

No. 831,768.

PATENTED SEPT. 25, 1906.

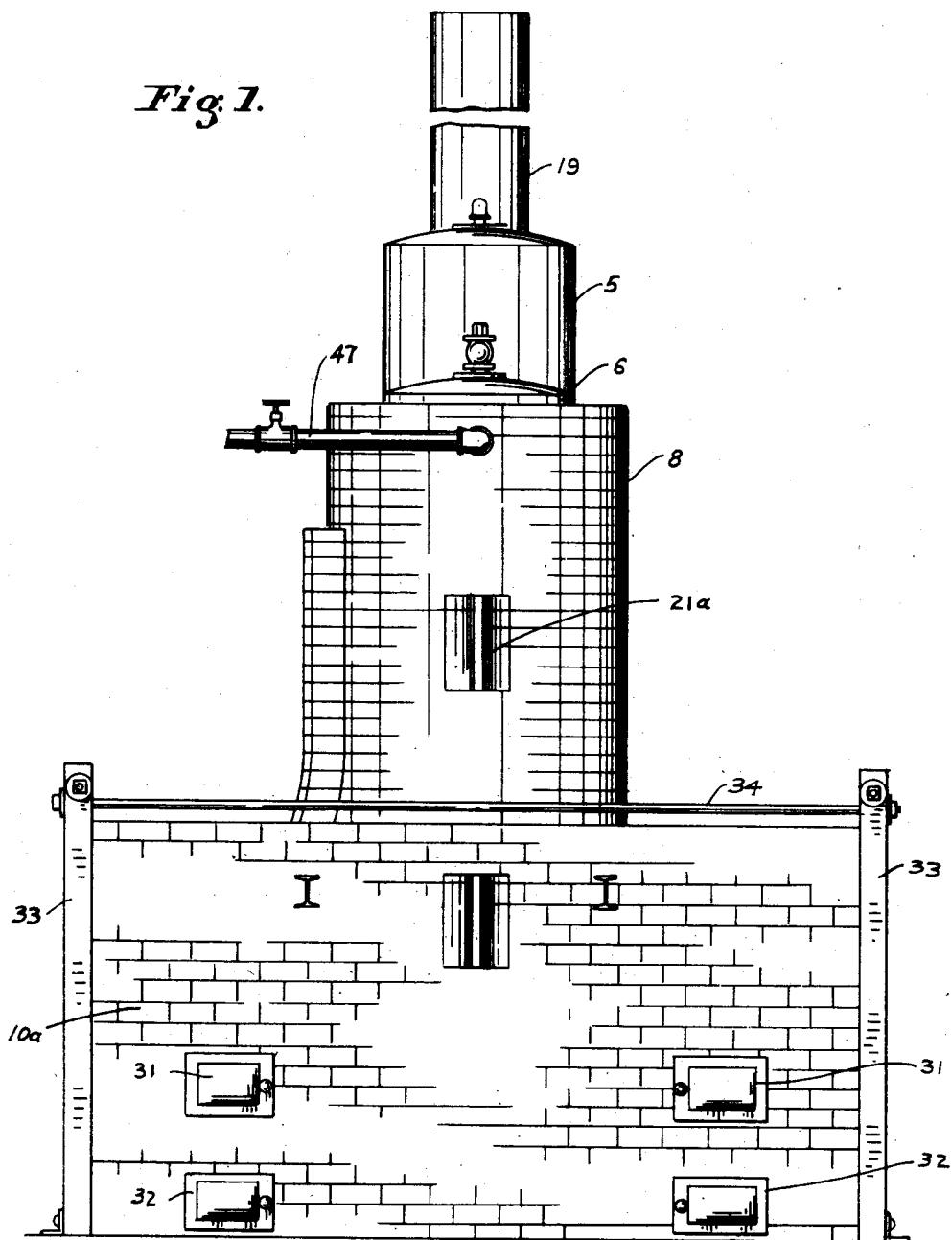
I. H. BOYER.

APPARATUS FOR THE GENERATION OF STEAM.

APPLICATION FILED DEO. 11, 1905.

4 SHEETS—SHEET 1.

Fig. 1.



WITNESSES:

Charles Norton
Clara A. Bridgeman

INVENTOR

Isaac H. Boyer,
BY Thomas L. Ryan

ATTORNEY

No. 831,768.

PATENTED SEPT. 25, 1906.

I. H. BOYER.
APPARATUS FOR THE GENERATION OF STEAM.

APPLICATION FILED DEC. 11, 1905.

4 SHEETS—SHEET 2.

Fig. 5.

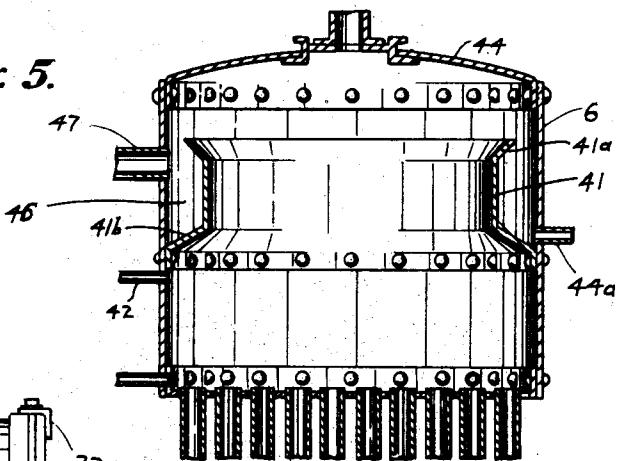
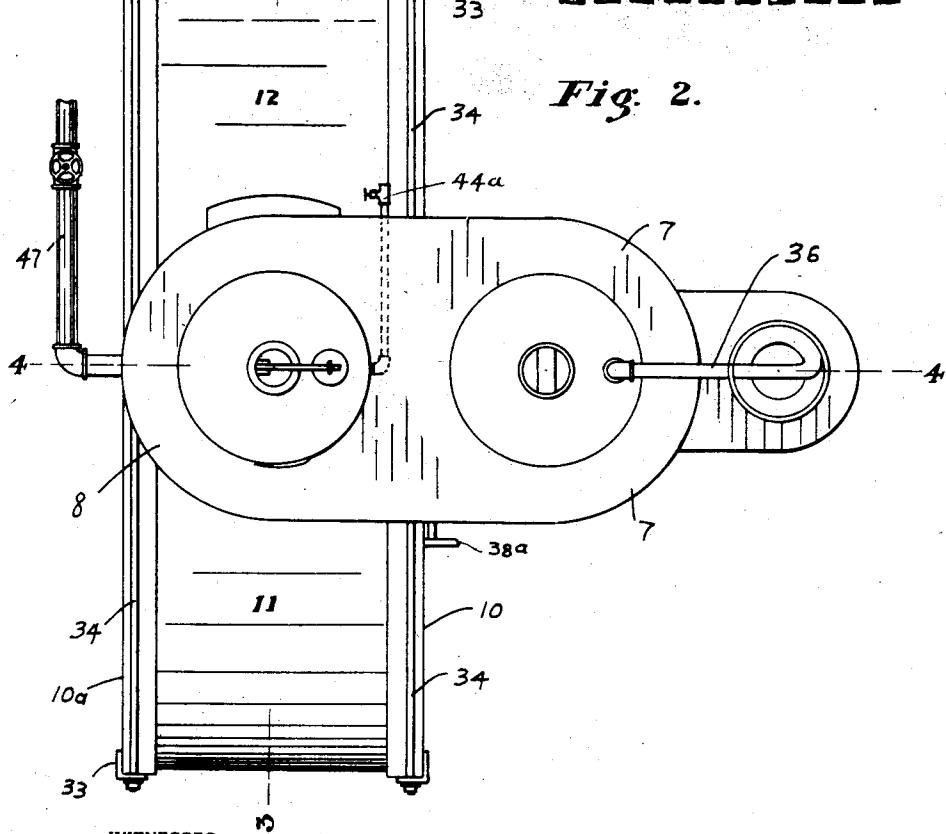


Fig. 2.



WITNESSES:

Ora L. Norton,
Clara A. Priddy.

INVENTOR

Isaac H. Boyer
BY *Thomas L. Ryan*

ATTORNEY

No. 831,768.

PATENTED SEPT. 25, 1906.

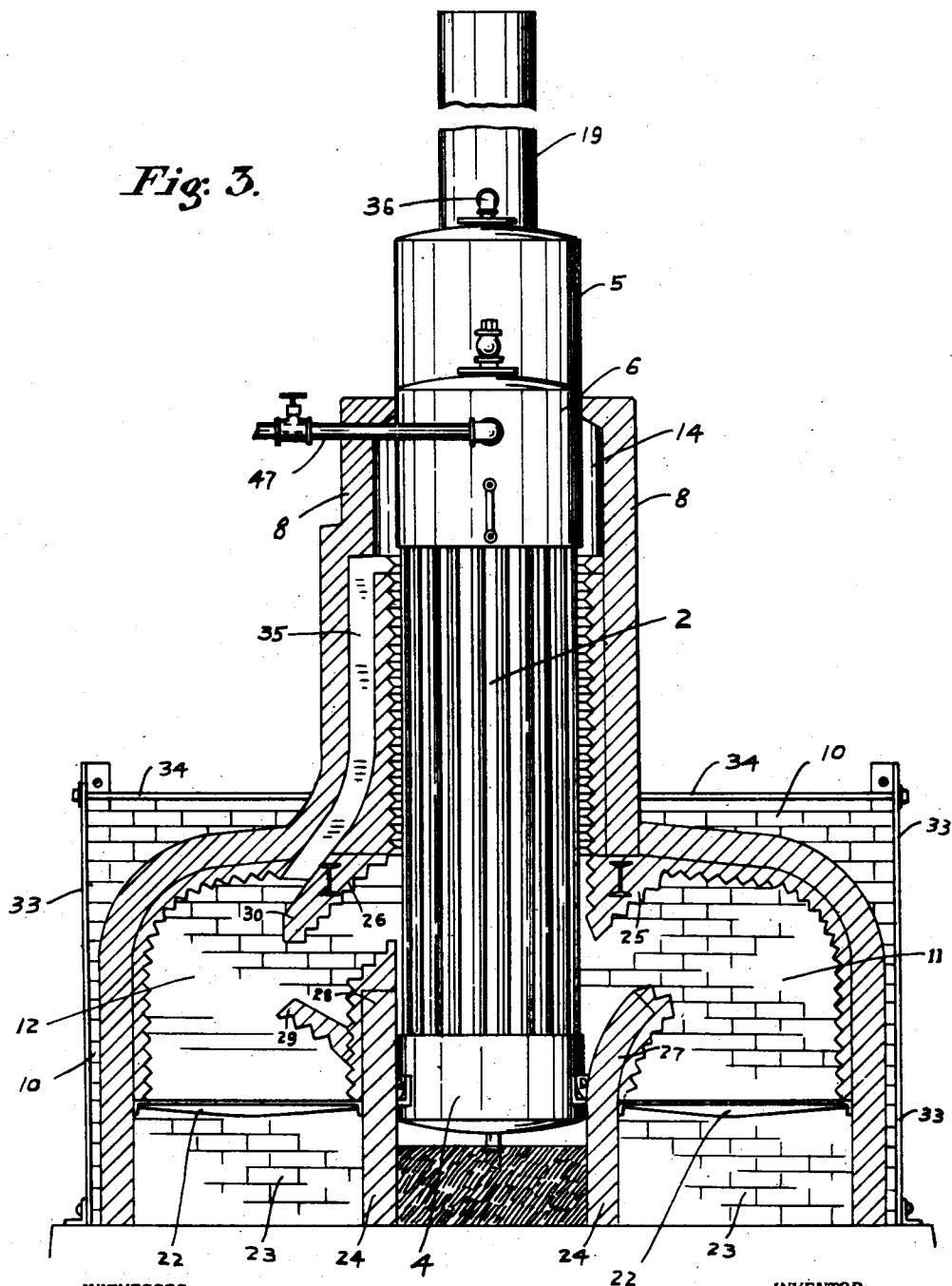
L. H. BOYER.

APPARATUS FOR THE GENERATION OF STEAM.

APPLICATION FILED DEO. 11, 1905.

4 SHEETS—SHEET 3.

Fig: 3.



WITNESSES:

Dr. L. Norton
Clara A. Bridgeman

INVENTOR

Isaac H. Boyer
BY Thomas L. Ryan

ATTORNEY

No. 831,768.

PATENTED SEPT. 25, 1906.

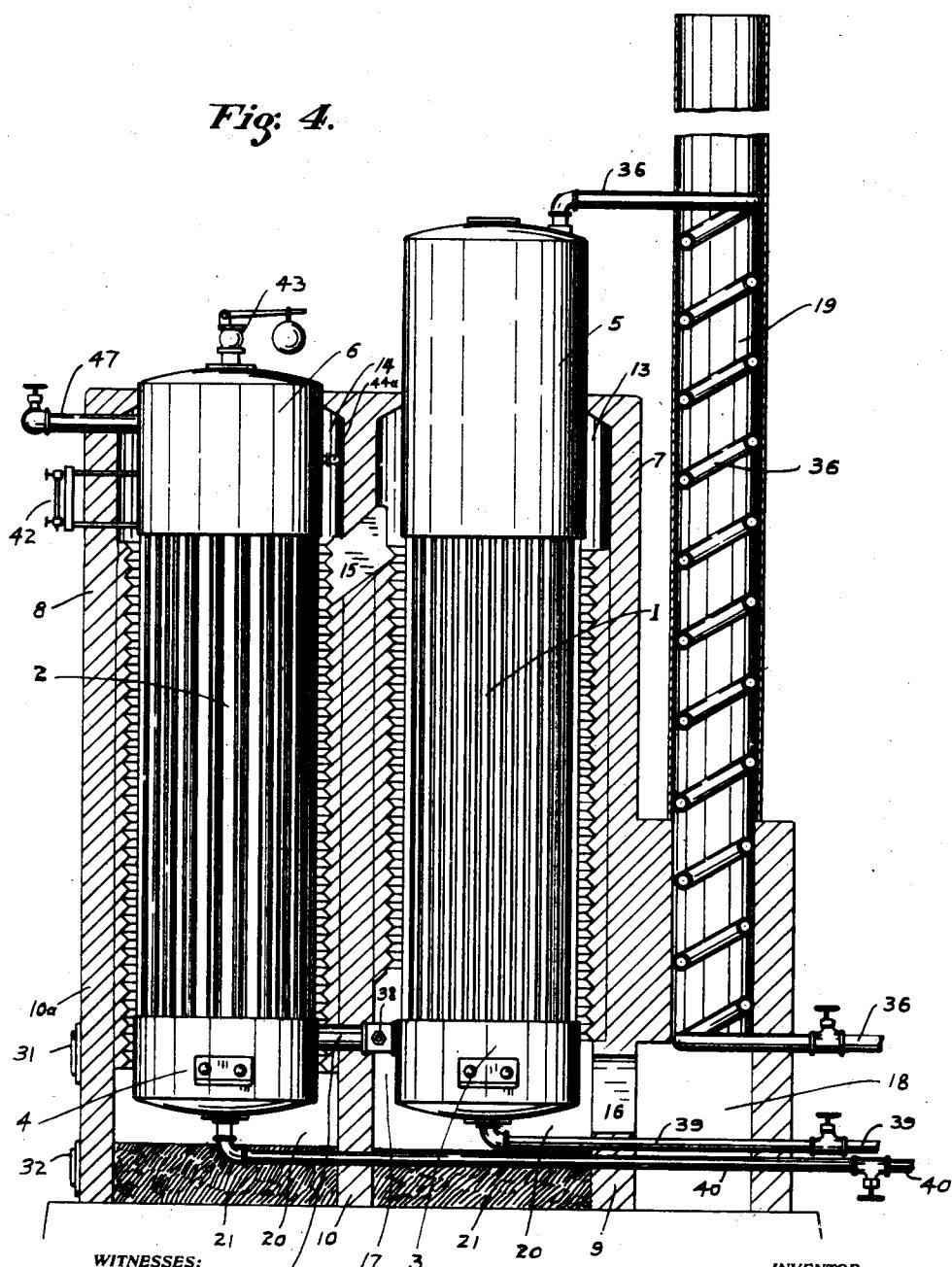
I. H. BOYER.

APPARATUS FOR THE GENERATION OF STEAM.

APPLICATION FILED DEC. 11, 1905.

4 SHEETS—SHEET 4.

Fig. 4.



WITNESSES:

Ora de Norton 37
Clara A. Biddy

INVENTOR
Isaac H. Boyer,
BY Thomas L. Ryan

ATTORNEY

UNITED STATES PATENT OFFICE.

ISAAC H. BOYER, OF MUNCIE, INDIANA.

APPARATUS FOR THE GENERATION OF STEAM.

No. 831,768.

Specification of Letters Patent.

Patented Sept. 25, 1906.

Application filed December 11, 1905. Serial No. 291,324.

To all whom it may concern:

Be it known that I, ISAAC H. BOYER, a citizen of the United States, residing at Muncie, in the county of Delaware and State of Indiana, have invented a new and useful Apparatus for the Generation of Steam, of which the following is a specification.

My invention has reference to improvements in apparatus for the generation of steam wherein are used the commonly known water-tube boilers.

In the adaptation of the water-tube boiler for the generation and superheating and supplying of steam for manufacturing purposes as now practiced the faults in the construction and arrangement of the boilers and superheating devices and the furnaces from which the heat is obtained are that the interior of the furnace is of such form and arrangement and the boilers are so formed and placed with reference thereto and the mode of the circulation of the water is such that the fire and heat in its initial and maximum degree of temperature as generated in the furnace for the purpose of producing the steam is directed against the boiler or boilers at the portions wherein is injected and retained the water having its initial degree of temperature, thereby causing unnecessarily a substantial portion of this initial heat to be absorbed. By reason of the excessive extent of the wall area of furnaces and the presence of air-currents of unheated and fluctuating degrees of temperature and the lack of means for the circulation of the heat with especial reference to the circulation of the water to be heated, which faults prevail in steam-generating apparatus now in use, a further portion of the initial and efficient part of the heat generated is by absorption unnecessarily consumed and wasted, the result being the extreme difficulty in maintaining an equable pressure and supply of steam except by the most skilful firing and handling and the consumption of large quantities of fuel.

My invention has for its purpose to provide an apparatus for the generation of steam consisting of water-tube boilers and furnaces so arranged and constructed that facility is afforded whereby with a predetermined course of circulation of the water in and through the boilers the degree of temperature of the water as it commences its circulation may be increased by the action of the waning or spent portion of the heat and gases generated in the firing portion of the furnaces, and the

further heating of the water is accomplished by the more intense degree of the heat, and the final heating to the degree of steam-generation is accomplished by that portion of the fire having the greatest intensity and the highest degree of efficiency, by which heat the water at this stage is with facility maintained in its high degree of temperature, and the steam so generated therefrom is simultaneously and by the same portion of the heat from the furnace superheated and maintained in its normal superheated condition. These and other objects are accomplished by the construction described in the following specification and illustrated in the accompanying drawings.

Similar figures of reference refer to corresponding parts throughout the several views, in which—

Figure 1 represents a front view of my improved steam-generating apparatus. Fig. 2 is a top plan view of the same. Fig. 3 is a vertical transverse section on the line 3 3, Fig. 2, showing the boilers and the pipes connected thereto in elevation. Fig. 4 is a vertical longitudinal section on the line 4 4, Fig. 2, showing the boilers and the pipes connected thereto in elevation; and Fig. 5 is a detached enlarged vertical cross-section view of the steam-drum.

1 and 2 designate two ordinary steam-boilers of the well-known water-tube type, vertically disposed, each provided at its lower portion with the mud-drums 3 and 4, respectively, and at their upper portions with the water-drum 5 and the steam-drum 6. The tubes constituting the boiler 1 are greater in number and smaller in size than the tubes of the boiler 2, the purpose of which will be hereinafter explained. These boilers are suitably supported upon the substantial metallic lugs secured thereto and resting upon proper foundation. Surrounding these boilers are the retorts 7 and 8, which provide a continuous air-space about the sides and bottoms of the boilers and which are supported upon the foundation-walls 9 and 10, the retort 7 being cylindrical continuously to its base, as shown in Fig. 4, and the retort 8 being cylindrical only in its upper portion, the lower portion of the same being enlarged, as shown in Fig. 3, into the furnaces 11 and 12. The walls of these furnaces and retorts are built up of ordinary building-bricks or other suitable constructive material, the inner surfaces of the walls being lined or covered with a veneer of

fire-brick. I have devised and employed the peculiar form of fire-brick shown, by which is obtained an interior surface for these retort and furnace walls which will present a 5 succession of angularly-disposed surfaces and successive horizontally-extending sharp lines. The object of this feature is to increase the wall area traversed by the heat and gases of combustion, whereby the same may be readily 10 absorbed by and retained in and radiated or refracted thereby. Moreover, these projecting portions of the fire-brick interior will readily become heated intensely and will by the latent heat contained therein contribute 15 to the complete combustion and burning of the effluvia and gases emanating from the furnace-fires. A further purpose is that these successively-disposed ridges will present such obstacle to the heat-currents that the same 20 will be directed away from the walls and inwardly through and about the tubes. For the reason that the heat should be kept circulating uniformly as possible in and among the tubes continuously throughout their extent these retorts and the heating devices described are provided.

The upper part of the retorts 7 and 8 are closed, as shown in Fig. 4, and brought to a 30 termination continuously about the water-drum 5 and the steam-drum 6. In these upper portions of the retorts the fire-brick lining is dispensed with, whereby the continuous cylindrical chambers 13 and 14 are provided, adapted to retain normally a portion 35 of the heat of the retorts. Provided in the wall separating the boilers is the passage 15, and in the foundation-wall 9 is the passage 16, these passages affording communication between the retorts 8 and 7 and between the 40 chamber 17 of the retort 7 and the stack-chamber 18. This stack-chamber is inclosed by suitable wall extending upwardly sufficiently to provide a receptacle for soot and ashes and for a base upon which is supported 45 and retained the smoke-stack 19. Beneath the boilers are the air-spaces 20, the filling 21, of sand or soil, being provided, so as to minimize the amount of air-space about the drums of the boilers.

50 In the furnace and retort walls are provided at suitable locations proper apertures 21^a, as shown in Fig. 1, affording access to the boilers for the purpose of inserting and operating means for cleaning and for purposes of the examination of the boilers. It will be understood that these openings 21^a are normally closed by such suitable means as may be conveniently employed. The furnaces 11 and 12, as plainly shown in Fig. 3, 55 are provided with the suitable gratings 22, supported by proper fastenings in the walls, and underneath the gratings are the suitable draft-chambers 23. The object of the longitudinal walls 24 is to provide separators by 60 which the contact with the boilers or the

drums or any portion thereof of unheated atmosphere is prevented. Extending transversely and suitably secured to the furnace-walls 10^a and 10 are the aprons 25 and 26, made of suitable fireproof material, supported on suitable girders properly secured in the walls. In the furnace 11 is provided the upwardly-extending dash 27, and in the furnace 12 is provided the vertically-disposed continuation 28 of the wall 24. Formed integral with the wall 28 and disposed diagonally is the dash 29, and formed integral with the apron 26 is the depending dash 30. The walls 10, forming the rear part of the furnaces, are continuous. Provided in the front 80 wall of the furnaces are the fire-doors 31 and the ash-pit doors 32. For bracing and stiffening the furnace structure the vertically-disposed angle-bars 33, secured at their bottoms to proper foundation and bound together at their tops longitudinally and transversely by the tie-rods 34, are provided. In the enlarged portion of the wall of the retort 8, adjacent and superincumbent the furnace 12, is provided the vertically-extending passage 35, adapted to conduct a portion of the freshly-generated heat of furnace 12 directly into the chamber 14.

36 designates the inlet-pipe through which 95 passes the water intended for use in the boilers. The entry for this pipe, which is provided with the suitable regulator and check valve 36^a, is provided at the chamber 18, whence it extends upwardly within the smoke-stack to a point above the height of 100 the water-drum of the boiler 1, into which drum 5 it is secured at its top. This pipe is shown as traversing the extent within the smoke-stack in a spiral. The object of this arrangement is to afford as much area as possible 105 for the water to travel through the heat contained in the stack and before reaching the boiler. Provided at the mud-drums 3 and 4 and communicating connecting the same at a point well above the bottoms of 110 these drums to permit the settling in drum 3 of such sediment as may be carried by the water, is the pipe 37, through which the water may freely pass from boiler 1 to boiler 2. This pipe is provided with the suitable regulator 115 and check valve 38, which is manipulated by the stem extended outside the apparatus and provided with the wheel 38^a, as shown in Fig. 2, the purpose of said valve being to afford means for the regulation and check of the 120 flow of the water from boiler 1 into boiler 2, and the prevention, under the steam-pressure within the boiler 2, of the backflow of the water from the boiler 2 to boiler 1. It will be understood that the boiler 1, drums 3 125 and 5, and pipe 36, as well as the drum 4 and tubes 2, are kept completely filled with water under pump-pressure and suitable system of check and regulator valves well known and universally used in steam-boilers, 130

so that under the high degree of furnace temperature maintained about the tubes 2 the height of the water-level therein may by the manipulation of the suitable regulator and 5 check valves be controlled, and thus may the evaporating-space be confined in the drum 6 to such extent as may be desired. The blow-off pipes 39 and 40 are provided at the bottoms of the boilers 1 and 2, respectively, for the discharge, when desired, of the 10 contents of the boilers.

Within the drum 6 is provided the shell 41, the body of the same being cylindrical and adapted to reside concentrically within the 15 drum. The upper portion 41^a of this shell is slightly flaring and the lower portion 41^b is bent obliquely outwardly, thence flanged vertically and rigidly secured to the walls of the drum, the lower line of this shell marking 20 the line which the water in the boiler is intended to assume and retain in the process of steam-generation.

42 designates an ordinary water-gage, and 25 43 represents a safety-valve suitably secured to the crown 44 of the drum 6.

44^a designates a suitable discharge-pipe through which may be discharged such refuse as may accumulate within the chamber 46.

Formed by the shell 41 is the chamber 46, 30 which constitutes a suitable receptacle for the generated steam and wherein the same is superheated before its final escape at such time and in such quantities as may be desired through the outlet-pipe 47.

35 In the operation of my invention water is injected into the inlet-pipe 36, and the boilers are filled. Fire is then started in the furnaces 11 and 12. By the peculiar formation of the interior of the furnaces the fire and 40 flame and the heated gases of combustion in their escape therefrom are impelled downwardly by the apron 25 and the dash 27 in and through the spaces between the boiler-tubes and by the apron 26 and the dashes 45 28 and 29 caused to ascend, a certain portion of the heat from furnace 12 ascending through passage 35 to and keeping a continuous heat of high degree in the chamber 14, whereby a continuous and intense heat is 50 maintained in contact with that portion of the boiler wherein the evaporation of the water takes place. By the means described of discharging the heat from the furnaces into the retort 8 such circulation of the same 55 is obtained that the maximum and highest degree of heat obtained and generated by the fire is available for this boiler. From the retort 8 the heat passes through the aperture 15 into the retort 7, thence downwardly it 60 circulates in and about the tubes around and under the drum 3, thence through the opening 16 into the chamber 18, thence upwardly escaping through the stack 19. In the boiler 1 the tubes are of comparatively greater 65 number and smaller in size than in the boiler

2, the purpose being to afford a more extensive area for the contact of the heat of the retort 7 with the same by reason of the lower temperature of the incoming water. It will be seen that by this arrangement of the boilers and the construction and adaptation of the furnaces and retorts with reference to the same the spent portion of the heat from the furnaces passing through the chamber 18 and the smoke-stack 1 is utilized in primarily 70 heating the incoming feed-water. As the water increases in temperature it reaches those portions of the retorts wherein the more intense heat exists, thus absorbing a minimum amount of the said heat and enabling the heat to stimulate and superheat the water and more easily maintain the high temperature of the same, and, further, the heat afforded for the superheating of the steam and the steam retained to be superheated both being at their highest temperature the maximum degree of superheating is not only accomplished, but is easily maintained in the highest degree.

What I claim as my invention, and desire 90 to secure by Letters Patent of the United States, is—

1. In a steam-generating apparatus, the combination with a primary and secondary boiler positioned slightly apart and connected to each other by a conduit for circulation and suitable means for supplying with and preventing the escape of water from the said boilers, retorts inclosing said boilers, the retort of the secondary boiler being broadened 95 into furnaces on the opposite sides of said secondary boiler, said furnace having means whereby the fire therein is directed downwardly and upwardly from said furnaces into contact with the secondary boiler, the 100 secondary upwardly-extending passage 35 from the fire-box adjacent the secondary boiler whereby the heat is directed against the drum 6 thereof, a dividing-wall between 105 said boilers having an aperture therein near 110 the upper portion of said boilers, a discharge-chamber in the lower portion adjacent to and communicating with the retort surrounding the primary boiler and leading to the smoke-stack.

2. In a steam-generating apparatus, the combination with a primary and secondary boiler of the water-tube type, positioned slightly apart and vertically disposed and communicatingly connected to each other, 115 retorts inclosing said boilers, the retort of the secondary boiler being broadened into furnaces on the opposite sides of said secondary boiler, said furnaces having means wherein the fire therein is directed downwardly and upwardly from said furnaces into contact with the secondary boiler, the secondary upwardly-extending passage 35 from the fire-box adjacent the secondary boiler whereby 120 the heat is directed against the drum 6 there- 125

of, a dividing-wall between said boilers having an aperture therein near the upper portion of the boilers, a discharge-chamber in the lower portion of the retort surrounding the primary boiler having a passage leading therefrom and communicating with the smoke-stack, a pipe arranged within the smoke-stack through which said pipe the feed-water for the boilers is projected upwardly and fed into the upper portion of the primary boiler, a pipe connecting the primary boiler to the secondary boiler at a point above and apart from the bottom part of said boilers and having a regulator and check valve therein, means in the upper drum of the secondary boiler whereby the generated steam may be retained in a receptacle apart from the wet steam of evaporation, suitable outlet from said drum affording egress of the steam generated, from the superheating-receptacle thereof.

3. In a steam-generating apparatus, the combination with a water-tube boiler vertically disposed within a suitable heating-furnace and having a drum at its upper portion and suitable secondary passage from the heating-furnace for directing and discharging a continuous and uniformly high degree of heat against the exterior of the same, of a cy-

lindrical shell of smaller diameter than the drum having its upper edge flaring and its lower portion bent obliquely outwardly and with vertically-disposed continuous flange secured to said drum upon a line substantially intermediate the bottom and top of said drum, said drum being provided with an outlet in the wall thereof at a point below the plane of the top of the said shell.

4. A furnace for the generation of steam in a water-tube boiler comprising oppositely-disposed fire-boxes and a superincumbent and communicating retort connected therewith and adapted to surround said boiler and having suitable escape-passage, said fire-boxes being respectively provided with the apron 25, the dash 27, the aprons 26 and 30, the dashes 28 and 29 so disposed that the fire generated will be conducted from the fire-boxes into the retort downwardly and upwardly.

In testimony whereof I have hereunto signed my name to this specification in the presence of two subscribing witnesses.

ISAAC H. BOYER.

Witnesses:

GEO. R. JONES,

SAMUEL M. SNODGRASS.