

(12) PATENT
(19) AUSTRALIAN PATENT OFFICE

(11) Application No. AU 199920681 B2
(10) Patent No. 747693

(54) Title
Active acoustic devices comprising panel members

(51)⁷ International Patent Classification(s)
H04R 007/06 H04R 007/10

(21) Application No: **199920681**

(22) Application Date: **1999.01.15**

(87) WIPO No: **WO99/37121**

(30) Priority Data

(31) Number	(32) Date	(33) Country
9801057	1998.01.20	GB
9801054	1998.01.20	GB
9811100	1998.05.23	GB
9813293	1998.06.20	GB

(43) Publication Date : **1999.08.02**

(43) Publication Journal Date : **1999.10.07**

(44) Accepted Journal Date : **2002.05.16**

(71) Applicant(s)
New Transducers Limited

(72) Inventor(s)
Farad Azima; Henry Azima; Martin Colloms; Graham Bank; Nicholas Patrick Roland Hill

(74) Agent/Attorney
SPRUSON and FERGUSON,GPO Box 3898,SYDNEY NSW 2001

(56) Related Art
WO 92/03024
US 3347335
WO 97/09842

20681/99

PCT

WORLD INTELLECTUAL PROPERTY ORGANIZATION
International Bureau

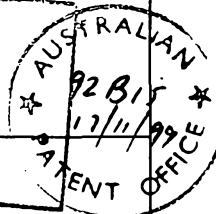
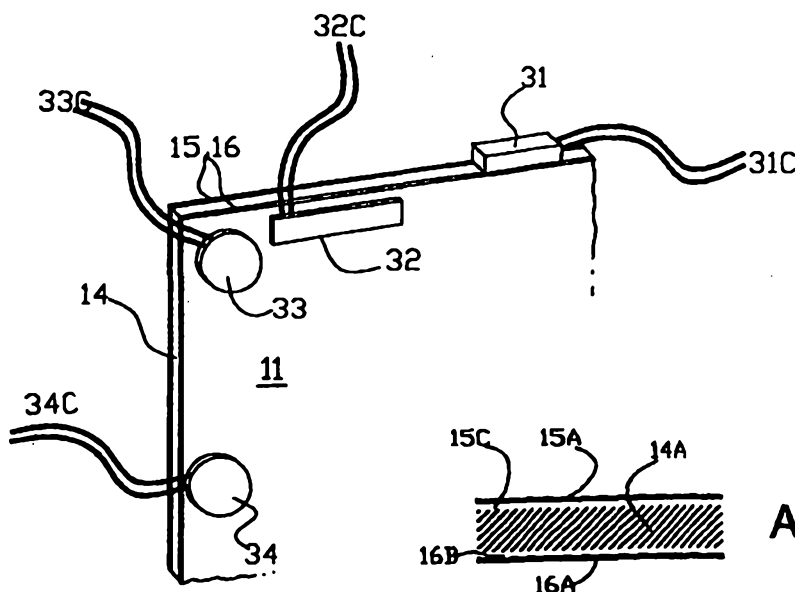
INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

(51) International Patent Classification ⁶ : H04R 7/06, 7/10		A1	(11) International Publication Number: WO 99/37121
			(43) International Publication Date: 22 July 1999 (22.07.99)
(21) International Application Number: PCT/GB99/00143		(74) Agent: MAGUIRE BOSS; 5 Crown Street, St. Ives, Cambridgeshire PE17 4EB (GB).	
(22) International Filing Date: 15 January 1999 (15.01.99)		(81) Designated States: AL, AM, AT, AU, AZ, BA, BB, BG, BR, BY, CA, CH, CN, CU, CZ, DE, DK, EE, ES, FI, GB, GE, GH, GM, HR, HU, ID, IL, IN, IS, JP, KE, KG, KP, KR, KZ, LC, LK, LR, LS, LT, LU, LV, MD, MG, MK, MN, MW, MX, NO, NZ, PL, PT, RO, RU, SD, SE, SG, SI, SK, SL, TJ, TM, TR, TT, UA, UG, US, UZ, VN, YU, ZW, ARIPO patent (GH, GM, KE, LS, MW, SD, SZ, UG, ZW), Eurasian patent (AM, AZ, BY, KG, KZ, MD, RU, TJ, TM), European patent (AT, BE, CH, CY, DE, DK, ES, FI, FR, GB, GR, IE, IT, LU, MC, NL, PT, SE), OAPI patent (BF, BJ, CF, CG, CI, CM, GA, GN, GW, ML, MR, NE, SN, TD, TG).	
(30) Priority Data: 9801057.2 20 January 1998 (20.01.98) GB 9801054.9 20 January 1998 (20.01.98) GB 9811100.8 23 May 1998 (23.05.98) GB 9813293.9 20 June 1998 (20.06.98) GB			
(71) Applicant (for all designated States except US): NEW TRANSDUCERS LIMITED [GB/GB]; Stonehill, Huntingdon, Cambridgeshire PE18 6ED (GB). <i>Ixworth House 37 Ixworth Place London SW3 3AH</i>			
(72) Inventors; and <i>United Kingdom</i>			
(75) Inventors/Applicants (for US only): AZIMA, Farad [GB/GB]; Flat 3, 1 Palace Gate, Kensington, London W8 5KS (GB). AZIMA, Henry [CA/GB]; 3 Southacre Close, Chaucer Road, Cambridge CB2 2TT (GB). COLLOMS, Martin [GB/GB]; 22 Burgess Hill, London NW2 2DA (GB). BANK, Graham [GB/GB]; 1 Boartree Way, Huntingdon, Cambridgeshire PE18 6GL (GB). HILL, Nicholas, Patrick, Roland [GB/GB]; The Flat, 206 Cherry Hinton Road, Cambridge CB1 4AW (GB).		Published With international search report. Before the expiration of the time limit for amending the claims and to be republished in the event of the receipt of amendments.	

IP AUSTRALIA

02 AUG 1999

RECEIVED

(54) Title: **ACTIVE ACOUSTIC DEVICES COMPRISING PANEL MEMBERS**

(57) Abstract

Active acoustic device comprises a panel member (11) having distribution of resonant modes of bending wave action determining acoustic performance in conjunction with transducer means (31-34). The transducer means (31-34) is coupled to the panel member (11) at a marginal position. The arrangement is such as to result in acoustically acceptable action dependent on said distribution of active said resonant modes. Methods of selecting the transducer location, or improvement by location of localised marginal clamping, rely on assessing best or better operative interaction of said transducer means (31-34) and the panel members (11) according to parameters of acoustic output for the device as an acoustic radiator.

5 **ACTIVE ACOUSTIC DEVICES COMPRISING PANEL MEMBERS**10 DESCRIPTION

15 FIELD OF THE INVENTION

 This invention relates to active acoustic devices and more particularly to panel members for which acoustic action or performance relies on beneficial distribution of resonant modes of bending wave action in such a panel member and
20 related surface vibration; and to methods of making or improving such active acoustic devices.

 It is convenient herein to use the term "distributed mode" for such acoustic devices, including acoustic radiators or loudspeakers; and for the term "panel-form"
25 to be taken as inferring such distributed mode action in a panel member unless the context does not permit.

 In or as panel-form loudspeakers, such panel members operate as distributed mode acoustic radiators relying on

bending wave action induced by input means applying mechanical action to the panel member; and resulting excitation of resonant modes of bending wave action causing surface vibration for acoustic output by coupling to ambient fluid, typically air. Revelatory teaching regarding such acoustic radiators (amongst a wider class of active and passive distributed mode acoustic devices) is given in our International patent application WO97/09842; and various of our later patent applications concern useful additions and developments.

BACKGROUND TO THE INVENTION

Hitherto, transducer locations have been considered as viably and optimally effective at locations in-board of the panel member to a substantial extent towards but offset from its centre, at least for panels that are substantially isotropic as to bending stiffness and exhibit effectively substantially constant axial anisotropy of bending stiffness(es). Aforementioned WO97/09842 gives specific guidance in terms of optimal proportionate co-ordinates for such in-board transducer locations, including alternatives; and preference for different particular co-ordinate combinations when using two or more transducers.

Various advantageous applications peculiar to the panel-form of acoustic devices have been foreshadowed, including carrying acoustically non-intrusive surfacing sheets or layers. For example, physically merging or incorporating into trim or cladding is feasible, including

as visually virtually indistinguishable. Also, functional combination is feasible with other purposes, such as display, including pictures, posters, write-on/erase boards, projection screens, etc. The capability effectively to hide in-board transducers from view is enough
 5 for many applications. However, there are potential practical applications where it could be useful to leave larger, particularly central, panel regions unobstructed even by hideable transducers. For example, for video or other see-through display use, pursuit of translucence, even transparency, of panel members is not worthwhile with such in-board instructions of transducers, though a panel-form acoustic device would be highly
 10 attractive if it could afford large medial areas of unobstructed visibility.

Object of the Invention

It is an object of the present invention to substantially overcome or at least ameliorate one or more of the disadvantages of the prior art, or at least to provide a useful
 15 alternative.

Summary of the Invention

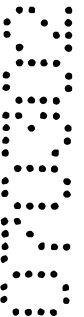
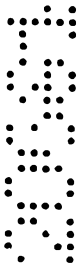
Accordingly, in a first aspect, the present invention provides a distributed mode active acoustic device comprising a plural-sided panel member and a transducer coupled
 20 to the panel member, the panel member being capable of sustaining bending waves in an operative frequency range over an active area of the transverse extent of the panel member with a distribution of resonant modes of bending wave vibration determining acoustic performance in conjunction with the transducer, the panel member having certain physical parameters the values of which affect said distribution of resonant modes, said
 25 parameters having selected values such that said resonant modes are beneficially distributed in frequency over said active area, and the panel member further having at least one in-board region of said active area where a plurality of lower frequency resonant bending wave modes in the operative frequency range have vibrationally active anti-nodes, the transducer being coupled to the panel member at a marginal position of the
 30 panel member for beneficial operative interaction of the transducer with the panel member, said marginal position corresponding to an orthogonal coordinate of said at least one in-board region.

The preferred embodiment provides a panel-form acoustic device comprising a distributed mode acoustic panel member with transducer means located at a marginal position, the arrangement being such as to result in acoustically acceptable effective



distribution and excitement of resonant mode vibration. Existence of suitable such marginal positions is established herein as locations are transducer means, along with valuable teaching as to judicious selection or improvement of one or more such locations.

- 5 Such judicious selection may advantageously be by or as would result from investigation of an acoustic radiator device or



loudspeaker relative to satisfactorily introducing vibrational energy into the panel member, say conveniently by assessing parameters of acoustic output from the panel member concerned when excited at marginal positions or 5 locations. At least best results also apply to microphones.

From the relevant background teaching as of the time of this invention, availability of successful such marginal locations is, to say the least, unexpected.

10 Indeed, main closest prior art cited against WO97/09842, is the start-point for its invention and revelatory teaching, namely WO92/03024 from which progress was made particularly in terms of departing from in-corner excitation thereof. Such progress involved appreciating

15 that distributed resonant mode bending wave action as required for viable acoustic performance results in high vibrational activity at panel corners; as is also a factor for panel edges generally. At least intuitively, and as greatly reinforced by practical success with

20 somewhat off-centre but very much in-board transducer locations, such high vibrational activity compounds strongly with panel margins self-evidently affording limited access, thus likely available effect upon, panel member material as a whole; this compounding combination

25 contributing to previously perceived non-viability of edge excitation.

For application of this invention, a suitable acoustic panel member, or at least region thereof, may be

transparent or translucent. Typical panel members may be generally polygonal, often substantially rectangular. Plural transducer means may be at or near different edges, at least for substantially rectangular panel members.

5 The or each transducer may be piezo-electric, electrostatic or electro-mechanical. The or each transducer may be arranged to launch compression waves into the panel edge, and/or to deflect the panel edge laterally to launch transverse bending waves along a panel
10 edge, and/or to apply torsion across a panel corner, and/or to produce linear deflection of a local region of the panel.

Assessment of acoustic output from panel members may be relative to suitable criteria for acoustic output
15 include as to amount of power output thus efficiency in converting input mechanical vibration (automatically also customary causative electrical drive) into acoustic output, smoothness of power output as measure of even-ness of excitation of resonant mode of bending wave action,
20 inspection of power output as to frequencies of excited resonant modes including number and distribution or spread of those frequencies, each up to all as useful indicators. Such assessments of viability of locations for transducer means constitute method aspects of this invention
25 individually and in combination.

As aid to assessment at least of smoothness of power output, it is further proposed herein to use techniques based on mean square deviation from some reference. Use

of the inverse of mean square deviation has the benefit of presenting smoothness for assessment according directly to positive values and/or representations. A suitable reference can be individual to each case considered, say a median-based, such as represented graphically by a smoothed line through actual measured power output over a frequency range of interest. It is significantly helpful to mean square deviation assessment for the reference to have a be normalised standard format; and for the measured acoustic power output to be adjusted to fit that standard format. The standard format may be a graphically straight line, preferably a flat straight line thus corresponding to some particular constant reference value; further preferably the same line or value as found naturally to apply to a distributed mode panel member at higher frequencies where modes and modal action are more or most dense.

In this connection it is seen as noteworthy that whatever function is required for such normalising to a substantially constant reference is effectively also a basis for an equalisation function applicable to input signals to improve lower frequency acoustic output. It is the case that viable distributed mode panel members as such, and with preferential aspect ratios and bending stiffness(es) as in our above patent application, may naturally have acoustic power output characteristics relative to frequency that show progressive droops towards and through lower frequencies where resonant modes and

modal action are less dense - but, as their frequency distribution as such is usually beneficial to acoustic action in such lower frequency range, such equalisation of input signal can be useful. This lower acoustic power output at lower frequencies is related to free edge vibration of the panel members as such, and consequential greater loss of lower frequency power, greater proportion of which tends to be poorly radiated and/or dissipated, including effectively short-circuited about free adjacent panel edges. As expected, these lower frequency power loss effects are significantly greater for panel members with transducer locations at or near their edges and/or lesser stiffnesses - compared with panel members using in-board transducer locations. However, and separately from any input signal equalisation, significant mitigation of these effects is available by mounting the panel members surrounded by baffles and/or by clamping at the edges of the panel members. Indeed, spaced localised edge clamps can have usefully selectively beneficial effects relative to frequencies with wavelengths greater than the spacing of the localised edge clamps.

Interestingly, for specific panel members of quite high stiffnesses, viable marginal transducer locations include positions having edge-wise correlation with normally in-board locations for transducer means arising as preferred by application of teachings or practice such as specifically in our above patent applications. When using transducer means in pairs, a first preference was

found for marginal transducer locations with said correlation as corresponding to notionally encompassing greatest area. For a substantially rectangular panel member, said correlation can be by way of correspondence with orthogonal or Cartesian co-ordinates, with said first preference represented by associating transducer means with diagonally opposite quadrants. However, this was in relation to a particularly high stiffness/high-Q panel member, and is not always true, even for quite (but less) stiff panels, see further below showing promising operation with association in some or adjacent quadrants. For an elliptical panel member said correlation/correspondence can be according to hyperbolic resonant mode related lines as going edge-wards through the in-board locations. Other variously less good, but feasibly viable, pairs of edge locations for transducer means were found by investigation based on rotating orthogonal vectors about in-board preferential transducer locations, including close to or at corner positions of panel members. Another inventive aspect regarding corner or near-corner excitation involves suitably mass-loading or clamping substantially at a known in-board optimal or preferential drive location, where it appears that such mass-loaded optimal drive location(s) effectively behave(s) to some useful extent as "virtual" source(s) of bending wave vibrations in the member. This latter may not avoid central intrusion by the mass loading but is clearly germane to successful marginal excitation at

corners.

Further investigations have been made, including of panel members having different stiffnesses, specifically again quite high but also much lower and intermediate stiffness panels, in each case of usual substantially rectangular configuration with aspect ratios and axial bending stiffnesses generally as in WO97/09842.

For the higher stiffness panel member, assessment based on smoothness of power output for single transducer locations along longer and shorter edges were generally confirmatory of above preferential coordinate positions, i.e. peaking as expected for best locations for single transducer means. However, additionally, longer edges had promising spreads of smoothness measure within about 15% of peak at transducer locations between the co-ordinate positions in each half of the edge and beyond those co-ordinate positions to about one-third length from each corner; and within about 30% along to at least the quarter length positions. For the shorter edges, spreads of smoothness measure were within about 10% between the co-ordinate positions, and within about 25% at quarter length positions. The shorter edges actually showed a better power smoothness measure than the longer edges showed at quarter length positions right through to within about one-tenth length of the corners.

Investigation of combinations of two transducers has also been extended particularly for same and adjacent quadrants with one transducer, for one on each of longer

and shorter edges. One transducer can be at one best position along one of the edges for a single transducer, with the other transducer varied along the other edge. For variation along the shorter edge, above preference for one of positions according to co-ordinates of in-board preferential transducer locations is confirmed by best smoothness measure at about six-tenths length. There are also near as good positions at three-quarter length and only a little less good at quarter and third length positions. Moreover, most positions other than below about one-tenth from a corner are better, similar, near as good, or not much worse, than for association with co-ordinates of preferred in-board locations in the same quadrant. For variation along the longer edge, the shorter edge transducer was located at about preferred near six-tenths position, there was then actually marked preference for combinations of transducer locations in adjacent quadrants, with best at just under one-fifth, and slightly better than the 0.42 position at the one-third length position with only a little worse at the one-tenth length position. The quarter length position is actually about the same as for the mid-length position and the adjacent quadrant position of the co-ordinate of preferred in-board location. Self-evidently, these procedures may be continued on an iterative basis, and may then reveal more favourable combinations.

Investigations of much lower stiffness panel members on the basis of smoothness of power output have shown

peaking for marginal transducer locations also at about the in-board co-ordinate position, but near as good at quarter length of panel edges, and generally markedly less criticality as to position along the edges in terms of actual achieved modal distribution. This is seen as explicable by interaction between the lower panel stiffness and compliance within the used transducer itself. It appears that the resonant modal distribution of the panel is affected and altered by the transducer location, at least to some extent going with such location. Higher panel stiffnesses substantially avoid such effects. However, such in-transducer compliance and possible interaction with panel stiffness/elasticity is clearly another factor to be taken into account, including exploited usefully.

Investigations of panel members with quite high and much lower stiffnesses clearly reveal rather different cases for application of marginal excitation, including as to more and less criticality as to transducer locations, whether singly or in pairs, and as to less or more interaction with in-transducer compliance. It is thus appropriate to consider a panel member of intermediate stiffness.

For such intermediate stiffness panel member, and much as expected, differences relative to the much lower stiffness panel member include increase in acoustic power output available by edge clamping, markedly increased power for mid-range frequency modes, and stronger modality

or peakiness for lower-frequency modes. Tendency towards characteristics of the higher stiffness panel member include stronger preference as best single transducer locations for edge positions on a co-ordinate of optimal in-board transducer locations, also promising feasibility for through the mid-point, but perhaps also at about one-tenth in from corners. For two marginally located transducer means, marked preference resulted for the co-ordinate related position of optimal in-board transducer location, with less good but likely viable spread to middle and two-thirds length positions and equality of same quadrant co-ordinate related and two-thirds length positions.

It is evident that differences in materials parameters of panel members beyond basic capability to sustain bending wave action are significant in determining marginal transducer locations; and that use of two or more such transducer locations produces highly individual solutions requiring experimental assessment such as now enabled by teachings hereof.

Also, at least specifically for tested substantially rectangular panel members, it has been found that many if not most, probably going on all, of edge or near-edge locations for transducer means that are unpromising as such can be significantly improved (as to bending wave dependent resonant mode distribution and excitement into acoustical response of the member) if associated with localised mass-loading or clamping at one or more selected

other marginal position(s) of the panel member concerned.

Inventive aspects thus includes association of a said drive means position with helpful other mass-loading or clamping position marginal of the panel member.

5 Regarding use of two or more transducer means, exhaustive investigation of combinations of marginal locations is impractical, but teaching is given as to how to find best and other viable marginal locations for second transducer means for any given first transducer
10 marginal location. Indeed, yet further marginal transducer locations could be investigated and assessed according to the teaching hereof. Somewhat likewise, use of localised marginal damping for improving performance for any given transducer marginal location is
15 investigatable and assessable to any extent and number using the teaching hereof, whether for enhancing or reducing contributions of some resonant mode(s), otherwise deliberately interfering with other resonant mode(s), or mainly to increase output power.

20 It believed to be worthwhile generally to take into account the fact that lowest resonant modes are related to length of the longest natural axis of any panel member, thus that longer edges of substantially rectangular panel members are sensibly always favoured for location of
25 transducer means, including doing so wherever feasible at the best position for operation with single transducer means. It is sensible to see this as applying even where use of another transducer means is encouraged or intended,

again whether for enhancing some resonant mode(s), deliberately interfering with other resonant mode(s) or mainly to increase output power.

Also relevant as a general matter is the fact that the operating frequency range of interest should be made part of assessment of location for transducer means, and may well affect best and viable such locations, i.e. could be different for ranges wholly above and extending below such as 500 Hz. Another influencing factor could be presence of an adjacent surface, say behind the panel member at a spacing affecting acoustic performance.

It is inferred or postulated that the nature of preferred said edge or edge-adjacent position(s) tend towards what is fore-shadowed in our above PCT and other patent applications, typically viewed as affording coupling to more approaching most frequency modes, and doing so more rather than less evenly, perhaps typically avoiding dominance of up to only a few frequency modes. Such suitability may be for lower rather than higher total actual vibrational energy locally in the panel member, but high as to population by frequency modes, i.e. rather than "dead" in the sense of little or no coupling to any or few modes.

BRIEF DESCRIPTION OF THE DRAWINGS

Specific implementation for the invention is now diagrammatically illustrated and described in and with reference to by way of example, in the accompanying drawings, in which:-

Figure 1 shows a distributed mode acoustic panel with a fitted transducer as generally described in the above PCT application;

Figure 2 shows outline indication of four different 5 ways of marginal or edge excitation an acoustic panel;

Figure 3 shows possible placements of transducers marginally of an acoustic panel to achieve actions shown in Figure 2, and Figure 3A shows transparent such panel;

Figure 4 shows four favoured marginal locations for 10 transducers shown in outline, relative to an in-board location of Figure 1 shown in phantom;

Figure 5 shows the same four favoured locations relative to another preferential in-board drive location and favoured pair of the complementary or phantom in-board 15 drive location;

Figure 6 indicates how any pairs and all four drive transducers at such favoured locations were connected for testing;

Figure 7 shows viable if less favoured pairs of 20 marginal drive transducer locations;

Figure 8 shows corner drive position and helpful mass-loading at an in-board preferential drive location;

Figures 9 and 9A show four normally unfavoured marginal drive transducer locations together with many 25 marginal mass-loading or clamping positions and how test masses and drive transducers were associated with the panel; and

Figure 10 shows in-board area unobstructed within

marginal positions for drive transducer(s), clamp termination(s) and resilient suspension/mounting.

Figures 11A, B are graphs of output power/frequency for a substantially rectangular panel member of quite high stiffness and single transducer positions along longer and shorter edges;

Figures 12A, B are related bar charts for measures of smoothness of output power;

Figures 13A, B are graphs of output power/frequency for two transducer positions with one varied along shorter or longer edges;

Figures 14A, B are related bar charts for measures of smoothness of output power;

Figures 15A, B are output power/frequency graphs and related power smoothness bar chart for a panel member of much lower stiffness and single transducer positions along the longer edge;

Figures 16A, B are output power/frequency graphs and power smoothness bar chart for second transducer positions along the shorter edge;

Figure 17 shows comparison of power outputs with transducers located preferentially in-board and at edge for the low stiffness panel member;

Figures 18A, B, C show effects of baffling, three-edge clamping and both;

Figures 19A, B are output power/frequency graphs and related power smoothness bar chart for the low stiffness panel member clamped along on three edges and transducer

positions on the fourth edge;

Figures 20A, B are output power/frequency graphs and related power smoothness bar chart for the low stiffness panel member clamped on two parallel edges sides and 5 transducer positions on another edge;

Figures 21A, B are output power/frequency graphs and related power smoothness bar chart for the low stiffness panel member with localised clamping at corners/mid-edges and transducer positions on other longer edge;

10 Figure 22 is a power smoothness bar chart for the low stiffness panel member with further localised clamping between other corner/mid-point clamping;

Figures 23A, B are bar charts for power assessment without normalisation for the low stiffness panel member 15 with three edge clamping of seven-point and full edge nature, respectively, and for position of another local clamp along the other edge at which transducer means has an unfavourable position;

Figures 24A, B are power output/frequency graphs and 20 related power smoothness bar chart for the three-edge clamped case assessed with normalisation;

Figures 25A, B are power output/frequency graphs and related power smoothness bar charts for a panel member of intermediate stiffness and single transducer positions 25 along the longer edge with normalisation;

Figures 26A, B are output power/frequency graphs and power assessment bar chart for the intermediate stiffness panel member with seven point localised clamping assessed

without normalisation;

Figures 27A, B are similar but with normalising for power smoothness assessment;

Figures 28A, B are power output graph and power smoothness bar chart for the intermediate stiffness panel member and a second transducer position along shorter edge;

Figure 29 indicates seven- and thirteen- point localised clamping as applied above;

10 Figure 30 is a schematic diagram useful in explaining impact of in-transducer compliance, and

Figures 31A-E are power efficiency bar charts for the lower stiffness panel member for different edge conditions.

15 DESCRIPTION OF ILLUSTRATED EMBODIMENTS

In Figure 1, distributed mode acoustic panel loud-speaker 10 is as described in WO97/09842 with panel member 11 having typical optimal near- (but off-) centre location for drive means transducer 12. The sandwich structure 20 shown with core 14 and skins 15, 16 is exemplary only, there being many monolithic and/or reinforced and other structural possibilities. In any event, normal in-board transducer placement potentially limits clear area available, e.g. for such as transmission of light in the 25 case of a transparent or translucent panel.

Mainly transparent or translucent resonant mode acoustic panel members might use known transparent piezo-electric transducers, e.g. of lanthanum doped titanium

zirconate. However these are relatively costly, hence the alternative approach thereof by which it is possible to leave the resonant mode acoustic panel member 10 mainly clear and unobstructed by optimising loudspeaker design 5 from a choice of four types of excitation shown in Figure 2 directed to the margins or perimeter of the panel, and labelled as types T1 - T4, as follows:-

T1 - launching compression waves into an edge (shown along 18A) of the panel member 11 - as available by
10 inertial action or reference plane related drive transducers

T2 - launching transverse bending waves along an edge (also shown along 18A) of the panel member 11 - as available by laterally deflecting the panel edge
15 using bender action drive transducers

T3 - applying torsion to the panel member 11 as shown across a corner between edges 18A, B - available by action of either of bender or inertial type drive transducers

20 T4 - producing linear deflection directly at an edge of the panel member 11 as shown at edge 18B - available at local region of contact by inertial action drive transducers.

Figure 3 is a scrap view of composite panel 11
25 showing high tensile skins 15, 16 and structural core 14 with drive transducers/excitors 31 - 34 for the above-mentioned four types T1 - T4 of edge/marginal drive. In practice, fewer than four drive types might be used at the

same time on a panel which can usefully be acoustically and mechanically optimised for the desired bandwidth of operation and for the particular type of drive employed.

Thus, an optimised panel may be driven by any one or more of the different drive types.

A transparent or translucent edge-driven acoustic panel could be monolithic, e.g. of glass, or of skinned core structure using suitable translucent/transparent core and skin materials, see Figure 11. Interpretation with a visual display unit (VDU) may enable the screen also to be used as a loudspeaker, can have suitably high bending stiffness along with low mass if comprising a pair of skins 15A, 16A sandwiching a lightweight core of aerogel material 14A using transparent adhesive 15B, 16B. Aerogel materials are extremely light porous solid materials, say of silica. Transparent or translucent skin or skins may be of laminated structure and/or made from transparent plastics material such a polyester, or from glass.

Conventional transparent VDU screens may be replaced by such a transparent acoustic radiator panel, including with acoustic excitation outside unobstructed main screen area.

A particular suitable silica aerogel core material is (RTM) BASOGEL from BASF. Other feasible core materials could include less familiar aerogel-forming materials including metal oxides such as iron and tin oxide, organic polymers, natural gels, and carbon aerogels. A particular suitable plastics skin laminates may be of polyethylene terephthalate (RTM) MYLAR, or other transparent materials

with the correct thickness, modulus and density. Very high shear modulus of aerogels allow extremely thin composites to be made to suit miniaturisation and other physically important factors and working under distributed 5 mode acoustic principles.

If desired, such transparent panel could be added to an existing VDU panel, say incorporated as an integral front plate. For a plasma type display the interior is held at low gas pressure, close to vacuum, and is of very 10 low acoustic impedance. Consequently there will be negligible acoustic interaction behind the sound radiator, resulting in improved performance, and the saving of the usual front plate. For film type display technologies, again the front transparent window may be built using a 15 distributed mode radiator while the display structures behind may be dimensioned and specified to include acoustic properties which aid the radiation of sound from the front panel. For example partial acoustic transparency for the rear display structures will reduce 20 back wave reflection and improve performance for the distributed mode speaker element. In the case of the light emitting class of display, these may be deposited on the rear surface of the transparent distributed mode panel, without significant impediment to its acoustic 25 properties, the images being viewed from the front side.

A transparent distributed mode loudspeaker may also have application for rear projection systems where it may be additional to a translucent screen or this function may

itself be incorporated with a suitably prepared surface for rear projection. In this case the projection surface and the screen may be one component both for convenience and economy but also for optimising acoustic performance.

5 The rear skin may be selected to take a projected image, or alternatively, the optical properties of the core may be chosen for projection use. For example in the case of a loudspeaker panel having a relatively thin core, full optical transparency may not be required or be ideal,

10 allowing the choice of alternative light transmitting cores, e.g. other grades of aerogel or more economical substitutes. Special optical properties may be combined with the core and/or the skin surface to generate directional and brightness enhancing properties for the

15 transmitted optical images.

Where the transparent distributed mode speaker has an exposed front face it may be enhanced, for example, by the provision of conductive pads or regions, visible, or transparent, for user input of data or commands to the

20 screen. The transparent panel may also be enhanced by optical coatings to reduce reflections and/or improve scratch resistance, or simply by anti scratch coatings. The core and skin for the transparent panel may be selected to have an optical tint, for colour shading or in

25 a neutral hue to improve the visual contrast ratios for the display used with or incorporated in the distributed mode transparent panel speaker. During manufacture of the transparent distributed model panel, invisible wiring,

e.g. in the form of micro-wires, or transparent conductive films, may be incorporated together with indicators, e.g. light emitting diodes (LED) or liquid crystal displays (LCD) or similar, allowing their integration into the transparent panel and consequent protection, the technique also minimising impairment to the acoustic performance. Designs may also be produced where total transparency is not required, e.g. where one skin only of the panel has transparency to provide a view to an integral display under that surface.

The transducers may be piezo-electric or electro-dynamic according to design criteria including price and performance considerations, and are represented in Figure 3 as simple outline elements simply bonded to the panel by suitable adhesive(s). For above T1 type drive excitation, inertial transducer 31 is shown driving vertically directed compression waves into the panel 30. For above T2 type of drive excitation, bending type of transducer 32 is shown operative for directly bending regionally to launch bending waves through the loudspeaker panel 30.

For above T3 type of drive excitation, inertial transducer 33 is shown serving to deflect the panel corner in driving into the diagonal and thence into the whole loudspeaker panel 30. For above type T4 drive excitation another inertial transducer 34 is shown of block or semi-circular form serving to deflect an edge of the loudspeaker panel 30.

Each type of excitation will engender its own

characteristic drive to the panel 30 which is accounted for in the overall loudspeaker design including parameters of the panel 30 itself. The placement of the transducers 31 - 34 along the panel edge is in practice iterated with the panel design parameters for optimum or at least operationally acceptable modal distribution of bending waves. It is envisaged that, according to the panel characteristics, including such as controlled loss for example, and the location(s) and type(s) of marginal edge or near-edge drive, more than one audio channel may be applied to the panel 30 concerned, e.g. via plural drive transducers. This multi-channel potential may be augmented by signal processing to optimise the sound quality, and/or to control the sound radiation properties and/or even to modify the perceived channel-to-channel separation and spatial effects.

Particularly satisfactory drive transducer locations along edges of a substantially rectangular panel member are at edge positions reached by orthogonal side-parallel lines or co-ordinates through an in-board optimal or preferential drive transducer position according to our above PCT application, see dashed at 42 to 45 - 48 in Figure 4. It is actually practical to use drive transducers at at least two such co-ordinate related edge locations 45 - 48. Figure 6 shows in-phase serial and serial/parallel connections for two and four drive transducers at A and B. Other driver connections are feasible, and may often be preferred, including directly

one-to-one to each transducer means; and any desirable signal conditioning may be applied, e.g. differential delay(s), filtering etc, say to suit reduction of undesirable interaction between transducers and/or with electrical signal source and favoured drive transducer positions CP1 - CP4 in Figure 5 relative to in-board preferential location PL. Pairing can be one from each co-ordinate, i.e. CP1 and CP2, CP2 and CP3, CP3 and CP4, CP4 and CP1, and a first favoured pairing is the one notionally defining included area that is greatest, indeed, contains the geometrical centre X. Such notional area will, of course, further pass through or contain other usual optimal or preferential in-board drive transducer position, see complementary location CL and indication at CP5 and CP6 for the first favoured pairing of drive transducer locations.

It has been interesting to note for a very high Q panel that preferred and most preferred pairs of orthogonal co-ordinate related drive locations can produce low frequency output that may be more extended and uniform even than prior preferential in-board much nearer centre positions, albeit with some moderate variation in the higher frequency range. Off-axis response is similar at higher frequencies but actually somewhat more symmetrical at lower frequencies.

Figure 7 shows select results of an experiment where pairs of transducers for which orthogonal angular relative relation is maintained centred on above normal inboard

preferential transducer location, specifically most beneficial for co-ordinate related marginal drive locations SP1 and SP4, but the transducers are tested at positions relatively translated round the panel edge.

5 Most viable/promising pairs of locations are indicated at pairs of positions 1a, 1b to 6a, 6d. Figure 7 actually also shows results of another experiment where pairs of transducers were at opposite ends of straight lines through the preferential in-board drive location SP1, 2.

10 Fewer viable/promising locations were found at positions 2a, 2d and 3a, 3d. More experimental work may well be worthwhile relative to other pairs or more of edge-drive positions, and theoretical/systematising work is being attempted. It will be appreciated from dimensions quoted

15 and as measured at pairs of positions giving viable/promising measured/assessed results that Figure 7 is not strictly to scale.

Figure 8 shows a panel 70 of core 74 and skins 75, 76 structure, and having near-corner-mounted transducer 72

20 with mass loading 78 substantially at an otherwise normal in-board preferential transducer, actually the one or in the group furthest away from the corner of excitation by the transducer 72, which is found to be particularly effective in appearing to behave as a "virtual" source of

25 bending wave vibrations. It can be advantageous for the transducer to avoid or at least couple outside a position with a co-ordinate location substantially centred at 5% of side dimensions from the corner as such, where it has been

established that many resonant mode(s) have nodes, i.e. low vibrational activity.

Turning to Figure 9, outline is indicated for an investigation involving select single positions for one edge or edge-adjacent transducer mounting, see at ST1 - ST4 for in-corner, half-side length, quarter-side length and three-eighths side-length, respectively; and select positions for edge-clamping/mass-loading at edge positions about the panel. An exciting transducer was used, see 92 10 in Figure 9A relative to panel 90, along with loads/clamps by way of panel flanking/gripping 93A/B magnets.

Performance using the corner exciting transducer position ST1 was aided by mass-loading as in Figure 9A at positions Pos. 13, 14, 18, 19 - including in further 15 combination with other positions. For exciting transducer position ST2, good single mass-loading positions are Pos. 6, 7, 8 perhaps 9, 11 particularly, 12, 15 - again including combinations with other positions. Combinations 5 = 11 and 6 + 11 were of particular value, including in 20 further combinations. For exciting transducer position ST3, good single mass-loading positions are Pos. 5, 6, 7, 13, especially the combinations 5 + 13 and 10 + 13, the combination 6 + 18, and combinations/further combinations.

For exciting transducer position ST4, best positions 25 appear to be 6, 18 but neither was as good as those for the other exciter positions ST1 - ST3.

Figure 10 shows a panel-form loudspeaker 80 having an in-board unobstructed region 81 extending throughout and

beyond normal in-board preferential drive transducer locations, and a marginally located transducer 82. The region 81 may serve for display purposes directly, or represent something carried by the panel 80 without affecting acoustic performance, or something behind which the loudspeaker panel 80 passes, say in close spacing and/or transparent or translucent. Both of loudness and quality are readily enhanced, the former by additional drive transducers judiciously placed (not shown), and quality by localised edge clamping(s) 83 beneficially to control particular modal vibration points effectively as panel termination(s). The panel 80 is further indicated with localised resilient suspensions 84 located neutrally or even beneficially regarding achieved acoustic performance. High pass filtering 85 is preferred for input signals to drive transducer(s) 82, conveniently to limit to range of best reproduction, say not below 100Hz for A4-size or similar panels. Then, there should not be any problematic low-frequency panel/exciter vibration.

20 It is advantageous in terms for acoustic performance to control acoustic impedance loading on the panel 80, say to be relatively low in the marginal or peripheral region, especially in the vicinity of the drive transducer(s) 82 where surface velocity tends to be high. Beneficial such control provision includes significant clearance to local planar members (say about 1 - 3 centimetre) and/or slots or other apertures in adjacent peripheral framing or support provision or grille elements.

It is further feasible and advantageous deliberately to arrange for such as mechanical damping to result in acoustic modification including loss in the area 81, or even also marginally thereof, not to be obstructed, at least for higher frequencies. This may be done by choice of materials, e.g. monolithic polycarbonate or acrylic and/or suitable surface coating or laminated construction.

Resulting effective concentration of acoustic radiation to marginal regions about plural drive transducers particularly facilitates reproduction of more than one sound channel, at least for near-field listening as for playing computer games or like localised virtual sound stage applications. Further away, merging even of multiple as-energised sound sources need not be problematic when summed, at least for such as audio visual presentations.

The following Table gives relevant physical parameters of actual panel members used for investigation to which Figures 11-28 relate.

	Lower Stiffness Panel	Higher Stiffness Panel	Intermediate Stiffness panel
Core material	Rohacell	Al honeycomb	Rohacell
Core thickness	1.5mm	4mm	1.8mm
Skin material	Melinex	Black glass	Black glass
Skin thickness	50 μm	102 μm	102 μm
Panel Area	0.06m ²	0.06m ²	0.06m ²
Aspect ratio	1:1.13	1:1.13	1:1.13
Bending stiffness	0.32 Nm	12.26 Nm	2.47 Nm
Mass density	0.35 kgm ⁻²	0.76 kgm ⁻²	0.6 kgm ⁻²
Zm	2.7 Nsm ⁻¹	24.4 Nsm ⁻¹	9.73 Nsm ⁻¹

Figures 11-14 relate to the higher stiffness panel member of the first column, Figures 15-24 to the much lower stiffness panel member of the second column, and Figures 25-28 to the intermediate stiffness panel member of the third column.

All of the graphs have acoustic output power (dB/W) as ordinate and frequency as abscissa, thus show measured acoustic output power as a function of frequency, typically as a truly plotted dotted line. Most of the graphs also show an upper adjustment of the true power line. As mentioned in the preamble, this adjustment is by way of applying functions that normalise to a flat straight line, and allows assessment of resonant modality free of often encountered effects of fall-off of power at

lower frequencies. It is found that smoothness of power makes significant contribution to quality of sound. From such normalised value of the actual power output, it is advantageous to produce assessment of smoothness by inverse of mean square deviation, and most of the bar plots are of that type.

The higher stiffness panel member for Figures 11-14 is actually somewhat less stiff than that used for previous Figures 7 and 9, but does clearly show preference for single transducers to be located at positions corresponding to co-ordinates of in-board transducer locations previously established as optimal, i.e. at about $3/7$, $4/9$ length from any corner or about 0.42-0.44. However, there are substantial spreads of promising potential location between and beyond such positions for each edge, actually within about 10% and 15% in the mid-regions of shorter and longer edges, respectively, and further within 28% and 30% at quarter-length positions.

At least for the most part, trial positions for transducer edge or near edge location are based on spacing substantially corresponding to the difference between the preferential co-ordinate value of 0.42 for in-board transducer location and the mid-point (0.5) of the edge, albeit with alternate spacings increased to 0.09. Usual trial locations are thus 0.08, 0.17, 0.28, 0.33, 0.42, 0.50.

In the main, it is believed that the illustrated graph and bar charts are substantially self-explanatory as

to showing best and presumably promising locations for transducers, and for localised clamping as feasible for improving less promising transducer locations, see Figures 23.

5 As far as single transducer edge or near-edge location is concerned, the other two tested panel members of much lower and intermediate stiffnesses also show the same in-board co-ordinate preference on a smoothness of power basis, see Figures 15 and 25. However, the lower
10 stiffness panel member shows another band of nearly as promising locations ranging from about quarter to below tenth length from corners. Interestingly, if assessment is based on efficiency, i.e. amount of power output - as would be the case for a median line through the true
15 output power plot being the basis used for mean square deviation - the above band becomes skewed to emphasise the quarter length position and is mostly preferential to the in-board coordinate related position, see inverse mean square deviation bar chart of Figure 31A. The
20 intermediate stiffness panel member veers towards the characteristic of the higher stiffness panel member in showing a promising spread between the in-board preferential coordinate positions, but also shows promise at about the one-tenth length positions.

25 It will be appreciated from inspection of true output power plots by those skilled in the art that there are differences between indicated best and viable transducer edge locations in terms of impact on expected quality of

sound reproduction - for which modality is normally taken as a significant factor, i.e. number and evenness of excitation of resonant modes. If characteristics such as modality are seen as more promising for locations indicated as preferential on the basis of assessing smoothness of output power, it is, of course, feasible to process input signals towards what is shown after above normalising - specifically selectively to amplify low frequency in a form of signal conditioning or equalising.

10 This would achieve, indeed exceed, power available using locations optimised on efficiency basis; but obviously not the efficiency itself as more input power has to be used.

Accordingly, other ways of increasing lower frequency power were investigated as foreshadowed above, namely baffling and/or selectively spaced local clamping or full edge clamping. Figures 18A, B, C give indication of generally beneficial raising of lower frequency output for surrounding baffling with an area over 60% greater than

20 the low stiffness panel, rigid clamping of all three edges not affording transducer location, and both of such baffling and clamping. Such baffling tends to maintain modality but may not always be feasible in specific applications. Accordingly, full investigation of clamping

25 seemed worthwhile for alternative transducer edge locations for the lower stiffness panel member. Results showed that assessment on an efficiency basis tended to emphasise the quarter length point for both of full edge

clamping at true parallel edges or three edges, and 7-point local edge clamping at corners and mid-points as at 'X' in Figure 29, with the edge of transducer location unclamped along its length, see bar charts of Figures 31B, 5 C and D, respectively. However, 13-point clamping as at 'X' + 'O' in Figure 29 shifted emphasis strongly to the in-board preferential coordinate position. Assessment of panel members with clamping on the basis of power smoothness produces much the same results for indication 10 of best transducer locations, see bar charts of Figures 19A, 20B, 21B and 22, but with considerable differences as to next favoured positions, as is generally confirmed by inspection of true output power plots.

Indeed, particularly strong general correlation is 15 found between preferences based on skilled inspection and assessment according to smoothness of power output. In turn, this tends to confirm at least slight preference for such assessment unless there are practical factors that lead to preference for efficiency rather than quality - 20 though that may not be much different anyway.

Another application for localised edge clamping is in relation to improving an unpromising transducer edge location, see bar charts Figures 23A, B showing right hand rather than left hand sides of the edge concerned as 25 otherwise in the drawings. The cases concerned relate to the lower stiffness panel member, and are full clamping of three edges and seven point clamping, with a localised clamp varied along the same edge as the transducer means.

In both cases, useful improvement results at about the quarter length position from the corner more remote from the exciter - see reference bar at right hand side of Figure 23B for no clamping condition. The spread is 5 greater for the full edge clamping case, see Figure 23A.

Where there is disagreement between assessments based on power efficiency and power smoothness, it is worth bearing in mind that any panel member with clamping of corners to the edge with which the transducer is 10 associates effectively has forced nulls at the corner.

There thus must be up to half wavelengths distance for resonant modes concerned before vibrational activity can reach anti-nodal peaks. If preference for a close-to-corner transducer location is indicated by power 15 smoothness assessment, it should be treated with caution as it could be of low power/efficiency, even though smooth by reason of coupling to all resonant mode waveform concerned at may be quite small rises in their waveforms.

Checking with the corresponding power/efficiency 20 assessment is thus recommended. Indeed, best is always likely to be where there is substantial agreement between the two bases of assessment, or some compromise particularly suited to a specific application; and preferably further taking account of skilled inspection of 25 power/frequency graphs perhaps advantageously with as well as without any normalisation for assessment purposes.

For the investigated panel members with higher and intermediate stiffnesses, there is a considerable measure

of consistency as to best transducer edge locations, but with quite marked difference as to other promising locations. The much lower stiffness panel member is markedly less critical as to promising transducer edge locations.

This position is yet more apparent when considering use of more than one transducer means associated with edges of the same panel member. The position for increased coupling to the resonant modes of a panel member is accompanied by complexity of their inevitable combined interaction with the natural distributed resonant vibration pattern of the panel member, and compounded by such distributed vibration pattern being available only at panel edges. There are notable variations from simple rules such as based on coordinates of established preferential in-board transducer location. However, the assessment procedures hereof afford valuable tools for finding good combinations of edge-associated transducer locations.

For the higher stiffness panel of the above Table, Figures 13A, 14A one transducer means is located at a position within the tolerance range of about 0.38-0.45 for the 0.42 preferred position for single transducer means along the longer edge. Second transducer means is varied along the closest shorter edge and Figure 14A shows marginal preference for the furthest 0.42 preferred position, i.e. centred at 0.58, compared with several other positions at about quarter, third and two-thirds

lengths from the common corner. Interestingly, fixing the second transducer means at such about 0.58 preferred position along the shorter panel edge, and varying the other transducer along the longer pane edge (see Figures 5 13B, 14B), produced best and next best preferences at about the one-fifth (0.17) and quarter length positions along the longer panel edge, both showing better than the start position (about 0.42) for power smoothness. This is a procedure clearly capable of further application in an
10 iterative manner, though it is recommended that either or both of power/efficiency assessment and skilled inspection be deployed, particularly if there is no convergence of location in the procedure or any indicated good position is less good in practice than hoped (or was before in the
15 procedure).

Figures 16A, B show results of investigation of the much lower stiffness panel member with the preferred about 0.42 transducer location used for the longer edge and a second transducer varied along the nearest shorter edge.
20 There were no great differences in power smoothness increase, the best three approaching corners and the nearest 0.42 preferential position, with some otherwise general preference for associations being in some quadrant.

25 The same investigation for the intermediate stiffness panel member showed strong preference for the adjacent quadrant preferential 0.42 transducer location (actually 0.58), see Figures 28A, B.

Reverting to the case of the much less stiff panel member, two effects are seen as contributing to much less well-defined best/near best exciter position. One is that the panel modes for the range of frequencies of the optimisation are higher than for stiffer panel members.

The panel member is therefore a closer approximation to a continuum, and smoothness of output power is less dependent on transducer position, particularly second transducer positions.

10 The other effect concerns the much lower mechanical impedance of the panel member, which leads to a less strong dependence on transducer position for energy transfer. The mechanism involved is now explained.

 The mechanical impedance (Z_m) of a panel member determines the movement resulting for an applied point force, see 100, 101 in Figure 30. An object associated with the panel with a mechanical impedance put very much less than, even approaching comparable to, the panel impedance will strongly offset panel motion where the object is located. Associating an exciting transducer of moving coil type with the panel is equivalent to connecting the panel to a grounded mass (the magnet cup of the transducer, see 102) via a spring (the voice coil suspension of the transducer, see 108). When the impedance of such spring is too close to the panel impedance, it will in some part determine the panel motion at the transducer. In the limit of this spring wholly determining the point motion at the transducer, there

would be no dependence of input power on exciter position.

In practice the ratio of spring impedance to panel impedance can so profoundly affect best transducer location, and results are no longer so clear for best/near 5 best transducer locations.

This low mechanical impedance has more effect for edge transducer location than for in-board transducer location as mechanical impedance is yet lower at the panel edge, which means that a transducer, voice coil suspension 10 has a larger effect. Specifically, for the lower stiffness panel of the above Table:

mechanical impedance in the body of the panel is

$$Z_{\text{body}} = 2.7 \text{ Nsm}^{-1}$$

mechanical impedance at the panel edge is approximately 15 half Z_{body} , i.e.

$$Z_{\text{edge}} = 1.3 \text{ Nsm}^{-1}$$

Compliance of the voice coil suspension of the transducer used is:

$$C_{\text{ms}} = 0.52 \times 10^{-3} \text{ mN}^{-1}$$

20 The mechanical impedance at each of modal frequencies can be an order of magnitude lower than the average impedance, Z_{edge} . It is therefore feasible to estimate a typical frequency, below which the exciter has a strong effect on the panel member, say where impedance of the 25 voice coil suspension is about one-fifth of the average impedance at the panel edge. Then,

$$\omega \times Cms = \frac{1}{5} \times Z_{medge}$$

and gives an estimate of 1200 Hz, below which the transducer and panel are intendedly coupled, which is 5 within the frequency range of optimisation.

Considering the transducer and such low mechanical impedance, panel member as one coupled system the transducer in part determines the impedance of the panel member, and smoothness of the output power is less 10 dependent on the position of the transducer.

Repeating such analysis for the high stiffness panel gives a corresponding frequency of 130Hz, which is outside the frequency range of the optimisation.

The claims defining the invention are as follows:

1. A distributed mode active acoustic device comprising a plural-sided
 5 panel member and a transducer coupled to the panel member, the panel member being
 capable of sustaining bending waves in an operative frequency range over an active area
 of the transverse extent of the panel member with a distribution of resonant modes of
 bending wave vibration determining acoustic performance in conjunction with the
 transducer, the panel member having certain physical parameters the values of which
 10 affect said distribution of resonant modes, said parameters having selected values such
 that said resonant modes are beneficially distributed in frequency over said active area,
 and the panel member further having at least one in-board region of said active area
 where a plurality of lower frequency resonant bending wave modes in the operative
 frequency range have vibrationally active anti-nodes, the transducer being coupled to the
 15 panel member at a marginal position of the panel member for beneficial operative
 interaction of the transducer with the panel member, said marginal position corresponding
 to an orthogonal coordinate of said at least one in-board region.

2. A distributed mode active acoustic device according to claim 1,
 20 wherein said panel member has edge clamping means.

3. A distributed mode active acoustic device according to claim 2, wherein
 said edge clamping means is localised.

4. A distributed mode active acoustic device according to claim 3, having
 25 plural said localised edge clamping means.

5. A distributed mode active acoustic device according to claim 4, wherein
 mutual spacing of said plural localised edge clamping means is related to wavelengths of
 30 lower frequency resonant modes so as to raise their contribution to acoustic action of the
 device.

6. A distributed mode active acoustic device according to claim 4 or 5,
 wherein said localised edge clamping means are associated with more than one side.



7. A distributed mode active acoustic device according to claim 6, wherein said panel member is substantially rectangular and said plural localised edge clamping means are associated with three sides not associated with said transducer means.

5

8. A distributed mode active acoustic device according to claim 7, wherein said plural localised edge clamping means are at each corner and at mid-points of said three sides.

10

9. A distributed mode active acoustic device according to claim 2, wherein said edge clamping means extends along said panel member.

15

10. A distributed mode active acoustic device according to claim 9, wherein said edge clamping means extends along at least one side not associated with said transducer means.

20

11. A distributed mode active acoustic device according to claim 10, wherein said panel member is substantially rectangular and said edge clamping means extends along two parallel sides.

12. A distributed mode active acoustic device according to claim 10, wherein said edge-clamping means extends along three sides.

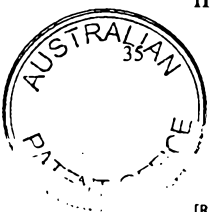
25

13. A distributed mode active acoustic device according to any preceding claim, wherein said panel member has at least two said transducer means in edge association therewith.

30

14. A distributed mode active acoustic device according to claim 13, wherein said panel member is of plural sided form with said transducer means associated with at least two side edges.

15. A distributed mode active acoustic device according to claim 13 or claim 14, wherein said panel member is substantially rectangular with said transducer means associated with longer and shorter sides thereof.



16. A distributed mode active acoustic device according to any previous claim, wherein said panel member is generally rectangular and said marginal position lies on a longer edge in the range 0.38 to 0.45 length from any corner of the panel member.

5

17. A distributed mode active acoustic device according to claim 16, wherein said marginal position is at about 0.42 to 0.44 length from any corner of the panel member.

10

18. A distribution mode active acoustic device according to any previous claim, wherein said panel member is generally rectangular and said marginal position of said transducer is within about 10% and 15% in the mid-regions of shorter and longer edges of the panel member respectively.

15

19. A distributed mode active acoustic device according to any previous claim, wherein said panel member is generally rectangular and said marginal position of said transducer is within 28% and 30% at quarter-length positions of the panel member, respectively.

20

20. Active acoustic device according to any preceding claim, further comprising baffle means extending about and beyond said panel member.

25

21. Active acoustic device according to any preceding claim, wherein said panel member is at least partially transparent or translucent.

22. Active acoustic device according to any preceding claim, wherein said transducer means is of electro-mechanical type.

30

23. Active acoustic device according to any preceding claim, wherein said transducer means is operative to launch compression waves into edge of said panel member and/or deflect edge of said panel member laterally to launch transverse bending waves along said panel member and/or to apply torsion across a corner of said panel member and/or to produced linear deflection of a local edge region of said panel member.



24. A distributed mode active acoustic device substantially as hereinbefore described with reference to the accompanying drawings.

5

Dated 5 March, 2002
New Transducers Limited
Patent Attorneys for the Applicant/Nominated Person
SPRUSON & FERGUSON

2002
MAR 5



1/27

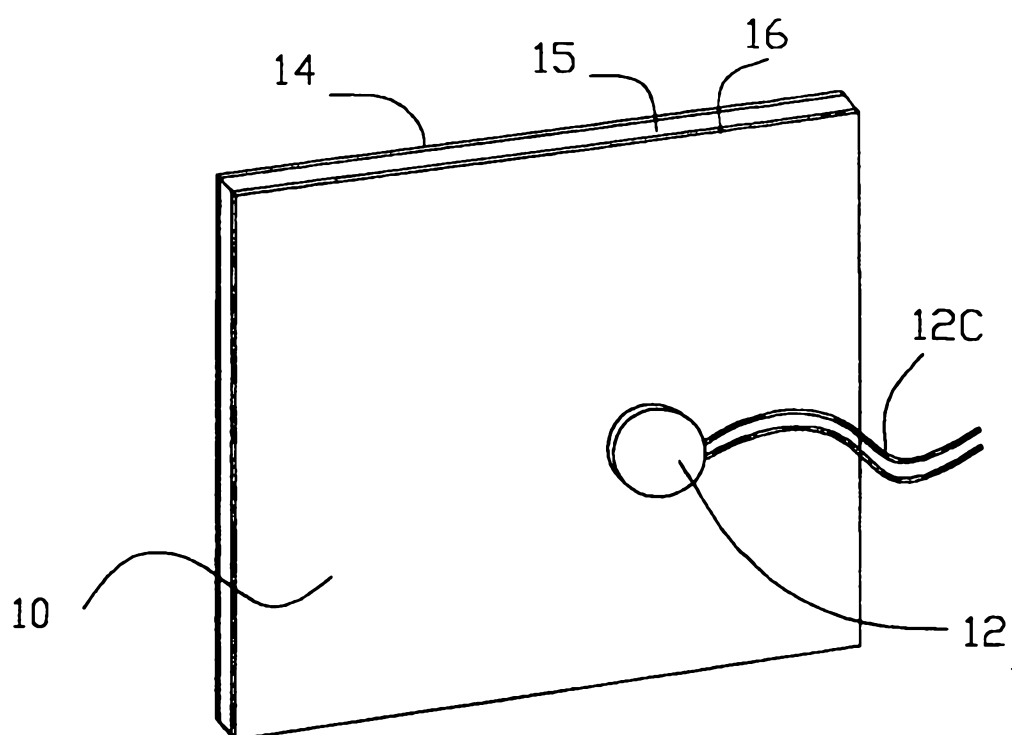


FIG. 1

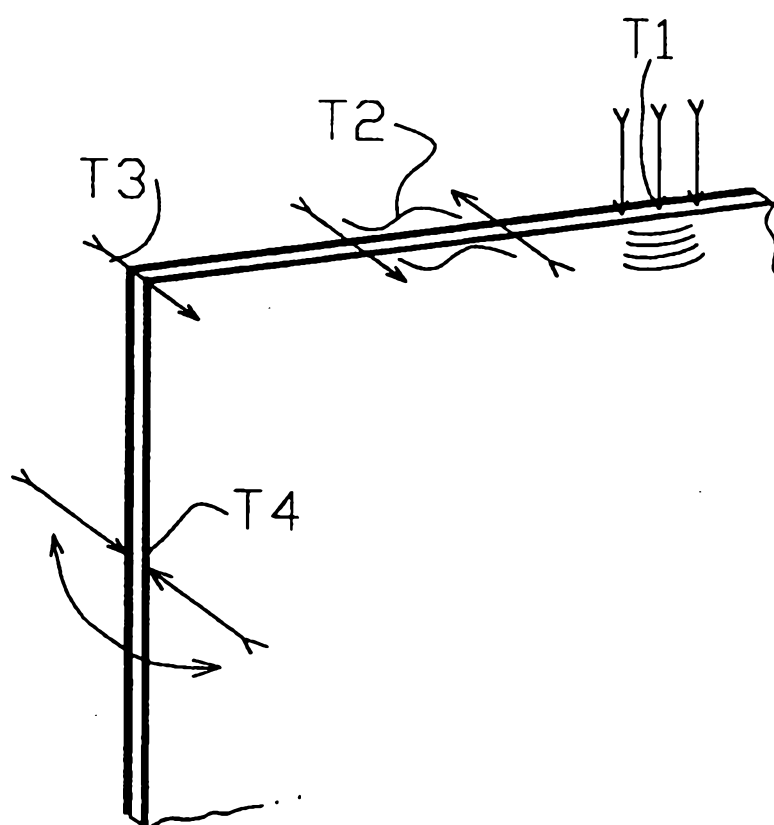
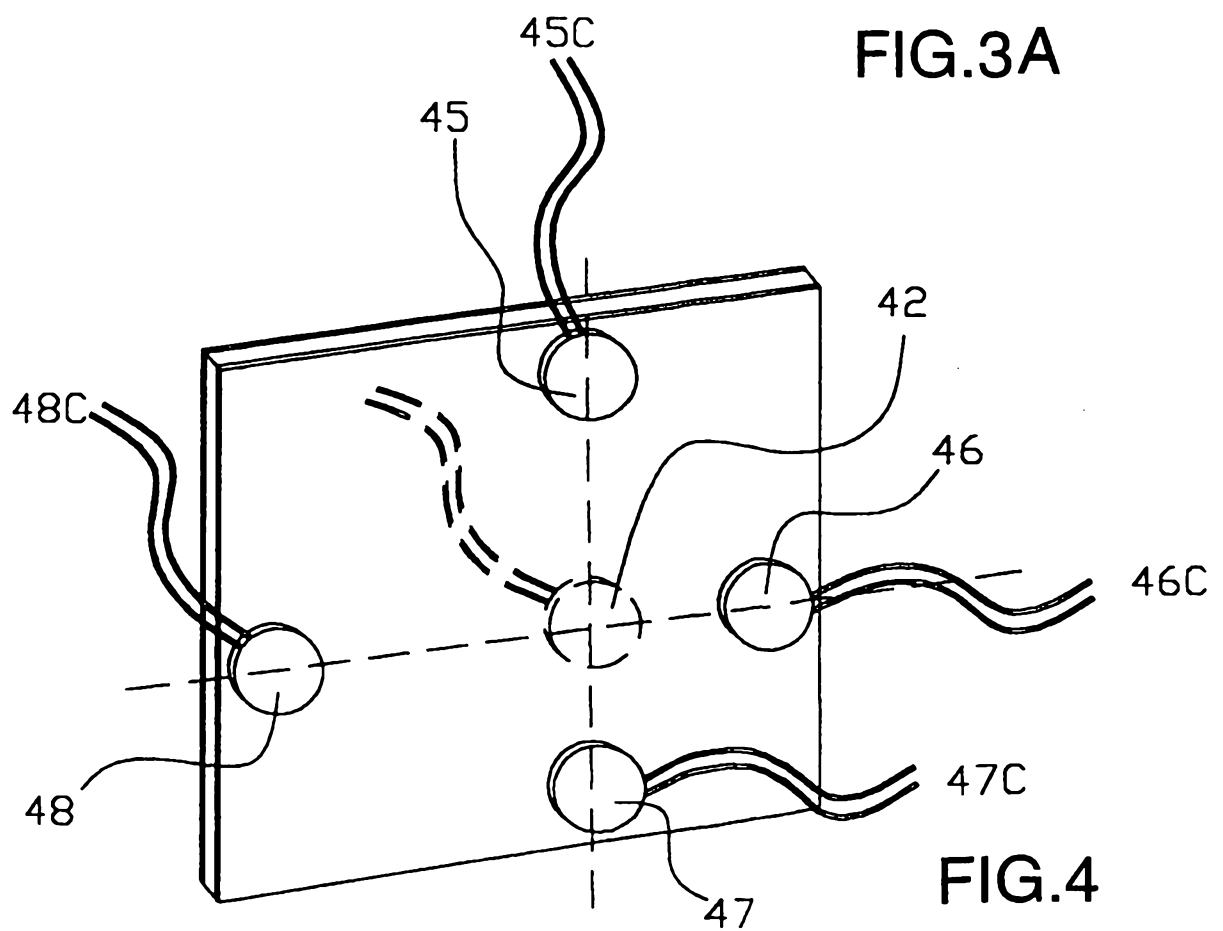
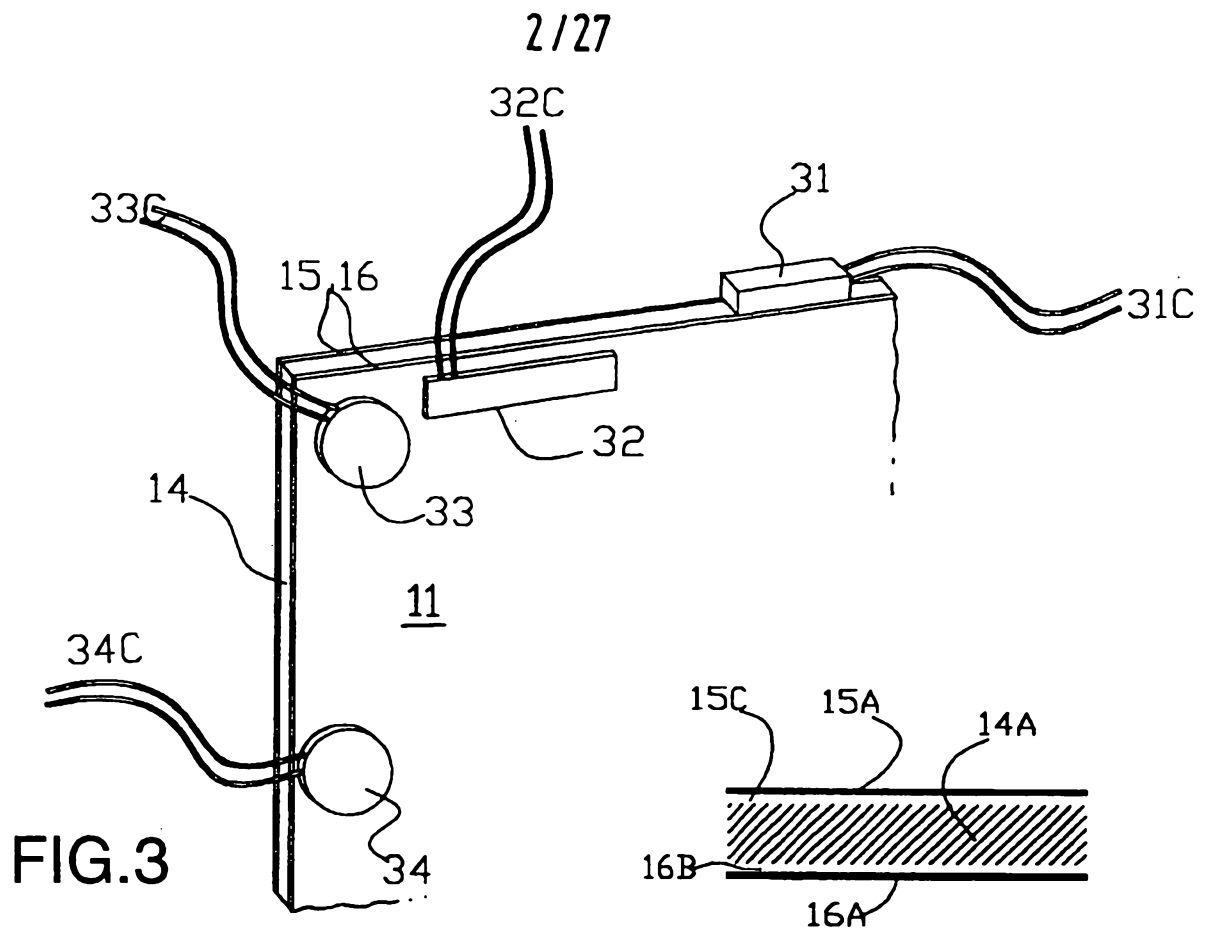


FIG. 2



3/27

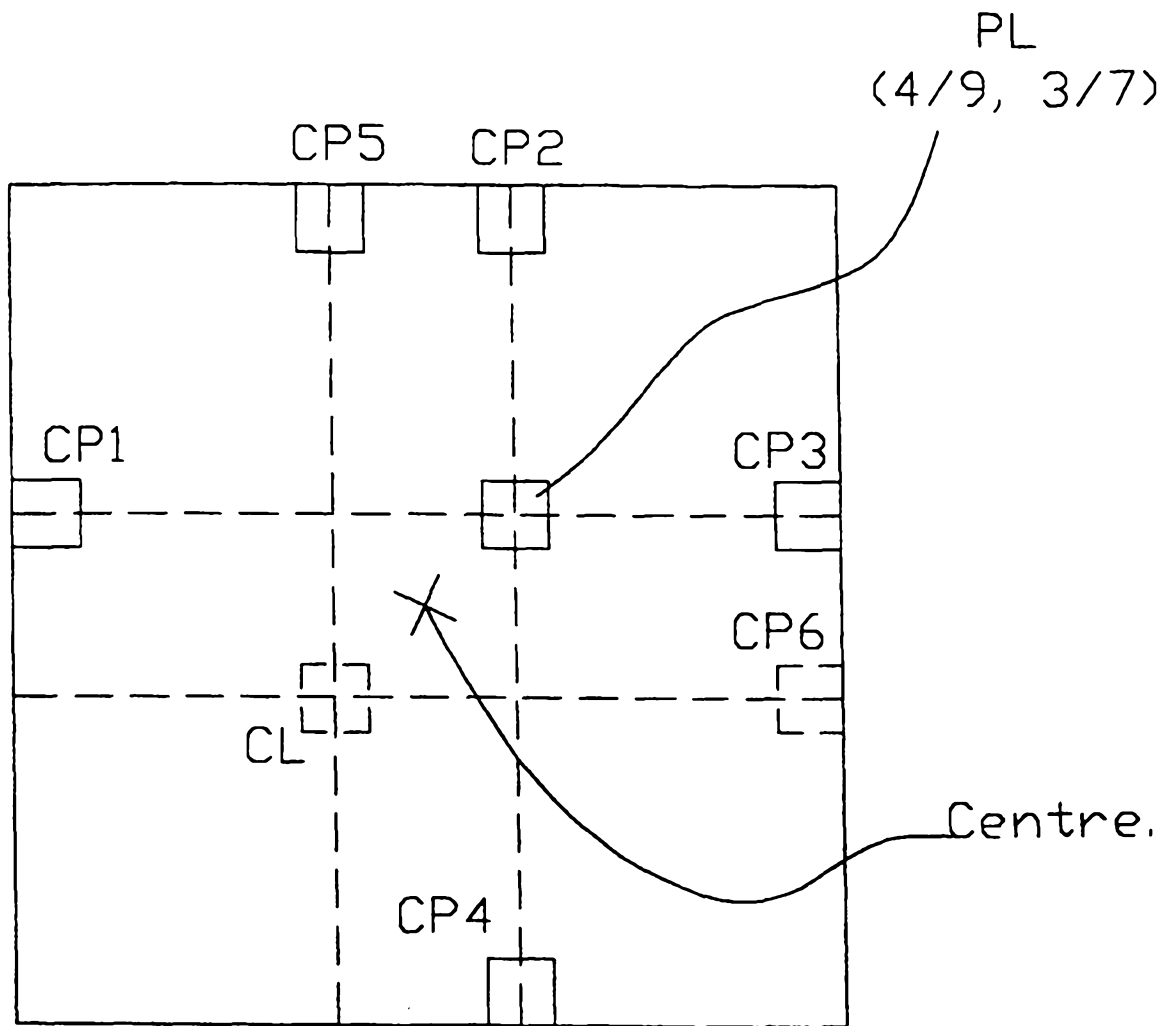


FIG. 5

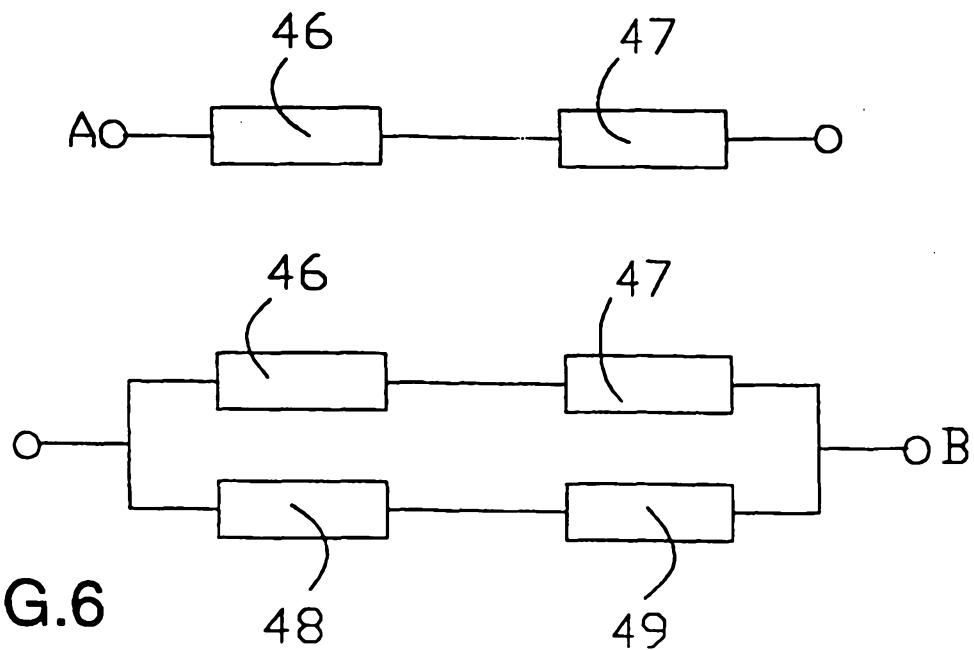
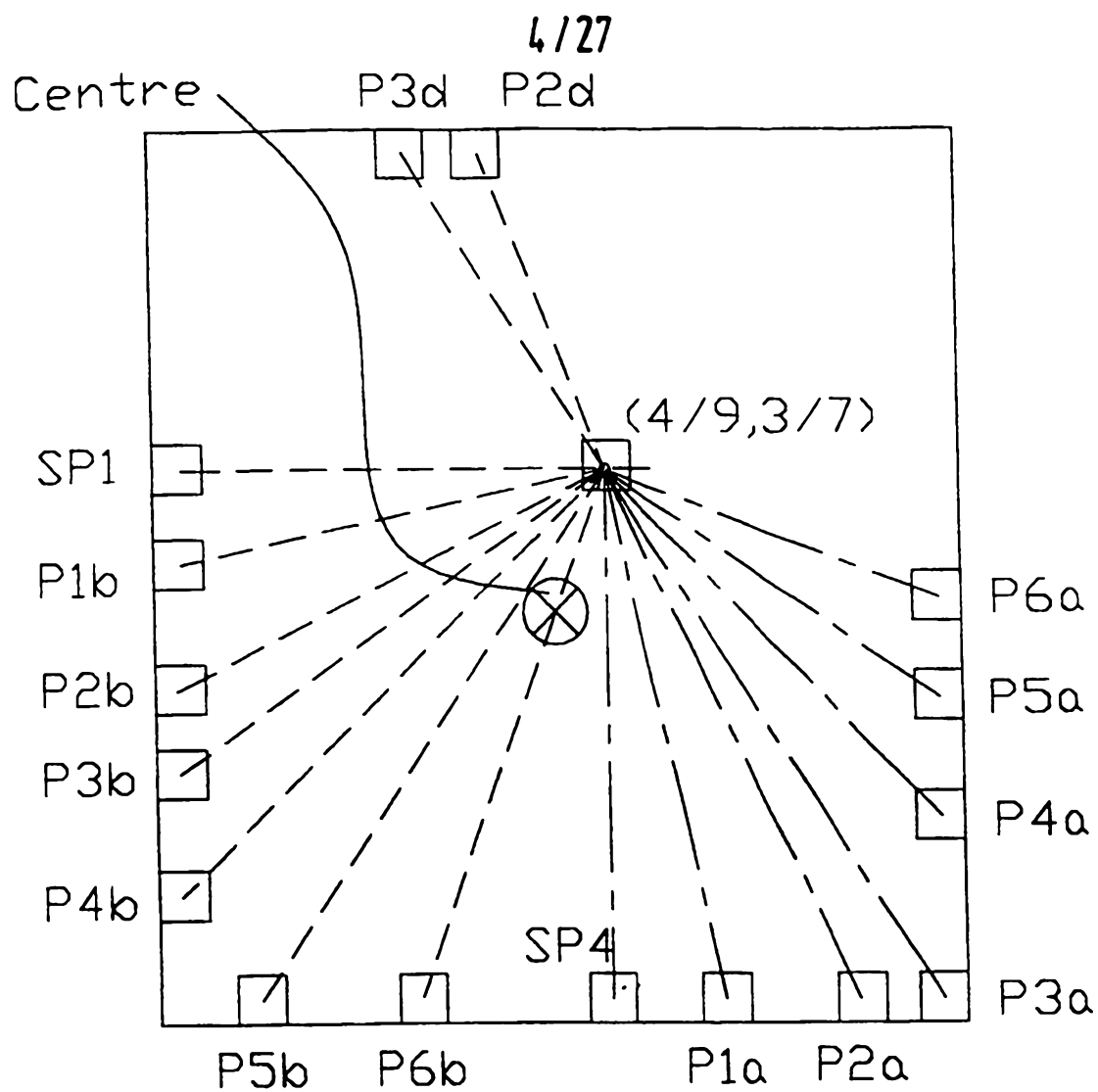
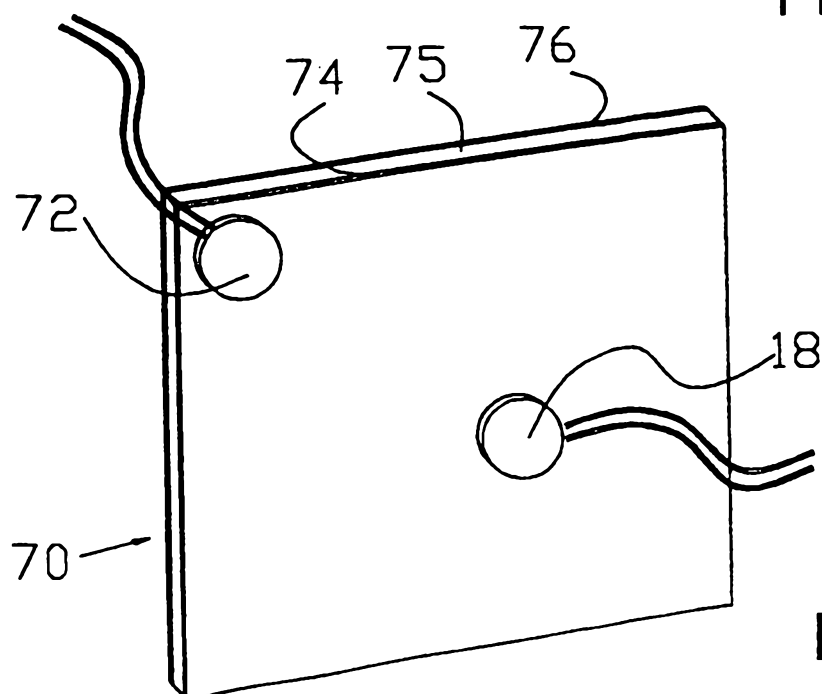


FIG. 6

**FIG. 7****FIG. 8**

5/27

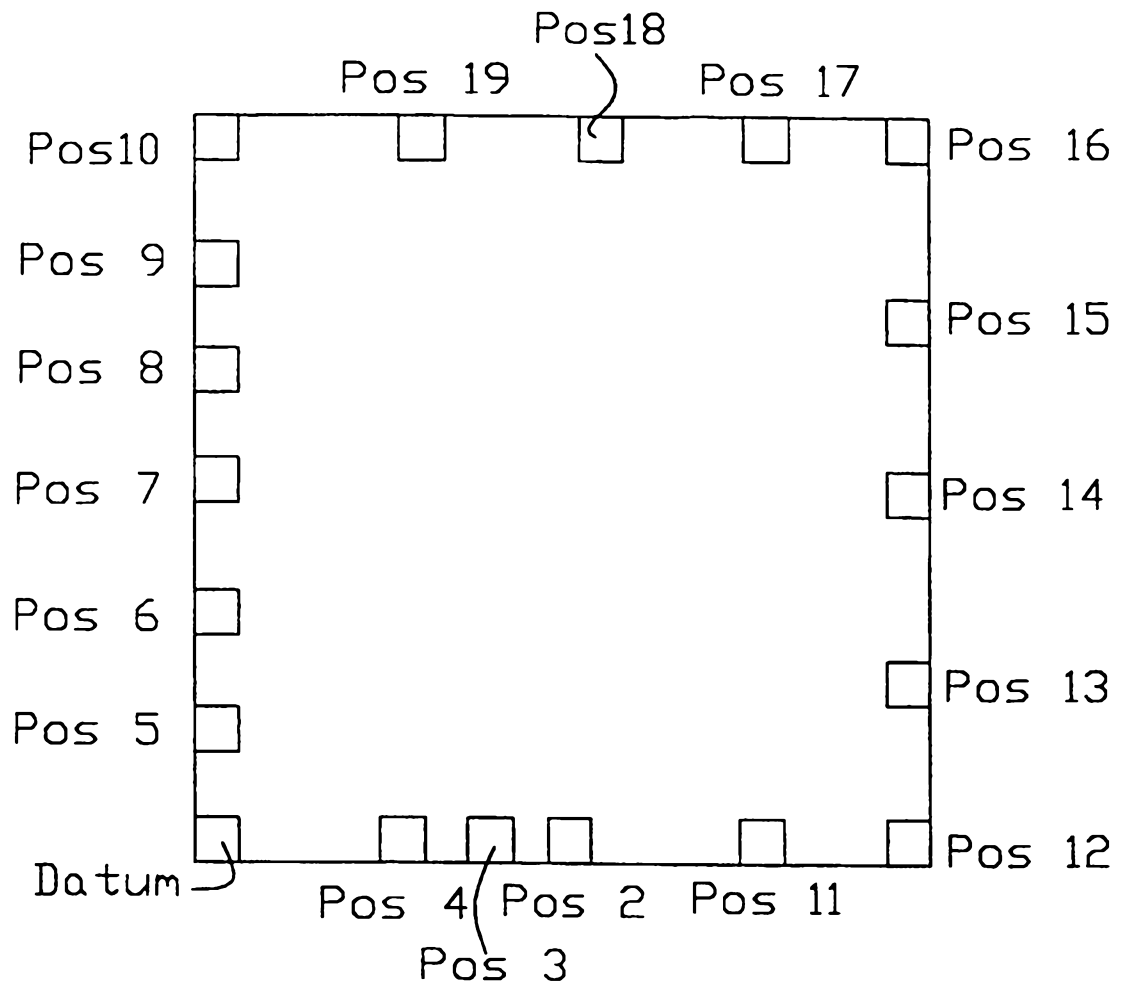


FIG. 9

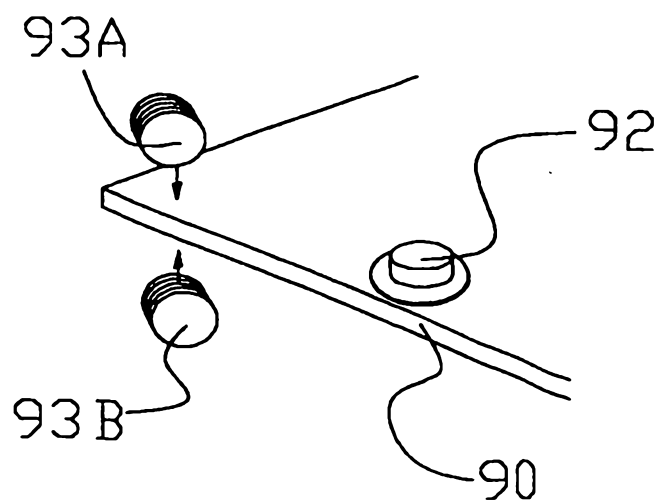


FIG. 9A

6/27

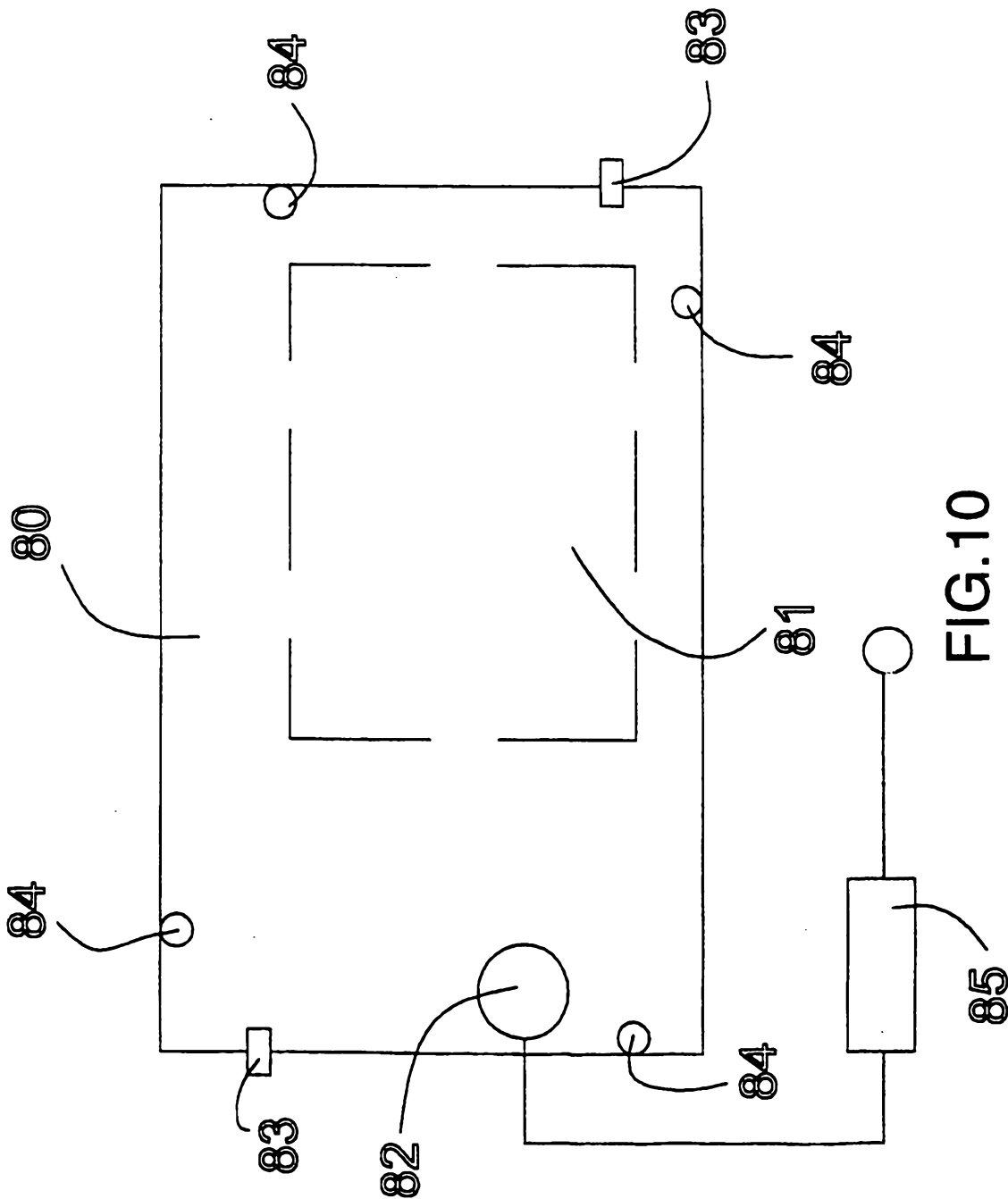


FIG.10

7/27

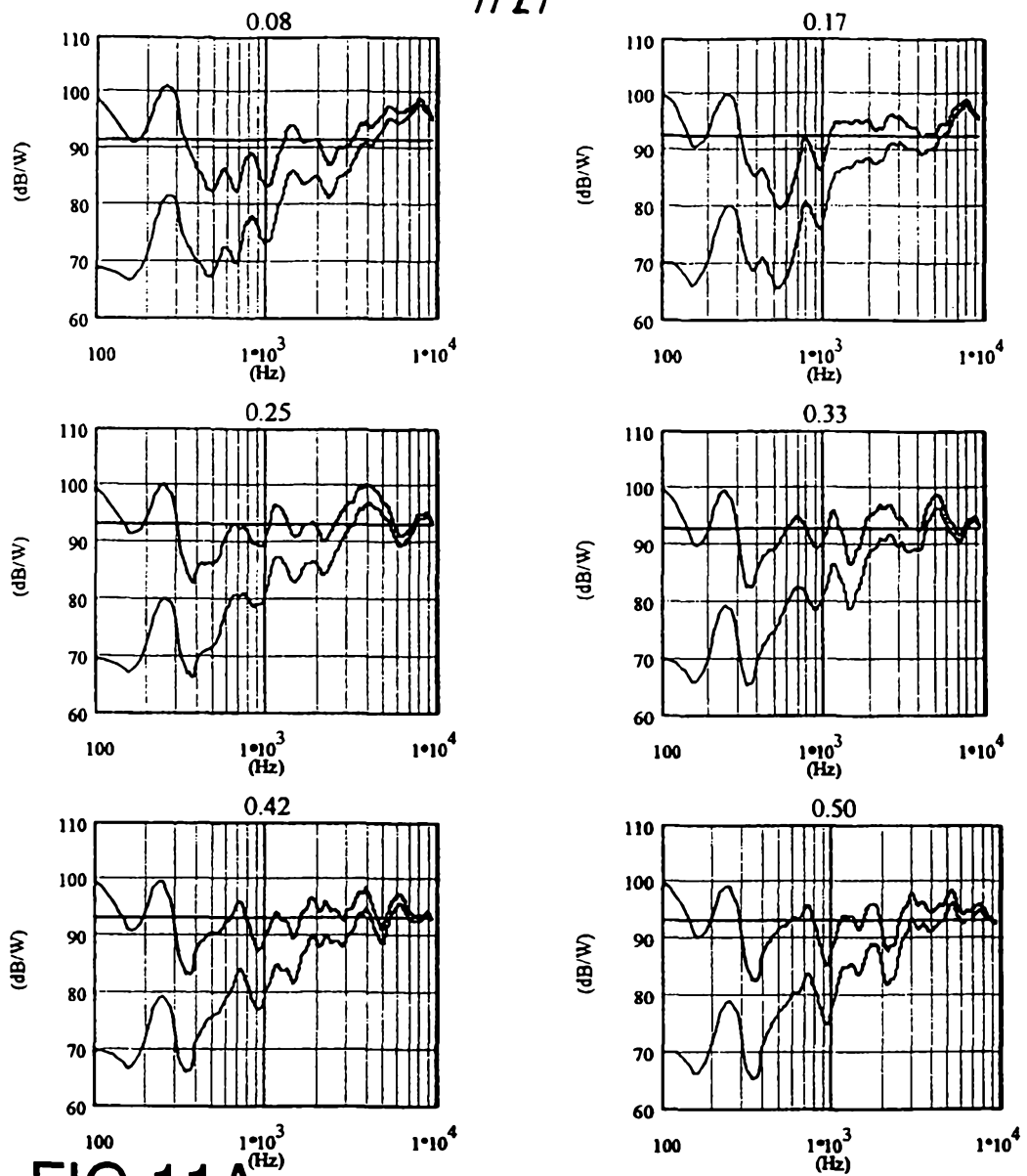
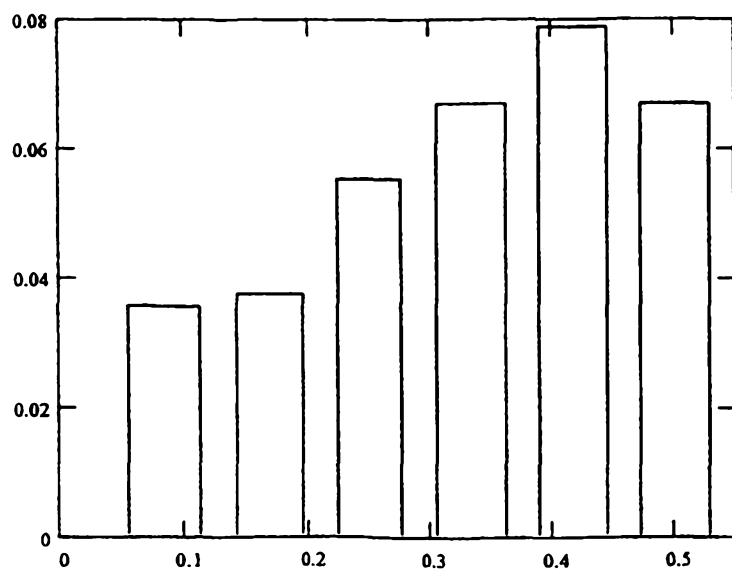


FIG.11A

FIG.12A



8/27

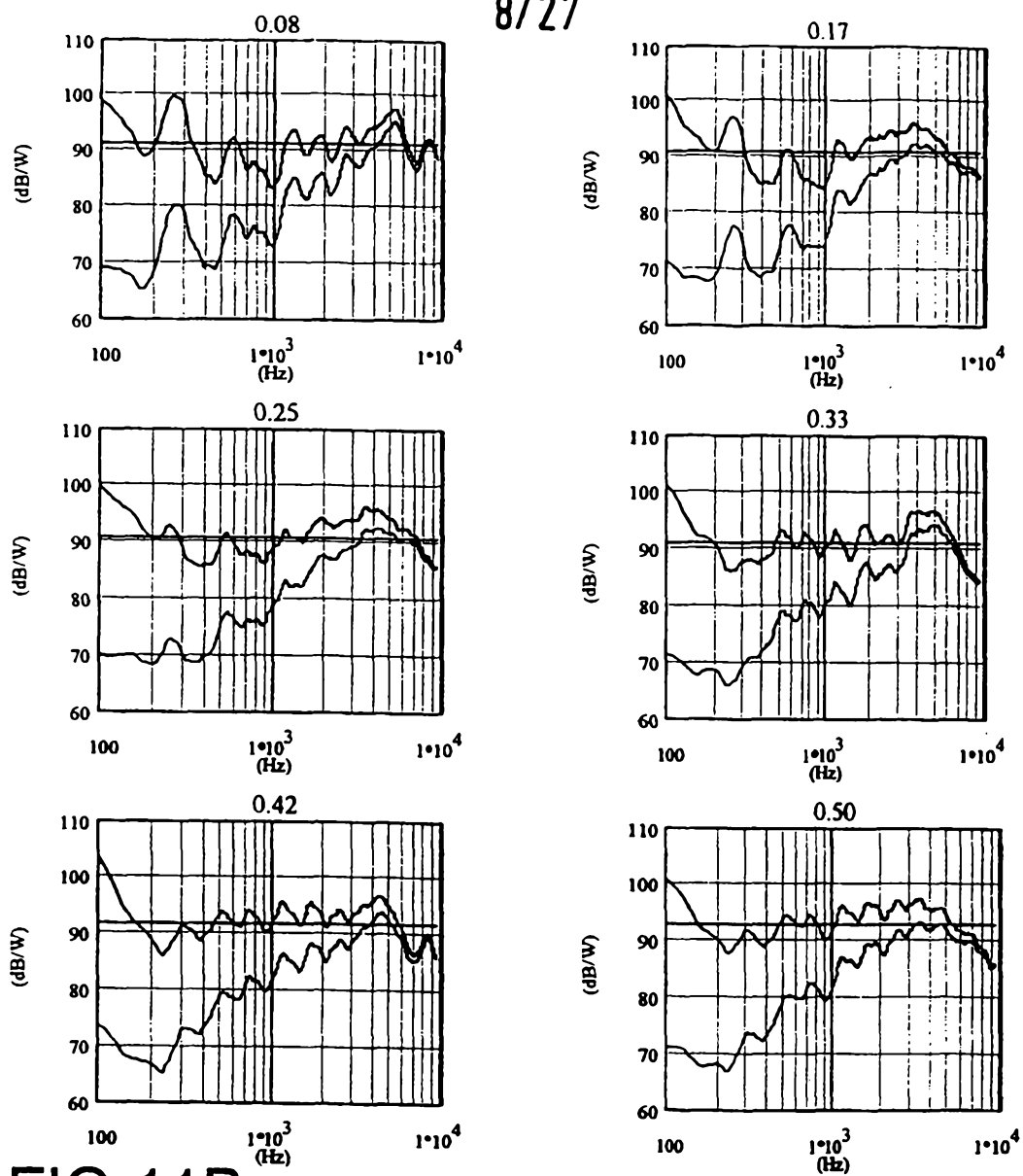
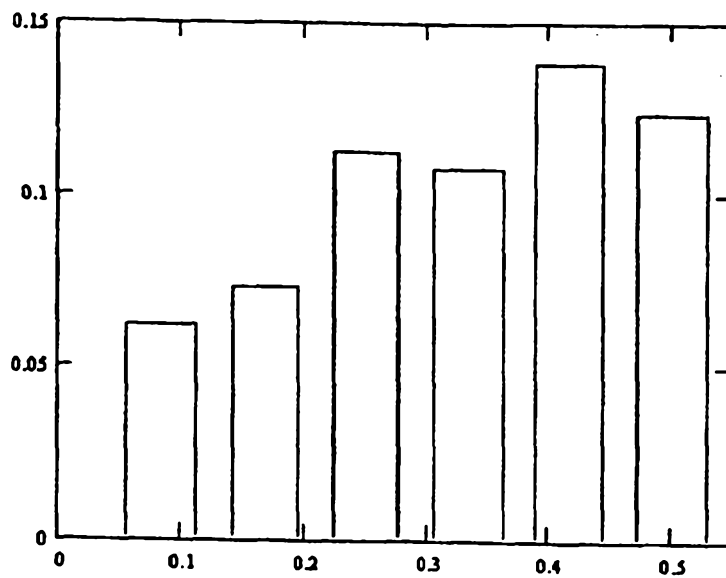


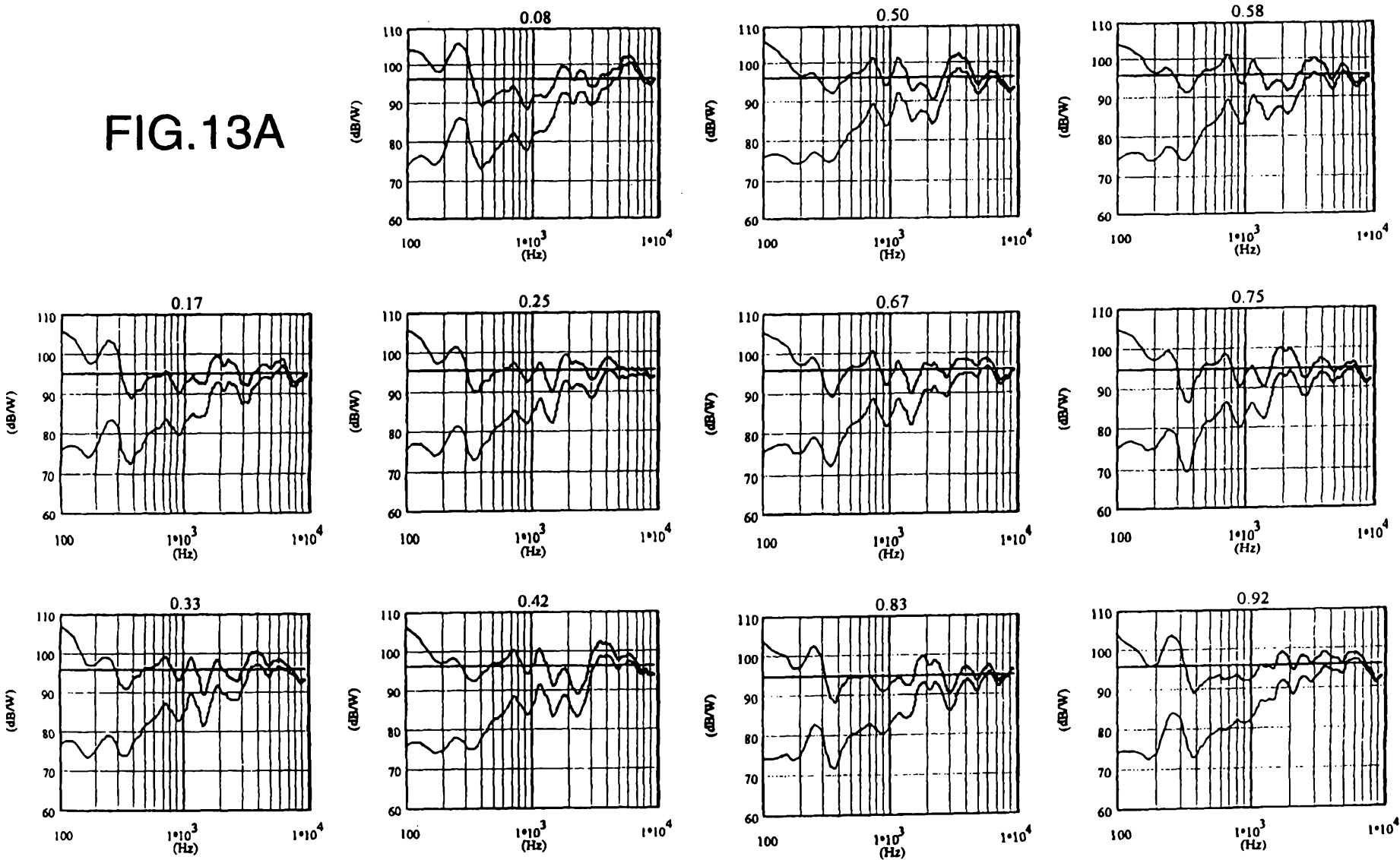
FIG.11B

FIG.12B



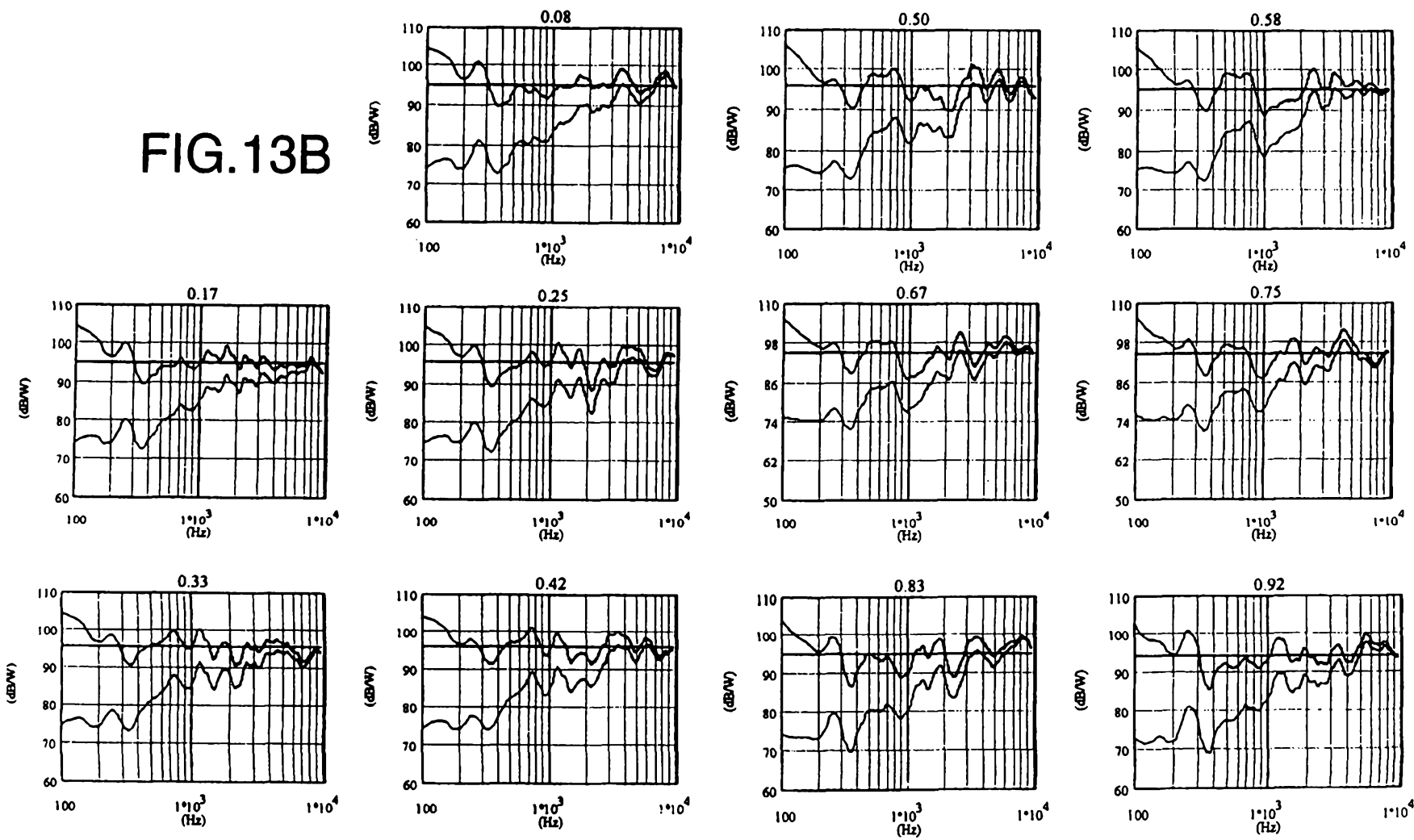
9 / 27

FIG.13A



10 / 27

FIG.13B



11/27

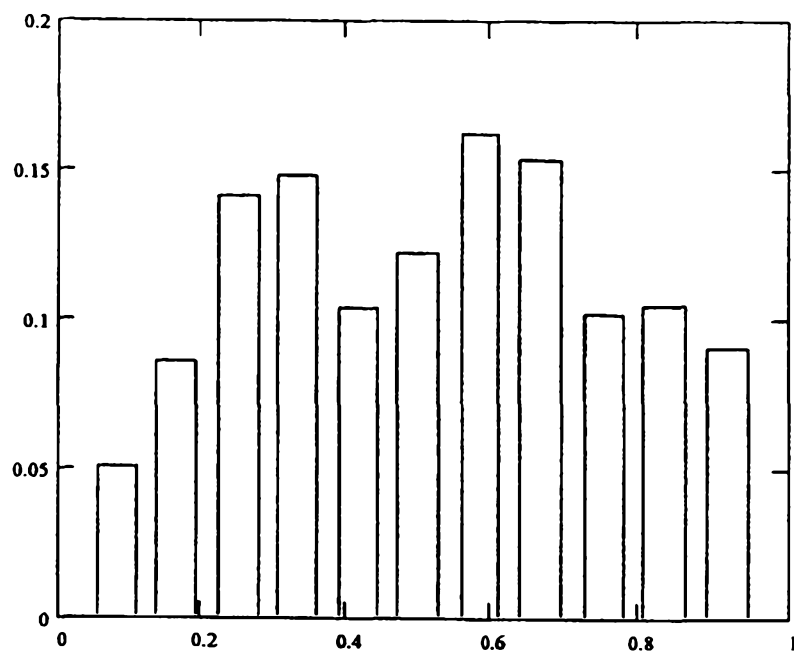


FIG. 14A

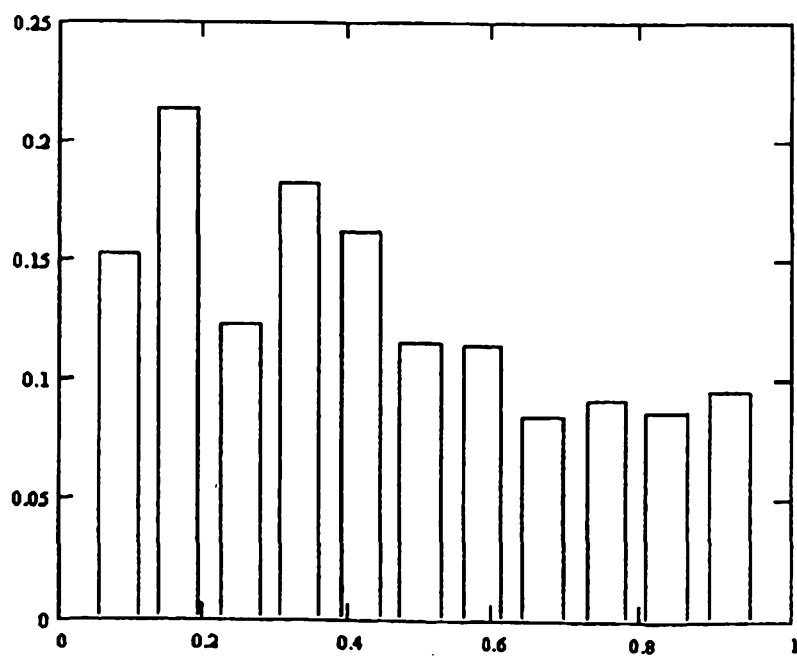


FIG. 14B

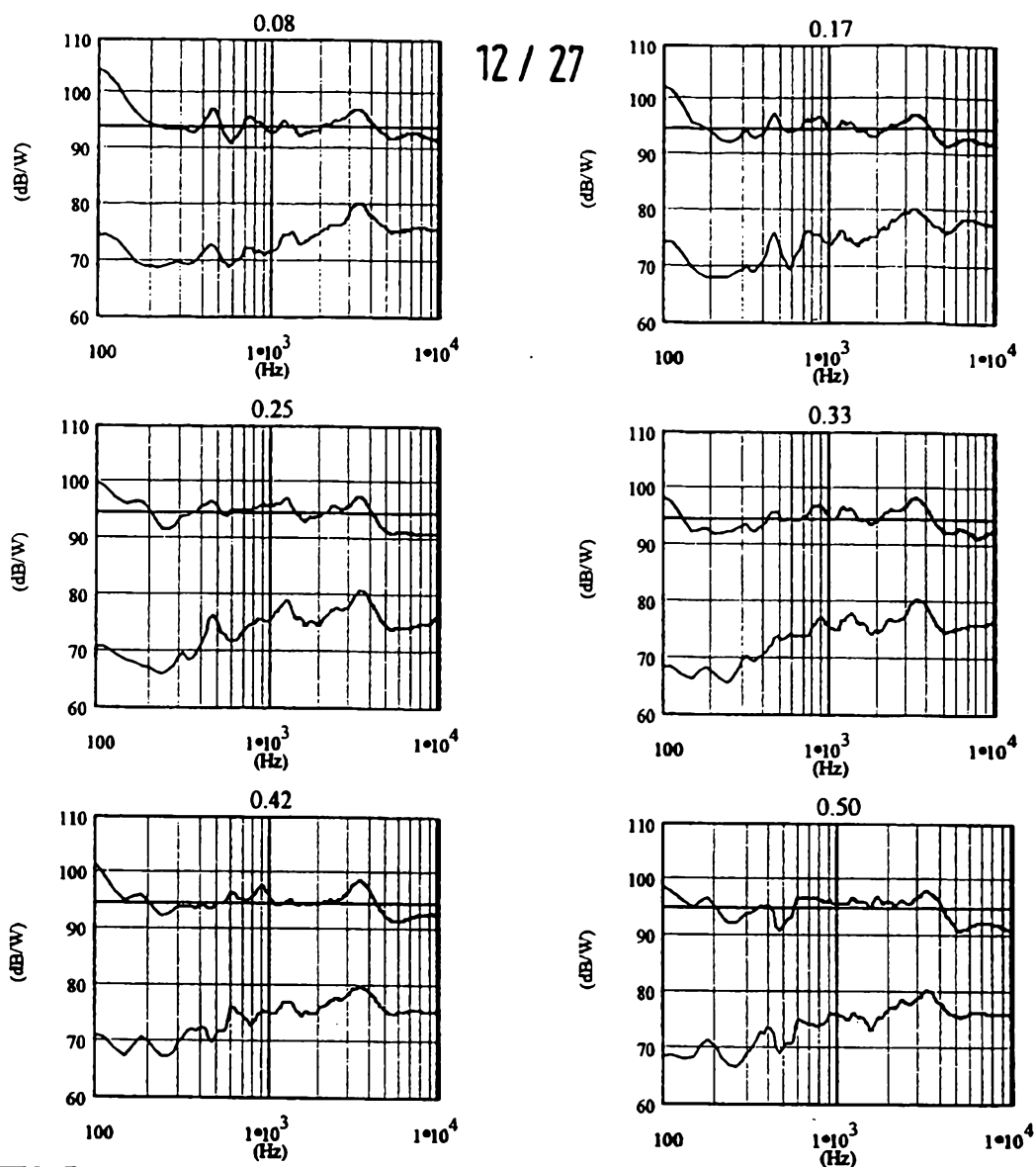


FIG.15A

FIG.15B

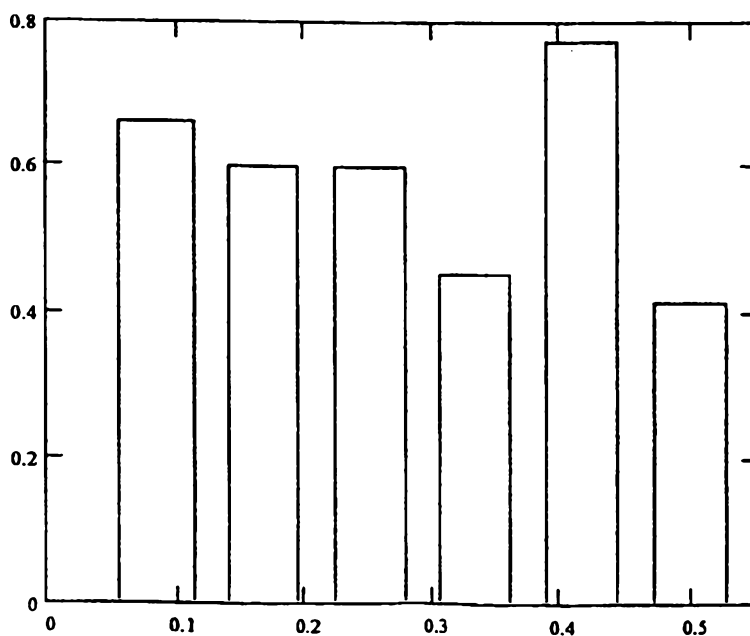
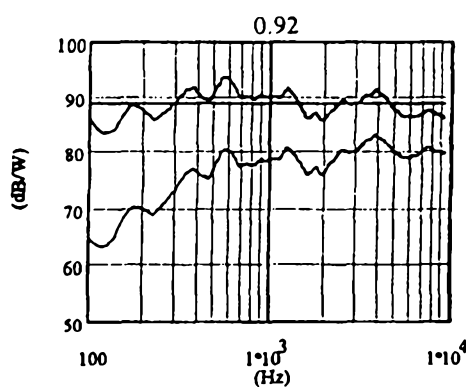


FIG.16A



14 / 27

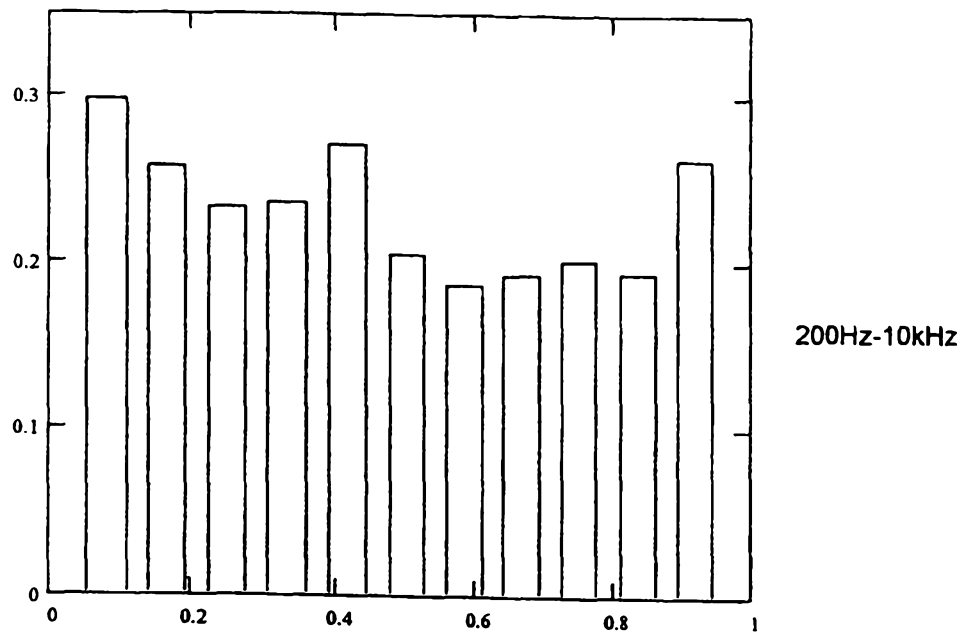


FIG.16B

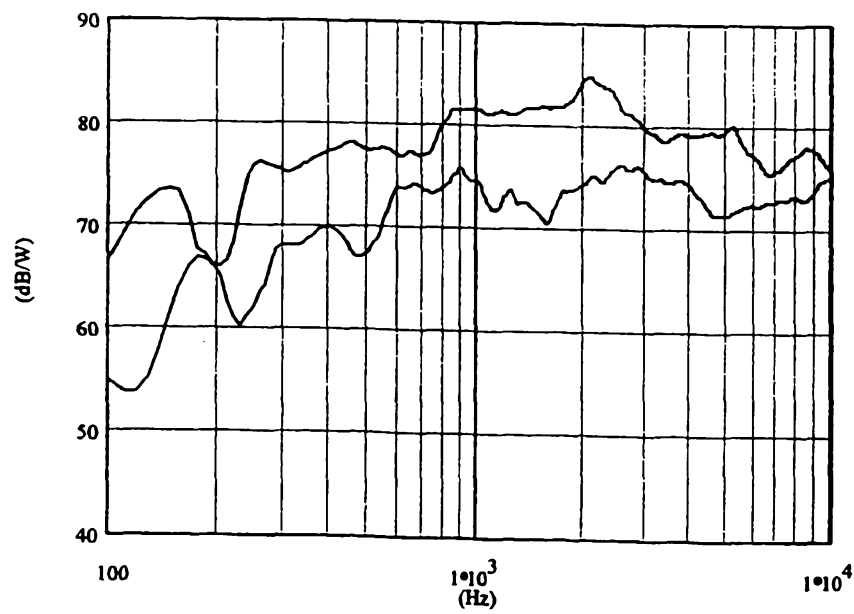


FIG.17

15 / 27

FIG.18A

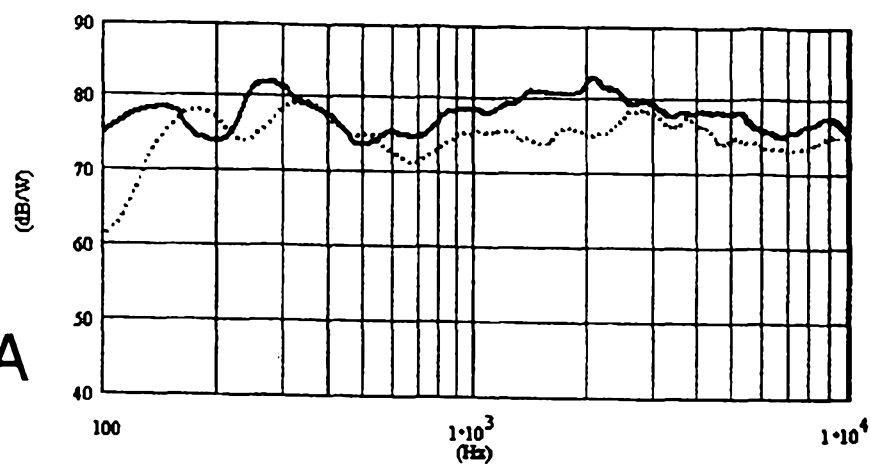


FIG.18B

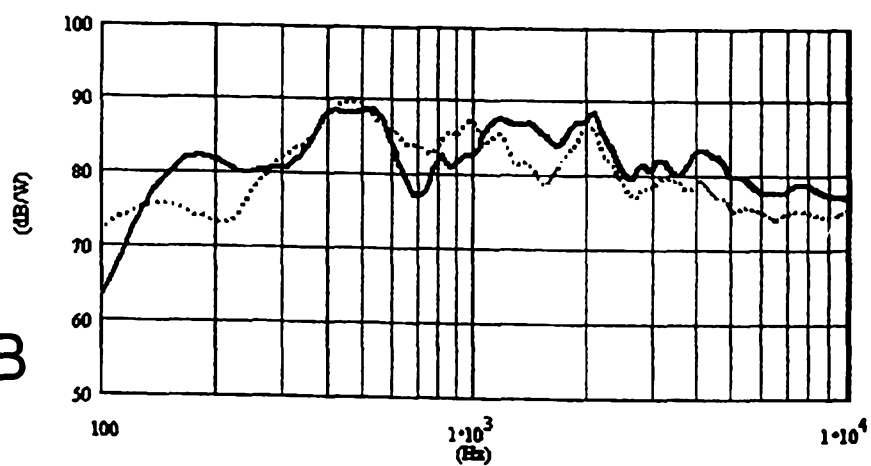
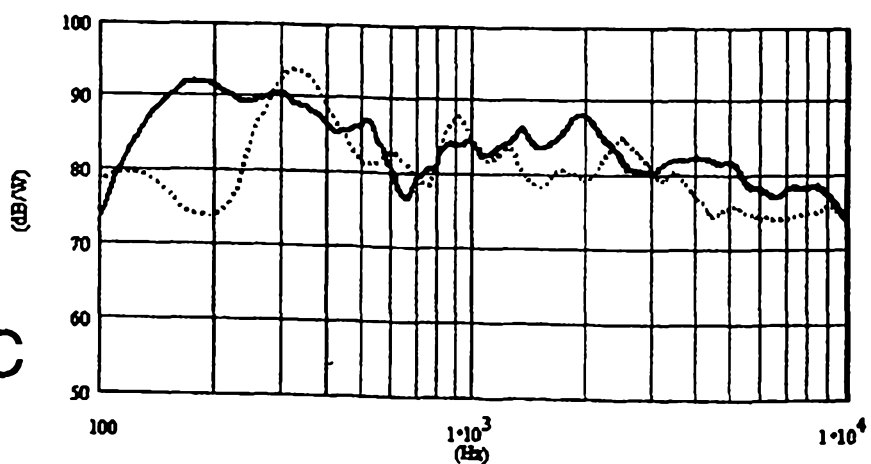


FIG.18C



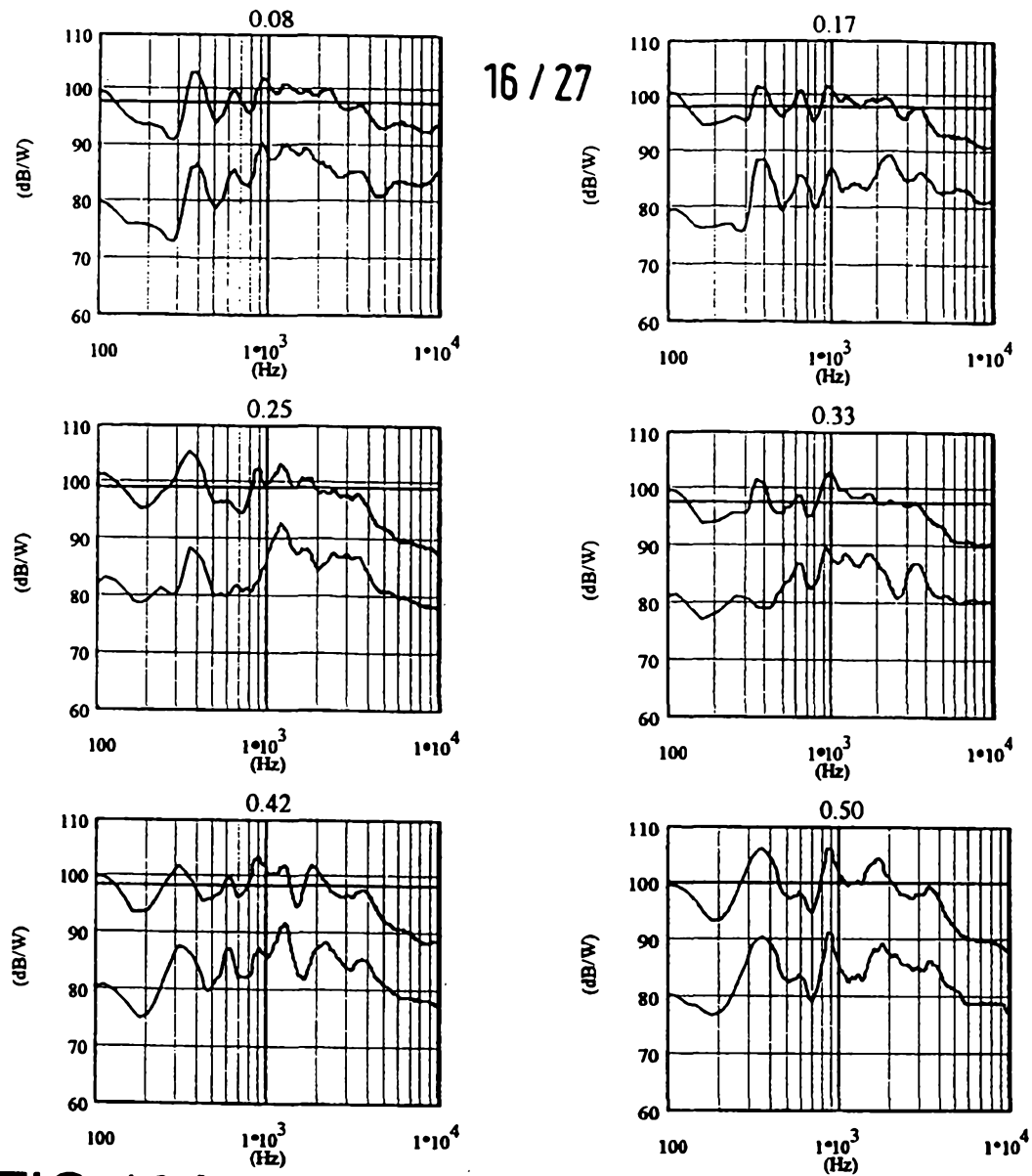
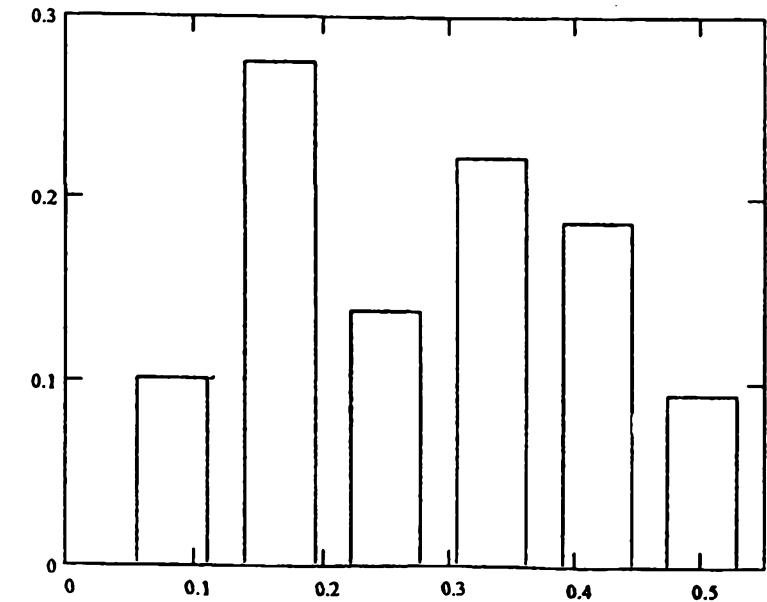


FIG.19A

FIG.19B

200Hz-4kHz



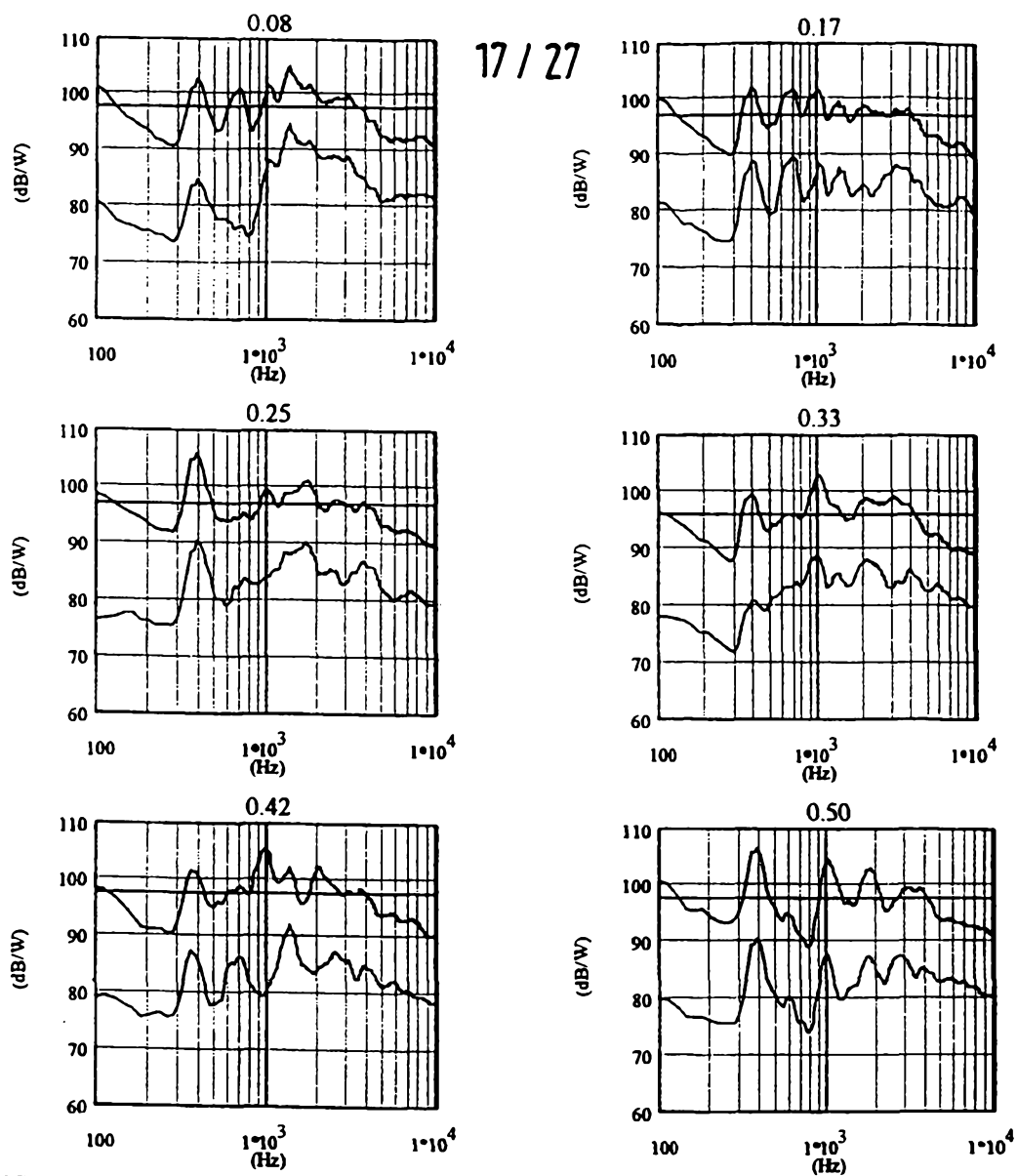
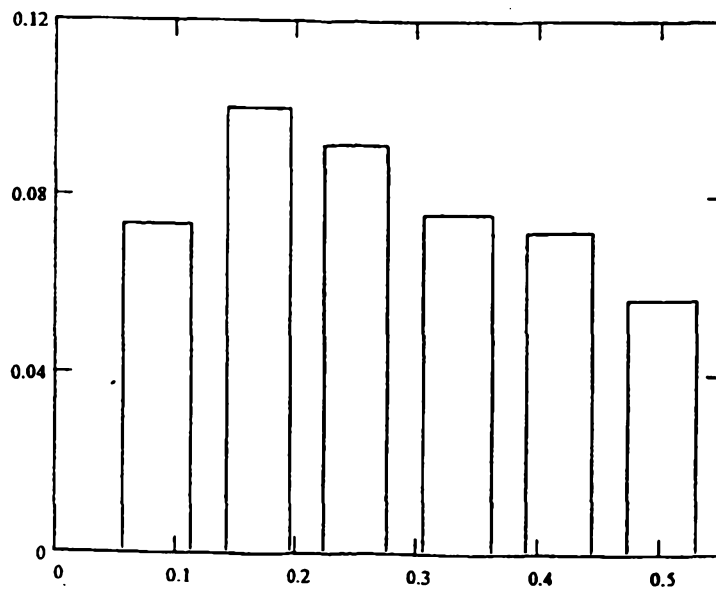


FIG. 20A

FIG. 20B

200Hz-4kHz



18/27

FIG.21A

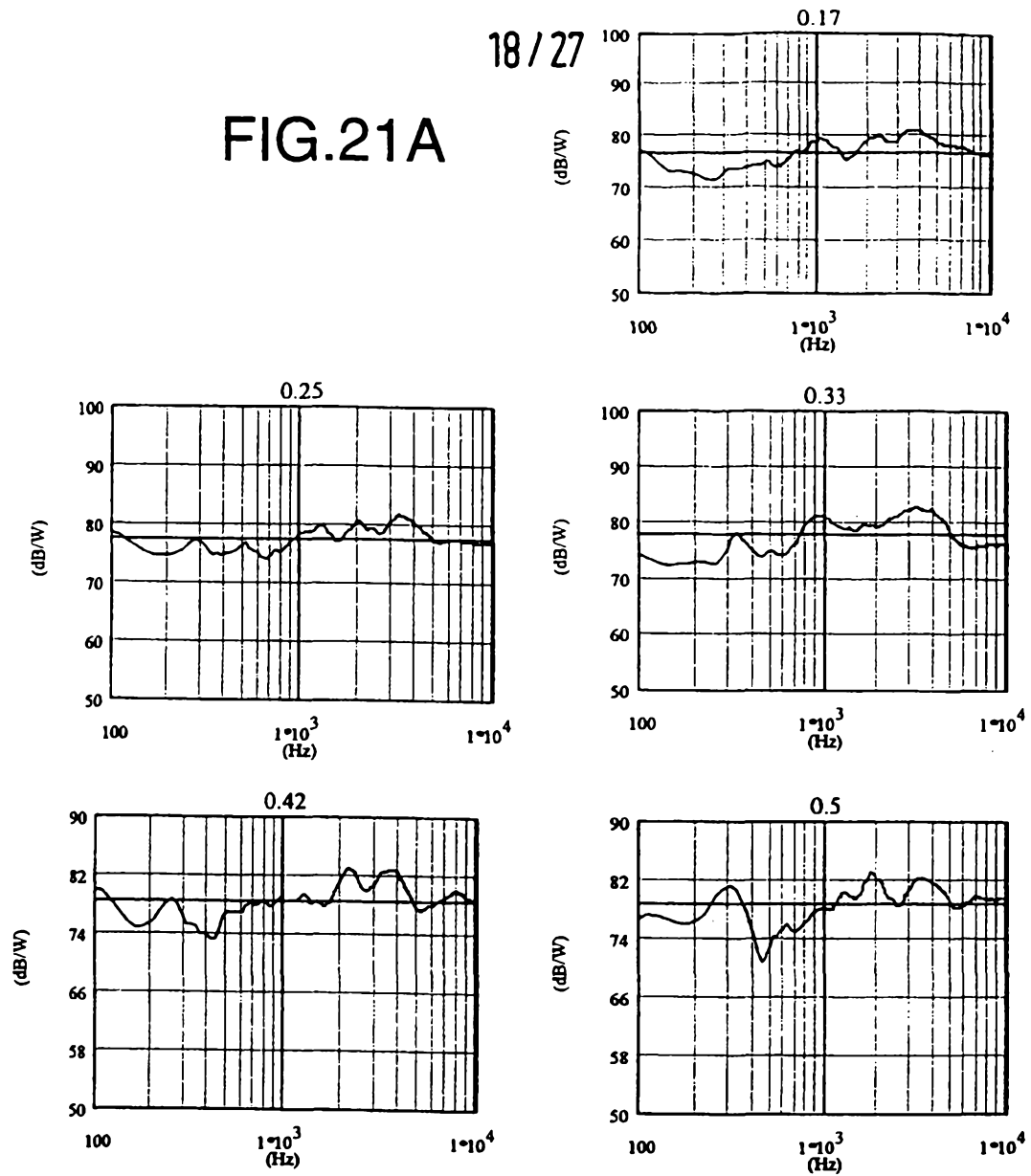
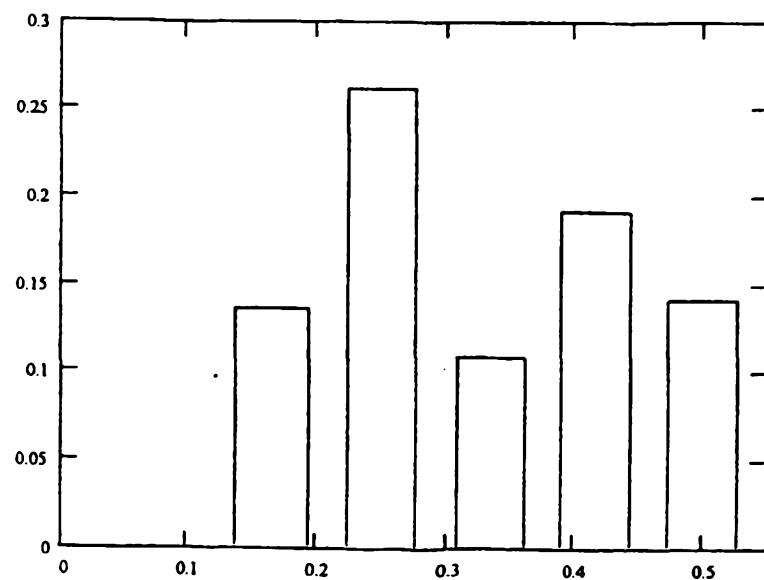
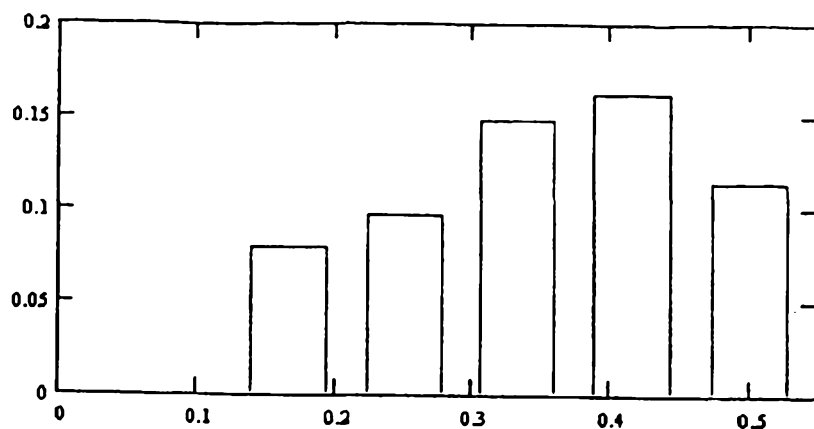
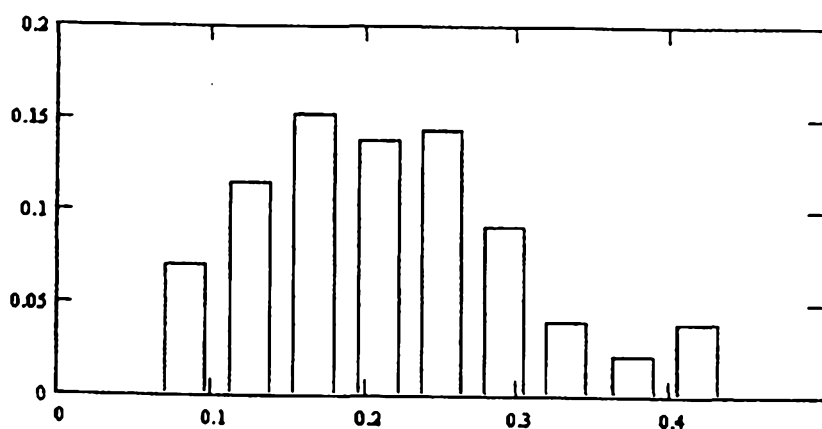
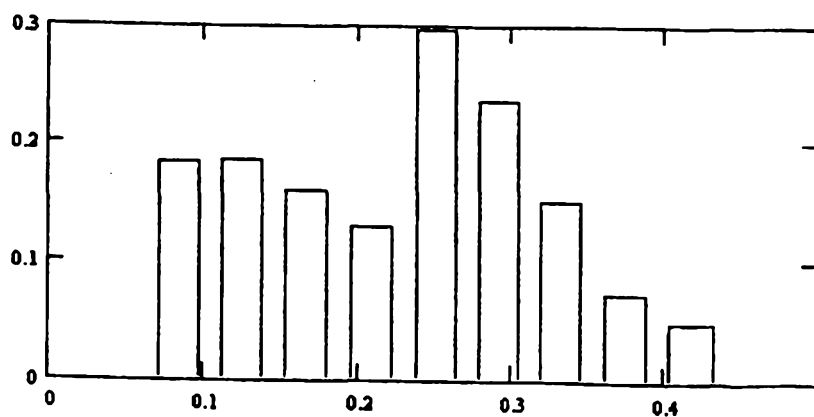


FIG.21B

200Hz-10kHz



19 / 27

**FIG.22**
300Hz-3kHz**FIG.23A**
300Hz-3kHz**FIG.23B**
300Hz-3kHz

20 / 27

FIG.24A

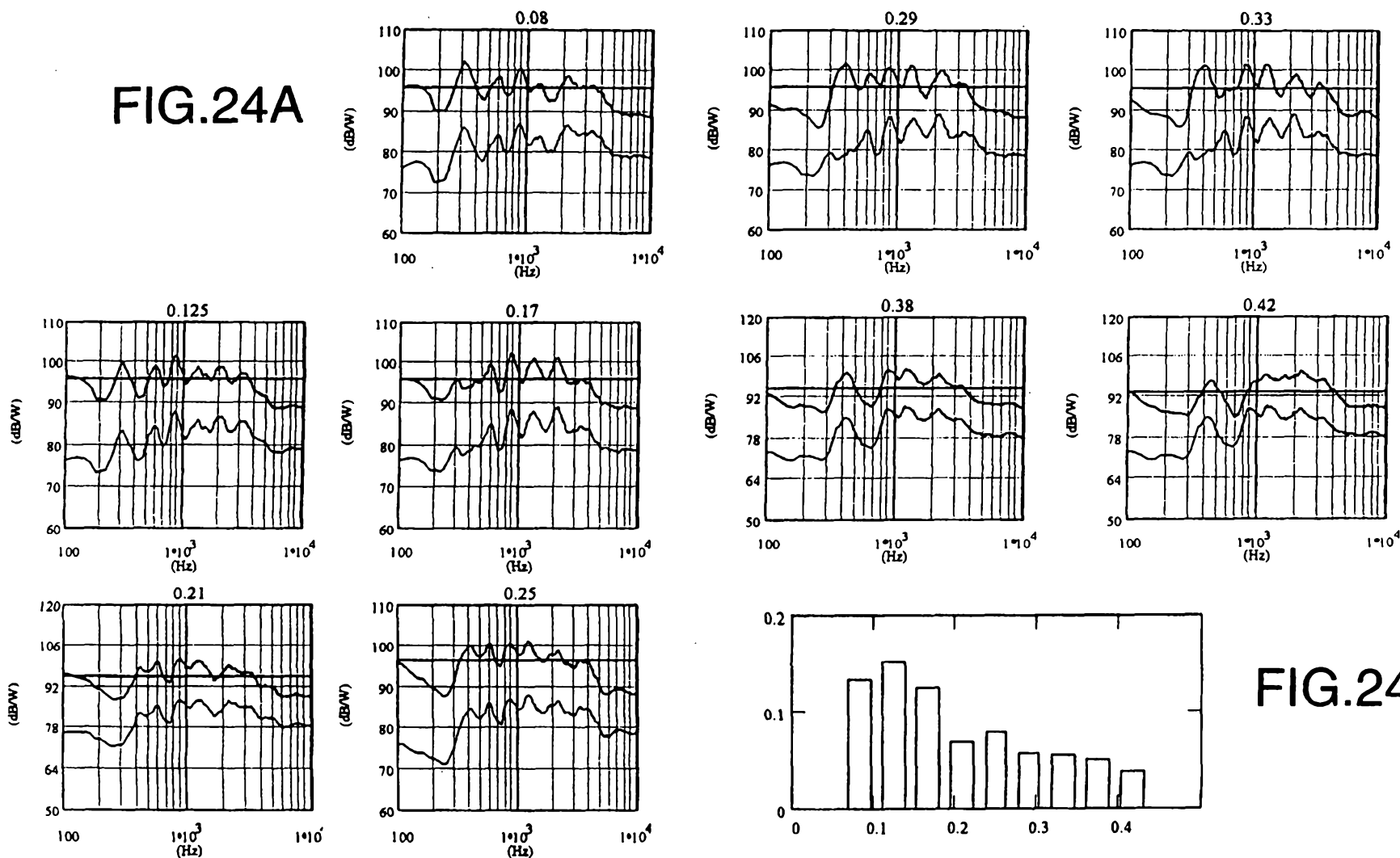


FIG.24B

21 / 27

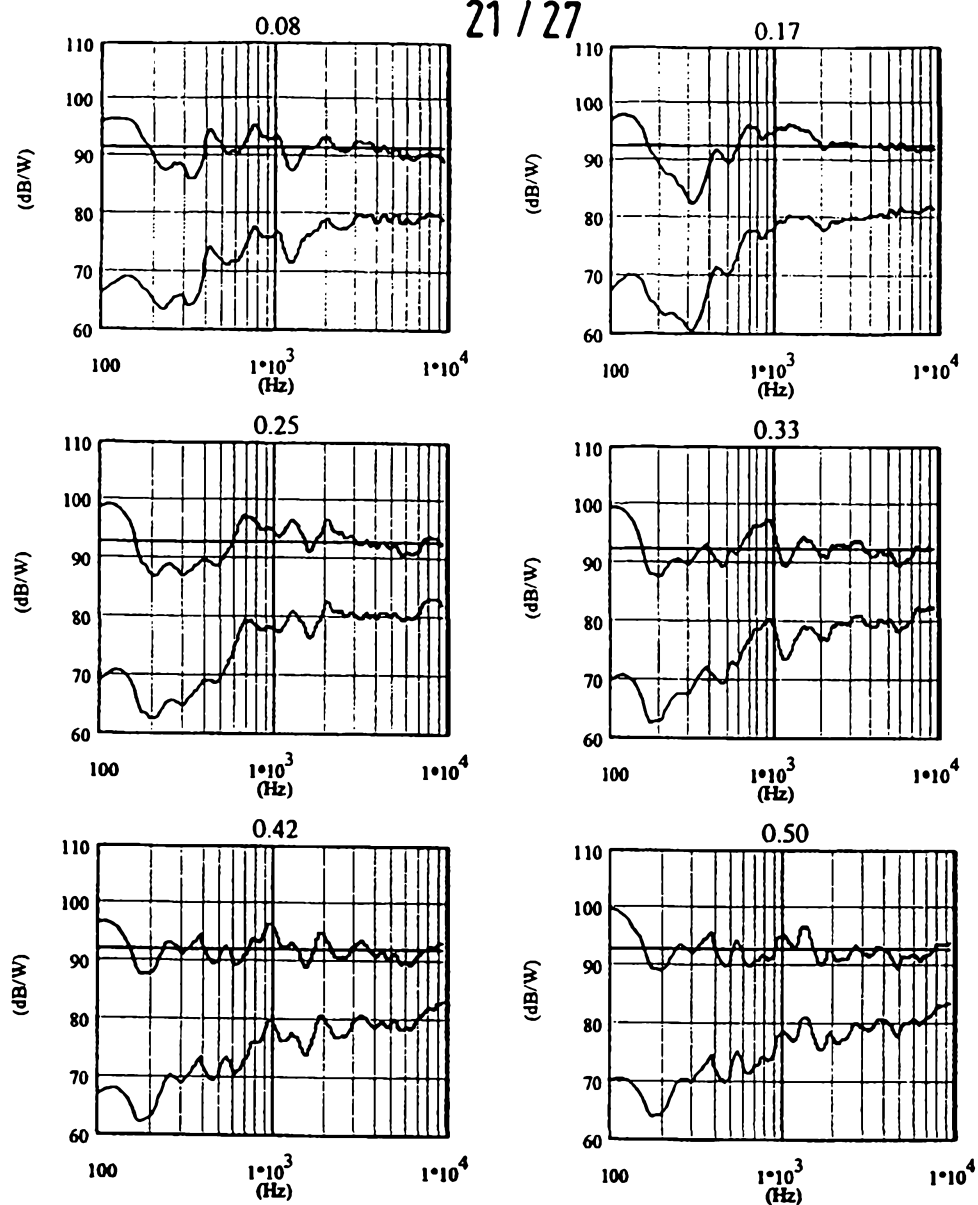
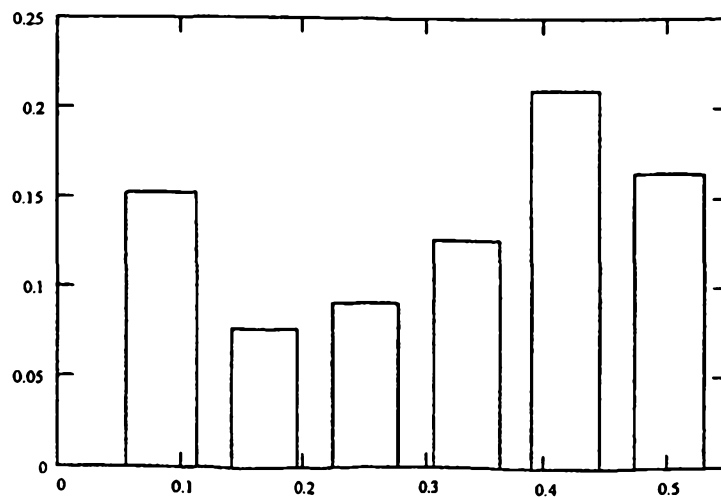


FIG.25A

FIG.25B

200Hz-10kHz



22/27

FIG.26A

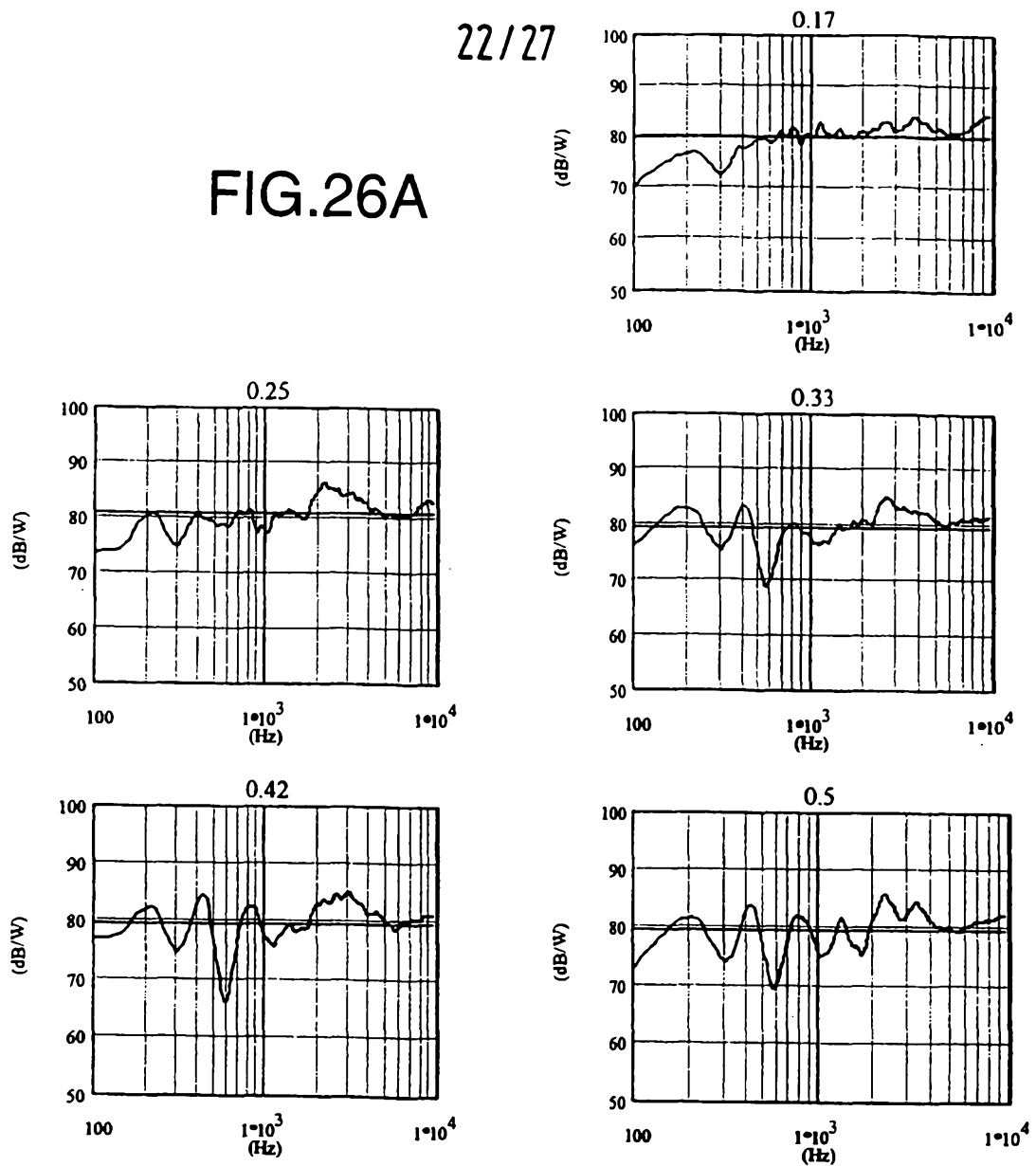
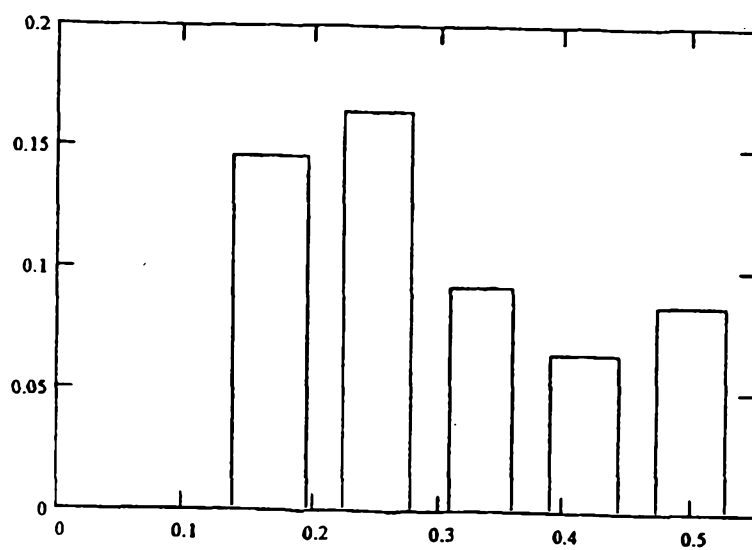


FIG.26B

200Hz-10kHz



23/27

FIG.27A

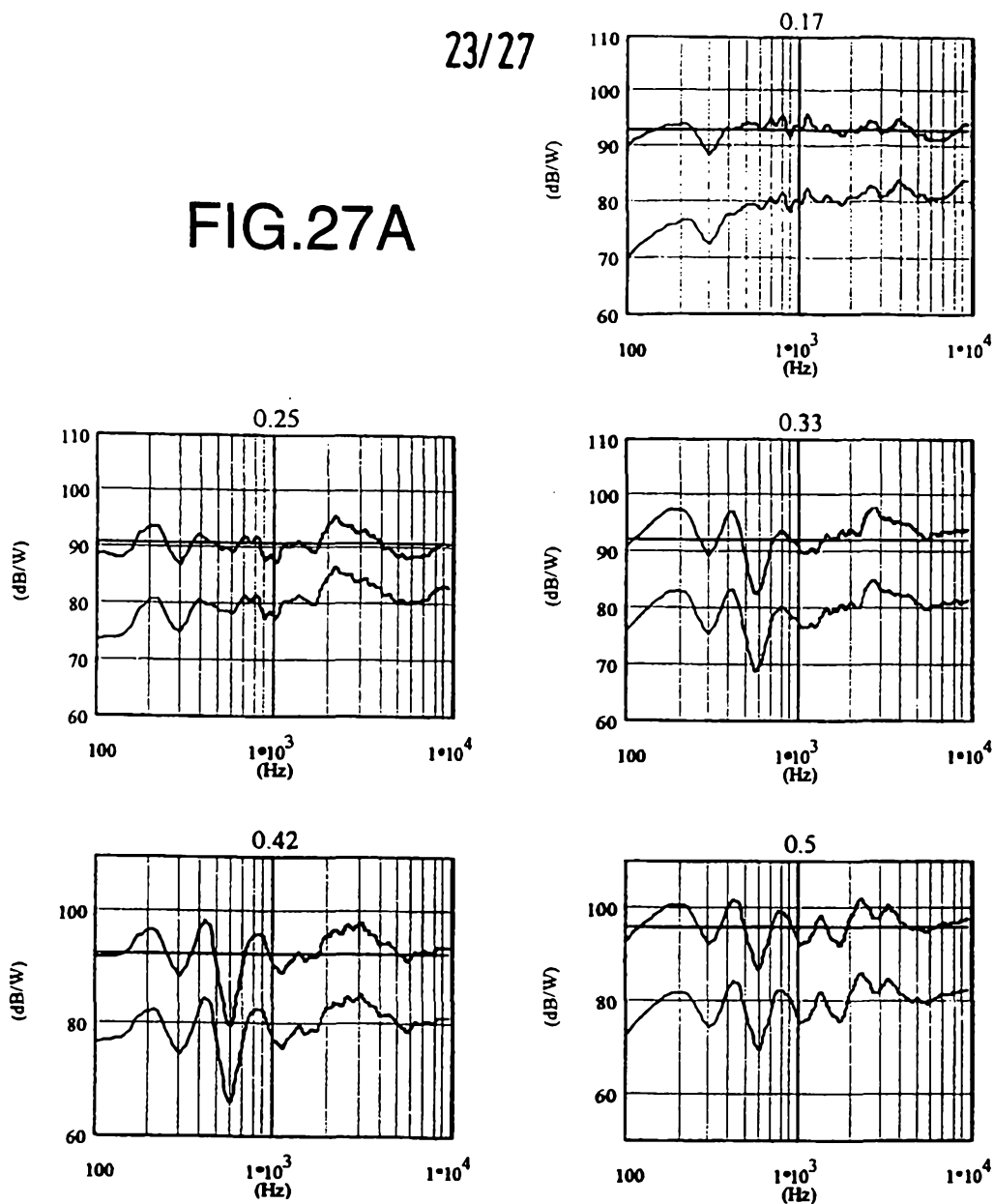
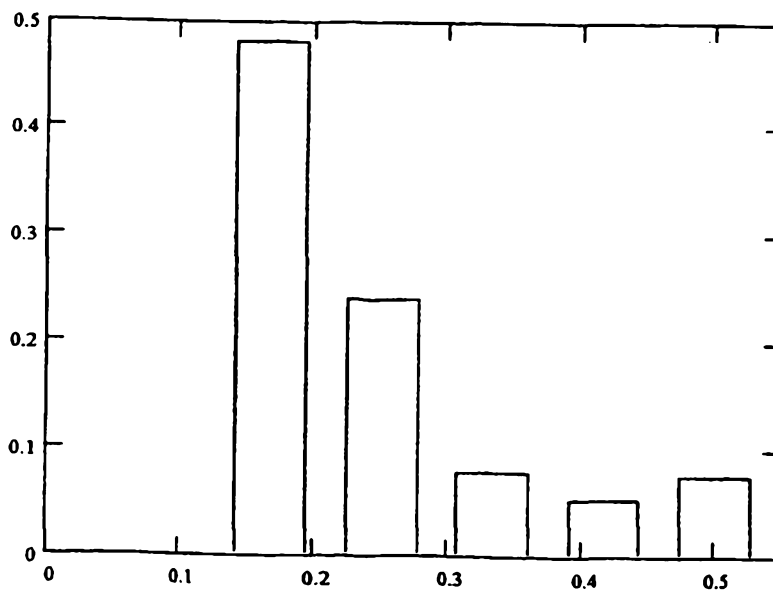


FIG.27B

200Hz-10kHz



24/27

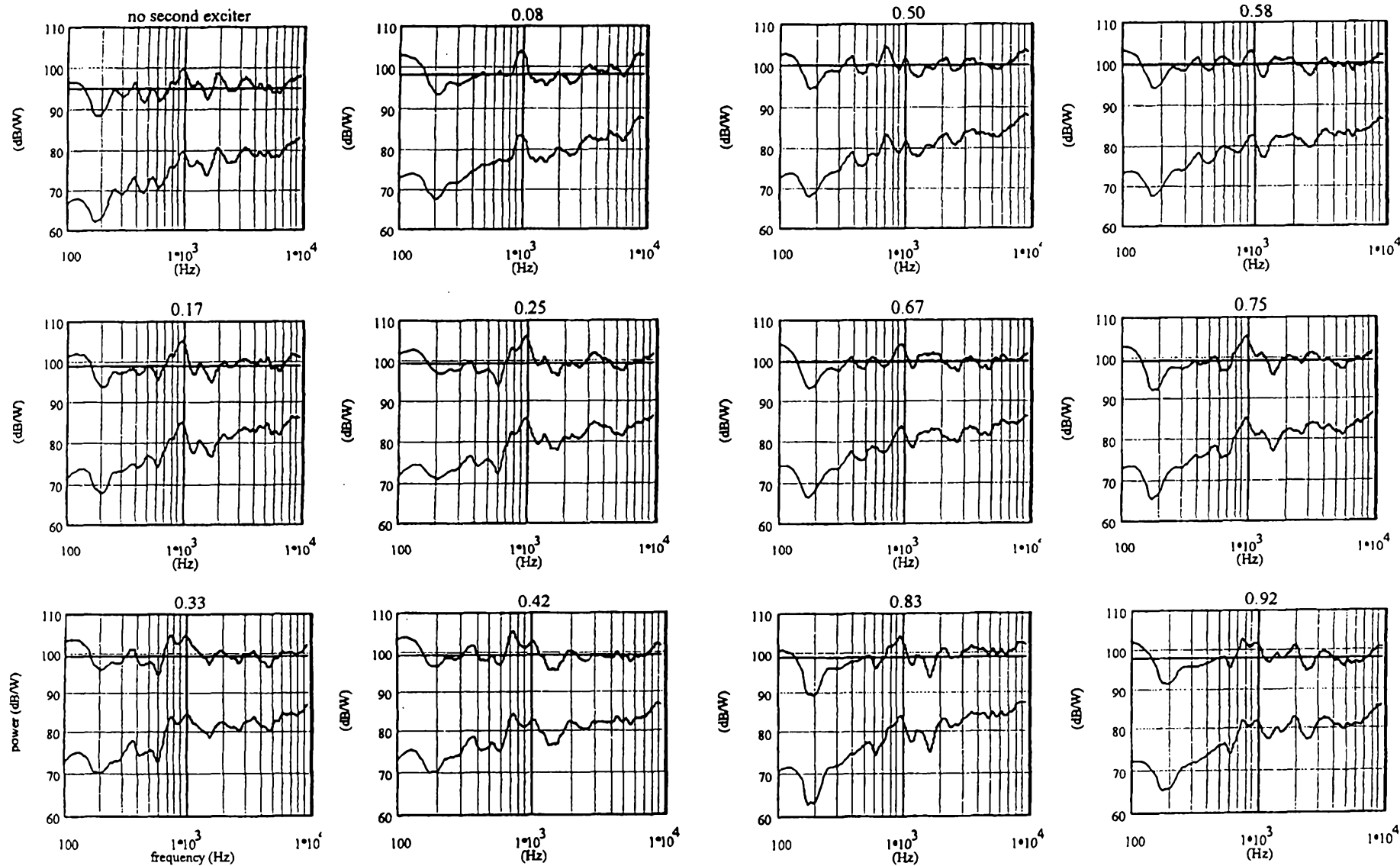
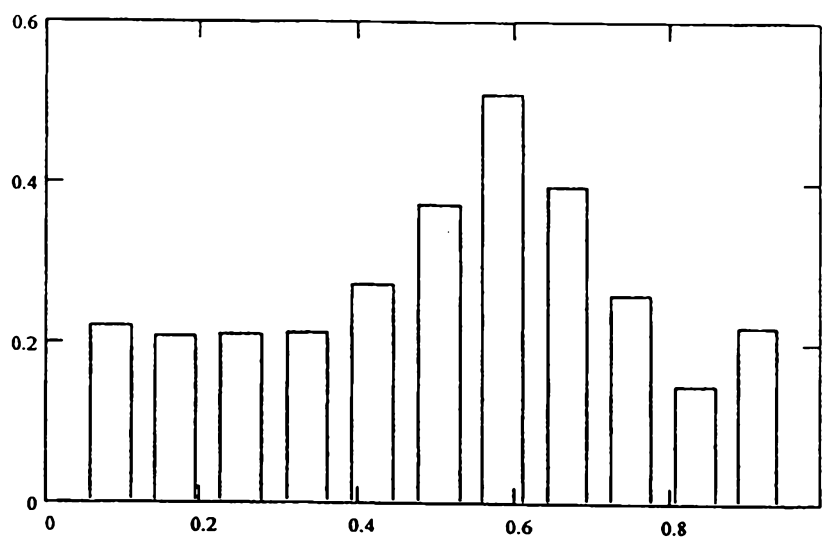
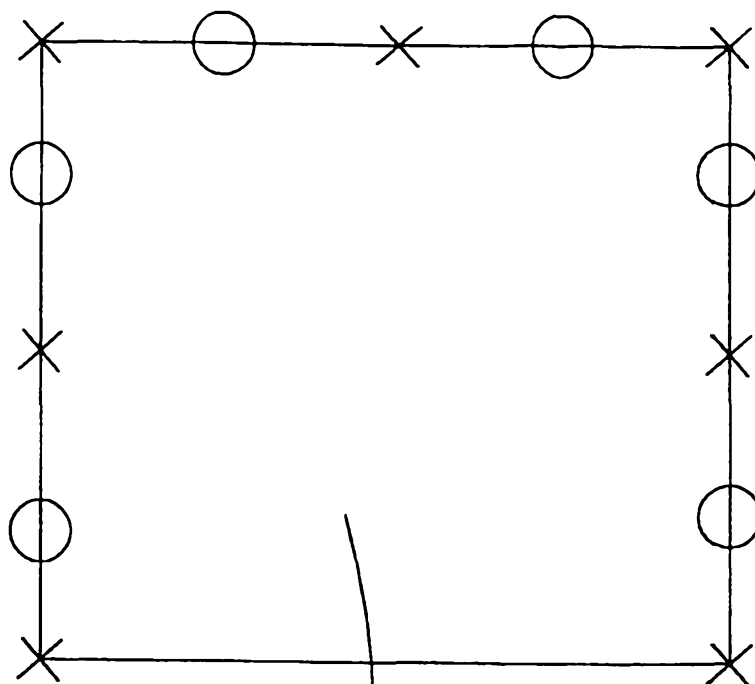


FIG.28A

25 / 27

**FIG.28B** 200Hz-10kHz

26 / 27



Panel.

FIG.29

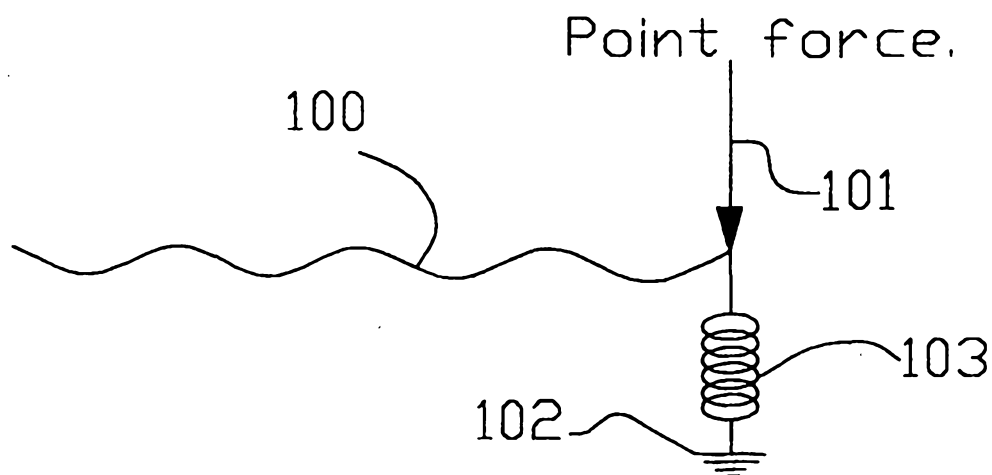
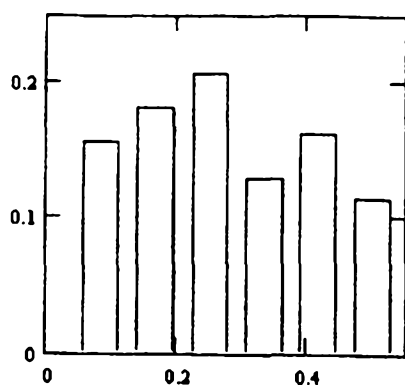
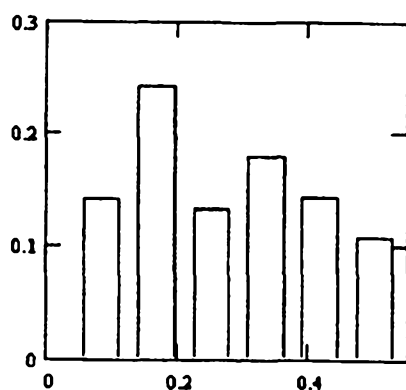


FIG.30

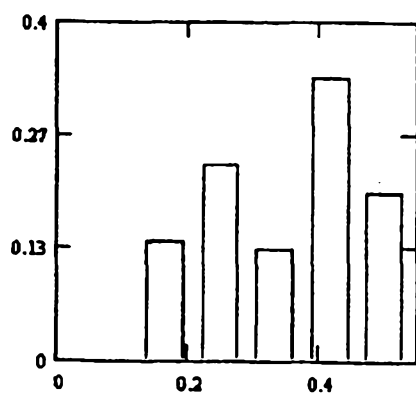
27 / 27

**FIG.31A**

300Hz-3kHz

**FIG.31C**

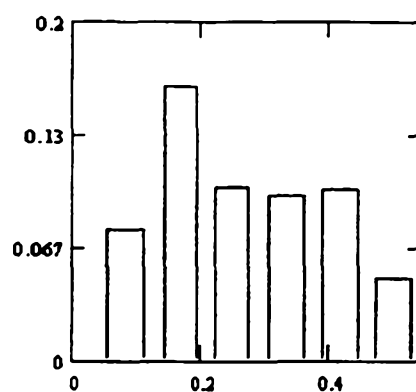
300Hz-3kHz

**FIG.31E**

300Hz-3kHz

FIG.31B

300Hz-3kHz

**FIG.31D**

300Hz-3kHz

