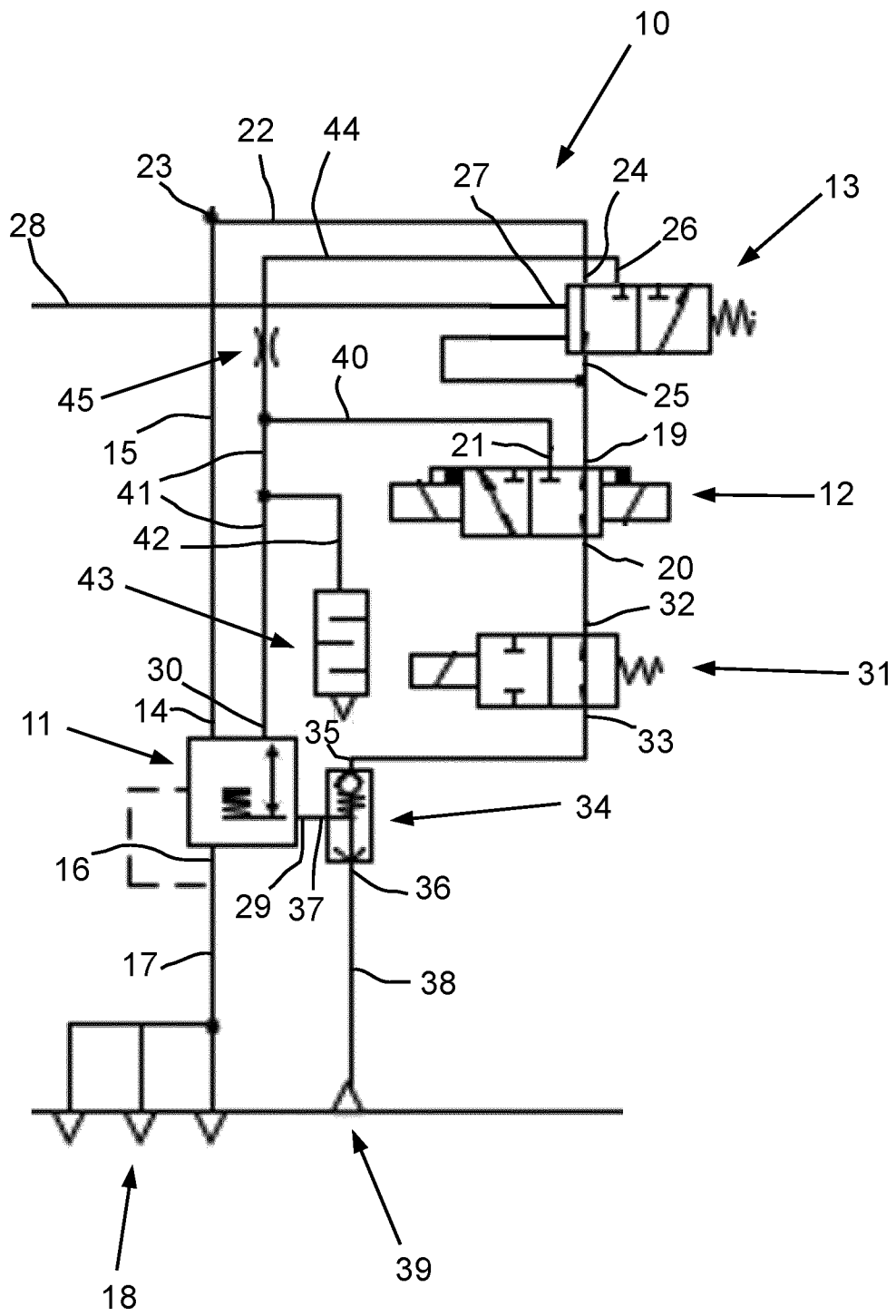


Fig. 1

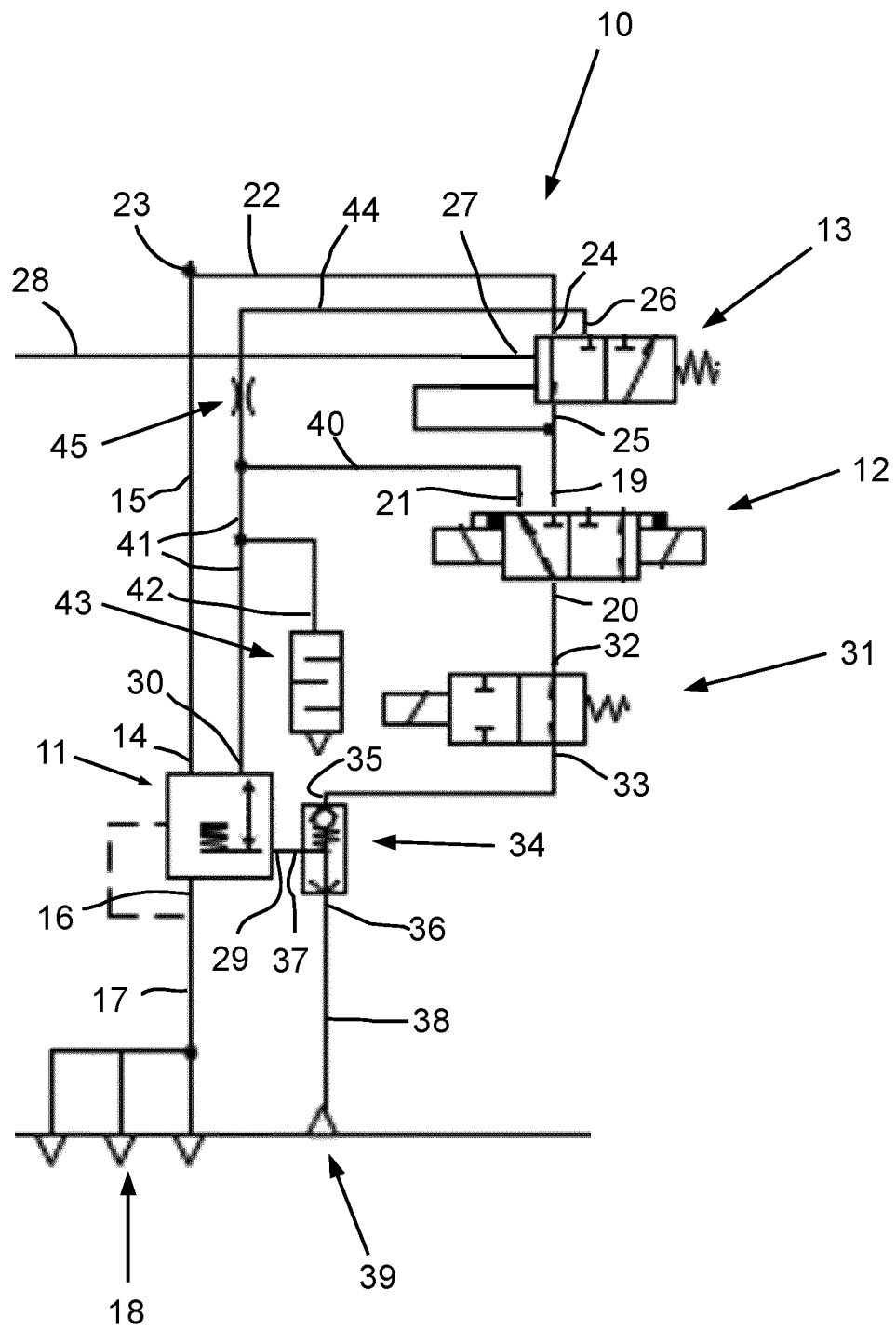


Fig. 2

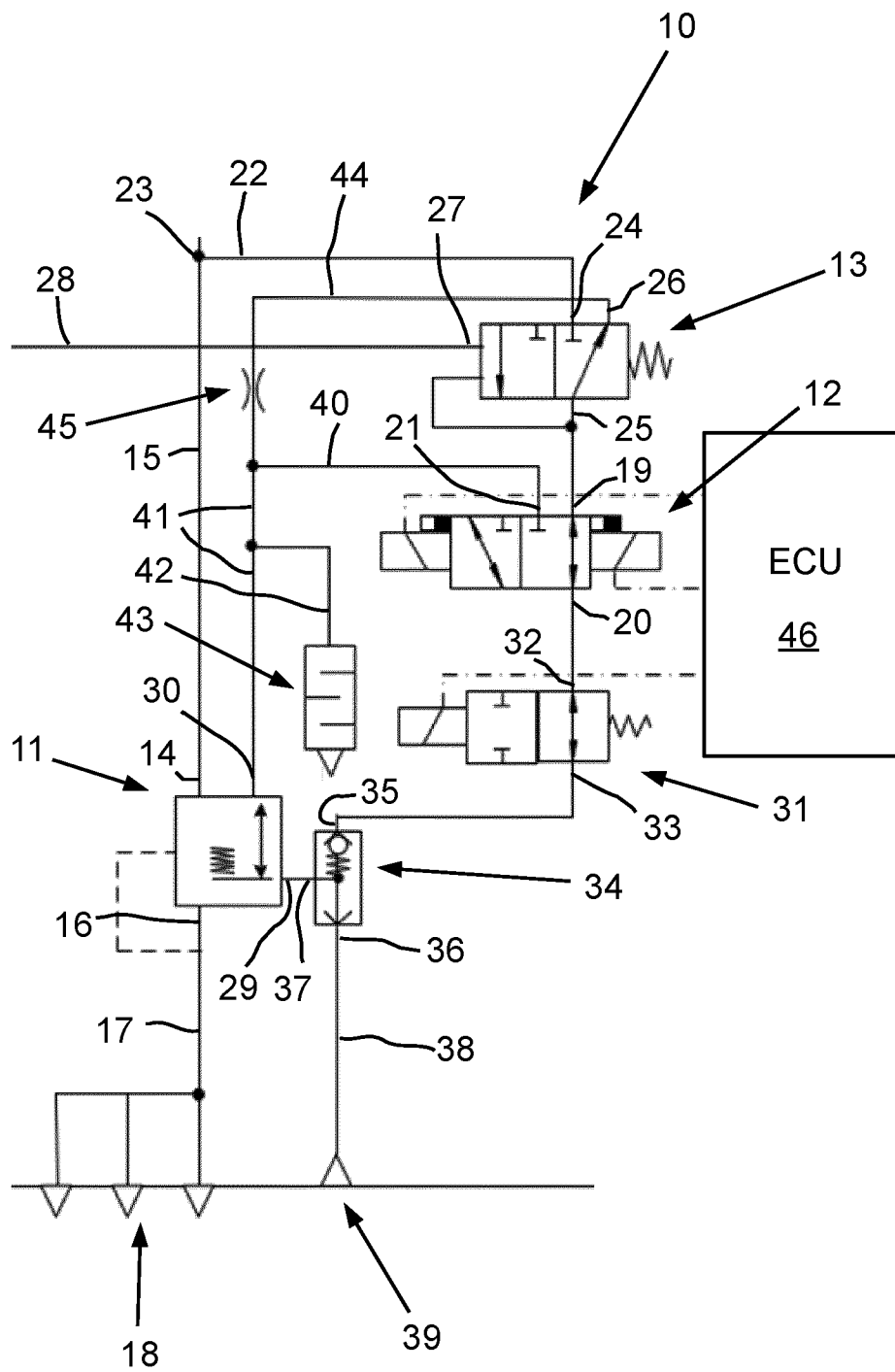


Fig. 3

INTERNATIONAL SEARCH REPORT

International application No
PCT/EP2020/087359

A. CLASSIFICATION OF SUBJECT MATTER
INV. B60T13/38 B60T13/68 B60T17/22
ADD.

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)
B60T

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

EPO-Internal

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	DE 10 2005 058799 A1 (WABCO GMBH [DE]) 14 June 2007 (2007-06-14)	1,7-16
A	paragraphs [0029], [0062] - [0066]; figures 1-3	2-6
E	----- WO 2021/058234 A1 (WABCO EUROPE BVBA [BE]) 1 April 2021 (2021-04-01)	1,2, 9-12, 14-16
	the whole document	
A	----- EP 2 998 177 A1 (KNORR BREMSE SYSTEME FÜR NUTZFAHRZEUGE GMBH [DE]) 23 March 2016 (2016-03-23) figure 3	13



Further documents are listed in the continuation of Box C.



See patent family annex.

* Special categories of cited documents :

"A" document defining the general state of the art which is not considered to be of particular relevance

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"X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone

"Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art

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Date of the actual completion of the international search

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