



(10) International Publication Number
WO 2006/096276 A1

(57) Abstract: A method for coupling a valve member (16) to an armature member (18) includes coupling the valve member (16) to a surface of a bore (28) of the armature member (18). The valve member (16) contacts the armature member (18) over less than an entire axial thickness of the surface of the bore (28) of the armature member (18). The method further includes aligning the armature member (18) to a desired alignment and securing the valve member (16) to the armature member (18) at the desired alignment.

Description

VALVE COUPLING SYSTEM

Technical Field

The present disclosure relates generally to a system and method for use in a valve assembly, and more particularly to a system and method for coupling a valve component to an armature component for use in a valve assembly.

Background

Within a work machine, valve assemblies typically control the flow of a gas or liquid. A simple valve assembly has two positions. In its first position, the valve assembly prevents flow of the gas or liquid. In its second position, the valve assembly allows flow. More sophisticated valve assemblies simply expand on this idea. For example, a valve assembly may move between two positions. In its first position, the valve assembly may allow communication between a first passage and a second passage and prevent communication to and from a third passage. In its second position, the valve may prevent communication to and from the first passage and allow communication between the second and third passages.

Internal combustion engines of work machines that use fuel injectors require valves within the fuel injector to operate precisely. For example, some fuel injectors utilize a valve for controlling over when fuel is injected into the combustion chamber. One problem with valves in fuel injectors occurs when the valve components do not respond as predicted. For example, typically solenoid actuators in a fuel injector act on an armature connected to a valve component to control the movement of the valve component in the fuel injector. If the armature and the valve component are coupled improperly, the

-2-

valve component may not respond as predicted. This could alter the timing of when fuel enters the combustion chamber and lead to inefficiencies and/or unwanted emissions.

Another problem an improperly coupled armature and valve component may cause is that the valve component may not seat properly. An improperly seated valve component may not fully close the valve, thereby allowing an undesired amount of fuel to enter the combustion chamber. This can also lead to inefficiencies and/or unwanted emissions in the work machine.

There are many ways to connect a valve component to an armature. U.S. Patent No. 5,271,371 ("the '371 patent") discloses using a threaded fastener to join an armature to a valve component for use in a control valve of a fuel injector. This connection, however, has drawbacks. First, if after the armature was connected to the valve component it was discovered that the two components were not properly aligned, the costs and/or difficulty associated with correcting the misalignment may warrant discarding the entire assembly. If an improperly aligned armature and valve component are installed, the valve component may not obtain proper valve component seating around the circumference of the valve component. As noted above, this may prevent the valve component from fully closing and may cause leaks. As also noted above, such an improperly coupled armature and valve component may not respond as predicted due to the improper connection between the armature and the valve component. A valve component that does not respond as predicted may cause fuel inefficiencies and/or unwanted emissions.

The method and apparatus of the present disclosure solves one or more of the problems set forth above.

Summary of the Disclosure

In accordance with one exemplary embodiment, a method for coupling a valve member to a surface of a bore of an armature member is provided. The valve member contacts the armature member over less than an

-3-

entire axial thickness of the surface of the bore of the armature member. The method further includes aligning the armature member to a desired alignment and securing the valve member to the armature member at the desired alignment.

In accordance with another exemplary embodiment of the present disclosure, a method is provided for coupling a valve member to an armature member. The method includes coupling the valve member to the armature member such that a substantial line contact is established between the valve member and a surface of a bore of the armature member. The method further includes aligning the armature member to a desired alignment and securing the valve member to the armature member at the desired alignment.

According to another exemplary embodiment, a valve member is provided that includes two reduced diameter portions separated by an intermediate portion having a maximum diameter greater than the diameter of either of the two reduced diameter portions. At least a portion of the intermediate portion includes an armature receiving portion.

According to yet another embodiment, a valve assembly is provided that includes a valve member having an intermediate portion positioned adjacent at least one reduced diameter portion. An armature member includes a bore sized to receive the valve member intermediate portion and sized to allow substantial angular movement relative to the valve member.

Brief Description of the Drawings

FIG 1. is a partial cross-sectional side view of an exemplary solenoid actuated control valve assembly in accordance with the present disclosure;

FIG. 2 is a partial cross-sectional side view of the armature and the valve member in accordance with the present disclosure;

FIG. 3A is a cross-sectional side view of the valve member in accordance with one embodiment of the present disclosure;

-4-

FIG. 3B is a cross-sectional side view of the valve member in accordance with an alternative embodiment of the present disclosure;

FIG. 3C is a cross-sectional side view of the valve member in accordance with yet another alternative embodiment of the present disclosure;

5 FIG. 3D is a cross-sectional side view of the valve member in accordance with another alternative embodiment of the present disclosure;

FIG. 4A is a partial cross-sectional side view of the armature and the valve member initially misaligned in accordance with the present disclosure;

10 FIG. 4B is another partial cross-sectional side view of the armature and the valve member misaligned in accordance with the present disclosure;

FIG. 5 is a partial cross-sectional side view of the armature and the valve member secured to one another in accordance with the present disclosure;

15 FIG. 6 is a flowchart illustrating an exemplary method of manufacturing in accordance with the present disclosure;

FIG. 7A is a partial cross-sectional side view of the armature and the valve member in accordance with another embodiment of the present disclosure; and

20 FIG. 7B is a partial cross-sectional side view of the armature and the valve member in accordance with another embodiment of the present disclosure.

Detailed Description

Reference will now be made in detail to exemplary embodiments of the disclosure, illustrated in the accompanying drawings. Wherever possible, 25 the same reference numbers will be used throughout the drawings to refer to the same or like parts.

Referring to the figures, a solenoid actuated control valve assembly 10 is shown generally in Fig. 1. Valve assembly 10 may be of the type utilized in admission valves, spill valves, check control valves, such as those

-5-

found in, but not limited to, hydraulically actuated fuel injectors, mechanically actuated fuel injectors, and common rail fuel injectors.

Valve assembly 10 may include a solenoid 12, a bias spring 14, a valve member 16, an armature 18, a first passage 20, a second passage 22, and a third passage 24. It should be understood, however, that the system disclosed here is not limited for use with this particular valve assembly 10. It should be understood that the disclosed system may be utilized to join any valve component and any armature component for use in a valve assembly.

Bias spring 14 may bias armature 18 and valve member 16 downward towards a first or closed position. When activated, the solenoid 12 urges armature 18 and valve member 16 upwards towards a second or open position.

In its closed position as shown in Fig. 1, valve member 16 prevents fluid communication between third passage 24 and first and second passages 20, 22 and allows fluid communication between first and second passages 20, 22. In its open position, valve member 16 allows fluid communication between third passage 24 and second passage 22 and prevents fluid communication between first passage 20 and second and third passages 22, 24.

Referring to Fig. 2, valve member 16 may be generally cylindrical in shape. Alternatively, valve member 16 may take another similar shape. Valve member 16 may have a seat end 30 positioned opposite from an armature end 32. Seat end 30 may include a varying diameter valve seat portion 36 where the diameter of valve member 16 tapers down on opposite sides, creating a first reduced diameter portion 38. Alternatively, the varying diameter valve seat portion 36 may include any other appropriate shape. For example, valve seat portion 36 may include a first increased diameter portion with a diameter larger than the maximum diameter of the seat end 30. Varying diameter valve seat portion 36 may be manufactured by cutting or grinding valve member 16.

-6-

Alternatively, varying diameter valve seat portion 36 may be formed when valve member 16 is originally created using a mold or similar method of manufacturing.

Valve member 16 may include a second varying diameter portion 26 located near armature end 32. Second varying diameter portion 26 may include a second reduced diameter portion 40 and a third reduced diameter portion 42. The diameter of valve member 16 may gradually reduce to form second and third reduced diameter portions 40, 42. For example, second and third reduced diameter portions may be formed by cutting a planar or curved groove into valve member 16. Valve member 16 may also include an intermediate portion 44 located between second and third reduced diameter portions 40, 42.

For use with this disclosure, the intermediate portion 44 will be defined as that portion of the valve member 16 extending from a point 49 in Fig. 3a to a point 53. The point 49 may be located on the side of the second reduced diameter portion 40 closest to the third reduced diameter portion 42, and positioned at half the depth of the second reduced diameter portion 40. The point 53 may be located on the side of third reduced diameter portion 42 closest to the second reduced diameter portion 40, and positioned at half the depth of the third reduced diameter portion 42. It should be understood that point 49 and point 53 may be located at different depths. Second and third reduced diameter portions 40, 42 and intermediate portion 44 may be manufactured by cutting or grinding valve member 16 or, may be formed when valve member 16 is originally created using a mold or similar method of manufacturing. As will be described in more detail below, intermediate portion 44 may include a diameter the same as or slightly larger than the maximum diameter of the remaining portions of valve member 16.

Intermediate portion 44 may be curved to include an elongated spherical or barrel shape as shown in Fig. 3A. Alternatively, intermediate portion

-7-

44 may be curved to include a substantially spherical shape as shown in Fig. 3B. Intermediate portion 44 may also taper substantially to form a point 46, shown in Fig. 3C, or a substantially straight portion 47, as shown in Fig. 3D. It should be understood that the intermediate portion 44 may take on any other shape and may include a maximum diameter of any size allowing for receipt and contact with armature 18. As shown in these figures, the maximum diameter of the intermediate portion 44 may be slightly larger than the minimum diameter of the remaining portions of valve member 16.

Referring to Fig. 4A, the intermediate portion 44 may include an armature receiving portion 34 for contacting the armature 18 so as to allow for angular rotation of the armature 18 with respect to the valve member 16. In the exemplary embodiments, the armature receiving portion 34 may extend over a portion of intermediate portion 44 less than the axial thickness of armature 18 (Figs. 3A-3C). For example, contact between the valve member 16 and the armature 18 at the armature receiving portion 34 may be substantially less than the axial thickness of the armature 18. For example, the armature 18 may contact armature receiving portion 34 over a substantially flat portion 47 of intermediate portion 44 as shown in Fig. 3D. Alternatively, the armature receiving portion 34 may provide substantially line contact with the armature 18 (Figs. 3A-3C). As described herein substantially line contact includes a mating of the armature 18 with the intermediate portion 44 of the valve member 16 that allows for angular rotation in the directions 48 and 50 in Figs. 4A and 4B in the magnitude of at least 20 degrees. The size of the armature receiving portion 34 may vary due to the inexactness of manufacturing these small components and the particular shape of the intermediate portion 44. It should be understood that the armature receiving portion 34 may take on any other shape and may include a maximum diameter of any size allowing for contact and angular rotation of the armature 18 with respect to the valve member 16.

-8-

Referring back to Fig. 2, armature 18 may be circular in shape and may include a bore 28. Bore 28 may be co-axially located at the center of armature 18. It should be understood that bore 28 could be located at another position on armature 18. Bore 28 may be created by cutting or grinding armature 18. Alternatively, armature 18 could be manufactured with bore 28 using a method known in the art. Bore 18 may be sized for receiving valve member 16 and may be generally cylindrical in shape and sized to allow for receipt of valve member 16 therethrough such that the intermediate portion 44 may be received within the bore 28.

It should be understood that the present disclosure may be practiced using an armature having a bore with varying diameters and a substantially constant diameter valve member as illustrated in Fig. 7A. Alternatively, both the bore and the valve member may include varying diameter portions for engaging one another, as illustrated in Fig. 7B.

Industrial Applicability

Referring to Fig. 2, to couple valve member 16 to armature 18, one end of valve member 16 may be inserted into bore 28 of armature 18 until the slightly larger armature receiving portion 34 of the intermediate portion 44 contacts the bore 28 of armature 18. Because the armature receiving portion 34 contacts the bore 28 over a thickness less than the axial thickness of the bore 28, for example providing substantially line contact (Fig. 3A-3C), the armature 18 and valve member 16 can be re-aligned after valve member 16 is inserted into armature 18. Thus, armature 18 may be aligned as desired, for example, aligned generally perpendicular to valve member 16. To align the two components generally perpendicularly, the armature 18 may be angularly rotated. For example, Fig. 4A and Fig. 4B illustrate two possible alignments between armature 18 and valve member 16 after valve member 16 has been inserted into bore 28. It should be understood that other alignments are possible. Rotating armature 18 as indicated by arrow 48 in Fig. 4A and by arrow 50 in Fig. 4B

allows for aligning armature 18 generally perpendicularly to valve member 16. Additionally, if required, armature 18 and valve member 16 may be rotated relative to one another about a longitudinal axis of valve member 16 as shown by arrow 51 in Fig. 4A.

5 Referring to Fig. 5, once aligned perpendicularly to one another, armature 18 and valve member 16 are secured together. In the preferred embodiment, armature 18 is secured to valve member 16 using a weld. Welding material 52 is positioned between bore 28 of armature 18 and third reduced diameter portion 42 of valve member 16. Next, the components are welded
10 together. Once the weld is completed, the welding material 52, armature 18, and valve member 16 form a secured or fixed unit. It should be understood that the weld may be alternatively applied by positioning the welding material 52 between bore 28 and second reduced diameter portion 40 and applying the weld at this position. It should be further understood that a weld may be placed
15 between bore 28 and both second and third reduced diameter portions 40, 42. Other methods known in the art may be used to secure armature 18 to valve member 16 after the desired alignment, such as using resins, mechanical fasteners and the like.

In the exemplary embodiments, valve member 16 may be pre-
20 hardened before being secured to armature 18. Pre-hardening valve member 16 may serve to add strength to the coupling between armature 18 and valve member 16 and may be achieved in any conventional manner.

The flow chart of Fig. 6 illustrates a method for coupling armature 18 to valve member 16. The first step may include cutting or grinding valve
25 member 16 to create first, second and third reduced diameter portions 38, 40, 42. (Step 100). This step may or may not be necessary based on the type of process used to form the valve member 16. Also if necessary, the first step may include cutting or grinding armature member 16 to create bore 28. (Step 100). Next, the valve member 16 may be pre-hardened. (Step 102). This may be practiced in

-10-

any conventional way, such as heating the valve member to a certain temperature to obtain a desired carbonization.

Valve member 16 is next inserted into bore 28 of armature 18 until the slightly larger armature receiving portion 34 of the intermediate portion 44 engages bore 28 of armature 18. (Step 104). It should be understood that the present disclosure may be practiced using an interference fit between armature 18 and valve member 16 such that valve member 16 may be pressed into bore 28 of armature 18. Alternatively, the bore 28 may be expanded by heating before valve member 16 is inserted into bore 28. Therefore, this step may or may not include heating up the bore 28 to allow for the formation of an interference fit between valve member 16 and armature 18. Next, armature 18 is angularly rotated to align armature 18 in a desired manner, such as substantially perpendicular to valve member 16. (Step 106). As noted above, the size of the contact between the armature receiving portion 34 of the valve member 16 and the bore 28 of the armature 18 allows for the angular rotation between the armature 18 and the valve member 16. The armature receiving portion 34 contacts the bore 28 over a thickness less than the axial thickness of the bore 28 or over substantially line contact. This may be achieved with the assistance of additional, removable hardware that assures the desired alignment. Welding material 52 is positioned between bore 28 and the valve member 16, i.e., at the third reduced diameter portion 42. (Step 108). Using the welding material 52, the armature 18 and the valve member 16 are welded together to form a secure or fixed unit. (Step 110). It should be understood, however, that the armature 18 and the valve member 16 may be welded together with a laser or similar welding process known in the art without using the welding material 52.

The system disclosed ensures proper alignment between the valve member 16 and the armature 18 before coupling the two components together. This allows for more accurate operation as the armature member and valve member will respond as predicted. Furthermore, proper alignment between the

-11-

valve member and the armature member ensures that the valve member will get proper valve seating. Proper valve member seating assists in preventing an undesired amount of fuel from entering the combustion chamber causing unwanted emissions and/or inefficiencies. Proper valve member seating also
5 assists in ensuring fuel enters the combustion chamber only at the desired time further preventing inefficiencies and unwanted emissions.

Other embodiments of the disclosure will be apparent to those skilled in the art from consideration of the specification and practice of the disclosure discussed herein. For example, it is understood that while the
10 intermediate portion 44 is described as being slightly larger than the valve member 16, intermediate portion 44 may include the same maximum diameter as other portions of the valve member 16. In such an instance, the maximum diameter of the intermediate portion 44 may be substantially equal to or larger than the bore 28 of the armature 18. As described above, when the intermediate
15 portion 44 is larger than the bore 28, the bore 28 could be heated, and thus expanded, to allow for assembly with the valve member 16. It is intended that the specification and examples be considered as exemplary only, with a true scope and spirit of the disclosure being indicated by the following claims and their equivalents.

-12-

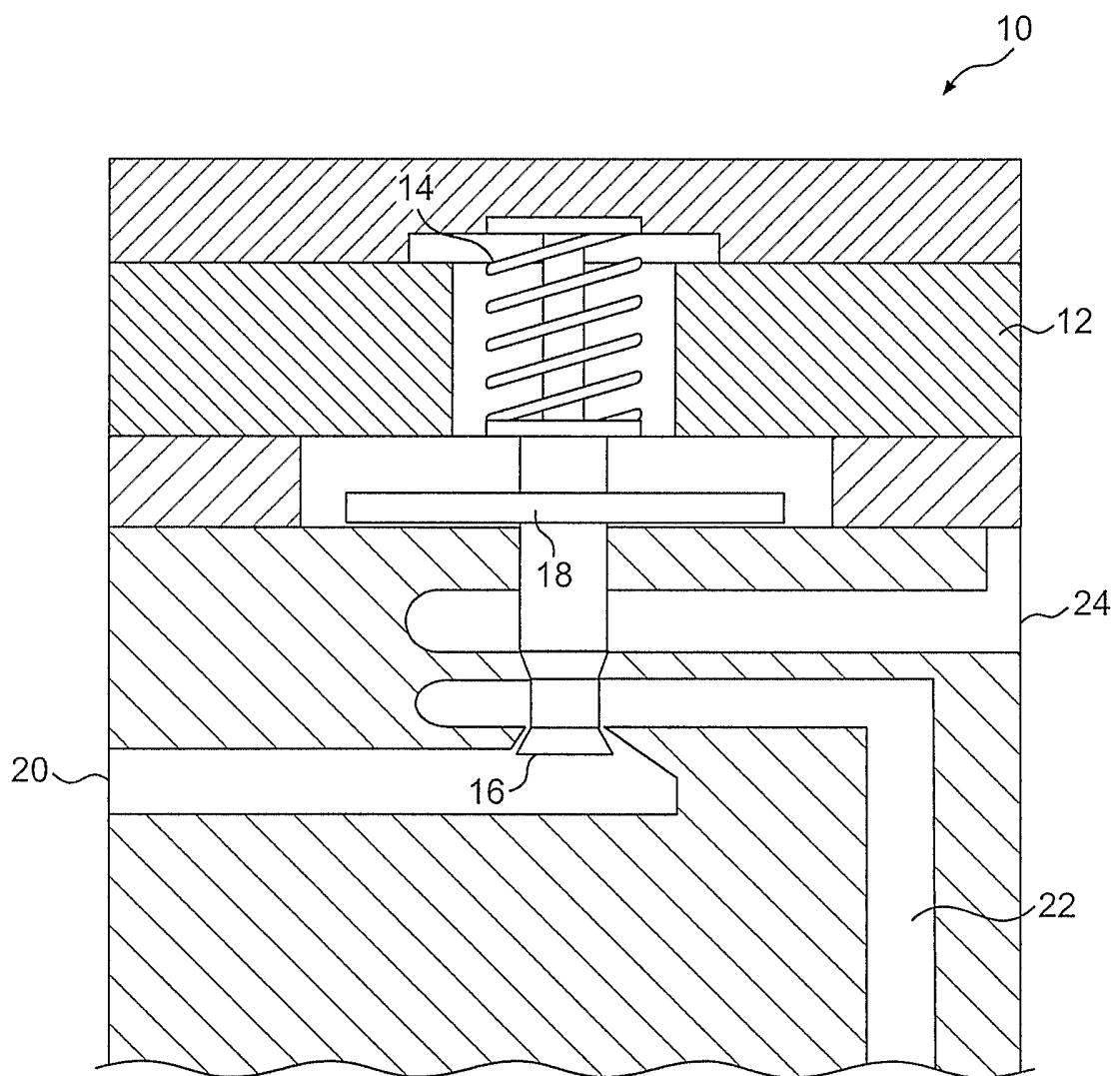
Claims

1. A method for coupling a valve member (16) to an armature member (18), comprising:
 - 5 coupling the valve member (16) to a surface of a bore (28) of the armature member (18) such that the valve member (16) contacts the armature member (18) over less than an entire axial thickness of the surface of the bore (28) of the armature member (18); and
 - aligning the armature member (18) to a desired alignment; and
 - 10 securing the valve member (16) to the armature member (18) at the desired alignment.
2. The method of claim 1, wherein the aligning includes aligning the armature member (18) to extend substantially perpendicular to the valve member (16).
- 15 3. The method of claim 2, wherein the valve member (16) includes at least one reduced diameter portion (40, 42) assisting in establishing the contact between the valve member (16) and the armature member (18).
4. The method of claim 3, wherein the at least one reduced diameter portion (40, 42) includes two reduced diameter portions (40, 42) located
20 on opposite sides of an intermediate portion (44), and the intermediate portion (44) includes a portion that provides the contact with the armature member (18).
5. The method of claim 4, wherein the intermediate portion (44) is generally spherical in shape.

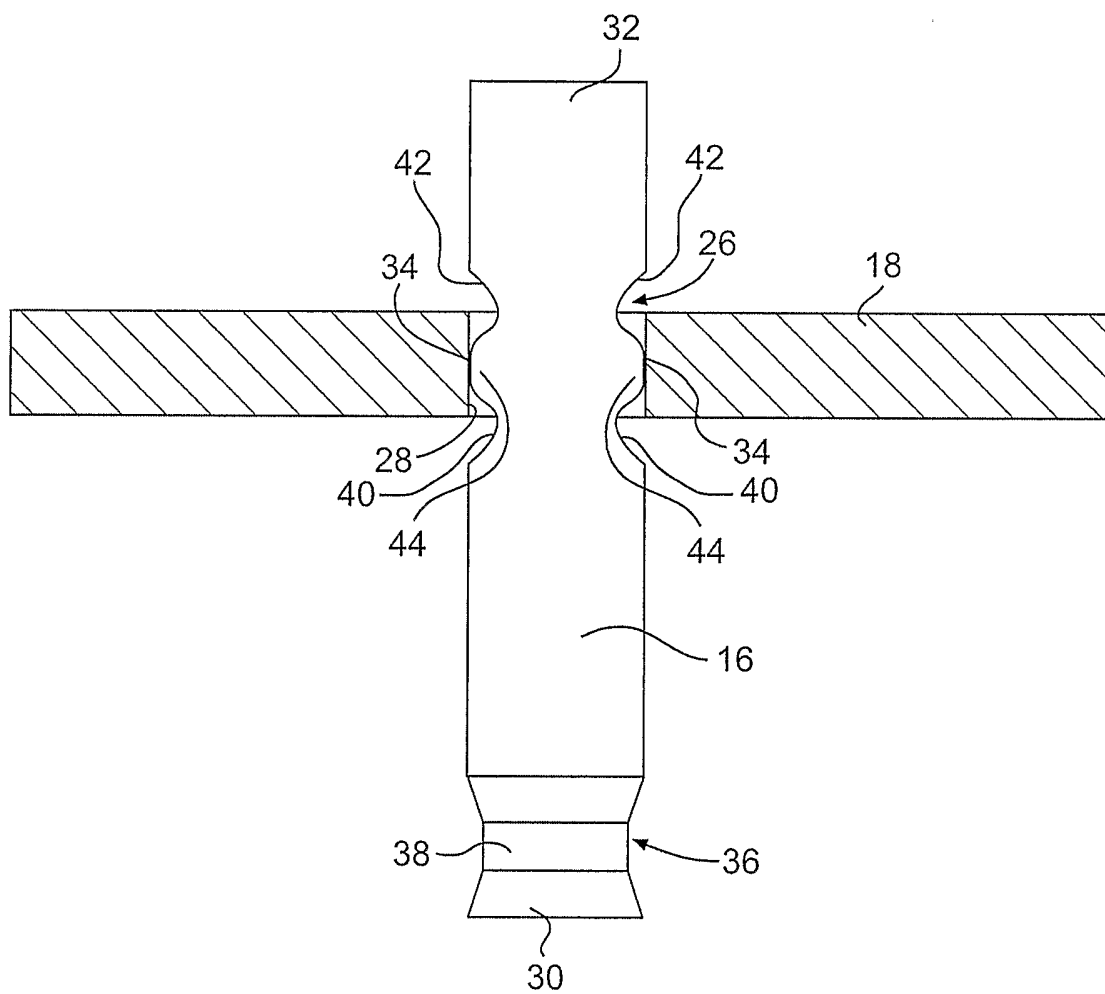
-13-

6. A valve assembly (10), comprising:
a valve member (16) having an intermediate portion (44)
positioned adjacent at least one reduced diameter portion (40, 42); and
an armature member (18) including a bore (28) sized to receive
5 and contact the valve member (16) intermediate portion (44) and allow
substantial angular movement relative to the valve member (16).
7. The valve assembly (10) of claim 6, wherein the at least one
reduced diameter portion (40, 42) includes two reduced diameter portions (40,
42) located on opposite sides of the intermediate portion (44).
- 10 8. The valve assembly (10) of claim 7, wherein the
intermediate portion (44) is generally spherical in shape.
9. The valve assembly (10) of claim 8, wherein the
intermediate portion (44) contacts the bore (28) of the armature member (18).
- 15 10. The valve assembly (10) of claim 9, wherein the armature
member (18) is coupled to the valve member (16) using a weld.

1/7

**FIG. 1**

2/7

**FIG. 2**

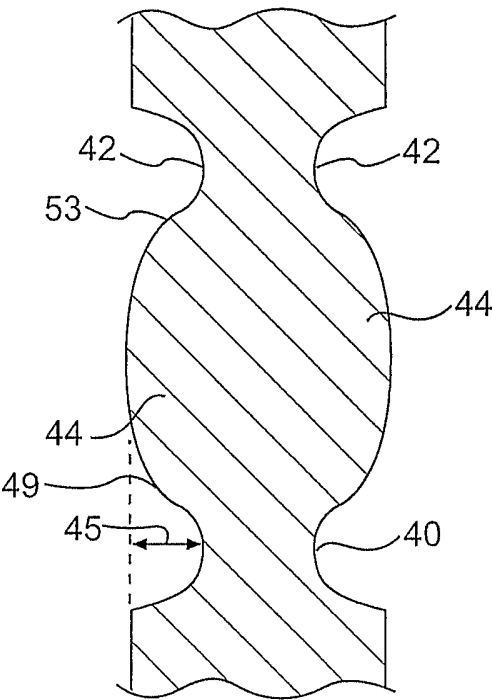


FIG. 3A

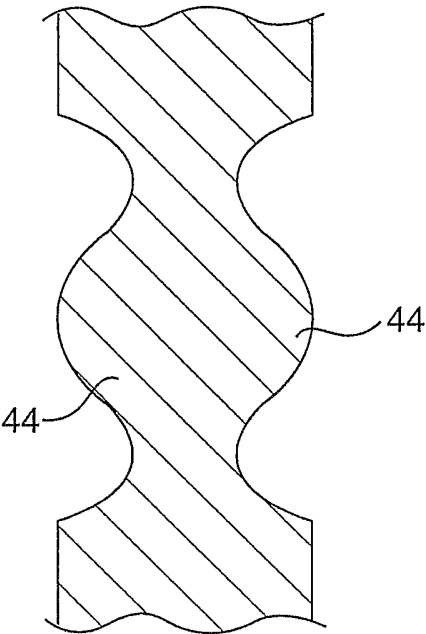


FIG. 3B

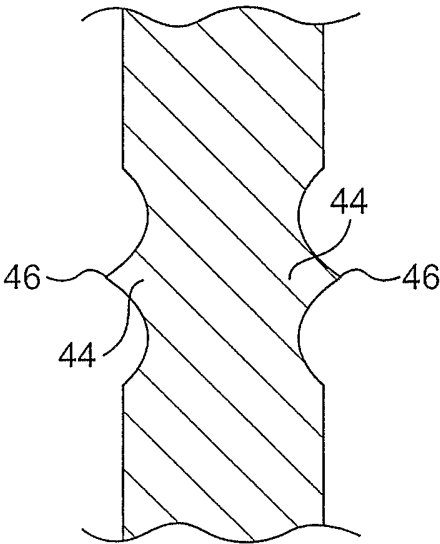


FIG. 3C

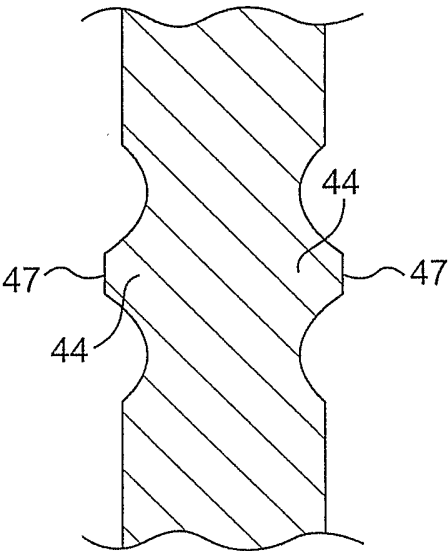


FIG. 3D

4/7

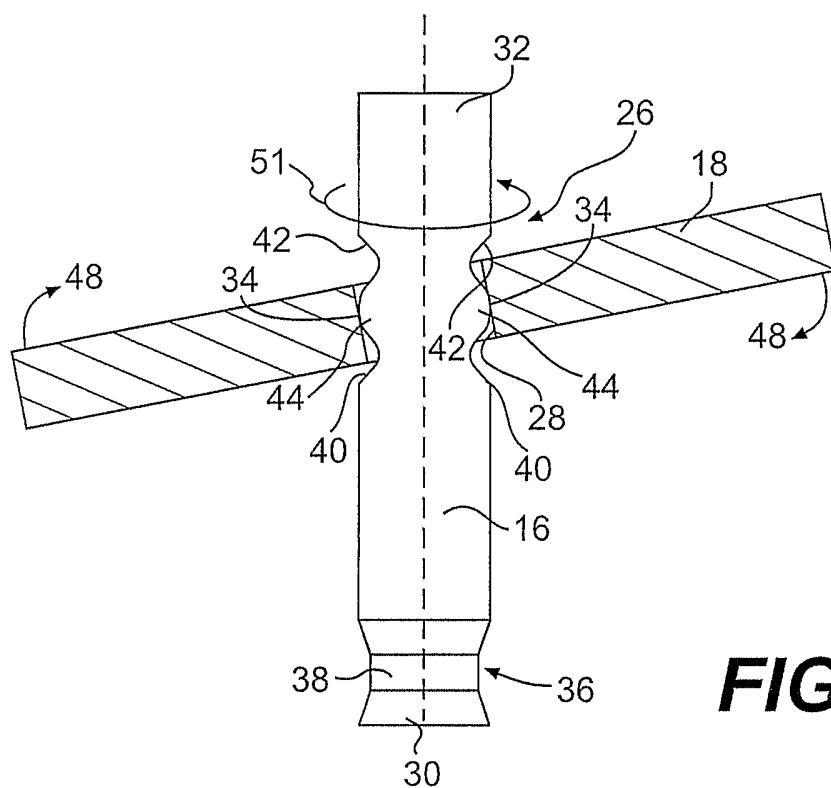


FIG. 4A

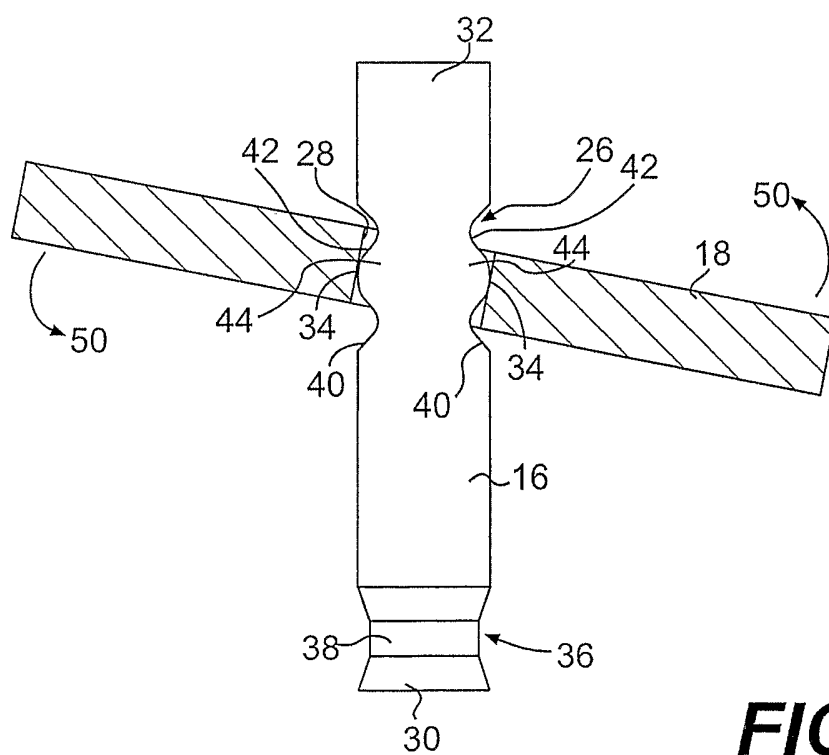
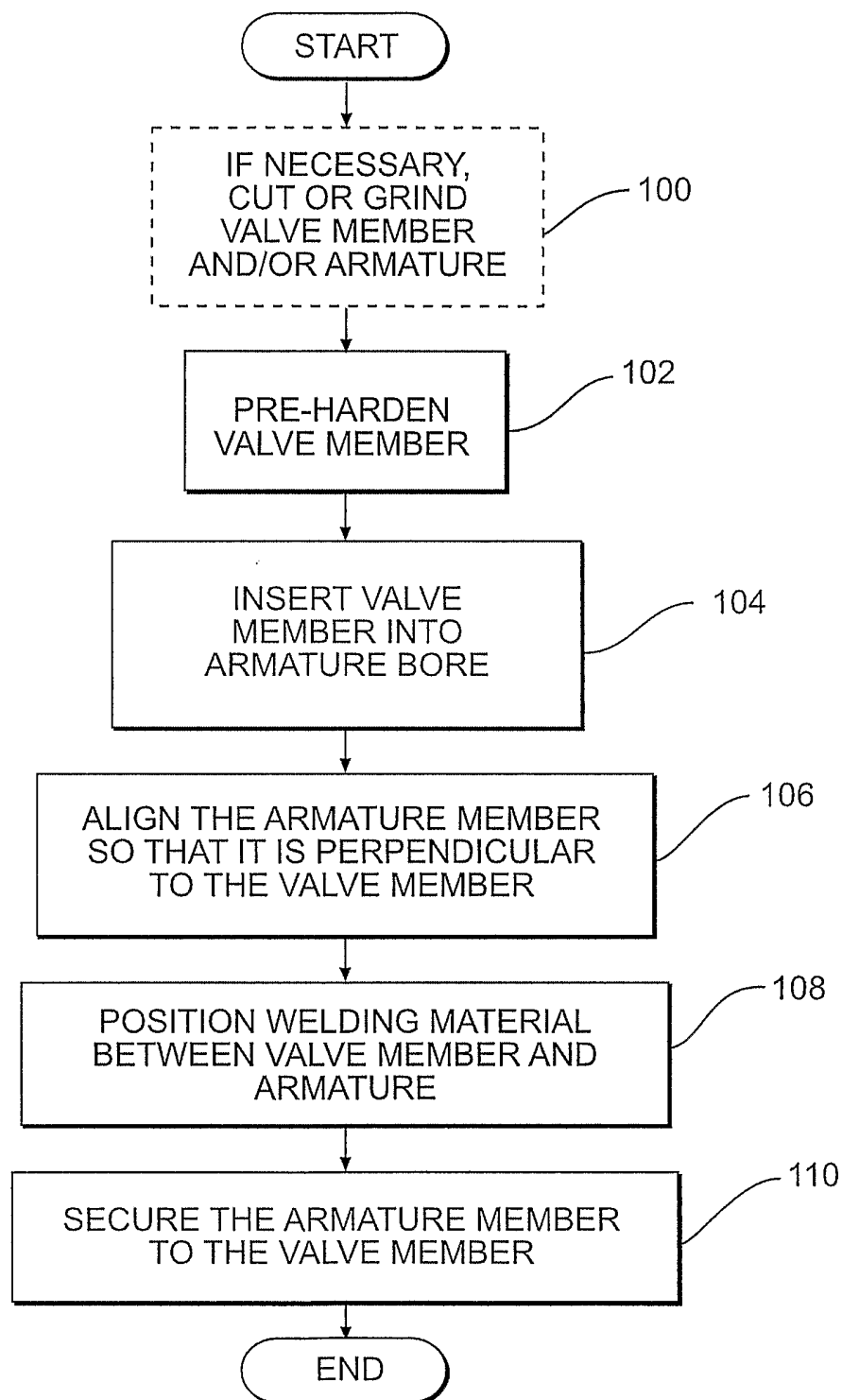


FIG. 4B

6/7

**FIG. 6**

7/7

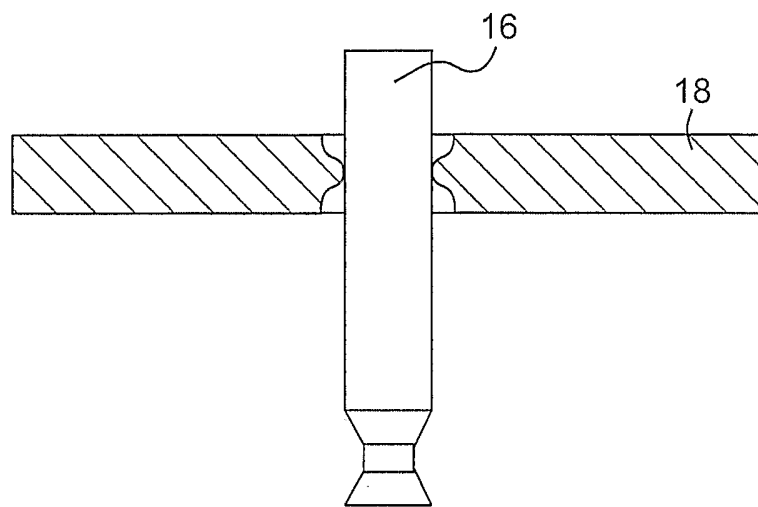


FIG. 7A

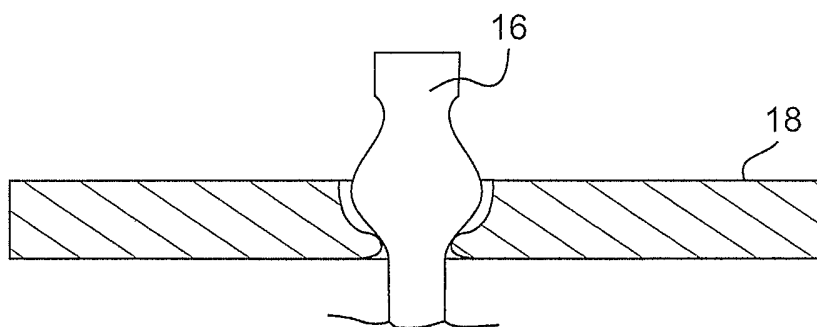


FIG. 7B

INTERNATIONAL SEARCH REPORT

International application No
PCT/US2006/004559

A. CLASSIFICATION OF SUBJECT MATTER

INV. F16K31/06

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)
F16K

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 2003/089873 A1 (MODIEN RUSSELL) 15 May 2003 (2003-05-15) paragraph [0030]; figure 1	1-4, 6, 7
X	US 6 496 092 B1 (SCHNATTERER JUERGEN ET AL) 17 December 2002 (2002-12-17) column 2, line 44 - line 54; figure 1	1-3, 6
A	US 4 852 853 A (TOSHIO ET AL) 1 August 1989 (1989-08-01) column 2, line 35 - line 38; figure 1	1-3, 5, 6, 8, 9

☐ Further documents are listed in the continuation of Box C.

☒ See patent family annex.

* Special categories of cited documents :

- *A* document defining the general state of the art which is not considered to be of particular relevance
- *E* earlier document but published on or after the international filing date
- *L* document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)
- *O* document referring to an oral disclosure, use, exhibition or other means
- *P* document published prior to the international filing date but later than the priority date claimed

- *T* later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention
- *X* document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone
- *Y* document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art.
- *&* document member of the same patent family

Date of the actual completion of the international search

2 June 2006

Date of mailing of the international search report

12/06/2006

Name and mailing address of the ISA/

European Patent Office, P.B. 5818 Patentlaan 2
NL - 2280 HV Rijswijk
Tel. (+31-70) 340-2040, Tx. 31 651 epo nl,
Fax: (+31-70) 340-3016

Authorized officer

Christensen, J

INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

PCT/US2006/004559

Patent document cited in search report		Publication date	Patent family member(s)		Publication date
US 2003089873	A1	15-05-2003	NONE		
US 6496092	B1	17-12-2002	DE	29905883 U1	17-06-1999
			WO	0060261 A1	12-10-2000
			EP	1082568 A1	14-03-2001
			JP	2002541402 T	03-12-2002
US 4852853	A	01-08-1989	JP	63180783 U	22-11-1988