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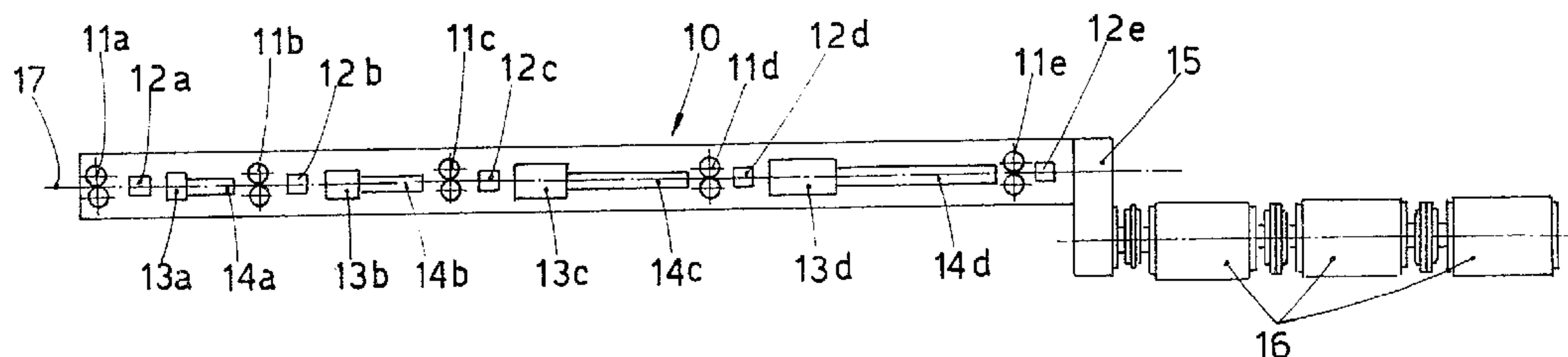
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(54) **TRAIN DE LAMINAGE A ROULEAUX MULTIPLES MONTES
SUR SUPPORT EN PORTE-A-FAX POUR LAMINAGE A
HAUTE VITESSE**

(54) **ROLLING STAND WITH MULTIPLE ROLLS SUPPORTED AS
CANTILEVERS FOR HIGH-SPEED ROLLING**



(57) Rolling stand with multiple rolls supported as cantilevers for high-speed rolling, which comprises a motor unit (16-15) that actuates at least one common shaft (22a) which transmits motion to a plurality of pairs of rolls (11-12) with their axes placed alternately substantially at a right angle to each other (11a, 12a, 11b, 12b.....11n, 12n), the stand including from six to ten pairs of rolls (11-12), the rolled stock being kept under traction between one pair of rolls (11-12) and the next pair (11-12), cooling chamber means (13) followed by temperature equalisation chamber means (14) being included between the pair of rolls (12) which processes round sections and the pair of rolls (11) which processes oval sections, the length of the cooling chamber means (13) being at least a function of the rolling speed of the pair of rolls (12) positioned upstream of that cooling chamber means (13), the cooling chamber means (13) being progressively longer in the direction of feed of the rolled stock.

ABSTRACT

Rolling stand with multiple rolls supported as cantilevers for high-speed rolling, which comprises a motor unit (16-15) that actuates at least one common shaft (22a) which transmits motion to a plurality of pairs of rolls (11-12) with their axes placed alternately substantially at a right angle to each other (11a, 12a, 11b, 12b.....11n, 12n), the stand including from six to ten pairs of rolls (11-12), the rolled stock being kept under traction between one pair of rolls (11-12) and the next pair (11-12), cooling chamber means (13) followed by temperature equalisation chamber means (14) being included between the pair of rolls (12) which processes round sections and the pair of rolls (11) which processes oval sections, the length of the cooling chamber means (13) being at least a function of the rolling speed of the pair of rolls (12) positioned upstream of that cooling chamber means (13), the cooling chamber means (13) being progressively longer in the direction of feed of the rolled stock.

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1 **"ROLLING STAND WITH MULTIPLE ROLLS SUPPORTED AS**
2 **CANTILEVERS FOR HIGH-SPEED ROLLING"**

3 * * * * *

4 This invention concerns a rolling stand with multiple
5 rolls supported as cantilevers for high-speed rolling. By
6 high-speed rolling is meant here a rolling speed of 120 to
7 140 metres per second or more with sections having a
8 diameter of 5.5 to 6.5/7 or equivalent.

9 The rolling stands with which the invention is concerned
10 are known commercially as "high-speed blocks" and comprise a
11 plurality of pairs of rolls supported as cantilevers, the
12 pairs having their axes rotated alternately by 90° to each
13 other so as to process an oval section, round section, oval
14 section, round section and so on.

15 The stands according to the invention include from six to
16 ten alternated pairs of rolls supported as cantilevers.

17 These stands comprise a source producing motion and one or
18 two transmission shafts which transmit the motion to all the
19 pairs of rolls.

20 One shaft advantageously transmits motion to the pairs of
21 rolls having, for instance, a horizontal axis, while another
22 shaft transmits motion to the pairs of rolls having, for
23 instance, a vertical axis.

24 The reduction ratio as between one pair of rolls and the
25 next is preset, so that the speed of rotation of the two
26 pairs keeps the rolled stock under slight traction between
27 the pairs.

28 The present applicant is not aware of rolling stands of
29 the above type which are suitable to process rolled stock,
30 normally round with sections having a diameter of 5.5 to
31 6.5/7 or equivalent shapes, at a speed of more than 120
32 metres per second and advantageously 140 metres per second
33 or more.

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1 The present applicant has found that when working under
2 the above conditions, for instance on a stand having ten
3 alternated pairs of rolls supported as cantilevers, a
4 considerable increase of temperature takes place in the
5 rolled stock, and it is therefore impossible, in fact, to
6 reach high speeds in the present high-speed blocks.

7 The present invention enables the speeds cited above to be
8 reached with the high-speed blocks processing the relative
9 sections.

10 According to the invention a fast-cooling chamber is
11 located immediately after the pair of rolls which processes
12 round sections and after each pair of intermediate rolls
13 which processes round sections.

14 The present applicant has found that after such cooling
15 chamber it is advantageous to place a temperature
16 equalisation chamber so as to prevent the rolled stock
17 arriving at the first pair of rolls (that processes oval
18 sections) with its surface layer too cold and its core hot.

19 The present applicant has also found that the length of
20 the cooling chamber and the length of the temperature
21 equalisation chamber are correlated respectively with the
22 speed of departure of the rolled stock from the pair of
23 rolls which processes round sections.

24 Moreover, the present applicant has found that the lengths
25 of the above two chambers become progressively greater.

26 According to the investigations and experiments carried
27 out by the present applicant in a case of a variant with the
28 purpose of reducing the dimensions of the high-speed block -

1 dimensions which depend also on the diameter of the common
2 transmission shaft or shafts - and with a view to enhancing
3 the versatility of the high-speed block, the invention
4 provides for the inclusion of an additional motor unit
5 downstream of the first pair of rolls, thus creating a first
6 section and a second section in the high-speed block itself.

7 This additional motor unit drives the first two, four or
8 six pairs of rolls, the common transmission shaft or shafts
9 of which remain connected to the like transmission shaft or
10 shafts of the remaining pairs of rolls.

11 According to a variant the mechanical connection between
12 the two sections of the high-speed block includes a
13 disconnectible joint which enables the pairs of rolls of
14 the first section and/or of the second section to be
15 employed independently and on their own.

16 The present applicant has also found that, when rolling
17 given alloys or superalloys, and in particular given types
18 of steel, the leading end of the rolled stock tends to
19 bifurcate and creates great problems of lead-in after the
20 fourth pair of rolls.

21 The invention therefore arranges, as a variant, that after
22 the fourth pair of rolls, which processes a round section,
23 and before the fifth pair of rolls, which processes an oval
24 section, a flying shears is fitted which is able to shear
25 the leading end of the rolled product passing through and
26 discharge that end.

27 The invention arranges equally well that the shears should
28 be located near to the outlet of the round pass or to the
29 inlet of the oval pass.

30 The attached figures, which are given as a non-restrictive
31 example, show the following:-

32 Fig.1 shows a diagram of a high-speed block according to the
33 invention;

1 Fig.2 shows a diagram of a high-speed block divided into two
2 sections according to the invention;

3 Fig.3 shows a diagram of a high-speed block divided into two
4 sections according to the invention, an intermediate
5 flying shears being included;

6 Fig.4 shows a diagram of a normal vertical section of a
7 high-speed block;

8 Fig.5 shows a diagrammatic front view of a high-speed block
9 according to the invention in the zone of connection
10 of the first and second sections.

11 In the figures a high-speed block 10 comprises, as an
12 example, one or more motors 16 which transmit motion through
13 a gearbox 15 to a first gearwheel 24 that actuates a second
14 gearwheel 23, which in turn sets in rotation two common
15 shafts 22a.

16 The common shafts 22a run along the whole high-speed
17 block 10 and transmit motion to units 26 which power,
18 support and position pairs of rolls 11 and 12.

19 The pairs of rolls 11 and 12 have a progressively greater
20 peripheral speed of rotation, so that the last pair 11e
21 rotates at the maximum speed of rotation of about 120 to 140
22 metres per second or more when processing sections having a
23 diameter of 5.5 to 6.5/7 mm.or equivalent.

24 In the example shown the pairs of rolls 11 serve to roll
25 oval sections, whereas the pairs of rolls 12 serve to roll
26 round sections.

27 The example shown provides for ten pairs of alternated
28 rolls 11 and 12 supported as cantilevers. The rolling axis
29 is referenced with 17.

30 The mechanical design embodiments shown are equally
31 suitable; those shown are given only as examples.

32 According to the invention a first cooling chamber 13 and
33 a second temperature equalisation chamber 14 are located in

1 strict coordination between a pair of rolls 12 that
2 processes round sections and a pair of rolls 11 that
3 processes oval sections and downstream of each intermediate
4 pair of rolls 12.

5 The length of the cooling chambers 13 and of the
6 temperature equalisation chambers 14 are a function of the
7 speed of departure of the rolled stock from the pairs of
8 rolls 12 that processes round sections and, in the case of
9 the temperature equalisation chambers 14, are a function
10 also of the gradient of cooling applied to the rolled stock
11 in the cooling chambers 13.

12 According to the variant of Fig.2 an auxiliary motor 18 is
13 included downstream of the fourth pair of rolls 12b and
14 governs the first four pairs of rolls 11a-12a-11b-12b by
15 means of a gearbox 19.

16 Respective common shafts 22a of two sections 20a and 20b
17 of the high-speed block 10 do not entail any mechanical
18 discontinuity owing to the inclusion of shafts 22 which
19 connect the two sections rigidly.

20 Joint 25 may be fitted on the shaft 22 and may be
21 disconnected so as to disconnect the section 20a from the
22 section 20b and to make these sections momentarily
23 independent.

24 Whether the gearboxes 15 and 19 are positioned at the head
25 or tail of the sections will depend on the design.

26 According to the variant of Fig.3 a flying shears 21
27 suitable to shear the leading end of the rolled stock
28 travelling through is located between the fourth pair of
29 rolls 12b and fifth pair of rolls 11c.

30 It is obvious that no physical lack of continuity can
31 exist between the cooling chamber 13 and temperature
32 equalisation chamber 14. Moreover, the temperature
33 equalisation chamber 14 may also possess a cooling function,

- 1 even though very modest or having a value which can be
- 2 varied along the length of the chamber 14.

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CLAIMS

1. Rolling stand for rolling a rolling stock with multiple rolls supported as cantilevers for high-speed rolling, which comprises a motor unit (16-15) that actuates at least one common shaft (22a) which transmits motion to a plurality of pairs of rolls (11-12) with their axes placed alternately substantially at a right angle to each other (11a, 12a, 11b, 12b...11n, 12n), the stand including from six to ten pairs of rolls (11-12), the rolled stock being kept under traction between one pair of rolls (11-12) and the next pair (11-12), the stand being characterized in that cooling chamber means (13) followed by temperature equalisation chamber means (14) are included between a first pair of rolls (12) which processes round sections and a second pair of rolls (11) which processes oval sections, the length of the cooling chamber means (13) being correlated to the rolling speed of said first pair of rolls (12), the cooling chamber means (13) being progressively longer in the direction of feed of the rolled stock.

2. Stand as claimed in Claim 1, in which the length of the temperature equalisation chamber means (14) is correlated to the rolling speed of said first pair of rolls (12) positioned upstream of that chamber means (14) and to the gradient of cooling imparted to the rolled stock by the cooling chamber means (13) positioned

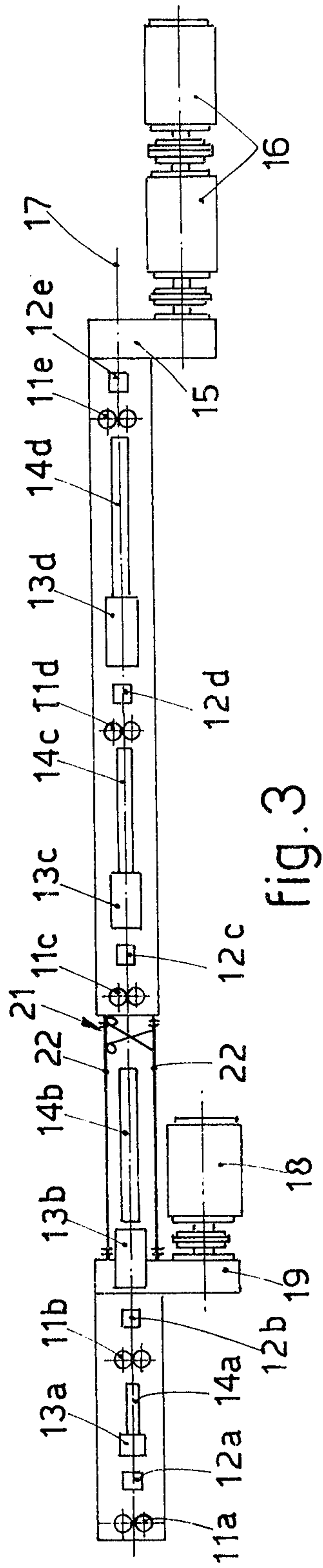
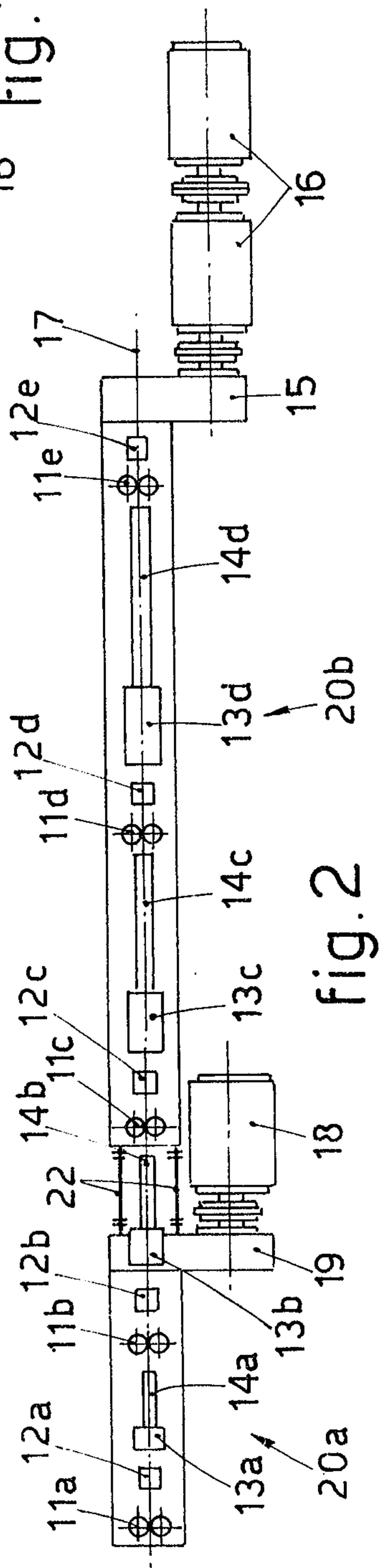
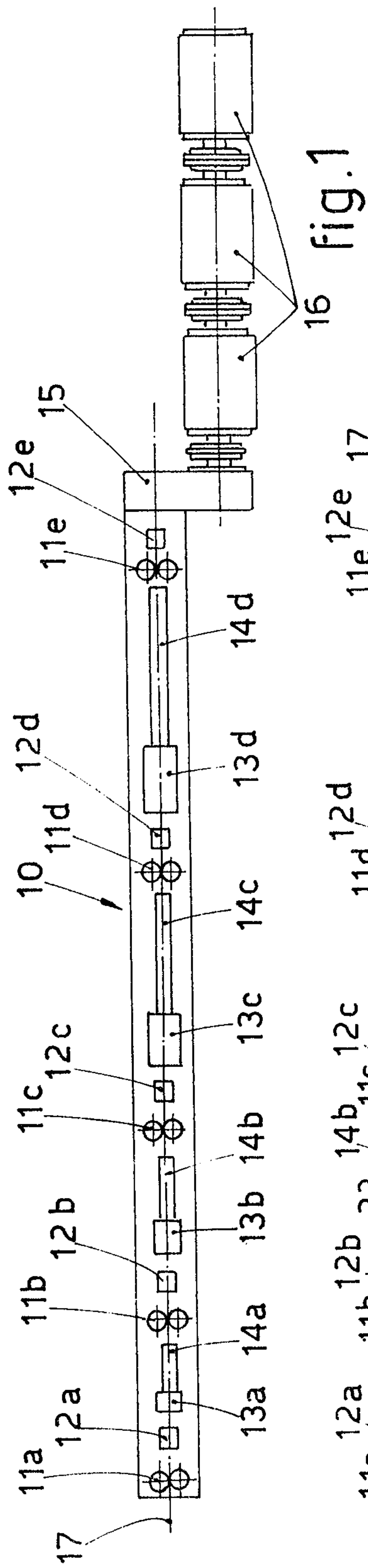
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upstream of the temperature equalisation chamber means (14) .

3. Stand as claimed in Claim 1 or 2, which is divided into two in-line sections (20a-20b) with independent drive means (16-15 and 18-19), the respective common shafts (22a) of the two sections (20a-20b) being connected mechanically and rigidly by a further shaft (22) .

4. Stand as claimed in any one of Claims 1 to 3, in which the further shaft (22) providing mechanical connection of the respective common shafts (22a) of the two sections (20a-20b) comprises joint means (25) which can be momentarily disconnected.

5. Stand as claimed in any one of Claims 1 to 4, in which a flying shears (21) to shear at least a leading end of the rolled stock travelling through is included between said first pair of rolls (12) that processes round sections and said second pair of roll (11) that processes oval sections.



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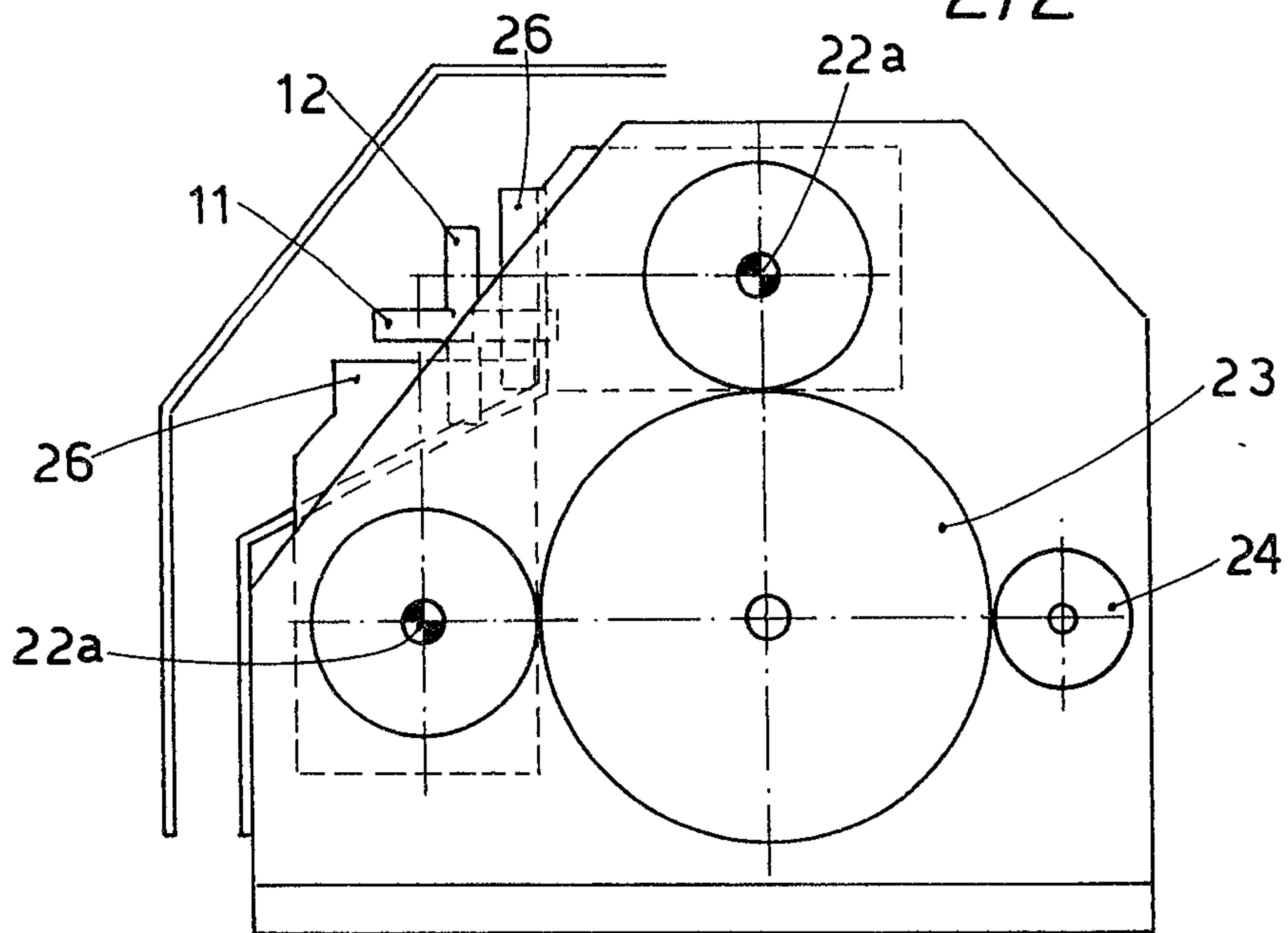


fig. 4

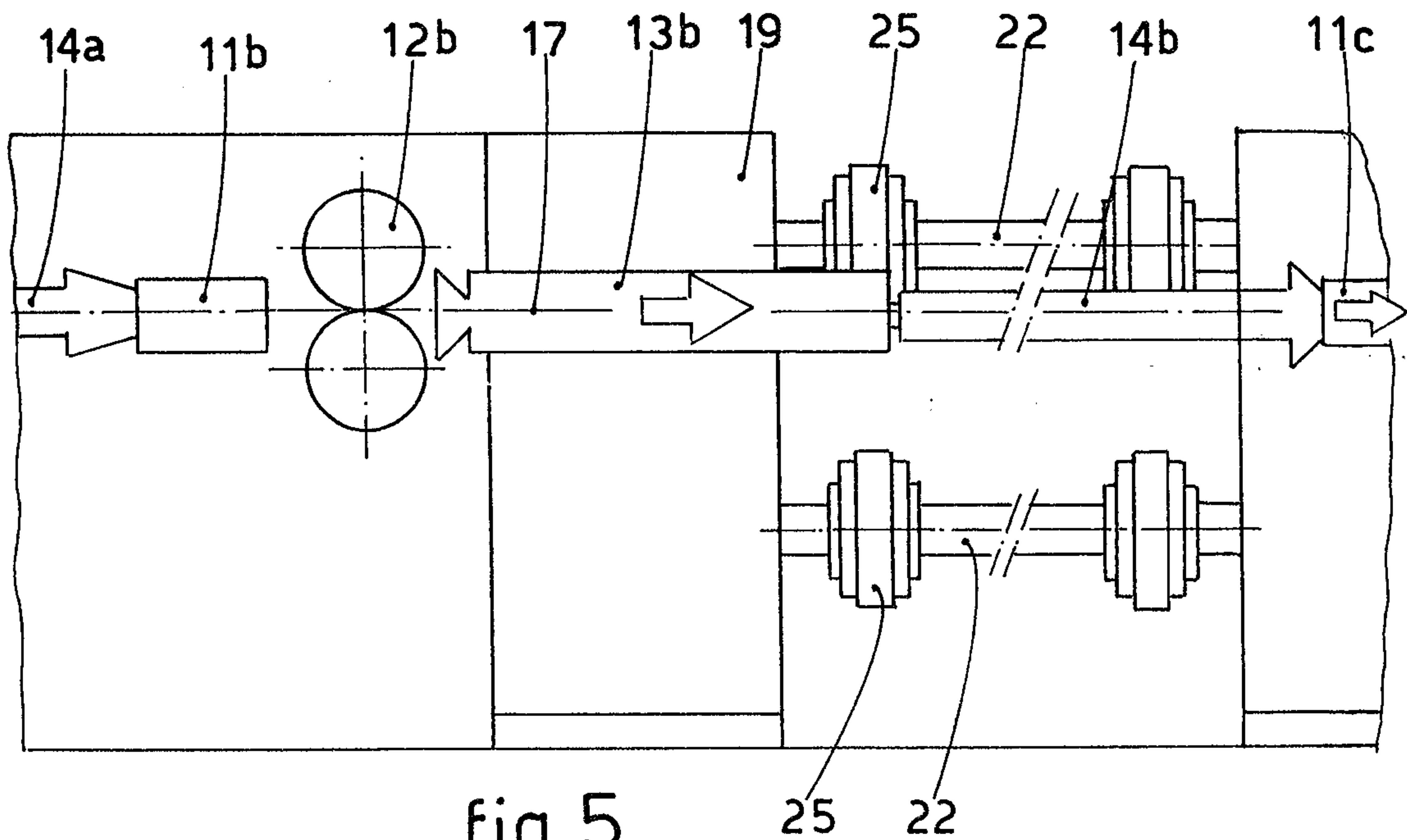


fig. 5

Andrew Lloyd Jones & Stephen Walker