METHOD FOR THE GASFICATION OF A LIQUID HYDROCARBON STREAM AND AN APPARATUS THEREFOR

Fig. 1

Abstract: The present invention provides a method of and apparatus for the gasification of a liquid hydrocarbon stream (5) to provide a gaseous hydrocarbon stream (55) and power, the method comprising at least the steps of: (a) pressurising a liquid hydrocarbon stream (5) at a feed pressure to provide a pressurised liquid hydrocarbon stream (15) at a cold recovery pressure; (b) heat exchanging the pressurised liquid hydrocarbon stream (15) against one or more vapour recycle streams (115) to provide a warmed liquid hydrocarbon stream (35) and one or more liquefied recycle streams (125); (c) passing the one or more liquefied recycle streams (125), after pressurisation if necessary, to one or both of the liquid hydrocarbon stream (5) and the pressurised liquid hydrocarbon stream (15); (d) pressurising one or more portions of the warmed liquid hydrocarbon stream (35) to provide one or more pressurised warmed liquid hydrocarbon streams (45), at least one of which is at a minimum pipeline pressure or higher, against one or more heat sources (345) to provide one or more gaseous hydrocarbon streams (55), at least one of which is at the minimum pipeline pressure or higher; (f) dynamically expanding at least a part (55, 55a, 55e) of at least one of the one or more gaseous hydrocarbon streams (55) to provide power and one or more expanded gaseous hydrocarbon streams (115), wherein at least a part of said one or more expanded gaseous hydrocarbon streams forms at least a part of the one or more vapour recycle stream (115).
METHOD FOR THE GASIFICATION OF A LIQUID HYDROCARBON 
STREAM AND AN APPARATUS THEREFOR 

The present invention provides a method for the 
gasification of a liquid hydrocarbon stream, such as a 
liquefied natural gas (LNG) stream, to provide a gaseous 
hydrocarbon stream, such as a natural gas stream, and 

power, and an apparatus therefor. The apparatus and 
method are particularly suitable for use at a LNG 
regasification facility. 

LNG is usually primarily liquefied methane 
containing varying quantities of ethane, propane and 

butanes with trace quantities of pentanes and heavier 
hydrocarbon components. Usually the LNG is low in 
aromatic hydrocarbons and non-hydrocarbons such as H₂O, 
N₂, CO₂, H₂S and other sulphur compounds, because these 
compounds have been removed at least partially before 

liquefying the natural gas stream, which is then stored 
and transported in liquid form. 

It is desirable to liquefy natural gas for a number 
of reasons. As an example, natural gas can be stored and 
transported over long distances more readily as a liquid 

than in gaseous form, because it occupies a smaller 
volume. It may not need to be stored at high pressures. 
Such liquefied natural gas can be stored at atmospheric 
pressure if maintained at cryogenic temperatures, such as 
at -160 °C or below. Alternatively, liquefied natural gas 
may be stored at temperatures above -160 °C if it is 

pressurised above atmospheric pressure. 

In order to regasify the LNG stream, it is usually 
pressurised and vaporised. If desired, a selected amount
of one or more additional components, for example nitrogen, can be added to obtain natural gas having a desired gas quality, for example a selected heating value (i.e. energy content when the natural gas is burned), according to gas specifications or the requirements of a consumer. Furthermore, the heating value of the natural gas may also be adjusted by removing or adding a desired amount of heavier hydrocarbons from natural gas.

A LNG regasification terminal pressurises and vaporises LNG so that pressurised natural gas can be injected into a gas network at ambient temperature. Although different kinds of heat exchangers are available for vaporising LNG, using an ambient heat source such as seawater is a common option. However, vaporising LNG against an ambient heat source represents lost energy. For instance, for each 1 MTPA (million metric tons per annum) LNG vaporised, a heat flow of between 20 and 30 MW is lost. This could be used for a number of practical purposes, such as power generation. It would be desirable if this unused a heat flow could be harnessed to generate electricity, rendering the regasification facility self-sufficient in terms of its power requirements, such that power input from an external electricity grid or the burning of hydrocarbon fuels would not be required. A gasification terminal which does not require the direct or indirect burning of hydrocarbon fuels for its operation will be carbon dioxide neutral.

WO 2007/011921 discloses a method of generating power in a LNG regasification facility utilising LNG. In the disclosed method, natural gas is used as a working fluid in which at least a portion of vaporised natural gas is expanded in a turbine to produce power and expanded vaporised natural gas. The expanded vaporised
natural gas is then condensed using cold from the liquefied natural gas, and subsequently combined with liquefied natural gas. The combined liquefied and condensed natural gas is then vaporised to produce the vaporised natural gas.

The method of WO 2007/011921 heats the portion of vaporised natural gas used in the expansion step to a temperature of 232 °C (450 °C) prior to expansion in the turbine. Although the disclosed method is said to be capable of generating 5 MW, this is not attainable using an ambient heat source, and additional energy input is required. In order to heat the vaporised natural gas to temperatures of 232 °C, a heat source such as a flue gas from a gas turbine, a waste heat recovery unit and/or a fired heater is required. The aforementioned heat sources all rely upon, for instance, the combustion of a hydrocarbon fuel and thus produce carbon dioxide emissions.

A problem of this known method of gasifying is that the combustion of hydrocarbons and the associated carbon dioxide emissions are required in order to produce sufficient power to meet the needs of the regasification facility.

The present invention seeks to address this problem by providing a method of regasifying a liquid hydrocarbon stream which is self-sustaining in terms of power consumption in steady state operation and is carbon dioxide neutral.

In particular, the method and apparatus of the present invention can generate sufficient power from the gasification process to operate an LNG import terminal in which LNG is converted into natural gas for consumption. Such an import terminal need not produce the carbon
dioxide emissions normally resulting from the generation of electricity required to operate the terminal and may be carbon dioxide emissions neutral as a result of energy generation from the gasification of the LNG.

It is desirable to minimise the carbon dioxide emissions from a facility, and more preferably to eliminate such emissions entirely to provide a "zero carbon dioxide emissions" regasification method. Such a facility would be autonomous in terms of its power requirements and give rise to carbon credits as a result of its carbon dioxide free power generation.

In a first aspect, the present invention provides a method for the gasification of a liquid hydrocarbon stream to provide a gaseous hydrocarbon stream and power, the method comprising at least the steps of:

(a) pressurising a liquid hydrocarbon stream at a feed pressure to provide a pressurised liquid hydrocarbon stream at a cold recovery pressure;

(b) heat exchanging the pressurised liquid hydrocarbon stream against one or more vapour recycle streams to provide a warmed liquid hydrocarbon stream and one or more liquefied recycle streams;

(c) passing the one or more liquefied recycle streams, after pressurisation if necessary, to one or both of the liquid hydrocarbon stream and the pressurised liquefied recycle stream;

(d) pressurising one or more portions of the warmed liquid hydrocarbon stream to provide one or more pressurised warmed liquid hydrocarbon streams, at least one of which is at a minimum pipeline pressure or higher;

(e) vaporising one or more of the pressurised warmed liquid hydrocarbon streams, at least one of which is
at the minimum pipeline pressure or higher, against one or more heat sources to provide one or more gaseous hydrocarbon streams, at least one of which is at the minimum pipeline pressure or higher; dynamically expanding at least a part of at least one of the one or more gaseous hydrocarbon streams to provide power and one or more expanded gaseous hydrocarbon streams, wherein at least a part of said one or more expanded gaseous hydrocarbon streams forms at least a part of the one or more vapour recycle streams.

In a further aspect, the present invention provides an apparatus for the gasification of a liquid hydrocarbon stream to provide a gaseous hydrocarbon stream and power, said apparatus comprising at least:

- a liquid hydrocarbon stream pump to pressurise a liquid hydrocarbon stream at a feed pressure to provide a pressurised liquefied hydrocarbon stream at a cold recovery pressure;
- a hydrocarbon stream heat exchanger to heat exchange the pressurised liquid hydrocarbon stream against one or more vapour recycle streams to provide a warmed liquid hydrocarbon stream and one or more liquefied recycle streams;
- a warmed hydrocarbon stream pump to pressurise one or more portions of the warmed liquid hydrocarbon stream to provide one or more pressurised warmed liquid hydrocarbon streams at least one of which is at a minimum pipeline pressure or higher;
- a heat source to vaporise one or more of the pressurised warmed liquid hydrocarbon streams, at least one of which is at a minimum pipeline pressure.
or higher, to provide one or more gaseous hydrocarbon streams;
- a recycle turbine to dynamically expand at least a part of at least one of the one or more gaseous hydrocarbon streams to drive a shaft and provide one or more expanded gaseous hydrocarbon streams, at least a part of which forms at least a part of the one or more vapour recycle streams; and
- one or more hydrocarbon stream mixing devices (20) to introduce the one or more liquefied recycle streams (125), or a stream derived therefrom (135), to the liquid hydrocarbon stream (5) or the pressurised liquefied hydrocarbon stream (15).

One major advantage of the present invention is that the method and apparatus operate at two pressure levels. An initial pressurisation step (a) is carried out to increase the pressure of the liquid hydrocarbon stream from a feed pressure to provide a pressurised liquid hydrocarbon stream at a cold recovery pressure.

The cold recovery pressure is the pressure used to carry out the heat exchange of the pressurised liquid hydrocarbon stream against the vapour recycle stream. This heat exchange warms the liquid hydrocarbon stream and liquefies the vapour recycle stream.

A second pressurisation of the warmed liquid hydrocarbon stream is then carried out in step (d) to bring the pressure of at least one portion of this stream to a minimum pipeline pressure or higher. The minimum pipeline pressure can be the pressure of the natural gas network to which the vaporised natural gas is to be supplied.

Second pressurisation step (d) enables the heat exchange in step (b) to be carried out at any desired
("cold recovery") pressure above the feed pressure. This allows an optimal LNG cold recovery pressure to be used in the heat exchange. In particular, the specific heat capacity of LNG decreases with pressure, such that LNG at lower pressures can provide additional cooling capacity compared to an identical composition at higher pressure. It is therefore desirable to carry out the heat exchange in step (b) at a cold recovery pressure which is less than the minimum pipeline pressure, in order to provide optimal cooling to the one or more vapour recycle streams in step (b). 

It has been found that cold recovery pressures of less than 60 bar are particularly advantageous. Such pressures can be significantly less than the minimum pipeline pressure, which can be in excess of 60 bar, and commonly up to 85 bar in some countries. Carrying out the heat exchange step (b) at lower than the pipeline pressure is also advantageous because the heat exchanger can be designed to operate at such lower pressures, reducing CAPEX. Preferably, the cold recovery pressure is less than the critical pressure of the pressurised hydrocarbon stream in order to retain the latent heat capacity of the hydrocarbon for the heat exchange. As used herein, the term "critical pressure" is the pressure of the pressurised hydrocarbon stream (or any other appropriate stream) at which the phase boundary between the gaseous and liquid states ceases to exist.

In contrast, the disclosure of WO 2007/011921 requires that the LNG is pumped to the pipeline pressure prior to the heat exchange against the expanded vaporised natural gas (vapour recycle stream). This means that the heat exchange of the LNG to liquefy the expanded vaporised natural gas must be carried out at the natural
gas export pressure (minimum pipeline pressure). This does not provide any flexibility in the pressure of the LNG during the heat exchange step with the expanded vaporised natural gas, preventing optimisation of the efficiency of the heat exchange step.

In addition, step (c) of the present invention adds the liquefied recycle stream to the liquid hydrocarbon stream or the pressurised liquid recycle stream. This combination of liquefied streams occurs upstream of the heat exchange step (b), such that the cold energy provided to liquefy the vapour recycle stream can be used to liquefy further natural gas vapour.

In contrast, the disclosure of WO 2007/011921 teaches that the condensed expanded vaporised natural gas is combined with the LNG downstream of the heat exchange step, such that the cooling duty of the condensed expanded vaporised natural gas is not utilised in the condensation of the expanded vaporised natural gas stream, leading to a less efficient use of the cold energy of the LNG.

The dynamic expansion of at least a part of at least one of the one or more gaseous hydrocarbon streams in step (f) is used to generate power. It is preferred that the gaseous hydrocarbon stream is expanded to a pressure of between 10 to 30 bars in a turbine.

The power produced is preferably generated as electrical power. The generation of electrical power is advantageous because any excess electricity not required to operate the gasification facility can be exported to other processing units or to an external electricity grid, such as a public electricity grid. If the power is generated as mechanical power, a system of gearing may be required to operate the pumps used in the facility.
The heat sources used herein, including any hydrocarbon streams generated in the gasification method which are used as heat sources, preferably have a temperature greater than the temperature of the stream to be warmed. Preferably, the heat source has a temperature of greater than -40 °C, more preferably greater than -30 °C, still more preferably in the range of 0 to 30 °C. In a preferred embodiment, the upper temperature limit of the heat source is substantially ambient temperature. One or more of the heat sources may be, for example one or both of an ambient heat source such as ambient air or ambient water, and a hydrocarbon stream generated in the gasification method. However it is possible to integrate the heat source with that from a combined power plant.

Embodiments of the present invention will now be described by way of example only and with reference to the accompanying non-limiting drawings in which:

Figure 1 is a diagrammatic scheme of a method of and apparatus for gasifying a liquid hydrocarbon stream according to a first embodiment;

Figure 2a is a diagrammatic scheme of a method of and apparatus for gasifying a liquid hydrocarbon stream according to a second embodiment;

Figure 2b is a diagrammatic scheme of a method of and apparatus for gasifying a liquid hydrocarbon stream according to a third embodiment;

Figure 3 is a diagrammatic scheme of a method of and apparatus for gasifying a liquid hydrocarbon stream according to a fourth embodiment; and

Figure 4 is a diagrammatic scheme of a method of and apparatus for gasifying a liquid hydrocarbon stream according to a fifth embodiment.
For the purpose of this description, a single reference number will be assigned to a line as well as a stream carried in that line. The same reference numbers used in different Figures represent identical lines and streams.

As discussed above, it is often desirable to liquefy hydrocarbon compositions, such as natural gas because it can be stored more conveniently and transported over long distances more readily as a liquid than in gaseous form. The liquefied natural gas can then be regasified at the desired destination and passed, under pressure, to a gas network.

Referring to the drawings, Figure 1 shows a method of, and apparatus 1 for the, gasification of a liquid hydrocarbon stream 5. The liquid hydrocarbon stream 5 may comprise a hydrocarbon or mixture of hydrocarbons, such as a liquefied natural gas stream, or a synthetic hydrocarbon composition, such as that provided by the reaction of synthesis gas in a Fischer-Tropsch process. Liquefied natural gas can be provided by the treatment and liquefaction of natural gas in a manner known in the art.

The liquid hydrocarbon stream 5 is provided at a feed pressure. Conventionally, liquid hydrocarbon, such as liquefied natural gas, is stored under cryogenic conditions. The liquefied gas may be pressurised at temperatures below 0 °C or at a temperature of approximately -160 °C and at or near atmospheric pressure. This storage is discussed in more detail in relation to Figure 2a. The feed pressure of the liquid hydrocarbon stream 5 will be dependent upon the pressure to which the stream is pumped to transport it to the gasification apparatus 1. Preferably, the feed pressure
is less than 8 bar, and more preferably in the range of 1 to less than 8 bar.

In step (a), the liquid hydrocarbon stream 5 at the feed pressure is passed to a liquid hydrocarbon stream pump 10, where it is pressurised to provide a pressurised liquid hydrocarbon stream 15 at a pressure higher than the feed pressure. The liquid hydrocarbon stream pump 10 is preferably an electrical pump. It is intended to raise the pressure of the liquid hydrocarbon stream 5 to the pressure at which the subsequent heat exchange step is to be carried out - the "cold recovery pressure". The cold recovery pressure is preferably in the range of 8 to less than 50 bar, more preferably in the range of 8 to less than 40 bar, even more preferably in the range of 8 to less than 30 bar.

The pressurised liquid hydrocarbon stream 15 is then passed to a hydrocarbon stream heat exchanger 30 in step (b), where it is warmed against one or more vapour recycle streams 115, to provide a warmed liquid hydrocarbon stream 35 and one or more liquefied recycle streams 125. In the embodiment shown in Figure 1, only a single vapour recycle stream 115 and liquefied recycle stream 125 are shown, although two, three or four or more of each stream may be present. Where there are two or more vapour recycle streams 115, these streams are advantageously provided at different pressures. This embodiment is discussed in greater detail in relation to the embodiment of Figure 3.

The hydrocarbon stream heat exchanger 30 may be selected from a shell and tube, a plate and fin and a printed circuit heat exchanger. It is important that the temperature of the pressurised liquid hydrocarbon stream 15 at the cold recovery pressure is not raised above the
bubble point (i.e. the temperature at which the pressurised liquid hydrocarbon stream 15 first produces bubbles of vapour) of the composition during the heat exchange, in order to ensure that the stream remains liquid, more preferably fully liquid. This is because it is more difficult and less efficient to compress a multi-phase vapour/liquid stream than a single phase liquid stream.

The requirement that the pressurised liquid hydrocarbon stream 15 remains a liquid in the hydrocarbon stream heat exchanger 30 places an upper limit on the cooling duty available from the pressurised liquid hydrocarbon stream 15 at a particular cold recovery pressure. Similarly, it is important that sufficient cooling is provided to the vapour recycle stream 115 to liquefy, preferably fully liquefy, the vapour recycle stream 115, such that an upper limit is placed upon the amount of vapour recycle stream which can be liquefied.

The warmed liquid hydrocarbon stream 35 exits the hydrocarbon stream heat exchanger 30 and one or more portions are passed to a warmed hydrocarbon stream pump 40. The warmed hydrocarbon stream pump 40 is preferably an electrical pump. It is preferred that all of the warmed liquid hydrocarbon stream 35 is passed to the warmed hydrocarbon stream pump 40. The warmed hydrocarbon stream pump 40 pressurises the one or more portions of the warmed liquid hydrocarbon stream 35 to provide one or more pressurised warmed liquid hydrocarbon streams 45, at least one of which is at a minimum pipeline pressure or higher. A single pressurised warmed liquid hydrocarbon stream 45 is shown in Figure 1, which will be at or above the minimum pipeline pressure.
The portion or portions of the pressurised warmed liquid hydrocarbon stream(s) 45 at the minimum pipeline pressure or higher can be gasified and passed to a gas consumer. Thus, the term "minimum pipeline pressure" is the required pipeline pressure of the consumer of the vaporised hydrocarbon, such as a gas network. This pressurisation step ensures that the portion or portions of the pressurised warmed liquid hydrocarbon stream 45 intended to provide the network gas is/are at a pressure equal to or higher than that of the pipeline pressure of the gas network. It is significantly more energy efficient to pressurise a liquid stream to the pipeline pressure or above, than a corresponding evaporated gaseous stream.

One or more of the one or more pressurised warmed liquid hydrocarbon streams 45, at least one of which is at the minimum pipeline pressure or higher, can be passed to one or more hydrocarbon stream heaters 50. The hydrocarbon stream heaters 50 are supplied with one or more heat sources 345 to vaporise the pressurised warmed liquid hydrocarbon. The one or more heat sources 345 preferably have a temperature of less than or equal to ambient temperature, such as an ambient air or ambient water sources, such as a seawater stream. The hydrocarbon stream heaters 50 produce one or more gaseous hydrocarbon streams 55, at least one of which will be at the minimum pipeline pressure or higher, and a cooled heat source 355, such as a cooled air or cooled water, such as seawater, stream. In a preferred embodiment the hydrocarbon stream heater 50 is an open rack vaporiser (ORV).

In some cases, the upper pressure limit to which the warmed liquid hydrocarbon stream(s) 35 can be pumped is
determined by the maximum operating pressure of the hydrocarbon stream heater 50. Commonly, open rack vaporisers have a maximum operating pressure of 140 to 150 bar. It is preferred that the pressure of the pressurised warmed liquid hydrocarbon stream(s) 45 which is at a minimum pipeline pressure or higher is between the minimum pipeline pressure and 140 bar.

In a preferred embodiment, the warmed liquid hydrocarbon stream 35 is pressurised to provide the pressurised warmed liquid hydrocarbon stream 45 as a supercritical hydrocarbon stream. Under supercritical conditions, the pressurised warmed liquid hydrocarbon stream 45 loses its latent heat capacity such that it can be more easily heated.

A gaseous hydrocarbon stream splitting device 60 can split at least one of the one or more gaseous hydrocarbon streams 55 into two or more parts 55a, 55b. Figure 1 shows the splitting of a single gaseous hydrocarbon stream 55, which is at or above the minimum pipeline pressure, into two parts, a first part 55a and a second part 55b.

The second part, 55b, can be passed to consumers of the gaseous hydrocarbon. If this is provided above the minimum pipeline pressure, then the pressure of the stream may be reduced by an appropriate expansion device. The embodiment of Figure 2a provides an example of such pressure reduction using a turbine to generate additional power.

The first part, 55a, is passed to a recycle turbine 110, where it is dynamically expanded to provide one or more expanded gaseous hydrocarbon streams 115. A single expanded gaseous hydrocarbon stream 115 is shown in Figure 1.
The recycle turbine 110 converts the useful work released by the dynamic expansion of the first part, 55a of the gaseous hydrocarbon stream 55 into kinetic energy to turn the shaft 112 of an electric generator 120. The electric generator can provide electrical power to meet the needs of the gasification facility.

In an alternative embodiment not shown in Figure 1, two or more gaseous hydrocarbon streams 55 can be provided by the one or more hydrocarbon stream heaters 50. A first gaseous hydrocarbon stream, which is at or above the pipeline pressure, can be passed to a gas network, while a second gaseous hydrocarbon stream can be passed to the recycle turbine for dynamic expansion.

Returning to the embodiment of Figure 1, the one or more expanded gaseous hydrocarbon streams 115 produced by the recycle turbine 110 can be provided at one or more expansion pressures. The one or more expansion pressures should be less than the suction pressure of the one or more gaseous hydrocarbon streams 55 passed to the turbine.

The power which can be generated from the recycle turbine 110 is dependent, amongst various factors, upon the pressure ratio of the turbine i.e. the ratio of the pressure of the expanded gaseous hydrocarbon stream 115 to the pressure of the part 55a of the one or more gaseous hydrocarbon streams sent to the recycle turbine 110. Gas expansion is more efficient at pressures less than the supercritical range, such that suction pressures in the range of 60 to 80 bar are preferred.

The one or more expanded gaseous hydrocarbon streams 115 are passed to the hydrocarbon stream heat exchanger 3 as at least a part of the one or more vapour recycle streams. Preferably the one or more vapour recycle
streams 115 are provided entirely from the one or more expanded gaseous hydrocarbon streams 115, such that the vapour recycle stream and expanded gaseous hydrocarbon stream are one and the same. The one or more vapour recycle streams 115 are liquefied in the hydrocarbon stream heat exchanger 30 against the pressurised liquid hydrocarbon stream 15 to provide a liquefied recycle stream 125. Figure 1 shows the liquefaction of a single vapour recycle stream 115 to a single liquefied recycle stream 125.

If the liquefied recycle stream 125 produced by the hydrocarbon stream heat exchanger 30 is at the feed pressure of the liquid hydrocarbon stream 5, then it can be passed directly to the liquid hydrocarbon stream 5. Such an embodiment is discussed in relation to Figure 2a below.

Alternatively, if the pressures of the one or more liquefied recycle streams 125 are above the feed pressure of the liquid recycle stream 5, then they can passed to the suction of one or more recycle stream pumps 130. The one or more recycle stream pumps 130 are preferably electrical pumps. The one or more recycle pumps 130 pressurise the one or more liquefied recycle streams 125 to provide one or more pressurised liquefied recycle streams 135 at the cold recovery pressure. The one or more liquefied recycle streams 125 are preferably fully liquefied streams, because the recycle stream pump 130 can operate most efficiently with a single phase liquid stream. Rather than using multiple recycle stream pumps 130 when there is more than one liquefied recycle stream 125, a multi-stage recycle stream pump 130 may be more economical. In the embodiment shown in Figure 1, a single
liquefied recycle stream 125 is passed to a single recycle stream pump 130.

The one or more pressurised liquefied recycle streams 135 can then be combined with the pressurised liquid hydrocarbon stream 15, via one or more hydrocarbon stream mixing devices 20, to provide pressurised liquid hydrocarbon stream 15a, which is a combined stream, and is passed to the hydrocarbon stream heat exchanger 30.

In order to maximise the power which can be generated by the electric generator 120, the part 55a of the one or more gaseous hydrocarbon streams 55 which is passed to the recycle turbine 110 should be maximised. The one or more expanded gaseous hydrocarbon streams 115 produced by the recycle turbine 110 must be subsequently liquefied so that they may be recycled. The portion of the one or more gaseous hydrocarbon streams 55 which can be recycled depends upon the cooling duty available in the hydrocarbon stream heat exchanger 30 from the pressurised liquid hydrocarbon stream 15 and the energy required to cool and liquefy the vapour recycle stream 115 in the hydrocarbon stream heat exchanger 30.

The cooling duty of the pressurised liquid hydrocarbon stream 15 is dependent upon the temperature change of the stream i.e. the extent to which it is warmed in the hydrocarbon stream heat exchanger 30, as well as the specific heat capacity of the stream.

Taking a liquefied natural gas stream as an example of the pressurised liquefied hydrocarbon stream 15, the specific heat capacity increases with decreasing stream pressure. As the specific heat capacity increases, the cooling capacity of the liquefied natural gas stream will also increase. For instance, the cooling capacity of liquefied natural gas at 10 bar is approximately 1.75%
more than the cooling capacity at 75 bar. This provides a significant effect in terms of the cooling duty of this stream. Thus lower cold recovery pressures are preferred for this reason.

The temperature change of the pressurised liquid hydrocarbon stream 15 in the heat exchanger will depend upon the temperature of the warmed liquid hydrocarbon stream 35. This temperature is related to the working pressure of the heat exchanger 30. Thus, the higher the temperature to which the pressurised liquid hydrocarbon stream 15 is warmed in the hydrocarbon stream heat exchanger 30, the higher the cold recovery pressure required. However, as discussed above, the higher the pressure, the lower the specific heat capacity of the pressurised liquid hydrocarbon stream 15, and thus the lower the cooling capacity. Consequently, the choice of cold recovery pressure is a balance between the specific heat capacity of the liquid hydrocarbon and the temperature increase of the liquid hydrocarbon in the heat exchanger 30.

A further limitation on the cooling duty provided by the pressurised liquefied hydrocarbon stream 15 is that this stream should remain in the liquid phase in order to allow it to be efficiently pressurised in the warmed hydrocarbon stream pump 40.

The energy required to cool and liquefy the vapour recycle stream 115 is a function of the specific heat capacity of the vapour recycle stream, the reduction in temperature of the stream in the hydrocarbon stream heat exchanger 30, the enthalpy of liquefaction of the vapour recycle stream at the discharge pressure of the recycle turbine 110, and the energy of any subcooling.
In this way, it is possible to determine the maximum proportion of the one or more gaseous hydrocarbon streams 55 which can be expanded and liquefied, allowing it to be recycled to upstream of the hydrocarbon stream heat exchanger 30.

Within the boundary presented by the maximum portion of the one or more gaseous hydrocarbon streams 55 which can be ultimately be reliquefied, there are a number of factors which can be considered with a view to maximising power generation.

The extent to which the one or more gaseous hydrocarbon streams 55 can be expanded through the recycle turbine 110 and reliquefied in the hydrocarbon stream heat exchanger 30 depends upon the degree of expansion which is carried out in the recycle turbine 110. The greater the expansion, the lower the pressure of the vapour recycle stream 115 passed to the hydrocarbon stream heat exchanger 30. The lower the pressure of the vapour recycle stream 115, the lower the temperature at which it will liquefy in the hydrocarbon stream heat exchanger 30.

However, the capacity of the pressurised liquid hydrocarbon stream 15 to liquefy the vapour recycle stream 115 mainly depends upon the pressure difference.

A ratio of the cold recovery pressure to the pressure of the vapour recycle stream 115 in the range of 2 to 3 is advantageous because it provides a neat cooling curve for the vapour recycle stream 115.

For example, when the liquid hydrocarbon is LNG, the cold recovery pressure is 20 bara, and the pressure of the vapour recycle stream 115 is 7 bara, the split ratio, i.e. the extent to which the liquid hydrocarbon can be recycled after vaporisation, can be approximately 27%
i.e. 27% by weight of the one or more gaseous hydrocarbon streams can be passed to the recycle turbine for energy production and then reliquefied in the hydrocarbon stream heat exchanger and returned upstream of the heat exchanger. In this case, approximately one third of the flow can be expanded.

However, it should be noted that the method disclosed herein may not have a single optimum point. There is a range for the expansion of the recycled first part of the one or more gaseous hydrocarbon streams of between 5 and 45 bara, with a corresponding split ratio for the part of the stream to be recycled of between 14% (at 5 bara) and 65% (at 45 bara).

Optimisation will depend upon the composition of the liquid hydrocarbon stream, its temperature, as well as other factors such as the temperature approach inside the hydrocarbon stream heat exchanger between the pressurised liquid hydrocarbon stream and the vapour recycle stream.

Thus, for example, a regasification plant with a 5 million metric ton per annum (MTPA) design (i.e. the production of second part of the one or more gaseous hydrocarbon streams), it would be more economical to provide a vapour recycle stream which has been expanded at a low pressure range and provided with a small flow rate to the recycle stream to minimise the size of the hydrocarbon stream heat exchanger and recycle turbine, with corresponding CAPEX savings.

Figure 2a shows further embodiments of the method and apparatus disclosed herein. In one such embodiment, the liquid hydrocarbon stream is provided from a storage tank by pumping a liquid hydrocarbon feed stream with a feed pump, such as a submerged pump. The feed pump is preferably an electrical pump.
The feed pump 410 provides a pumped liquid hydrocarbon stream 415, which can be passed through a feed valve 420, to provide the liquid hydrocarbon stream 5 at feed pressure.

The liquid hydrocarbon is normally stored in tank 400 under cryogenic conditions. For instance, when the liquid hydrocarbon is liquefied natural gas, it can be stored at temperatures of less than -150 °C in a thermally insulated tank. During storage at such cryogenic conditions, the liquid hydrocarbon can be at or near atmospheric pressure, such as approximately 1 bar.

Temperature variations during storage can result in the vaporisation of a portion of the liquefied hydrocarbon as hydrocarbon gas vapour, also called boil off gas. Such hydrocarbon gas vapour can be withdrawn from at or near the top of storage tank 400 as a boil off gas stream 455, and passed to the suction side of a boil off gas compressor 460. The boil off gas compressor 460 can be mechanically driven by a boil off gas compressor driver 470, such as an electric driver powered by electricity generated by the recycle stream electric generator 120.

The boil off gas compressor 460 compresses the boil off gas stream 455 to provide a compressed boil off gas stream 465. The heat of compression can then be removed from the compressed boil off gas stream 465, and this condensed in a boil off gas condenser 480. The boil off gas condenser 480 provides a liquefied boil off gas stream 485, which can be returned to the storage tank 400. The compressed boil off gas stream 465 may be condensed against a liquefied hydrocarbon stream (not shown), for instance a liquefied hydrocarbon stream drawn from the tank 400.
With regard to the gasification line-up shown in Figure 2a, this differs from that of Figure 1 in that an optional further heater 70 is provided to warm the gaseous hydrocarbon stream 55 against a heat source to provide (warmed) gaseous hydrocarbon stream 55c prior to passing it to the gaseous hydrocarbon stream splitting device 60. The heat source preferably has a temperature of less than or equal to ambient temperature, such as an ambient air or ambient water sources, such as a seawater stream or a hydrocarbon stream produced in the gasification process. The optional further heater 70 is advantageous if the gaseous hydrocarbon stream 55 is produced at too low a temperature for a portion, 55e, to be passed to gas consumers. For instance if the gaseous hydrocarbon stream 55 is provided at or below 0 °C, additional warming could be required to prevent condensation of water vapour on associated pipework.

A gaseous hydrocarbon stream splitting device 60 provides a first part 55d of the (warmed) gaseous hydrocarbon stream 55c for dynamic expansion and power generation in recycle turbine 110.

The embodiment of Figure 2a shows a further optional aspect of the method and apparatus in which the second part 55e of the (warmed) gaseous hydrocarbon stream 55c is passed to the suction of a pipeline turbine 80. The pipeline turbine 80 dynamically expands the second part 55e of the (warmed) gaseous hydrocarbon stream 55c to provide an expanded gaseous hydrocarbon stream 85. The useful work extracted from the second part 55e of stream 55c is used to mechanically drive a pipeline electric generator 82 via shaft 81 to produce electrical power.

If required, the expanded gaseous hydrocarbon stream 85 may be passed to a further heater 87, in which it can
be heated to, for example, 10-15 °C in order to redress any cooling resulting from the expansion in the hydrocarbon stream turbine 80. The further heater 87 provides a gaseous hydrocarbon pipeline stream 95 which can be passed to hydrocarbon gas consumers. The further heater 87 may be provided with a heat source, such as an ambient heat source, having a temperature of less than or equal to ambient temperature. The heat source may be one or more of an ambient air source, an ambient water source such as a seawater stream, and a hydrocarbon stream produced in the gasification process.

The optional hydrocarbon stream turbine 80 is advantageous when the warmed liquid hydrocarbon stream 35 is pumped to a pressure above the pipeline pressure by warmed hydrocarbon stream pump 40. In such a situation, the second part 55e of the (warmed) gaseous hydrocarbon stream 55c would be provided at a pressure above the pipeline pressure, and thus require pressure reduction before it can be passed to the gas network. The pressure reduction of the second part 55e of stream 55c can thus be used to generate additional power. The warmed liquid hydrocarbon stream 35 may be pumped to a pressure above the pipeline pressure in order to provide a particular suction pressure for the recycle turbine 110, which is fed by the first part 55c of the (warmed) gaseous hydrocarbon stream 55c.

Figure 2b provides alternative embodiments of the method and apparatus disclosed herein in which optional further heaters 90, 140 are provided to warm the first and second parts 55a, 55b of the gaseous hydrocarbon stream 55. Optional further heaters 90, 140 are preferably supplied with heat sources having a temperature of less than or equal to ambient temperature.
In contrast to the embodiment of Figure 2a, where a single further heater 70 is provided to warm the gaseous hydrocarbon stream 55 upstream of the gaseous hydrocarbon stream splitting device 60, further heaters 90, 140 are provided in each of the part streams 55a, 55b so that the temperature of each stream can be individually adjusted. It will be evident to the skilled person that only one or other of these optional further heaters 90, 140 may be present as required.

Thus, the first part 55a of gaseous hydrocarbon stream 55 can be warmed by further heater 140 which preferably has a heat source having a temperature of less than or equal to ambient temperature, to provide a (warmed) first part 55f of the gaseous hydrocarbon stream 55 to the recycle turbine 110. This can be advantageous in controlling the suction temperature of the stream to be expanded in the recycle turbine 110 in order to maintain efficient operation.

Similarly, the second part 55b of the gaseous hydrocarbon stream 55 can be warmed in optional further heater 90 to provide gaseous hydrocarbon pipeline stream 95a. The optional further heater 90 is advantageous if the second part 55b of gaseous hydrocarbon stream 55 is too cold to be passed to the gas network, and must be warmed prior to distribution.

In a further optional embodiment shown in Figure 2b, the liquefied recycle stream 125, produced by liquefying the vapour recycle stream 115 against the pressurised liquid hydrocarbon stream 15, is passed directly to liquid hydrocarbon stream 5 via hydrocarbon stream mixing device 20a to provide a (combined) liquid hydrocarbon stream 5a. This line-up may be used when the recycle turbine 110 provides an expanded gaseous hydrocarbon
stream (vapour recycle stream) 115 at the feed pressure of the liquid hydrocarbon stream 5.

In this case, there is no requirement for a recycle stream pump to pressurise the liquefied recycle stream 125 to the cold recovery pressure so that it can be combined with the pressurised liquefied hydrocarbon stream 15.

Although the power requirements of liquid hydrocarbon stream 10 would have to be increased to handle (combined) liquefied hydrocarbon stream 5a, comprising both the liquefied hydrocarbon stream 5 and the liquefied recycle stream 125, CAPEX savings are provided because no recycle stream pump is required.

Figure 2c provides an alternative embodiment of the method and apparatus disclosed herein in which all, rather than a part, of a gaseous hydrocarbon stream 55 is passed to the recycle turbine 110a for power generation.

In such an embodiment, the recycle turbine 110 may be a multi-stage turbine. Such a multi-stage recycle turbine 110a can provide two discharge streams, a first expanded gaseous hydrocarbon stream 115a, and a second expanded gaseous hydrocarbon stream 115b. The first expanded recycle stream 115a can be an intermediate pressure stream which can be used to provide the gaseous hydrocarbon for export. In order to avoid subsequent repressurisation of the first expanded recycle stream 115a, it should be provided at a pressure at or above, preferably at, the pipeline pressure. The second expanded gaseous hydrocarbon stream 115b can be an intermediate pressure stream which can be used as a vapour recycle stream.

If the first expanded gaseous hydrocarbon stream 115a is at the pipeline pressure, no further pressure
reduction is required. The first expanded gaseous hydrocarbon stream 115a can be passed to a further heater 87b, in which it can be heated to, for example 10-15 °C, in order to raise the gas to a temperature suitable for export. The further heater 87b may be provided with a heat source having a temperature of less than or equal to ambient temperature. The further heater 87b provides a gaseous hydrocarbon pipeline stream 95b which can be passed to hydrocarbon gas consumers.

If the first expanded gaseous hydrocarbon stream 115a is at a pressure above the pipeline pressure, it can be further expanded in a hydrocarbon stream turbine (not shown) in order to generate additional power and reduce the pressure of this stream to the pipeline pressure for export. This embodiment is less preferred because two turbines are thus required.

More preferred is the use of a single turbine which would be a multi-stage recycle turbine 110a, which would provide the first expanded gaseous hydrocarbon stream 115a at pipeline pressure, as well as the second expanded gaseous hydrocarbon stream 115b which is to be used as the vapour recycle stream. The use of a single turbine in the line-up shown in Figure 2c provides CAPEX savings.

The second expanded gaseous hydrocarbon stream 115b can be passed to hydrocarbon stream heat exchanger 30 for liquefaction against the liquid hydrocarbon stream 15 as described for the embodiment of Figure 1.

Figure 3 shows a further embodiment of the method and apparatus 1 disclosed herein in which multiple vapour recycle streams 115, operating at different pressures, are liquefied in the hydrocarbon stream heat exchanger 30. The provision of two or more vapour recycle streams 115 at different pressures is advantageous because it
allows improved thermal transfer in the heat exchanger 30. In particular, the cooling curves of multiple vapour recycle streams 115 can be used to provide a better match to the heating curve of the (combined) pressurised liquid hydrocarbon stream 15a.

In one example of this line-up, the liquid hydrocarbon stream 5 can be a liquefied natural gas stream, provided at a feed pressure of, for example, 1 bar, and pressurised to a cold recovery pressure of, for example, 30 bar by liquid natural gas stream pump 10. The pressurised LNG stream 15 is then combined with a liquefied recycle stream 125 at a mixing temperature of, for example, -146 °C by natural gas stream mixing device 20 to provide the (combined) pressurised LNG stream 15a with a mass flow of, for example, 50 kg/s. The (combined) pressurised LNG stream 15a is passed to the natural gas stream heat exchanger 30, which can be a condenser.

The condenser 30 heats the (combined) pressurised LNG stream 15a to provide a warmed LNG stream 35 at a temperature of, for example, -95 °C. The warmed LNG stream 35 is passed to warmed LNG stream pump 40, where it is pressurised to a pressure of, for example, 75 bar to provide pressurised warmed LNG stream 45. The pressurised warmed LNG stream 45 can be vaporised in an LNG stream heater 50 with a heat source, for example to a temperature in the range of 10-15 °C to provide a gaseous natural gas stream 55. The heat source preferably has a temperature of less than or equal to ambient temperature, and is more preferably an ambient heat source. For instance, the LNG stream heater 50 is preferably an open rack vaporiser.

The gaseous natural gas stream 55 can have a mass flow of, for example, 50 kg/s, and is passed to a gaseous
natural gas stream splitting device 60. The gaseous natural gas stream 55 is split into two parts. A first part, 55a, is passed to a recycle turbine 110, for example at a mass flow of 19 kg/s. A second part 55b, can be passed to a gas network. The second stream will be at the pipeline pressure of 75 bar, if the pressurised warmed LNG stream 45 is pumped to this pressure.

The first part 55a of the gaseous natural gas stream 55 is dynamically expanded in the recycle turbine 110 from the pipeline pressure, for example 75 bar. The recycle turbine 110b can be a multi-stage recycle turbine. Such a multi-stage recycle turbine 110b can provide two discharge streams, a first expanded gaseous natural gas stream 115c, for example with a pressure of 25 bar and a mass flow of 7 kg/s, and a second expanded gaseous natural gas stream 115d, for example with a pressure of 10 bar and a mass flow of 12 kg/s. Both first and second expanded gaseous natural gas streams 115a, 115b can then be passed to condenser 30 for liquefaction against the liquid LNG stream 15.

The first expanded gaseous natural gas stream 115c can be liquefied in condenser 30, for example at -115 °C, to provide a first liquefied recycle stream 125c. The second expanded gaseous natural gas stream 115d can be liquefied in condenser 30, for example at -130 °C, to provide a second liquefied recycle stream 125d. The first expanded gaseous natural gas stream 115c will be more easily liquefied in the condenser 30 than the second stream 115d, because the former stream is at a higher pressure. The first expanded stream 115c will therefore have better heat exchanger temperature heat transfer, and avoid too low a minimum internal temperature approach to the exchanger 30.
The first and second liquefied recycle streams 125c, 125d can then be passed to a multi-stage recycle stream pump 130 where they are pressurised, for example to 31 bar, to provide a pressurised liquefied recycle stream 135, for example with a mass flow of 19 kg/s. The pressurised liquefied recycle stream 135 can then be mixed with the pressurised LNG stream 15 via natural gas stream mixing device 20.

The pressurised LNG stream 15 is provided by LNG stream 5, which can be drawn from an LNG source such as a storage tank. When the LNG stream 5 has a mass flow of 31 kg/s, the pressurised LNG stream 15 will have a similar mass flow. Thus, (combined) pressurised LNG stream 15a, comprising the pressurised LNG stream 15 and pressurised liquefied recycle stream 135, will have the required mass flow of 50 kg/s. Consequently, this embodiment provides a line-up in which 38 (i.e. 19/50 kg/s) % by mass of the natural gas is recycled to the condenser 30.

The multi-stage recycle turbine 110 can produce 2.5 MW/MTPA power when used with the exemplary mass flows, pressures and temperatures provided in this embodiment. The power consumption of pumps 10, 40 and 130 is 0.4 MW/MTPA in total, such that a net power production of 2.1 MW/MTPA is provided. This is above the power threshold necessary to provide an autonomous gasification terminal and thus would not require any external power, or utilise any fuel to generate additional power, during steady state operation.

In an alternative embodiment not shown in Figure 3, one or both of the multi-stage recycle turbine 110 and the multi-stage recycle stream pump 130 can be replaced by two or more recycle turbines 110b and two or more recycle stream pumps respectively.
Figure 4 provides a further embodiment of the method and apparatus 1 disclosed herein. In this line-up, a Rankine cycle is incorporated into the regasification process according to the embodiment of Figure 1 to increase the generated power. It will be apparent that such a Rankine cycle can be used with any embodiment disclosed herein.

One or more Rankine circuits 200 can be added to generate additional power from the pressurised warmed liquid hydrocarbon stream 45. The pressurised warmed liquid hydrocarbon stream 45 is provided by warmed hydrocarbon stream pump 40, in which it has been pressurised to the minimum pipeline pressure or higher. In a preferred embodiment, the pressurised warmed liquid hydrocarbon stream is pressurised to supercritical pressure, for example in the range of 65 to 85 bars.

Prior to vaporisation in the one or more LNG stream heaters 50, the pressurised warmed liquid hydrocarbon stream 45 can be passed to a working fluid heat exchanger 210, where it is warmed against an expanded vaporised working fluid stream 245 in the Rankine circuit 200, to provide a pressurised (further) warmed liquid hydrocarbon stream 45a and a liquefied working fluid stream 215.

The working fluid preferably has a condensing temperature between -20 and -50 °C. In some cases, a refrigerant is useful as the working fluid. Preferred working fluids are selected from one or more of the group comprising: tetrafluoromethane (R14), ethane, pentafluoroethane (R125) and propane.

The cold energy from the pressurised warmed liquid hydrocarbon stream 45 is used to liquefy the working fluid in the expanded vaporised working fluid stream 245. The liquefied working fluid stream 215 thus produced can
then be passed to a working fluid pump 220, in which it is pressurised to provide a pressurised liquefied working fluid stream 225. Preferably, the liquefied working fluid stream 215 is pumped to a pressure in the range of 5 to 12 bar, more preferably about 6.2 bars. It is generally preferred to pump the liquefied working fluid stream 215 to its dew point pressure at ambient temperature. Such a liquefied working fluid stream can then be vapourised against a heat source at 15 °C.

The pressurised liquefied working fluid stream 225 can then be passed to a working fluid heater 230, where it is vaporised against a heat source. The heat source preferably has a temperature of less than or equal to ambient temperature, such as an ambient air or ambient water feed stream 325. In an alternative embodiment, the heat source may be a hydrocarbon stream generated as part of the gasification method. The working fluid heater 230 is preferably an open rack vaporiser. The working fluid heater 230 provides a vaporised working fluid stream 235, and a cooled heat source, such as a water, air or hydrocarbon stream 345a, which is a cooled stream.

The vaporised working fluid stream 235 can then passed to a working fluid turbine 240, where it is dynamically expanded to provide power and the expanded vaporised working fluid stream 245. The power is produced by mechanically driving a working fluid electric generator 250 with the working fluid turbine 240 via shaft 242. Preferably, the vaporised working fluid stream 235 is expanded to a pressure in the range of 1 to 3 bar, more preferably about 1.2 bars.

Ideally, expansion to a minimum pressure in the working fluid turbine would be preferred in order to maximise power generation. However, the lower the
pressure of the expanded vapourised working fluid stream 245, the harder it will be to liquefy. In contrast, the higher the pressure of the expanded vapourised working fluid stream 245, the easier it will be to liquefy, such that although less power will be generated in the working fluid turbine 240, a smaller, less expensive working fluid heat exchanger 210 can be used.

The pressurised (further) warmed liquid hydrocarbon stream 45a can then be vaporised against a heat source 345a to provide a cooled heat source, such as a cooled air or water stream 355a, and a gaseous hydrocarbon stream 55. The heat source 345a preferably has a temperature of less than or equal to ambient temperature, and may be the air, water or hydrocarbon stream 345a produced by the working fluid heater 230.

Such an embodiment is capable of producing 4 MW/MTPA power. Approximately 2 MW/MTPA can be generated from the recycle turbine 110 and recycle stream electric generator 120, in a similar manner to the embodiment of Figure 1. A further 2 MW/MTPA can be generated from the working fluid turbine 240 and working fluid electric generator 250. 4 MW/MTPA power generation is equivalent to the emission of 12200 ton/year of carbon dioxide produced by electricity generation from the combustion of natural gas in a combined cycle power generation apparatus. This elimination of carbon dioxide emissions represents a significant advantage, with the further benefit that the additional electricity generated which is not used in the steady state operation of the gasification facility can be exported to other facilities or an external electricity grid.

The person skilled in the art will understand that the present invention can be carried out in many various
ways without departing from the scope of the appended claims. For instance, the power generated from the embodiment of Figure 4 incorporating a Rankine circuit could be increased by the use of two or more expanded gaseous hydrocarbon streams 115 at different pressure levels, as described in relation to the embodiment of Figure 3.
1. A method for the gasification of a liquid hydrocarbon stream (5) to provide a gaseous hydrocarbon stream (55) and power, comprising at least the steps of:

(a) pressurising a liquid hydrocarbon stream (5) at a feed pressure to provide a pressurised liquid hydrocarbon stream (15) at a cold recovery pressure;

(b) heat exchanging the pressurised liquid hydrocarbon stream (15) against one or more vapour recycle streams (115) to provide a warmed liquid hydrocarbon stream (35) and one or more liquefied recycle streams (125);

(c) passing the one or more liquefied recycle streams (125), after pressurisation if necessary, to one or both of the liquid hydrocarbon stream (5) and the pressurised liquid hydrocarbon stream (15);

(d) pressurising one or more portions of the warmed liquid hydrocarbon stream (35) to provide one or more pressurised warmed liquid hydrocarbon streams (45), at least one of which is at a minimum pipeline pressure or higher;

(e) vaporising one or more of the pressurised warmed liquid hydrocarbon streams (45), at least one of which is at the minimum pipeline pressure or higher, against one or more heat sources (345) having a temperature of less than or equal to ambient temperature to provide one or more gaseous hydrocarbon streams (55), at least one of which is at the minimum pipeline pressure or higher;

(f) dynamically expanding at least a part (55, 55a, 55e) of at least one of the one or more gaseous
hydrocarbon streams (55) to provide power and one or more expanded gaseous hydrocarbon streams (115), wherein at least a part of said one or more expanded gaseous hydrocarbon streams forms at least a part of the one or more vapour recycle streams (115).

2. The method according to claim 1 wherein all heat exchange and vaporisation steps resulting in an increase in the temperature of any of the hydrocarbon streams (15, 45, 55) are carried out against one or both of a heat source having a temperature of less than or substantially equal to ambient temperature and a hydrocarbon stream (115) generated in the gasification method.

3. The method according to claim 1 or claim 2, wherein between steps (e) and (f), a gaseous hydrocarbon stream (55) is split into a first part (55a, 55d) of the gaseous hydrocarbon stream (55) and a second part (55b, 55e) of the gaseous hydrocarbon stream (55), and wherein in step (f) the first part (55a, 55d) of the gaseous hydrocarbon stream (55) is dynamically expanded to provide one or more first expanded gaseous hydrocarbon streams as the one or more vapour recycle streams (115).

4. The method according to claim 3, wherein in step (f) the dynamic expansion provides at least two first expanded gaseous hydrocarbon streams as at least two vapour recycle streams (115) each said streams at different pressures, in step (b) the heat exchange provides two first liquefied hydrocarbon streams (125), and in pressurising step (c) the at least two first liquefied recycle streams (125) are pressurised to provide a pressurised liquefied recycle stream (135), which is passed to the pressurised liquid hydrocarbon stream (15).
5. The method according to claim 4 wherein the dynamic expansion in step (f) is carried out in a multistage recycle turbine (110) and in step (c) the pressurisation is carried out in a multistage recycle stream pump (130).

6. The method according to any one of the preceding claims, further comprising the step of heating the gaseous hydrocarbon stream (55) against a heat source (70) between steps (e) and (f).

7. The method according to any one of claims 3 to 5 further comprising the steps of:
   - heating the first part (55a) of the gaseous hydrocarbon stream (55) against a heat source prior to the dynamic expansion in step (f); and
   - heating the second part (55b) of the gaseous hydrocarbon stream (55) against a heat source.

8. The method according to any of the preceding claims wherein step (c) comprises one or both of:
   (i) passing at least one of the one or more liquefied recycle streams (125) to the liquid hydrocarbon stream (5) if the one or more liquefied recycle streams (125) are at the feed pressure or
   (ii) pressurising at least one of the one or more liquefied recycle streams (125) to provide a pressurised liquefied recycle stream (135) at the cold recovery pressure and passing the pressurised liquefied recycle stream to the pressurised liquid hydrocarbon stream (15), if the pressure of the one or more liquefied recycle streams (125) is above the feed pressure.

9. The method according to any of the preceding claims, wherein at least one (55c) of the one or more gaseous hydrocarbon streams (55) is at a pressure higher than the
minimum pipeline pressure, the method further comprising the steps of:

(g) dynamically expanding at least a part (55e) of the at least one gaseous hydrocarbon stream (55c) which is at a pressure higher than the minimum pipeline pressure in a pipeline turbine (80), said pipeline turbine mechanically driving a pipeline electric generator (82) to provide electrical power and an expanded gaseous hydrocarbon stream (85) which is at a pressure of at least the minimum pipeline pressure; and

(h) heating the expanded gaseous hydrocarbon stream (85) against a heat source to provide a heated gaseous hydrocarbon stream (95);

(i) passing the heated gaseous hydrocarbon stream (95) to a gas network.

10. The method according to any of the preceding claims further comprising, between steps (d) and (e), the steps of:

- heat exchanging at least one of the one or more pressurised warmed liquid hydrocarbon streams (45) against an expanded vaporised working fluid stream (245) to provide a liquefied working fluid stream (215);

- pressurising the liquefied working fluid stream (215) to provide a pressurised liquefied working fluid stream (225);

- vaporising the pressurised liquefied working fluid stream (225) against a heat source (325) to provide a vaporised working fluid stream (235); and

- dynamically expanding the vaporised working fluid stream (235) in a working fluid turbine (240), said working fluid turbine (240) mechanically driving a
working fluid electric generator (250) to provide electrical power and the expanded vaporised working fluid stream (245).

11. The method according to any of the preceding claims in which the cold recovery pressure is in the range of 10 to less than 60 bar, more preferably 20 to less than 50 bar and/or the minimum pipeline pressure is greater than or equal to 60 bar, more preferably greater than or equal to 70 bar, still more preferably greater than or equal to 75 bar.

12. The method according to any of the preceding claims wherein the liquid hydrocarbon stream (5) is liquefied natural gas and the one or more gaseous hydrocarbon streams (55) comprise a natural gas stream (55b).

13. An apparatus (1) for the gasification of a liquid hydrocarbon stream (5) to provide a gaseous hydrocarbon stream (55), said apparatus comprising at least:
- a liquid hydrocarbon stream pump (10) to pressurise a liquid hydrocarbon stream (5) at a feed pressure to provide a pressurised liquefied hydrocarbon stream (15) at a cold recovery pressure;
- a hydrocarbon stream heat exchanger (30) to heat exchange the pressurised liquid hydrocarbon stream (15) against one or more vapour recycle streams (115) to provide a warmed liquid hydrocarbon stream (35) and one or more liquefied recycle streams (125);
- a warmed hydrocarbon stream pump (40) to pressurise one or more portions of the warmed liquid hydrocarbon stream (35) to provide one or more pressurised warmed liquid hydrocarbon streams (45) at least one of which is at a minimum pipeline pressure or higher;
- a heat source (345) to vaporise one or more of the pressurised warmed liquid hydrocarbon streams (45), at
least one of which is at a minimum pipeline pressure or higher, to provide one or more gaseous hydrocarbon streams;

- a recycle turbine (110) to dynamically expand at least a part (55, 55a, 55e) of at least one of the one or more gaseous hydrocarbon streams (55) to drive a shaft (112) and provide one or more expanded gaseous hydrocarbon streams, at least a part of which forms at least a part of the one or more vapour recycle streams (115); and

- one or more hydrocarbon stream mixing devices (20) to introduce the one or more liquefied recycle streams (125), or a stream derived therefrom (135), to the liquid hydrocarbon stream (5) or the pressurised liquefied hydrocarbon stream (15).

14. The apparatus (1) according to claim 13, further comprising:

- a recycle stream pump (130) to pressurise the one or more liquefied recycle streams (125) to provide a pressurised liquefied recycle stream (135) at the cold recovery pressure; and

wherein hydrocarbon stream mixing device (20) introduces the pressurised liquefied recycle stream (135) to the pressurised liquid hydrocarbon stream (15).

15. The apparatus (1) according to claim 13 or claim 14, wherein the heat source (345) has a temperature of less than or substantially equal to ambient temperature.
A. CLASSIFICATION OF SUBJECT MATTER
   INV. F17C9/04
   ADD.
   According to International Patent Classification (IPC) or to both national classification and IPC.

B. FIELDS SEARCHED
   Minimum documentation searched (classification system followed by classification symbols)
   F17C
   Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched
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C. DOCUMENTS CONSIDERED TO BE RELEVANT

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Further documents are listed in the continuation of Box C

Date of the actual completion of the international search
30 July 2010

Date of mailing of the international search report
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Name and mailing address of the ISA
European Patent Office P B 5818 Patentlaan 2 NL - 2280 HV Rijswijk Tel (+31-70) 340-2040, Fax (+31-70) 340-3016

Authorized officer
Ott, Thomas
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